

HUMIDITY CONTROLLER APPARATUS

TECHNICAL FIELD

The present invention generally relates to a humidity controller apparatus and
5 specifically to an air distribution mechanism.

BACKGROUND ART

Conventionally, humidity controller apparatuses for regulating the humidity of air
by making use of an adsorbent material have been known in the prior art. One such
humidity controller apparatus is disclosed in Japanese Patent Application *Kokai*
10 Publication No. 1996-189667.

This humidity controller apparatus has an air passage through which a stream of
outdoor or room air flows and passes. And, a section of the piping of a refrigerant circuit
which performs a vapor compression refrigeration cycle by the circulation of refrigerant is
disposed in the inside of the air passage. This piping section in the air passage functions
15 as an evaporator or as a condenser. In addition, disposed on the periphery of the piping in
the air passage is a mesh member which is formed of a net-like member in which is
encapsulated an adsorbent material.

The adsorbent material of the mesh container is cooled by a flow of refrigerant
traveling through the piping section, when the piping section functions as an evaporator.
20 By the action of such cooling, moisture present in the room or outdoor air is adsorbed via
the net-like member. In addition, when the piping section functions as a condenser,
moisture adsorbed onto the adsorbent material is desorbed when heated by the refrigerant
flowing through the piping section. As a result, the adsorbent material is regenerated.

PROBLEMS THAT INVENTION INTENDS TO SOLVE

25 However, in the above-described conventional humidity controller apparatus, air

distribution systems have not been considered at all, and size reduction has been demanded. That is, it is necessary to switch the air distribution systems between the case where the piping section functions as an evaporator and the case where the piping section functions as a condenser. Especially, continuous adsorption and regeneration of the adsorbent material requires more complicated air systems.

In the above-described conventional humidity controller apparatus, however, the piping as a means for cooling and heating air and the mesh container as a moisture-adsorbing means are formed and arranged separately. Therefore, the air flow systems become more complicated, and in addition, the size of the conventional humidity controller increases.

The present invention was conceived in view of the above problems. An object of the present invention is to improve the air systems for the purpose of downsizing.

DISCLOSURE OF INVENTION

As illustrated in Figure 1, a first invention is directed to a humidity controller apparatus comprising: a first heat exchange chamber (69) for accommodating a first heat exchanger (3), an adsorbent material being supported on a surface of the first heat exchanger (3); and a second heat exchange chamber (73), formed adjacently to the first heat exchange chamber (69), for accommodating a second heat exchanger (5), an adsorbent material being supported on a surface of the second heat exchanger (5). The humidity controller apparatus further comprises a first air inflow passage (63) and a first air outflow passage (65) which are formed along one end surface in a thickness direction in which respective one surfaces of the two heat exchange chambers (69, 73) continue and which are arranged in a superimposed manner in the thickness direction of both the heat exchange chambers (69, 73). The humidity controller apparatus further comprises a second air inflow passage (57) and a second air outflow passage (59) which are formed along another

end surface which is an end surface where respective one surfaces of the two heat exchange chambers (69, 73) continue and which is situated face to face with the one end surface, and which are arranged in a superimposed manner in the thickness direction of both the heat exchange chambers (69, 73). The humidity controller apparatus further
5 comprises opening/closing means (35, ..., 47, ...) for opening and closing openings (31a, ..., 33a, ...) for communication between the first heat exchange chamber (69) and the second heat exchange chamber (73), and each inflow passage (57, 63) and each outflow passage (59, 65).

In the first invention, the distribution of air of the first heat exchange
10 chamber (69) and the second heat exchange chamber (73) is switched by opening/closing control by means of the opening/closing means (35, ..., 47, ...). As the result of this, moisture absorption and moisture release are effected in the first and second heat exchange chambers (69, 73).

The second invention provides a humidity controller apparatus according to the
15 first invention, wherein four openings (33a-33d) which allow communication between the first heat exchange chamber (69) and the second heat exchange chamber (73) and the first air inflow passage (63) and the first air outflow passage (65) are positioned in close proximity to one another in a matrix direction. Four openings (31a-31d) which allow communication between the first heat exchange chamber (69) and the second heat
20 exchange chamber (73) and the second air inflow passage (57) and the second air outflow passage (59) are positioned in close proximity to one another in a matrix direction. The eight opening/closing means (35, ..., 47, ...) are each formed by a damper.

In the second invention, the direction in which air is distributed is switched by opening/closing the eight dampers (35, ..., 47, ...). As a result, moisture absorption and
25 moisture release are realized in the first heat exchange chamber (69) and the second heat

exchange chamber (73).

The third invention provides a humidity controller apparatus according to the first invention, wherein the first air inflow passage (63) and the first air outflow passage (65) are arranged symmetrically with the second air inflow passage (57) and the second air outflow passage (59).

In the third invention, the first air inflow passage (63), the first air outflow passage (65), the second air inflow passage (57), and the second air outflow passage (59) are placed in a compact arrangement, and the air distribution resistance is decreased.

The fourth invention provides a humidity controller apparatus according to the first invention, wherein the first heat exchanger (3) and the second heat exchanger (5) are provided in a refrigerant circuit (1) which performs a vapor compression refrigeration cycle by the circulation of a refrigerant therethrough such that refrigerant condensation and evaporation alternately occur in the first heat exchanger (3) and the second heat exchanger (5).

In the fourth invention, moisture absorption and regeneration of the adsorbent material are realized by refrigerant condensation and evaporation occurring in the first heat exchanger (3) and the second heat exchanger (5).

The fifth invention provides a humidity controller apparatus according to the fourth invention, wherein dehumidifier means (80) for switching refrigerant circulation in the refrigerant circuit (1) and air distribution is provided so that moisture in a stream of air flowing through a refrigerant evaporating heat exchanger (3, 5) is adsorbed by the adsorbent material of the refrigerant evaporating heat exchanger (3, 5) while the adsorbent material of a refrigerant condensing heat exchanger (5, 3) is regenerated by the release of moisture therefrom to a stream of air flowing through the refrigerant condensing heat exchanger (5, 3), whereby the stream of air dehumidified by the adsorbent material of the

refrigerant evaporating heat exchanger (3, 5) is supplied to an indoor space.

In the fifth invention, the dehumidifier means (80) switches refrigerant circulation in the refrigerant circuit (1) and air distribution, whereby air dehumidification takes place in one heat exchanger (i.e., the refrigerant evaporating heat exchanger (3, 5)) and
5 adsorbent-material regeneration takes place in the other heat exchanger (i.e., the refrigerant condensing heat exchanger (5, 3)). As the result of this, dehumidification is carried out without interruption.

The sixth invention provides a humidity controller apparatus according to the fourth invention, wherein humidifier means (81) for switching refrigerant circulation in the
10 refrigerant circuit (1) and air distribution is provided so that moisture in a stream of air flowing through a refrigerant evaporating heat exchanger (3, 5) is adsorbed by the adsorbent material of the refrigerant evaporating heat exchanger (3, 5) while the adsorbent material of a refrigerant condensing heat exchanger (5, 3) is regenerated by the release of moisture therefrom to a stream of air flowing through the refrigerant condensing heat
15 exchanger (5, 3), whereby the stream of air humidified by the adsorbent material of the refrigerant evaporating heat exchanger (5, 3) is supplied to an indoor space.

In the sixth invention, humidifier means (81) switches refrigerant circulation in the refrigerant circuit (1) and air distribution, whereby air dehumidification takes place in one heat exchanger (i.e., the refrigerant evaporating heat exchanger (3, 5)) and air
20 humidification takes place in the other heat exchanger (i.e., the refrigerant condensing heat exchanger (5, 3)) and the adsorbent material thereof is regenerated. As the result of this, humidification is carried out without interruption.

