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BUCKET ELEVATOR CONVEYORS

The present invention relates to improvements in bucket elevator conveyor systems used for the gentle transport of loose materials and is particularly concerned with the systems that allow for continuous feeding of the loose material into the carrying bucket without spillage and without damage thereto.

Bucket elevator conveyor systems are known in which buckets, in spaced apart relationship, are rotatably supported between two chains that enables them to negotiate corners on a path in one plane, the buckets being arranged to be inverted at a desired discharge station. In order to achieve continuous filling of the buckets, necessary for efficient operation of the system, the chains, in one type of system, have to be collapsed in order to bring the buckets close together and minimise spillage, adequate chain control to produce this effect is difficult to achieve.

Elevators of this type are used for handling a very wide range of materials that are required to be handled gently during production and/or packaging to avoid damage. Such materials include, for example, peanuts, confectionery, milk powder, tile dust, carbon black, precious metal powders, cereals, pasta, chocolate, semi-liquids and delicate or volatile materials.

We have now devised a bucket elevator conveyor system which overcomes the problems associated with known systems such as those described above. In our system there is provided a body housing a track to route at least one chain through an endless circuit, a

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means for driving the chain and buckets pivotally mounted thereon for movement therewith, the said chain comprising a multiplicity of links connected together by pins pivoting in elongated holes such that the length of a section of the chain increases when in tension compared with its length when in compression, means being provided to cause a said section of the chain to be in compression at one part of the circuit where materials can be continuously loaded (a loading station). The spacing of the buckets is such that when the chain is in this section it is in compression and the buckets are in contact one with another. When the buckets traverse the upper sections of the circuit the chain is in tension and the buckets can be rotated through 360° about the pivotal mountings to discharge their contents at any given part of the circuit, e.g. a discharge station, without colliding with each other.

Preferably the buckets are pivotally suspended between two chains.

Preferably also means are provided for ensuring that when a said section of the chain is in compression it adopts a relatively stiff state. Suitable means for achieving this effect may comprise guide elements surrounding the track on three sides and/or, preferably, adjacent links being provided with interlocking portions that engage when the chain is in compression to reduce the tendency of the chain to buckle and causing it to become rigid in the plane of movement.

The driving means may comprise one or more sprockets driven by an electric motor; the means for bringing a section of the chain into compression at a loading station may comprise a pair of sprockets

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rotatably mounted at either end of the said loading station and spaced at a distance apart equivalent to the compressed length of the chain in that section. The chain drive should be on the entry side of the said section and may comprise the appropriate one of said sprockets.

One embodiment of the invention will now be described with reference to the accompanying drawings in which:-

Figure 1 is a diagrammatic side view of a bucket elevator conveyor system according to the present invention;

Figure 2 is a diagrammatic view of the loading station of the circuit of Figure 1; and

Figure 3 is an exploded diagrammatic illustration of a chain and bucket assembly for use with the system of Figures 1 and 2.

Referring to the drawings, a bucket elevator system of the present invention comprises an outer framework generally indicated at 1 having a horizontal first section 1a at one level, a vertical section 1b and a second horizontal section 1c at the second higher level. The said framework houses a circuit of parallel flexible chains 2 consisting of chain links 12 having slotted pivot locations 25 and joined by pivot pins 11 (see Figure 3). The chains move through a loading station 14, in frame section 1a, along a U-shaped guide 35 and in a substantially horizontal path, a vertical section 1b and a discharge station 20 in frame section 1c, in which the chains are again substantially horizontal and at a higher level.



The chains are driven by a pair of sprockets 6, located immediately before the loading station 14; and an electric motor 15 coupled to sprockets 6 by chain 16. The chains are maintained in engagement with sprockets 6 by an encapsulating shroud 13. From sprocket 6 the chains pass over a first pair of control sprockets 8 positioned after the loading station 14 and are maintained in engagement therewith by an encapsulating shroud 22. The chains then move upwardly over guide sprockets 7 and thence horizontally to pass round a terminal sprocket 18 to move horizontally in the opposite direction to a discharge station 20. The chains turn round a second pair of guide sprockets 27 downwardly through chain guide 4 to a second pair of control sprockets 37 and thence horizontally back to drive sprockets 6.

