

Description

This invention relates generally to a multi wire dot printer print head and more particularly to an improved print wire driving armature construction and pivotal support for use in such a print head.

Multi wire dot matrix wire printers have been in use for some years and often have print heads which include a circular arrangement of electromagnets each of which is selectively energised to attract a respective cylindrical movable core mounted intermediate opposite ends of a respective print wire actuator lever. The magnetic gap between the fixed core and the movable core is small but provides a longer stroke to the print wire because of the mechanical advantage provided by the pivoting actuator lever. Examples of this general type of dot wire printer print head are disclosed in United States Patents Nos. A—3,770,092; A—3,892,175; and A—4,244,658.

The actuator levers of the prior art type of dot wire printer print head are pivotally supported at their outer ends and extend inwardly across electromagnetic coils with the inner ends of the actuator levers engaging the ends of the print wires and moving them towards a print position when the corresponding electromagnetic actuators are energised. When the cylindrical movable core moves into and out of the electromagnetic coil, it follows an arcuate path of movement in the cylindrical bore of the electromagnetic coil. In prior art types of dot printer print head, the pivotal support of the actuator lever is provided in a plane which is normal to the cylindrical bore and positioned above the open end of the cylindrical bore in the electromagnetic actuator. As a result a substantially large circumferential clearance must be provided between the outer periphery of the cylindrical movable core and the inner periphery of the cylindrical bore to ensure that the core does not make contact with the inner surface of the cylindrical bore during its movement. This large clearance reduces the electromagnetic efficiency of the print actuator. This reduced efficiency in turn tends to adversely affect the size of the print head, the energy it requires, and the heat it generates in operation.

The object of the present invention is to provide a multi-wire dot print head having an improved print wire driving armature and pivot support means.

According to the present invention a multi-wire dot print head comprises a frame, a plurality of print wires supported for longitudinal sliding movement in the frame, and a wire driving armature associated with each of the print wires and including a pivotally mounted actuator lever having an outer end, an inner end engaging the print wire, and a cylindrical core mounted intermediate the ends of the actuator lever. The print head also includes an electromagnetic actuator associated with each of the armatures for imparting pivotal movement thereto to thereby move the associated print wire. Each electromagnetic actuator has a cylindrical bore that is open at one

end thereof for receiving the associated cylindrical core therein and pivot support means cooperating with the outer end of the associated actuator lever and defining a fulcrum for the pivotal movement of the associated actuator lever. Each pivot support means lies in a plane normal to the associated cylindrical bore. Such a print head is known from US—A—4244658. The print head is characterised in that the plane in which each pivot support means lies is intermediate the open end of the cylindrical bore and the position of maximum penetration of the associated cylindrical core therein.

This positioning of each pivot support means below the upper level of the open end of the cylindrical bore in the associated electromagnetic actuator reduces the amount of transverse movement of the movable core as it is drawn into and moves outwardly of the bore and thereby reduces the circumferential clearance needed to be provided between the outer periphery of the cylindrical movable core and the inner periphery of the cylindrical bore of the electromagnetic actuator. This reduced amount of clearance provides an efficient magnetic coupling with a consequent savings of energy, space and heat.

According to a preferred embodiment of the invention each pivot support means is positioned approximately one-fifth of the distance from the open end of the associated cylindrical bore to the position of maximum penetration of the associated cylindrical core in the cylindrical bore.

According to one embodiment of the invention the dot print head includes a metallic cup in which the electromagnetic actuator are supported in spaced relationship. Each actuator includes an electromagnetic coil with a fixed core at its lower end and the metallic cup forms an electromagnetic flux force yoke for constraining the magnetic field of each actuator coil and directing it in a closed circuit through the actuator lever so as to maximise the intensity of the magnetic flux through the movable core. Thus, the only gap in the magnetic flux path occurs inside the electromagnetic coil and between the inner end of the cylindrical movable core and the inner end of the fixed core in the coil, when the actuator lever is raised to the rest position. Each electromagnetic coil is wound on a bobbin. The upper rim of the metallic cup forms the pivot or fulcrum surface for the actuator arms of each of the electromagnetic actuators. The upper flange of each bobbin is provided with an integrally moulded extension surrounding the outer end of the associated actuator lever and extending outwardly over the upper rim of the metallic cup to provide a pivot constraint for the outer end of the actuator lever. The pivot constraint integral with the coil bobbin provides a close dimensional relationship between the actuator lever and the bobbin and allows the clearance between the cylindrical bore of the bobbin and the movable core to be further reduced. The downwardly extending pivot leg of the actuator lever is of substantially the same width as the width of the upper edge of the

