

TITLE

5

Diaphragm vacuum pump

TECHNICAL FIELD

10 The present invention relates to a diaphragm vacuum pump according to the preamble of patent claim 1.

PRIOR ART

15 Diaphragm vacuum pumps of this type are used for various medicinal applications, in particular for drainage applications, such as wound drainage or thorax drainage. They are also well known as breastpumps for expressing human breast milk. Examples thereof include WO 96/22116, US 2009/0099511, US 2008/0287037, US 7 094 217 and US 2008/0039781.

20

The drive customarily consists of an electric motor which transmits the rotational movement of the motor to the diaphragm via an eccentric, a connecting rod or another force transmission unit and cyclically deflects said diaphragm. A disadvantage of many such pumps is that they are relatively large and loud. However, in particular in “hand-free” applications, the pump should be designed to be as discrete as possible and therefore as small and quiet as possible. In this context, “hands-free” means that, after being switched on, the entire device functions without hands, *i.e.* neither the pump nor the breast shield have to be held by hand.

25 30 WO 02/102437 and WO 2008/137678 show examples of “hands-free” expressing devices. The breast shield here is in each case integrated in a pump housing and serves at the same time as a diaphragm for generating a negative pressure. A stepper motor is used in each case as the drive.

SUMMARY OF THE INVENTION

- 5 It is an object of the invention to provide a vacuum pump which is designed to be as small as possible.

This object is achieved by a device having the features of claim 1.

- 10 The diaphragm vacuum pump according to the invention has an electrically operated drive unit and a diaphragm which separates a pump chamber into a drive-side part and a drive-remote part and which can be deflected by means of the drive unit. According to the invention, the drive unit is an electromagnetic drive unit and the membrane is deflected in the direction of a linear movement that is generated electromagnetically in the drive unit.

15

Drive units of this type can be of very small design. Owing to the direct coupling to the diaphragm and the deflection of the diaphragm in the same direction as the electrically movable part of the unit, complicated and space-consuming force transmission elements are also not necessary.

20

- It is furthermore advantageous that, in comparison to the otherwise customary rotating drives of diaphragm vacuum pumps, said linear movement is quieter and fewer vibrations and structure-borne sound are generated. In addition, in contrast to the known diaphragm vacuum pumps, the stroke length can be changed. It can be controlled in particular
25 electronically. This permits precise control even under high vacuum levels.

- In a preferred embodiment, the drive unit has at least one permanent magnet and a coil former with a coil, wherein the coil is arranged on the coil former, wherein the coil former together with the coil is held in a linearly displaceable manner along a longitudinal axis
30 with respect to the magnet, and wherein the coil former is connected to the diaphragm and deflects the latter during the abovementioned displacement of the coil former in the direction of the displacement of said coil former.

The coil former preferably has a length which is much greater than the width thereof. It may be formed as a single piece or in a number of pieces. The diaphragm preferably has a diameter that is much larger than the piston. The diaphragm is preferably of substantially circular design and is composed of a thin material, in particular of silicone.

5

The coil former preferably serves as piston for the deflection of the diaphragm. The piston has a first end at which the diaphragm is arranged. Force is therefore transmitted very directly and precisely. The deflection of the diaphragm can be controlled in a specific manner and the vacuum pump operates within a very narrow tolerance limit. The precision
10 can be further increased if the diaphragm is fastened centrally to the first end of the piston.

The vacuum pump typically operates up to a negative pressure of 0 to 300 mmHg . The diaphragm can be driven at a frequency of 5 to 120 cycles per minute.

15 The piston has a second end, which, in a first preferred embodiment, is held in a linearly displaceable manner.

As an alternative, a second diaphragm can also be fastened to said second end. Said second diaphragm is preferably formed identically to the first diaphragm and is likewise arranged
20 in a pump chamber. There is therefore a double pump, which is of mirror-symmetrical design preferably centrally with respect to the longitudinal axis thereof.

In a preferred embodiment, the movable part of the unit is operatively connected to a ventilation valve, and therefore the linear movement of the movable part of the unit is
25 optionally also used at the same time in order to actuate the ventilation valve. In a preferred embodiment, the ventilation valve is arranged for this purpose at that end of the movable unit part which is opposite the vacuum diaphragm. Upon an appropriate stroke, the movable unit part opens the ventilation valve. For this purpose, the unit part preferably has at least one pin which acts on at least one valve flap such that the latter opens up a
30 ventilation opening. The valve flap is preferably part of a ventilation diaphragm, that is fastened to the movable unit part.

Since the movable unit part may also be used for actuating the ventilation valve, the

construction of the unit is simplified, and relatively few individual parts are required. It is advantageous that a separate motor and a separate controller are not required in order to actuate the ventilation valve. This increases the reliability and reduces the costs.

