

A TEMPERATURE DIFFERENTIAL ENGINE DEVICE

BACKGROUND OF THE INVENTION

Field of the Invention

The invention relates to a temperature differential engine device which
5 belongs to a type of device that transfers thermal energy into mechanical energy.

Background Information

A most widely used manner of utilizing energy by human beings is to transfer
thermal energy into mechanical energy. A traditional manner for transferring energy
is to transfer thermal energy into pressure potential energy first and then to do work
10 externally. This leads to energy losses; besides energy losses, additional fossil energy
resources will be consumed.

SUMMARY OF THE INVENTION

In order to overcome the above mentioned disadvantages associated with prior
art, the invention discloses an engine device which dispenses with the pressure rising
15 process during the course of transferring thermal energy into mechanical energy and
thus directly transfers the energy emerging from molecular thermal motions into
mechanical energy.

In addition, the invention aims to transfer thermal energy carried by fluids in
natural environment into mechanical energy efficiently by means of the engine device
20 proposed by the invention.

The aims of the invention are realized by means of the following
countermeasures.

The temperature differential engine device according to the present invention
consists of a low-boiling-point medium steam turbine 1, a heat absorber 2, a thermal-
25 insulating type low-temperature countercurrent heat exchanger 3, a circulating pump
4, and a refrigerating system 5, which are interconnected to constitute a closed
circulating system filled with a low-boiling-point medium fluid, wherein:

a. the low-boiling-point medium steam turbine 1 and the heat absorber 2
30 constitute a low-density-medium heat-absorbing working system, and the circulating
pump 4 and the refrigerating system 5 constitute a high-density-medium refrigerating-
circulating system; transverse heat transfer is realized between the fluids via the
thermal-insulating type low-temperature countercurrent heat exchanger 3;

b. the thermal-insulating type low-temperature countercurrent heat
35 exchanger 3 comprises a high-temperature end A and a low-temperature end B; the
heat exchanger walls are longitudinally heat-insulated with each other by means of
heat-insulating layers within the heat exchanger; the temperatures of the fluids
gradually change from the high-temperature end to the low-temperature end
longitudinally; and the fluids exchange heat transversely with each other via the heat
40 exchanger walls;

c. the low-boiling-point medium flows from the heat-absorbing working
system into the thermal-insulating type low-temperature countercurrent heat
exchanger 3, and flows from the high-temperature end A to the low-temperature end
B during which the medium transfers from the gaseous state to the liquid state or a
45 state in a higher density; then the medium flows through the refrigerating-circulating
system, and recirculates from the low-temperature end B to the high-temperature end
A during which the medium transfers from the liquid state or a state in a higher
density to the gaseous state; there exists a heat exchange in a counterflow manner in
the thermal-insulating type low-temperature countercurrent heat exchanger 3 between
50 the medium flows;

d. the heat absorber 2 may be placed before or after the low-boiling-point
medium steam turbine 1 so as to compensate for thermal energy consumption of the
system which is utilized to do work externally by the low-boiling-point medium steam
turbine 1; in this way system's energy maintains balanced.

55 The low-boiling-point medium steam turbine 1 is a device which outputs
mechanical energy externally, and the heat absorber 2 absorbs heat from outside.
Among other things, the thermal-insulating type low-temperature countercurrent heat
exchanger 3 mainly functions to prevent the medium which flows to the low-
temperature end B from carrying heat thereto as much as possible, so as to ensure that
60 the medium in the low-temperature zone is in liquid state or in a higher density state
and thus the workload of the refrigerating-circulating system is reduced. The
circulating pump 4 mainly functions to ensure that the medium circulates in the

system along the desired direction. Since the differential pressure of the system is small and the medium in liquid state flows at a low speed, the power consumption of the circulating pump 4 is low. The refrigerating system 5 mainly functions to dissipate the heat which is carried with the fluid, the heat introduced therein due to imperfect heat-insulation, and the heat generated during the working of circulating pump 4, such that the medium is maintained at low temperatures.

The thermal-insulating type low-temperature countercurrent heat exchanger 3 divides the system into a low-temperature portion which refrigerates and circulates and a high-temperature portion which absorbs heat and works. The temperatures of both the high-temperature and the low-temperature portions are lower than that of the outside fluid which supplies heat to the heat absorber 2. The low-temperature circumstance of the high-density-medium refrigerating-circulating system is maintained via the refrigerating system and the thermal-insulating layers thereof. The high-temperature portion consists of the low-boiling-point medium steam turbine 1 and the heat absorber 2. The heat absorber 2 absorbs heat from the outside fluid, so that the low-boiling-point medium steam turbine 1 utilizes the heat absorbed by the heat absorber 2 to do work, after which the heat absorber 2 absorbs heat from the outside fluid again, such that heat consumption of the system is compensated and thus system's energy maintains balanced.

