

Description**VALVE ACTUATING DEVICE WITH OPENING REDUCED SPEED****Technical field**

[0001] The invention is directed to the field of valves for fluids, like compressed gas. More particularly, the invention is directed to the field of such valves that are operated by means of an actuator such as a compressed air actuator. The invention is directed to an actuator for a valve.

Background art

[0002] Prior art patent document published WO 2010/097456 A1 discloses a valve actuating device that is operated by compressed air. This device comprises a piston that converts the pressure of the compressed air into a larger pressure acting on an auxiliary fluid like lubricating oil that itself acts on a larger surface of an actuating element of the device. In other words, a limited pressure of the compressed air entering the actuating device is converted into a larger force that moves the actuating element of the device on a reduced stroke. That force acts against a stack of Belleville disk springs. The actuating device of this teaching results in a quick opening and closing of the valve. However for certain applications, like for UHP (Ultra High Purity) gases, more generally for gases that produce a substantial temperature increase during an adiabatic compression and that are likely to react exothermically (ignite and/or explode) further to that temperature increase, it can be desirable to achieve a progressive opening of the valve.

Summary of inventionTechnical Problem

[0003] The invention has for technical problem to provide a valve actuating device that achieves a slow opening of the valve and this with a simple and robust construction.

Technical solution

[0004] The invention is directed to a valve actuating device comprising: a body with a bore and a bottom; an actuating element sealingly housed in the bore of the body and forming a first chamber with the bottom of the body; a piston with a portion sealingly extending through a bore in the actuating element,

said portion and bore delimiting a second actuating chamber; an auxiliary fluid in the first and second chambers; an auxiliary fluid passage between the first and second chambers so that a movement of the piston in the bore of the actuating member can increase the pressure in the first chamber and move the actuating element; remarkable in that the fluid passage between the first and second chambers comprises a restriction and a check-valve both arranged in parallel, so as to limit the flow of the auxiliary fluid from the second chamber to the first chamber.

- [0005] According to a preferred embodiment of the invention, the actuating element comprises a grip portion to be attached to a movable element of the valve, said grip portion sealingly protruding through an opening of the bottom of the body, the restriction and/or the check-valve being arranged on a face of the actuating element that is in vis-à-vis with the bottom of the body and that surrounds said grip portion.
- [0006] According to a preferred embodiment of the invention, the restriction and the check-valve are arranged at distinct locations at the periphery of the grip portion, said distinct locations being preferably opposed diametrically.
- [0007] According to a preferred embodiment of the invention, the fluid passage between the first and second chambers comprises an internal groove in the bore of the actuating element.
- [0008] According to a preferred embodiment of the invention, the bore of the actuating element comprises a bottom, the internal groove being at said bottom.
- [0009] According to a preferred embodiment of the invention, the distance between a or the face of the actuating element that is in vis-à-vis with the bottom of the body and the cavity at the bottom level is less than 6 mm, preferably less 5 mm, more preferably less than 4 mm.
- [0010] According to a preferred embodiment of the invention, the restriction and/or the check-valve has/have a longitudinal axis that is forms an angle of less than 30°, preferably less than 20°, more preferably less than 10° with a longitudinal axis of the bore and the groove of the actuating element.
- [0011] According to a preferred embodiment of the invention, the actuating element comprises at least one offset bore on a or the face of the actuating element

that is in vis-à-vis with the bottom of the body, said at least one offset bore intersecting the groove and housing the check valve or a plug with a restricted drill hole forming the restriction. The plug can be cap-shaped with a bottom wall and a side wall. The hole can be formed in the bottom wall. The side wall can comprise a thread on its external surface. It can also comprise a collar at its circular edge distal from the bottom wall, said collar being configured for resting in a tight manner on a shoulder portion formed in the actuating element.

