

Description

The invention relates to a pipe propelling device according to the preamble of claim 1. Such pipe propelling devices are used for propelling pipes from vertical shafts in order to force the pipe into the ground by the operation of the hydraulic jack.

A pipe propelling device as described in the preamble of claim 1 known from DE—B 25 59 853.

From FR—A—2 526 882 a cylinder device is known. Upper and lower cylinders of the first stage jacks of this device have capacities equal to each other, and the cylinder of a second stage jack has capacity equal to the sum of those of the first stage jack cylinders.

It is object of the present invention to provide a pipe propelling device as described which needs a small-dimensioned vertical shaft without interposing any struts while providing a sufficient operative space within the vertical shaft.

This object is solved by the invention as claimed in claim 1.

The total length of the pipe propelling device is minimize while the operative stroke of the jack assembly can be made longer than the length of the pipe by extending operatively said first and second stage jacks. The size of the vertical shaft can be minimized and further the pipe can be forced entirely into the ground by one operation of the jack assembly without interposing the strut, thereby the efficiency of operation can be remarkably improved.

Further, only one first stage jack can be formed with a port connected directly to a pressurized liquid supply source by interconnecting liquid chambers for extruding and returning the first stage jack and liquid chambers for extruding and returning the second stage jack respectively through pressurized liquid pipe paths. Thus, compared with the case in which ports communicating to the pressurized liquid supply source are provided in the respective jacks, the number of pipings can be lessened, and damages due to mutual friction of the pipings can be prevented. Since the cylinders of the first and second stage jacks are interconnected, the first and second stage jack cylinders are moved always integrally with each other. Thereby, no friction takes place between the pipings for these cylinders.

Further, by disposing a push ring connected to the second stage jack to be pivoted about said jack can be adjusted the moving speed of both jacks in the contracting operation of the second stage jack forming a part of each assembly to be approximately equal to each other.

The other objects and features of the present invention will become apparent from the following description of a preferred embodiment of the invention with reference to the accompanying drawings.

Fig. 1 is a front view showing a pipe propelling device according to the present invention;

Fig. 2 is a left side view showing the pipe propelling device under the completely contracted condition;

Fig. 3 is a left side view showing the pipe propelling device under the extended condition; and

Fig. 4 is a longitudinal sectional view of the pipe propelling device as viewed from the side, showing schematically a pressurized liquid pipe path.

As shown in Fig. 1, a pipe propelling device 10 according to the present invention comprises two sets of jack assemblies 12, 14 spaced from each other in a vertical shaft and extending laterally (in the direction of propelling the pipe) of the vertical shaft. Both jack assemblies are slidably mounted on a pair of frames 16 disposed on the bottom of the vertical shaft and extending laterally. The frames 16 are made of H-type steel and interconnected by a plurality of connecting members 18 made of H-type steel arranged longitudinally spaced from each other. Also, the frames 16 are on the rear ends fixed by bolts to a box-like reacting support 20 fixed to the reacting shaft wall of the vertical shaft (see Figs. 2 and 3).

As shown in Figs. 1 to 3, each jack assembly comprises a pair of first stage jacks 22, 24 arranged spaced from each other vertically and a second stage jack 26 disposed between the first stage jacks, said first and second stage jacks being interconnected by connecting means 28. The first stage jacks 22, 24 have the same operative stroke and total length.

The first stage jacks 22, 24 have respectively the rod ends 22a, 24a connected to the reacting support 20, and a rod end 26a of the second stage jack 26 are directed forward in the direction of propulsion. Further, a cylinder 26b of the second stage jack extends through the reacting support 20 from the front portion to the rear portion. Thus, the front ends of the cylinders 22b, 24b of the first stage jacks are arranged to be located in front of the rod end 26a of the second stage jack so that a gap between said reacting shaft wall and the rod end 26a of the second stage jack can be lessened, compared with the case of arranging said front ends reversely, provided the second stage jack 26, when subjected to extending operation, needs to be able to extend the rod end 26a forward more than the front ends of cylinders 22b, 24b of the first stage jacks. By such an arrangement, a pipe 30 disposed in front of the rod end 26a to be propelled can be located nearer said reacting shaft wall, and thus the lateral length of the vertical shaft can be set small. In consideration of this point of view, the total length of the second stage jack under the completely contracted condition is preferably shorter than that of the first stage jack under the completely contracted condition. Further, by providing the multiple stage jack like the embodiment shown in the drawings, the second stage jack 26 can be obtained the short total length under the contracted condition, but the large operative stroke.