Terminal sprocket 18 is carried for free rotation about shaft 18a journaled into bearings with sliding mounts and spring loaded at 19 to apply a small tension to the extended chain circuit.

Sprockets 6, 7, 8, 27 and 37 are rotatably carried on common shafts 6a, 7a, 8a, 27a and 37a respectively, each shaft being journaled into bearings secured in frame 1. The sprockets have the same number and pitch of teeth in order that each forwards the chain at the same speed.

Suspended between the chains are buckets 3 carried on inwardly extending end portion 11a of pivot pins 11 that engage in hollow spigots 30 extending outwardly from each side of buckets 3. Gears 21 are non-rotatably secured to spigots 30 on one side of each bucket. Alternatively the male pins can be on the

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chains and the hollow spigots formed in the bucket trunnions.

The parallel chains 2 are maintained at a constant distance apart by use of tie bars 10 which locate in hollow ends 11c of pivot pins 11 and are positioned between any or all pairs of buckets.

To avoid spillage at the feeding station buckets 3 are provided with overlap member lips 23 located along one side thereof and extending outwardly to extend over an adjacent bucket side edge to provide cover for the space between buckets when they are juxtaposed at the loading station 14.

The chain links 12 are formed with V-shaped portions 40 at one end that interlock with complimentary shaped portions 41, formed in the opposite end of an adjacent link, when the chain is in compression causing the links to lock into a rigid section. When the chain is under tension portions 40 and 41 separate to allow articulation.

Guide sprockets 8 and 37 are connected by chain 31 passing over the snubs 32 and 33 located either side of drive elements 24 on sprockets 8 and corresponding elements 38 on sprockets 37 whereby sprockets 8 and 37 are caused to rotate at the same speed and maintain correct rotation and correct closed or compressed chain pitch control throughout this section.

In setting up the conveyor system the relative positions of drive sprockets 6 and control sprockets 8 are arranged such that the chain is in a closed rigid and compressed state at the feed station 14 causing the buckets 3 to lie in near touching relationship with the

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overlap portions 23 of one bucket extending over the opposed edge of an adjacent bucket. To further control the positioning of the buckets 3 in a horizontal position at the feed station 14, and to provide stability to the buckets, a spring-loaded hollow frame 24 contacts the side top edges of the buckets as they pass through the feed station.

In use, with the chains 2 at the feed station 14 in a compressed and rigid condition, buckets 3 are continuously filled with the material to be transferred and drive sprocket 6 propels the chain around control sprocket 8 into the vertical section 1b of frame 1. Terminal sprocket 18 is carried on bearings with sliding mounts and spring-loaded to apply a tension to the chain circuit other than the circuit chain positioned between central sprockets 8 and 37 as it passes round sprocket 6. The chain links slide into the extended state through the upper section of the circuit allowing the buckets 3 to swing freely about the support pivots 11b. Buckets 3 pass around the upper sprockets 7 and 37 and the terminal sprocket 18 to the discharge point where bucket gears 21 engage a toothed rack (not shown) which causes the buckets to rotate through 360° during which rotation material contained in the buckets is discharged. Thereafter the buckets continue around the circuit back to the feed station.



CLAIMS

1. A bucket elevator conveyor system comprising a track to route at least one chain (2) through an endless circuit including a loading station (14) and discharge station (20); buckets (3) pivotally mounted (11a, 30) on the said chain (2) for movement therewith and to receive material at the loading station (14) and discharge material at the discharge station (20) and means (6, 15, 16) to drive the said chain (2), characterised in that:

(i) the said chain (2) comprises a multiplicity of links (12) connected together by pins (11) pivoting in elongated holes (25) formed in the said links such that the length of a section of the chain (2) is greater when in tension compared with its length when in compression;

(ii) means (6, 8) are provided to cause a said section of the chain to be in compression at the said loading station (14);

(iii) the spacing of the said buckets (3) is such that when the chain (2) is in compression adjacent edge portions of the buckets (3) are in contact and when the chain (2) is in tension the buckets (3) can be rotated through 360° about the pivotal mountings (11a, 30).

2. A bucket elevator conveyor according to claim 1 wherein the buckets (3) are pivotally suspended between two chains (2).

3. A bucket elevator conveyor according to claim 1 or claim 2 including means (35, 40, 41) to cause the chain to adopt a stiff state when in compression.