Alternatively, the aims of the invention can also be realized by the following countermeasures.

In the temperature differential engine device according to the invention, the thermal-insulating type low-temperature countercurrent heat exchanger 3 is chosen from any one or various combinations of one or more of a plate type heat exchanger, a pipe type heat exchanger and a fin type heat exchanger.

In the temperature differential engine device according to the invention, the thermal-insulating type low-temperature countercurrent heat exchanger 3 consists of a plurality of stages along the flow direction, with thermal-insulating pad layers interposed therebetween. In this way heat transferring toward the low-temperature zone along the heat exchanger walls is prevented. The in-coming and out-going (counterflow) fluids exchange heat transversely with each other via the heat exchanger walls.

In the temperature differential engine device according to the invention, the heat absorber 2 and the low-boiling-point medium steam turbine 1 may be

consecutively assembled in multiple groups. The more the heat absorber 2 and the low-boiling-point medium steam turbine 1 are consecutively assembled in multiple groups, the more mechanical energy will be output externally, without increasing the workload of the refrigerating system 5 and the circulating pump 4 in the low-temperature portion.

In the temperature differential engine device according to the invention, the thermal-insulating type low-temperature countercurrent heat exchanger 3 may be omitted. In case that multiple groups of the heat absorber 2 and the low-boiling-point medium steam turbine 1 are serially connected in multiple groups, the thermal-insulating type low-temperature countercurrent heat exchanger 3 may also be omitted, because more mechanical work will be output externally. In this case, the refrigerating system 5 can fulfill the task of maintaining a low-temperature condition in place of the thermal-insulating type low-temperature countercurrent heat exchanger 3. The energy consumed by the refrigerating system 5 and the circulating pump 4 will be less than the sum of energy output from the plurality of steam turbines 1.

In the temperature differential engine device according to the invention, the flow speed of the medium fluid that drives the low-boiling-point medium steam turbine 1 to rotate can be adjusted by changing the diameter of the inlet pipe of the steam turbine. In this way, different technical conditions of engines with a variety of different requirements can be accommodated.

The temperature differential engine device according to the invention is adapted to be used in any circumstances in natural environment comprising fluids as heat sources, including air as heat sources and water as heat sources.

The temperature differential engine device according to the invention is adapted to be used in engines of automobiles, ships, airplanes and the engines of thermal power plants.

The solutions of the temperature differential engine device according to the invention and the application thereof disclosed as above provide prominent substantive features and a notable progress over the prior art as follows:

1. The invention provides an engine device which dispenses with the pressure rising process during the course of transferring thermal energy into mechanical energy and thus directly transfers the energy emerging from molecular thermal motions into mechanical energy;

2. The invention can transfer thermal energy carried by fluids existing in natural environment into mechanical energy efficiently; and

3. The invention provides an engine device which is not constrained by the temperate of the natural environment and the sunshine.

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BRIEF DESCRIPTION OF THE DRAWINGS

The present invention will be illustrated in more detail from the following description taken in conjunction with the accompanying drawings, in which:

Figure 1 is an explanatory view of the principle of the temperature differential engine device according to the invention, wherein the reference numbers are defined as follows:

- 1 a low-boiling-point medium steam turbine 1;
- 2 a heat absorber;
- 3 a thermal-insulating type low-temperature countercurrent heat
- 15 exchanger;
- 4 a circulating pump;
- 5 a refrigerating system.

DETAILED DESCRIPTION OF AN ILLUSTRATIVE EMBODIMENT

20 Hereinafter, the invention will be further explained with reference to the following embodiment.

The First Embodiment of the Invention

The temperature differential engine device according to the present invention consists of a low-boiling-point medium steam turbine 1, a heat absorber 2, a thermal-insulating type low-temperature countercurrent heat exchanger 3, a circulating pump

25 4, and a refrigerating system 5, which are interconnected to constitute a closed circulating system filled with a low-boiling-point medium fluid, wherein:

