

IMPROVED FIXING ANCHOR

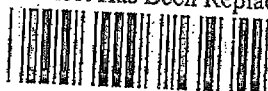
This invention relates to improvements in fixing anchors, in particular threaded fasteners for fixing into brick, blocks, masonry or the
5 like.

It is well known that with the exception of resin anchors, fixing anchors for use by the construction industry or hobbyists are generally designed to work in specific substrates such as concrete or masonry. One
10 well known method for fixing to relatively hard and/or brittle substrates such as concrete or brick or stone, is to first drill a hole in the substrate and then insert an anchor which includes an expansion mechanism. A standard screw may then be inserted within the anchor which causes the anchor to expand as it is tightened. The expansion of the anchor exerts a
15 radially directed compressive force on the walls of the pre-drilled hole which holds the anchor in place.

An alternative method of fixing into concrete is to employ a self tapping screw. However, such a fixing is suitable where only light loads
20 are to be carried by the fixing, and is not well suited to soft substrates.

According to a first aspect of the invention, a threaded fastener for anchoring into a substrate comprises a core portion having at least one
helical continuous thread provided along a substantial portion of the
25 length of the core, in which the helical thread has a substantially V-shaped cross section defining two flanks which subtend an angle of between 60° and 90° , and the thread has a helix angle of between approximately 20° and 45° .

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The helix angle between 20° and 45° allows the fastener to be inserted into the substrate with a small number of turns compared to a small helix angle of, say, 5° to 10° . The choice of 60° to 90° for the thread ensures that tensile forces applied along the axis of the fastener are
5 directed into the substrate at an angle well away from the axis providing good tensile performance in soft substrates and indeed across a wide range of substrates.

The threaded fastener is suitable for use with any substrate, but is
10 most preferably used for fastening in relatively soft substrates such as brick or blockwork, sand stone or lime stone.

Most preferably, the helix angle of the thread is 25° .

15 The two flanks of the thread may meet at an edge defining a crest. Alternatively and preferably the crest of the thread may be flattened or rounded. A flat/radius of 0.01-0.015 inches is preferred.

An advantage of the provision of only a single helical thread is that
20 it provides a reduced insertion torque when compared to the provision of a dual thread. Also, a dual thread offers a reduced distance between the threads for any given helix angle which is disadvantageous.

Preferably, the ratio of the outside diameter of the thread to the
25 pitch of the thread is substantially within the range 0.5 to 0.9. This ensures good levels of performance in soft substrates.

Preferably, the ratio of the outside diameter of the thread to the
inside diameter of the thread (i.e. the outside diameter of the core) is
30 substantially in the range 1.25 to 1.5. Thus, the diameter of the thread is



quite large compared to the diameter of the core, which ensures that the fastener remains fast within the substrate under large tensile loads.

The threaded fastener may further include a secondary balancing
5 thread which may be located in between the turns of the first helical thread. This thread can help guide the fastener when inserting into a pre-drilled hole, and may be chosen to have a diameter approximately equal to the hole diameter whilst the core is slightly smaller in diameter than the hole. The crest of the secondary thread may be flattened or rounded.

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Preferably, the core portion is substantially smooth and cylindrical over a substantial portion of its length other than the provision of the first (and optional second) helical threads. Thus, no grooves are provided at the join between the sides of the thread and the shank which could clog
15 with loose debris during installation.

The two flanks of the first and/or second threads may subtend equal angles with the core. Alternatively, the angle between one thread flank and the core and the other thread flank and the core may be different
20 provided the angle subtended by the two flanks is in the range 60° to 90°.

A bolt head or other head form (such as a stud, hook or eye) may be provided on one end of the core which has a hexagonal outline for engagement with a suitably shaped insertion tool.

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The other end of the core may finish at a point or taper, and may be provided with one or more axially extending grooves spaced around the circumference of the core. The first and/or second threads may extend to
30 a point or taper at the tip of the core to facilitate self-tapping of the

fastener into the substrate. These notches help to facilitate ease of thread cutting in harder substrates.

