

Description

The present invention relates to a compact printer of the kind having a housing, means mounted in said housing for printing on lines of sheet media moved through a print zone, means for moving sheets from a sheet supply region to and through said print zone and a sheet storage and supply assembly including a sheet cassette having top and bottom wall portions and side wall portions that define a sheet stack enclosure.

As computers have become less-expensive and more compact, the number of personal computer units has increased. Portable, "lap-top" computers are available that enable convenient owner transport between home and work, or on business trips. The portability of these computers raises a clear need for compatible printer units that are portable with equal convenience. For both the computer units and printer units of such portable systems, one premium feature is compactness. Whether from the viewpoint of briefcase-volume or desk footprint area occupied, the users' desire for smallness is clear.

Several compact printer units have emerged successfully into the market, see e.g. the compact printer described in my U.S. Patent No. 4,759,646, which shows the general configuration of the Diconix Model 150 printer. This printer configuration has been particularly attractive because of its smallness; however, it does not have the capability to store and feed sheets from an integral supply station.

U.S. Patent Nos. 4,763,138 and 4,783,669 describe a compact printer having a built-in sheet supply station and feeder. This printer gains compactness by combining sheet feed and transport functions in a single platen and by providing a drawer to receive and position a sheet supply. The sheet supply drawer can be moved to a position flush with the printer housing, when empty of sheets.

It is the object of the present invention to provide compact printers that incorporate the desirable features of built-in sheet storage and mechanized sheet feed, but also have enhanced compactness and capability for containing a sheet supply in the carrying disposition.

In accordance with the invention, this object is attained in that means are provided for mounting said sheet cassette to the bottom of said printer housing for rotation about an axis normal to the bottom of the printer and the cassette between: a first non-operational position wherein its side walls are approximately flush with the side walls of said printer housing; and a second operational position rotated approximately 90° from said first position with said cassette side walls centered with respect

to said print zone so that the sheet media can be directly fed from this position into the printer.

Brief Description of Drawings

The subsequent description of preferred embodiments refers to the accompanying drawings wherein:

FIG. 1 is a perspective view of one preferred printer embodiment in accord with the present invention, having certain portions removed to facilitate the illustration of other portions;

FIG. 2 is a cross-sectional schematic view illustrating components forming the sheet media feed path of the FIG. 1 printer;

FIG. 3 is a perspective view, like FIG. 1, but with its sheet cassette in the printing orientation rather than the storage and carrying orientation;

FIG. 4 is a perspective view of the sheet cassette structure of the FIG. 1 printer, detached from the printer bottom; and

FIGS. 5a, 5b and 6a, 6b are schematic diagrams useful in explaining the interface constructions of the sheet cassette and printer.

Modes for Carrying Out the Invention

The printer shown in FIG. 1 has a housing designated generally 1 with top, bottom and side walls that enclose the other printer components and whose inner surfaces can provide main frame support for the assembly of the printer components. Thus, printer platen 4 is mounted for rotation to advance print media from the bottom region of the printer, up through a print zone and out an opening in the top of the printer housing (or through an open top wall lid of the housing, not shown). A print carriage 3 is mounted on guide rail 10 to traverse the print zone 13 and print lines upon media fed therepast by platen 4. Stepper motors 5 and 6 are respectively coupled via drive transmission elements to traverse the print carriage 3 and rotate platen 4, under the control of the electronics on circuit board 2, which is located in the top front of the housing enclosure. The printer can be powered by batteries located within platen 4 as described in more detail in my U.S. Patent No. 4,759,646. The print carriage 3 is constructed to receive, position and connect an ink jet print cartridge in the manner described in more detail in U.S. Patent No. 4,736,213. A sheet supply cassette 8 constructed in accord with the present invention is shown in its carrying/storage position in FIG. 1 and will be described in more detail subsequently.

