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POWER ACTUATED TOOLS WITH POWER ADJUSTMENT MEANS

The present invention relates to power actuated tools for driving fasteners into a substrate, and more particularly to such tools which operate by detonation of an explosive
5 charge.

Power actuated tools for driving fasteners such as nails or pins, into a substrate such as a concrete or steel beam, conventionally comprise a barrel from which the fastener is expelled by means of a piston driven by detonation of an explosive charge. Previously
10 proposed tools of this type are described for example in our International patent applications Nos. PCT/AU88/00255 and PCT/AU90/00018, the disclosure of which is hereby incorporated by reference.

In power actuated tools of this general type, the power output of the tool can be
15 controlled by utilising explosive charges of different power. Alternatively a variable power adjustment system may be incorporated into the tool itself. Such power adjustment systems may operate by altering the volume of the firing chamber defined in the barrel of the tool rearwardly of the piston, or by controlled venting of combustion gases from the portion of the barrel rearwardly of the piston. The present invention relates to an improved power
20 adjustment system which operates by providing a controlled venting of the combustion gases.

According to the present invention there is provided an explosively-actuated tool for use in setting fasteners into a substrate, comprising a barrel, a driving piston mounted within the barrel for driving the fastener into a substrate upon detonation of an explosive charge
25 positioned at or adjacent the rear of the barrel, a firing chamber defined in the barrel rearwardly of the piston, and means for controlling the power of the tool by controlled venting of combustion gases from the firing chamber, said power control means comprising an exhaust port arrangement leading from the firing chamber through the wall of the barrel, and a control member for controlling the effective open area of the exhaust port arrangement.

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Advantageously, the exhaust port arrangement comprises one or more radial bores extending through the wall of the barrel. When there are two or more such bores, the bores are in axially-spaced arrangement relative to the barrel.

5 Advantageously, the exhaust port arrangement opens into an axial bore or other passage within the wall of the barrel, with the control member being defined by a control rod which extends into the passage by an adjustable distance.

Preferably, the control rod extends into the passage from the rear end of the passage
10 with the front end of the passage being open for discharge of the gases from the firing chamber via the exhaust port arrangement. Advantageously, the position of the control rod within the passage is controlled by a mechanism operative to cause infinite or stepwise movement of the control rod within the passage. Such a mechanism may be a screw mechanism.

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Particularly advantageously, the end of the control rod is provided with a concave arcuate control edge which provides a relatively uniform relationship between displacement of the control rod and the degree of opening of the exhaust port arrangement. Such a control edge can be formed by a simple chamfer at the end of a cylindrical control rod with the
20 passage also being cylindrical.

An embodiment of the invention will now be described, by way of example only, with reference to the accompanying drawings in which:-

Figure 1 is a schematic sectional view showing the rear end portion of the barrel of
25 a power actuated tool in accordance with the preferred embodiment of the invention and illustrating in particular a control mechanism for providing controlled venting of combustion gases from the firing chamber;

Figure 2 is an enlarged view similar to Figure 1 and illustrating in greater detail a control rod of the control mechanism and its relationship with the exhaust port leading from
30 the firing chamber; and

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Figures 3A to 3C illustrate schematically the progress of the control edge of the control rod across the array of holes forming the exhaust port.

Figure 1 shows part of the barrel 2 of a power actuated tool, the barrel 2 being mounted within a main body 4 of the tool for axial movement. The barrel 2 houses a fastener driving piston 6. A charge chamber 8 is defined at the rear end of the barrel 2 for receiving an explosive charge, and a firing chamber 10 is defined between the rear face 6a of the piston 6 and the rear end of the barrel 2. The basic operation of the tool is substantially as described in our earlier applications as aforesaid, with the axial movement of the barrel 2 within the body 4 serving to facilitate cocking of the tool when the forward end of the barrel 2 is pressed against the work surface, and also to reset the piston 6 within the rear part of the barrel 2 after firing by drawing the barrel 2 forwardly of the body 4.