- 5 It is furthermore advantageous that both diaphragms, i.e. the vacuum diaphragm and the ventilation diaphragm, serve as a mounting for the movable part of the unit, thus avoiding friction. Said pump unit can therefore be used for a relatively long time without maintenance.
- 10 In a preferred exemplary embodiment, in particular in the abovementioned examples, the coil is a flat coil and there are at least two permanent magnets which are arranged on either side of the coil and of the coil former. Preferably there is a pair of permanent magnets each on either side of the coil.
- 15 In another preferred exemplary embodiment, the second end of the piston is mounted displaceably between an iron core and a magnet.

The coil may be a moving coil, wherein the moving coil and the permanent magnet are of rotationally symmetrical design and the iron core passes through the moving coil.

20

- There is preferably a position detector for determining the relative position of the coil former with respect to the permanent magnet. As a result, the particular deflection of the diaphragm can also be determined unambiguously. The position detector is preferably an optical sensor. For example, a position scale, which is monitored by the optical sensor, can
- 25 be arranged on the coil former. The position scale can consist, for example, of one, two, or more grayscale bands which move together with the coil former relative to the sensor.

- If the position detector generates a signal, which is used to control the pump, as a function of the relative position of the coil former, the deflection of the diaphragm can be controlled
- 30 very precisely. The vacuum pump operates within a very narrow tolerance range with respect to frequency and amplitude, *i.e.* with respect to the vacuum obtained.

Said vacuum pump is suitable for a very wide variety of areas of application, in particular

for medicinal use. A further preferred application is the expressing of human breast milk, *i.e.* in the area of breastpumps. Preferred applications are thorax drainage and wound drainage.

- 5 In a preferred breastpump, which has the unit according to the invention, there is an outlet in the drive-remote part of the pump chamber, said outlet being connected to an inlet of a second chamber, wherein the chamber has a second diaphragm which separates said chamber into two parts. This diaphragm serves as a means of separating media and for transferring the generated vacuum outward. Said second diaphragm can give rise to a
- 10 system which changes from an initial pneumatic negative pressure in a breast shield bearing against the breast to a hydraulic negative pressure. In this case, milk which is already being expressed acts as fluid of the hydraulic system which expresses further milk out of the breast.
- 15 However, the vacuum pump according to the invention can also be used for breastpumps which apply a cyclic vacuum to the breast shield and in which the milk passes into a milk collecting container via a path that is separate from the air line.

Further advantageous embodiments are specified in the dependent claims.

20

BRIEF DESCRIPTION OF THE DRAWINGS

- Preferred embodiments of the invention are described below with reference to the drawings that serve merely for explanation and should not be interpreted as being limiting. In the
- 25 drawings:

- Figure 1 shows a longitudinal section through a vacuum pump according to the invention;
- 30 Figure 2 shows a longitudinal section through a vacuum pump according to the invention in a second embodiment;

- Figure 3 shows an exploded view of a device for expressing breast milk with the vacuum pump according to the invention according to figure 1;
- Figure 4 shows a device for expressing breast milk in a second embodiment;
- 5 Figure 5 shows an exploded view of a vacuum pump according to the invention in a first embodiment with part of the device according to figure 3;
- Figure 6 shows a longitudinal section through part of the device according to figure 5;
- 10 Figure 7 shows a schematic view of a vacuum pump according to the invention in a third embodiment including the control unit;
- 15 Figure 8 shows an exploded view of the vacuum pump according to figure 7;
- Figure 9 shows a side view of the vacuum pump according to figure 7 in a partially assembled state;
- 20 Figure 10 shows a longitudinal section through the vacuum pump according to figure 7 in a first position during the generation of the vacuum;
- Figure 11 shows an enlarged detail A according to figure 10 with a closed ventilation flap;
- 25 Figure 12 shows an enlarged detail E according to figure 10 with an open exhaust flap and closed vacuum flap.
- Figure 13 shows a longitudinal section through the vacuum pump according to figure 7 in a second position during the generation of a vacuum;
- 30 Figure 14 shows an enlarged detail D according to figure 13 with a closed exhaust flap and open vacuum flap;

- Figure 15 shows a longitudinal section through the vacuum pump according to figure 7 during the ventilation;
- 5 Figure 16 shows an enlarged detail B according to figure 15 with an open ventilation flap, and
- Figure 17 shows an enlarged detail C according to figure 15.
- 10 The same parts are provided with the same reference numbers in the figures.

DESCRIPTION OF PREFERRED EMBODIMENTS

- 15 Figure 1 illustrates a first embodiment of the vacuum pump according to the invention and of a pump unit according to the invention with an electromagnetic drive, respectively.

The pump unit has a housing 90, which is preferably manufactured from metal or plastic. The housing 90 preferably has a cuboidal design. A flat iron core and an iron plate 911, respectively, and a permanent magnet 91 attached to the latter are arranged on one side of
 20 the housing 90. The iron plate 911 rests upon the housing 90 such that the permanent magnet 91 is in a recess of the housing. The permanent magnet 91 consists of two parts, which are spaced from one another such that there is a recess 910 between both parts.

