

HYBRID DYNAMIC PRESSURE GAS THRUST BEARING

TECHNICAL FIELD

The invention relates to a dynamic pressure gas thrust bearing, in particular to a hybrid dynamic pressure gas thrust bearing, which relates to a rigid characteristic of a high-limit rotating speed of a thrust bearing of a groove-type dynamic pressure gas and the flexible characteristic of the high impact resistance and the load capacity of the foil type dynamic pressure gas thrust bearing, belongs to the technical field of gas bearings.

BACKGROUND TECHNOLOGY

The gas bearing has the advantages of being high in speed, high in precision, resistant to high temperature and small in friction loss, long in service life and the like. After the rapid development of recent decades, the gas bearing has been widely applied to the fields of high-speed supporting, high-precision supporting and the like. At present, various types of gas bearings have been developed, and the gas bearings are mainly divided into a dynamic pressure type and a static pressure type.

The dynamic pressure gas bearing takes gas as a lubricant, and a gas film is formed between the shaft and the bearing. It is a bearing form that the moving face is not in direct contact with the static face, and it has many advantages such as no pollution, low friction loss, wide temperature range, smooth operation, long use time and high working speed. Due to the small friction loss, and not necessary to use liquid lubricating oil, it is widely used in the field of high-speed rotary applications. In particular, it is often used in ultra-high-speed application field which is usually supported by a rolling bearing and where liquid lubricants are not suitable.

The dynamic pressure gas thrust bearing is formed by the relatively moving two working faces forming a wedge-shaped space. When they are relatively moved, the gas is driven by its own viscous action and compressed into the wedge-shaped gap, thereby generating dy-

dynamic pressure to support the load. Different structural forms of gas dynamic pressure thrust bearings due to structural differences, the work process is slightly different. At present, the more common types of dynamic pressure thrust bearing structure are a tilting pad type, a groove type and a foil type.

5 The tilting pad type dynamic pressure gas thrust bearing is a kind of excellent dynamic pressure gas bearing with self-adjusting performance, which can work safely in the space of smaller air gap, and is not sensitive to thermal deformation, elastic deformation and so on, and the machining precision is easy to be guaranteed, and it has the outstanding advantages of being capable of automatically tracking the change of the load. At present, it is mainly used
10 in large-scale high-speed rotating machinery and turbine machinery at home and abroad. But its bearing structure is more complex, the installation process is complex, is higher than the general thrust bearing requirements, thus its application is limited.

 Although the foil type dynamic pressure thrust bearing with elastic support, can make the bearing to obtain a certain bearing capacity and alleviate the impact vibration capacity, but the
15 foil bearing is usually made of metal foil sheets, there are still some problems in the material manufacturing technology and processing technology, the damping value of the bearing can not be greatly improved, so that the rigidity of the bearing is not enough, the critical rotating speed of the bearing is low, and the bearing is easy to instability and even stuck during high-speed operation.

20 The groove type dynamic pressure gas thrust bearing has good stability, and the certain stability is achieved even under no load; moreover, at a high speed, the static bearing capacity is larger than that of other forms of bearings. At present, it is mostly used for small-sized high-speed rotating machines, and used as a bearing in precision machinery, such as a gyroscope and a magnetic drum. However, due to the fact that the groove type dynamic pressure gas thrust bearing is high in rigidity, so that the impact resistance is not good enough, the
25 load capacity is not large enough, and high-speed operation under a large load cannot be realized.

How to realize the hybrid dynamic pressure gas thrust bearing which has the rigid char-

acteristics of high-limit rotating speed of the thrust bearing of groove-type dynamic pressure gas, and has the flexible characteristic of the high impact resistance and the load capacity of the foil type dynamic pressure gas thrust bearing, is not only a target which is expected to be realized by researchers in the field, but also to achieve the application of dynamic pressure thrust bearings in the larger load under the ultra-high-speed field has important value and far-reaching significance.

CONTENT OF THE INVENTION

In view of the problems and requirements of the prior art discussed above, it is an object of the present invention to provide hybrid dynamic pressure gas thrust bearing which has the rigid characteristics of high-limit rotating speed of the thrust bearing of groove -type dynamic pressure gas, and has the flexible characteristic of the high impact resistance and the load capacity of the foil type dynamic pressure gas thrust bearing, and to achieve the application of dynamic pressure gas thrust bearings in the ultra-high-speed areas under larger load.

To achieve the above object, the subject matter of the present invention is as follow:

A hybrid dynamic pressure gas thrust bearing, comprises two outer discs, an inner disc clamped between the two outer discs, and foil type elastic parts arranged between each outer disc and the inner disc; and the two end faces of the inner disc are respectively provided with regular groove-type patterns, and the groove-type pattern in one end face is in mirror symmetry with the groove-type pattern in the other end face.

An implementation scheme, groove-type patterns are also arranged in the outer circumferential face of the inner disc, and the shape of the groove-type pattern in the outer circumferential face is the same as the shape of the groove-type pattern in the left and right end faces, and the axial contour lines of the groove-type patterns in the outer circumferential face and the radial contour lines of the groove-type patterns in the left and right end faces are in one-to-one correspondence and are connected with each other.

An implementation scheme, the axial high-position lines of the groove-type patterns in the outer circumferential face correspond to the radial high-position lines of the groove-type

patterns in the left and right end faces, and are connected with each other in front of the circumferential chamfer in the end face; and the axial middle-position lines of the groove-type patterns in the outer circumferential face correspond to the radial middle-position lines of the groove-type patterns in the left and right end faces, and are connected with each other in front of the circumferential chamfer in the end face; and the axial low-position lines of the groove-type patterns in the outer circumferential face correspond to the radial low-position lines of the groove-type patterns in the left and right end faces, and are connected with each other in front of the circumferential chamfer in the end face.