The seventh invention provides a humidity controller apparatus according to the fourth invention, wherein dehumidifier means (80) for switching refrigerant circulation in
25 the refrigerant circuit (1) and air distribution is provided so that moisture in a stream of air

flowing through a refrigerant evaporating heat exchanger (3, 5) is adsorbed by the adsorbent material of the refrigerant evaporating heat exchanger (3, 5) while the adsorbent material of a refrigerant condensing heat exchanger (5, 3) is regenerated by the release of moisture therefrom to a stream of air flowing through the refrigerant condensing heat exchanger (5, 3), whereby the stream of air dehumidified by the adsorbent material of the refrigerant evaporating heat exchanger (3, 5) is supplied to an indoor space. Humidifier means (81) for switching refrigerant circulation in the refrigerant circuit (1) and air distribution is provided so that moisture in a stream of air flowing through a refrigerant evaporating heat exchanger (3, 5) is adsorbed by the adsorbent material of the refrigerant evaporating heat exchanger (3, 5) while the adsorbent material of a refrigerant condensing heat exchanger (5, 3) is regenerated by the release of moisture therefrom to a stream of air flowing through the refrigerant condensing heat exchanger (5, 3), whereby the stream of air humidified by the adsorbent material of the refrigerant evaporating heat exchanger (5, 3) is supplied to an indoor space. The dehumidifier means (80) and the humidifier means (81) are configured to operate switchably between a dehumidification mode of operation and a humidification mode of operation.

In the seventh invention, the dehumidifier means (80) switches refrigerant circulation in the refrigerant circuit (1) and air distribution, whereby air dehumidification takes place in one heat exchanger (i.e., the refrigerant evaporating heat exchanger (3, 5)) and adsorbent-material regeneration takes place in the other heat exchanger (i.e., the refrigerant condensing heat exchanger (5, 3)). As the result of this, dehumidification is carried out without interruption. In addition, when switched to the humidifier means (81), refrigerant circulation in the refrigerant circuit (1) and air distribution are switched. Then, air dehumidification takes place in one heat exchanger (i.e., the heat exchanger (3, 5)) and air humidification takes place in the other heat exchanger (i.e., the heat exchanger (5, 3))

and the adsorbent material thereof is regenerated. As the result of this, humidification is carried out without interruption.

EFFECTS OF INVENTION

Thus, in accordance with the first invention, it is arranged that: the first heat
5 exchange chamber (69) and the second heat exchange chamber (73) are adjacently
arranged; the inflow passages (57, 63) and the outflow passages (59, 65) are arranged in a
superimposed manner in the thickness direction of the first and second heat exchange
chambers (69, 73). This provides a downsized humidity controller apparatus. That is,
air distribution systems are made simple in construction and the reduction of size is
10 achieved.

Further, it is arranged that adsorbent materials are supported on the surfaces of the
first and second heat exchangers (3) and (5), thereby allowing integral formation of a
heating/cooling means and an adsorption/desorption means. Such an arrangement makes
it possible to provide dehumidification and humidification with the omission of an
15 adsorbent-material container. This results in a reduction in the number of component
parts, thereby making it possible to provide not only a simplified structure but also a
downsized humidity controller apparatus.

Furthermore, it is arranged that adsorbent materials are supported on the surfaces
of the first and second heat exchangers (3) and (5). Such an arrangement makes it
20 possible to allow the refrigerant to directly cool or heat the adsorbent materials. As the
result of this, the adsorption/desorption performance of the adsorbent materials is brought
to a maximum. This makes it possible to improve the efficiency of adsorption/desorption
and to provide a downsized humidity controller apparatus.

In accordance with the second invention, the openings (31a-31d, 33a-33d) are
25 positioned in close proximity to one another in a matrix direction and are opened/closed by

the dampers (35, ..., 47, ...), respectively. Accordingly, air distribution systems are made simple in construction and the reduction of size is achieved.

In accordance with the third invention, the first inflow and outflow passages (63, 65) are arranged symmetrically with the second inflow and outflow passages (57, 59).

5 Therefore, the distribution resistance is reduced. As a result, it is possible to perform dehumidification and other like operation with high efficiency.

In accordance with the fourth invention, the vapor compression refrigeration cycle refrigerant circuit (1) is employed. Therefore, the adsorption and regeneration of adsorbent materials are carried out with high efficiency.

10 In accordance with the fifth or sixth invention, it is possible to continuously perform a dehumidification operation or a humidification operation with the omission of an adsorbent material container. This reduces the number of component parts, thereby making it possible to provide not only a simplified structure but also a downsized humidity controller apparatus. Besides, a dehumidification operation or a humidification operation
15 are performed with high efficiency.

In accordance with the seventh invention, the operation is continued by switching between dehumidification and humidification. This reduces the number of component parts, thereby making it possible to provide not only a simplified structure but also a downsized humidity controller apparatus. Besides, dehumidification or humidification is
20 provided with high efficiency.

BRIEF DESCRIPTION OF DRAWINGS

Figure 1 is a circuit diagram showing a refrigerant circuit of a humidity controller apparatus according to an embodiment of the present invention;

Figure 2 is a perspective view of a heat exchanger according to the embodiment of
25 the present invention;

Figure 3 is a plan view of a casing, with the omission of its top surface plate;

Figure 4 is an end elevation view of the casing taken on line A-A of Fig. 3;

Figure 5 is an end elevation view of the casing taken on line B-B of Fig. 3;

Figure 6 is a side view of a damper when placed in the closed state;

5 Figure 7 is a side view of the damper when placed in the open state;

Figure 8 is a side view of a first modification of the damper when placed in the closed state;

Figure 9 is a side view of the first damper modification when placed in the open state;

10 Figure 10 is a side view of a second modification of the damper when placed in the closed state;

Figure 11 is a side view of the second damper modification when placed in the open state;

15 Figure 12 is a plan view of the casing with the omission of its top surface plate, illustrating a first operation during dehumidification;

Figure 13 is a plan view of the casing with the omission of its top surface plate, illustrating a second operation during dehumidification;

Figure 14 is a plan view of the casing with the omission of its top surface plate, illustrating a first operation during humidification;

20 Figure 15 is a plan view of the casing with the omission of its top surface plate, illustrating a second operation during humidification;

Figure 16 is a psychrometric chart representing an air state of the humidity controller apparatus of the present embodiment and that of a conventional humidity controller apparatus during dehumidification operation; and

25 Figure 17 is a table showing respective data of the humidity controller apparatus

of the present embodiment and the conventional humidity controller apparatus during dehumidification operation.

BEST MODE FOR CARRYING OUT INVENTION

Hereinafter, an embodiment of the present invention will be described in detail
5 with reference to the drawings.

As shown in Figures 1-5, a humidity controller apparatus of the present embodiment effects room-air dehumidification and humidification, and has a hollow, rectangular parallelepiped box-like casing (17). The casing (17) accommodates a refrigerant circuit (1) and other component parts.

10 Referring to Figure 1, the refrigerant circuit (1) is a closed circuit formed by a sequential connection of a compressor (7), a four-way switching valve (9) which is a flow path switching means, a first heat exchanger (3) which is a first heat exchanger, an expansion valve (11) which is an expansion mechanism, and a second heat exchanger (5) which is a second heat exchanger.

15 In addition, the refrigerant circuit (1) is charged with a refrigerant. The refrigerant circuit (1) is configured, such that a vapor compression refrigeration cycle is performed by the circulation of the refrigerant therethrough.

One end of the first heat exchanger (3) is connected to the four-way switching valve (9). The other end of the first heat exchanger (3) is connected to one end of the
20 second heat exchanger (5) via the expansion valve (11). The other end of the second heat exchanger (5) is connected to the four-way switching valve (9).

With reference to Figure 2, the first heat exchanger (3) and the second heat exchanger (5) are each formed by a respective fin and tube heat exchanger of the cross fin type. More specifically, each of the first and second heat exchangers (3) and (5) is
25 provided with a large number of aluminum fins (13) each shaped like a rectangular plate

and copper heat transfer tubes (15) which penetrate the fins (13).

An adsorbent material is supported by means of dip forming (immersion forming) on the external surface of each fin (13) as well as on the external surface of each heat transfer tube (15).

5 As the adsorbent material, zeolite, silica gel, activated carbon, materials selected from the organic polymer family having hydrophilic properties or water-absorbing properties, materials selected from the ion-exchange resin family having carboxylic acid group or sulfonic acid group, and functional polymeric materials such as temperature-sensitive polymers may be used.

