

Description

This invention relates to cup infeeders and more particularly relates to a continuously rotating feeder which transports cups to a receiving station where the cups are indexed for engagement by a reciprocating tool that moves parallel to the axis of rotation for the rotating feeder.

The main section or body of a so-called two piece metal container includes an elongated cylindrical sidewall, an integral bottom and an open top. Such bodies are often formed in drawing and ironing machines of the type described in the E. Paramonoff, United States Patents Nos. 3,704,619 issued December 5, 1972 and 3,735,629 issued May 29, 1973 entitled, respectively, Redraw Blankholder Positioning Mechanism for Cup-Shaped Article Formers Such as Metallic Can Body Formers and the Like and Apparatus for Forming One Piece Metallic Can Bodies. Such machines produce can bodies from blanks, in the form of shallow cups, by having a reciprocated ram drive each cup through a die pack which is a series of die elements having openings that are graduated so that the blank passes through the largest opening first and each subsequent opening that the blank is driven through is slightly smaller than the preceding opening through which the blank has been driven.

In prior art apparatus of this type the cups are transferred from a gravity feed chute to a receiving station through which the ram travels. A linearly reciprocated feed element is disclosed in U.S. Patent No. 4,534,202 issued August 13, 1985 to W. W. Snyder for Cup Feeding Mechanism, and a pivoted feed member is disclosed in U.S. Patent No. 4,061,012 issued December 8, 1977 to E. F. Wessman for a Drawing and Ironing Machine With Positive Cup Feeder.

US-A-3,289,453 discloses an apparatus for processing circular-sided cup-like parts comprising supply means, tool means, means for reciprocating the movable section, feeding means, a rotatable member having pocket means with an arcuate lead-in formation, which extends from the pocket means in the rotational direction of the member and recedes on moving past the exit and supports a circular-sided part as it moves into the receiving station, and means for coordinating the rotations of the rotatably mounted member with the movement of the movable section. The rotatable members are caused to rotate intermittently in operation of the apparatus.

Utilization of reciprocated and/or rocking type feed mechanisms severely limits production rates of prior art machines and often presented maintenance problems.

The instant invention overcomes the limitations which linearly reciprocated and rocking type feed

mechanisms imposed on prior art drawing and ironing machines by providing a feed mechanism that comprises a single continuously rotating member having a pocket that receives a blank which is a circular-sided cup-like part as it is gradually lowered from a feed chute by a lead-in surface that extended into the pocket. The feed member positively moves the cup along an arcuate guide and into a receiving station where the blank is positioned for engagement by the ram as it moves forward in its working stroke. A stationary stripper removes the blank from the pocket upstream of the receiving station means but the feed member continues to positively drive the blank to engagement with registry means at the receiving station. When the blank engages the registry means a portion of the feed member cooperates with the registry means to hold the blank in registered position until it is engaged by the ram and/or by a movable, clamping pad that reciprocates in association with the ram.

Accordingly, the primary object of the instant invention is to provide a continuously rotating mechanism for feeding blanks to a receiving station through which tool means operates.

Another object is to provide feeding means of this type that is adapted for a drawing and ironing machine which transforms metal cups into one piece can bodies.

Still another object is to provide a feeder of this type that rotates through a complete revolution for each operation of a reciprocated ram that engages a workpiece which is delivered by the feed member after being released by the latter.

A further object is to provide a feed mechanism of this type having means that cooperates with registry means to hold a blank in position for engagement by a tool.

A still further object is to provide a feed mechanism of this type which limits abrupt movement of the blanks that are disposed within a gravity feed chute.