An implementation scheme, the groove-type pattern is of an impeller shape.

As a preferable scheme, a fit clearance between the foil type elastic part and the inner disc is 0.003-0.008 mm.

As a preferable scheme, at least one end of the foil type elastic part is fixed in the inner end face of the corresponding outer disc.

As a preferable scheme, the number of the foil type elastic parts on each outer disc is multiple, and the foil type elastic parts are uniformly distributed along the inner end face of the corresponding outer disc.

As a more preferable scheme, the foil type elastic part fixed on one outer disc is in mirror symmetry with the foil type elastic part fixed on the other outer disc.

As a more preferable scheme, a clamping groove used for fixing the foil type elastic part is formed in the inner end face of the outer disc.

As a preferable scheme, the foil type elastic part is subjected to surface heat treatment.

An implementation scheme, the foil type elastic part is composed of a wave foil and a flat foil, the top end of the arc-shaped protrusion of the wave foil is attached to the flat foil, the transition bottom edge of the wave arch of the wave foil is attached to the inner end face of the corresponding outer disc.

Compared to the prior art, the present invention offers the following benefits:

The foil type elastic part is arranged between the outer disc and the inner disc, and groove-type patterns with regular shape are arranged in the left and right end faces of the inner disc, and the groove-type pattern in one end face is in mirror symmetry with the
5 groove-type pattern in the other end face, so that the hybrid dynamic pressure gas thrust bearing is obtained, which has the rigidity characteristic of the high-limit rotating speed of the groove-type dynamic pressure gas thrust bearing, and has the flexible characteristic of the high impact resistance and the load capacity of the foil type dynamic pressure gas thrust bearing. Compared with an existing simple groove-type dynamic pressure gas thrust bearing,
10 it has the anti-impact capacity and the load capacity which are increased in a multiplied mode at the same rotating speed; and compared with an existing pure foil type dynamic pressure gas thrust bearing, it has the limit rotating speed which is increased by times under the same load; through the test, the hybrid dynamic pressure gas thrust bearing provided by the invention can be realized the limit rotating speed of 200, 000 rpm to 450,000 rpm under a load of
15 1-3 kg, but the existing dynamic pressure gas thrust bearing can only achieve a load capacity of 0.5-1.5 kg, the maximum limit rotating speed can only reach 100,000 rpm to 200,000 rpm. It can be seen that the present invention can realize the application of dynamic pressure gas thrust bearings in ultra high-speed areas under larger loads. Compared with the prior art, the present invention has made significant progress, and made the study of the dynamic bearing
20 gas thrust bearing technology take a new step.

BRIEF DESCRIPTION OF THE DRAWINGS

FIG. 1 is a schematic cross-sectional view of a hybrid dynamic pressure gas thrust bearing according to embodiment 1 of the present invention.

FIG. 2a is a left view of the inner disc of embodiment 1.

25 FIG. 2b is a right view of the inner disc of embodiment 1.

FIG. 3a is a right view of the left outer disc with a foil type elastic part fixed in a first embodiment.

FIG. 3b is left view of the right outer disc with a foil type elastic part fixed in a first em-

bodiment.

FIG. 4 is a schematic cross-sectional view of a foil type elastic part as described in embodiment 1.

5 FIG. 5 is a three-dimensional schematic diagram of a foil type elastic part as described in embodiment 1.

FIG. 6a is a left-view three-dimensional schematic diagram of a hybrid dynamic pressure gas thrust bearing according to embodiment 2 of the present invention.

FIG. 6b is a right-view three-dimensional schematic diagram of a hybrid dynamic pressure gas thrust bearing provided by embodiment 2.

10 FIG. 7 is a schematic diagram of a partial division three-dimensional of a hybrid dynamic pressure gas thrust bearing provided by embodiment 2.

FIG. 8 is a schematic diagram of a left-view three-dimensional of the inner disc according to embodiment 2.

FIG. 9 is an enlarged view of the part A in FIG. 8.

15 FIG. 10 is a schematic diagram of a right-view three-dimensional of the inner disc according to embodiment 2.

FIG. 11 is an enlarged view of the part B in FIG. 10.

Reference numerals in the figures denote as follows:

1, outer disc; 11, left outer disc; 12, right outer disc; 13, clamping groove; 2, inner disc; 21, 20 groove-type pattern in the left end face; 211, radial high-position line; 212, radial middle-position line; 213, radial low-position line; 22, groove-type pattern in the right end face; 221, radial high-position line; 222, radial middle-position line; 223, radial low-position line; 23, groove-type pattern in the outer circumferential face; 231, axial high-position line; 232, axial middle-position line; 233, axial low-position line; 3, foil type elastic part; 3a, foil type elastic part 25 fixed on the left outer disc; 3b, foil type elastic part fixed on the right outer disc; 31, wave foil; 311, arc-shaped protrusions; 312, transition bottom edges between the wave arches; 32, flat foils; 33, fixed end.

DETAILED DESCRIPTION

The subject matter of the present invention will be described in further detail below with reference to several embodiments and the accompanying drawings.

EXAMPLE 1

5 As shown in FIG. 1, the hybrid dynamic pressure gas thrust bearing provided by the embodiment of the invention, includes two outer discs 1, an inner disc 2 is clamped between the two outer discs 1, a foil type elastic part 3 is arranged between each outer disc 1 and the inner disc 2, the regular groove-type patterns 21 are arranged in the left end face and a regular groove-type patterns 22 are arranged in the right end face of the inner disc 2.