- [0012] According to a preferred embodiment of the invention, the restriction is formed by a drill hole extending from a or the face of the actuating element that is in vis-à-vis with the bottom of the body, to the groove.
- [0013] According to a preferred embodiment of the invention, the restriction comprises an average diameter that is less than 0.5 mm, preferably less than 0.2 mm, more preferably less than 0.1 mm.
- [0014] According to a preferred embodiment of the invention, the check valve is configured so as to open when the auxiliary fluid flows from the first chamber to the second chamber.
- [0015] According to a preferred embodiment of the invention, the auxiliary fluid in the first and second chambers is liquid, preferably oil or an oil-like silicone liquid.
- [0016] According to a preferred embodiment of the invention, it comprises a spring urging the actuating element against the bottom of the body, said spring preferably comprising a stack of Belleville disk springs.
- [0017] According to a preferred embodiment of the invention, the portion of the piston that extends through the bore of the actuating member is a first portion, said piston comprising a second portion of a larger diameter and sealingly housed in a further bore, said bore and second portion delimiting a third chamber with an port for compressed air for moving said piston towards the bottom of the body. The pressure of the compressed air can be comprised between 6 and 10 bar.
- [0018] According to a preferred embodiment of the invention, the port for compressed air comprises a device that restricts the passage when the air

flows to fill the third chamber and that essentially does not restrict the passage when the air escapes from said chamber.

- [0019] According to a preferred embodiment of the invention, the diameter of the second portion of the piston is at least 2 times, preferably at least 4 times, larger than the diameter of the first portion.
- [0020] According to a preferred embodiment of the invention, the diameter of bore of the body, that houses the actuating element is at least 2 times, preferably at least 3 times, larger than the diameter of the bore of said element.
- [0021] According to a preferred embodiment of the invention, the bore of the body, that houses the actuating element; the bore of the actuating element; if present, the grip portion; and, if present, the further bore of the body; are aligned.
- [0022] The invention has also for object a valve for a fluid, comprising: a seat; a closure member that is movable in translation along a main axis for cooperating with the seat for shutting-off of a fluid passage; an actuating device for actuating the closure member; remarkable in that the actuating device is according to the invention.
- [0023] According to a preferred embodiment of the invention, the movement of the actuating element, when the auxiliary fluid flows from the second chamber to the first chamber, opens the passage.
- [0024] The invention is also directed to a system for outputting a gas stored in a compressed state in a gas cylinder, wherein said system comprises: a valve according to the invention; and a pneumatic control unit to be pneumatically connected between a compressed air source and the first chamber of the actuating device of the valve, said pneumatic control unit comprising a flow restrictor and in parallel a check valve oriented such as to close when filling the first chamber so as force the compressed air towards the flow restrictor, and to open when emptying said chamber so as to by-pass said restrictor.
- [0025] The invention can also be directed to such a system for a valve with any kind of pneumatic actuating device, i.e. to a system for outputting a gas stored in a compressed state in a gas cylinder, wherein said system comprises: a valve with a pneumatic actuating device with a first actuating chamber; and a pneumatic control unit to be pneumatically connected

between a compressed air source and the first chamber of the actuating device of the valve, said pneumatic control unit comprising a flow restrictor and in parallel a check valve oriented such as to close when filling the first chamber so as to force the compressed air towards the flow restrictor, and to open when emptying said chamber so as to by-pass said restrictor. The first chamber can be delimited by a piston that cooperates with an hydraulic amplifier of the actuating effort, said amplifier preferably comprising an auxiliary fluid, a flow restriction and a check-valve both arranged in parallel, so as to limit the flow of the auxiliary fluid when opening the valve.

[0026] According to a preferred embodiment of the invention, the pneumatic control unit further comprises a calibrated valve in parallel to the check valve and the flow restrictor, said calibrated valve being configured to open and by-pass said flow restrictor and said check valve during filling of the chamber as long as the difference of pressure between the compressed air source and the first chamber is above a first value.

[0027] According to a preferred embodiment of the invention, the first value is comprised between 2 and 10 bar, preferably between 3 and 8 bar, more preferably between 4 and 6 bar.

[0028] According to a preferred embodiment of the invention, the calibrated valve is configured to close when the pressure in the first chamber is above a second value.

[0029] According to a preferred embodiment of the invention, the second value is comprised between 1 and 5 bar, preferably between 2 and 4 bar.

[0030] According to a preferred embodiment of the invention, the pneumatic control unit is mounted on the valve, preferable on the valve actuator.