Referring now to FIG. 2, the paper feed path is shown in more detail, with the sheet supply cassette 8 rotated 90° from FIG. 1, to its sheet feed position better shown in FIG. 3. Thus, upon ac-

tuation of sheet feed roller 9, a top sheet of paper is fed from stack 15 through buckle separators 17 on side members 90 and into the print path ingress defined by sheet guides 16, formed in the base of the printer. The lead edge of the fed sheet proceeds along the dotted arrow path A (shown in FIG. 2) between the extension of guide 16 and paper guide shim 14. As can be seen in FIGS. 1 and 2, edge portions 14a of guide shim 14 extend to the nips formed between platen 4 and bail rollers 12, and the central portion 14b of the shim plate is relieved to expose the print zone 13. The sheet supply container has a force plate 31 coupled to its lower wall to urge the leading portion of a contained sheet stack toward feed roller 9. FIG. 2 also shows how continuous print media can be fed over sprocket wheels 7 and beneath platen 4 from an opening in the rear wall of the printer, along the dotted arrow path B.

Considering now the adjustable sheet cassette feature of the present invention, it can be seen, by comparing FIGS. 1 and 3, that the cassette 8 is rotatable 90° with respect to the remainder of the printer, between the storage/carrying position shown in FIG. 1 and the sheet feeding position shown in FIG. 3. In the preferred embodiment shown in those Figures, such rotatable indexing of the cassette 8 is accomplished by the spring-loaded mounting (denoted generally 29) which couples the cassette 8 to the printer housing, with the cassette top wall urged toward the bottom wall of the printer.

The detail construction of the sheet cassette 8 and its specific construction for rotation and detenting on the bottom of the printer can be seen more clearly in FIGS. 4, 5a and 5b. Thus, the cassette top wall has an attached, threaded pivot pin 11 that extends upwardly to protrude through the mounting hole in the bottom wall of the printer. Coil spring 61 is located around the portion of pin 11 that extends through the bottom hole and, during assembly, is compressed between a nut on the top of pin 11 and the bottom printer wall so as to urge the top of the container toward the bottom of the printer.

To provide for a shift between the FIG. 1 and FIG. 3 conditions by a simple rotation movement (of 90°), the locations of the cassette pin 11 and hole in the bottom of the printer wall cooperation are predeterminedly located as shown in FIGS. 6a and 6b. Also, the perimeters of cassette 8 and printer housing 1 are selected to be substantially the same size as the sheet media utilized in the printer (e.g., approximately 216 x 279mm (8-1/2 x 11 inches)). In the storage position the housing and cassette perimeters are approximately coincident, one above the other (FIG. 1 and FIG. 6a) and the pin/hole rotation axis is located so that, in the 90° rotated sheet feed position (shown in FIGS. 3 and

6b), the cassette's longitudinal sides are centered with respect to the printer width (and thus the sheet feed path of the printer).

In order to allow easy sliding movement of the cassette during its condition-shift rotation (and to provide clearance for printer base guide 16 and the sheet feed roller which extends through a central portion of the printer bottom wall), bosses 26 are formed on the top wall of the cassette. As shown in FIGS. 6a and 6b, the printer bottom wall has two indent recesses 27 that are configured to receive two of bosses 26 in detent relation, when the cassette 8 has been rotated into the proper sheet feed position. Also the printer bottom wall has a slot 28a configured to receive a key portion 28b formed on the bottom of pivot pin 11, so that the cassette, and the exposed forward portion of its sheet stack, can move toward the printer bottom when properly indexed into the sheet feed condition. This allows the top sheet of a stack in the cassette to properly engage the feed roller 9, under upward bias of force plate 31.

From the foregoing discussion, it will be appreciated that proper location of the axis of rotation between the sheet cassette and the bottom of the printer is an important aspect of the present invention. FIGS. 5a and 5b are useful to explain one mode by which one skilled in the art can construct the proper interface axis. In this embodiment the rotation of the cassette is intended to be counterclockwise (90°) from the FIG. 5a storage condition to the FIG. 5b use position and the cassette is to be centered with respect to the dimension W of the printer housing when in the use position. Thus, the position of the cassette across the width of the printer (W) in the operational mode (FIG. 5b) can be determined. For example, if $W = 279\text{mm}$ (216 x 279 paper) (8-1/2" x 11" paper), then $C_1 = C_2 = 1.25"$. As one constructional parameter, the rotation axis should lie along line L_3 shown in FIG. 5a, that runs from the intersection of C_2 on the front edge of the printer to the intersection of C_1 along, the back edge of the printer. This constructional parameter is also used when the cassette is not centered vis-a-vis the printer width W, i.e. where C_1 is not equal to C_2 .

