

Selvage-knitting needle mechanism for use in a shuttleless loom.

9

The present invention relates to a selvage-knitting needle mechanism for use in a shuttleless loom for weaving a narrow fabric such as a slide fastener stringer tape.

Weaving a narrow fabric or tape on a shuttleless loom requires that the selvage-knitting needle catch a weft loop inserted through a warp shed and projected therefrom, and draw the engaged weft loop through a previous weft loop on the needle, thereby knitting a selvage along an edge of the tape being woven. To perform this task, it is necessary that the selvage-knitting needle reliably engage or catch such projected weft loops without fail.

A variety of selvage-knitting needle mechanisms have been devised to meet such requirements. For example, U.S. patent No. 3,102,557, patented September 3, 1963 discloses a selvage-knitting needle that moves upwardly as it moves rearwardly until its hook is above an inserted weft loop, which is then engaged by the hook when the needle starts moving forwardly and downwardly. The upward movement of the needle, however, causes a previous weft loop on a needle shank thereof to be pulled upwardly and enlarged excessively, with the result that the resulting selvage can be irregular in shape, loose, and unsightly. To solve such a disadvantage, a driving mechanism shown in U.S. Patent No. 3,978,895, patented September 7, 1976, enables the shank of a selvage-knitting needle to be held substantially in a vertical sense whereas the hook moves up and down for reliable weft-loop catching as the needle reciprocates back and forth. The disclosed driving mechanism, however, comprises a complex linkage which is not suitable for a high-speed operation.

According to the invention, there is provided a selvage-knitting needle mechanism for use in a shuttleless loom, comprising: a support bar; a selvage-knitting needle attached at a fixed angle to said support bar, said support bar and said needle having their longitudinal axes crossing at an intersection; rocker means for angularly driving said support bar such that the locus of said intersection substantially describes an arc of a circle; and pivot means which maintain the longitudinal axis of said support bar passing substantially through a fixed point on said circle while said support bar is being angularly moved.

The present invention seeks to provide a selvage-knitting needle mechanism for forming a tight and slightly selvage along a tape edge at a high speed.

The invention further seeks to provide a selvage-knitting needle mechanism which is relatively simple in structure.

The invention will now be described by way of example with reference to the accompanying drawings, wherein:—

Figure 1 is a perspective view of a selvage-

knitting needle mechanism according to the present invention;

Figure 2 is a side elevational view of the selvage-knitting needle mechanism shown in Figure 1;

Figure 3 is a side elevational view of a selvage-knitting needle mechanism according to a second embodiment;

Figure 4 is a side elevational view of a selvage-knitting needle mechanism according to a third embodiment;

Figure 5 is a side elevational view of a selvage-knitting needle mechanism according to a fourth embodiment; and

Figure 6 is a diagrammatic view illustrative of the principles of operation of the selvage-knitting needle mechanism of the present invention.

The principles of the present invention are particularly useful when embodied in a selvage-knitting needle mechanism such as shown in Figures 1 and 2, generally indicated by the reference numeral 10. The selvage-knitting needle mechanism 10 is incorporated in a shuttleless or ribbon loom 11 for weaving a ribbon or a narrow fabric, the shuttleless loom 11 including harnesses (not shown) for raising and lowering selected groups of parallel warp threads 12, 13 to form a succession of sheds 14 in which loops of a weft thread 15 are placed by a reciprocating weft inserter 16 thereby weaving a narrow fabric 17 for use as a slide fastener stringer tape, for example. A reed 18 beats the inserted weft loops into a fell 19 of the fabric 17 being woven.

The selvage-knitting needle mechanism 10 comprises a selvage-knitting needle 21 in the form of a latch needle having a hook 23 for catching a weft loop 22 inserted through the shed 14, and drawing the weft loop 22 through a previous weft loop which has been a shank 25 thereof and which is then cast off the needle 21. The needle 21 is attached at an angle θ to a support bar 26 by a screw 27. The support bar 26 is guided for longitudinal reciprocating movement by a pivotable guide 28 having a hole 29 through which the support bar 26 is slidably received. The guide 28 has a pin 30 pivotably mounted on a loom frame (not shown). A pivotable lever 31 is supported on a shaft 32 which is pivotably mounted on the loom frame for being angularly reciprocally moved by a suitable drive mechanism (not shown) that may be of any known structure. The lever 31 is pivotably connected at its distal end 33 to the support bar 26 by a pin 34 located substantially at the intersection of the axes of the needle 21 and support bar 26.