Advantages of the invention

[0031] The invention is particularly interesting in that it provides a slower movement in one direction than in the other one, and this with a compact and robust construction. Indeed, the presence of a restriction in the passage for the auxiliary fluid between the first and second chamber is quite efficient for achieving a reliable slowdown of the movement on one stroke. The way of providing the restriction in accordance with the invention is particularly advantageous in that it can be easily performed industrially.

Brief description of the drawings

- [0032] Figure 1 is a sectional view of a valve equipped with an actuating device in accordance with the invention.
- [0033] Figure 2 is a section view of the valve and the actuating device of figure 1 where however the valve is in an open position.
- [0034] Figure 3 shows a diagram of a system for delivering gas from a gas cylinder.
- [0035] Figure 4 shows an alternative of the pneumatic control unit of the diagram of figure 3.

Description of an embodiment

- [0036] Figure 1 is a sectional view of a valve equipped with an actuating device according to an embodiment of the invention. The section is made along a plane that comprises the longitudinal axis of the valve and the device. This axis is vertical in the present figures.
- [0037] In figure 1, the valve is kept in a closed state by the actuating device, in the absence of compressed air in the device.
- [0038] The valve 2 comprises a valve body 4 with an inlet 6 and an outlet 8 for a fluid like a pressurized gas. The valve body 4 comprises also a fluid passage 10 interconnecting the inlet 6 with the outlet 8. A seat 12 is formed in the valve body 4 and surrounds the passage 10. A closure member 14 is movable in a cavity of the valve body 4 so as to cooperate with the seat 12 in order to shut-off the passage 10. In the present embodiment, the closure member 14 is attached to an intermediate member 16 that is guided by a sleeve 18 screwed on the valve body 4. The closure member 14 is attached via the intermediate member 16 to an actuating element of the actuating device 20.
- [0039] The actuating device 20 comprises a body 22 with a bore 22¹ that houses the actuating element 24 in a fluid tight manner. To that end, the actuating element 24 can comprise on its external generally cylindrical surface a groove with a seal that contacts the bore 22¹ of the body 22. The bore 22¹ of the body 22 comprises a bottom 22² that delimits with the lower face of the actuating element 24 that is in vis-à-vis with said bottom 22², a first chamber 25 that is filled with an auxiliary fluid. The fluid can be liquid, e.g. lubricating oil.

- [0040] The actuating device 20 comprises also a piston 26 with a first portion 26¹ that cooperates in a fluid tight manner with a bore 24¹ in the actuating element 24. The bore 24¹ of the actuating element 24 opens at a face of said element that is opposite to the lower one, i.e. vis-à-vis with the bottom 22² of the body 22. The first portion 26¹ of the piston 26 delimits with the bore 24¹ a second chamber 27 that communicates with the first chamber 25 via a fluid restriction 28 and a check-valve 30, said restriction and check-valve being fluidly arranged in parallel between the two chambers.
- [0041] The actuating element 24 is resiliently biased towards the bottom 22² of the body 22 by a spring 32. This latter can comprise a stack of Belleville washer springs. These washers are fitted around a cylindrical portion of the actuating element 24 and abut at one end on a shoulder of the cylindrical portion and at the other end on a ring 34 that is fastened to the body by a threaded engagement.
- [0042] The piston 26 comprises also a second portion 26² that cooperates in a fluid tight manner with a further bore 22³ of the body 22. In the present particular embodiment, that body part is made in a separate cap-shaped element 36 that is fixed to the body 22 via a threaded engagement. The second portion 26² of the piston 26 forms with the further bore 22³ a third chamber 40 that can be filled with compressed air. To that end the cap-shaped element 36 comprises on its upper wall a gas passage 36¹. A cap-shaped hand-wheel 38 is rotatably fitted around the cap-shaped element 36 and comprises on its upper wall a port 42 for connecting a source of compressed air. A gasket 44 is housed in a groove on the interior side of the upper wall of the hand-wheel 38 and circumvents the port 42. The gasket 44 provides therefore a gas tight connection between the gas passage 36¹ in the upper wall of the cap-shaped element 36 and the port 42 on the upper wall of the hand-wheel 38 when these gas passages are aligned. Since these are offset relative to the rotation axis of the hand-wheel 38, compressed air supplied at the port 42 of the hand-wheel 38 can flow into the chamber 40 only when the port 42 is aligned with the gas passage 36¹. In other words, by adequately rotating the hand-wheel 38, compressed air can be supplied or not to the chamber 40.