The next step is to determine the distance D_1 that the cassette overlaps the printer footprint in the operational mode. D_1 is determined from the component layout of the feed mechanism, e.g. the location of the feed roller 9 and ingress 16. Given a dimension for D_1 , a specific point along line L_3 can be calculated for the rotation axis. Thus for the illustrated embodiment, where $C_1 = C_2$ and L_3 is therefore at 45° to the printer walls the relation $D_1 = C_2 + 2X$, can be used to calculate a distance X (see FIGS. 5b and 6b). The value for X thus calculated is the desired distance that a rotation point

(lying along line L_3) should be from the line L_2 , drawn parallel to the edge of the printer at a distance C_2 from the edge. The intersection of end of segment X and L_3 defines the point for the axis of rotation.

Similar calculations can be performed in cases where C_1 is not equal to C_2 ; however, in this instance the angle between the line L_3 and the side of the printer (e.g., say θ) will not necessarily equal 45° . In such situations the relation $D_1 = C_2 + Y \sin \theta + Y \cos \theta$ can be solved to determine a value of a segment length Y (along line L_3 from L_2 - L_3 intercept point to the rotation axis).

FIGS. 6a and 6b illustrate one preferred mode for selecting the location of the detents 27 in the bottom of the printer and the bosses 26 on the cassette. Thus, the dimensions D_2 and D_3 , as shown in FIG. 6b are chosen to provide good stability for the cassette. D_3 is of course $< D_1$. If $D_3 < X$, the bosses 26 are closer to the center of the printer than the rotation axis, in the storage mode. If $X < D_3 < 2X + C_2$ the bosses 26 would be closer than the pivot point to the side edge of the printer. As previously mentioned, where C_1 is not equal to C_2 , L_3 will not be a 45° segment; however, similar analysis can be used to determine desired positions of the bosses 26.

FIG. 4 shows how the bottom wall of cassette 8 is segmented with a leaf spring connection 32 to provide an upward bias to the exposed, forward edge of a sheet stack edge. However, other more conventional force plates comprising an internally mounted pivot plate and spring can be used. The sheet feed roller 9 is preferably activated by selective coupling to the drive for platen 4, via lever 18; however other sheet feed drive systems can be utilized, e.g., one that selectively energizes a feed roller clutch by means of a switch on the printer keyboard.

As shown best in FIG. 1, one preferred way to load paper into cassette 8 is to index the cassette to the storage carrying position (shown in that Figure), manually depress the force plate and insert the sheet stack.

As will be clear from the foregoing description, the printer and coupled cassette are best transported and stored (e.g., on a desk or shelf) in the compact condition shown in FIG. 1. When it is desired to operate in a mechanized sheet feed print mode, the cassette 8 is indexed to the FIG. 3 position. After proper start-up procedures for the printer, the sheet feed roller is actuated, e.g., by moving lever 18 and stepper motor 6 advances a top sheet in the cassette past buckler separators 15, into ingress 16, under shim 14 and into the nip of bail roller 12 and platen 4. The platen rotation and sheet feed roller rotation are synchronized to yield a common peripheral velocity to avoid jam-

ming a fed sheet between platen 4 and guide shim 14. Once the lead edge of the sheet is in the nip formed by platen 4 and roller 12, the sheet feed roller 9 is allowed to idle. Further sheet advance is effected by rotation of platen 4 as line-by-line printing is effected.

Industrial Applicability

Thus, an important industrial advantage of printers constructed according to the present invention is that they are portable in a highly compact condition with a sheet stack stored therein.