4. A bucket elevator conveyor according to claim 3 wherein said means comprises guide elements (35) partially encapsulating the said chain (2).

5. A bucket elevator conveyor according to claim 3 wherein said means comprises interlocking portions (40, 41) provided on adjacent links (12) for engagement when the chain (2) is in compression, which portions reduce the tendency of the chains to buckle in the plane of movement.

6. A bucket elevator conveyor according to any one of the preceding claims wherein the means for driving the chain comprises a sprocket (6) member powered by a motor (15).

7. A bucket elevator conveyor according to claim 6 wherein the means for maintaining a section of the chain in compression comprises a pair of sprockets (6, 8) rotatably mounted at the entry and exit ends of said section (14) and spaced apart at a distance equivalent to the compressed length of the chain (2) in that section; the chain drive means (6) being positioned on the entry side of said section.

8. A bucket elevator conveyor according to claim 7 wherein the chain drive comprises the sprocket (6) mounted at the entry end of said section (14).

9. A bucket elevator conveyor according to claim 7 or claim 8 wherein a third control sprocket (37) is rotatably mounted in the section of the chain circuit approaching the said drive means (6) and is arranged to rotate at the same speed as the sprocket (8) on the exit side of the compressed chain section, all said sprockets (6, 8, 37) having the same number and pitch





of teeth whereby the chain is forwarded at the same speed by each said sprocket.

10. A bucket elevator conveyor wherein means (24) are provided to stabilise the buckets in a substantially horizontal position when passing through the compressed section of the chain.

11. A bucket elevator conveyor according to claim 10 wherein said stabilising means comprises a spring-loaded hollow frame member (24) dimensioned to contact and bear down on the top edge of the buckets during their passage through the said section.

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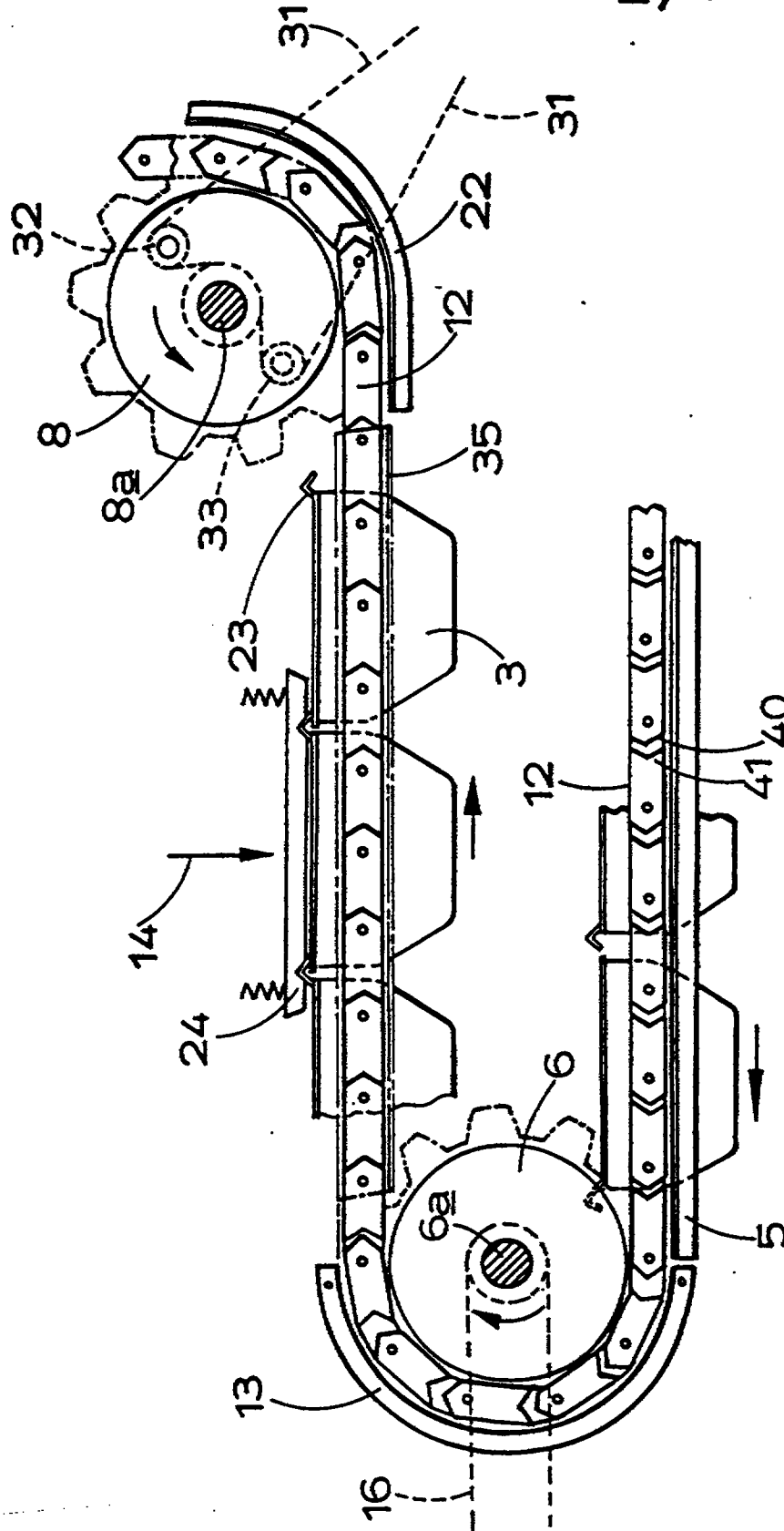