The power control system for the tool in accordance with the preferred embodiment of the invention will now be described.

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The wall 2a of the barrel 2 around the firing chamber 10 is of an increased thickness so that its outer diameter closely matches that of the inner surface of the body 4. This section of the barrel wall is provided with an axial through-bore 12 which communicates with the firing chamber 10 via one, two, or more radial passages 14 spaced axially along the bore 12, the or each passage 14 collectively defining an exhaust port. A control rod 16 extends into the axial bore 12 through the rear end thereof and can be positioned to a variable distance within the bore 12 so as to control the effective area of communication between the firing chamber 10 and the axial bore 12 as defined by the open area of the exhaust port, the cross-section of the rod 16 closely conforming to that of the bore 12. As will be apparent, axial movement of the control rod 16 within the bore 12 will cause the forward end portion 16a of the rod 16 to move across the outer ends of the passages 14 and hence to open or close the passages 14 to a variable extent. The forward end of the bore 12 opens into an annular space 18 defined between the barrel 2 and the body 4, and this annular space communicates with a main outlet port (not shown) which discharges into the atmosphere. By varying the extent to which the exhaust port defined by the array of passages 14 is open, by varying the axial

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position of the control rod 16 within the bore 12, a variable quantity of the gases generated on detonation of the explosive charge will be exhausted directly from the firing chamber 10 thereby varying the effective power of the tool. When the exhaust port is fully closed, the tool will be at maximum power for a given charge, and when the exhaust port is fully open, 5 the tool will be at minimum power for a given charge. The control rod 16 is capable of infinitely variable movement or, alternatively, stepwise movement, between the fully open and fully closed configurations of the exhaust port whereby to provide a range of intermediate power positions between the maximum and minimum.

10 The position of the control rod 16 within the bore 12 can be altered by any form of suitable adjustment mechanism. However, as shown, this is achieved by means of a simple screw mechanism comprising a screw 20 which is journaled in the body of the tool, and which is rotatable by means of a knurled knob 22 which is either located externally of the body, or which projects through a slot in the body to facilitate manipulation by an operator. 15 The threaded shank of the screw 22 is engaged in a threaded block 24 carried at the rear end of the control rod 16. Forwardly of the threaded block 24, the control rod 16 passes through an axial bore within a breach block (not shown) which is mounted within the body 4 immediately rearwardly of the charge chamber 8. The bore in the breach block accurately locates the control rod 16 so that when the barrel 2 is moved forwardly relative to the body 20 to reset the piston 6 after firing, the rear end of the barrel 2 will move forwardly out of engagement with the control rod 16, and when the barrel 2 is returned to its rear position its axial bore 12 will align with the end of the control rod 16 and move over the control rod 16. The knob 22 or the screw 20 may be associated with a detent device to provide a number of discrete adjustment positions between maximum and minimum. The knob 22 may carry 25 suitable markings, or the control rod may be linked to an indicator slide (not shown) so that the power adjustment can easily be determined by the operator.

The passages 14 which define the exhaust port can simply be formed by radial holes formed by drillings extending between the outside and inside surfaces of the barrel, with the 30 drillings being axially spaced. This is a much simpler and less expensive process than cutting

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an elongate slot between the firing chamber and axial bore 12 to provide a port which is capable of selective opening and closure by means of the control rod. As illustrated, the individual holes are spaced so that a land 14b is formed between adjacent holes. The formation of an exhaust port of elongate form by means of two or more drilled holes with a
5 land separating adjacent holes also provides advantages over an exhaust port in the form of an elongate slot in that the presence of the or each land between the adjacent holes forms a barrier within the exhaust port itself to unwanted by-pass flow of combustion gas across the peripheral surface of the rear end of the piston where a seal exists between the piston and wall of the barrel.