10 In the present embodiment, the first heat exchanger (3) and the second heat exchanger (5) are fin and tube heat exchangers of the cross fin type. Alternatively, different types of heat exchangers may be employed. For example, heat exchangers of the corrugated fin type may be used.

In addition, in the present embodiment, the adsorbent material is supported, by
15 means of dip forming, on the external surface of each of the fins (13) and on the external surface of each of the heat transfer tubes (15). Alternatively, the adsorbent material may be supported on the external surfaces by any other techniques as long as its ability to provide adsorption is not reduced.

The four-way switching valve (9) is so configured as to switch between a first
20 state that allows communication between the first and third ports and, at the same time, communication between the second and fourth ports (as indicated in Figure 1(A)), and a second state that allows communication between the first and fourth ports and, at the same time, communication between the second and third ports (as indicated in Figure 1(B)). And, the four-way switching valve (9) is switched for establishing a switch between a first
25 operation in which the first heat exchanger (3) functions as a condenser while,

simultaneously, the second heat exchanger (5) functions as an evaporator and a second operation in which the second heat exchanger (5) functions as a condenser while, simultaneously, the first heat exchanger (3) functions as an evaporator.

Next, with reference to Figures 3-5, the inner structure of the casing (17) is described. Referring to Figure 3, the casing (17) has a front side (which is shown as a lower side in the figure), a backside (upper side), a left-hand side (left side), and a right-hand side (right side). In addition, in Figures 4 and 5, the casing (17) has a top surface (which is shown as an upper side in the figure) and a bottom surface (lower side).

The casing (17) is square when viewed in plan and is shaped like a flat box. The casing (17) has a left side plate (17a) which is provided with a first suction opening (19) through which outdoor air *OA* is drawn in and a second suction opening (21) through which room air *RA* (return air) is drawn in. The casing (17) further has a right side plate (17b) which is provided with a first outlet opening (23) through which emission air *EA* is expelled outside the room and a second outlet opening (25) through which humidity-conditioned air *SA* is supplied to an indoor space.

Disposed inside the casing (17) is a partition plate (27) which is a partition member. The partition plate (27) defines an air chamber (29a) and an equipment chamber (29b) inside the casing (17). The partition plate (27) is disposed perpendicularly, i.e., in the direction of the thickness of the casing (17). Referring to Figures 4 and 5, the partition plate (27) is disposed so as to extend from a top plate (17e) of the casing (17) (upper side) to a bottom plate (17f) of the casing (17) (lower side). Furthermore, with reference to Figure 3, the partition plate (27) is disposed so as to extend from a front plate (17c) of the casing (17) (lower side) to a back plate (17d) of the casing (17) (upper side). In addition, in Figure 3, the partition plate (27) is positioned, such that it lies somewhat right relative to the central part of the casing (17).

The equipment chamber (29b) accommodates the compressor (7) and other devices in the refrigerant circuit (1) with the exclusion of the heat exchangers (3, 5). In addition, the equipment chamber (29b) houses a first fan (79) and a second fan (77). The first fan (79) is connected to the first outlet opening (23) while, on the other hand, the
5 second fan (77) is connected to the second outlet opening (25).

The air chamber (29a) of the casing (17) is provided with a first end surface plate (33) which is a partition member, a second end surface plate (31) which is a partition member, and a central dividing plate (67) which is a partition member. The first end surface plate (33), the second end surface plate (31), and the dividing plate (67) are all
10 disposed perpendicularly, i.e., in the direction of the thickness of the casing (17) and, as shown in Figures 4 and 5, these plates are disposed, such that they extend from the top surface plate (17e) to the bottom surface plate (17e) in the casing (17).

As shown in Figure 3, the first end surface plate (33) and the second end surface plate (31) are disposed so as to extend to the partition plate (27) from the left side surface plate (17a) in the casing (17). In addition, the first end surface plate (33) is positioned,
15 such that it slightly overlies the central part of casing (17) in Figure 3. On the other hand, the second end surface plate (31) is positioned, such that it slightly underlies the central part of casing (17), as shown in Figure 3.

The dividing plate (67) is disposed so as to extend between the first end surface plate (33) and the second end surface plate (31), as shown in Figure 3.
20

And, in the casing (17), a first heat exchange chamber (69) is divisionally formed as a compartment by the first end surface plate (33), the second end surface plate (31), the dividing plate (67), and the partition plate (27). In addition, in the casing (17), a second heat exchange chamber (73) is divisionally formed as a compartment by the first end
25 surface plate (33), the second end surface plate (31), the dividing plate (67), and the left

side surface plate (17a) of the casing (17). In other words, the first heat exchange chamber (69) lies on the right-hand side in Figure 3; the second heat exchange chamber (73) lies on the left-hand side in Figure 3; and the first heat exchange chamber (69) and the second heat exchange chamber (73) are formed adjacently parallelly with each other.

The first heat exchanger (3) is disposed in the first heat exchange chamber (69). The second heat exchanger (5) is disposed in the second heat exchange chamber (73).

A horizontal plate (61), serving as a partition member, is disposed between the first end surface plate (33) and the back plate (17d) of the casing (17), such that it defines a first inflow passage (63) and a first outflow passage (65). In addition, a horizontal plate (55), serving as a partition member, is disposed between the second end surface plate (31) and the front plate (17c) of the casing (17), such that it defines a second inflow passage (57) and a second outflow passage (59).

The interior space of the casing (17) is divided vertically (i.e., in the direction of the thickness of the casing (17)) into upper and lower spaces by the horizontal plates (61, 55). And, in Figure 4, the first inflow passage (63) is formed on the top surface side while, on the other hand, the first outflow passage (65) is formed on the bottom surface side. In Figure 5, the second inflow passage (57) is formed on the top surface side while, on the other hand, the second outflow passage (59) is formed on the bottom surface side.

That is, the first inflow passage (63) and the first outflow passage (65) are formed along one end surface in a thickness direction in which respective one surfaces of the first and second heat exchange chambers (69) and (73) continue, and are arranged in a superimposed manner in the thickness direction of the first and second heat exchange chambers (69, 73).

In addition, a second air inflow passage (57) and a second air outflow

passage (59) which are formed along another end surface which is an end surface where respective one surfaces of the heat exchange chambers (69, 73) continue and which is situated face to face with the one end surface, and which are arranged in a superimposed manner in the thickness direction of the heat exchange chambers (69, 73).

5 And, in Figure 3, the first inflow passage (63) and the first outflow passage (65) are arranged symmetrically with the second inflow passage (57) and the second outflow passage (59), in other words, they are arranged in plane symmetry on the basis of a central line that crosses the first and second heat exchange chambers (69) and (73).

In addition, the first inflow passage (63) is in communication with the first suction
10 opening (19). The first outflow passage (65) is in communication with the first fan (79), thereby being in communication with the first outlet opening (23). The second inflow passage (57) is in communication with the second suction opening (21). The second outflow passage (59) is in communication with the second fan (77), thereby being in communication with the second outlet opening (25).

15 As shown in Figure 4, the first end surface plate (33) is provided with first to fourth openings (33a, 33b, 33c, 33d). The first to fourth openings (33a, 33b, 33c, 33d) are provided with a first damper (47), a second damper (49), a third damper (51), and a fourth damper (53), respectively. These four openings (33a-33d) are positioned in close proximity to one another in a matrix direction, in other words they are vertically laterally
20 arranged in the form of squares in pairs. The first opening (33a) and the third opening (33c) open to the first heat exchange chamber (69) while, on the other hand, the second opening (33b) and the fourth opening (33d) open to the second heat exchange chamber (73).

The first opening (33a) allows communication between the first inflow
25 passage (63) and the first heat exchange chamber (69). The third opening (33c) allows

communication between the first outflow passage (65) and the first heat exchange chamber (69). In addition, the second opening (33b) allows communication between the first inflow passage (63) and the second heat exchange chamber (73). The fourth opening (33d) allows communication between the first outflow passage (65) and the
5 second heat exchange chamber (73).