- a. the low-boiling-point medium steam turbine 1 and the heat absorber 2 constitute a low-density-medium heat-absorbing working system, and the circulating
- 30 pump 4 and the refrigerating system 5 constitute a high-density-medium refrigerating-

circulating system; transverse heat transfer between the fluids is realized via the thermal-insulating type low-temperature countercurrent heat exchanger 3;

b. the thermal-insulating type low-temperature countercurrent heat exchanger 3 comprises a high-temperature end A and a low-temperature end B; the heat exchanger walls are longitudinally heat-insulated with each other by means of heat-insulating layers within the heat exchanger; the temperatures of the fluids gradually change from the high-temperature end to the low-temperature end longitudinally; and the fluids exchange heat transversely with each other via the heat exchanger walls;

c. the low-boiling-point medium flows from the heat-absorbing working system into the thermal-insulating type low-temperature countercurrent heat exchanger 3, and flows from the high-temperature end A to the low-temperature end B during which the medium transfers from the gaseous state to the liquid state or a state in a higher density (liquefaction); then the medium flows through the refrigerating-circulating system, and recirculates from the low-temperature end B to the high-temperature end A during which the medium transfers from the liquid state or a state in a higher density to the gaseous state (vaporization); there exists a heat exchange in a counterflow manner in the thermal-insulating type low-temperature countercurrent heat exchanger 3 between medium flows;

d. the heat absorber 2 may be placed before or after the low-boiling-point medium steam turbine 1 so as to compensate for thermal energy consumption of the system which is utilized to do work externally by the low-boiling-point medium steam turbine 1; in this way system's energy maintains balanced.

The thermal-insulating type low-temperature countercurrent heat exchanger 3 is a fin type heat exchanger.

The thermal-insulating type low-temperature countercurrent heat exchanger 3 may consist of a plurality of stages, with heat-insulating pad layers interposed therebetween.

The heat absorber 2 and the low-boiling-point medium steam turbine 1 may be consecutively assembled in multiple groups.

The medium which is used to absorb, transfer heat, and transfer thermal energy into mechanical energy is refrigerating medium R22.

The engine device may find applications in engines of automobiles, ships, airplanes and engines of electricity generating plants.

CLAIMS

1. A temperature differential engine device, characterized in that, it consists of a low-boiling-point medium steam turbine (1), a heat absorber (2), a thermal-insulating type low-temperature countercurrent heat exchanger (3), a circulating pump (4), and a refrigerating system (5), which are interconnected to constitute a closed circulating system filled with a low-boiling-point medium fluid, wherein:

a. the low-boiling-point medium steam turbine (1) and the heat absorber (2) constitute a low-density-medium heat-absorbing working system, and the circulating pump (4) and the refrigerating system (5) constitute a high-density-medium refrigerating-circulating system; transverse heat transfer is realized through the thermal-insulating type low-temperature countercurrent heat exchanger (3);

b. the thermal-insulating type low-temperature countercurrent heat exchanger (3) comprises a high-temperature end (A) and a low-temperature end (B); the heat exchanger walls are longitudinally heat-insulated with each other by means of heat-insulating layers within the heat exchanger; the temperatures of the fluids gradually change from the high-temperature end to the low-temperature end longitudinally; and the fluids exchange heat transversely with each other via the heat exchanger walls;

c. the low-boiling-point medium flows from the heat-absorbing working system into the thermal-insulating type low-temperature countercurrent heat exchanger (3), and flows from the high-temperature end (A) to the low-temperature end (B) during which the medium transfers from the gaseous state to the liquid state or a state in a higher density; then the medium flows through the refrigerating-circulating system, and recirculates from the low-temperature end (B) to the high-temperature end (A) during which the medium transfers from the liquid state or a state in a higher density to the gaseous state; there exists a heat exchange in a counterflow manner in the thermal-insulating type low-temperature countercurrent heat exchanger (3) between the medium flows;

d. the heat absorber (2) may be placed before or after the low-boiling-point medium steam turbine (1) so as to compensate for thermal energy consumption of the system which is utilized to do work externally by the low-boiling-point medium steam turbine (1), so that system's thermal energy maintains balanced.

2. The temperature differential engine device according to claim 1, characterized in that, the thermal-insulating type low-temperature countercurrent heat exchanger (3) is chosen from any one or various combinations of one or more of a plate type heat exchanger, a pipe type heat exchanger and a fin type heat exchanger.

3. The temperature differential engine device according to claim 1, characterized in that, the thermal-insulating type low-temperature countercurrent heat exchanger (3) consists of a plurality of stages along the flow direction, with thermal-insulating pad layers interposed therebetween.

4. The temperature differential engine device according to claim 1, characterized in that, the heat absorber (2) and the low-boiling-point medium steam turbine (1) may be consecutively assembled in multiple groups.

5. The temperature differential engine device according to claim 1, characterized in that, the flow speed of the medium fluid w drives the low-boiling-point medium steam turbine (1) to rotate can be adjusted by changing the diameter of the inlet pipe of the steam turbine.

6. The use of the temperature differential engine device according to claim 1 in any circumstance in the natural environment which includes fluid as a heat source.