- 25 An identical construction is also present on an opposite side of the housing. An iron plate 911 and a two-part permanent magnet 91 attached to the latter are also arranged here.

A flat coil former 92 with a coil 921 inserted in the coil former 92 runs between said two mutually opposite pairs of permanent magnets 91.

30

The coil former 92 is of substantially rod-shaped or plate-shaped design. At one end, it is held in a fixed position in a guide bearing 93. The guide bearing 93 is displaceable together with the coil former 92 including the coil 921 relative to the housing 90 and therefore

relative to the permanent magnet 91 along the longitudinal axis of the housing 90. For this purpose, the housing 90 has a plain bearing 900. The movement of the coil former 92 is illustrated by a double arrow in Figure 1.

5 The other end of the coil former 92 is connected fixedly to a diaphragm, referred to here as the vacuum diaphragm 94. The vacuum diaphragm 94 bears against the housing 90 on the end side thereof and is clamped securely between the housing 90 and a valve plate 95. The diaphragm 94 separates a pump chamber 96 from the coil former 92. The vacuum diaphragm 94 preferably has a circular outline, it preferably having a shape customary for
10 diaphragms of diaphragm vacuum pumps. The valve plate 95 is held between the housing 90 and a cover 99. Said three parts are preferably connected, for example screwed, tightly to one another in a detachable or nondetachable manner. The corresponding holes are provided with the reference number 991 in the cover 99. The cover 99 has a port opening 990 for the vacuum line 12' which is connected to the pump chamber 96. An air inlet
15 opening 992 in the cover 99 likewise connects the environment to the pump chamber 96 via the valve plate 95. The valve plate 95 has the valves, inlets and outlets customary for diaphragm vacuum pumps. These are not described in detail here.

If an alternating electric current flows through the coil 921, the electromagnetic field
20 changes and the coil former 92 moves relative to the permanent magnet 91. The coil former 92 acts like a piston or ram and moves the vacuum diaphragm 94 cyclically to and fro. In this case, the force acting on the vacuum diaphragm 94 is proportional to a current applied to the coil. By means of the movement of the vacuum diaphragm 94, a cyclically changing vacuum is built up in the pump chamber 96 and is present at the output 990.

25 The vacuum diaphragm 94 of the pump unit 9 is therefore driven via a linear movement that is generated electromagnetically, wherein the coil former 92 acts as a piston. It is advantageous that, in comparison to the rotating drives that are otherwise customary, the movement is quieter and fewer vibrations and less structure-borne sound are generated. In
30 contrast to the prior art, the stroke length can be changed. It can be electronically controlled. This permits precise control even under low vacuum levels.

In order for the stroke length to be able to be controlled in a specific manner, the

displacement and the position of the coil former 92 is monitored, respectively. Position and/or movement sensors are used for this purpose. In this example, this takes place by means of an optical sensor which detects a position scale. The position scale 920 is preferably provided on the coil former 92. A light source 97 sends its light perpendicularly
 5 to the longitudinal direction of the coil former 92 to an opposite detector 98, said light passing through the position scale 920. The coil former 92 in said region is of a transparent design.

The position scale may be formed, for example, by one, two or more grayscale bars which,
 10 together with the coil former, pass the sensor. If there are two grayscale bars running parallel to each other, wherein one has a rising graduation and one has a falling graduation, and two sensors are used, the difference between said two signals can be used as a very precise determination of the position.

15 The light source 97 and the detector 98 are preferably arranged in the recesses 910 of the permanent magnet 91. Other types of position measurement are possible. The measured signal is sent to an electronic controller of the vacuum pump and the current is applied to the coil according to said signal. As a result, the position, the deflection amplitude, and the frequency can be controlled independently of one another. Vacuum values of 0 to 300
 20 mmHg are customarily obtained. The frequencies are customarily 0 to 150 cycles per minute.

As an alternative, instead of the guide bearing 93, there may also be a second diaphragm which is of similar or identical design to the vacuum diaphragm 94. This results in a
 25 symmetrical construction, which likewise ensures guidance and therefore a linear movement of the coil former 92 within the housing 90. In addition, the second diaphragm can likewise be used to build up the vacuum such that the flow rate can be increased.

Figure 2 shows a second embodiment of a vacuum pump with an electromagnetically
 30 generated linear driving movement of the diaphragm. In contrast to the above-described flat coil, a moving coil is used here. The magnet and coil are of rotationally symmetrical, in particular annular and cylindrical design, respectively. A housing 90' is also present here. A permanent magnet 91' is arranged at the rear end of the moving coil remote from

the breast shield. Said permanent magnet is passed through by an iron core 911', which is in contact with the permanent magnet 91' on a first end side thereof. Against the opposite end side of the permanent magnet bears an iron ring 912'. The coil former 92' engages around the iron core 911' and is axially guided by said iron core. The coil former 92' extends between the iron core 911' and the permanent magnet 91' and the iron ring 912', respectively. Said coil former is guided without contact in the permanent magnet 91' and in the iron ring 912'. The wound coil 921' in this case surrounds the coil former 92' in this region.