- [0043] The upper and exterior surface of the upper wall of the cap-shaped element 36 can be flat enough to remain in gas tight contact with the gasket 44 while the hand-wheel 38 is positioned such that the port 42 is out of alignment with the gas passage 36¹, so that the compressed air supplied at the 42 does not unintentionally flow at the ambient.
- [0044] The first and second portions 26¹ and 26² of the piston are preferably rigidly linked to each other. The effective external diameter of the second portion 26² is larger, preferably at least two times larger, more preferably at least four times larger, than the diameter of the first portion 26¹. The pressure that is applied to the second portion 26² of the piston can therefore be amplified in the second chamber 27 delimited by the first portion 26¹.
- [0045] The operation of the valve is as follows. When compressed air, or any compressed fluid, is supplied to the third chamber 40, the force resulting from the pressure of the air on the second portion 26² of the piston 26 is transmitted to the first portion 26¹ which generates a higher pressure in the second chamber 27. The auxiliary fluid in that chamber can flow to the first chamber 25 only via the restriction 28. Indeed, the check-valve 30 is arranged so as to open only when the auxiliary fluid flows from the first chamber 25 to the second chamber 27. The restriction can have an average or constant diameter that is less than 0.5 mm, preferably less than 0.2 mm, more preferably less than 0.1 mm. The flow of the auxiliary fluid from the second chamber 27 to the first one 25 is therefore slowed down so as to provide a progressive opening of the valve.
- [0046] The auxiliary fluid flowing from the second chamber 27 to the first one 25 increases the pressure in the first chamber to the level of the pressure in the second one. The effective surface of the actuating element 24 that is subject to the pressure of the auxiliary fluid in the first chamber 27 can be larger than the effective surface of the first portion 26¹ of the piston 26, thereby providing an additional amplification of the force that is exerted to the actuating element 24. Further to the increase of pressure in the first chamber 25, the resulting force exerted on the actuating element 24 has for effect to move it away from the bottom 22² of the body 22, against the resilient force of the spring 32.

- [0047] The actuating element 24 comprises a grip portion 24² that protrudes away from the second chamber through an opening in the bottom 22² of the body 22. The grip portion 24² can comprise at least one groove on its external surface so as to contact, in a fluid tight manner, the interior surface of the opening in the bottom 22². The above mentioned intermediate member 16 is mechanically engaged with the grip portion 24². The lift movement of the actuating element 24 has therefore for effect to lift also the closure member 14 and to open the valve 2.
- [0048] Figure 1 comprises a magnified view of an area of the actuating device that comprises the first and second chambers 25 and 27, and the restriction 28. We can observe that the restriction is offset with regard to the longitudinal axis of the actuating element 24. We can also observe that the restriction has its longitudinal axis essentially parallel to the main longitudinal axis. An internal groove 24⁴ is made in the bore 24¹, preferably at the level of the bottom 24³ of said bore. The groove forms a passage between the bore 24¹ of the actuating element 24 and the restriction 28 and also the check-valve which are both laterally distant from the said bore. Indeed, the grip portion 24² for actuating the closure member of the valve occupies the central portion of the first chamber so that the restriction and the check-valve have to be shifted to offset positions.
- [0049] In the present embodiment, the restriction is formed by a plug 28 that is screwed in an offset bore 24⁵ in the (lower) face of the actuating element 24 that delimits the first chamber 25. The offset bore 24⁵ intersects the groove 24⁴ so as to provide a passage for the auxiliary fluid between the first and second chambers.
- [0050] As is apparent in the magnified area in figure 1, the plug 28 is cap-shaped and comprises in its bottom wall a through-hole 28¹ of a reduced diameter. The side wall of the plug 28 is threaded and comprises a collar 28² for resting on a shoulder portion formed around the offset bore 24⁵. The plug can be made of any metal, in particular stainless steel. The through-hole 28¹ can be made by drilling or via a laser beam. A laser beam is particularly interesting from diameters of less than 0.2 mm, more particularly for less than 0.1 mm.