FIG. 2

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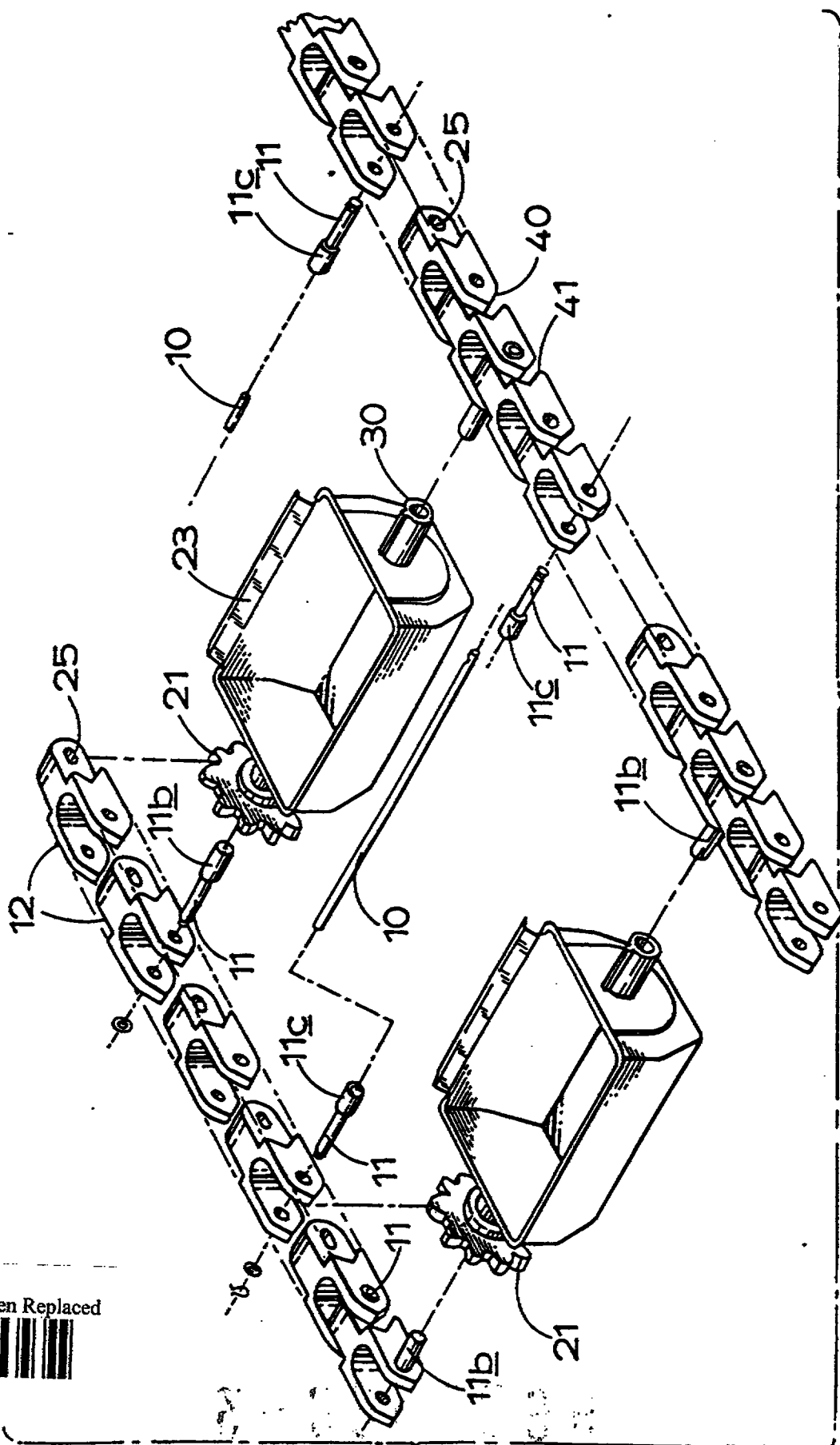