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As will be apparent, when the exhaust port is defined by one or more drilled holes, a control rod of a cylindrical shape at its forward end will exhibit an essentially transverse "straight" control edge. As the control edge moves across the circular face of each hole and across the land separating adjacent holes, the relationship between displacement of the control
15 rod and the degree of opening of the exhaust port is very significantly non-linear to the extent that during part of the movement of the control rod where the control edge moves from one hole across the land and onto the adjacent hole, this will not result in any alteration in the effective open area of the exhaust port arrangement. We have overcome this disadvantage by forming the forward end of the control rod 16 with a chamfer 16b whereby the effective
20 control edge of the control rod as defined at the base of the chamfer and which traverses the outer faces of the holes 14 is of a concave arcuate form (particularly an elliptical shape) and it is this concave edge which effectively forms the control edge. This is illustrated schematically in Figures 3A to 3C which illustrates the progression of control edge (designated as 16c) across the array of holes 14 when the control rod 16 is moving in a
25 direction indicated by the arrow to close the exhaust port. It is to be noted that while the control edge 16c is moving across the land separating two adjacent holes 14, it is still acting to progressively close the preceding hole 14 and will still do so when it starts to close the following hole. It will be appreciated that throughout the range of movement of the control rod 16 there will always exist a progressive opening or closure of the exhaust port without
30 the significant "flat spot" which is present when a control rod with a plain cylindrical end

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traverses a land between two adjacent holes. As will also be apparent with an essentially "straight" transverse control edge provided by a plain cylindrical end of a control rod there will be substantial variation in the rate of change of opening/closure as it moves across the face of a cylindrical hole, but with a concave arcuate control edge this rate of change becomes
5 rather more uniform as the control edge moves across the hole. Accordingly the presence of the arcuate concave control edge provided by the chamfer 16b will lead to a more uniform relationship between the axial adjustment of the control rod 16 provided by rotation of the screw mechanism and the degree of opening/closure of the exhaust port and hence the power control adjustment; this is achieved irrespective of whether there is a single drilling to form
10 the exhaust port or a series of two or more axially-spaced drillings to form the exhaust port. When the exhaust port consists of two or more drillings, they may be of the same or different diameters to obtain a desired relationship between control rod position and power output.

The embodiment has been described by way of example only and modifications are
15 possible within the scope of the invention.



CLAIMS:-

1. An explosively-actuated tool for use in setting fasteners into a substrate, comprising a barrel, a driving piston mounted within the barrel for driving the fastener into a substrate
5 upon detonation of an explosive charge positioned at or adjacent the rear of the barrel, a firing chamber defined in the barrel rearwardly of the piston, and means for controlling the power of the tool by controlled venting of combustion gases from the firing chamber, said power control means comprising an exhaust port arrangement leading from the firing chamber through the wall of the barrel, and a control member for controlling the effective open area
10 of the exhaust port arrangement.
2. A tool according to claim 1, wherein the exhaust port arrangement comprises at least one radial bore extending through the wall of the barrel.
- 15 3. A tool according to claim 2, wherein there are two or more said bores, said bores being in axially-spaced arrangement relative to the barrel.
4. A tool according to any one of claims 1 to 3, wherein the exhaust port arrangement opens into a passage within the wall of the barrel, the control member being defined by a
20 control rod which extends into the passage by an adjustable distance.
5. A tool according to claim 4, wherein the passage is defined by an axial bore within the wall of the barrel.
- 25 6. A tool according to claim 5, wherein the control rod extends into the passage from a rear end of the passage, a front end of the passage being open for discharge of the gases from the firing chamber via the exhaust port arrangement.
7. A tool according to any one of claims 4 to 6, wherein the position of the control rod
30 within the passage is controlled by a mechanism operative to cause infinite or stepwise