The coil former 92' is also fixedly connected here to the vacuum diaphragm 94' and acts as a piston for the linear drive of the vacuum diaphragm 94'. The pump chamber is provided with the reference number 96', the valve plate with the reference number 95' and the cover with the reference number 99'. The port opening has the reference number 990' and the air inlet opening has the reference number 992'. Also here, there is again a position sensor which indicates the position of the coil former 92' relative to the magnet 91' and therefore the movement and the position of the vacuum diaphragm 94', respectively, to a controller in order to control the vacuum. The light source is denoted here by the reference number 97', the detector by 98' and the position scale by 920'. The position scale is preferably in a transparent design. In contrast to the previous example, said sensor here is arranged outside the region of the permanent magnet 91'.

In this embodiment, vacuum values of 0 to 300 mmHg are customarily obtained. The frequencies are customarily 0 to 150 cycles per minutes.

Figures 7 to 17 show a third embodiment of a vacuum pump according to the invention with electromagnetically generated linear driving movement of the diaphragm. Said embodiment essentially constitutes a development of the pump unit according to figure 1.

Figure 7 shows the pump unit 9'' according to third embodiment, said pump unit being connected to a control unit 15 via lines 16 for supplying current and for the transfer of data. The control unit 15 has operating elements 150 for operating the breastpump.

Figure 9 shows said pump unit 9'' in a partially exploded view. The individual parts of the

unit 9'' can readily be seen in figure 8. The pump unit 9'' again has a housing 90'', that is preferably manufactured from metal or plastic. The housing 90'', formed by two side parts 90'', is closed on both sides by a planar cover 99'', 902. The corresponding fastening holes 908 in the housing 90'' and in the covers 99'', 902 can readily be seen in figure 8. The
 5 associated screws are not illustrated.

A coil carrier or coil former 92'' with a flat coil 921'' fastened thereto is arranged in the housing. End windings 922 of the coil 921'' surround a rod-shaped end of the coil former 92''. Said rod-shaped end preferably has a round cross section.

10

The coil 921'' has a central recess, in which a position scale 920'' is arranged. The position of the coil former 92'' relative to the housing can be ascertained by means of an opposite transmitter and receiver or detector 98'' on the basis of said position scale 920''. The transmitter/receiver 98'' preferably comprises a light-emitting diode and a photodiode.
 15 The position scale 920'' may be, for example, a grayscale scale. The ports for an associated measuring board are denoted in the figure by the reference numeral 980. Other means of determining the position are possible.

An iron plate 911'' is arranged on one side of the coil former 92''. There are permanent
 20 magnets 91'' and an iron plate 911'', that is connected to said permanent magnets and connects the permanent magnets 91'' to one another, on the other side. In this embodiment, the permanent magnets 91'' are preferably of cuboidal design. They may also be arranged on both sides of the coil 921'', as in figure 1. Similarly, the magnets 91 of the embodiment according to figure 1 may also be present only on one side of the coil.

25

A vacuum diaphragm 94'' is arranged on one side of the elongate coil former 92'' and is connected fixedly to the latter. The vacuum diaphragm 94'' has a known, preferably round shape with grooves or beads. It is therefore preferably substantially plate-shaped. It is preferably manufactured from an elastic material, in particular from silicone. For the
 30 fastening to the coil former 92'', the diaphragm is provided with a central hole, as can be seen in figure 12.

The vacuum diaphragm 94'' preferably has two wings which are diametrically opposite

each other and are formed integrally with the rest of the diaphragm 94''. Said two wings form an inlet valve flap or vacuum flap 942 and an outlet valve flap 943.

5 The vacuum diaphragm 94'' is fastened to the first end of the coil former 92'' with clamping means. There is a first spacer 940 and a counterpart 941 for this purpose, as can be seen in figures 8 and 12. For this purpose, the counterpart 941 has a plate and a stud which is integrally formed thereon. The first spacer 940 is of cuboidal design with a hole for receiving the stud. The counterpart 941 is arranged on the lower side of the diaphragm 94'', the stud penetrates the diaphragm 94'' and the first spacer 940 and engages in an
10 opening in the lower end of the coil holder 92''. The stud and said opening preferably each have a thread.

A valve plate 95'' is arranged in the cover 99''. Said valve plate may be inserted into a corresponding recess of the cover 99'' or may be formed integrally in said cover. The
15 valve plate 95'' has valve openings and valve channels that are customary for diaphragm pumps and are connected to the valve flaps 942, 943.

A vacuum port 952, which can readily be seen in figures 8 and 12, leads outward such that a vacuum generated in the pump unit, or more precisely in a pump chamber 96'', can be
20 applied externally, for example in a breast shield of a breastpump. Pump chamber 96'' and vacuum port 952 are connected to each other via a vacuum channel 953. There is a corresponding port opening 990 in the housing 90'' in order to guide the vacuum port 952, here a nozzle, outward.

