

SCROLL-TYPE FLUID HANDLING MACHINE

The present invention relates to a scroll-type fluid machine for use in air conditioners, refrigerators or the like.

The scroll-type fluid handling machine of prior art generally has a hermetically sealed casing enclosing therein an electromotor consisting of a rotor integrally mounted on a drive shaft and a stator provided around this rotor, a crankshaft eccentrically connected to the drive shaft of said electromotor, a stationary scroll member fixed to the casing, and a movable scroll member positioned to be engaged with the stationary scroll member and connected to said crankshaft.

Eccentric movement of the crankshaft occurs as the drive shaft of the electromotor rotates and causes the movable scroll member to be rotated in engagement with the stationary scroll member so that a stream of coolant is compressed successively in a suction pressure chamber, an intermediate pressure chamber and a discharge pressure chamber defined between the both scroll members.

With such scroll-type fluid handling machine of well known art, however, a rotational output of the electromotor is transmitted by the crankshaft to the movable scroll member to rotate the latter, so not only its construction is necessarily complicated but also excessive bending moment and other factors exerted on the crankshaft cause a mechanical loss, thus limiting improvement of overall mechanical efficiency.

Furthermore, the conventional construction that the electromotor comprises the rotor and the stator mounted around the rotor, on one hand, and the movable scroll

member is rotationally driven from the drive shaft via the crankshaft, on the other hand, inevitably increases the axial dimension of the machine and results in a bulky machine.

Accordingly, it is a primary object of the invention to provide an improved scroll-type fluid handling machine allowing both the drive shaft of the electromotor and the crankshaft required by the prior art to be eliminated so as to reduce a mechanical loss, thereby to improve a mechanical efficiency, and allowing the axial dimension of the machine itself to be sufficiently reduced to achieve a compact and light weight machine of fewer parts and thereby to reduce a manufacturing cost.

According to one aspect of the invention, there is provided a scroll-type fluid machine comprising:

- a casing;
- a stationary scroll member in the casing;
- a movable scroll member in the casing and interengaging with the stationary scroll member; and
- a motor in the casing for rotating the movable scroll member, the motor comprising a rotor portion on the movable scroll member and a stator portion which is stationary in the casing.

According to another aspect of the invention, there is provided, a scroll-type fluid handling machine having a hermetically sealed casing enclosing therein a stationary scroll member fixed to the casing, a movable scroll member adapted to be rotatable in engagement with the stationary scroll member, and an electromotor to drive the movable scroll member, wherein said electromotor comprises a motor consisting of movable magnets and stationary coils;

wherein said movable magnets face the associated stationary coils in the axial direction of the machine itself; and wherein said movable magnets are mounted on said movable scroll member and said stationary coils are fixed to the casing.

These and the other features, objects and advantages of the invention will be apparent from the following description of preferred embodiments made in reference with the accompanying drawings.

BRIEF DESCRIPTION OF THE DRAWINGS

Figs. 1 through 6 illustrate an embodiment of the invention, in which

Fig 1 is a vertical sectional view of the scroll-type fluid handling machine;

Fig 2 is a sectional view taken along a line II-II in Fig 1, showing an arrangement of coils;

Fig 3 is a sectional view taken along a line III-III in Fig 1, showing an arrangement of magnets;

Fig 4 is a sectional view taken along a line IV-IV in Fig 6, showing an electromotor;

Fig 5 is a perspective view of the magnets; and

Fig 6 is a sectional view as viewed along a line VI-VI in Fig 4, showing the coil;

Figs 7 through 10 illustrate another embodiment of the invention, in which

Fig 7 is a vertical sectional view of the scroll-type fluid handling machine;

Fig 8 is a sectional view taken along a line VIII-VIII in Fig. 7;

Fig 9 is a sectional view taken along a line IX-IX in Fig 7; and

Fig 10 is a sectional view taken along a line X-X in Fig 7, showing a mechanism for prevention of uncontrolled rotation; and

Fig 11 is a fragmentary sectional view of the scroll-type fluid handling machine as a further another embodiment of the invention.

DETAILED DESCRIPTION OF PREFERRED EMBODIMENTS:

Referring to Fig 1, reference numeral 1 designates a scroll-type fluid handling machine, reference numeral 2a designates an upper casing, and reference numeral 2b designates a lower casing, these casings 2a, 2b being joined to each other as by means of welding so as to provide a hermetically sealed space therebetween.