As shown in Figure 5, the second end surface plate (31) is provided with fifth to eighth openings (31a, 31b, 31c, 31d). The fifth to eighth openings (31a, 31b, 31c, 31d) are provided with a fifth damper (35), a sixth damper (37), a seventh damper (39), and an eighth damper (41), respectively. The four openings (31a, 31b, 31c, 31d) are positioned
10 in close proximity to one another in a matrix direction, in other words they are vertically laterally arranged in the form of squares in pairs. The fifth opening (31a) and the seventh opening (31c) open to the first heat exchange chamber (69) while, on the other hand, the sixth opening (31b) and the eighth opening (31d) open to the second heat exchange chamber (73).

15 The fifth opening (31a) allows communication between the second inflow passage (57) and the first heat exchange chamber (69). The seventh opening (31c) allows communication between the second outflow passage (59) and the first heat exchange chamber (69). In addition, the sixth opening (31b) allows communication between the second inflow passage (57) and the second heat exchange chamber (73). The eighth
20 opening (31d) allows communication between the second outflow passage (59) and the second heat exchange chamber (73).

The first to eighth dampers (47-53, 35-41) constitute opening and closing means for opening and closing the openings (33a-33d, 31a-31d). The description is made with a focus on the fifth to eighth dampers (35-41). The damper (35-41) has a vane part (43)
25 shaped like a rectangle and a shaft part (45) arranged centrally in the vane part (43), as

shown in Figures 6 and 7. The shaft part (45) supports the vane part (43) on the first end surface plate (33) or on the second end surface plate (31), such that the vane part (43) is rotatable. And, as shown in Figure 7, the damper (35-41) is configured so that the opening (31a-31d) is placed in the open state when the vane part (43) is placed in a horizontal position. The other first to fourth dampers (47-53) are formed so as to have the same structure as that of the fifth to eighth dampers (35-41).

The structure of the dampers (35-41) is not limited to the ones shown in Figures 6 and 7. That is, each damper (47-53, 35-41) may employ a structure shown in Figures 10 and 11 or a structure shown in Figures 8 and 9.

Referring to Figures 8 and 9, there are shown fifth to eighth dampers (35-41) each of which is provided with two vane parts (43). Each damper (35-41) is configured, such that its two vane parts (43) rotate individually upwards and downwards respectively, thereby placing the associated opening (31a-31d) in the open state.

Referring to Figures 10 and 11, there are shown fifth to eighth dampers (35-41) each of which is provided with two vane parts (43). Each damper (35-41) is configured, such that its two vane parts (43) is folded upwards, thereby placing the associated opening (31a-31d) in the open state.

In addition, the present humidity controller apparatus is provided with a dehumidifier means (80) and a humidifier means (81). And, it is configured that the dehumidifier means (80) may be switched to function as the humidifier means (81), and vice versa, thereby allowing a switch between a dehumidification mode of operation and a humidification mode of operation.

The dehumidifier means (80) switches the circulation of refrigerant in the refrigerant circuit (1) and the distribution of air by the damper (47-53, 35-41) so that moisture present in a stream of air flowing through the first heat exchanger (3) or the

second heat exchanger (5), whichever functions as a refrigerant evaporating heat exchanger, is adsorbed by adsorbent material while, on the other hand, adsorbent material is regenerated by the release of moisture present in the adsorbent material to a stream of air flowing through the second heat exchanger (5) or the first heat exchanger (3), whichever
5 functions as a refrigerant condensing heat exchanger, and the stream of air thus converted into a stream of dehumidified air by the adsorbent material is supplied into the room.

On the other hand, the humidifier means (81) switches the circulation of refrigerant in the refrigerant circuit (1) and the distribution of air by the damper (47-53, 35-41) so that moisture present in a stream of air flowing through the first heat exchanger (3)
10 or the second heat exchanger (5), whichever functions as a refrigerant evaporating heat exchanger, is adsorbed by adsorbent material while, on the other hand, adsorbent material is regenerated by the release of moisture present in the adsorbent material to a stream of air flowing through the second heat exchanger (5) or the first heat exchanger (3), whichever
functions as a refrigerant condensing heat exchanger, and the stream of air thus converted
15 into a stream of humidified air by the adsorbent material is supplied into the room.

RUNNING OPERATION

Next, the running operation of the above-described humidity controller apparatus is described. The humidity controller apparatus takes in a stream of first air and a stream of second air and performs selectively a dehumidification operation or a humidification
20 operation. In addition, the humidity controller apparatus continuously performs dehumidification and humidification operations by alternately repeating a first operation and a second operation. Besides, the humidity controller apparatus performs dehumidification and humidification operations in full ventilation mode and, in addition, humidification and dehumidification operations in circulation mode.

DEHUMIDIFICATION OPERATION IN FULL VENTILATION MODE

In a dehumidification operation during full ventilation mode by the dehumidifier means (80), outdoor air *OA* is brought in as the first air and is supplied to the room while, on the other hand, room air *RA* is brought in as the second air and is discharged outside the
5 room.

FIRST OPERATION

In the first operation in which the first fan (79) and the second fan (77) are driven, the process of moisture adsorption in the second heat exchanger (5) and the process of adsorbent-material regeneration (moisture desorption) in the first heat exchanger (3) are
10 carried out. In other words, in the first operation, moisture present in the outdoor air *OA* is adsorbed in the second heat exchanger (5) and moisture desorbed out of the first heat exchanger (3) is given to the room air *RA*.

Referring to Figure 1(A) and Figure 12, during the first operation the second damper (49), the third damper (51), the eighth damper (41), and the fifth damper (35) are
15 placed in the open state while, on the other hand, the first damper (47), the fourth damper (53), the sixth damper (37), and the seventh damper (39) are placed in the closed state. The first heat exchanger (3) is supplied with a stream of room air *RA*. The second heat exchanger (5) is supplied with a stream of outdoor air *OA*.

In addition, the four-way switching valve (9) changes state to a state shown in
20 Figure 1 (A). As a result, in the refrigerant circuit (1) the first heat exchanger (3) functions as a condenser and the second heat exchanger (5) functions as an evaporator.

That is, high-temperature, high-pressure refrigerant expelled out of the compressor (7) flows into the first heat exchanger (3) as a heat carrier for heating. In the first heat exchanger (3), the adsorbent materials supported, respectively, on the external
25 surface of each fin (13) and on the external surface of each heat transfer tube (15) are

heated. This heating causes moisture desorption from the adsorbent materials, whereby the adsorbent materials are regenerated.

On the other hand, the refrigerant condensed in the first heat exchanger (3) is decompressed by the expansion valve (11). The post-decompression refrigerant flows
5 into the second heat exchanger (5) as a heat carrier for cooling. In the second heat exchanger (5), heat of adsorption is generated when the adsorbent material supported on the external surface of each fin (13) and the adsorbent material supported on the external surface of each heat transfer tube (15) adsorb moisture. The refrigerant in the second heat exchanger (5) absorbs the heat of adsorption and then evaporates. The refrigerant
10 evaporated is directed back to the compressor (7) and repeats such a circulation.

In addition, an inflow of room air *RA* entering through the second suction opening (21) by drive of the first and second fans (79) and (77) travels through the second inflow passage (57) and flows to the first heat exchange chamber (69) from the fifth opening (31a). In the first heat exchange chamber (69), moisture desorbed from the
15 adsorbent material of the first heat exchanger (3) is released to the room air *RA* and, in this way, the room air *RA* is humidified. The room air *RA* thus humidified becomes a stream of emission air *EA*. The emission air *EA* exiting the first heat exchange chamber (69) flows through the first outflow passage (65) by way of the third opening (33c). Then, the emission air *EA*, after passing through the first fan (79), is discharged outdoors through the
20 first outlet opening (23).