The parts mentioned above of the selvage-knitting needle mechanism 10 are dimensioned such that the pin 30 has an axis F located at a point on a circle C which can imagi-

10

narly be described by the axis O of the pin 34 when the latter is angularly moved by the lever 31 which is pivotably moved about the axis P of the shaft 32, whereas the needle 21 intersects the circle C at a point E adjacent to the fell 19, where selvage knitting is carried out.

When the shaft 32 is angularly moved back and forth in the directions indicated by the arrow *a* through a predetermined angle, the support bar 26 is caused to angularly move about the pin 30 in the directions of the arrow *c* and at the same time to reciprocate longitudinally through the guide 28 in the directions of the arrow *b*. The needle 21 moves with the support bar 26 back and forth along an arc while tilting up and down, and yet passes through the point E which remains vertically stationary at all times, so that weft loops knitted by the needle 21 are kept tight and firm.

A selvage-knitting needle mechanism 36 according to a second embodiment shown in Figure 3 comprises a selvage-knitting latch needle 37 attached to a support bar 40 by a screw 41. A lever 42 is supported by a pin 43 pivotably mounted on a non-illustrated loom frame and is pivotably attached by a pin 44 to an end portion 45 of the support bar 40. The support bar 40 is driven for angular movement and longitudinal reciprocation by a pivotable lever 46 mounted on a drive shaft 47 and pivotably attached to the support bar 40 by a pivot pin 48 located substantially at the intersection of the axes of the needle 37 and support bar 40. With such an arrangement, the support bar 40 is caused by angular movement of the lever 46 to angularly move and simultaneously longitudinally reciprocate as the lever 42 is angularly moved about the pin 43 in the direction of the arrow *e*, the axis of the support bar 40 passing substantially through a point F on a circle C that is imaginarily describable by the axis O of the pin 48 when angularly moving along an arc of the circle C about the shaft 47. The needle 37 is enabled to intersect the circle C at a point E which remains substantially stationary while the needle 37 reciprocates along an arc with pivotal movement of the lever 46. In this embodiment, the pin 43 is located out of the circle C.

Figure 4 illustrates a third embodiment in which a selvage-knitting needle mechanism 50 comprises a selvage-knitting latch needle 51 attached by a screw 55 to a curved support bar 54 substantially in the shape of a J having a distal end portion 49 pivotably connected by a pin 56 to a lever 57 which is pivotably connected by a pin 56 to a lever 57 which is pivotably connected by a pin 58 to a non-illustrated loom frame. A pivotable lever 59 is supported on a drive shaft 60 pivotably mounted on the loom frame and is pivotably connected by a pin 61 to the support bar 54 remotely from the needle 51. When the drive lever 59 is caused to pivot to and fro, the intersection Q of the axis of the needle 51 and the line extending through

the axes O', O'' of the pins 61, 56 substantially described an arc of a circle C to enable the needle 51 to reciprocate longitudinally through a vertically stationary point E on the circle C. The support bar 54, while being angularly moved, is allowed to move vertically by the lever 57 as the latter pivots about the pin 58. With this arrangement, the shaft 60 has an axis P' located eccentrically with respect to the circle C.

According to a fourth embodiment shown in Figure 5, a selvage-knitting needle mechanism 62 comprises a selvage-knitting latch needle 63 attached to a support bar 64 pivotably connected to a distal end thereof by a pin 66 to a lever 65 which is supported on a pin 67 pivotably mounted on a loom frame (not shown). A pivotable lever 68 is supported on a drive shaft 69 which is driven for angularly moving the lever 68 back and forth. The pivotable lever 68 is pivotably connected by a pin 70 to the support bar 64 remotely from the needle 63. Upon pivotal movement of the lever 68, the support bar 64 is angularly moved to enable the intersection of the axes of the needle 63 and support bar 64 to describe an arc of a circle C, and at the same time is caused by the lever 65 to move back and forth longitudinally. The needle 63 passes substantially through a fixed point E on the circle C during such swinging movement of the support bar 64. The pin 67 is located within the circle C according to the embodiment of Figure 5.