According to the present invention there is provided an apparatus for processing circular-sided parts comprising

supply means wherein circular-sided cup like parts are disposed side-by-side in a stack, the supply means including an exit through which the circular-sided parts leave one at a time to be transferred to a receiving station,

tool means comprising a movable section mounted to move along a path that extends through the receiving section,

a first means for reciprocating the movable section along the path between a forward position and a return position behind the receiving station, said movable section, when moving forward in a working stroke, engaging a circular-sided part dis-

posed at the receiving station and,

feeding means for separating the most downstream cup-like part in the stack from the remaining cup like parts in the stack and transferring the circular sided parts from the said exit to the receiving station

in which the feeding means comprise

a rotatably mounted member having pocket means to receive a circular-sided part and an arcuate lead-in formation which extends from the pocket means in the rotational direction of the member and gradually recedes on moving past the exit and supports a circular-sided part as it moves into the receiving station, and

means for coordinating the rotations about an axis of the rotatably mounted member with the movement of the movable section whereby a circular-sided part is disposed at the receiving station for each working stroke of the movable section, characterised in that the rotatably mounted member is rotated continuously.

These objects as well as other objects of this invention shall become readily apparent after reading the following description the accompanying drawings of one embodiment of the invention in which:

Fig. 1 is a perspective of a cup-shaped article forming machine incorporating the continuous rotary feed mechanism constructed in accordance with the instant invention.

Fig. 2 is an enlarged fragmentary somewhat diagrammatic view of the die pack section in the machine of Fig. 1 looking in the direction of arrows 2-2 of Fig. 1 with the ram and redraw blank holder pad in their forward or can body forming positions.

Fig. 3 is an enlarged rear elevation of the rotary feed mechanism.

Fig. 4 is a cross-section taken through line 4-4 of Fig. 3 looking in the direction of arrows 4-4.

Fig. 5 is an enlarged fragmentary horizontal section showing the ram and blank holder pad of the tool means retracted prior to engagement with a blank.

Fig. 6 is a cross-section taken through lines 6-6 of Fig. 5 looking in the direction of arrows 6-6 with the blank holder pad in its forward clamping position.

Figs. 7, 8 and 9 are simplified rear elevations of the rotary feed mechanism showing various positions for a blank as it leaves the gravity feed chute (Fig. 7) until it is in engagement with the registry means at the receiving station and disposed for engagement by the movable tool elements (Fig. 9).

Now referring to the Figures. Rotary feed mechanism 10 (Fig. 3) of this invention is incorporated in otherwise conventional drawing and iron-

ing machine 11 of Fig. 1. The latter includes main frame 20 having main drive mechanism 22 mounted thereon for reciprocating a movable member which is a ram 24 (Fig. 2) along a horizontal feed path from a rearward reversing position (Fig. 5) forwardly through receiving station 25 of feeding mechanism 10, the forward direction being from right to left with respect to Fig. 2. Forward of feeding mechanism 10 ram 24 passes through die pack 28 and, upon reaching the position illustrated in Fig. 2, reverses and returns to the position illustrated in Fig. 5.

Body former 11 receives shallow circular-sided cup-shaped parts 30 that are disposed side-by-side in gravity chute 26. Parts 30 exit one at a time from the bottom of chute 26 and are transformed into elongated one piece can bodies 32 (Fig. 2). That is, the cylindrical sidewall of cup 30 is elongated and ironed by passing through a series of ring-shaped dies 33a - 33d, being driven into ram 24. During initial forward movement of ram 24 in its working stroke, ram 24 is preceded by blank holder pad 34 (Figs. 5 and 6). The latter is mounted to movable bend holder frame assembly 98 positioned in front of cross-arm 99 of main frame 20. As assembly 98 moves forward relative to cross-arm 99 from its return or retracted position of Fig. 5 to its clamping position of Fig. 6, pad 34 enters part 30 through its rear facing open end thereof (Fig. 6) and clamps bottom 31 of part 30 against the first die ring 33a. Then ram 24 moves through assembly 98 including central guide bore 35 of pad 34, engages bottom 31 and drives it forward through die rings 33a - 33d and finally into engagement with doming formation 36 (Fig. 2).