25 According to the invention, a second diaphragm 903 is arranged at that end of the coil former 92'' which is opposite the vacuum diaphragm 94''. Said diaphragm 903 together with the rear-side cover 902 forms a valve, that serves to ventilate the unit, or more precisely the vacuum channel 953 and therefore the vacuum port 952. With the aid of said ventilation diaphragm 903, an externally applied vacuum can be reduced rapidly and in a
30 specific manner such that the pressure in the vacuum channel 953 and at the vacuum port 952 is increased, depending on the duration of the ventilation, to atmospheric pressure. The air for the ventilation originates from the interior of the housing 90'', said housing not being hermetically closed to the outside except in the regions relevant to the generation of

the vacuum.

Said ventilation diaphragm 903 is likewise connected fixedly to the coil former 92''. It is preferably round with grooves or beads, i.e. is of plate-shaped design, wherein it also has
 5 two diametrically opposite wings, wherein at least one of said wings, preferably both wings, forms or form a ventilation flap 903'. The ventilation diaphragm 903 may be of substantially identical design to the vacuum diaphragm 94''. However, it may also have a different shape and/or size.

10 The ventilation diaphragm 903 has a central hole so that it can be fastened to the rear-side cover 902 with suitable clamping means. The clamping means are preferably a second spacer 904 and a counterpart 909. Said second spacer 904 and the counterpart 909 are preferably of identical design to the corresponding parts for the fastening of the vacuum diaphragm 94''. A stud of the counterpart 909 is also received here in a corresponding
 15 opening in the second end of the coil former 92''.

At least one, here two ventilation actuators 923, protrude over said second end of the coil former 92''. In this example, the ventilation actuators 923 are in each case a pin which is directed towards the ventilation flaps 903' and runs in the axial direction of the coil former
 20 92'. The pins 923 are arranged on the coil former 92' and are preferably formed integrally therewith. They are arranged diametrically opposite one another. The distance of the pins 923 from the center axis of the coil former 92' is dimensioned such that the pins can touch the ventilation flaps 903'. It is also possible to use only a single pin. However, for reasons of symmetry, in particular in order to permit a uniform movement of the coil former 92'',
 25 the use of two or more pins is recommended.

Figure 10 shows a cross section through the assembled unit. Said unit is in a first maximum stroke position, in which a vacuum is generated. The coil former 92'' is in a position close to the vacuum chamber 96'', called the lower position here. It can be seen in
 30 figure 11 that the ventilation diaphragm 903 rests on the housing 90'' and the ventilation flaps 903' close a ventilation channel 901 running in the housing 90''. For the purpose of simpler production, the ventilation channel 901 is provided at one end with a closure stopper 906 in order to close a corresponding bore 907 in the housing 90''. For the same

reason, the other end of the ventilation channel 901 is likewise provided with a closure stopper 994. The ventilation diaphragm 903 is preferably prestressed in order optimally to ensure said closure.

5 Figure 12 shows the situation at the same point in time in the lower region containing the vacuum diaphragm 94''. The vacuum diaphragm 94'' is in the lowermost position thereof, in which it is at its closest to the cover 99''. The vacuum valve flap 942 is closed, and the connection between pump chamber 96'' and vacuum channel 953 is interrupted. By contrast, the outlet valve flap 943 is open. As a result, air still in the cover-side region of
10 the pump chamber 96'' is released into the housing 90'' or to the outside via an outlet channel 993 running in the valve plate 95' and in the cover 99'', respectively.

Figure 13 shows the coil former 92'' in a second stroke position. As before, the ventilation diaphragm 903 closes the ventilation channel 901 such that air cannot pass from the
15 outside into the vacuum channel 953. Although the ventilation actuator 923 has come closer to the ventilation flap 903', it is not yet raised.

Figure 14 shows this situation in the region of the vacuum diaphragm 94'' on an enlarged scale. Owing to the raised vacuum diaphragm 94'', the cover-side region of the vacuum
20 chamber 96' has expanded and has a negative pressure. The outlet valve flap 943 is closed, by contrast the vacuum flap 942 is open, and therefore the negative pressure is applied via the vacuum channel 953 to the vacuum port 952. It is true that the ventilation channel 901 forms an additional dead volume, that also has to be evacuated. Since, however, the ventilation channel has a relatively small volume, the channel 901 has scarcely any
25 negative effect on the efficiency of the pump unit.

Figure 15 shows the coil former 92'' in a third position in which it is raised even further. In this position, the actuators 923 raise the ventilation flaps 903' and thus open a connection to the ventilation channel 901. Air can pass from the housing 90'' and from the outside
30 through the channel 901 to the vacuum port 952 and can thus reduce the vacuum applied and raise the pressure to atmospheric pressure or to a desired base vacuum. This can readily be seen in figure 16. The arrow shows the path of the inflowing air.

As can be seen in figure 17, the vacuum diaphragm 94'' is raised even further in this situation and is therefore ready for the next, vacuum-generating stroke. The vacuum flap 942 and also the outlet flap 943 are both raised and therefore open up the corresponding openings.