A disc-like stationary member 3 is secured to the inside of the lower casing 2b as by means of welding and an oil reservoir 4 extends over a bottom of the lower casing 2b. A stationary scroll member 5 configured to be suited to the upper and lower casings 2a, 2b is secured by means of bolts 6 to the top of the stationary member 3 at a plurality of locations circumferentially spaced from one another adjacent the outer edge thereof. Between the stationary member 3 and the stationary scroll member 5 there is provided a movable scroll member 7 with a movable scroll 7a being engaged with a stationary scroll 5a.

The stationary member 3 holds at its center a stationary shaft 10 by means of a bearing 9. The shaft 10 is formed with a pit 11 being eccentric with respect to the center O_1 of said stationary shaft 10 by a distance 1 so as to rotatably support a rotatable shaft 12 projecting from the

rear side (i.e. lower surface as viewed in Fig 1) of the movable scroll member 7 and being correspondingly eccentric by the same distance 1 by means of a metallic bearing 13 so that the movable scroll member 7 may be rotatably driven with its scroll 7a being engaged with the stationary scroll 5a. The lower casing 2b is provided with a suction pipe 15 extending through the side wall thereof and a stream of coolant is introduced from this suction pipe 15 through a suction portion 16 formed in the stationary scroll member 5 into the machine 1. The stream of coolant is successively compressed in a suction pressure chamber 17, an intermediate pressure chamber 18 and a discharge pressure chamber 19 all being formed between the both scrolls 5a, 7a, and then flows through a discharge port 20 formed in the stationary scroll member, a discharge chamber 21 and is finally discharged out from a discharge pipe 22 extending through the upper casing 2a. Reference numeral 10A designates a balancing weight provided integrally with the stationary shaft 10 to compensate a rotary imbalance possibly occurring as the movable scroll member 7 is rotated. Reference numerals 10a, 25a designate oiling passageways, respectively.

This scroll-type fluid handling machine 1 further includes a mechanism 24 to prevent uncontrolled rotation of the movable scroll member 7. In this specific embodiment, the mechanism 24 comprises three pins 25 which are, as seen in Figs 2 and 3, positioned at regular intervals so that a diameter-enlarged portion 25A of each pin 25 may be rotatably inserted into a correspondingly diameter-enlarged opening 26 formed in the stationary member 3 while a diameter-reduced portion 25b of each pin 25 may be rotatably inserted into a correspondingly diameter-reduced opening 27 formed in the movable

scroll member 7 to prevent the movable scroll member 7 from uncontrollably rotating.

The scroll-type fluid handling machine 1 further includes an electromotor 30 serving for direct rotational drive of the movable scroll member 7. This electromotor 30 comprises six sets of motors each consisting of a coil 31 on the primary, i.e. non-rotating portion and a magnet 32 on the secondary, i.e. rotary portion. The coil 31 and the magnet 32 of each set have substantially the same rectangular shape with a predetermined width as viewed in Figs. 2 and 3, and the coil 31 is provided on the upper side of the stationary member 3 while the magnet 32 is provided on the lower side of the movable scroll member 7 so that the coil 31 and the magnet 32 of each set are opposed to each other in the axial direction of the machine itself.

Referring to Figs. 4 and 5, said magnet 32 comprises a plurality of square pillar-shaped magnet components 33 parallelly spaced from one another by predetermined distances and carried by a support plate 34. These magnet components 33 are so magnetized that their polarities (N and S) alternate along the side facing the coil 31.

Said coil 31 is formed on the side facing the magnet 32 comprising laminated steel plates 35 alternately with slots 35a and projection poles 35b in association with the arrangement of said magnet components 33 and a zigzag 36 is inserted into the slots 35a, as seen in Figs. 4 and 6. Energization, for example, in a direction indicated by an arrow in Fig 6 magnetizes said projecting poles 35b alternately as N and S poles. In consequence, the projecting pole 35b and the adjacent magnet component 33 of said magnet 32 having the same polarity repel each

other while the projecting pole 35b and the adjacent magnet component 33 having the different polarities attract each other, linearly thrusting the magnet 32 in a direction indicated by an arrow A in Fig 4.

As will be apparent from Figs. 2 and 3, these sets of the coils 31 and the magnets 32 are arranged at regular intervals and the magnetic poles of each set are arranged in a tangential direction. It should be understood that each pair of the diagonally opposed coils are driven at a same phase, namely, each pair of the diagonally opposed linear motors corresponds to one phase and respective phases form a star connection. Thus, a drive mode thereof is similar to that for the three-phase rotary electromotor.

Each coil 31 is received in a recess 41 formed in the upper side of the stationary member 3 while each magnet 32 is received in a recess 42 formed in lower side (rear side) of the movable scroll member 7 so that each pair of the mutually associated coil 31 and magnet 32 are opposed to each other.

In such scroll-type fluid handling machine 1, the movable scroll member 7 may be directly driven to be rotated together with the magnets 32 by properly controlling three-phase AC supplied to the respective coils 31.