With particular reference to Figs. 3, 4 and 7 through 9, it is seen that feed mechanism 10 includes rotary feed member 40 that is keyed to continuously rotating horizontal shaft 41. The periphery of feed member 40 is disposed to move below and in proximity to the bottom or exit end of chute 26. Part 30 shown in phantom, in Fig. 3 is positioned at the bottom of chute 26 for removal through the exit thereof. At this time part 30 is supported by lead-in surface portion 40a along the edge of feed member 40. Lead-in surface portion 40a is so shaped that while it supports can 30 the latter gradually moves downward away from the exit of chute 26. Finally, part 30 is received in pocket 40b (Fig. 7) of feed member 40 and is driven along the upper main arcuate portion 42 of guide wall 43. The lower or terminal portion 44 of guide wall 43 is generally straight and generally parallel to edge 46 of stripper 45. Narrow slot 51 (Fig. 4) in the edge of rotating feed member 40 provides clearance for stripper 45. When part 30 moves between guide portion 44 and edge 46, stripper 45 forces blank 30 out of pocket 40b.

However, feed member 40 continues to drive blank 30 downward until it reaches receiving station 25 where arcuate indexing or registry formation 48 arrests movement of part 30 in a position aligned with ram 24 and clamping pad 34 (Fig. 5). While clamping pad 34 moves from the rear position of Fig. 5 to the clamping position of Fig. 6, edge portion 40c of member 40 that extends immediately upstream from pocket 40b locks part 30 against registry formation 48 (Fig. 9).

Edge portion 40c which constitutes a holding means, is the part of feed member 40 that is most distant from the rotational axis thereof, and shaft 41 is positioned so that no portion of member 40 passes across the feed path of ram 24 so that the latter cannot engage feed member 40 in the event these elements are out of synchronization.

In the event of a malfunction, solenoid operated plunger 55 (Fig. 3) is actuated to extend into feed chute 26 and stop parts 30 from moving therethrough.

While rotary feed member 40 is a single lobe element having a single pocket 40b, it is noted that in the absence of size considerations a multilobed, multipocketed feed member may be provided so long as its rotational speed is coordinated with the reciprocating motion of ram 24 and clamping pad 35. The single lobe construction illustrated is appealing in that the feed member 40 and movable tool elements 24 and 35 operate on a one to one basis. That is, for each complete revolution of feed member 40 tool members 24 and 35 move forward and rearward through a complete cycle.

The power to rotate shaft 41 of feed member 40 is supplied by main drive 202 (Fig. 4) which is connected through clutch 203 and a timing belt 204 to normally rotate sprocket 205 that is keyed to shaft 41. When feed member 40 engages a blank having an oval sidewall or other defect that causes it to wedge against guide wall 43, the force required to rotate feed member 40 increases. When this required force exceeds a predetermined value the driving connection broken between the input 206 and output 207 of clutch 203 whereby the driving connection between main drive 202 and feed member 40 is broken so that the latter ceases to deliver blanks to receiving station 25.

A braking force is applied automatically to clutch output 207 when the forward or working stroke of ram 24 fails to produce a properly formed elongated can body 32. For example, when a blank 30 has a defective sidewall, it is not uncommon for the front of blank 30 to separate from the remainder (rear) of blank 30 as ram 24 moves forward with this remainder of blank 30 remaining at the receiving station 25 to interfere with entry of the next blank 40 into station 25. More particularly, when ram 24 reaches the end of its forward stroke

sensor 208 (Fig. 2) determines whether cam body 32 is defective by detecting if its sidewall is too short. If this defect condition is found to exist sensor 208 generates a signal which actuates brake 209 which applies an overboard force to clutch output 207. This breaks the driving connection between clutch input 206 and clutch output 207 so that the driving connection between main drive 202 and feed member 40 is broken.

Since the rotating feed member 40 has relatively low inertia the braking force applied to clutch output 207 coupled with removal of driving power from the latter causes feed member 40 to stop very quickly, say approximately a half cycle. Because the reciprocating ram 24 has relatively high inertia, it takes much longer to stop, say approximately one and a half cycles after driving power is removed therefrom and a braking force is applied thereto. However, this failure of ram 24 to stop instantaneously does not cause additional damage. Thus, for a second time ram 24 will merely pass through the rear portion of part 30 that was left behind at receiving station 25.