The connecting means 28 comprises steel blocks 28a, 28b in which the cylinders 22b, 24b of the first stage jacks 22, 24 fit firmly and a steel block 28c in which the cylinder 26b of the second stage jack 26 fits firmly. The blocks 28a, 28c and 28b, 28c

are respectively fixed to each other by a plurality of bolts 31, and further between the respective blocks are arranged keys 32 to prevent positively slippage between the first and second stage jacks.

Further, a plurality of pairs other than one pair of the first stage jacks in each jack assembly may be provided and the quantity of the second stage jack may be at least one, provided it is preferable that to give stable thrust to the pipe 30 and properly pipe 30 efficiently, the respective first stage jacks have the equal capacity, i.e., produced thrust and further the sum of these capacities is equal to the capacity of the second stage jack or to the sum of these capacities when the second stage jacks are plural.

To the second stage jack is connected a push ring 34 for transmitting thrust to the pipe 30. This push ring 34 is arranged on the connecting member 18 of the frame 16 and mounted slidably on a pair of supports 36 extending laterally. The longitudinal dimension of the push ring is maximized in the intermediate position between two sets of jack assemblies 12, 14 and decreased gradually from said intermediate position toward the respective jack assemblies. Also, the push ring 34 is provided with a hole 38 for passing laser beam or the like to measure the direction of propelling the pipe 30.

For the extruding and returning operations of the respective jacks are provided ports communicating to liquid chambers for the extrusion and return in the cylinders. Though pressurized liquid supply pipes may be connected to the respective ports, it is preferable, as shown in Fig. 4, that the extruding and returning liquid chambers of the first stage jacks 22, 24 are connected respectively to the extruding and returning liquid chambers of the second stage jack 26 through pressurized liquid pipe paths.

Extruding and returning ports corresponding to holes for supplying pressurized liquid are provided in the rod ends 22a of the upper first stage jacks.

The extruding pressurized liquid pipe path is made of a path 44 communicating to an extruding liquid chamber 42 in front of a piston 40 from said extruding port through a rod 39 and the piston 40 of the first stage jack, a conduit 48 communicating to the liquid chamber 42 and an extruding liquid chamber 46 in the inner rear portion of the cylinder 26b of the second stage jack, a path 54 extending from the liquid chamber 46 through a piston portion 50a of a first rod 50 of the second stage jack to communicate to an extruding liquid chamber 52 of the first rod 50, a path 60 extending from the liquid chamber 46 through the piston portion 50a of the first rod, a rod portion 50b communicating to the piston portion and a piston portion 50c communicating to the rod portion and received in a hollow portion of a second rod 56 to communicate to an extruding liquid chamber 58 in front of the piston portion 50c and a conduit 64 communicating to the liquid chamber 46 and an

extruding liquid chamber 62 in the inner front portion of the cylinder 24b of the lower first stage jack. Further, referring to said capacity again, the sum of pressure receiving areas of both first stage jack pistons respectively in the extrusion and return is equal to the pressure receiving areas of the piston portions 50a, 50c and rear portion of the second rod 56 of the second stage jack.

When pressurized liquid is supplied to said extruding port through a pressurized liquid supplying conduit 66 connected to a pressurized liquid supply source (not shown), the cylinders 22b, 24b, the first rod 50 and the second rod 56 are simultaneously started to move forward in the propelling direction.

On the other hand, the returning pressurized liquid pipe path is made of a path 70 communicating to a returning liquid chamber 68 behind the piston 40 from said returning port through the rod 39 of the first stage jack, a conduit 74 communicating to the liquid chamber 68 and a returning liquid chamber 72 in the inner front portion of the cylinder 26b of the second stage jack, a path 78 communicating to a returning liquid chamber 76 of the second rod from the liquid chamber 72 through the piston portion 50a and the rod portion 50b of the first rod and a conduit 82 communicating to the liquid chamber 72 and a liquid chamber 80 in the inner rear portion of the cylinder 24b of the lower first stage jack.

When pressurized liquid is supplied to said returning port through a pressurized liquid supplying conduit 84 connected to the pressurized liquid supply source, the cylinders 22b, 24b, the first rod 50 and the second rod 56 are started to move simultaneously backward in the propelling direction.

When said first and second stage jacks are operatively extended and contracted, the connecting means 28 slides on the frame 16 with a pair of lower projecting ends of the block 28b contacting the frame 16, and the push ring 34 and the pipe 30 contacting the push ring on the end slide on the support 36.