10 With combined reference to Figs. 2a and 2b, it can be seen that: the groove-type patterns 21 in the left end face of the inner disc 2 are in mirror symmetry with the groove-type patterns 22 in the right end face of the inner disc 2. The radial contour lines of the groove-type patterns 21 in the left end face and the groove-type patterns 22 in the right end face are in one-to-one correspondence and are connected with each other. The groove-type patterns 21 and 22 are
15 the same in shape. In the embodiment, the shape of the impeller is the shape of an impeller.

Further, in combination with FIG. 3a and FIG. 3b, it can be seen that: the foil type elastic part 3 is fixed in the inner end face of the corresponding outer disc 1 (for example, the left outer disc 11 fixed with a foil type elastic part 3a in FIG. 3a and the right outer disc 12 fixed with a foil type elastic part 3b in FIG. 3b), and the foil type elastic part 3a fixed on the left outer
20 disc 11 is in mirror symmetry with the foil type elastic part 3b fixed on the right outer disc 12. The number of the foil type elastic parts on each outer disc may be plural (four as shown in FIG. 3a and FIG. 3b), they are uniformly distributed along the inner end face of the outer disc.

In combination with FIG. 1 and FIG. 4, FIG. 5, the foil type elastic part 3 can be composed of a wave foil 31 and a flat foil 32, the top end of the arc-shaped protrusion 311 of the
25 wave foil 31 is attached to the flat foil 32, the wave arch transition bottom edge 312 of the wave foil 31 is attached to the inner end face of the corresponding outer disc 1; at least one end of each foil type elastic part 3 is fixed in the inner end face of the corresponding outer disc (shown in the embodiment, one end of each foil type elastic part 3 is fixed, as shown the 33 in

the figure, and the other end is a free end).

EXAMPLE 2

As shown in FIGS. 6a, 6b, 7, 8 and 10, the only difference from the hybrid dynamic pressure gas thrust bearing in the first embodiment is that the groove-type patterns 23 are also arranged in the outer circumferential face of the inner disc 2, and the shape of the groove-type pattern 23 in the outer circumferential face is the same as the shapes of the groove-type patterns (21, 22) in the left and the right end faces (in the embodiment, the shape of the groove-type patterns are all the shape of the impeller), and an axial contour lines of the groove-type patterns 23 in the outer circumferential face and the groove-type patterns (21, 22) in the left and the right end faces are in one-to-one correspondence and are connected with each other; namely,

The axial high-position lines 231 of the groove-type patterns 23 in the outer circumferential face correspond to the radial high-position lines 211 of the groove-type patterns 21 in the left end face, and are connected with each other in front of the circumferential chamfer in the end face; and the axial middle-position lines 232 of the groove-type patterns 23 in the outer circumferential face correspond to the radial middle-position lines 212 of the groove-type patterns 21 in the left end face, and are connected with each other in front of the circumferential chamfer in the end face; and the axial low-position lines 233 of the groove-type patterns 23 in the outer circumferential face correspond to the radial low-position lines 213 of the groove-type patterns 21 in the left end face, and are connected with each other in front of the circumferential chamfer in the end face (as shown in FIG. 9);

The axial high-position lines 231 of the groove-type patterns 23 in the outer circumferential face correspond to the radial high-position lines 221 of the groove-type patterns 22 in the right end face, and are connected with each other in front of the circumferential chamfer in the end face; and the axial middle-position lines 232 of the groove-type patterns 23 in the outer circumferential face correspond to the radial middle-position lines 222 of the groove-type patterns 22 in the right end face, and are connected with each other in front of the circumferential chamfer in the end face; and the axial low-position lines 233 of the groove-type patterns 23 in the outer circumferential face correspond to the radial low-position lines 223 of the

groove-type patterns 22 in the right end face, and are connected with each other in front of the circumferential chamfer in the end face (as shown in FIG. 11).

The inner end face of the outer disc 1 is provided with a clamping groove 13 for fixing the foil type elastic part 3 (as shown in FIG. 7).

5 By providing with a foil type elastic part 3 arranged between the outer disc 1 and the inner disc 2, and the groove-type patterns of regular shape arranged in the left end face and the right end face of the inner disc 2, and the groove-type pattern 21 in the left end face is in mirror symmetry with the groove-type pattern 22 in the right end face, so that the hybrid dynamic pressure gas thrust bearing is obtained, which has the rigidity characteristic of the high-limit
10 rotating speed of the groove-type dynamic pressure gas thrust bearing, and has the flexible characteristic of the high impact resistance and the load capacity of the foil type dynamic pressure gas thrust bearing; Due to a wedge-shaped space is formed between the foil type elastic part 3 and the inner disc 2, when the inner disc 2 rotates, the gas is driven and compressed into the wedge-shaped space due to the viscosity action of the gas, so that the axial
15 dynamic pressure can be obviously enhanced. Compared with an existing simple foil type dynamic pressure gas thrust bearing, the hybrid dynamic pressure gas thrust bearing can increase the limit rotating speed by times under the same load; meanwhile, due to the foil type elastic part 3, under the elastic action of the foil type elastic part 3, the load capacity, the impact resistance and the capability of inhibiting the axial vortex are all remarkably improved.
20 Compared with an existing simple groove-type dynamic pressure gas thrust bearing, the hybrid dynamic pressure gas thrust bearing can increase the impact resistance and the load capacity in a multiplied mode at the same rotating speed. Particularly, when the groove-type patterns 23 are also arranged in the outer circumferential face of the inner disc 2, and the shape of the groove-type pattern 23 in the outer circumferential face is the same as the
25 groove-type pattern 21 in the left end face and the groove-type pattern 22 in the right end face, and the axial contour lines of the groove-type patterns 23 in the outer circumferential face and the radial contour lines of the groove-type patterns(21, 22) in the left end and the right end faces are in one-to-one correspondence and are connected with each other, so that the pressurized air generated by the groove-type patterns (21 and 22) in the left and right end
30 faces of the inner disc 2 is continuously conveyed from a center of the shaft along the radial

direction into groove-type-like channels formed by the groove-type patterns 23 in the outer circumferential face, and the required air film to provide a stronger support to the high-speed running bearing can be formed, and the air film serves as a lubricant for the dynamic pressure gas thrust bearing, then the high-speed stable operation of the hybrid dynamic pressure gas thrust bearing under the air floatation state can be ensured, and provides a further guarantee for achieving the high-limit rotating speed.