Claims

1. An apparatus for processing circular-sided parts comprising

supply means wherein circular-sided cup-like parts (30) are disposed side-by-side in a stack, the supply means including an exit through which the circular-sided parts leave one at a time to be transferred to a receiving station (25),

tool means comprising a movable section (24) mounted to move along a path that extends through the receiving section (25),

a first means for reciprocating the movable section along the path between a forward position and a return position behind the receiving station (25), said movable section, when moving forward in a working stroke, engaging a circular-sided part (30) disposed at the receiving station (25) and,

feeding means for separating the most downstream cup-like part in the stack from the remaining cup-like parts in the stack and transferring the circular sided parts (30) from the said exit to the receiving station (25)

in which the feeding means comprise

a rotatably mounted member (40) having pocket means (40b) to receive a circular-sided part (30) and an arcuate lead-in formation (40a) which extends from the pocket means in the rotational direction of the member and gradually recedes on moving past the exit and supports a circular-sided part (30) as it moves into the receiving station, and



However, feed member 40 continues to drive blank 30 downward until it reaches receiving station 25 where arcuate indexing or registry formation 48 arrests movement of part 30 in a position aligned with ram 24 and clamping pad 34 (Fig. 5). While clamping pad 34 moves from the rear position of Fig. 5 to the clamping position of Fig. 6, edge portion 40c of member 40 that extends immediately upstream from pocket 40b locks part 30 against registry formation 48 (Fig. 9).

Edge portion 40c which constitutes a holding means, is the part of feed member 40 that is most distant from the rotational axis thereof, and shaft 41 is positioned so that no portion of member 40 passes across the feed path of ram 24 so that the latter cannot engage feed member 40 in the event these elements are out of synchronization.

In the event of a malfunction, solenoid operated plunger 55 (Fig. 3) is actuated to extend into feed chute 26 and stop parts 30 from moving therethrough.

While rotary feed member 40 is a single lobe element having a single pocket 40b, it is noted that in the absence of size considerations a multilobed, multipocketed feed member may be provided so long as its rotational speed is coordinated with the reciprocating motion of ram 24 and clamping pad 35. The single lobe construction illustrated is appealing in that the feed member 40 and movable tool elements 24 and 35 operate on a one to one basis. That is, for each complete revolution of feed member 40 tool members 24 and 35 move forward and rearward through a complete cycle.

The power to rotate shaft 41 of feed member 40 is supplied by main drive 202 (Fig. 4) which is connected through clutch 203 and a timing belt 204 to normally rotate sprocket 205 that is keyed to shaft 41. When feed member 40 engages a blank having an oval sidewall or other defect that causes it to wedge against guide wall 43, the force required to rotate feed member 40 increases. When this required force exceeds a predetermined value the driving connection broken between the input 206 and output 207 of clutch 203 whereby the driving connection between main drive 202 and feed member 40 is broken so that the latter ceases to deliver blanks to receiving station 25.

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sensor 208 (Fig. 2) determines whether cam body 32 is defective by detecting if its sidewall is too short. If this defect condition is found to exist sensor 208 generates a signal which actuates brake 209 which applies an overboard force to clutch output 207. This breaks the driving connection between clutch input 206 and clutch output 207 so that the driving connection between main drive 202 and feed member 40 is broken.

Since the rotating feed member 40 has relatively low inertia the braking force applied to clutch output 207 coupled with removal of driving power from the latter causes feed member 40 to stop very quickly, say approximately a half cycle. Because the reciprocating ram 24 has relatively high inertia, it takes much longer to stop, say approximately one and a half cycles after driving power is removed therefrom and a braking force is applied thereto. However, this failure of ram 24 to stop instantaneously does not cause additional damage. Thus, for a second time ram 24 will merely pass through the rear portion of part 30 that was left behind at receiving station 25.