Claims

1. Compact printer of the kind having a housing (1), means (3) mounted in said housing for printing on lines of sheet media moved through a print zone (13), means for moving sheets from a sheet supply region to and through said print zone and a sheet storage and supply assembly including a sheet cassette (8) having top and bottom wall portions and side wall portions that define a sheet stack enclosure, characterized by: means (11) for mounting said sheet cassette to the bottom of said printer housing for rotation about an axis (11) normal to the bottom of the printer and the cassette between: a first non-operational position wherein its side walls are approximately flush with the side walls of said printer housing; and a second operational position rotated approximately 90° from said first position with said cassette side walls centered with respect to said print zone so that the sheet media can be directly fed from this position into the printer.
2. Compact printer defined in claim 1, characterized by means (26, 27) for detenting said sheet cassette (8) in said first and second positions.
3. Compact printer defined in claim 1, characterized in that said mounting means (11) includes means (28a, 28b) for coupling said cassette and said printer for movements: (i) between said first and said second position in directions approximately parallel to the bottom of said printer and (ii) between detented and non-detented positions in directions approximately normal to the bottom of said printer.
4. Compact printer defined in claim 3, characterized in that said mounting means includes spring means (61) for urging said cassette toward said printer bottom.



(lying along line L_3) should be from the line L_2 , drawn parallel to the edge of the printer at a distance C_2 from the edge. The intersection of end of segment X and L_3 defines the point for the axis of rotation.

Similar calculations can be performed in cases where C_1 is not equal to C_2 ; however, in this instance the angle between the line L_3 and the side of the printer (e.g., say θ) will not necessarily equal 45° . In such situations the relation $D_1 = C_2 + Y \sin \theta + Y \cos \theta$ can be solved to determine a value of a segment length Y (along line L_3 from L_2 - L_3 intercept point to the rotation axis).

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ming a fed sheet between platen 4 and guide shim 14. Once the lead edge of the sheet is in the nip formed by platen 4 and roller 12, the sheet feed roller 9 is allowed to idle. Further sheet advance is effected by rotation of platen 4 as line-by-line printing is effected.

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4. Compact printer defined in claim 3, characterized in that said mounting means includes spring means (61) for urging said cassette toward said printer bottom.



5. Compact printer defined in claim 4, characterized by means (26, 27) for detenting said cassette (8) in said first and second positions.
6. Compact printer defined in claim 5, characterized in that said detent means (26, 27) comprises cooperative recess and protrusion means.
7. Compact printer defined in claim 6, characterized in that said protrusion means (26) is constructed to displace said cassette against said spring means (61) during movement between said first and second positions.

Patentansprüche

1. Kompakter Drucker mit einem Gehäuse (1), einer in dem Gehäuse gelagerten Einrichtung (3) zum zeilenweisen Bedrucken von Blättern, die durch eine Druckzone (13) bewegt werden, Mitteln zum Transport von Blättern aus einem Vorratsbereich zu der Druckzone und durch diese hindurch und einer Blattvorrats- und -zuführeinheit, die ein Blattmagazin (8) mit einer oberen und einer unteren Wandung sowie Seitenwandungen umfaßt, die zusammen einen einen Blattstapel umgebenden Behälter bilden, gekennzeichnet durch eine Einrichtung (11), mittels der das Blattmagazin am Boden des Druckergehäuses um eine senkrecht zum Boden des Druckers verlaufende Achse (11) so drehbar gelagert ist, daß es zwischen einer ersten unwirksamen Stellung, in der seine Seitenwandungen mit den Seitenwandungen des Druckergehäuses annähernd fluchten, und einer zweiten wirksamen Stellung bewegbar ist, in der es um etwa 90° aus der ersten Stellung herausgeschwenkt ist, wobei die Seitenwandungen des Magazins gegenüber der Druckzone so zentriert sind, daß die Blätter aus dieser Stellung heraus direkt in den Drucker einführbar sind.
2. Kompakter Drucker nach Anspruch 1, dadurch gekennzeichnet, daß Mittel (26, 27) vorgesehen sind, die das Blattmagazin (8) in der ersten und in der zweiten Stellung arretieren.
3. Kompakter Drucker nach Anspruch 1, dadurch gekennzeichnet, daß die Lagereinrichtung (11) Mittel (28a, 28b) umfaßt, die das Magazin und den Drucker so miteinander verbinden, daß sie (i) zwischen der ersten und der zweiten Stellung in etwa parallel zum Boden des Druckers verlaufenden Richtungen und (ii) zwischen einer arretierten und einer nicht arretierten Stellung in etwa senkrecht zum Boden des Druck-

kers verlaufenden Richtungen bewegbar sind.