5

Therefore, in said pump unit according to the invention, not only is the linear movement of the coil former 92'' of the solenoid converted into a parallel movement of the vacuum diaphragm. The same coil former 92'' is also used for opening the ventilation valve, the movement of said coil former in the same direction as for the vacuum strokes being used
10 for this purpose. Depending on the position of the coil former 92'' relative to the permanent magnet 91'' and therefore to the housing 90'', a vacuum is generated or the ventilation is activated. To this end, only three different stroke positions of the coil former 99'' are necessary. In this case, the third stroke, that activates the ventilation, is larger than each of the two other strokes, or it is at least larger than the stroke in the same direction for
15 generating the vacuum. Said three strokes can be generated by the electronic controller 15 in a controlled and targeted manner, in particular in terms of the magnitude thereof.

Said third stroke, like the two other strokes, is generated by the electronic controller 15 of the unit 9''. However, the third stroke does not have to take place in each cycle. As a
20 result, pump sequences having different and varying curve courses and intervals can be generated.

Owing to the position scale, the position of the coil former 92'' relative to the housing 90'' can be precisely identified and used for controlling the strokes. However, other ways of
25 precisely identifying the position of the coil former 92'' and the magnitude of the strokes are also possible.

It is furthermore advantageous that the two diaphragms, i.e. the vacuum diaphragm 94'' and the ventilation diaphragm 903, serve as bearings for the movement of the coil former
30 92'' and that therefore friction is avoided.

The inventive vacuum pump explained according to the above-described embodiments can now be used, for example, in devices for expressing human breast milk. Examples of

devices of this type are illustrated in figures 3 to 6.

Figure 3 illustrates a first embodiment of a device of this type. Said device has a breastpump 1, a first line 2, a coupling part 3, a breast shield 4, a nonreturn valve 5, a
5 second line 6 and a milk collecting container 7.

The breast shield 4 is connected to the breastpump 1 via the coupling part 3 and the first flexible line 2. The second flexible line 6 leads from the breastpump 1 to the milk collecting container 7, said connection being provided with the nonreturn valve 5. The two
10 flexible lines 2, 6 are preferably tubes, in particular made of silicone.

As illustrated in figure 4, the milk collecting container 7 may alternatively also be fastened directly to the breastpump 1. For this purpose, there is preferably a suitably shaped adaptor 70 on the milk collecting container 7, said adaptor being detachably connectable to a
15 housing 10 of the breastpump.

The breastpump 1 has the abovementioned housing 10, in which the vacuum pump according to the invention, referred to below as a pump unit, is arranged together with a controller. The pump unit and the controller can be operated from the mains and/or battery
20 operated.

There are control elements (not illustrated here) on the housing 10. The control elements may contain, inter alia, a switch on/off element, keys or buttons for selecting the pumping frequency and the applied vacuum and the period of duration of the expressing operation.
25 There may also be a display.

A second chamber 8 (see figure 6), which likewise acts as a pump chamber, is formed between a recess of the housing 10 and a cover 13 covering said recess. The cover 13 is preferably connected detachably to the housing 10. A diaphragm 14 which is held in its
30 position by the cover 13 is arranged in said second chamber 8. The diaphragm 14 divides said pump chamber 8 into a unit-side part and a unit-remote part 80, 81, said diaphragm sealing said two parts with respect to each other.

As can be seen in figure 5, the diaphragm 14 illustrated here preferably has a substantially circular outline. There are preferably laterally protruding wings 140. There are three wings 140 here. The baseplate, which is preferably part of the housing 10, has lateral stops 110, between which said wings 140 are held. As a result, the diaphragm 14 can be held in an unambiguous position in the second chamber 8. This facilitates assembling.

The cover 13 has connecting ports for the first and second line 2, 6 and for the milk collecting container, respectively. Said ports are provided with the reference numbers 130 and 131. The second port 131 is preferably provided with a nonreturn valve 5. The connection to the counterplate, which is part of the housing 10, preferably takes place via snap-in or screw connections, wherein the corresponding holes are provided with the reference numbers 132 and 111 in figure 5.

A vacuum, which has been generated in the pump unit, is transmitted via a line 12 to said second chamber 8. The line 12 is connected to the output 990. The changes in pressure are passed on via the vacuum line 12 to the second chamber 8 where the diaphragm 14 is moved analogously to the diaphragm 94.

The pump here can be operated with a temporally constant cycle or, as is known in the prior art, the suction curve may be adapted in the shape, frequency and intensity thereof to the suction behavior of the baby and/or to the requirements of the mother.

The second chamber 8 has inputs and outputs which are not all visible in the figures. The cover 13 may be formed as a single piece or in a number of pieces. It not only forms a tight closure but also serves as a valve plate for said second pump chamber 8. Passages and valves (not illustrated in detail here) enabling the vacuum to build up in the unit-remote part, *i.e.* breast-side part 81, of the second pump chamber are therefore arranged in the cover 13.