Claims

1. An apparatus for processing circular-sided parts comprising

supply means wherein circular-sided cup like parts (30) are disposed side-by-side in a stack, the supply means including an exit through which the circular-sided parts leave one at a time to be transferred to a receiving station (25),

tool means comprising a movable section (24) mounted to move along a path that extends through the receiving section (25),

a first means for reciprocating the movable section along the path between a forward position and a return position behind the receiving station (25), said movable section, when moving forward in a working stroke, engaging a circular-sided part (30) disposed at the receiving station (25) and,

feeding means for separating the most downstream cup-like part in the stack from the remaining cup-like parts in the stack and transferring the circular sided parts (30) from the said exit to the receiving station (25)

in which the feeding means comprise

a rotatably mounted member (40) having pocket means (40b) to receive a circular-sided part (30) and an arcuate lead-in formation (40a) which extends from the pocket means in the rotational direction of the member and gradually recedes on moving past the exit and supports a circular-sided part (30) as it moves into the receiving station, and



means for co-ordinating the rotations about an axis of the rotatably mounted member (40) with the movement of the movable section (24) whereby a circular-sided part is disposed at the receiving station for each working stroke of the movable section, characterised in that the rotatably mounted member is rotated continuously.

2. Apparatus according to claim 1 also including a guide (42) and indexing means (48) for guiding a circular-sided part into the receiving station (25) for engagement by the movable section (24).

3. Apparatus according to claim 1 or claim 2 in which the rotatably mounted member also includes a holding formation to maintain a circular-sided part (30) in position in the receiving station (25) until engaged by the movable section and

the holding formation is provided along an edge of the rotatably mounted member and extends from the pocket means in a direction opposite to the direction of rotation of the member, the said edge being the part of rotatably mounted member that is most distant from the rotational axis thereof.

4. Apparatus according to claim 2 or claim 3 in which the exit is positioned above the indexing means (48).

5. Apparatus according to any preceding claim also including stripper means (45) to remove circular-sided parts from the pocket means (40b) before such parts reach said indexing means (48).

6. Apparatus according to any of claims 2 to 5 in which the feeding means drives circular-sided parts (30) while they move along the guide means (42) and even after they are removed from the pocket means (40b) by the stripper means (45).

7. Apparatus according to any preceding claim in which the rotatably mounted member (40) comprises a single lobe and the pocket means (40b) comprises a single pocket and the rotatably mounted member (40) moves through a single revolution each time the movable section (24) moves through its working stroke and back to the return position.

8. Apparatus according to any preceding claim in which the circular-sided parts (30) are cups that have rear facing entrances and the movable section (24) comprises a ram that enters the cups during the forward stroke of the movable section;

the tool means also comprise a stationary section disposed along the path forward of the receiving station to cooperate with the ram in operating on the cups; and

the stationary section includes die means (33) that encircle the path and engages the exterior of each cup along its sidewall when the cup is driven forward by the ram.

9. Apparatus according to any preceding claim in which the rotatably mounted member (40) is continuously rotated by

a second means which includes clutch means (203) to interrupt driving engagement between the second means and the member (40) automatically when the power required to drive the member (40) exceeds a predetermined level, and

a third means for automatically operating the clutch means (203) to interrupt driving engagement between the second means and the member (40) upon detecting that travel of the movable section (24) to the forward position has failed to convert a circular-sided part (30) to a properly formed product.

10. Apparatus according to claim 8 or claim 9 in which the die means (33) includes a plurality of die sections disposed one in front of the other

and each of the die sections (33a, 33b, 33c, 33d) has an aperture that encircles the path, and the apertures are graduated in diameter and each of the apertures is arranged behind the apertures of smaller diameter and forward of the apertures of larger diameter.