Further, the extruding conduits 48, 64 and the returning conduits 74, 82 can be formed to extend respectively through the block 28c, a part of the connecting means 28 for said first and second stage jacks or, as shown in Fig. 4, a path extending through the block 28c may be provided to which said conduit communicates. In either case, since said first and second stage jacks are fixed to each other in these cylinders, each conduit is never affected by the extending and contracting operations. Also, by forming the pressurized liquid pipe path in such a manner, damages of pipings due to friction between pipings are not needed to be taken into consideration.

In the extending and contracting operations of all jacks, particularly in the returning operation of contraction, each jack may not retreat with equal speed due to the fitting condition of a packing for

preventing pressurized liquid from leakage, dimensional error or the like. To avoid such a phenomenon as far as possible, the push ring 34 is preferably connected to the second stage jack 26 through a connecting portion 86 to permit pivoting about said jack.

As shown in Fig. 4, the connecting portion 86 comprises a socket 88 provided in the push ring 34 and a slip-out preventing member 92 received in the socket and fixed to a reduced diameter portion 90 of the second stage jack rod end 26a having the outer diameter converging forward in said propelling direction to prevent the reduced diameter portion 90 from slipping out of said socket. The reduced diameter portion 90 is formed on the front end with threads onto which is screwed the nut-like slip-out preventing member 92. Also, the slip-out preventing member 92 is received in a recess 94 communicating to the socket 88, and a slight gap is provided between the wall surface of the recess and the slip-out preventing member 92.

Accordingly, when speed difference between both second stage jacks 26 is produced when said both jacks 26 are contacted backward in the propelling direction after the pipe 30 is propelled, the push ring 34 is allowed to pivot in the reduced diameter portion 90 within the range of the gap between the socket 88 and the reduced diameter portion 90 or between the slip-out preventing member 92 and the wall surface of the recess 94. Thereby, the push ring 34 is inclined and one of the second stage jack rod ends 26a regulates the retreat movement of the other second stage jack rod end 26a so that both rod ends 26a will move with approximately equal speed.

Instead of above-mentioned construction, said connecting portion may be formed such that the socket 88 has the inner diameter converging forward in said propelling direction and the reduced diameter portion 90 has the constant outer diameter.

The connecting portion having such a construction may be also applied between the first stage jack rod ends 22a, 24a and a base member 96 mounted on the reacting support 20 (see Fig. 4).

Thus, when the frame 16 supporting the jack assemblies 12, 14 is not extended straight, but bent slightly irregularly in the lateral or longitudinal direction, the respective jack assemblies during movement can swing as a whole with respect to the reacting support 20 thus said reacting shaft wall. Thereby, damages of the device caused by said irregularity can be prevented and the propulsion of the pipe can be maintained. Also, by applying said connecting portion, compared with a ball joint used in place of that, the shorter axial length of the jack will do, working and mounting are easily carried out and further the number of parts is reduced.

Claims

1. A pipe propelling device of the type, in which a pipe (30) is propelled from a vertical shaft

having a reacting shaft wall, comprising two sets of jack assemblies (12, 14) supported spaced from each other in said vertical shaft and extending laterally of said vertical shaft, each said jack assembly including: a plurality of first stage jacks (22, 24) having rod ends (22a, 24a) directed backward in the propelling direction of said pipe and connected to said reacting shaft wall, characterized by at least one second stage jack (26) having a cylinder (26b) connected to each of cylinders (22b, 24b) of the first stage jacks and the rod end (26a) directed forward in the propelling direction of said pipe to give a thrusting force to said pipe.

2. A pipe propelling device according to claim 1, characterized in that the front ends of said first stage jack cylinders being located in front of said second stage jack rod end when all said jacks are contracted.

3. A pipe propelling device according to claim 1 or 2, characterized in that extruding liquid chambers (42, 62) and returning liquid chambers (68, 80) of said first stage jack being connected respectively to extruding liquid chambers (46, 52 and 58) and returning liquid chambers (72, 76) of said second stage jack through pressurized liquid pipe paths (48, 54, 60, 64, 74, 78 and 82).

4. A pipe propelling device of the type according to one of claims 1 to 3, characterized by a push ring (34) connected to the second stage jack through a connecting portion (86) such that said push ring is allowed to pivot with respect to the second stage jack.

5. A pipe propelling device as claimed in claim 4, characterized in that said connecting portion (86) comprises a socket (88) provided in said push ring (34) and a member (92) received in said socket and fixed to a reduced diameter portion (90) of said second stage jack rod end (26a) having the outer diameter converging forward in said propelling direction to prevent said reduced diameter portion from slipping out of said socket.