Figure 6 illustrates the principles of operation of the selvage-knitting needle mechanisms 10, 36, 50, and 62 shown in Figures 1 and 2, 3, 4, and 5, respectively. Given a circle I having a center K and a radius D, a line A is drawn thereacross through a fixed joint J on the circle I and a line B is also drawn across the circle I through a point L thereon, forming a vertex with the line A on the circle I, where the lines A, B form an angle θ . As the vertex moves arcuately from points 1 through 6 on the circle I, with the angle θ unchanged, the line B is caused at all times to pass through the points L. Assuming that the points L, J, K correspond to the points E, F, P, respectively, in Figures 1 and 2, and the lines A, B, D correspond to the lengths of the support bar 26 — i.e. the distance between the pins 30 and 34 — needle 21 and lever 46, respectively, the selvage-knitting needle mechanism 10 shown in Figures 1 and 2 is effective in enabling the needle 21 to pass through the fixed point E adjacent to the fell 19 of the tape 17 being woven.

Where the line A is constrained to pass successively through points 1'' through 6'' that are spaced an equal distance from the points 1 through 6 as the vertex moves from the points 1 through 6, the points 1'' through 6'' can substantially described an arc I'' of a circle having a radius G and a center K'' outside the circle I, and the line B is still caused to pass substantially through the fixed point L. Thus, also assuming

(11)

that the center K" and the radius G correspond to the axis P" and the length of the lever 42, respectively, in Figure 3, the needle 37 passes substantially through the fixed point E at all times during operation of the mechanism 36 while the axis O" of the pin 44 passes through the points 1" through 6".

It is assumed that points 1' through 6' are marked on the line A which are spaced an equal distance from the points 1 through 6, and a substantial arc I' is drawn through the points 1' through 6' which is part of a circle having a center K' which is eccentric with respect to the center K of the circle I. A line H is drawn from the center K' to the arc I' forming a vertex with the line A on the arc I'. As the vertex is advanced along the arc I' from the points 1' through 6', the above-mentioned vertex formed by the line A and B is also caused to move through the points 1 through 6, permitting the line B to be directed toward the point L on the circle I. Accordingly, the selvage-knitting needle mechanism 50 illustrated in Figure 4 can have the needle 51 to reciprocate through the fixed point E provided the length of the lever 59 corresponds substantially to the length H and the point Q corresponds substantially to the vertex by the lines A, B, and also provided the axis P' corresponds to the center K' and the point O' corresponds to the vertex by the lines H, A.

There are further marked points 1''' through 6''' on the line A which are at an equal distance from the points 1' through 6' and on a substantial arc I''' of a circle having a center K''' within the circle I. When a segment of the line A between the points 1', 1''' is shifted laterally through the points 2'—6' and the points 2'''—6''', the vertex by the lines A, B moves from the points 1 through 6, causing the line B to pass through the point L. The needle 63 shown in Figure 5 therefore will pass substantially through the fixed point E as long as the axis P''' corresponds to the point K''', and the axis O''' has a path of movement corresponding to the arc I''' and also provided the length of the lever 65 corresponds to the radius of the arc I'''.

- 10 — needle mechanism
- 11 — ribbon loom
- 12 — warp threads
- 13 — warp threads
- 14 — shed
- 15 — weft thread
- 16 — weft inserter
- 17 — fabric
- 18 — reed
- 19 — fell
- 21 — needle
- 22 — weft loop
- 23 — hook
- 25 — shank
- 26 — support bar
- 27 — screw
- 28 — guide