Patentansprüche

1. Vorrichtung zum Verarbeiten von Teilen mit kreisförmigen Seiten, umfassend einer Zuführvorrichtung, in welcher büchsenartige Teile (30) mit kreisförmigen Seiten nebeneinander in einem Stapel angeordnet sind, wobei die Zuführvorrichtung einen Auslaß aufweist, durch welchen die Teile mit kreisförmigen Seiten jeweils einzeln hindurch treten, um einer Aufnahmestation (25) zugeführt zu werden;

Werkzeuge, umfassend einen beweglichen Abschnitt (24), der derart angeordnet ist, daß er sich entlang einer Wegstrecke bewegt, die sich durch die Aufnahmestation (25) hindurch erstreckt;

ein erstes Mittel zum hin- und hergehenden Bewegen des beweglichen Abschnittes entlang



- der Wegstrecke zwischen einer Vorwärtsposition und einer Rückwärtsposition hinter der Aufnahmestation (25), wobei der bewegliche Abschnitt beim Vorwärtsbewegen in einem Arbeitshub ein Teil (30) mit kreisförmiger Seite, das in der Aufnahmestation (25) angeordnet ist, erfaßt, und
- ein Förderelement zum Trennen dem am weitesten stromabwärts befindlichen büchsenförmigen Teiles im Stapel und zum Übertragen der Teile (30) die kreisförmige Seitenwände aufweisen, Von dem genannten Ausgang zur Aufnahmestation (25),
- wobei die Fördervorrichtung umfaßt:
- ein drehbar gelagertes Element (40) in einer Tasche (40b) zur Aufnahme eines kreisförmigen Teiles (30) und mit einer bogenförmigen Einführung (40a), die sich von der Tasche in Drehrichtung des Elementes aus erstreckt und langsam bei Bewegung nach dem Auslaß zurücktritt und ein mit einer kreisförmigen Seite versehenes Teil (30) dann trägt, wenn sich dies in eine Aufnahmestation hineinbewegt, und
- Mittel zum Koordinieren der Umläufe um eine Achse des drehbar gelagerten Elementes (40) mit der Bewegung des beweglichen Abschnittes (24), wobei ein mit einer kreisförmigen Seite versehenes Teil in der Aufnahmestation für jeden Arbeitshub des beweglichen Abschnittes angeordnet ist,
- dadurch gekennzeichnet, daß das drehbar gelagerte Element kontinuierlich umläuft.
2. Vorrichtung nach Anspruch 1, weiterhin umfassend eine Führung (42) sowie ein Indexiermittel (48) zum Führen eines mit einer kreisförmigen Seite versehenen Teiles in einer Aufnahmestation (25) hinein zum Erfassen durch den beweglichen Abschnitt (24).
3. Vorrichtung nach Anspruch 1 oder 2, dadurch gekennzeichnet, daß das drehbar gelagerte Element einen Halter aufweist, um ein mit einer kreisförmigen Seite versehenes Teil (30) so lange in der Aufnahmestation (25) in Position zu halten, bis es vom beweglichen Abschnitt erfaßt wird,
- und daß der Halter entlang einer Kante des beweglich gelagerten Teiles angeordnet ist und sich von der Tasche in einer Richtung entgegen der Drehrichtung des Elementes erstreckt, und daß die genannte Kante jenes Teil des beweglich gelagerten Elementes ist, das von der Drehachse am weitesten entfernt ist.
4. Vorrichtung nach Anspruch 2 oder 3, dadurch gekennzeichnet, daß der Auslaß oberhalb des

Indexiermittels (48) angeordnet ist.