metallic cup and the lower surface of the downwardly extending pivot leg is cut at a slight angle so that it is substantially flush with and in surface contact with the upper surface of the cup when the actuator lever is in the raised or non-print position. As a result a very efficient flux pattern or path is provided thereby permitting a relatively strong magnetic force to be produced by a relatively small electromagnetic coil.

In order that the invention may be more readily understood an embodiment will now be described with reference to the accompanying drawings, in which:

Figure 1 is an isometric view looking view looking downwardly on a multi-wire dot print head according to the present invention,

Figure 2 is an enlarged vertical sectional view of the print head illustrated in Figure 1 taken substantially along the line 2—2 in Figure 1.

Figure 3 is a fragmentary isometric view of the print head illustrated in Figure 1 showing the electromagnetic coil and actuator lever associated with one of the print wires, and

Figure 4 is an enlarged vertical sectional view through one of the electromagnetic coils and associated actuator lever of the print head illustrated in Figure 1 with the actuator lever being shown in the print position in solid lines and in the non-print position in dotted lines.

A multi-wire dot print head, as illustrated in the drawings, includes a moulded main support frame 10 with a downwardly depending support leg 10a supporting a ribbon guide 11 and upper and lower print wire guide plates 12, 13 (Figure 2). The lower ends of print wires 14 are supported for vertical sliding movement in the guides 12, 13 and the upper ends thereof extend through guide openings in an upstanding hub portion of the main frame 10. Enlarged impact heads 15 are fixedly connected to the upper ends of the wires 14 and compression springs 16 surround the upper ends of the print wires 14 and normally urge the print wires to an upper or non-print position.

A magnetic flux conducting or constraining yoke, in the form of a metallic cup 20, surrounds and is supported on the upstanding hub of the main frame 10 (Figure 4) and supports electromagnetic actuators around the peripheral surface thereof. The electromagnetic actuators each include a bobbin with an upper flange 21, a lower flange 22 and a barrel 23 having an inner cylindrical bore 24 which is open at its upper end. A fixed metallic core 25 is supported in the lower portion of the open bore 24 and its lower end is fixed in the cup 20. Wire windings form a coil 27 around the barrel 23. The coil 27 is electrically connected to operator means for energising the same through contacts 29 extending through a spacer plate 28 (Figure 2) and joined to the end 30 of a flexible cable 31.

An actuator lever guide sleeve 32 surrounds and extends upwardly from the upstanding hub of the main frame 10 and the upper portion thereof is provided with notched openings 33 in which

the inner end portions of the print wire driving armatures are positioned. The print wire driving armatures each include a radially extending magnetically permeable actuator lever 34 and a cylindrical movable core 35 mounted intermediate opposite ends of the actuator lever 34 and extending downwardly therefrom. The inner end of the actuator lever 34 engages and rests upon the enlarged drive head 15 of the print wire 14. A compression spring 38 is supported in a suitable bore in the sleeve 32 and its upper end engages and urges the inner end portion of the actuator lever 34 upwardly to the non-print or dotted line position shown in Figure 4. The outer end of the actuator lever 34 is provided with a downwardly extending pivot leg 40. The lower surface of the pivot leg 40 is cut at an angle of approximately three degrees, as indicated in Figure 4, from the horizontal and relative to the upper end of the upper surface of the cup 20 so that the pivot support or fulcrum point of the outer end of the actuator lever 34 is located at the inner edge portion of the cup 20, as indicated at 41 in Figure 4. The armature pivot support 41 is positioned normal to the cylindrical bore 24 and intermediate the level of the open end of the cylindrical bore 24 and the position of maximum penetration of the movable core 35, and is illustrated in Figure 4 as being at approximately one-fifth of this distance.