On the other hand, an inflow of outdoor air *OA* entering through the first suction opening (19) travels through the first inflow passage (63) and flows to the second heat exchange chamber (73) from the second opening (33b). In the second heat exchange chamber (73), moisture in the outdoor air *OA* is adsorbed by the adsorbent material of the
25 second heat exchanger (5) and, in this way, the outdoor air *OA* is dehumidified. The

outdoor air *OA* thus dehumidified becomes a stream of humidity conditioned air *SA*. The humidity conditioned air *SA* exiting the second heat exchange chamber (73) flows through the second outflow passage (59) by way of the eighth opening (31d). Then, the humidity conditioned air *SA*, after passing through the second fan (77), is supplied indoors through
5 the second outlet opening (25).

Upon completion of the execution of the first operation, the second operation is carried out.

SECOND OPERATION

In the second operation in which the first fan (79) and the second fan (77) are
10 driven, the process of adsorption in the first heat exchanger (3) and the process of regeneration in the second heat exchanger (5) are carried out. In other words, in the second operation, moisture present in outdoor air *OA* is adsorbed in the first heat exchanger (3) and moisture desorbed out of the second heat exchanger (5) is given to room air *RA*.

15 Referring to Figure 1(B) and Figure 13, during the second operation the first damper (47), the fourth damper (53), the seventh damper (39), and the sixth damper (37) are placed in the open state while, on the other hand, the third damper (51), the second damper (49), the fifth damper (35), and the eighth damper (41) are placed in the closed state. And, the first heat exchanger (3) is supplied with a stream of outdoor air *OA*. The
20 second heat exchanger (5) is supplied with a stream of room air *RA*.

In addition, the four-way switching valve (9) changes state to a state shown in Figure 1(B). As a result, in the refrigerant circuit (1) the second heat exchanger (5) functions as a condenser and the first heat exchanger (3) functions as an evaporator.

That is, high-temperature, high-pressure refrigerant expelled out of the
25 compressor (7) flows into the second heat exchanger (5) as a heat carrier for heating. In

the second heat exchanger (5), the adsorbent materials supported, respectively, on the external surface of each fin (13) and on the external surface of each heat transfer tube (15) are heated. This heating causes moisture desorption from the adsorbent materials, whereby the adsorbent materials are regenerated.

5 On the other hand, the refrigerant condensed in the second heat exchanger (5) is decompressed by the expansion valve (11). The post-decompression refrigerant flows into the first heat exchanger (3) as a heat carrier for cooling. In the first heat exchanger (3), heat of adsorption is generated when the adsorbent material supported on the external surface of each fin (13) and the adsorbent material supported on the external
10 surface of each heat transfer tube (15) adsorb moisture. The refrigerant in the first heat exchanger (3) absorbs the heat of adsorption and then evaporates. The refrigerant evaporated is directed back to the compressor (7) and repeats such a circulation.

 In addition, an inflow of room air *RA* entering through the second suction opening (21) by drive of the first and second fans (79) and (77) travels through the second
15 inflow passage (57) and flows to the second heat exchange chamber (73) from the sixth opening (31b). In the second heat exchange chamber (73), moisture desorbed from the adsorbent material of the second heat exchanger (5) is released to the room air *RA* and, in this way, the room air *RA* is humidified. The room air *RA* thus humidified becomes a stream of emission air *EA*. The emission air *EA* exiting the second heat exchange
20 chamber (73) flows through the first outflow passage (65) by way of the fourth opening (33d). Then, the emission air *EA*, after passing through the first fan (79), is discharged outdoors from the first outlet opening (23).

 On the other hand, an inflow of outdoor air *OA* entering through the first suction opening (19) travels through the first inflow passage (63). The outdoor air *OA* flows into
25 the first heat exchange chamber (69) from the first opening (33a). In the first heat

exchange chamber (69), moisture in the outdoor air *OA* is adsorbed by the adsorbent material of the first heat exchanger (3), whereby the outdoor air *OA* is dehumidified. The outdoor air *OA* thus dehumidified becomes a stream of humidity conditioned air *SA*. The humidity conditioned air *SA* exiting the first heat exchange chamber (69) flows through the
5 second outflow passage (59) by way of the seventh opening (31c). Then, the humidity conditioned air *SA*, after passing through the second fan (77), is supplied indoors through the second outlet opening (25).

After execution of the second operation, the first operation is carried out again. And, indoor space dehumidification is continuously carried out by repetition of the first
10 operation and the second operation.

HUMIDIFICATION OPERATION IN FULL VENTILATION MODE

In a humidification operation in full ventilation mode by the humidifier means (81), room air *RA* is brought in as the first air and is discharged outside the room while, on the other hand, outdoor air *OA* is brought in as the second air and is supplied into
15 the room.

FIRST OPERATION

In the first operation in which the first fan (79) and the second fan (77) are driven, the process of adsorption in the second heat exchanger (5) and the process of regeneration in the first heat exchanger (3) are carried out. In other words, in the first operation,
20 moisture present in the room air *RA* is adsorbed in the second heat exchanger (5) and moisture desorbed out of the first heat exchanger (3) is given to the outdoor air *OA*.

Referring to Figure 1(A) and Figure 14, during the first operation the first damper (47), the fourth damper (53), the seventh damper (39), and the sixth damper (37) are placed in the open state while, on the other hand, the third damper (51), the second
25 damper (49), the fifth damper (35), and the eighth damper (41) are placed in the closed

state. The first heat exchanger (3) is supplied with a stream of outdoor air *OA*. The second heat exchanger (5) is supplied with a stream of room air *RA*.

In addition, the four-way switching valve (9) changes state to a state shown in Figure 1 (A). As a result, in the refrigerant circuit (1) the first heat exchanger (3) functions as a condenser and the second heat exchanger (5) functions as an evaporator.

That is, high-temperature, high-pressure refrigerant expelled out of the compressor (7) flows into the first heat exchanger (3) as a heat carrier for heating. In the first heat exchanger (3), the adsorbent materials supported, respectively, on the external surface of each fin (13) and on the external surface of each heat transfer tube (15) are heated. This heating causes moisture desorption from the adsorbent materials, whereby the adsorbent materials are regenerated.

On the other hand, the refrigerant condensed in the first heat exchanger (3) is decompressed by the expansion valve (11). The post-decompression refrigerant flows into the second heat exchanger (5) as a heat carrier for cooling. In the second heat exchanger (5), heat of adsorption is generated when the adsorbent material supported on the external surface of each fin (13) and the adsorbent material supported on the external surface of each heat transfer tube (15) adsorb moisture. The refrigerant in the second heat exchanger (5) absorbs the heat of adsorption and then evaporates. The refrigerant evaporated is directed back to the compressor (7) and repeats such a circulation.

In addition, an inflow of room air *RA* entering through the second suction opening (21) by drive of the first and second fans (79) and (77) travels through the second inflow passage (57) and flows into the second heat exchange chamber (73) from the sixth opening (31b). In the second heat exchange chamber (73), moisture in the room air *RA* is adsorbed by the adsorbent material of the second heat exchanger (5), whereby the room air *RA* is dehumidified. The room air *RA* thus dehumidified becomes a stream of

emission air *EA*. The emission air *EA* exiting the second heat exchange chamber (73) flows through the first outflow passage (65) by way of the fourth opening (33d). Then, the emission air *EA*, after passing through the first fan (79), is discharged outdoors through the first outlet opening (23).

5 On the other hand, an inflow of outdoor air *OA* entering through the first suction opening (19) travels through the first inflow passage (63) and flows to the first heat exchange chamber (69) from the first opening (33a). In the first heat exchange chamber (69), moisture desorbed out of the adsorbent material of the first heat exchanger (3) is released to the outdoor air *OA*, whereby the outdoor air *OA* is humidified.

10 The outdoor air *OA* thus humidified becomes a stream of humidity conditioned air *SA*. The humidity conditioned air *SA* exiting the first heat exchange chamber (69) flows through the second outflow passage (59) by way of the seventh opening (31c). Then, the humidity conditioned air *SA*, after passing through the second fan (77), is supplied indoors through the second outlet opening (25).

15 Upon completion of the execution of the first operation, the second operation carried out.

SECOND OPERATION

 In the second operation in which the first fan (79) and the second fan (77) are driven, the process of adsorption in the first heat exchanger (3) and the process of
20 regeneration in the second heat exchanger (5) are carried out. In other words, in the second operation, moisture present in room air *RA* is adsorbed in the first heat exchanger (3) and moisture desorbed out of the second heat exchanger (5) is given to outdoor air *OA*.