FIG. 3.

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REPLACEMENT SHEETS		SG
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BUCKET ELEVATOR CONVEYORS DATE OF RECEIPT 13/3/97

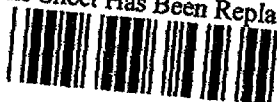
The present invention relates to improvements in bucket elevator conveyor systems used for the gentle transport of loose materials and is particularly concerned with the systems that allow for continuous feeding of the loose material into the carrying bucket without spillage and without damage thereto.

Bucket elevator conveyor systems are known in which buckets, in spaced apart relationship, are rotatably supported between two chains that enables them to negotiate corners on a path in one plane, the buckets being arranged to be inverted at a desired discharge station. In order to achieve continuous filling of the buckets, necessary for efficient operation of the system, the chains, in one type of system, have to be collapsed in order to bring the buckets close together and minimise spillage, adequate chain control to produce this effect is difficult to achieve.

Elevators of this type are used for handling a very wide range of materials that are required to be handled gently during production and/or packaging to avoid damage. Such materials include, for example, peanuts, confectionery, milk powder, tile dust, carbon black, precious metal powders, cereals, pasta, chocolate, semi-liquids and delicate or volatile materials.

We have now devised a bucket elevator conveyor system which overcomes the problems associated with known systems such as those described above. In our system there is provided a body housing a track to route at least one chain through an endless circuit, a

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Date: 13 MAR 1997



means for driving the chain and buckets pivotally mounted thereon for movement therewith, the said chain comprising a multiplicity of links connected together by pins pivoting in elongated holes such that the length of a section of the chain increases when in tension compared with its length when in compression, means being provided to cause a said section of the chain to be in compression at one part of the circuit where materials can be continuously loaded (a loading station). The spacing of the buckets is such that when the chain is in this section it is in compression and the buckets are in contact one with another. When the buckets traverse the upper sections of the circuit the chain is in tension and the buckets can be rotated through 360° about the pivotal mountings to discharge their contents at any given part of the circuit, e.g. a discharge station, without colliding with each other.

Preferably the buckets are pivotally suspended between two chains.

Preferably also means are provided for ensuring that when a said section of the chain is in compression it adopts a relatively stiff state. Suitable means for achieving this effect may comprise guide elements surrounding the track on three sides and/or, preferably, adjacent links being provided with interlocking portions that engage when the chain is in compression to reduce the tendency of the chain to buckle and causing it to become rigid in the plane of movement.

The driving means may comprise one or more sprockets driven by an electric motor; the means for bringing a section of the chain into compression at a loading station may comprise a pair of sprockets

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Date: 13 MAR 1997



rotatably mounted at either end of the said loading station and spaced at a distance apart equivalent to the compressed length of the chain in that section. The chain drive should be on the entry side of the said section and may comprise the appropriate one of said sprockets.

One embodiment of the invention will now be described with reference to the accompanying drawings in which:-

Figure 1 is a diagrammatic side view of a bucket elevator conveyor system according to the present invention;

Figure 2 is a diagrammatic view of the loading station of the circuit of Figure 1; and

Figure 3 is an exploded diagrammatic illustration of a chain and bucket assembly for use with the system of Figures 1 and 2.

