partition wall position in a conventional washing machine shown as Comparative Example 2. FIG. 11 is a characteristic graph showing changes in vibration system amplitude during spin-drying in the Comparative Example.

As shown in FIG. 10, balancer device 207 of Comparative Example 2, in common with balancer device 7 of the exemplary embodiment, has an outside diameter of 470 mm, with internal height H2 of annular chambers 233, 234 set to 70 mm. Meanwhile, balancer device 207 has an inside diameter of 381 mm, with a radial width of annular body 235 set to 44.5 mm. Distance H1 from liquid level 38a of liquid 38 to inner peripheral-side lower ends of upper portions 236a, 237a of partition walls 236, 237 is set to 5 mm.

In other words, balancer device 207 of Comparative Example 2 is similar to the balancer device of Comparative Example 1 in distance H1 from liquid level 38a to lower ends of the upper portions 236a, 237a of partition walls 236, 237. Meanwhile, the inside diameter (381 mm) of balancer device 207 is smaller than the inside diameter (404 mm) in Comparative Example 1 shown in FIG. 8, so that annular body 235 has an increased radial width of 44.5 mm. Thus, a channel for liquid 38 in Comparative Example 2 is wider than the channel for liquid 38 in Comparative Example 1. This enables liquid 38 to smoothly move in the wide channel after a start of spin-drying. This also enables liquid 38 to smoothly move to a position opposite an imbalance (a delay of 180°) when the rotation speed of wash and spin tub 4 increases and reaches a spin-drying rotation speed of 200 r/min. As a result, as shown in FIG. 11, the balancer device provides an improved vibration damping effect at a secondary resonance point.

However, balancer device 207 of Comparative Example 2, as described above, has a decreased inside diameter of 381 mm. This configuration lowers the workability for loading and unloading laundry into and from wash and