The upper flange 21 is provided with an integrally moulded extension in the form of an upwardly extending open frame 42 which is thicker than the upper flange 21 and is provided with a rectangular opening 43 for closely surrounding and confining the lower portion of the pivot leg 40 of the actuator lever 34 in direct alignment with the upper surface of the upstanding wall of the cup or yoke 20. The extension 42 and opening 43 thus forms a pivot positioning means for the pivoting of the outer end of the actuator lever 34. When the lever 34 is in the raised or non-print position, the lower surface of the pivot leg 40 closely engages the upper rim of the metal cup 20 to provide a closed circuit and an efficient flux path for the magnetic field of the electromagnetic actuator.

A finned outer housing 50 (Figures 1 and 2) closely surrounds the main frame 10 and extends around the same. The finned housing 50 is preferably formed of a heat conducting metal, such as aluminium, and provides a heat sink for dissipating heat generated by the operation of the electromagnetic actuators in the print head. A cap 51 surrounds an upstanding post 52 integral with the main frame 10 and extending upwardly from the hub thereof. After assembly, a head 53 is formed on post 52 extending into a recess in cap 51 in order to retain the cap against sleeve 32. An O-ring 54 is supported in an annular groove in the lower surface of the cap 51 and bears against the outer ends of the actuator levers 34 to resiliently maintain the vertical pivot legs 40 in firm contact with the upper surface of the cup 20. An energy absorbing stop ring 55 (Figure 2) is supported



below the cap 51 and forms an upper stop and damper for the inner ends of the actuator levers 34. To aid in transfer of heat from the coils 27 to the heat sink 50, it is preferred that a potting compound, not shown, be positioned around the coils 27. This potting compound may be poured into the cup 20 to surround the coils 27 and set in rigid or semi-rigid condition.

As best shown in Figure 4, the vertical centre lines of each moveable cylindrical core 35 and the associated fixed core 25 are axially aligned and concentric. When the coil 27 is energised, a magnetic field is produced within the coil and attracts the movable core 35 inwardly against the fixed core 25. The strength of the magnetic field is greatest when the gap between the movable core 35 and the fixed core 25 is positioned within the middle one-third of the windings of the coil 27.

The distance from the pivot or fulcrum point 41 of each actuator lever 34 to the centre of the associated movable core 35, as indicated by the dimension A in Figure 4, is approximately one-third of the overall length of the dimension from the pivot point of the actuator lever 34, as indicated by the dimension B in Figure 4. Thus a lever ratio of approximately three to one is provided to increase the displacement and thus the velocity imparted to the upper end of the print wire 14. Since the pivot or fulcrum point 41 of the actuator lever 34 is positioned in a plane normal to the cylindrical bore 24 and intermediate the open upper end of the cylindrical bore 24 and the position of maximum penetration of the cylindrical movable core 35 as opposed to providing a pivot or fulcrum point above the upper end of the cylindrical bore 24, the amount of transverse or lateral movement of the cylindrical movable core 35 is very small when the actuator lever 34 moves between the non-print and print positions and vice versa. With such a small amount of transverse or lateral movement of the movable core 35, the amount of circumferential clearance provided between the cylindrical movable core 35 and the cylindrical bore 24 of the electromagnetic actuator can be reduced.

The multi-wire dot print head of the present invention thus includes actuator levers 34 with the pivot support point 41 at the outer end of each of the actuator levers 34 being positioned below the plane of the open end of the cylindrical bore 24 of the associated electromagnetic actuator so that the clearance provided between the cylindrical movable core 35 and the cylindrical bore 24 of the electromagnetic actuator may be reduced. This repositioning of the pivot support point for each actuator lever permits a highly efficient magnetic force to be applied to the lever and the print wire and permits a reduction of the size of the electromagnetic coil, thereby permitting a reduced consumption of energy in operating the print wire, as well as a reduced amount of heat being generated by the print head.

Claims

1. A multi-wire dot print head comprising a frame (10),
a plurality of print wires (14) supported for longitudinal sliding movement in said frame,
a wire driving armature associated with each of said print wires and including a pivotally mounted actuator lever (34) having an outer end, an inner end engaging said print wire, and a cylindrical core (35) mounted intermediate said ends, and
an electromagnetic actuator (20—24) associated with each of said armatures for imparting pivotal movement thereto to thereby move the associated print wire, each electromagnetic actuator having a cylindrical bore (24) that is open at one end thereof for receiving the associated cylindrical core therein, and pivot support means (41) cooperating with the outer end of the associated actuator lever and defining a fulcrum for the pivotal movement of the associated actuator lever, each pivot support means lying in a plane normal to the associated cylindrical bore, characterised in that the plane in which each pivot support means lies is intermediate the open end of said cylindrical bore and the position of maximum penetration of the associated cylindrical core therein.