Referring to the drawings, a bucket elevator system of the present invention comprises an outer framework generally indicated at 1 having a horizontal first section 1a at one level, a vertical section 1b and a second horizontal section 1c at the second higher level. The said framework houses a circuit of parallel flexible chains 2 consisting of chain links 12 having slotted pivot locations 25 and joined by pivot pins 11 (see Figure 3). The chains move through a loading station 14, in frame section 1a, along a U-shaped guide 35 and in a substantially horizontal path, a vertical section 1b and a discharge station 20 in frame section 1c, in which the chains are again substantially horizontal and at a higher level.

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Date: 13 MAR 1997



The chains are driven by a pair of sprockets 6, located immediately before the loading station 14; and an electric motor 15 coupled to sprockets 6 by chain 16. The chains are maintained in engagement with sprockets 6 by an encapsulating shroud 13. From sprocket 6 the chains pass over a first pair of control sprockets 8 positioned after the loading station 14 and are maintained in engagement therewith by an encapsulating shroud 22. The chains then move upwardly over guide sprockets 7 and thence horizontally to pass round a terminal sprocket 18 to move horizontally in the opposite direction to a discharge station 20. The chains turn round a second pair of guide sprockets 27 downwardly through chain guide 4 to a second pair of control sprockets 37 and thence horizontally back to drive sprockets 6.

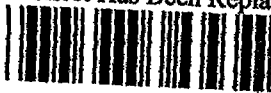
Terminal sprocket 18 is carried for free rotation about shaft 18a journaled into bearings with sliding mounts and spring loaded at 19 to apply a small tension to the extended chain circuit.

Sprockets 6, 7, 8, 27 and 37 are rotatably carried on common shafts 6a, 7a, 8a, 27a and 37a respectively, each shaft being journaled into bearings secured in frame 1. The sprockets have the same number and pitch of teeth in order that each forwards the chain at the same speed.

Suspended between the chains are buckets 3 carried on inwardly extending end portion 11a of pivot pins 11 that engage in hollow spigots 30 extending outwardly from each side of buckets 3. Gears 21 are non-rotatably secured to spigots 30 on one side of each bucket. Alternatively the male pins can be on the

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Date: 13 MAR 1997



chains and the hollow spigots formed in the bucket trunnions;

The parallel chains 2 are maintained at a constant distance apart by use of tie bars 10 which locate in hollow ends 11c of pivot pins 11 and are positioned between any or all pairs of buckets.

To avoid spillage at the feeding station buckets 3 are provided with overlap member lips 23 located along one side thereof and extending outwardly to extend over an adjacent bucket side edge to provide cover for the space between buckets when they are juxtaposed at the loading station 14.

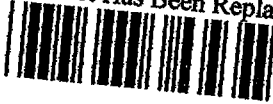
The chain links 12 are formed with V-shaped portions 40 at one end that interlock with complimentary shaped portions 41, formed in the opposite end of an adjacent link, when the chain is in compression causing the links to lock into a rigid section. When the chain is under tension portions 40 and 41 separate to allow articulation.

Guide sprockets 8 and 37 are connected by chain 31 passing over the snubs 32 and 33 located either side of drive elements 24 on sprockets 8 and corresponding elements 38 on sprockets 37 whereby sprockets 8 and 37 are caused to rotate at the same speed and maintain correct rotation and correct closed or compressed chain pitch control throughout this section.

In setting up the conveyor system the relative positions of drive sprockets 6 and control sprockets 8 are arranged such that the chain is in a closed rigid and compressed state at the feed station 14 causing the buckets 3 to lie in near touching relationship with the

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Date: 13 MAR 1997



overlap portions 23 of one bucket extending over the opposed edge of an adjacent bucket. To further control the positioning of the buckets 3 in a horizontal position at the feed station 14, and to provide stability to the buckets, a spring-loaded hollow frame 24 contacts the side top edges of the buckets as they pass through the feed station.

In use, with the chains 2 at the feed station 14 in a compressed and rigid condition, buckets 3 are continuously filled with the material to be transferred and drive sprocket 6 propels the chain around control sprocket 8 into the vertical section 1b of frame 1. Terminal sprocket 18 is carried on bearings with sliding mounts and spring-loaded to apply a tension to the chain circuit other than the circuit chain positioned between central sprockets 8 and 37 as it passes round sprocket 6. The chain links slide into the extended state through the upper section of the circuit allowing the buckets 3 to swing freely about the support pivots 11b. Buckets 3 pass around the upper sprockets 7 and 37 and the terminal sprocket 18 to the discharge point where bucket gears 21 engage a toothed rack (not shown) which causes the buckets to rotate through 360° during which rotation material contained in the buckets is discharged. Thereafter the buckets continue around the circuit back to the feed station.