- 29 — hole
- 30 — pin
- 31 — lever
- 32 — shaft
- 33 — distal end
- 34 — pin
- 36 — needle mechanism
- 37 — latch needle
- 40 — support bar
- 41 — screw
- 42 — lever
- 43 — pin
- 44 — pin
- 45 — end portion
- 46 — lever
- 47 — drive shaft
- 48 — pivot pin
- 49 — end portion
- 50 — needle mechanism
- 51 — latch needle
- 54 — support bar
- 55 — screw
- 56 — pin
- 57 — lever
- 58 — pin
- 59 — lever
- 60 — drive shaft
- 61 — pin
- 62 — pin
- 62 — needle mechanism
- 63 — latch needle
- 64 — support bar
- 65 — lever
- 66 — pin
- 67 — pin
- 68 — lever
- 69 — drive shaft
- 70 — pin

Claims

1. A selvage-knitting needle mechanism for use in a shuttleless loom, comprising:

(a) a support bar (26; 40; 54; 64),

(b) a selvage-knitting needle (21; 37; 51; 63) attached at a fixed angle to said support bar (26; 40; 54; 64), said support bar and said needle having their longitudinal axes crossing at an intersection (O),

(c) rocker means (31; 46; 59; 68) for angularly driving said support bar (26; 40; 54; 64) such that the locus of said intersection (O) substantially describes an arc of a circle (C), and

(d) pivot means (28; 42; 57; 65) which maintain the longitudinal axis of said support bar (26; 40; 54; 64) passing substantially through a fixed point (F) on said circle (C) while said support bar is being angularly moved.

2. A selvage-knitting needle mechanism according to claim 1, said rocker means comprising a pivotable lever (31; 46) having a distal end thereof pivotably connected to said support bar (26; 40) at said intersection (O).

3. A selvage-knitting needle mechanism according to claim 1, said rocker means com-

(11)

that the center K" and the radius G correspond to the axis P" and the length of the lever 42, respectively, in Figure 3, the needle 37 passes substantially through the fixed point E at all times during operation of the mechanism 36 while the axis O" of the pin 44 passes through the points 1" through 6".

It is assumed that points 1' through 6' are marked on the line A which are spaced an equal distance from the points 1 through 6, and a substantial arc I' is drawn through the points 1' through 6' which is part of a circle having a center K' which is eccentric with respect to the center K of the circle I. A line H is drawn from the center K' to the arc I' forming a vertex with the line A on the arc I'. As the vertex is advanced along the arc I' from the points 1' through 6', the above-mentioned vertex formed by the line A and B is also caused to move through the points 1 through 6, permitting the line B to be directed toward the point L on the circle I. Accordingly, the selvage-knitting needle mechanism 50 illustrated in Figure 4 can have the needle 51 to reciprocate through the fixed point E provided the length of the lever 59 corresponds substantially to the length H and the point Q corresponds substantially to the vertex by the lines A, B, and also provided the axis P' corresponds to the center K' and the point O' corresponds to the vertex by the lines H, A.

There are further marked points 1''' through 6''' on the line A which are at an equal distance from the points 1' through 6' and on a substantial arc I''' of a circle having a center K''' within the circle I. When a segment of the line A between the points 1', 1''' is shifted laterally through the points 2'—6' and the points 2'''—6''', the vertex by the lines A, B moves from the points 1 through 6, causing the line B to pass through the point L. The needle 63 shown in Figure 5 therefore will pass substantially through the fixed point E as long as the axis P''' corresponds to the point K''', and the axis O''' has a path of movement corresponding to the arc I''' and also provided the length of the lever 65 corresponds to the radius of the arc I'''.

- 10 — needle mechanism
- 11 — ribbon loom
- 12 — warp threads
- 13 — warp threads
- 14 — shed
- 15 — weft thread
- 16 — weft inserter
- 17 — fabric
- 18 — reed
- 19 — fell
- 21 — needle
- 22 — weft loop
- 23 — hook
- 25 — shank
- 26 — support bar
- 27 — screw
- 28 — guide