 Referring to Figure 1(B) and Figure 15, during the second operation the second
25 damper (49), the third damper (51), the eighth damper (41), and the fifth damper (35) are

placed in the open state while, on the other hand, the fourth damper (53), the first damper (47), the sixth damper (37), and the seventh damper (39) are placed in the closed state. The first heat exchanger (3) is supplied with a stream of room air *RA*. The second heat exchanger (5) is supplied with a stream of outdoor air *OA*.

5 In addition, the four-way switching valve (9) changes state to a state shown in Figure 1 (B). As a result, in the refrigerant circuit (1) the second heat exchanger (5) functions as a condenser and the first heat exchanger (3) functions as an evaporator.

That is, high-temperature, high-pressure refrigerant expelled out of the compressor (7) flows to the second heat exchanger (5) as a heat carrier for heating. In the
10 second heat exchanger (5), the adsorbent materials supported, respectively, on the external surface of each fin (13) and on the external surface of each heat transfer tube (15) are heated. This heating causes moisture desorption from the adsorbent materials, whereby the adsorbent materials are regenerated.

On the other hand, the refrigerant condensed in the second heat exchanger (5) is
15 decompressed by the expansion valve (11). The post-decompression refrigerant flows to the first heat exchanger (3) as a heat carrier for cooling. In the first heat exchanger (3), heat of adsorption is generated when the adsorbent materials supported, respectively, on the external surface of each fin (13) and on the external surface of the heat transfer tube (15) adsorb moisture. The refrigerant in the first heat exchanger (3) absorbs the heat
20 of adsorption and then evaporates. The refrigerant evaporated is directed back to the compressor (7) and repeats such a circulation.

In addition, an inflow of air *RA* entering through the second suction opening (21) by drive of the first and second fans (79) and (77) travels through the second inflow passage (57). Then, the room air *RA* flows to the first heat exchange chamber (69) from
25 the fifth opening (31a). In the first heat exchange chamber (69), moisture in the room

air *RA* is adsorbed by the adsorbent material of the first heat exchanger (3), whereby the room air *RA* is dehumidified. The room air *RA* thus dehumidified becomes a stream of emission air *EA*. The emission air *EA* exiting the first heat exchange chamber (69) flows through the first outflow passage (65) by way of the third opening (33c). Then, after
5 passage through the first fan (79), the emission air *EA* is discharged outdoors through the first outlet opening (23).

On the other hand, an inflow of outdoor air *OA* entering through the first suction opening (19) travels through the first inflow passage (63). Then, the outdoor air *OA* flows into the second heat exchange chamber (73) from the second opening (33b). In the
10 second heat exchange chamber (73), moisture desorbed from the adsorbent material of the second heat exchanger (5) is released to the outdoor air *OA*, whereby the outdoor air *OA* is humidified. The outdoor air *OA* thus humidified becomes a stream of humidity conditioned air *SA*. The humidity conditioned air *SA* exiting the second heat exchange chamber (73) flows through the second outflow passage (59) by way of the eighth
15 opening (31d). Then, after passage through the second fan (77), the humidity conditioned air *SA* is supplied indoors through second outlet opening (25).

Upon completion of the execution of the second operation, the first operation is carried out again. And, indoor space humidification is continuously carried out by repetition of the first operation and the second operation.

20 DEHUMIDIFICATION OPERATION IN CIRCULATION MODE

During a dehumidification operation in circulation mode by the dehumidifier means (80), room air *RA* is brought in as the first air and is supplied indoors while, on the other hand, outdoor air *OA* is brought in as the second air and is discharged outdoors. The circulation of refrigerant in the refrigerant circuit (1) is the same as that in full
25 ventilation mode and the description thereof is omitted accordingly.

FIRST OPERATION

In the first operation, the process of adsorption in the second heat exchanger (5) and the process of regeneration (desorption) in the first heat exchanger (3) are carried out. In other words, in the first operation, moisture present in room air *RA* is adsorbed in the second heat exchanger (5) and moisture desorbed out of the first heat exchanger (3) is given to outdoor air *OA*.

During the first operation, the first damper (47), the third damper (51), the sixth damper (37), and the eighth damper (41) are placed in the open state while, on the other hand, the second damper (49), the fourth damper (53), the fifth damper (35), and the seventh damper (39) are placed in the closed state. The first heat exchanger (3) is provided with a stream of outdoor air *OA*. The second heat exchanger (5) is provided with a stream of room air *RA*.

An inflow of outdoor air *OA* entering through the first suction opening (19) travels through the first inflow passage (63). Then, the outdoor air *OA* flows into the first heat exchange chamber (69) from the first opening (33a). In the first heat exchange chamber (69), moisture desorbed from the adsorbent material of the first heat exchanger (3) is released to the outdoor air *OA*, whereby the outdoor air *OA* is humidified. The outdoor air *OA* thus humidified becomes a stream of emission air *EA*. The emission air *EA* exiting the first heat exchange chamber (69) flows through the first outflow passage (65) by way of the third opening (33c). Then, after passage through the first fan (79), the emission air *EA* is discharged outdoors through the first outlet opening (23).

On the other hand, an inflow of room air *RA* entering through the second suction opening (21) travels through the second inflow passage (57). Then, the room air *RA* flows into the second heat exchange chamber (73) from the sixth opening (31b). In the second heat exchange chamber (73), moisture in the room air *RA* is adsorbed by the

adsorbent material of the second heat exchanger (5), whereby the room air *RA* is dehumidified. The room air *RA* thus dehumidified becomes a stream of humidity conditioned air *SA*. The humidity conditioned air *SA* exiting the second heat exchange chamber (73) flows through the second outflow passage (59) by way of the eighth opening (31d). Then, after passage through the second fan (77), the humidity conditioned air *SA* is supplied indoors through the second outlet opening (25).

Upon completion of the execution of the first operation, the second operation is carried out.

SECOND OPERATION

10 In the second operation, the process of adsorption in the first heat exchanger (3) and the process of regeneration in the second heat exchanger (5) are carried out. In other words, in the second operation, moisture present in room air *RA* is adsorbed in the first heat exchanger (3) and moisture desorbed out of the second heat exchanger (5) is given to outdoor air *OA*.

15 During the second operation, the second damper (49), the fourth damper (53), the fifth damper (35), and the seventh damper (39) are placed in the open state while, on the other hand, the first damper (47), the third damper (51), the sixth damper (37), and the eighth damper (41) are placed in the closed state. The first heat exchanger (3) is provided with a stream of room air *RA*. The second heat exchanger (5) is provided with a stream of outdoor air *OA*.

20 An inflow of outdoor air *OA* entering through the first suction opening (19) travels through the first inflow passage (63). Then, the outdoor air *OA* flows into the second heat exchange chamber (73) from the second opening (33b). In the second heat exchange chamber (73), moisture desorbed from the adsorbent material of the second heat exchanger (5) is released to the outdoor air *OA*, whereby the outdoor air *OA* is humidified.

The outdoor air *OA* thus humidified becomes a stream of emission air *EA*. The emission air *EA* exiting the second heat exchange chamber (73) flows through the first outflow passage (65) by way of the fourth opening (33d). Then, after passage through the first fan (79), the emission air *EA* is discharged outdoors through the first outlet opening (23).

5 On the other hand, an inflow of room air *RA* entering through the second suction opening (21) travels through the second inflow passage (57). Then, the room air *RA* flows into the first heat exchange chamber (69) from the fifth opening (31a). In the first heat exchange chamber (69), moisture in the room air *RA* is adsorbed by the adsorbent material of the first heat exchanger (3), whereby the room air *RA* is dehumidified. The
10 room air *RA* thus dehumidified becomes a stream of humidity conditioned air *SA*. The humidity conditioned air *SA* exiting the first heat exchange chamber (69) flows through the second outflow passage (59) by way of the seventh opening (31c). Then, after passage through the second fan (77), the humidity conditioned air *SA* is supplied indoors through the second outlet opening (25).