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Date: 13 MAR 1997



CLAIMS

1. A bucket elevator conveyor system comprising a track to route at least one chain (2) through an endless circuit including a loading station (14) and discharge station (20); buckets (3) pivotally mounted (11a, 30) on the said chain (2) for movement therewith and to receive material at the loading station (14) and discharge material at the discharge station (20) and means (6, 15, 16) to drive the said chain (2), characterised in that:

(i) the said chain (2) comprises a multiplicity of links (12) connected together by pins (11) pivoting in elongated holes (25) formed in the said links such that the length of a section of the chain (2) is greater when in tension compared with its length when in compression;

(ii) means (6, 8) are provided to cause a said section of the chain to be in compression at the said loading station (14);

(iii) the spacing of the said buckets (3) is such that when the chain (2) is in compression adjacent edge portions of the buckets (3) are in contact and when the chain (2) is in tension the buckets (3) can be rotated through 360° about the pivotal mountings (11a, 30).

2. A bucket elevator conveyor according to claim 1 wherein the buckets (3) are pivotally suspended between two chains (2).

3. A bucket elevator conveyor according to claim 1 or claim 2 including means (35, 40, 41) to cause the chain to adopt a stiff state when in compression.



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Date: 13 MAR 1997



4. A bucket elevator conveyor according to claim 3 wherein said means comprises guide elements (35) partially encapsulating the said chain (2).
5. A bucket elevator conveyor according to claim 3 wherein said means comprises interlocking portions (40, 41) provided on adjacent links (12) for engagement when the chain (2) is in compression, which portions reduce the tendency of the chains to buckle in the plane of movement.
6. A bucket elevator conveyor according to any one of the preceding claims wherein the means for driving the chain comprises a sprocket (6) member powered by a motor (15).
7. A bucket elevator conveyor according to claim 6 wherein the means for maintaining a section of the chain in compression comprises a pair of sprockets (6, 8) rotatably mounted at the entry and exit ends of said section (14) and spaced apart at a distance equivalent to the compressed length of the chain (2) in that section; the chain drive means (6) being positioned on the entry side of said section.
8. A bucket elevator conveyor according to claim 7 wherein the chain drive comprises the sprocket (6) mounted at the entry end of said section (14).
9. A bucket elevator conveyor according to claim 7 or claim 8 wherein a third control sprocket (37) is rotatably mounted in the section of the chain circuit approaching the said drive means (6) and is arranged to rotate at the same speed as the sprocket (8) on the exit side of the compressed chain section, all said sprockets (6, 8, 37) having the same number and pitch



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Date: 13 MAR 1997



of teeth whereby the chain is forwarded at the same speed by each said sprocket.

10. A bucket elevator conveyor wherein means (24) are provided to stabilise the buckets in a substantially horizontal position when passing through the compressed section of the chain.

11. A bucket elevator conveyor according to claim 10 wherein said stabilising means comprises a spring-loaded hollow frame member (24) dimensioned to contact and bear down on the top edge of the buckets during their passage through the said section.



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Date: 13 MAR 1997



BUCKET ELEVATOR CONVEYORS

ABSTRACT

A bucket elevator conveyor comprises a frame housing (1) incorporating a track to route a pair of parallel chains (2) carrying buckets (3) pivotally mounted (11a, 30) therebetween through an endless circuit including loading (14) and discharge (20) stations. The chains consist of links (12) connected together by pins (11) pivoting in elongated holes (25) such that the chains are shorter in compression than when under tension, the ends of adjacent links (12) including interengaging portions (40, 41) to cause the chains to adopt a relatively stiff state when in compression. The chains (2) are maintained under compression in a section of track at the loading station (14) by two pairs of sprockets (6, 8) positioned at the entry and exit ends of the section and spaced apart to maintain the chains under compression, the sprockets (6) at the entry end being driven by an electric motor drive (15). The spacing of the buckets (3) is such that when the chains (2) are in compression at the loading station (14) the end edge portions of adjacent buckets (3) are in contact, and when the chain is under tension at the discharge station (20) the buckets (3) can be rotated through 360° without colliding.

