

ACTIVE VIBRATION ATTENUATION

Field of the invention

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The invention relates to the attenuation or reduction of vibration caused when moving a mass with a motor due to a reaction force in response to the driving force. Such vibration attenuation is achievable by attempting to cancel or reduce the effects of such reaction force concurrently with the generation of a driving force to move the mass.

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Background and Prior Art

According to Newton's third law of motion, for every action, there is an equal and opposite reaction. In practical application in an industrial machine, when a motor is activated to move a mass or object, the object exerts a reaction force on the motor equal to the force generated to the object by the motor. This reaction force acts on the motor as well as on parts connected to the motor. As a result, the reaction force received by the motor, if sufficiently large, causes vibration or shock to the machine.

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Vibration and shock are undesirable and will affect machine performance, especially in the semiconductor industry where speed and precision are important. An example is when creating wire bonds to make electrical connections between electrical contacts on a semiconductor die and a leadframe using a wire bonder machine. A motor has to move an object, namely a bond head and attached ultrasonic transducer of the machine, between different predetermined electrical contacts on the die and the leadframe. To achieve higher throughput, the motor has to move the bond head at a high speed. However, reaction forces created by the driving movement acts on the motor and motor platform, causing vibration of the same. Thus, precision may be sacrificed at the expense of speed if such

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vibration is not attenuated. Conversely, to achieve higher precision, the bond head has to be driven at a lower speed.

5 A conventional solution is to separate a body of a driving motor from its platform or base. A vibration insulator is inserted between the motor body and its base. In this way, vibration forces from the motor body to the rest of the machine can be dampened by the vibration insulator. However, since the body of the motor is no longer rigidly fixed, there is a need for more critical control requirements and complicated control algorithms.

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There have also been attempts in other industries to attempt to cancel vibrational forces. An example of this is described in US Patent Number 5,231,336 for "Actuator for Active Vibration Control". An electromagnetic actuator is arranged so that a moving element undergoes reciprocal motion in
15 response to an electrical input signal. The moving element is decoupled from any mechanical load in one embodiment so as to produce reaction forces which may be used to cancel vibrational forces. However, a drawback of the device described therein is that it requires a vibration sensor to sense the vibrations of a vibrating surface before the vibrations are processed by a
20 processing unit to produce a suitable feedback. The processing unit generates a driving signal to electromagnetic coils to impart motion to an armature in such a manner as to produce reaction forces to reduce the sensed vibrations. This means that vibrations need to be sensed first, then measured, before a counteracting force is generated to reduce the vibrations.
25 This solution is not sufficiently responsive where vibration forces are continually changing, and there is not sufficient time to sense an amount of vibration before seeking to counteract such vibration.

In the light of the above disadvantages, it would be desirable to develop a
30 solution wherein the effects of vibrational forces are reduced in an efficient and substantially instantaneous manner while having a motor's main body fixed to the rest of the machine for stability.

Summary of the invention

5 It is therefore an objective of the invention to seek to provide an apparatus and method for vibration attenuation such that the effects of reaction forces caused by an object being moved are reduced substantially concurrently with the generation of such reaction or vibrational forces.

10 According to a first aspect of the invention, there is provided an apparatus for attenuating vibration when driving an object, comprising: driving means for generating a driving force to drive the object and an attenuation actuator mounted in reciprocal relationship with the driving means for generating an attenuation force substantially concurrently with the generation of the said
15 driving force; wherein the attenuation force opposes and at least partially cancels a reaction force exerted by the object in response to the driving force.

According to a second aspect of the invention, there is provided a method for attenuating vibration when driving an object, comprising the steps of:
20 generating a driving force to drive the object; and generating an attenuation force substantially concurrently with the generation of the driving force, such that the attenuation force opposes and at least partially cancels a reaction force exerted by the object in response to the driving force.

25 It will be convenient to hereinafter describe the invention in greater detail by reference to the accompanying drawings, which illustrate one embodiment of the invention. The particularity of the drawings and the related description is not to be understood as superseding the generality of the broad identification of the invention as defined by the claims.

