

INSTANTANEOUS DISPLAY MECHANISM FOR A TIMEPIECE

FIELD OF THE INVENTION

The invention relates to the field of horology. It concerns, more specifically, a mechanism for the instantaneous display of a time indication,
5 such as a simple or annual calendar, an indication of the day of the week or the month.

BACKGROUND OF THE INVENTION

Such display mechanisms are well known to those skilled in the art.
10 They generally comprise a display member which is fitted with a tothing, mobile in rotation, and driven by a wheel set arranged for delivering a brief impulse thereto, upon the passage from one time indication to the next. The wheel set is conventionally formed of a finger that cooperates with the tothing to move the latter forward one step, a cam secured to the finger in
15 rotation, and a drive wheel which draws its drive force from a movement.

An example of this type of instantaneous display mechanism is illustrated in Figure 1. The mechanism includes, in a conventional manner, a display member 10, formed of a date ring 12 fitted with a tothing 14, mounted to move in rotation on a plate that is not shown. The display
20 member 10 is positioned angularly using a jumper spring 16. It is actuated in rotation by a wheel set 18, which is itself driven in rotation clockwise by an intermediate wheel 20, meshed with an hour wheel 22.

Wheel set 18 includes a finger 24 and a cam 26 secured in rotation to finger 24 and oriented angularly relative to finger 24. For this purpose,
25 cam 26 is fitted with a stud 28 engaged in a cut out portion 30 of finger 24. Wheel set 18 further includes a drive wheel 32, meshed with intermediate wheel 20, in which an aperture 34, in the form of an arc of a circle, is made, comprising an active end 34a and a passive end 34b. Stud 28 passes

through aperture 34, such that the assembly of finger 24 – cam 26 forms a coupling with play, wherein the finger 24 – cam 26 assembly is free to pivot at a certain angle. A lever 36 abuts on cam 26 via the action of a spring 38.

Cam 26, illustrated in Figure 2, includes a portion 26a of increasing
5 radius, a portion 26b of rapidly decreasing radius and a portion 26c of minimum radius. The various portions define, for a given point of abutment of lever 36, different angular positioning sectors of finger 24, corresponding to distinct advancement modes of the finger 24 – cam 26 assembly. The first two portions 26a and 26b define two angular sectors, respectively BA
10 and AB, delimited by two angular positions A and B of finger 24. In position A, finger 24 is oriented upstream of but released from tothing 14 and in position B, finger 24 is oriented downstream of but engaged in tothing 14. It should be noted that finger 24 is said to be “engaged” in tothing 14 if it is located on the path of tothing 14, and “released” from
15 tothing 14 if it is located outside the path of tothing 14. The last portion 26c defines a stable position of the finger 24 – cam 26 assembly, wherein finger 24 is in position B.

Over the first angular sector BA, the finger 24 – cam 26 assembly is driven in rotation by drive wheel 32, via the action of stud 28 which
20 cooperates with the active end 34a of aperture 34. Over this entire angular sector, lever 36 abuts on portion 26a, such that spring 38 is gradually strained via the effect of the increase in radius of cam 26.

When finger 24 reaches position A, the point of abutment of lever 36 passes from portion 26a to portion 26b. Via the effect of the rapid decrease
25 in radius of cam 26, the spring abruptly slackens, driving in rotation the finger 24 – cam 26 assembly over the entire second angular sector AB.

After pivoting, finger 24 is in position B and lever 36 is abutting on portion 26c, which corresponds to the minimum strain position of spring 38. The finger 24 – cam 26 assembly is in a stable position. This configuration
30 is illustrated in Figure 3. Stud 28 is now located at about two thirds of the way along aperture 34, such that drive wheel 32 stops driving the

finger 24 – cam 26 assembly during the period of time necessary for active end 34a of aperture 34 to catch up with stud 28. This period of time is typically of the order of 8 hours.