4. Kompakter Drucker nach Anspruch 3, dadurch gekennzeichnet, daß die Lagereinrichtung eine Feder (61) umfaßt, die das Magazin zum Boden des Druckers hin vorspannt.
5. Kompakter Drucker nach Anspruch 4, gekennzeichnet durch Mittel (26, 27) zum Arretieren des Magazins (8) in der ersten und in der zweiten Stellung.
6. Kompakter Drucker nach Anspruch 5, dadurch gekennzeichnet, daß die Arretiermittel (26, 27) aus einem mit einer Vertiefung zusammenwirkenden Vorsprung bestehen.
7. Kompakter Drucker nach Anspruch 6, dadurch gekennzeichnet, daß der Vorsprung (26) so ausgebildet ist, daß er das Magazin während der Bewegung zwischen der ersten und der zweiten Stellung entgegen der Kraft der Feder (61) verschiebt.

Revendications

1. Imprimante de faible encombrement, du type qui comporte un boîtier (1), un dispositif (3) monté dans le boîtier et destiné à imprimer sur des lignes de supports d'impression sous forme de feuilles, déplacés dans une zone d'impression (13), un dispositif de déplacement des feuilles d'une région de réserve de feuilles à la zone d'impression et dans celle-ci, et un ensemble de stockage et de transmission de feuilles comprenant une cassette (8) de feuilles ayant des parties de parois supérieure et inférieure et des parties de parois latérales qui délimitent une enceinte pour une pile de feuilles, caractérisée par un dispositif (11) de montage de la cassette de feuille sur le fond du boîtier de l'imprimante afin qu'elle puisse tourner autour d'un axe (11) qui est perpendiculaire au fond de l'imprimante et à la cassette, entre une première position de repos dans laquelle ces parois latérales sont pratiquement au niveau des parois latérales du boîtier d'imprimante, et une seconde position de travail obtenue par rotation d'environ 90° par rapport à la première position et dans laquelle les parois latérales de la cassette sont centrées par rapport à la zone d'impression, si bien que les supports d'impression formés par les feuilles peuvent être transmis directement de cette position à l'imprimante.
2. Imprimante de faible encombrement selon la revendication 1, caractérisée par un dispositif



(26, 27) d'encliquetage de la cassette (8) de feuilles dans la première et dans la seconde position.

3. Imprimante de faible encombrement selon la revendication 1, caractérisée en ce que le dispositif de montage (11) comporte un dispositif (28a, 28b) destiné à accoupler la cassette et l'imprimante de manière qu'elles se déplacent (i) entre la première et la seconde position en directions sensiblement parallèles au fond de l'imprimante, et (ii) entre des positions encliquetée et non encliquetée en directions approximativement perpendiculaires au fond de l'imprimante. 5 10 15
4. Imprimante de faible encombrement selon la revendication 3, caractérisée en ce que le dispositif de montage comporte un dispositif (61) à ressort destiné à repousser la cassette vers le fond de l'imprimante. 20
5. Imprimante de faible encombrement selon la revendication 4, caractérisée par un dispositif (26, 27) destiné à encliqueter la cassette (8) lorsqu'elle est dans la première position et dans la seconde position. 25
6. Imprimante de faible encombrement selon la revendication 5, caractérisée en ce que le dispositif d'encliquetage (26, 27) comporte un dispositif à cavité et saillie coopérantes. 30
7. Imprimante de faible encombrement selon la revendication 6, caractérisée en ce que le dispositif à saillie (26) a une construction telle qu'il provoque un déplacement de la cassette malgré la force antagoniste du dispositif (61) à ressort lors du déplacement entre la première et la seconde position. 35 40

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