In addition, the foil type elastic part 3 is preferably subjected to surface heat treatment, so that the performance requirements of high-speed operation can be better met; the matching clearance between the foil type elastic part 3 and the inner disc 2 is preferably 0.003-0.008 mm, so that the reliability and the stability of high-speed operation of the bearing can be further ensured.

Other, the structure of the foil type elastic part 3 is not limited to that of the above embodiment, so as to meet the substantive requirements of the present invention as long as the matching relation between the inner disc and the outer disc is ensured.

Through the test, the hybrid dynamic pressure gas thrust bearing provided by the invention can be realized the limit rotating speed of 200, 000 rpm to 450,000 rpm under a load of 1-3 kg, but the existing dynamic pressure gas thrust bearing can only achieve a load capacity of 0.5-1.5 kg, the maximum limit rotating speed can only reach 100,000 rpm to 200,000 rpm. It can be seen that the present invention can realize the application of dynamic pressure gas thrust bearings in ultra high-speed areas under larger loads. Compared with the prior art, the present invention has made significant progress, and made the study of the dynamic bearing gas thrust bearing technology take a new step.

At last, it is noted here that the above disclosure is intended merely to describe the subject matter of the present invention in further detail and is not to be construed as limiting the scope of the invention. Non-substantive improvements and modifications made by those skilled in the art in accordance with the above disclosure all fall within the scope of the present invention.

1. A hybrid dynamic pressure gas thrust bearing, comprising:
two outer discs, and
an inner disc clamped between the two outer discs,
characterized in that
- 5 foil type elastic parts are arranged between each one of the two outer discs and the inner disc;
two end faces of the inner disc are respectively provided with regular groove-type patterns;
the regular groove-type patterns on one of the two end face is in mirror symmetry with
- 10 another one of the regular groove-type patterns on another end face;
the regular groove-type patterns are also arranged on the outer circumferential face of the inner disc;
a shape of the regular groove-type pattern on the outer circumferential face is the same as a shape of the regular groove-type pattern on a left end and a right end of the two end
- 15 faces; and
axial contour lines of the regular groove-type patterns on the outer circumferential face and radial contour lines of the regular groove-type patterns on a left end and a right end of the two end faces are connected with each other in one-to-one correspondence.
- 20 2. The hybrid dynamic pressure gas thrust bearing according to claim 1,
characterized in that
axial high-position lines of the regular groove-type patterns on the outer circumferential face correspond to radial high-position lines of the regular groove-type patterns on the left end face and the right end face, and are connected with each other in front of a
- 25 circumferential chamfer on one of the two end faces;
axial middle-position lines of the regular groove-type patterns on the outer circumferential face correspond to radial middle-position lines of the regular groove-type patterns on the left end face and the right end face, and are connected with each other in front of the circumferential chamfer on one of the two end faces; and
- 30 axial low-position lines of the regular groove-type patterns on the outer circumferential face correspond to radial low-position lines of the regular groove-type patterns on the left end face and the right end face, and are connected with each other in front of the circumferential chamfer on one of the end faces.

3. The hybrid dynamic pressure gas thrust bearing according to claim 1 or 2,
characterized in that:
the regular groove-type patterns are of impeller shape.
- 5 4. The hybrid dynamic pressure gas thrust bearing according to any of the preceding
claims,
characterized in that
a fit clearance between the foil type elastic parts and the inner disc is 0.003-0.008 mm
substantially.
- 10 5. The hybrid dynamic pressure gas thrust bearing according to any of the preceding
claims,
characterized in that
at least one end of the foil type elastic parts is fixed in an inner end face of the two
15 corresponding outer discs.
6. The hybrid dynamic pressure gas thrust bearing according to claim 5,
characterized in that
a number of the foil type elastic parts on each one of the corresponding outer discs is
20 multiple, and the foil type elastic parts are uniformly distributed along the inner end face of
the two corresponding outer discs.
7. The hybrid dynamic pressure gas thrust bearing according to claim 5 or 6,
characterized in that
25 the foil type elastic parts fixed on one of the two outer disc are in mirror symmetry with the
foil type elastic parts fixed on another one of the two outer discs.
8. The hybrid dynamic pressure gas thrust bearing according to claim 5 or 6,
characterized in that
30 a clamping groove used for fixing the foil type elastic parts is formed on the inner end face
of the two corresponding outer discs.

9. The hybrid dynamic pressure gas thrust bearing according to any of the preceding claims,
characterized in that
5 the foil type elastic parts are subjected to surface heat treatment.
10. The hybrid dynamic pressure gas thrust bearing according to claim 1, 4, 5, 6 or 9,
characterized in that
the foil type elastic parts comprise a wave foil and a flat foil;
10 a top end of an arc-shaped protrusion of the wave foil is attached to the flat foil; and
a transition bottom edge between wave arches of the wave foil is attached to the inner
end face of the two corresponding outer discs.