6. A pipe propelling device as claimed in claim 4, characterized in that said connecting portion (86) comprises a socket provided in said push ring (34) and having the inner diameter converging forward in said propelling direction and a member fixed to said second stage jack rod end received in said socket to prevent said rod end from slipping out of said socket.

7. A pipe propelling device as claimed in one of claims 1 to 6, wherein said cylinders (22b, 24b) of the first stage jacks have capacities equal to each other and said cylinder (26b) of the second stage jack has capacity equal to the sum of those of said first stage jack cylinders.

8. A pipe propelling device as claimed in one of claims 1 to 7, wherein said rod ends (22a, 24a) of the first stage jacks are connected such that said rods are allowed to pivot with respect to said reacting shaft wall.

Patentansprüche

1. Rohrvorschubvorrichtung von dem Typ, in

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The connecting portion having such a construction may be also applied between the first stage jack rod ends 22a, 24a and a base member 96 mounted on the reacting support 20 (see Fig. 4).

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Patentansprüche

1. Rohrvorschubvorrichtung von dem Typ, in



dem ein Rohr (30) von einem vertikalen Schacht mit einer rückwirkenden Wand vorgeschoben wird, mit zwei Arbeitszylinderaufbauten (12, 14), die in einem Abstand voneinander in dem vertikalen Schacht getragen sind und sich quer zu dem vertikalen Schacht erstrecken, wobei jeder Arbeitszylinderaufbau eine Mehrzahl von Arbeitszylindern erster Stufe (22, 24) mit sich rückwärts zu der Vorschubrichtung des Rohres erstreckenden und mit der rückwirkenden Schachtwand verbundenen Stangenenden (22a, 24a) aufweist, gekennzeichnet durch einen Arbeitszylinder zweiter Stufe (26) mit einem Zylinder (26b), der mit jedem von Zylindern (22b, 24b) der Arbeitszylinder erster Stufe verbunden ist, und einem in die Richtung des Vorschubes des Rohres vorwärts gerichteten Stangenende (26a) zum Ausüben einer Druckkraft auf das Rohr.

2. Rohrvorschubvorrichtung nach Anspruch 1, dadurch gekennzeichnet, daß die vorderen Enden der Zylinder der Arbeitszylinder erster Stufe vor den Stangenenden der Arbeitszylinder zweiter Stufe angeordnet sind, wenn alle Arbeitszylinder zusammengezogen sind.

3. Rohrvorschubvorrichtung nach Anspruch 1 oder 2, dadurch gekennzeichnet, daß Ausstoßflüssigkeitskammern (42, 62) und Rückkehrflüssigkeitskammern (68, 80) des Arbeitszylinders erster Stufe mit entsprechenden Ausstoßflüssigkeitskammern (46, 52 und 58) und Rückkehrflüssigkeitskammern (72, 76) des Arbeitszylinders zweiter Stufe durch Rohrwege für unter Druck stehende Flüssigkeit (48, 54, 60, 64, 74, 78 und 82) verbunden sind.

4. Rohrvorschubvorrichtung nach einem der Ansprüche 1 bis 3, gekennzeichnet durch einen mit dem Arbeitszylinder zweiter Stufe durch einen verbindenden Abschnitt (86) derart verbundenen Druckring (34), daß der Druckring sich in Beziehung auf den Arbeitszylinder zweiter Stufe schwenken kann.

5. Rohrvorschubvorrichtung nach Anspruch 4, dadurch gekennzeichnet, daß der verbindende Abschnitt (86) eine in dem Druckring (34) vorgesehene Fassung (88) und ein in der Fassung aufgenommenes und an einem Abschnitt (90) mit reduziertem Durchmesser des Stangenendes (26a) des Arbeitszylinders zweiter Stufe aufgenommenes Teil (92) aufweist, mit einem vorwärts in die Vorschubrichtung konvergierenden äußeren Durchmesser zum Verhindern, daß der Abschnitt mit reduziertem Durchmesser aus der Fassung herausrutscht.

6. Rohrvorschubvorrichtung nach Anspruch 4, dadurch gekennzeichnet, daß der verbindende Abschnitt (86) eine in dem Druckring (34) vorgesehene Fassung, bei der der innere Durchmesser vorwärts in die Vorschubrichtung konvergiert, und ein an dem Stangenende des Arbeitszylinders zweiter Stufe befestigtes Teil, das in der Fassung aufgenommen ist um zu verhindern, daß das Stangenende aus der Fassung rutscht, aufweist.