5 The abrupt passage of finger 24 from position A to position B has driven display member 10 through one step in rotation. For approximately 8 hours after this rotation of display member 10, finger 24 is locked in position B via the effect of spring 38, which holds lever 36 abutting against portion 26c. In this position, it has been observed that there is a risk of poor manipulation causing significant damage.

10 Indeed, the time indication display mechanism is generally provided with a quick correction device for driving display mechanism 10 directly in its current operating direction, namely clockwise in this case. When the user actuates the quick correction device while finger 24 is in position B, engaged in tothing 14, display member 10 drives finger 24 in rotation, via
15 tothing 14. It will be noted that this manipulation is possible since stud 28 is not stopped against the passive end 34b of aperture 34, but at a sufficient distance for finger 24 to be able to be released from tothing 14 without being blocked by drive wheel 32.

If the quick correction device is partially actuated, finger 24 may be
20 oriented in a singular position referenced S. In this position S, finger 24 is abutting on the tip of tothing 14, on which it exerts a significant force because of the action of spring 38, which holds lever 36 abutting on portion 26a of increasing radius of cam 26. Display member 10 is locked in an intermediate position, since the action of jumper spring 16, which tends
25 to make it rotate, is not enough to counter the action of spring 38, which is locking said member This configuration is illustrated in Figure 4.

Since finger 24 is positioned in singular position S, the user may wish to go backwards, if he observes that he should really have corrected the display in the opposite direction of rotation to the current operating
30 direction. Generally, quick correction devices do not allow quick correction in the opposite direction of rotation, because of the high risk of breaking the

display mechanism. For this reason, they are provided with disconnecting means or sliding gear means for switching the correction of one display mechanism to another, depending upon the direction of rotation. However, correction in the opposite direction to the operating direction is possible via
5 correction of the time. If, therefore, the user tries to correct the position of display member 10 in the opposite direction of rotation to the current operating direction, while finger 24 is immobilised in singular position S, this manipulation will inevitably lead to the breakage of finger 24 or part of tothing 14. The instantaneous display mechanism is then no longer
10 operational.

SUMMARY OF THE INVENTION

It is an object of the present invention to overcome this drawback by proposing an instantaneous display mechanism wherein the singular
15 position S is made impossible. More specifically, the invention concerns an instantaneous display mechanism for a timepiece including:

- a display member provided with a tothing, mounted so as to move in rotation, positioned using a jumper spring,
 - a drive wheel set, mounted so as to move in rotation including a
20 cam and a finger secured to the cam in rotation, the finger cooperating with the tothing to drive the display member in steps,
 - a spring cooperating at least indirectly with the cam,
- the cam being arranged to pivot the finger – cam assembly, via the unwinding action of the spring, so as to drive the display member through
25 one step.

According to the invention, the cam is also arranged for pivoting the finger – cam assembly via the unwinding action of the spring, so as to release the finger from the tothing after the finger – cam assembly has driven the display member through one step.

Owing to this feature of the cam, finger 24 can no longer be immobilised in the singular position S wherein it blocks the rotation of display member 10. As a result, even in the event of backward correction by the user, any risk of finger 24 or toothing 14 breaking is removed.

- 5 Other features and advantages of the present invention will appear more clearly from the following detailed description of an example embodiment of an instantaneous display mechanism according to the invention, this example being given purely by way of non limiting illustration with reference to the annexed drawing:

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BRIEF DESCRIPTION OF THE DRAWINGS

- Figure 1 – 4 already described show an instantaneous display mechanism. - Figure 1 – 4 already described show an instantaneous display mechanism.

- 15 - Figure 5 is a top view of a display mechanism according to the invention,

- Figure 6 is a view of a cam belonging to the display mechanism, and

- 20 - Figures 7 and 8 illustrate the operation of the display mechanism according to the invention.

Like the mechanism illustrated in Figures 1 to 4, the instantaneous display mechanism shown in Figure 5 includes a display member 10 mounted so as to move in rotation on a plate that is not shown, angularly positioned using a jumper spring 16, and actuated in rotation by a wheel set 39. Wheel set 39 is identical to wheel set 18 described above, as regards structure, but differs as regards the features of the cam 40, shown in a top view in Figure 6.