A method of inserting the fastener according to the first aspect into
5 a substrate may comprise pre-drilling a hole having a diameter greater than the core of the fastener but smaller than the outside diameter of the main thread and inserting the fastener into the hole. When a secondary thread is also provided, the diameter of the pre-drilled hole may be slightly smaller than or substantially equal to the outside diameter of the
10 secondary thread. This allows the second threads to act as a guide in the hole or possibly bite into the substrate.

The fastener may be fabricated from a material, i.e. steel which is then hardened. This is particularly suitable for use of the fastener in
15 relatively hard and/or abrasive substrates such as concrete. Alternatively, the fastener may be made from a plastics material, e.g. for use with aercrete substrates.

The fastener may be made by rolling to produce the thread or by
20 turning or casting. The fastener may be hardened after it is produced.

There will now be described by way of example only two embodiments of the invention with reference to the accompanying drawings in which:

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Figure 1 is an illustration of a first embodiment of a threaded fastener;

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Figure 2 is a cross sectional view of the thread profile of the threaded fastener of Figure 1;

Figure 3 is an illustration of a second embodiment of a threaded fastener;

5 Figure 4 is an end view of the fastener of Figure 3;

Figure 5 shows a flat portion provided at the crest of the thread;
and

10 Figure 6a,b,c show various possible thread cross sections.

A first embodiment of a threaded fastener is shown in Figures 1 and 2 of the accompanying drawings.

15 The fastener 1 comprises a solid elongate cylindrical core 2 having a central axis of rotation. A main thread 3 having a helical form is provided along a length of the core 2 from a point proximal a first end of the core 2 to a point proximal a second end of the core 2. A hexagonal head 5 is provided on the second end of the core to allow the fastener to
20 be rotated using a spanner or grips, whilst the other end of the core 2 tapers to a point 6.

The core and thread are formed in one piece using a rolling die, and it is preferably made from steel. The steel is then hardened to
25 produce a hard surface layer of approximately 0.008"-0.012".

The main thread 3 provided on the core has a helix angle A of between substantially 20 and 45 degrees, and preferably 25 degrees. The dimensions of the fastener are such that the ratio of the outside diameter F

of the thread to the outside diameter E of the core is between 1.25 and 1.5.

The ratio of the outside diameter of the thread F to the pitch of the thread is between 0.5 and 0.9 approximately, and the two sides 4a, 4b of the thread subtend an inclusive angle of between substantially 60° and 90° and preferably 60° approximately. This ensures that when a tensile load is applied to the fastener it causes it to lock into place in the substrate rather than fracturing the surrounding substrate. A flat portion 8 may be provided at the crest of the thread as shown in Figure 5. The flat portion 8 is preferably for use with harder substrates such as concrete with flint aggregates. In hard substrates, a sharp crest may become burred or damaged which could remove the hard outer surface causing the required insertion force to rise or failure to cut a clean thread.

In addition to the first thread 3 a second thread 7 is provided which is disposed in between the first thread 3 and is also of a helical, spaced form. The second thread 7 has an outside diameter G smaller than the outside diameter F of the first thread 3 and acts as a guide to ensure the bolt is accurately guided during insertion into a hole in the substrate. The ratio of the outside diameter of the first thread 3 to the second thread 7 is between 1.05 and 1.2 with the two sides of the second thread subtending an angle of substantially 60° .

It is envisaged that several size fasteners can be manufactured and the fastener shown in the accompanying figures has a length H of approximately 100mm, with the core diameter being between 7.5mm and 7.6mm. This allows the fastener to be inserted into a 8mm predrilled hole made using a masonry/hammer drill. Of course, larger and smaller fasteners are possible within the scope of the invention, up to 20mm core

diameter or larger being possible. Tensile loads of up to and in excess of 2 tonnes may be carried by the fixing, depending on dimensions chosen.

- 5 An alternative embodiment 100 is shown in Figures 3 and 4 of the accompanying drawings.

10 The fastener comprises a core 101, a main thread 102, a second thread 106 and a head 103 which correspond to the core 2, main thread 3 and second thread 7 and head 5 of the first embodiment. In this embodiment, a number of grooves 104 which run from a point adjacent the first end of the core 101 in an axial direction along the length of the core 101 are provided. The grooves 104 may have a U-shaped or V-shaped cross-section or other cross-section and help the fastener 100 to
15 cut a thread as opposed to forming a thread. In addition, or alternatively, the grooves may run along the length of the fastener to define a plurality of notches 107 in the thread 102. These notches 107 help to allow dust and other debris to escape as the thread cuts into the substrate. The notches may only be in the thread and not penetrate the core of the
20 fastener.