15 After execution of the second operation, the first operation is carried out again. And, indoor space dehumidification is continuously carried out by repetition of the first operation and the second operation.

HUMIDIFICATION OPERATION IN CIRCULATION MODE

 In a humidification operation in circulation mode by the humidifier means (81),
20 outdoor air *OA* is brought in as the first air and is discharged outdoors while, on the other hand, room air *RA* is brought in as the second air and is supplied indoors. The circulation of refrigerant in the refrigerant circuit (1) is the same as that in full ventilation mode and the description thereof is omitted accordingly.

FIRST OPERATION

25 In the first operation, the process of adsorption in the second heat exchanger (5)

and the process of regeneration in the first heat exchanger (3) are carried out. In other words, in the first operation, moisture present in outdoor air *OA* is adsorbed in the second heat exchanger (5) and moisture desorbed out of the first heat exchanger (3) is given to room air *RA*.

5 During the first operation, the second damper (49), the fourth damper (53), the fifth damper (35), and the seventh damper (39) are placed in the open state while, on the other hand, the first damper (47), the third damper (51), the sixth damper (37), and the eighth damper (41) are placed in the closed state. The first heat exchanger (3) is supplied with a stream of room air *RA*. The second heat exchanger (5) is supplied with a stream of
10 outdoor air *OA*.

An inflow of room air *RA* entering through the second suction opening (21) travels through the second inflow passage (57). Then, the room air *RA* flows into the first heat exchange chamber (69) from the fifth opening (31a). In the first heat exchange chamber (69), moisture desorbed from the adsorbent material of the first heat
15 exchanger (3) is released to the room air *RA*, whereby the room air *RA* is humidified. The room air *RA* thus humidified exits the first heat exchange chamber (69) and flows through the second outflow passage (59) by way of the seventh opening (31c). Then, after passage through the second fan (77), the room air *RA* is supplied indoors through the second outlet opening (25).

20 On the other hand, an inflow of outdoor air *OA* entering through the first suction opening (19) travels through the first inflow passage (63). Then, the outdoor air *OA* flows into the second heat exchange chamber (73) from the second opening (33b). In the second heat exchange chamber (73), moisture in the outdoor air *OA* is adsorbed by the adsorbent material of the second heat exchanger (5), whereby the outdoor air *OA* is
25 dehumidified. The outdoor air *OA* thus dehumidified becomes a stream of emission

air *EA*. The emission air *EA* exiting the second heat exchange chamber (73) flows through the first outflow passage (65) by way of the fourth opening (33d). Then, after passage through the first fan (79), the emission air *EA* is discharged outdoors through the first outlet opening (23).

- 5 Upon completion of the execution of the first operation, the second operation is carried out.

SECOND OPERATION

 In the second operation, the process of adsorption in the first heat exchanger (3) and the process of regeneration in the second heat exchanger (5) are carried out. In other
10 words, in the second operation, moisture present in outdoor air *OA* is adsorbed in the first heat exchanger (3) and moisture desorbed out of the second heat exchanger (5) is given to room air *RA*.

 During the second operation, the first damper (47), the third damper (51), the sixth damper (37), and the eighth damper (41) are placed in the open state while, on the other
15 hand, the second damper (49), the fourth damper (53), the fifth damper (35), and the seventh damper (39) are placed in the closed state. The second heat exchanger (5) is supplied with a stream of room air *RA*. The first heat exchanger (3) is provided with a stream of outdoor air *OA*.

 An inflow of room air *RA* entering through the second suction opening (21)
20 travels through the second inflow passage (57). Then, the room air *RA* flows into the second heat exchange chamber (73) from the sixth opening (31b). In the second heat exchange chamber (73), moisture desorbed from the adsorbent material of the second heat exchanger (5) is released to the room air *RA*, whereby the room air *RA* is humidified. The room air *RA* thus humidified becomes a stream of humidity conditioned air *SA*. The
25 humidity conditioned air *SA* exiting the second heat exchange chamber (73) flows through

the second outflow passage (59) by way of the eighth opening (31d). Then, after passage through the second fan (77), the humidity conditioned air *SA* is supplied indoors through the second outlet opening (25).

On the other hand, an inflow of outdoor air *OA* entering through the first suction opening (19) travels through the first inflow passage (63). Then, the outdoor air *OA* flows into the first heat exchange chamber (69) from the first opening (33a). In the first heat exchange chamber (69), moisture present in the outdoor air *OA* is adsorbed by the adsorbent material of the second heat exchanger (5), whereby the outdoor air *OA* is dehumidified. The outdoor air *OA* thus dehumidified becomes a stream of emission air *EA*. The emission air *EA* exiting the first heat exchange chamber (69) flows through the first outflow passage (65) by way of the third opening (33c). Then, after passage through the first fan (79), the emission air *EA* is discharged outdoors through the first outlet opening (23).

Upon completion of the execution of the second operation, the first operation is carried out again. And, indoor space humidification is continuously carried out by repetition of the first operation and the second operation.

PERFORMANCE COMPARISON

Referring to Figure 16, there is shown a psychrometric chart for comparison between the dehumidification operation of a humidity controller apparatus of the present embodiment and the dehumidification operation of a conventional humidity controller apparatus. Both the humidity controller apparatus of the present embodiment and the conventional humidity controller apparatus are capable of dehumidifying a volume of air (about 150 m³) per hour.

Referring to Figure 17, there is shown a table for comparison between the data of the dehumidification operation of a humidity controller apparatus of the present

embodiment and the data of the dehumidification operation of a conventional humidity controller apparatus. The data shown in the table include information about the inlet temperature of outdoor air *OA* and other pieces of information. In addition, the symbols "①"- "⑧" indicated in Figure 17 correspond to the symbols "①"- "⑧" indicated in Figure 16, respectively.

As is clear from Figures 16 and 17, the amount of dehumidification of the humidity controller apparatus according to the present embodiment is greater than the amount of dehumidification of the conventional humidity controller apparatus. More specifically, the amount of dehumidification of the humidity controller apparatus according to the present embodiment is twice the amount of dehumidification of the conventional humidity controller apparatus.

EFFECTS OF EMBODIMENT 1

As described above, according to the present embodiment, it is arranged that adsorbent materials are supported on the external surface of each heat transfer tube (15) as well as on the external surface of each fin (13) in the first and second heat exchangers (3) and (5), thereby allowing integral formation of a heating/cooling means and an adsorption/desorption means. Such an arrangement makes it possible to continuously provide dehumidification and humidification with the omission of an adsorbent-material container. This results in a reduction in the number of component parts, thereby making it possible to provide not only a simplified structure but also a downsized humidity controller apparatus.

Furthermore, it is arranged that adsorbent materials are supported on the external surface of each heat transfer tube (15) as well as on the external surface of each fin (13) in the first and second heat exchangers (3) and (5). Such an arrangement makes it possible to allow the refrigerant to directly cool or heat the adsorbent materials. As the result of

this, the adsorption/desorption performance of the adsorbent materials is brought to a maximum. This makes it possible to improve the efficiency of adsorption/desorption and to provide a downsized humidity controller apparatus.

That is, the contact thermal resistance is large if the adsorbent material is brought
5 into contact with only the external surface of each fin (13). Thus, the effects of cooling and heating cannot be expected. In the present embodiment, adsorbent materials are supported on the external surface of each heat transfer tube (15) and the external surface of each fin (13) in the first and second heat exchangers (3) and (5), whereby the effects of cooling and heating are sufficiently achieved.

10 Furthermore, it is arranged that adsorbent materials are supported on the external surface of the first heat exchanger (3) as well as on the external surface of second heat exchanger (5). Such arrangement makes it possible to continuously perform dehumidification and humidification. As the result of this, it becomes possible to perform dehumidification and humidification with high efficiency.

15 In addition, it is arranged that: the first heat exchange chamber (69) and the second heat exchange chamber (73) are adjacently arranged; the inflow passages (57, 63) and the outflow passages (59, 65) are arranged in a superimposed manner in the thickness direction of the first and second heat exchange chambers (69, 73). Such an arrangement provides a downsized humidity controller apparatus.