- [0051] The restriction can also be made by directly drilling the material of the actuating element 24. Generally speaking, performing drill holes of reduced diameters, e.g. less than 1 mm, is difficult and is, a general practical rule, limited to a drill depth that corresponds to about 10 times the diameter. This means that for a hole with a diameter of 0.5 mm, it is desirable, for industrial and practical reasons, that the drill depth does not exceed 5 mm. Similarly, for a diameter of 0.4 mm, the drill depth shall not exceed 4 mm. In other words, this means that for practicing a restriction between the first and second chambers 25 and 27, it is particularly advantageous to perform the groove 24⁴ that provides a passage from the bore 24¹ to a portion of material of the actuating element 24 that is in contact with the first chamber 25 and that has a reduced thickness, e.g. less than 5 or 4 mm.
- [0052] The restriction can also be made by laser drilling, i.e. by applying a laser beam to the material of the actuating element 24. Similar constraints to those mentioned here above apply to laser drilling.
- [0053] Figure 2 is a sectional view of the valve of figure 1 in the opened state. Compressed gas has filled the third chamber 40 and has lowered the piston 26. This movement of the piston 26 has moved a portion of the auxiliary fluid from the second chamber 27 to the first chamber 25. This movement of the fluid has then exerted an effort on the actuating element 24 and has lifted it against the elastic biasing effect of the spring 32. The grip portion 24² has then moved the intermediate member 16 and the closure member 14 away from the seat 12 so as to open the passage 10 between the inlet 6 and the outlet 8.
- [0054] During the above mentioned opening movement, the flow of the auxiliary fluid from the second chamber 27 to the first one 25 was slowed down due to the restriction 28. Indeed, dependent on the viscosity of the fluid, the restriction can provide an opening time that exceeds 1 second, preferably 2 seconds, more preferably 3 seconds.
- [0055] When the pressure of the compressed air in the third chamber 40 is released, the pressure in the second chamber 27 drops and the auxiliary fluid can flow through the check-valve from the first chamber to the second chamber. The biasing force of the spring 32 is converted into pressure in

the first and second chambers, and said pressure pushes the piston back to its initial position when the valve is closed.

- [0056] Although not shown in the figures, the port for compressed air can comprise a device that restricts the passage when the air flows to fill the third chamber and that essentially does not restrict the passage when the air escapes from said chamber.
- [0057] Figure 3 illustrates a diagram of a system for delivering or outputting gas from a gas cylinder. The system comprises a valve 2 with a pneumatic actuator that comprises a pneumatic actuating chamber receiving a piston and an auxiliary fluid, like oil or any other liquid, that amplifies the pneumatic force produced on the piston. It comprises a restriction and a check-valve both arranged in parallel, so as to limit the flow of the auxiliary fluid when opening the valve. That actuator can be in accordance with the actuator 20 of figures 1 and 2. The valve 2 is mounted on, or connected to, the gas cylinder 50. The system comprises also a pneumatic control unit 52 that controls the flow of pneumatic air that actuates the valve 2. An electromagnetic valve 54 is arranged between the pneumatic control unit 52 and a compressed air source. That source can be a usual one, at a pressure comprised between 5 and 10 bar.
- [0058] The pneumatic control unit 52 comprises essentially a flow restrictor 52¹ and a check-valve 52². Both are mounted in parallel and the check-valve 52² is oriented so as to close and thereby block a flow of air from the compressed air source to the actuator, and therefore force it through the flow restrictor 52¹. The check-valve opens and thereby allows the flow of air from the actuator to the environment of the system through the valve 54.
- [0059] For opening the valve, the electromagnetic valve 54 is opened so that the compressed air flows through the flow restrictor 52¹, the check-valve 52² being closed. The pressure in the actuating pneumatic chamber increases then progressively to reach a pressure close to the pressure of the compressed air source. The valve can be a normally close valve against the resilient force of a spring or any other resilient means. While the pressure increases in the pneumatic actuating chamber, the resulting effort in the

opening direction reaches the resilient effort of the spring and thereby progressively opens the valve.