With this embodiment, the movable scroll member is directly driven, instead of being eccentrically driven from the rotary electromotor via the crankshaft, so that the mechanical loss can be minimized and thereby the mechanical efficiency can be improved. Further, the electromotor which comprises the stator coils and rotor magnets makes the drive shaft as well as the crankshaft

which have conventionally been essential to the machine unnecessary and allows the axial dimension of the machine itself to be substantially reduced. In this manner, it is possible to obtain a compact, light weight machine of fewer parts, to simplify the construction and thereby to reduce a manufacturing cost.

Now another embodiment will be discussed.

With the scroll-type fluid handling machine 44 according to this embodiment, as shown in Figs. 7 through 9, the movable magnets 32 of the respective motors are provided on the lower face of the movable scroll member 7 while the stationary coils 31 are provided on the stationary member 3, and the mechanism 24 to prevent the movable scroll member 7 from uncontrollably rotating is provided in the form of a Scotch yoke.

More specifically, the mechanism 24 is so arranged, as seen in Fig 7, that the stationary member 3 is eccentrically provided with a boss 3a projecting downward and holding therein the stationary shaft 10 by means of a metallic bearing 45. An eccentric opening 11 formed through said stationary shaft 10 receives the rotatable shaft 12 extending therethrough by means of the metallic bearing 13. Said boss-like portion 3a formed therein at its lower end with a first guide groove 3b, as seen in Fig 10, adapted to loosely receive a movable member 46 which is, in turn, adapted to be slidably movable within said guide groove 3b in one direction (indicated by an arrow X) in a plane being parallel to said movable scroll member 7.

The movable member 46 is formed with a second guide groove 47 extending transversely of said first guide groove 3b and an interlocking portion 12a formed on the lower end of said rotatable shaft 12 is loosely received in said guide groove 47 so that said interlocking portion 12A may be slidably movable in the guide groove 47 in a direction (indicated by an arrow Y) transversely of said first guide groove 3b.

Thus, as in the previous embodiment, it is possible to prevent the movable scroll member which eccentrically moves from uncontrollably rotating and thereby to achieve the same effect as achieved by the previous embodiment.

Though the movable magnets 32 of the motors are provided in the lower (i.e., rear) face of the movable scroll member 7 and the stationary coils 31 are provided on the upper face of the stationary member 3 in the above-mentioned both embodiments, it is also possible to provide the movable magnets 32 on the upper face of the movable scroll member 7 while to provide the stationary coils 31 on the lower face of the stationary scroll member 5, as seen in Fig. 11, so as to achieve the same effect as achieved by the previously mentioned embodiments so far as said magnets and said coils are positioned so as to face one another in the axial direction of the machine itself. In the last-mentioned embodiment, the coils and the magnets attract one another in the axial direction to enhance sealing of the compressing chambers between the scrolls, to reduce resistance occurring in the thrust direction and thereby to improve durability.

Although the motor has been described as a three-phase linear motor, the number of phases is not critical and the number of motors for each phase may comprise three or more sets.



CLAIMS

1. A scroll-type fluid handling machine having a hermetically sealed casing enclosing therein a stationary scroll member fixed to the casing, a movable scroll member adapted to be rotatable in engagement with the stationary scroll member and an electromotor used to drive the movable scroll member, said scroll-type fluid handling machine being characterized in that said electromotor comprises a motor consisting of movable magnets and stationary coils; that said movable magnets face the stationary coils in the axial direction of the machine itself; and that said movable magnets are mounted on said movable scroll member and said stationary coils are fixed to the casing.
2. A scroll-type fluid machine comprising:
 - a casing;
 - a stationary scroll member in the casing;
 - a movable scroll member in the casing and interengaging with the stationary scroll member; and
 - a motor in the casing for rotating the movable scroll member, the motor comprising a rotor portion on the movable scroll member and a stator portion which is stationary in the casing.
3. A machine according to claim 2 wherein said motor comprises magnets mounted on said movable scroll member and coils fixed relative to said stationary scroll member.



4. A machine according to any preceding claim further comprising means for preventing uncontrolled movement of said movable scroll member.

5. A machine according to claim 4 wherein said preventing means comprises a pin on one scroll member engaging in an aperture on the other scroll member.

6. A machine according to claim 4 wherein said preventing means comprises a scotch yoke.

7. A scroll-type machine substantially as hereinbefore described with reference to Figs. 1 to 6 or to Figs. 7 to 10 or to Fig. 11 of the accompanying drawings.

8. An air conditioner or refrigerator comprising a compressor constructed in accordance with any preceding claim.