20 Besides, the eight dampers (35, ..., 47, ...) are provided, whereby the direction in which air is distributed is switched. This makes it possible to achieve a switch in the distribution direction of air by a simplified structure.

Furthermore, the first inflow and outflow passages (63, 65) are arranged symmetrically with the second inflow and outflow passages (57, 59). As a result of such
25 arrangement, the distribution resistance is reduced. This makes it possible to perform

dehumidification and other like operation with high efficiency.

In addition, the openings (31a-31d, 33a-33d) are positioned in close proximity to one another in a matrix direction and are opened/closed by the dampers (35, ..., 47, ...), respectively. Accordingly, air distribution systems are made simple in construction and
5 the reduction of size is achieved.

Furthermore, the vapor compression refrigeration cycle refrigerant circuit (1) is employed, the adsorption and regeneration of adsorbent materials are carried out with high efficiency.

OTHER EMBODIMENTS OF INVENTION

10 In the foregoing embodiment, in each of the first heat exchanger (3) and the second heat exchanger (5), adsorbent materials are supported on the external surface of each fin (13) as well as on the external surface of each heat transfer tube (15). In the present invention, however, it may be arranged that adsorbent materials are supported on at least either one of the external surface of the fin (13) and the external surface of the heat
15 transfer tube (15).

In addition, in the foregoing embodiment, adsorbent materials are supported on both the first heat exchanger (3) and the second heat exchanger (5); however, it may be arranged that adsorbent material is supported on one of the heat exchangers, i.e., only on the first heat exchanger (3). In this case, moisture adsorption and moisture desorption
20 (adsorbent material regeneration) take place at intervals.

Although in the foregoing embodiment the vapor compression refrigeration cycle refrigerant circuit (1) is employed to cool or heat the adsorbent materials of the first heat exchanger (3) and the second heat exchanger (5), cold water and hot water may be used instead. Specifically, the adsorbent materials may be cooled or heated by introducing a
25 passage of cold or hot water in the first heat exchanger (3) and the second heat

exchanger (5).

Furthermore, the present invention is not limited to the arrangement of the embodiment shown in plan view (see Figure 3). That is, the orientation of arrangement is not limited to that of the embodiment.

5 In addition, in the above-described embodiment, the first fan (79) is connected to the first outlet opening (23). Alternatively, the first fan (79) may be provided so as to communicate with the first suction opening (19). In addition, the second fan (77) is connected to the second outlet opening (25). Alternatively, the second fan (77) may be provided so as to communicate with the second suction opening (21). In other words, the
10 first fan (79) and the second fan (77) may be of the draw-through type or the forced draft type.

INDUSTRIAL APPLICABILITY

As has been described above, the humidity controller apparatuses of the present invention are useful in dehumidifying or humidifying an indoor space or the like and are
15 particularly suited for a structure where an adsorbent material is supported on the surface of a heat exchanger.

CLAIMS

1. A humidity controller apparatus, comprising:

a first heat exchange chamber (69) for accommodating a first heat exchanger (3),
5 an adsorbent material being supported on a surface of said first heat exchanger (3);

a second heat exchange chamber (73), formed adjacently to said first heat
exchange chamber (69), for accommodating a second heat exchanger (5), an adsorbent
material being supported on a surface of said second heat exchanger (5);

a first air inflow passage (63) and a first air outflow passage (65) which are
10 formed along one end surface in a thickness direction in which respective one surfaces of
said two heat exchange chambers (69, 73) continue and which are arranged in a
superimposed manner in the thickness direction of both said heat exchange chambers (69,
73);

a second air inflow passage (57) and a second air outflow passage (59) which are
15 formed along another end surface which is an end surface where respective one surfaces of
said two heat exchange chambers (69, 73) continue and which is situated face to face with
said one end surface, and which are arranged in a superimposed manner in the thickness
direction of both said heat exchange chambers (69, 73); and

opening/closing means (35, ..., 47, ...) for opening and closing openings (31a, ...,
20 33a, ...) for communication between said first heat exchange chamber (69) and said
second heat exchange chamber (73), and each said inflow passage (57, 63) and each said
outflow passage (59, 65).

2. The humidity controller apparatus of claim 1, wherein:

25 four openings (33a-33d) which allow communication between said first heat

exchange chamber (69) and said second heat exchange chamber(73) and said first air inflow passage (63) and said first air outflow passage (65) are positioned in close proximity to one another in a matrix direction;

four openings (31a-31d) which allow communication between said first heat
5 exchange chamber(69) and said second heat exchange chamber(73) and said second air inflow passage (57) and said second air outflow passage (59) are positioned in close proximity to one another in a matrix direction; and

said eight opening/closing means (35, ..., 47, ...) are each formed by a damper.

10 3. The humidity controller apparatus of claim 1, wherein said first air inflow passage (63) and said first air outflow passage (65) are arranged symmetrically with said second air inflow passage (57) and said second air outflow passage (59).

15 4. The humidity controller apparatus of claim 1, wherein said first heat exchanger (3) and said second heat exchanger (5) are provided in a refrigerant circuit (1) which performs a vapor compression refrigeration cycle by the circulation of a refrigerant therethrough such that refrigerant condensation and evaporation alternately occur in said first heat exchanger (3) and said second heat exchanger (5).

20 5. The humidity controller apparatus of claim 4, wherein dehumidifier means (80) for switching refrigerant circulation in said refrigerant circuit (1) and air distribution is provided so that moisture in a stream of air flowing through a refrigerant evaporating heat exchanger (3, 5) is adsorbed by the adsorbent material of said refrigerant evaporating heat exchanger (3, 5) while the adsorbent material of a refrigerant condensing heat exchanger (5,
25 3) is regenerated by the release of moisture therefrom to a stream of air flowing through

said refrigerant condensing heat exchanger (5, 3), whereby the stream of air dehumidified by the adsorbent material of said refrigerant evaporating heat exchanger (3, 5) is supplied to an indoor space.

5 6. The humidity controller apparatus of claim 4, wherein humidifier means (81) for switching refrigerant circulation in said refrigerant circuit (1) and air distribution is provided so that moisture in a stream of air flowing through a refrigerant evaporating heat exchanger (3, 5) is adsorbed by the adsorbent material of said refrigerant evaporating heat exchanger (3, 5) while the adsorbent material of a refrigerant condensing heat exchanger (5, 10 3) is regenerated by the release of moisture therefrom to a stream of air flowing through said refrigerant condensing heat exchanger (5, 3), whereby the stream of air humidified by the adsorbent material of said refrigerant evaporating heat exchanger (5, 3) is supplied to an indoor space.

15 7. The humidity controller apparatus of claim 4, wherein:

dehumidifier means (80) for switching refrigerant circulation in said refrigerant circuit (1) and air distribution is provided so that moisture in a stream of air flowing through a refrigerant evaporating heat exchanger (3, 5) is adsorbed by the adsorbent material of said refrigerant evaporating heat exchanger (3, 5) while the adsorbent material 20 of a refrigerant condensing heat exchanger (5, 3) is regenerated by the release of moisture therefrom to a stream of air flowing through said refrigerant condensing heat exchanger (5, 3), whereby the stream of air dehumidified by the adsorbent material of said refrigerant evaporating heat exchanger (3, 5) is supplied to an indoor space;

humidifier means (81) for switching refrigerant circulation in said refrigerant 25 circuit (1) and air distribution is provided so that moisture in a stream of air flowing

through a refrigerant evaporating heat exchanger (3, 5) is adsorbed by the adsorbent material of said refrigerant evaporating heat exchanger (3, 5) while the adsorbent material of a refrigerant condensing heat exchanger (5, 3) is regenerated by the release of moisture therefrom to a stream of air flowing through said refrigerant condensing heat exchanger (5, 3), whereby the stream of air humidified by the adsorbent material of said refrigerant evaporating heat exchanger (5, 3) is supplied to an indoor space; and

said dehumidifier means (80) and said humidifier means (81) are configured to operate switchably between a dehumidification mode of operation and a humidification mode of operation.