2. A multi-wire dot print head as claimed in Claim 1 characterised in that each pivot support means is positioned approximately one-fifth of the distance from the open end of the associated cylindrical bore to the position of maximum penetration of the associated cylindrical movable core in said cylindrical bore.

3. A multi-wire dot print head as claimed in either of the preceding claims including a metal cup (20) surrounding said electromagnetic actuators and formed with an upper edge,

characterised in that each of said electromagnetic actuators comprises

a bobbin with upper and lower bobbin flanges (21, 22), the lower surface of said upper bobbin flange being located at the same level as the upper edge of said metal cup,

and an extension (42) formed integrally with said upper bobbin flange and extending over the upper edge of said cup and providing pivot support means for the associated actuator lever.

4. A multi-wire dot print head as claimed in Claim 3 characterised in that said extension includes an upwardly extending open frame forming with the upper edge of said cup said pivot support means.

5. A multi-wire dot print head as claimed in any one of the preceding claims characterised in that the outer end of each of said actuator levers includes a downwardly depending pivot leg (40) engaging in said pivot support means.

6. A multi-wire dot print head as claimed in Claim 5 as dependent on Claim 4.

characterised in that said pivot leg includes a lower surface which is cut at an angle so that said lower surface engages the upper edge of said cup when said inner end of said actuator lever is in a

below the cap 51 and forms an upper stop and damper for the inner ends of the actuator levers 34. To aid in transfer of heat from the coils 27 to the heat sink 50, it is preferred that a potting compound, not shown, be positioned around the coils 27. This potting compound may be poured into the cup 20 to surround the coils 27 and set in rigid or semi-rigid condition.

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The distance from the pivot or fulcrum point 41 of each actuator lever 34 to the centre of the associated movable core 35, as indicated by the dimension *A* in Figure 4, is approximately one-third of the overall length of the dimension from the pivot point of the actuator lever 34, as indicated by the dimension *B* in Figure 4. Thus a lever ratio of approximately three to one is provided to increase the displacement and thus the velocity imparted to the upper end of the print wire 14. Since the pivot or fulcrum point 41 of the actuator lever 34 is positioned in a plane normal to the cylindrical bore 24 and intermediate the open upper end of the cylindrical bore 24 and the position of maximum penetration of the cylindrical movable core 35 as opposed to providing a pivot or fulcrum point above the upper end of the cylindrical bore 24, the amount of transverse or lateral movement of the cylindrical movable core 35 is very small when the actuator lever 34 moves between the non-print and print positions and vice versa. With such a small amount of transverse or lateral movement of the movable core 35, the amount of circumferential clearance provided between the cylindrical movable core 35 and the cylindrical bore 24 of the electromagnetic actuator can be reduced.

The multi-wire dot print head of the present invention thus includes actuator levers 34 with the pivot support point 41 at the outer end of each of the actuator levers 34 being positioned below the plane of the open end of the cylindrical bore 24 of the associated electromagnetic actuator so that the clearance provided between the cylindrical movable core 35 and the cylindrical bore 24 of the electromagnetic actuator may be reduced. This repositioning of the pivot support point for each actuator lever permits a highly efficient magnetic force to be applied to the lever and the print wire and permits a reduction of the size of the electromagnetic coil, thereby permitting a reduced consumption of energy in operating the print wire, as well as a reduced amount of heat being generated by the print head.

Claims

1. A multi-wire dot print head comprising a frame (10),
a plurality of print wires (14) supported for longitudinal sliding movement in said frame,
a wire driving armature associated with each of said print wires and including a pivotally mounted actuator lever (34) having an outer end, an inner end engaging said print wire, and a cylindrical core (35) mounted intermediate said ends, and
an electromagnetic actuator (20—24) associated with each of said armatures for imparting pivotal movement thereto to thereby move the associated print wire, each electromagnetic actuator having a cylindrical bore (24) that is open at one end thereof for receiving the associated cylindrical core therein, and pivot support means (41) cooperating with the outer end of the associated actuator lever and defining a fulcrum for the pivotal movement of the associated actuator lever, each pivot support means lying in a plane normal to the associated cylindrical bore, characterised in that the plane in which each pivot support means lies is intermediate the open end of said cylindrical bore and the position of maximum penetration of the associated cylindrical core therein.