5. Vorrichtung nach einem der vorausgegangenen Ansprüche, dadurch gekennzeichnet, daß auch ein Abstreifer (45) vorgesehen ist, um mit kreisförmigen Seiten versehene Teile von der Tasche (40b) zu entfernen, bevor diese Teile das Indexiermittel (48) erreichen.
6. Vorrichtung nach einem der Ansprüche 2 bis 5, dadurch gekennzeichnet, daß die Fördervorrichtung mit kreisförmigen Seiten versehene Teile (30) vorschiebt, während diese sich entlang der Führung (42) bewegen, und selbst dann noch, wenn sie aus der Tasche (40b) vom Abstreifer (45) entnommen wurden.
7. Vorrichtung nach einem der vorausgegangenen Ansprüche, dadurch gekennzeichnet, daß das drehbar gelagerte Element (40) einen einzigen Nocken aufweist, daß die Tasche (40b) eine einzige Tasche ist, und daß sich das drehbar gelagerte Element (40) einen einzigen Umlauf jedesmal dann ausführt, wenn sich der bewegliche Abschnitt (24) durch seinen Arbeitshub sowie zurück in die Rückkehrposition bewegt.
8. Vorrichtung nach einem der Vorausgegangenen Ansprüche, dadurch gekennzeichnet, daß mit kreisförmigen Seiten versehene Teile (30) Büchsen sind, welche rückwärtige Eingangs-Stirnflächen haben, daß der bewegliche Abschnitt (24) eine Ramme umfaßt, die in die Büchsen während des Vorwärtshubes des beweglichen Abschnittes eintritt, daß das Werkzeug weiterhin einen stationären Abschnitt aufweist, der entlang einer Wegstrecke vor der Aufnahmestation angeordnet ist, um mit der Ramme bezüglich des Einwirkens auf die Büchsen zusammen zu arbeiten, und daß der stationäre Abschnitt Gesenke (33) aufweist, die die Wegstrecke umhüllen und den Außenumfang einer jeden Büchse entlang deren Seitenwand dann erfassen, wenn die Büchse durch die Ramme nach vorn bewegt wird.
9. Vorrichtung nach einem der Vorausgegangenen Ansprüche, dadurch gekennzeichnet, daß das drehbar gelagerte Element (40) kontinuierlich angetrieben ist durch ein zweites Mittel, das eine Kupplung (203) enthält, um die Triebverbindung zwischen dem zweiten Mittel und dem Element (40) automatisch dann zu unterbrechen, wenn die zum Antrieb des Elementes (40) erforderliche Leistung einen vorgegebenen Wert überschreitet, und



ein drittes Mittel zum automatischen Betätigen der Kupplung (203) zum Unterbrechen der Triebverbindung zwischen dem zweiten Mittel und dem Element (40), sobald erfaßt wird, daß ein Hub des beweglichen Elementes (24) zur vorderen Position nicht ausgeführt wurde, um ein mit einer kreisförmigen Seite versehenes Teil (30) in ein sauber ausgeformtes Produkt umzuwandeln.

10. Vorrichtung nach den Ansprüchen 8 oder 9, dadurch gekennzeichnet, daß die Gesenke (33) eine Mehrzahl von Gesenkabschnitten aufweisen, die in Reihe angeordnet sind, und daß jeder Gesenkabschnitt (33a, 33b, 33c, 33d) eine Öffnung aufweist, die die Wegstrecke umhüllt, und daß die Öffnungen im Durchmesser abgestuft sind und jede Öffnung hinter den Öffnungen kleinerer Durchmesser sowie vor den Öffnungen größerer Durchmesser liegt.

Revendications

1. Machine pour le traitement de pièces à paroi circulaire comprenant :

un moyen d'introduction dans laquelle des pièces (30) analogues à des boîtes à paroi circulaire sont disposées côte à côte dans une pile, le moyen d'introduction comprenant une sortie par laquelle les pièces à paroi circulaire sortent une à une pour être transférées vers un poste récepteur (25),

un moyen d'outil comprenant une section mobile (24) montée en déplacement le long d'un trajet qui traverse le poste récepteur (25),

un premier moyen pour soumettre la section mobile à un mouvement alternatif le long du trajet entre une position avant et une position de retour derrière le poste récepteur (25), ladite section mobile, quand elle se déplace en avant pendant sa course de travail, venant en prise avec une pièce (30) à paroi circulaire placée au poste récepteur (25), et

un moyen d'alimentation pour séparer la pièce en forme de boîte la plus en aval dans la pile des autres pièces en forme de boîtes restant dans la pile et pour transférer les pièces (30) à paroi circulaire de ladite sortie vers le poste récepteur (25)

dans laquelle les moyens d'alimentation comprennent

un élément (40) monté en rotation comportant un moyen de poche (40b) pour recevoir une pièce (30) à paroi latérale circulaire et une formation d'entrée cintrée (40a) qui part du moyen de poche dans la direction de rotation de l'élément et recule progressivement en passant devant la sortie et supporte une pièce (30)

à paroi circulaire au cours de son mouvement dans le poste récepteur, et

un moyen pour coordonner les rotations autour d'un axe de l'élément (40) monté en rotation avec le mouvement de la section mobile (24) de façon telle qu'une pièce à paroi circulaire est disposée au poste récepteur à chaque course de travail de la section mobile, caractérisé en ce que l'élément monté en rotation est en rotation permanente.