This helps to substantially eliminate any radial or axial stresses which may otherwise be imposed into the surrounding substrate.

25 Figure 6 is an illustration of the possible variations in thread cross-section which are envisaged within the scope of the invention. In Figure 6(a), the two flanks 4a,4b of the thread 3 subtend identical angles with the core 2. In Figures 6(b) and 6(c), the flanks subtend unequal angles so that the thread cross section is no longer symmetrical.

CLAIMS

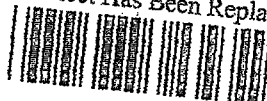
1. A threaded fastener (1) for anchoring into a substrate comprising a core portion (2) having at least one helical continuous thread (3) provided
5 along a substantial portion of the length of the core, characterised in that the helical thread (3) has a substantially V-shaped cross section defining two flanks which subtend an angle of between substantially 60° and 90° , and the thread (3) has a helix angle of between approximately 20° and 45° .
- 10 2. A threaded fastener (1) according to claim 1 characterised in that the helix angle of the thread is 25° .
3. A threaded fastener (1) according to claim 1 or claim 2
15 characterised in that the two flanks of the thread (3) meet at an edge defining a crest.
4. A threaded fastener according to claim 3 characterised in that the crest of the thread is flattened.
- 20 5. A threaded fastener according to claim 1 or claim 2 characterised in that the crest is rounded.
6. A threaded fastener according to claim 4 or claim 5 characterised
25 in that the crest has a flat/radius of 0.01 inches to 0.015 inches.
7. A threaded fastener according to any preceding claim characterised in that the ratio of the outside diameter of the thread to the pitch of the thread is substantially within the range 0.5 to 0.9.

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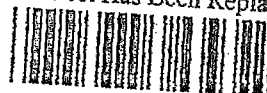


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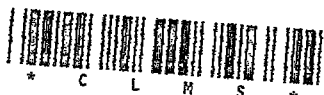


8. A threaded fastener according to any preceding claim characterised in that the ratio of the outside diameter of the thread to the inside diameter of the thread is substantially in the range 1.25 to 1.5.
- 5 9. A threaded fastener according to any preceding claim which is characterised by further including a secondary balancing thread located in between the turns of the first helical thread.
- 10 10. A threaded fastener according to claim 9 characterised in that the crest of the secondary thread is flattened or rounded.
- 15 11. A threaded fastener according to any preceding claim characterised in that the core portion is substantially smooth and cylindrical over a substantial portion of its length other than the provision of the first helical thread.
- 20 12. A threaded fastener according to any preceding claim characterised in that the two flanks of the first thread subtend equal angles with the core.
- 25 13. A threaded fastener according to claim 9 or claim 10 characterised in that the two flanks of the secondary thread subtend equal angles with the core.
14. A threaded fastener according to any preceding claim characterised in that the angle between one thread flank and the core are different whilst the angle subtended by the two flanks is in the range 60° to 90° .

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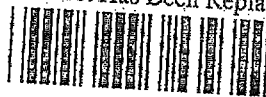


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15. A threaded fastener according to any preceding claim characterised in that a head form is provided on one end of the core which has a hexagonal outline for engagement with a suitably shaped insertion tool.
- 5 16. A threaded fastener according to claim 15 characterised in that the other end of the core finishes at a point or taper.
17. A threaded fastener according to claim 16 characterised in that the said other end of the fastener is provided with one or more axially
10 extending grooves spaced around the circumference of the core.
18. A threaded fastener according to any preceding claim characterised by being fabricated from a material which is then hardened.
- 15 19. A threaded fastener according to any one of claims 1 to 17 characterised by being made from a plastics material.
20. A method of inserting a fastener according to claims 9 or 10 into a substrate comprising the steps of pre-drilling a hole having a diameter
20 greater than the core of the fastener but smaller than the outside diameter of the main thread and inserting the fastener into the hole, the diameter of the pre-drilled hole being substantially equal to the outside diameter of the secondary thread.

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