Like cam 26, cam 40 illustrated in Figure 6 includes a first portion 40a of increasing radius, a second portion 40b of rapidly decreasing

radius and a portion 40c of minimum radius. It further includes a second portion 40d of increasing radius and a second portion 40e of rapidly decreasing radius, together forming a beak 42 located between portions 40c and 40a. At the junction between portions 40e and 40a there is
5 a point 40f of minimum local radius of cam 40.

The two portions 40d and 40e define two new angular sectors, respectively BC and CD, delimited by two angular positions C and D of finger 24, and they correspond to two distinct advancement modes of the finger 24 – cam 40 assembly. In position C, finger 24 is oriented
10 downstream of position B, but still engaged in tothing 14, and in position D, it is oriented downstream of position C, but released from tothing 14. The point 40f of minimum local radius defines a second stable position of the finger 24 – cam 40 assembly, wherein finger 24 is oriented in angular position D.

15 The operation of the instantaneous display mechanism provided with cam 40 is illustrated in Figures 7 and 8.

The cycle of rotation of wheel set 39 starts at point D. Over the first angular sector DA, the driving and operation of wheel set 39 is identical to that of wheel set 18 over sector BA. The same is true for sector AB. After
20 pivoting over sector AB, finger 24 is in position B and lever 36 is abutting on portion 40c. The finger 24 – cam 40 assembly occupies its first stable position, and stud 28 is located two thirds of the way along aperture 34.

When the finger is in position B, two situations may arise.

In a first situation, which particularly interests us, the user
25 manipulates the quick correction device. Tothing 14 then drives the finger 24 – cam 40 assembly via finger 24, over the angular sector BC corresponding to cam portion 40d. Spring 38 strains via the effect of increase in radius of cam 40. Figure 7 illustrates this step of the rotation cycle of wheel set 39.

If the user stops his manipulation before finger 24a has reached or passed angular position C, the finger 24 – cam 40 assembly returns, via the action of spring 38, to its first stable position wherein finger 24 occupies angular position B.

5 Reference will now be made to Figure 8. If the user continues his manipulation, finger 24 passes position C, wherein it is still engaged in tothing 14, and lever 36 passes from portion 40d to portion 40e. The spring abruptly slackens via the effect of the decrease in radius of cam 40, and drives the finger 24 – cam 40 assembly in rotation over the entire
10 angular sector CD. After pivoting, finger 24 is in position D, released from tothing 14, and the finger 24 – cam 40 assembly occupies its second stable position. Moreover, display member 10, driven by jumper spring 16, has rotated through one step.

Owing to the second portion 40e of rapidly decreasing radius,
15 finger 24 is thus driven, without any action by the user, and without any possibility of stopping in singular position S, from an angular position C upstream of angular position S, to an angular position D downstream of singular position S. The user may then go backwards using the time correction, without any risk of breaking finger 24 or tothing 14.

20 In the second situation, the user does not manipulate the rapid correction device, and end 34a of aperture 34 catches up with stud 28 in a time period of approximately eight hours. It is then drive wheel 32 that drives the finger 24 – cam 40 assembly over angular sector BC. Then the finger 24 – cam 40 assembly pivots rapidly via the action of spring 38, over
25 angular sector CD. As finger 24 is in position D, the finger 24 – cam 40 assembly occupies its second stable position.

Stud 28 is now held slightly downstream of active end 34a of aperture 34, such that the drive wheel stops driving the finger 24 – cam 40 assembly for the time period necessary for active end 34a of aperture 34 to
30 catch up with stud 28. This time period is typically of the order of two hours.

After these two hours, the drive wheel controls the finger 24 – cam 40 assembly again, and the cycle of rotation starts again from position D.