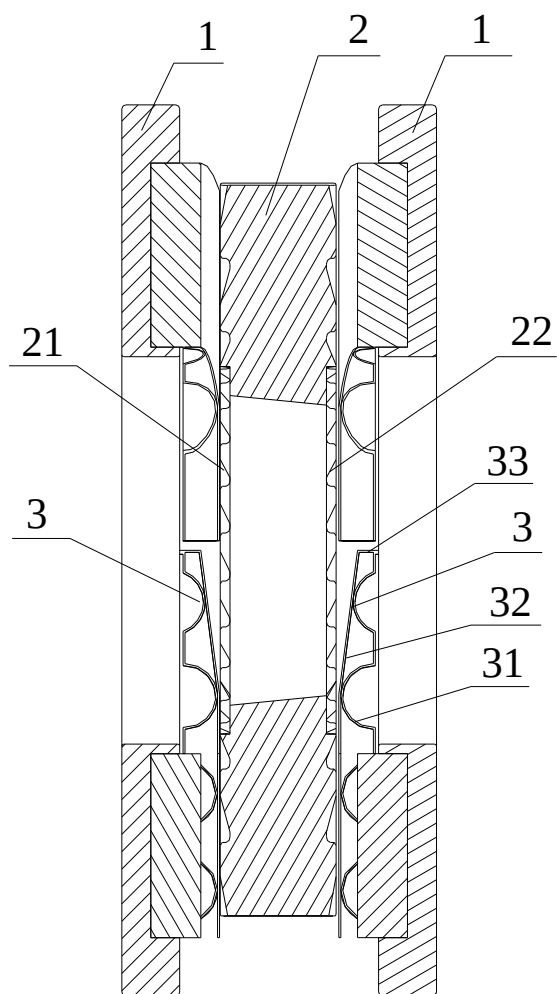


Figure 1

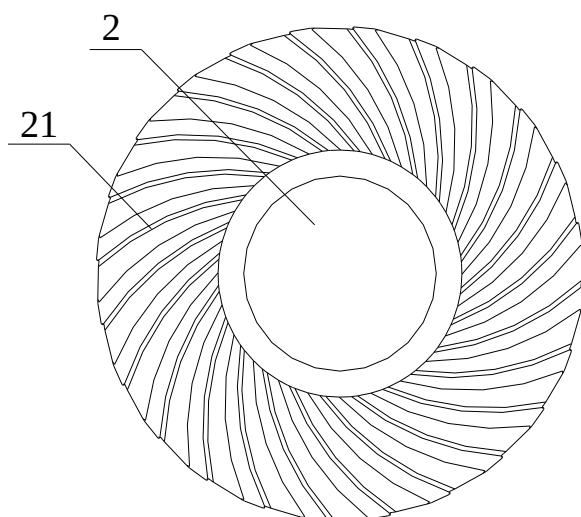


Figure 2a

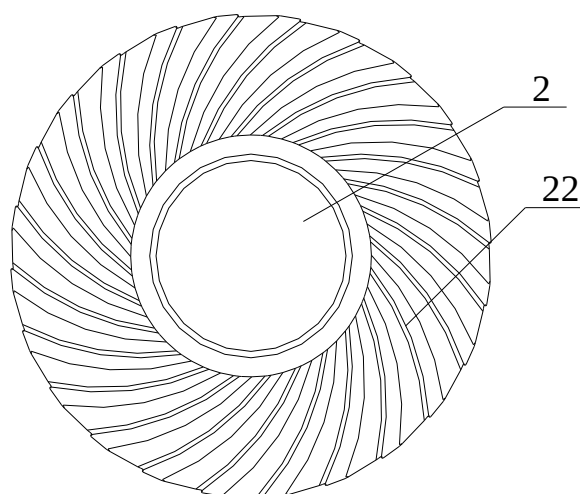


Figure 2b

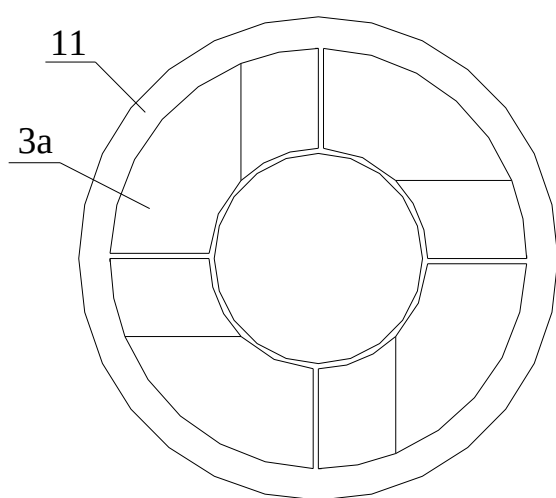