2. A multi-wire dot print head as claimed in Claim 1 characterised in that each pivot support means is positioned approximately one-fifth of the distance from the open end of the associated cylindrical bore to the position of maximum penetration of the associated cylindrical movable core in said cylindrical bore.

3. A multi-wire dot print head as claimed in either of the preceding claims including a metal cup (20) surrounding said electromagnetic actuators and formed with an upper edge,

characterised in that each of said electromagnetic actuators comprises
a bobbin with upper and lower bobbin flanges (21, 22), the lower surface of said upper bobbin flange being located at the same level as the upper edge of said metal cup,
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4. A multi-wire dot print head as claimed in Claim 3 characterised in that said extension includes an upwardly extending open frame forming with the upper edge of said cup said pivot support means.

5. A multi-wire dot print head as claimed in any one of the preceding claims characterised in that the outer end of each of said actuator levers includes a downwardly depending pivot leg (40) engaging in said pivot support means.

6. A multi-wire dot print head as claimed in Claim 5 as dependent on Claim 4.

characterised in that said pivot leg includes a lower surface which is cut at an angle so that said lower surface engages the upper edge of said cup when said inner end of said actuator lever is in a



raised non-print position.

Patentansprüche

1. Mehrnadel-Matrixdruckkopf mit einem Rahm (10), einer Anzahl von für eine Gleitbewegung in Längsrichtung im Rahmen gehaltenen Drucknadeln (14),

einem Nadelantriebsanker in Zuordnung zu jeder der Drucknadeln und enthaltend einen Schwenkbar angebrachten Betätigungshebel (34) mit einem äußeren Ende, einem an der Drucknadel angreifenden inneren Ende und einem zwischen den Enden angebrachten zylindrischen Kern (35), und,

in Zuordnung zu jedem der Anker, einem elektromagnetischen Steller (20—24), welcher jenem eine Schwenkbewegung erteilt, um so die zugehörige Drucknadel zu bewegen, wobei jeder elektromagnetische Steller eine an einem ihrer Enden offene zylindrische Bohrung (24) zur Aufnahme des zugehörigen zylindrischen Kerns und Schwenklagermittel (41), welche mit dem äußeren Ende des zugehörigen Betätigungshebels zusammenwirken und einen Drehpunkt für die Schwenkbewegung des zugehörigen Betätigungshebels bestimmen, aufweist, wobei jedes Schwenklagermittel in einer zur zugehörigen zylindrischen Bohrung senkrechten Ebene liegt,

dadurch gekennzeichnet, daß

die Ebene, in welcher jedes Schwenklagermittel liegt, sich zwischen dem offenen Ende der zylindrischen Bohrung und der Lage maximaler Eindringung des zugehörigen zylindrischen Kerns in diese befindet.

2. Mehrnadel-Matrixdruckkopf nach Anspruch 1, dadurch gekennzeichnet, daß jedes Schwenklagermittel ungefähr ein Fünftel des Abstandes vom offenen Ende der zugehörigen zylindrischen Bohrung zur Lage maximaler Eindringung des zugehörigen zylindrischen beweglichen Kerns in die zylindrische Bohrung liegt.

3. Mehrnadel-Matrixdruckkopf nach irgendeinem der vorstehenden Ansprüche, mit einem die elektromagnetischen Steller umgebenden und mit einem oberen Rand ausgebildeten Metallnapf (20),

dadurch gekennzeichnet, daß jeder der elektromagnetischen Steller

eine Spule mit einem oberen und unteren Spulenflansch (21, 22), wobei die untere Fläche des oberen Spulenflansches auf gleicher Höhe wie der obere Rand des Metallnapfes liegt,

und eine einstückig mit dem oberen Spulenflansch ausgebildete und sich über den oberen Rand des Napfes erstreckende und Schwenklagermittel für den zugehörigen Betätigungshebel liefernde Verlängerung (42) umfaßt.

4. Mehrnadel-Matrixdruckkopf nach Anspruch 3, dadurch gekennzeichnet, daß die Verlängerung einen sich nach oben erstreckenden offenen Rahmen enthält, welcher mit dem oberen Rand des Napfes die Schwenklagermittel bildet.