- 29 — hole
- 30 — pin
- 31 — lever
- 32 — shaft
- 33 — distal end
- 34 — pin
- 36 — needle mechanism
- 37 — latch needle
- 40 — support bar
- 41 — screw
- 42 — lever
- 43 — pin
- 44 — pin
- 45 — end portion
- 46 — lever
- 47 — drive shaft
- 48 — pivot pin
- 49 — end portion
- 50 — needle mechanism
- 51 — latch needle
- 54 — support bar
- 55 — screw
- 56 — pin
- 57 — lever
- 58 — pin
- 59 — lever
- 60 — drive shaft
- 61 — pin
- 62 — pin
- 62 — needle mechanism
- 63 — latch needle
- 64 — support bar
- 65 — lever
- 66 — pin
- 67 — pin
- 68 — lever
- 69 — drive shaft
- 70 — pin

Claims

1. A selvage-knitting needle mechanism for use in a shuttleless loom, comprising:

(a) a support bar (26; 40; 54; 64),

(b) a selvage-knitting needle (21; 37; 51; 63) attached at a fixed angle to said support bar (26; 40; 54; 64), said support bar and said needle having their longitudinal axes crossing at an intersection (O),

(c) rocker means (31; 46; 59; 68) for angularly driving said support bar (26; 40; 54; 64) such that the locus of said intersection (O) substantially describes an arc of a circle (C), and

(d) pivot means (28; 42; 57; 65) which maintain the longitudinal axis of said support bar (26; 40; 54; 64) passing substantially through a fixed point (F) on said circle (C) while said support bar is being angularly moved.

2. A selvage-knitting needle mechanism according to claim 1, said rocker means comprising a pivotable lever (31; 46) having a distal end thereof pivotably connected to said support bar (26; 40) at said intersection (O).

3. A selvage-knitting needle mechanism according to claim 1, said rocker means com-



12

prising a lever (59; 68) pivotable about an axis (P') eccentric with respect to said circle (C) and having a distal end thereof pivotably connected to said support bar (54; 64) intermediately of the intersection (O) and the pivot means (57; 65).

4. A selvage-knitting needle mechanism according to claim 1, said pivot means comprising a guide (28) pivotable about an axis (30) located substantially on said circle (C) and having a hole (29) through which said support bar (26) extends axially movably.

5. A selvage-knitting needle mechanism according to claim 1, said pivot means comprising a lever (42; 57) angularly movable about an axis (P'') disposed outside of said circle (C) and having a distal end thereof pivotably connected to said support bar, (40; 54) at a point (O'') located outside of said circle (C).

6. A selvage-knitting needle mechanism according to claim 3, said pivot means comprising a lever (65) angularly movable about an axis (P''') disposed within said circle (C) and having a distal end thereof pivotably connected to said support bar (64) at a point (O''') located within said circle (C).

Revendications

1. Mécanisme d'aiguille de tricotage de lisière destiné à être utilisé dans un métier sans navette, comprenant:

a) une barre de support (26; 40; 54; 64);
b) une aiguille de tricotage de lisière (21; 37; 51; 63) fixée suivant un angle invariable à ladite barre de support (26; 40; 54; 64), les axes longitudinaux de ladite barre de support et de ladite aiguille se coupant en un point d'intersection (O);

c) un moyen d'oscillation (31; 46; 59; 68) pour entraîner angulairement ladite barre de support (26; 40; 54; 64) de telle sorte que le lieu géométrique de ladite intersection (O) décrit sensiblement un arc d'un cercle (C), et

d) un moyen d'articulation (28; 52; 57; 65) qui maintient le passage de l'axe longitudinal de ladite barre de support (26; 40; 54; 64) sensiblement par un point fixe (F) dudit cercle (C) pendant que ladite barre de support est déplacée angulairement.

2. Mécanisme d'aiguille de tricotage de lisière suivant la revendication 1, ledit moyen d'oscillation comprenant un levier pivotant (31; 46) dont l'extrémité libre est articulée à ladite barre de support (26; 40) à ladite intersection (O).

3. Mécanisme d'aiguille de tricotage de lisière suivant la revendication 1, ledit moyen d'oscillation comprenant un levier (59; 68) qui peut pivoter autour d'un axe géométrique (P') excentré par rapport audit cercle (C) et dont l'extrémité libre est articulée à ladite barre de support (54; 64) en un point intermédiaire entre l'intersection (O) et le moyen d'articulation (57; 65).