Figure 3a

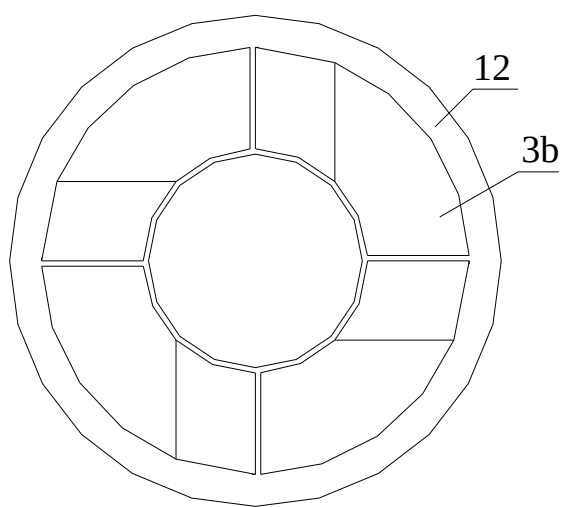


Figure 3b

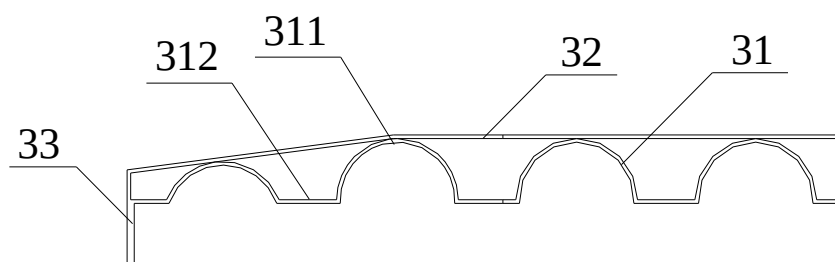


Figure 4

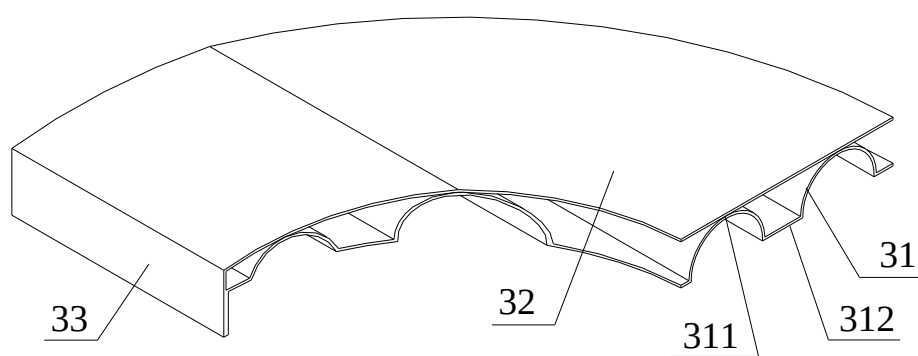


Figure 5