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Brief description of the drawings

Examples of a method and apparatus for attenuation of vibration in accordance with the invention will now be described with reference to the accompanying drawings, in which:

- 5 Fig. 1 is an isometric view of an X-Y table of a wire bonder machine incorporating attenuation actuators according to a preferred embodiment of the invention;
- Fig. 2 is an isometric view of the X-Y table incorporating attenuation actuators according to another preferred embodiment of the invention;
- 10 Fig. 3 is a side view of the X-Y table looking from direction A of Fig. 2; and
- Fig. 4 is a cross-sectional view of an attenuation actuator that may be used with the preferred embodiments of the invention.

Detailed description of the preferred embodiments

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Fig. 1 is an isometric view of an X-Y table 10 of a wire bonder machine incorporating attenuation actuators 24, 26 according to a preferred embodiment of the invention. For simplicity of illustration, the rest of the wire bonder machine is not illustrated. The X-Y table 10 comprises generally of an

20 X-Y table base 12, on which is mounted an X-Y stage 14 of the X-Y table. A wire bonder bond head 16 is mounted onto the X-Y stage 14. An ultrasonic transducer 18 for ultrasonically forming wire bonds is mounted on one side of the bond head 16. Movement of the X-Y stage 14 in the X direction or axis and Y direction or axis carries the bond head 16 and ultrasonic transducer 18

25 to bonding positions on a semiconductor die and leadframe (not shown), whereby wire bonding is performed.

An X motor 20 drives the X-Y stage 14 and bond head 16 in the X direction whereas a Y motor 22 drives X-Y stage 14 and bond head 16 in the Y

30 direction. As the bond head 16 is driven in either direction by the respective motors 20, 22, it generates reaction forces in directions opposite to driving forces from the motors. Consequently, the reaction forces are transmitted through the respective motors 20, 22 to the X-Y table base 12, and the whole

X-Y table starts to vibrate. Such vibration may also be transmitted to other parts of the wire bonder machine that are connected to the X-Y table 10. The drawbacks of vibration on the wire bonding process have already been mentioned above. One way to reduce the effects of vibration is to strengthen the rigidity of the X-Y table base 12, but this is not ideal. It would be desirable to seek to at least partially cancel the effects of the reaction forces to limit vibration of the X-Y table 10.

According to the first embodiment of the invention, an X attenuation actuator 24 is attached to the X motor 20 and a Y attenuation actuator 26 is attached to the Y motor 22. The function of the attenuation actuators 24, 26 is to generate a force that opposes the reaction force acting on each of the respective motors 20, 22, that is, in a direction that is opposite to the reaction force. Ideally, the value of the attenuation force is substantially equal to the reaction force, but if that is not possible due to design requirements, the value of the attenuation force may be a portion of the reaction force instead. It is possible to generate an attenuation force that is substantially equal to the reaction force because the reaction force is equal in value to the driving force from the respective motors 20, 22, which is known or determinable. The attenuation actuator 24, 26, is thus designed to produce a force that is essentially equal to the driving force, but mechanically decoupled from the driving force.

Fig. 2 is an isometric view of the X-Y table 10 incorporating attenuation actuators 24, 26 according to another preferred embodiment of the invention. In this embodiment, the X attenuation motor 24 is attached to the X-Y table base 12 instead of to the X motor 20. This layout may be more desirable or necessary if there is insufficient space next to the X motor 20 for attachment of the X attenuation motor 24, or because of other design considerations. For comparison, the Y attenuation actuator 26 is still attached to the Y motor 26. Mechanically, it is more desirable to locate the attenuation motor 24, 26 in alignment with the driving force of the motors 20, 22, since a resulting turning moment may otherwise be present if the force of the attenuation actuator 24

and the reaction force are not aligned, as will be explained in more detail below. Nevertheless, the effect of the reaction force is still significantly reduced notwithstanding the non-alignment. This second embodiment illustrates that the attenuation actuator 24, 26 may be attached at any position
5 in relation to the X-Y table 10, as long as it is capable of producing an attenuation force equal to and in an opposite direction to a reaction force created by a driving motion of the motors 20, 22, and this force can be transmitted to the components affected by the reaction force.