In the cover 13 there is the first outlet opening 130 which connects the environment to the cover-side part 81 of the pump chamber 8. Said outlet opening 130 serves as a first port for the first line 2. The second outlet opening 131, which likewise connects the cover- and the breast-side part 81 of the second pump chamber 8 to the surroundings, respectively, is

designed as a second port. Said second port is provided with the nonreturn valve 5. Use is made here of a spout valve which is plugged onto a connector. However, other types of valve can also be used.

- 5 If the device is now used, the breast shield 4 is placed onto the mother's breast such that said breast shield at least surrounds the nipple. Preferably, at maximum the areola is additionally surrounded by the breast shield 4. The breastpump 1 is switched on and operated in the manner described above. The vacuum transmitted to the second pump chamber 8 evacuates the first line 2 such that there is a negative pressure in the breast
10 shield 4. As a result, milk is expressed from the mother's breast and passes through the breast shield 4 and the coupling part 3 into the first line 2. The milk flows through the first port 130 into the cover-side part of the second pump chamber. The expressed milk leaves the second pump chamber 8 through the second port 131 and the nonreturn valve 5 and passes into the milk collecting container 7 via the second line 6 (see Figure 3) or else,
15 depending on the embodiment, directly (see Figure 4). There is therefore no separate line for transporting the milk. The first line 2 serves at the same time as a suction line and as a milk transport line. After initial pneumatic pumping, the device therefore changes to hydraulic pumping. This is a further approximation to the natural sucking of babies.
- 20 The diaphragm 14 in the second pump chamber has two functions. Firstly, it serves as a partition wall between the air in the pump-side part of the second pump chamber and the milk in the cover-side part of the second pump chamber. It therefore serves as a means of separating media. It thereby prevents milk from being able to pass into the vacuum line 12 and therefore into the pump unit. However, it also prevents dirt from the pump unit from
25 being able to pass into the first and second lines 2, 6. Secondly, the cyclic movement of said diaphragm within the second pump chamber results in said diaphragm conveying and transporting the milk. Owing to said second function, during the expressing operation, the milk collecting container 7, breast shield 4 and breastpump 1 can be arranged in positions which are independent of one another. For example, the milk collecting container 7 can be
30 located above the breastpump 1 and/or the breast shield 4. The breastpump 1 may also be above the milk collecting container 7 and/or the breast shield 4. This enables the mother to express milk even while lying down or, if she is seated, to place the milk collecting container 7 and the breastpump 1 onto a shelf or another raised platform out of the reach of

small children.

The nonreturn valve 5 preferably first opens at a sufficient pressure, *i.e.* when the second pump chamber 8 is sufficiently filled with milk. In this way, the dead volume which must
5 be evacuated can be kept to a minimum.

In addition, the dead volume can be reduced by using a small breast shield 4 which only surrounds the nipple and as small a part of the remaining breast as possible, if any part at all. A suitable breast shield 4 is illustrated in the figures. Alternative forms of breast
10 shields can also be used.

The elements of the above-described embodiments can be combined with one another individually or in groups in order to form further embodiments.

15 The vacuum pump according to the invention is relatively small and compact and operates quietly. It is suitable in particular for breastpumps used "hands-free".

LIST OF DESIGNATIONS

1	breastpump	81	breast shield-side chamber
10	housing		part
110	lateral stop		
111	hole	9, 9', 9''	pump unit
12	vacuum line	90, 90', 90''	housing
13	cover	900	plain bearing
130	first port	901	ventilation channel
131	second port	902	rear-side cover
132	hole	903	ventilation diaphragm
14	diaphragm	903'	ventilation flap
140	lateral wing	904	second spacer
15	control unit	906	closure stopper
150	operating elements	907	bore
16	lines	908	fastening holes
		909	second counterpart
2	first line	91, 91', 91''	permanent magnet
		910	recess
3	coupling part	911'	iron core
		911, 911''	iron plate
4	breast shield	912'	iron ring
		92, 92', 92''	coil former with coil
5	nonreturn valve	920, 920'	position scale
		920''	position scale
6	second line	921, 921'	coil
60	valve cap	921''	coil
		922	winding ends
7	milk collecting container	923	ventilation actuator
70	adaptor	93	guide bearing
		94, 94', 94''	vacuum diaphragm
8	pump chamber	940	first spacer
80	drive-side chamber part	941	first counterpart

942	vacuum valve flap (inlet valve flap)	98, 98', 98''	detector
943	outlet valve flap	980	port to measuring board
95, 95', 95''	valve plate	99, 99', 99''	cover
952	vacuum port	990	port opening
953	vacuum channel	991	hole
96, 96', 96''	pump chamber	992	air inlet opening
97, 97'	light source	993	outlet channel
		994	closure stopper