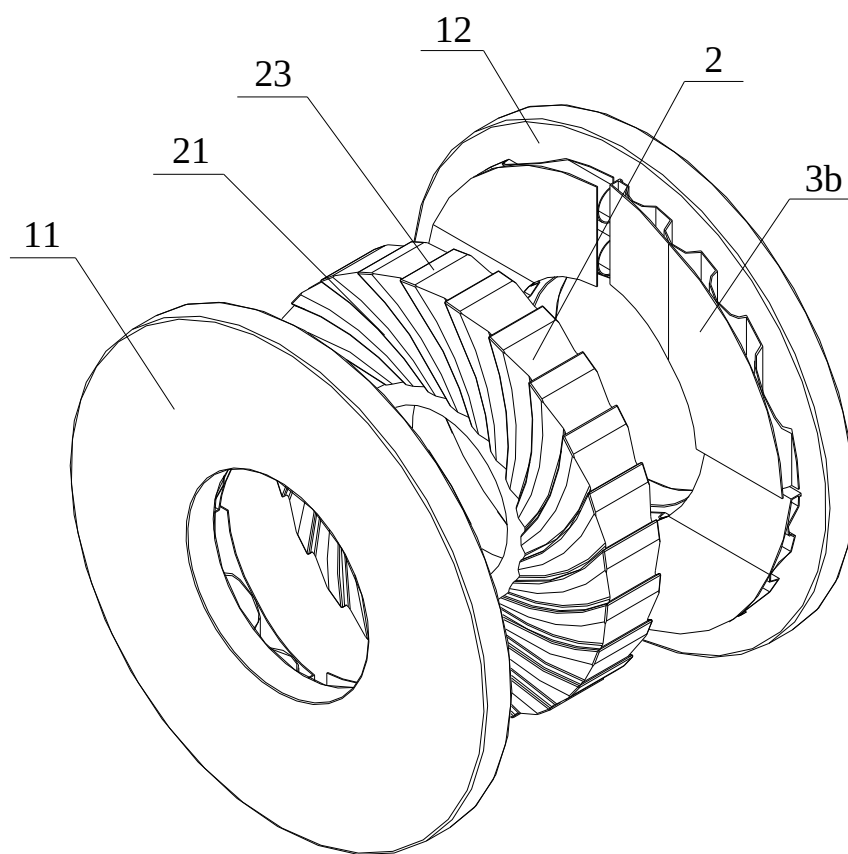


Figure 6a

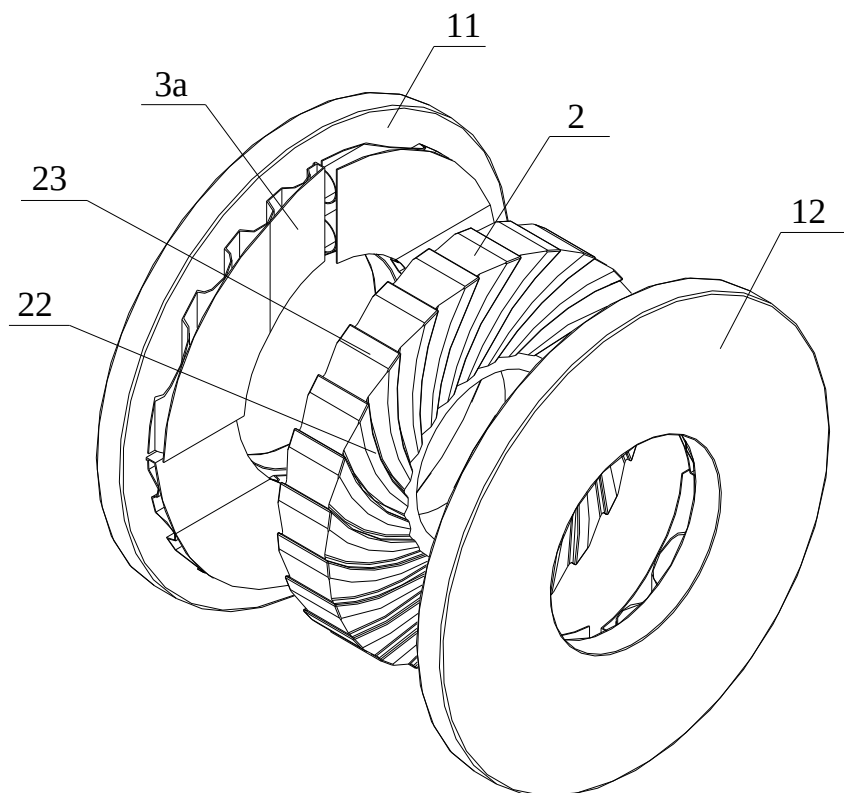


Figure 6b

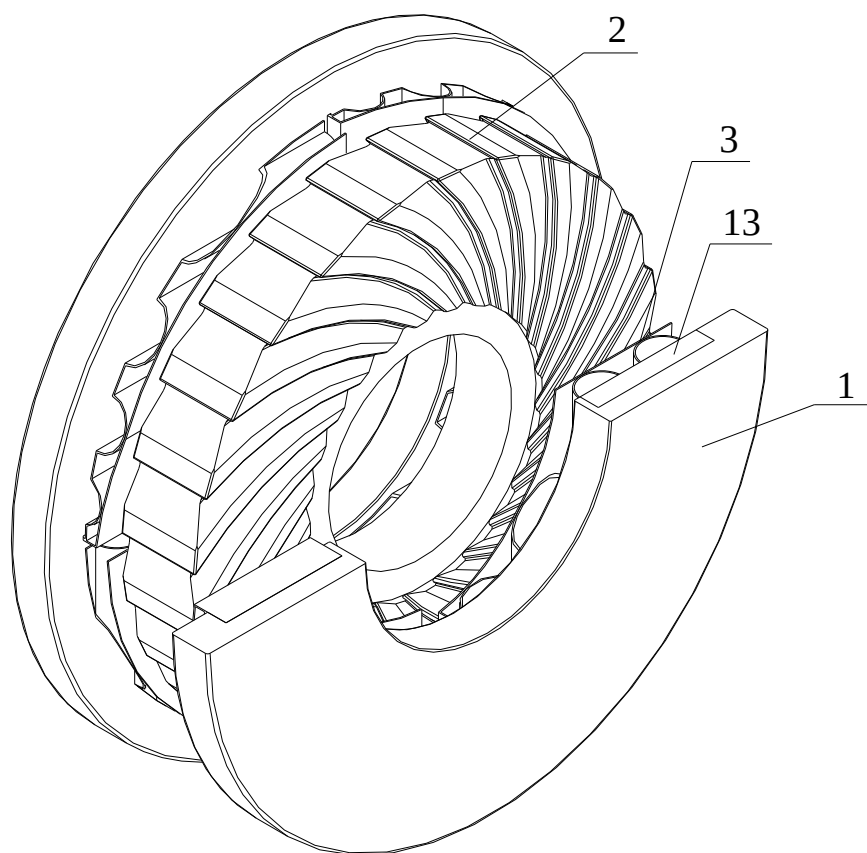


Figure 7

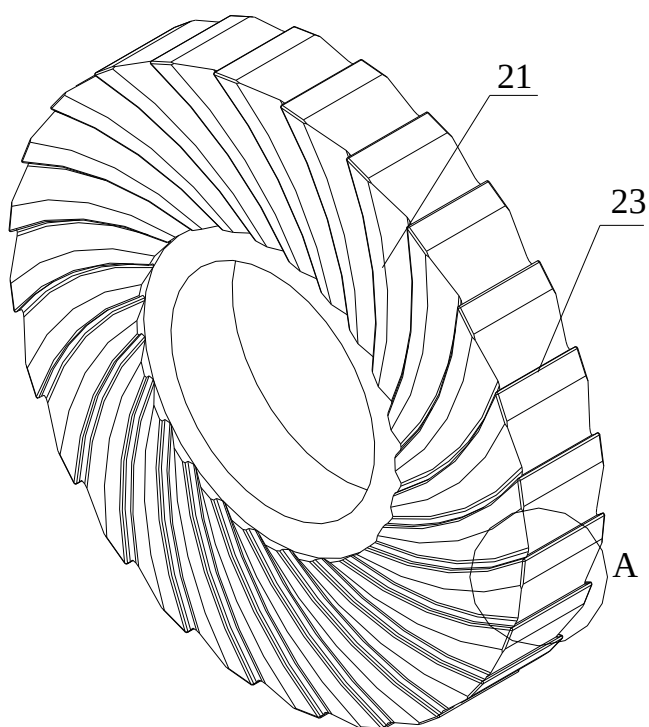