- 10 Fig. 3 is a side view of the X-Y table 10 looking from direction A of Fig. 2. This view exclusively illustrates the forces acting in the X direction, but forces acting in the Y direction follow a similar principle. A driving force F is generated by the X motor 20, which creates a reaction force, F' . The X attenuation actuator 24 produces a driving force N , which in turn generates a
15 reaction or cancellation force N' in a direction opposite to that of the reaction force, F' . N' is preferably equal and opposite to F' , therefore in essence serving to cancel the effect of the reaction force.

Referring to Fig. 3, the following is a more detailed description of the various
20 forces. The driving force required to drive a mass, such as the bond head 16, in the X direction is F . Therefore, the reaction force acting on the X motor 20 is F' . The attenuation force required to cancel the effect of the reaction force F' is N' . Since the values of these forces are equivalent, $F = F' = N = N'$.

- 25 Assuming that the driving force N of the attenuation actuator is generated by a linear motor, $N = SI$, where S is the force constant of the attenuation actuator 24, 26, and I is the current flow through the linear motor of the attenuation actuator 24, 26. As the driving force, F , is known or can be determined, the attenuation force N' can be generated accordingly by providing an appropriate
30 current flow I through its motor.

Furthermore, a turning moment is created by the reaction force, which is equivalent to $F' \times L_1$. With the addition of the attenuation actuator 24, 26, the

countering turning moment provided by the attenuation force is $N' \times L_2$. In the event, since $F' = N'$, the value of the resultant turning moment is $F' \times (L_1 - L_2)$. The initial turning moment is hence reduced as compared to when there is no attenuation force. It would be appreciated that the smaller the difference
5 between L_1 and L_2 , the lower would be the turning moment. The turning moment may thus be almost eliminated if the attenuation actuator 24, 26 is attached to the respective driving motor 20, 22 as in Fig. 1.

Fig. 4 is a cross-sectional view of an attenuation actuator 24, 26 that may be
10 used with the preferred embodiments of the invention. The actuator 24, 26 has a mounting plate 30 for attaching the actuator 24, 26 to a mounting surface, such as a driving motor 20, 22 or an X-Y table base 12. Preferably, the mounting plate 30 is attached to the surface using screws (not shown) although other means of attachment are possible. Extending from the
15 mounting plate 30 is a shaft 32 with a cavity at its center to house a compression spring 44 and an extension spring 46. A moving element 34 is connected by the compression spring 44 and extension spring 46 to the mounting plate 30 and shaft 32. The compression spring 44 and extension spring 46 help to ensure that the moving element 34 is biased at an
20 equilibrium position substantially at the center of the shaft 32 so as to have enough travel distance on either side of the shaft 32 depending on the direction of a force generated.

The moving element 34 substantially covers the shaft 32, and is slidably
25 located along the shaft 32. Bushings 36 help to reduce friction due to the sliding motion. The moving element 34 further incorporates a magnet 40 whereas the shaft incorporates a coil 42. The magnet 40 and coil 42 form an electromagnetic linear motor, wherein the magnet is a movable component. Other configurations are possible, such as a moving coil linear motor. When
30 current is passed through the coil 42 within a field of magnetic flux generated by the magnet 40, an electromagnetic force, N , is created which tends to drive the magnet 40 and moving element 34 in the direction of such force. A reaction force N' is created that is equal and opposite to electromagnetic force

N, and N' will serve to counter or cancel the reaction force F' due to the driving force F of a driving motor 20, 22 (see Fig. 3). After discontinuance of the electromagnetic force N, the moving element is moved back to its equilibrium position by the compression spring 44 and extension spring 46.

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The attenuation actuator 24, 26 can be introduced as an add-on mechanism for a machine, or built as an integral part of a motor to be mounted onto the machine. Although the attenuation actuator 24, 26 is shown to comprise a linear motor by way of example, it should be appreciated that other types of
10 motors may be used if they are capable of generating a controlled force in an opposite direction to a reaction force. Some examples are pneumatic linear drivers or motors, or rotational electromagnetic motors with linear elements, such as a ball screw. The reaction force attenuation principles herein can also be implemented in other types of machines, with suitable modifications if
15 necessary. The various embodiments of the invention make it possible to cancel the effects of reaction forces significantly. Since the body of the driving motor is rigidly fixed to the machine, the need for feedback control is also obviated.