A time indication display mechanism, free of the breakage risks that exist in known mechanisms, has thus been described. It goes without
5 saying that the present invention is not limited to the embodiment that has just been described, and that various simple alterations and variants could be envisaged by those skilled in the art, without departing from the scope of the present invention, as defined by the annexed claims.

It will be noted, in particular, that in a simplified embodiment, the first
10 and second portions of rapidly decreasing radius 40b and 40e of cam 40 are adjoining. The second portion 40d of increasing radius does not exist, and the portion of minimum radius 40c is merged with point 40f of minimum local radius.

Owing to this structure of cam 40, the finger 24 – cam 26 assembly is
15 driven in rotation by spring 38, from angular position A to angular position D, without stopping in stable position B, and without passing through another drive mode between positions B and C.

This embodiment prevents the risk of breaking finger 24 and
tooth 14, in the same way as the previously described embodiment.
20 However, it is less advantageous, since, after driving display member 10, finger 24 is no longer engaged in tooth 14. Consequently, finger 24 is no longer able to block the rotation of display member 10, and in particular to prevent the risk of several consecutive jumps.

WHAT IS CLAIMED IS

1. An instantaneous display mechanism for a timepiece, including:

- a display member fitted with a tothing mounted so as to move in rotation, positioned using a jumper spring,

- a drive wheel set mounted so as to move in rotation, including a
5 cam and a finger secured in rotation to the cam, said finger cooperating with said tothing to drive the display member by steps,

- a spring cooperating at least indirectly with said cam,

said cam being arranged for pivoting, via the slackening action of said spring, the finger – cam assembly so as to drive said display member
10 through one step,

wherein said cam is further arranged for pivoting the finger – cam assembly, via the slackening action of said spring, so as to release said finger from said tothing after the finger – cam assembly has driven said display member through one step.

15 2. The display mechanism according to claim 1, wherein said cam is further arranged for holding, via the pressure action of said spring, said finger engaged in said tothing, after the finger – cam assembly has driven said display member through one step.

20 3. The display mechanism according to claim 1 or 2, wherein said cam includes a first portion of decreasing radius arranged for causing the finger – cam assembly to pivot, via the slackening action of said spring, so as to drive said display member through one step, and a second portion of decreasing radius arranged for causing the finger – cam assembly to pivot, via the slackening action of said spring, so as to release said finger from
25 said tothing, after the finger – cam assembly has driven said display member through one step.

4. The display mechanism according to claim 3, wherein said cam further includes a portion of minimum radius inserted between said first and second portions of decreasing radius, and arranged for holding, via the pressure action of said spring, said finger engaged in said toothing, after
5 finger – cam assembly has driven said display member through one step.

5. The display mechanism according to claim 1, wherein said drive wheel set further includes a drive wheel provided with an aperture in the form of an arc of a circle, in which a stud, secured to the finger – cam assembly, is engaged, said aperture cooperating with said stud to drive the
10 finger – cam assembly in rotation or to leave said assembly free to pivot via the slackening action of said spring.

6. The display mechanism according to claim 5, wherein said cam further includes a first portion of increasing radius inserted between said second and first portions of decreasing radius, and arranged for straining
15 said spring via the rotational drive action of the drive wheel.

7. The display mechanism according to claim 5 or 6, wherein said cam further includes a second portion of increasing radius inserted between said portion of minimum radius and said second portion of decreasing radius, and arranged for straining said spring via the rotational drive action
20 of the drive wheel.

8. The display mechanism according to claim 1, wherein said first and second portions of decreasing radius are adjoining.

9. The display mechanism according to claim 8, wherein said drive wheel further includes a drive wheel provided with an aperture in the shape
25 of an arc of a circle, in which a stud, secured to the finger – cam assembly, is engaged, said aperture cooperating with said stud to drive the finger – cam assembly in rotation or to leave said assembly free via the slackening action of said spring.

10. The display mechanism according to claim 9, wherein said cam
30 further includes a portion of minimum radius arranged for holding finger

released from said tothing, via the pressure action of said spring, after the finger – cam assembly has driven said display member through one step.