- [0060] The presence of a flow restrictor in the passage of the auxiliary hydraulic fluid, in addition to the flow restrictor 52¹ in the pneumatic control unit 52, provides an additional interesting effect for obtaining a slow opening movement of the valve. Indeed, the flow restrictor 52¹ can then be dimensioned larger so as to more rapidly fill the actuating chamber. The time for filling the chamber until reaching a pressure high enough to open the valve is indeed a time that does not count to the opening time.
- [0061] For closing the valve, the valve 54 is switched to a state where it closes the source of compressed air and connected the pneumatic actuating chamber with the ambient. The air in the actuating chamber flows out of the chamber through the check-valve 52², by-passing the flow restrictor 52¹. The valve closes therefore rapidly, contrary to the opening movement.
- [0062] For reducing the time for filling the actuating chamber with the compressed air, the pneumatic control unit 52 of figure 3 can be replaced by the unit 152 of figure 4. Similarly to the unit 52 of figure 3, the unit 152 comprises a flow restrictor 152¹ and a check-valve 152², both being mounted in parallel. It comprises, in addition, a calibrated valve 152³ that is configured to open only when the difference of pressure between the compressed air source and the actuating chamber is greater than a first value. That value can be comprised between 2 and 10 bar, preferably between 3 and 8 bar, more preferably between 4 and 6 bar. This means that at the beginning of the opening operation, the pressure in the actuating chamber is close to zero so that the difference of pressures on both sides of the valve 152³ is high enough for said valve to open and permit a rapid filling of the chamber until it reaches a pressure of a second value that is close to the pressure necessary for starting opening the valve. The calibrated valve 152³ closes then and the compressed air flow then only through the flow restrictor 152¹.

Claims

1. Valve actuating device (20) comprising:
 - a body (22) with a bore (22¹) and a bottom (22²);
 - an actuating element (24) sealingly housed in the bore (22¹) of the body (22) and forming a first chamber (25) with the bottom (22²) of the body (22);
 - a piston (26) with a portion (26¹) sealingly extending through a bore (24¹) in the actuating element (24), said portion (26¹) and bore (24¹) delimiting a second actuating chamber (27);
 - an auxiliary fluid in the first and second chambers (25, 27);
 - an auxiliary fluid passage between the first and second chambers (25, 27) so that a movement of the piston (26) in the bore (24¹) of the actuating member (24) can increase the pressure in the first chamber (25) and move the actuating element (24);
 characterized in that
 the fluid passage between the first and second chambers (25, 27) comprises a restriction (28) and a check-valve (30), both arranged in parallel, so as to limit the flow of the auxiliary fluid from the second chamber (27) to the first chamber (25).
2. Valve actuating device (20) according to claim 1, characterized in that the actuating element (24) comprises a grip portion (24²) to be attached to a movable element (14, 16) of the valve (2), said grip portion sealingly protruding through an opening of the bottom (22²) of the body (22), the restriction (28) and/or the check-valve (30) being arranged on a face of the actuating element (24) that is in vis-à-vis with the bottom (22²) of the body (22) and that surrounds said grip portion (24²), preferably the restriction (28) and the check-valve (30) are arranged at distinct locations at the periphery of the grip portion (24²), said distinct locations being more preferably opposed diametrically.
3. Valve actuating device (20) according to one of claims 1 and 2, characterized in that the fluid passage between the first and second chambers (25, 27) comprises an internal groove (24⁴) in the bore (24¹) of the actuating element (24), preferably the bore (24¹) of the actuating element (24) comprises a bottom (24³), the internal groove (24⁴) being at said bottom (24³), and more preferably the

distance between a or the face of the actuating element (24) that is in vis-à-vis with the bottom (22²) of the body (22) and the groove (24⁴) at the bottom (24³) level is less than 6 mm, preferably less 5 mm, more preferably less than 4 mm.