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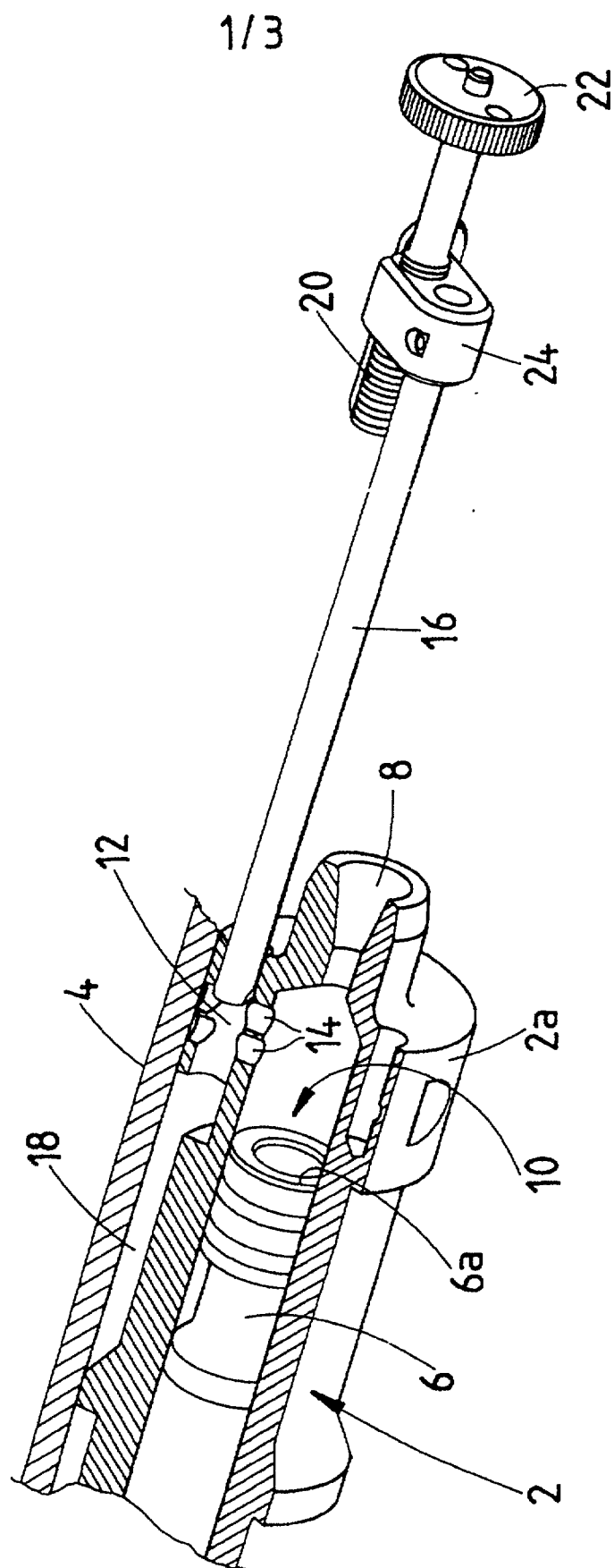
movement of the control rod within the passage.