4. Mécanisme de tricotage de lisière suivant la revendication 1, ledit moyen d'articulation comprenant un guide (28) qui peut pivoter autour d'un axe géométrique (30) situé sensiblement sur ledit cercle (C) et qui comporte un trou (29) à travers lequel ladite barre de support (26) s'étend de façon mobile axialement.

5. Mécanisme d'aiguille de tricotage de lisière suivant la revendication 1, ledit moyen d'articulation comprenant un levier (42; 57) qui peut se déplacer angulairement autour d'un axe géométrique (P'') disposé à l'extérieur dudit cercle (C) et dont l'extrémité libre est articulée à ladite barre de support (40; 54) en un point (O'') situé à l'extérieur dudit cercle (C).

6. Mécanisme d'aiguille de tricotage de lisière suivant la revendication 3, ledit moyen d'articulation comprenant un levier (65) qui peut se déplacer angulairement autour d'un axe géométrique (P''') disposé à l'intérieur dudit cercle (C) et dont l'extrémité libre est articulée à ladite barre de support (64) en un point (O''') située à l'intérieur dudit cercle (C).

Patentansprüche

1. Webkantenstrick-Nadelmechanismus für eine Band-Webmaschine mit

a) einem Abstützarm (26; 40; 54; 64),

b) einer unter einem unveränderlichen Winkel am Abstützarm (26; 40; 54; 64) befestigten Webkantenstricknadel (21; 37; 51; 63), wobei sich die Längsachsen des Abstützarms und der Stricknadel in einem Punkt (O) schneiden,

c) einer Schwinghebeleinrichtung (31; 45; 59; 68) zum derartigen Verschwenken des Abstützarms (26; 40; 54; 64), daß der Schnittpunkt (O) im wesentlichen einen Kreisbogen (C) beschreibt, und

d) einem Gelenk (28; 42; 57; 65), das während der Verschwenkung des Abstützarmes (26; 40; 54; 64) bewirkt, daß sich dessen Längsachse im wesentlichen durch einen auf dem Kreisbogen (C) liegenden Fixpunkt (F) bewegt.

2. Webkantenstrick-Nadelmechanismus nach Anspruch 1, dadurch gekennzeichnet, daß die Schwinghebeleinrichtung einen Schwenkhebel (31; 46) aufweist, dessen freies Ende im Schnittpunkt (O) mit dem Abstützarm (26; 40) gelenkig verbunden ist.

3. Webkantenstrick-Nadelmechanismus nach Anspruch 1, dadurch gekennzeichnet, daß die Schwinghebeleinrichtung einen Schwenkhebel (59; 68) aufweist, der um eine zum Kreisbogen (C) exzentrisch angeordnete Achse (P') verschwenkbar ist und mit seinem freien Ende zwischen dem Schnittpunkt (O) und dem Gelenk (57; 65) mit dem Abstützarm (54; 64) gelenkig verbunden ist.

4. Webkantenstrick-Nadelmechanismus nach Anspruch 1, dadurch gekennzeichnet, daß das Gelenk aus einer Führung (28) besteht, die um eine im wesentlichen auf dem Kreisbogen (C)



liegende Achse (30) schwenkbar ist und eine Öffnung (29) aufweist, die der Abstützarm (26) axial beweglich durchsetzt.

4. Webkantenstrick-Nadelmechanismus nach Anspruch 1, dadurch gekennzeichnet, daß das Gelenk aus einem Hebel (42; 47) besteht, der um eine außerhalb des Kreisbogens (C) liegende Achse (P'') verschwenkbar ist und dessen freies Ende an einem außerhalb des Kreisbogens (C) liegenden Punkt (O'') mit dem Abstützarm (40;

54) gelenkig verbunden ist.

6. Webkantenstrick-Nadelmechanismus nach Anspruch 3, dadurch gekennzeichnet, daß das Gelenk aus einem Hebel (65) besteht, der um eine innerhalb des Kreisbogens (C) liegende Achse (P''') verschwenkbar ist und dessen freies Ende an einem innerhalb des Kreisbogens (C) liegenden Punkt (O''') mit dem Abstützarm (64) gelenkig verbunden ist.

10

15

20

25

30

35

40

45

50

55

60

65

6