2. Machine selon la revendication 1, comprenant également un guide (42) et un moyen d'avance (48) pour guider une pièce à paroi circulaire dans le poste récepteur (25) de manière que la section mobile (24) puisse venir en prise avec elle.

3. Machine selon la revendication 1 ou 2, dans laquelle l'élément monté en rotation comprend également une formation de maintien qui maintient une pièce (30) à paroi circulaire en position au poste récepteur (25) jusqu'à ce que la section mobile vienne en prise avec elle, et

la formation de maintien est formée le long d'une arête de l'élément monté en rotation et s'étend depuis le moyen de poche dans une direction opposée à la direction de rotation de l'élément, ladite arête faisant partie d'un élément monté en rotation qui est le plus éloigné de son axe de rotation.

4. Machine selon la revendication 2 ou 3, dans laquelle la sortie est placée au-dessus du moyen d'avance (48).

5. Machine selon l'une quelconque des revendications précédentes comprenant également un moyen de démoulage (45) pour sortir des pièces à paroi circulaire du moyen de poche (40b) avant que ces pièces atteignent ledit moyen d'avance (48).

6. Machine selon l'une quelconque des revendications 2 à 5, dans laquelle le moyen d'alimentation entraîne des pièces (30) à paroi circulaire pendant qu'elles se déplacent le long du moyen de guidage (42) et même après qu'elles soient sorties du moyen de poche (40b) sous l'action du moyen de démoulage (45).

7. Machine selon l'une quelconque des revendications précédentes, dans laquelle l'élément monté en rotation (40) comporte un seul lobe et le moyen de poche (40b) comporte une seule poche

et l'élément monté en rotation (40) fait un



seul tour chaque fois que la section mobile (24) décrit sa course de travail et revient à sa position de retour.

8. Machine selon l'une quelconque des revendications précédentes, dans laquelle les pièces (30) à paroi circulaire sont des boîtes ayant des entrées orientées vers l'arrière et dans laquelle la section mobile (24) est constituée d'un mouton qui pénètre dans les boîtes pendant la course en avant de la section mobile ;
 le moyen d'outil comprend également une section fixe placée le long de la trajectoire avant du poste récepteur pour coopérer avec le mouton en agissant sur les boîtes ; et
 la section fixe comprend un moyen de matrice (33) qui entoure la trajectoire et vient en prise avec l'extérieur de chaque boîte le long de sa paroi latérale quand la boîte est entraînée en avant par le mouton.
9. Machine selon l'une quelconque des revendications précédentes, dans laquelle l'élément monté en rotation (40) est entraîné en rotation continue par
 un second moyen qui comprend un moyen d'embrayage (203) pour interrompre la prise d'entraînement entre le second moyen et l'élément (40) automatiquement quand l'énergie nécessaire pour entraîner l'élément (40) dépasse un niveau déterminé d'avance, et
 un troisième moyen pour actionner automatiquement le moyen d'embrayage (203) afin d'interrompre la prise d'entraînement entre le second moyen et l'élément (40) lorsque l'on détecte que la course de la section mobile (24) vers la position avant n'a pas réussi à transformer une pièce (30) à paroi circulaire en un produit convenablement mis en forme.
10. Machine selon la revendication 8 ou 9, dans laquelle le moyen de matrice (33) comprend une pluralité de sections de matrice disposées l'une en face de l'autre
 et dans laquelle chacune des sections de matrice (33a, 33b, 33c, 33d) présente une ouverture qui entoure la trajectoire, les ouvertures ayant des diamètres progressifs et chacune des ouvertures étant disposée derrière les ouvertures de diamètre plus petit et devant les ouvertures de plus grand diamètre.