4. Valve actuating device (20) according to claim 3, characterized in that the restriction (28) and/or the check-valve (30) has/have a longitudinal axis that forms an angle of less than 30°, preferably less than 20°, more preferably less than 10° with a longitudinal axis of the bore (24¹) and the groove (24⁴) of the actuating element (24), and/or the actuating element (24) comprises at least one offset bore (24⁵) on a or the face of the actuating element (24) that is in vis-à-vis with the bottom (22²) of the body (22), said at least one offset bore (24⁵) intersecting the groove (24⁴) and housing the check valve (30) or a plug (28) with a restricted drill hole (28¹) forming the restriction, and/or the restriction (28) is formed by a drill hole extending from a or the face of the actuating element (24) that is in vis-à-vis with the bottom (22²) of the body (22), to the groove (24⁴).
5. Valve actuating device (20) according to any one of claims 1 to 4, characterized in that the restriction (28) comprises an average diameter that is less than 0.5 mm, preferably less than 0.2 mm, more preferably less than 0.1 mm.
6. Valve actuating device (20) according to any one of claims 1 to 5, characterized in that the check valve (30) is configured so as to open when the auxiliary fluid flows from the first chamber (25) to the second chamber (27).
7. Valve actuating device (20) according to any one of claims 1 to 6, characterized in that the auxiliary fluid in the first and second chambers (25, 27) is liquid, preferably an oil-like silicone based liquid.
8. Valve actuating device (20) according to any one of claims 1 to 7, characterized in that it comprises a spring (32) urging the actuating element (24) against the bottom (22²) of the body (22), said spring preferably comprising a stack of Belleville disk springs (32).
9. Valve actuating device (20) according to any one of claims 1 to 8, characterized in that the portion (26¹) of the piston (26) that extends through the bore (24¹) of the actuating member (24) is a first portion, said piston (26) comprising a second

portion (26²) of a larger diameter and sealingly housed in a further bore (22³), said bore (22³) and second portion (26²) delimiting a third chamber (40) with an port (42) for compressed air for moving said piston (26) towards the bottom (22²) of the body (22), preferably the port (42) for compressed air comprises a device that restricts the passage when the air flows to fill the third chamber (40) and that essentially does not restrict the passage when the air escapes from said chamber, and/or the diameter of the second portion (26²) of the piston (26) is at least 2 times, preferably at least 4 times, larger than the diameter of the first portion (26¹).

10. Valve actuating device (20) according to any one of claims 1 to 9, characterized in that the diameter of the bore (22¹) of the body (22), that houses the actuating element (24) is at least 2 times, preferably at least 3 times, larger than the diameter of the bore (24¹) of said element (24).
11. Valve actuating device (20) according to any one of claims 1 to 10, characterized in that the bore (22¹) of the body (22), that houses the actuating element (24); the bore (24¹) of the actuating element (24); if present, the grip portion (24²); and, if present, the further bore (22³) of the body (22); are aligned.
12. Valve (2) for a fluid, comprising:
 - a seat (12);
 - a closure member (14) that is movable in translation along a main axis for cooperating with the seat (12) for shutting-off of a fluid passage (10);
 - an actuating device (20) for actuating the closure member (14);
 characterized in that
the actuating device (20) is according to any one of claims 1 to 11.
13. Valve (2) according to claim 12, characterized in that the movement of the actuating element (24), when the auxiliary fluid flows from the second chamber (27) to the first chamber (25), opens the passage (10).
14. System for outputting a gas stored in a compressed state in a gas cylinder, characterized in that said system comprises:
 - a valve (2) according to any one of claims 19 and 20; and

- a pneumatic control unit (52; 152) to be pneumatically connected between a compressed air source and the first chamber (40) of the actuating device (20) of the valve (2), said pneumatic control unit (52; 152) comprising a flow restrictor (52¹; 152¹) and, in parallel, a check valve (52²; 152²) oriented such as to open when filling the first chamber (40) so as to force the compressed air towards the flow restrictor (52¹; 152¹), and to open when emptying said chamber (40) so as to by-pass said restrictor (52¹; 152¹).

15. System according to claim 14, characterized in that the pneumatic control unit further comprises a calibrated valve (152³) in parallel to the check valve (152²) and the flow restrictor (152¹), said calibrated valve (152³) being configured to open and by-pass said flow restrictor (152¹) and said check valve (152²) during filling of the chamber (40) as long as the difference of pressure between the compressed air source and the first chamber is above a first value, preferably the first value is comprised between 2 and 10 bar, preferably between 3 and 8 bar, more preferably between 4 and 6 bar, and/or the calibrated valve (152³) is configured to close when the pressure in the first chamber is above a second value, the second value being comprised between 1 and 5 bar, preferably between 2 and 4 bar.