PATENT CLAIMS

1. A diaphragm vacuum pump with an electrically operated drive unit and a vacuum diaphragm (94, 94', 94''), which separates a pump chamber (96, 96', 96'') into a drive-side part and a drive-remote part and which can be deflected by means of a movable part (92, 92', 92'') of the drive unit, wherein the drive unit is an electromagnetic drive unit, wherein the movement of the movable part (92, 92', 92'') is a linear movement and wherein the vacuum diaphragm (94, 94', 94'') is deflected in the direction of said linear movement which is generated electromagnetically in the drive unit.
2. The diaphragm vacuum pump as claimed in claim 1, wherein the movable part (92'') is operatively connected to a ventilation valve (903) for ventilating the diaphragm vacuum pump, wherein the linear movement generated in the drive unit optionally actuates the ventilation valve (903).
3. The diaphragm vacuum pump as claimed in claim 2, wherein the movable part (92'') has a first end and a second end opposite the first end, wherein the vacuum diaphragm (94'') is operatively connected to the first end, and the ventilation valve (903) is operatively connected to the second end.
4. The diaphragm vacuum pump as claimed in either of claims 2 and 3, wherein the ventilation valve has a ventilation diaphragm (903).
5. The diaphragm vacuum pump as claimed in claim 4, wherein the vacuum diaphragm (94'') is connected fixedly to the first end of the movable part (92'') and the ventilation diaphragm (903) is connected fixedly to the second end thereof, and wherein said diaphragms form the mounting of the movable part (92'') within the drive unit.
6. The diaphragm vacuum pump as claimed in one of claims 2 to 5, wherein the movable part (92'') is movable by means of a controller in a first and second stroke

in order to generate a vacuum in the pump chamber (96'') and is movable in a third stroke in order to actuate the ventilation valve (903), wherein the third stroke takes place in the direction of the second stroke but is larger than said second stroke.

7. The diaphragm vacuum pump as claimed in one of claims 1 to 6, wherein the drive unit has at least one permanent magnet (91, 91', 91'') and a coil former (92, 92', 92'') with a coil, wherein the coil is arranged on the coil former (92, 92', 92''), wherein the coil former (92, 92', 92'') together with the coil is held in a linearly displaceable manner in two directions along a longitudinal axis with respect to the magnet (91, 91', 91''), and wherein the coil former (92, 92', 92'') forms the movable part and is fixedly connected to the vacuum diaphragm (94, 94', 94'') and deflects the latter during the abovementioned displacement of the coil former (92, 92', 92'') in both directions of the displacement of said coil former.
8. The diaphragm vacuum pump as claimed in claim 7, wherein the coil former (92, 92', 92'') forms a piston with a first end at which the vacuum diaphragm (94, 94', 94'') is arranged.
9. The diaphragm vacuum pump as claimed in claim 8, wherein the vacuum diaphragm (94, 94', 94'') has a diameter that is much larger than the piston (92, 92', 92'').
10. The diaphragm vacuum pump as claimed in claim 9, wherein the vacuum diaphragm (94, 94', 94'') is fastened centrally on the first end of the piston (92, 92', 92'').
11. The diaphragm vacuum pump as claimed in one of claims 8 to 10, wherein the piston (92, 92', 92'') has a second end which is held in a linearly displaceable manner.
12. The diaphragm vacuum pump as claimed in one of claims 8 to 11, wherein the piston (92, 92'') has a second end to which a further diaphragm (903) is fastened.
13. The diaphragm vacuum pump as claimed in one of claims 8 to 12, wherein the coil (921, 921'') is a flat coil and wherein there is at least one permanent magnet (91, 91''), which is held in a fixed position in a housing (90, 90'') of the diaphragm

vacuum pump.

14. The diaphragm vacuum pump as claimed in one of claims 8 to 11, wherein the piston (92') has a second end, which is mounted displaceably between an iron core (911') and a magnet (91').
15. The diaphragm vacuum pump as claimed in one of claims 8 to 11 or 14, wherein the coil is a moving coil, the moving coil and the permanent magnet (91') are of rotationally symmetrical design, and an iron core (911') passes through the moving coil.
16. The diaphragm vacuum pump as claimed in one of claims 1 to 15, wherein there is at least one position detector (97, 97'; 98, 98', 98'') for determining the relative position of the movable part (92, 92', 92'') of the drive unit with respect to the remaining part of the drive unit which is in a fixed position.
17. The diaphragm vacuum pump as claimed in claim 16, wherein the position detector (97, 97'; 98, 98', 98'') generates a signal, which is used to control the vacuum pump, as a function of the relative position of the coil former (92, 92', 92'').
18. The diaphragm vacuum pump as claimed in one of claims 1 to 17, wherein there is an outlet (990) in the drive-remote part of the pump chamber (96, 96', 96''), said outlet being connected to an inlet of a second chamber (8), wherein the second chamber (8) has a further diaphragm (14), which separates said second chamber (8) into two parts (80, 81) and wherein the further diaphragm (14) serves as a means of separating media and for transferring the vacuum generated in the pump chamber.