20 Another advantage of such apparatus and method is that a machine base need not be as strong as otherwise required because vibration forces are reduced. Furthermore, precision of a machine can be improved as explained above.

25 The invention described herein is susceptible to variations, modifications and/or additions other than those specifically described and it is to be understood that the invention includes all such variations, modifications and/or additions which fall within the spirit and scope of the above description.

CLAIMS:

1. An apparatus for attenuating vibration when driving an object,
comprising:
 - 5 driving means for generating a driving force to drive the object; and
an attenuation actuator mounted in reciprocal relationship with the
driving means for generating an attenuation force substantially concurrently
with the generation of the said driving force;
wherein the attenuation force opposes and at least partially cancels a
10 reaction force exerted by the object in response to the driving force.
2. An apparatus according to claim 1, wherein the attenuation actuator
comprises a motor.
- 15 3. An apparatus according to claim 2, wherein the motor is selected from
the group consisting of a pneumatic motor, linear motor and rotational motor.
4. An apparatus according to claim 3, wherein the attenuation actuator
comprises a moving magnet linear motor.
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5. An apparatus according to claim 4, wherein the attenuation actuator
comprises a moving element slidably located on a shaft of the motor.
6. An apparatus according to claim 5, wherein the moving element is
25 biased in an equilibrium position relative to the shaft by resilient means.
7. An apparatus according to claim 6, wherein the resilient means
comprises a compression spring and an extension spring connected in series.
- 30 8. An apparatus according to any one of the preceding claims, wherein
the attenuation actuator is mounted onto the driving means.

9. An apparatus according to any one of the preceding claims, wherein the attenuation actuator is mounted onto a base supporting the driving means.

5 10. An apparatus according to any one of the preceding claims, wherein the driving means comprises a linear motor.

11. An apparatus according to any one of the preceding claims, wherein the driving means drives the object along a linear axis.

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12. An apparatus according to claim 11, wherein a second driving means drives the object along another axis perpendicular to said linear axis.

13. An apparatus according to claim 12, including an attenuation actuator
15 for each of the driving means and second driving means respectively to at least partially cancel reaction forces arising both driving means.

14. An apparatus according to any one of the preceding claims, wherein the attenuation force is substantially equal to the driving force.

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15. An apparatus according to any one of the preceding claims, wherein the object is a bond head of a wire bonder machine.

16. A method for attenuating vibration when driving an object, comprising
25 the steps of:

generating a driving force to drive the object; and

generating an attenuation force substantially concurrently with the
generation of the driving force, such that the attenuation force opposes and at
least partially cancels a reaction force exerted by the object in response to the
30 driving force.

17. A method according to claim 16, wherein the attenuation force is generated by a motor.

18. A method according to claim 17, wherein the motor is selected from the group consisting of a pneumatic motor, linear motor and rotational motor.

5 19. A method according to claim 18, wherein the motor comprises a moving magnet linear motor.

20. A method according to claim 19, wherein the motor comprises a moving element slidably located on a shaft of the motor.

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21. A method according to claim 20, including biasing the moving element in an equilibrium position relative to the shaft by using resilient means.

22. A method according to claim 21, wherein the resilient means comprises
15 a compression spring and an extension spring connected in series.

23. A method according to any one of claims 16 to 22 including aligning the attenuation force with the driving force.

20 24. A method according to any one of claims 16 to 23, wherein the driving force is generated by a linear motor.

25. A method according to any one of claims 16 to 24, including generating the driving force to drive the object along a linear axis.

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26. A method according to claim 25, including generating a second driving force to drive the object along another axis perpendicular to said linear axis.

27. A method according to claim 26, including a generating an attenuation
30 force for each of the driving forces respectively to at least partially cancel reaction forces arising both driving forces.

28. A method according to any one of claims 16 to 27, wherein the attenuation force is substantially equal to the driving force.
- 5 29. A method according to any one of claims 16 to 28, wherein the object is a bond head of a wire bonder machine.