8. A tool according to claim 7, wherein the mechanism is a screw mechanism.

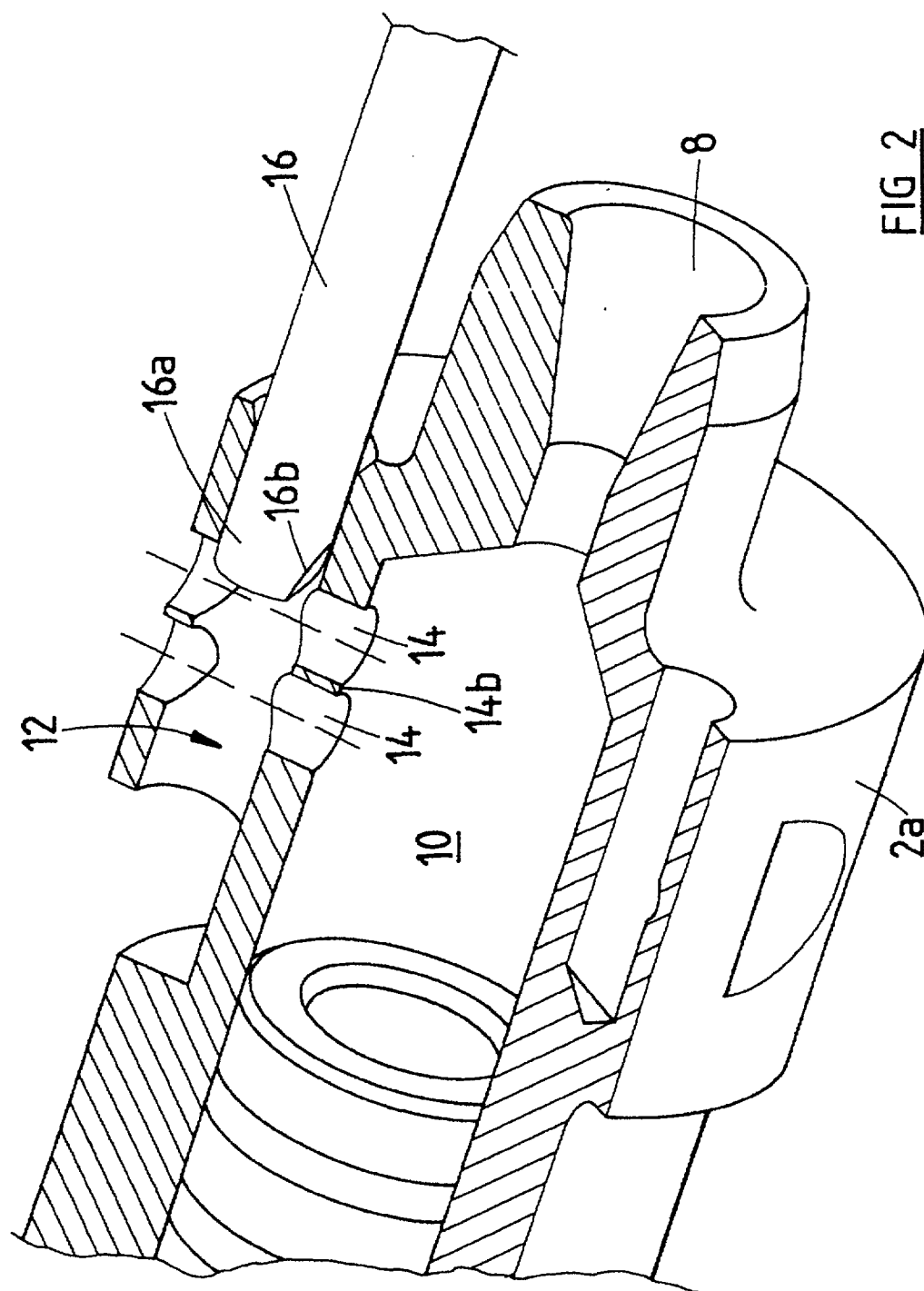
5 9. A tool according to any one of claims 4 to 8, wherein a forward end of the control rod is provided with a concave arcuate control edge operative to provide a generally uniform relationship between displacement of the control rod and the degree of opening of the exhaust port arrangement whereby to achieve progressive opening or closure of the exhaust port arrangement substantially throughout the range of movement of the control rod.

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10. A tool according to claim 9, wherein the control rod and passage are cylindrical and the control edge is formed by a chamfer at the end of the cylindrical control rod.



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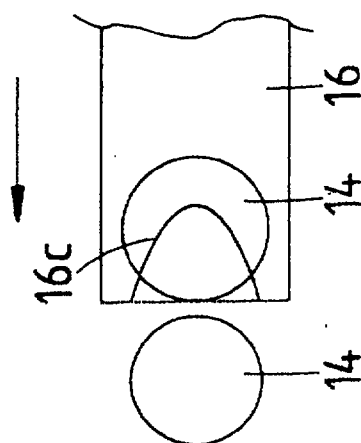