5. Mehrnadel-Matrixdruckkopf nach irgendei-

nem der vorstehenden Ansprüche, dadurch gekennzeichnet, daß das äußere Ende eines jeden der Betätigungshebel einen nach unten verlaufenden Schwenkschenkel (40) aufweist, welcher in die Schwenklagermittel eingreift.

6. Mehrnadel-Matrixdruckkopf nach Anspruch 5 in Rückbezug auf Anspruch 4,

dadurch gekennzeichnet, daß der Schwenkschenkel eine untere Fläche aufweist, welche unter einem Winkel geschnitten ist, so daß die untere Fläche am oberen Rand des Napfes angreift, wenn das innere Ende des Betätigungshebels in einer angehobenen Nicht-Druckstellung ist.

Revendications

1. Tête d'impression par points à aiguilles multiples comprenant:

une carcasse (10),

une pluralité d'aiguilles d'impression (14) montées pour décrire un mouvement de coulissement longitudinal dans ladite carcasse,

une armature d'attaque de l'aiguille associée à chacune desdites aiguilles d'impression et comprenant un levier actionneur (34) monté pivotant, qui possède une extrémité extérieure, une extrémité intérieure attaquant ladite aiguille d'impression et un noyau cylindrique (35) monté entre lesdites extrémités et

un actionneur électromagnétique (20—24) associé à chacune desdites armatures pour lui imprimer un mouvement de pivotement, pour mettre ainsi l'aiguille d'impression correspondante en mouvement, chaque actionneur électromagnétique présentant un alésage cylindrique (24) qui s'ouvre à l'une de ses extrémités pour recevoir intérieurement le noyau cylindrique correspondant, et des moyens supports d'articulation (41) qui coopèrent avec l'extrémité extérieure du levier actionneur correspondant et définissent un point d'appui pour le mouvement de pivotement du levier actionneur correspondant, chaque moyen support d'articulation se trouvant dans un plan normal à l'alésage cylindrique correspondant,

caractérisée en ce que le plan dans lequel se trouve chaque moyen support d'articulation est placé entre l'extrémité ouverte dudit alésage cylindrique et le point de pénétration maximum de noyau cylindrique correspondant dans cet alésage.

2. Tête d'impression par points à aiguilles multiples selon la revendication 1,

caractérisée en ce que

chaque moyen support d'articulation est placé à peu près à un cinquième de la distance entre l'extrémité ouverte de l'alésage cylindrique correspondant et le point de pénétration maximum du noyau mobile cylindrique correspondant dans ledit alésage cylindrique.

3. Tête d'impression par points à aiguilles multiples selon l'une quelconque des revendications précédentes, comprenant un godet métallique (20) qui entoure lesdits actionneurs électro-



magnétiques et comprend un bord supérieur, caractérisée en ce que chacun desdits actionneurs électromagnétiques comprend

un corps de bobines possédant des joues de corps de bobine supérieure et inférieure (21, 22), la surface inférieure de ladite joue supérieure du corps de bobine étant placée au même niveau que le bord supérieur du godet métallique,

et un prolongement (42) venu d'une seule pièce avec ladite joue supérieure de corps de bobine et qui se prolonge au-dessus du bord supérieur dudit godet et forme des moyens de support d'articulation pour le levier actionneur correspondant.

4. Tête d'impression par points à aiguilles multiples selon la revendication 3, caractérisée en ce que ledit prolongement comprend un cadre ouvert en saillie vers le haut qui forme avec le bord supérieur dudit godet lesdits moyens de

support d'articulation.

5. Tête d'impression par points à aiguilles multiples selon l'une quelconque des revendications précédentes, caractérisée en ce que l'extrémité extérieure de chacun desdits leviers actionneurs comprend une branche d'articulation (40) dirigée vers le bas qui est engagée dans lesdits moyens de support d'articulation.

6. Tête d'impression par points à aiguilles multiples selon la revendication 5, rattachée à la revendication 4,

caractérisée en ce que ladite branche d'articulation comprend une surface inférieure qui est taillée selon un angle, de sorte que ladite surface inférieure s'appuie sur le bord supérieur dudit godet lorsque ladite extrémité intérieure dudit levier actionneur est dans une position relevée de non impression.

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