7. Rohrvorschubvorrichtung nach einem der Ansprüche 1 bis 6, bei der die Zylinder (22b, 24b)

der Arbeitszylinder erster Stufe Kapazitäten aufweisen, die einander gleich sind, und der Zylinder (26b) des Arbeitszylinders zweiter Stufe eine Kapazität aufweist, die gleich der Summe der von den Zylindern des Arbeitszylinders erster Stufe ist.

8. Rohrvorschubvorrichtung nach einem der Ansprüche 1 bis 7, bei der die Stangenenden (22a, 24a) der Arbeitszylinder erster Stufe so verbunden sind, daß die Stangen im Verhältnis zu der rückwirkenden Schachtwand schwenken können.

Revendications

1. Dispositif de propulsion de tubes du type dans lequel un tube (30) est propulsé à partir d'un puits vertical qui possède une paroi de réaction, comprenant deux jeux d'ensembles de vérins (12, 14) supportés à un certain écartement l'un de l'autre dans ledit puits vertical et qui s'étendent dans la direction transversale par rapport audit puits vertical, chacun des ensembles de vérins comprenant une pluralité de vérins de premier étage (22, 24) ayant des extrémités de tiges (22a et 24a) qui sont dirigées vers l'arrière relativement à la direction de propulsion dudit tube et qui sont reliées à ladite paroi de réaction du puits, caractérisé par au moins un vérin de deuxième étage (26) qui possède un cylindre (26b) relié à chacun des cylindres (22b, 24b) des vérins de premier étage, et dont l'extrémité de tige (26a) est dirigée vers l'avant, relativement à la direction de propulsion dudit tube, pour imprimer audit tube une force de poussée.

2. Dispositif de propulsion de tubes selon la revendication 1, caractérisé en ce que les extrémités avant desdits cylindres de vérins de premier étage sont placées en avant de ladite extrémité de tige du vérin de second étage lorsque tous lesdits vérins sont rétractés.

3. Dispositif de propulsion de tubes selon l'une des revendications 1 et 2, caractérisé en ce que des chambres à liquide d'extension (42, 62) et des chambres à liquide de retrait (68, 80) desdits vérins de premier étage sont respectivement raccordées à des chambres à liquide d'extension (46, 52 et 58) et à des chambres à liquide de retrait (72, 76) dudit vérin de deuxième étage par l'intermédiaire de circuits de canalisations de liquide sous pression (48, 54, 60, 64, 74, 78 et 82).

4. Dispositif de propulsion de tubes de type selon l'une des revendications 1 à 3, caractérisé par un anneau de poussée (34) relié au vérin de deuxième étage par l'intermédiaire d'une partie de liaison (86) telle, que ledit anneau de poussée puisse pivoter par rapport audit vérin de deuxième étage.

5. Dispositif de propulsion de tubes selon la revendication 4, caractérisé en ce que ladite partie de liaison (86) comprend un alvéole (88) ménagé dans ledit anneau de poussée (34) et un élément (92) logé dans ledit alvéole et fixé à une partie de diamètre réduit (88) de ladite extrémité de tige (26a) du vérin de deuxième étage dont le diamètre extérieur converge vers l'avant, relativement à



ladite direction de propulsion, pour empêcher ladite partie de diamètre réduit de s'échapper dudit alvéole.

6. Dispositif de propulsion de tubes selon la revendication 4, caractérisé en ce que ladite partie de liaison (86) comprend un alvéole ménagé dans ledit anneau de poussée (34) et a son diamètre intérieur qui converge vers l'avant, relativement à ladite direction de propulsion, et un élément fixé à ladite extrémité de tige du vérin de deuxième étage logé dans ledit alvéole pour empêcher ladite extrémité de tige de s'échapper dudit alvéole.

7. Dispositif de propulsion de tubes selon l'une des revendications 1 à 6, caractérisé en ce que lesdits cylindres (22b, 24b) des vérins de premier étage ont des capacités identiques et ledit cylindre (26b) du vérin de deuxième étage possède une capacité égale à la somme de celle desdits cylindres des vérins de premier étage.

8. Dispositif de propulsion de tubes selon l'une des revendications 1 à 7, caractérisé en ce que lesdites extrémités de tiges (22a, 24a) des vérins de premier étage sont reliées de telle manière que lesdites tiges puissent pivoter par rapport à ladite paroi de réaction du puits.

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