Figure 8

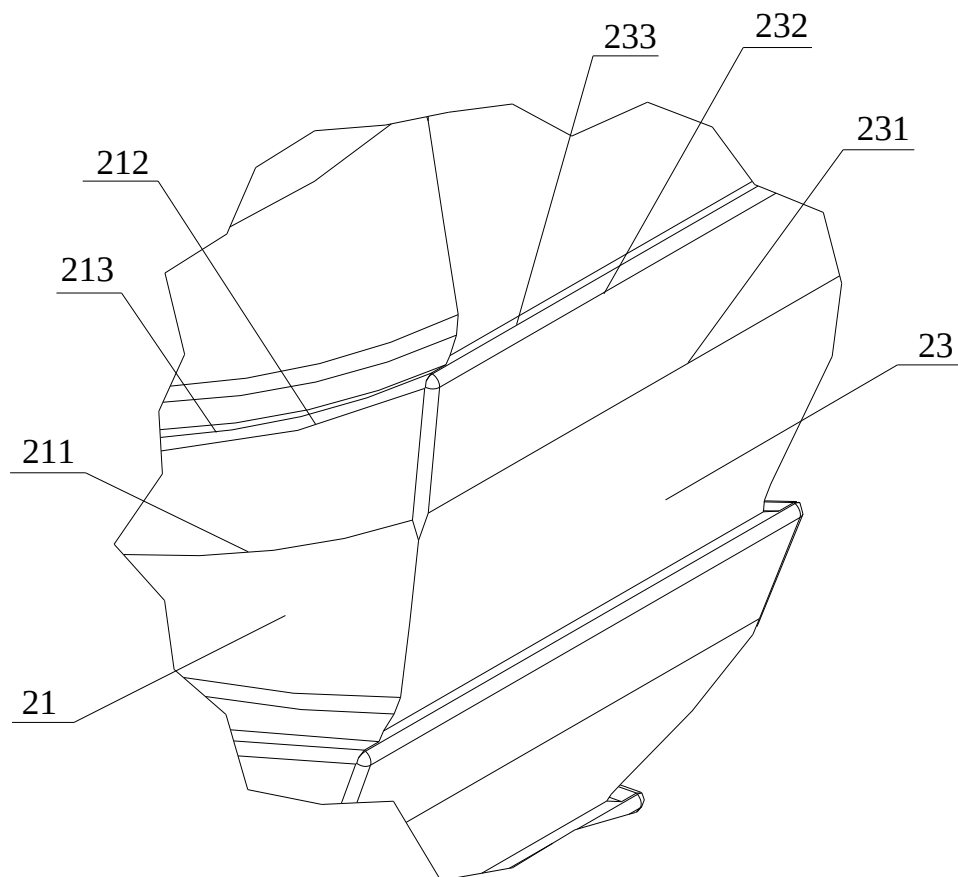


Figure 9

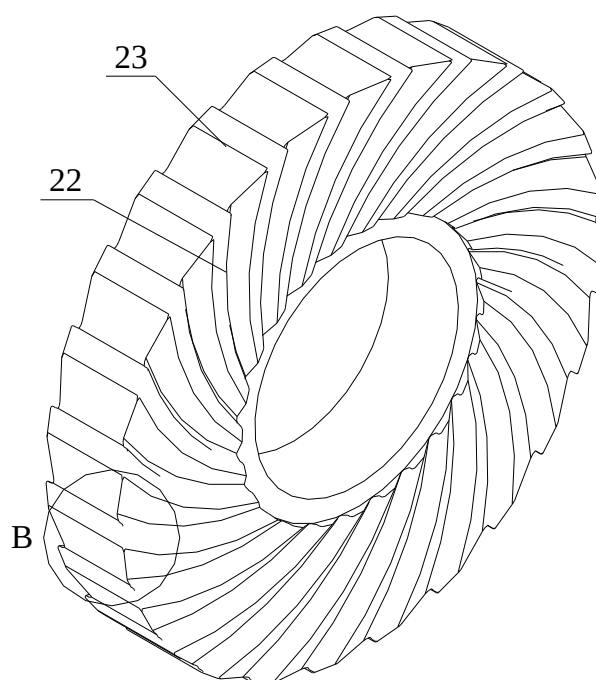


Figure 10

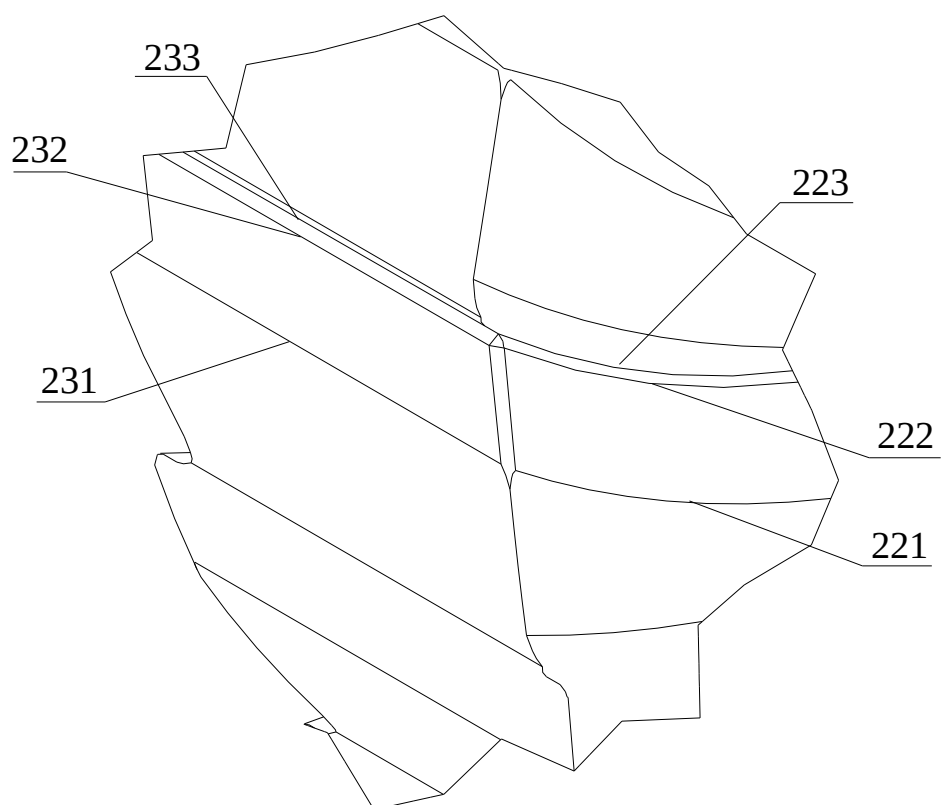


Figure 11