FIG 3A

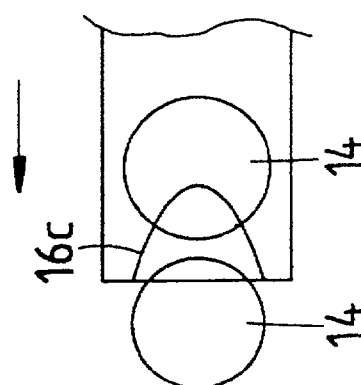


FIG 3B

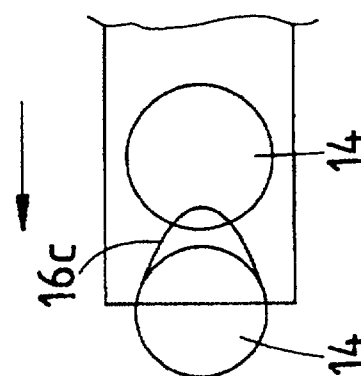


FIG 3C

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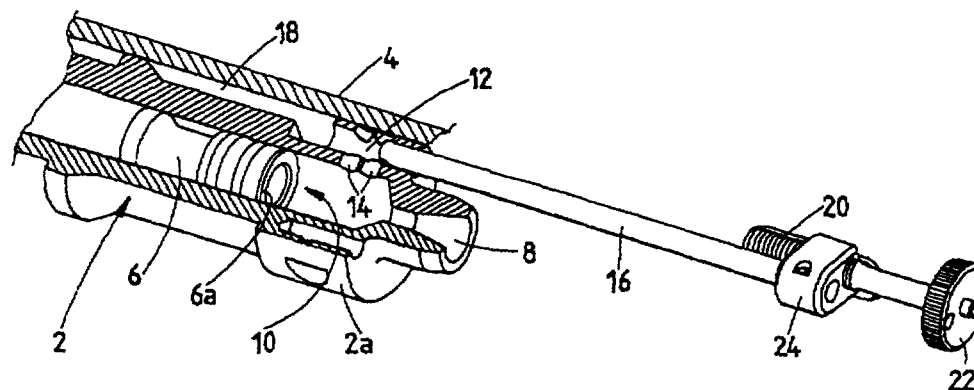
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(71) Applicant (for all designated States except US): RAMSET FASTENERS (AUST.) PTY. LIMITED [AU/AU]; Maroon-dah Highway, Croydon North, VIC 3136 (AU).			
(71)(72) Applicants and Inventors (for all designated States except US): RENSHAW, Brian, Douglas [AU/AU]; 11 Jennifer Drive, Morphett Vale, S.A. 5162 (AU). CONNELL, Robert, Urquhart [AU/AU]; 11 Goodall Street, Hawthorn, VIC 3122 (AU).			
(72) Inventor; and (75) Inventor/Applicant (for US only): CLARK, Philip, Charles [AU/AU]; 33 Wildwood Avenue, Vermont South, VIC 3133 (AU).			
(74) Agents: HIND, Raymond, Stenton et al.; Davies Collison Cave, 1 Little Collins Street, Melbourne, VIC 3000 (AU).			

(54) Title: POWER ACTUATED TOOLS WITH POWER ADJUSTMENT MEANS



(57) Abstract

According to the present invention there is provided an explosively-actuated tool for use in setting fasteners into a substrate, comprising a barrel (2), a driving piston (6) mounted within the barrel for driving the fastener into a substrate upon detonation of an explosive charge positioned at or adjacent the rear of the barrel, a firing chamber (8) defined in the barrel rearwardly of the piston, and means for controlling the power of the tool by controlled venting of combustion gases from the firing chamber, said power control means comprising an exhaust port arrangement (14) leading from the firing chamber through the wall of the barrel, and a control member (16) for controlling the effective open area of the exhaust port arrangement. Particularly advantageously, the end of the control rod is provided with a concave arcuate control edge (16b) which provides a relatively uniform relationship between displacement of the control rod and the degree of opening of the exhaust port arrangement. Such a control edge can be formed by a simple chamfer at the end of a cylindrical control rod (16) with the passage also being cylindrical.