FIGURE 1

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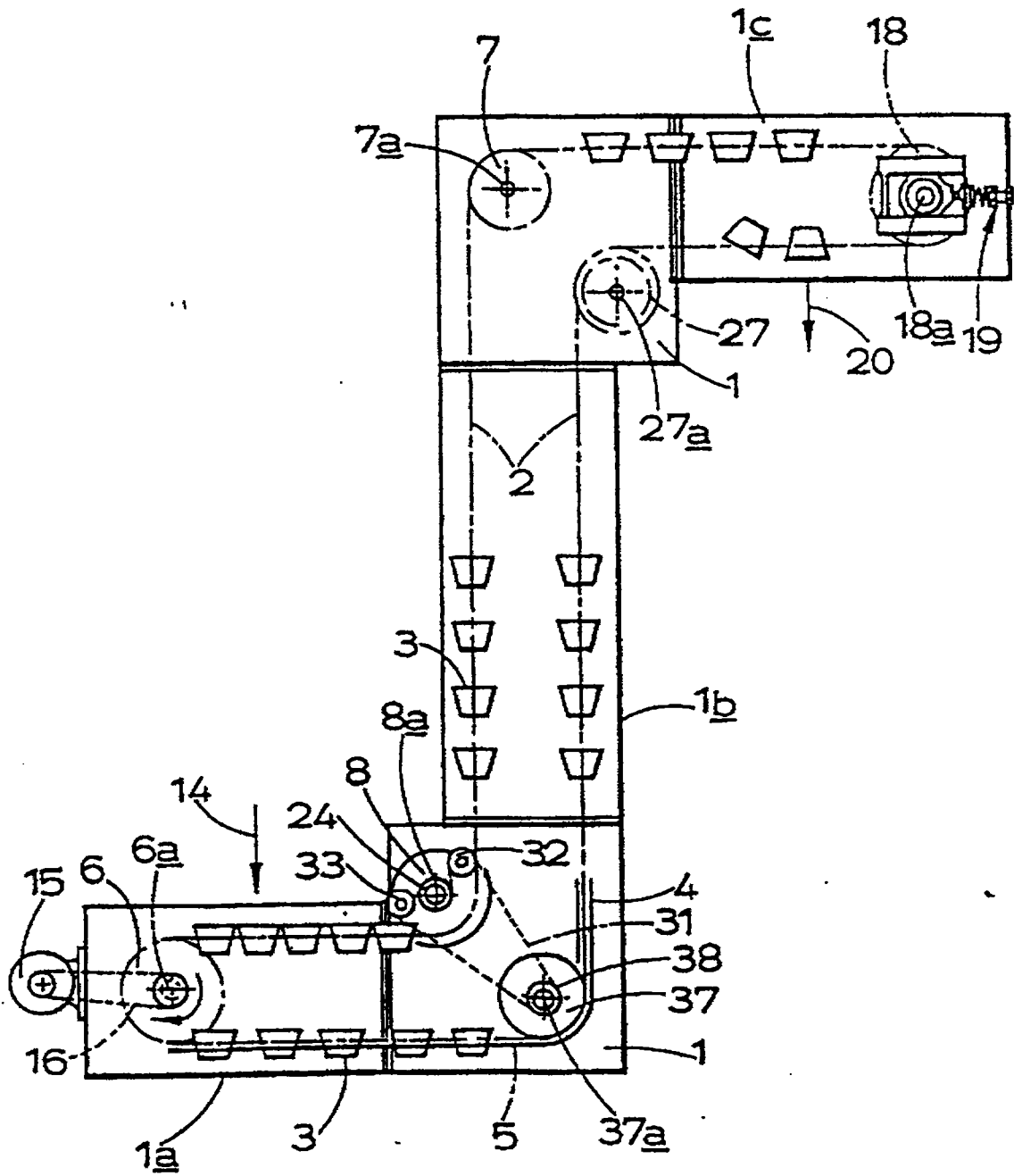
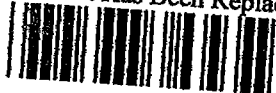


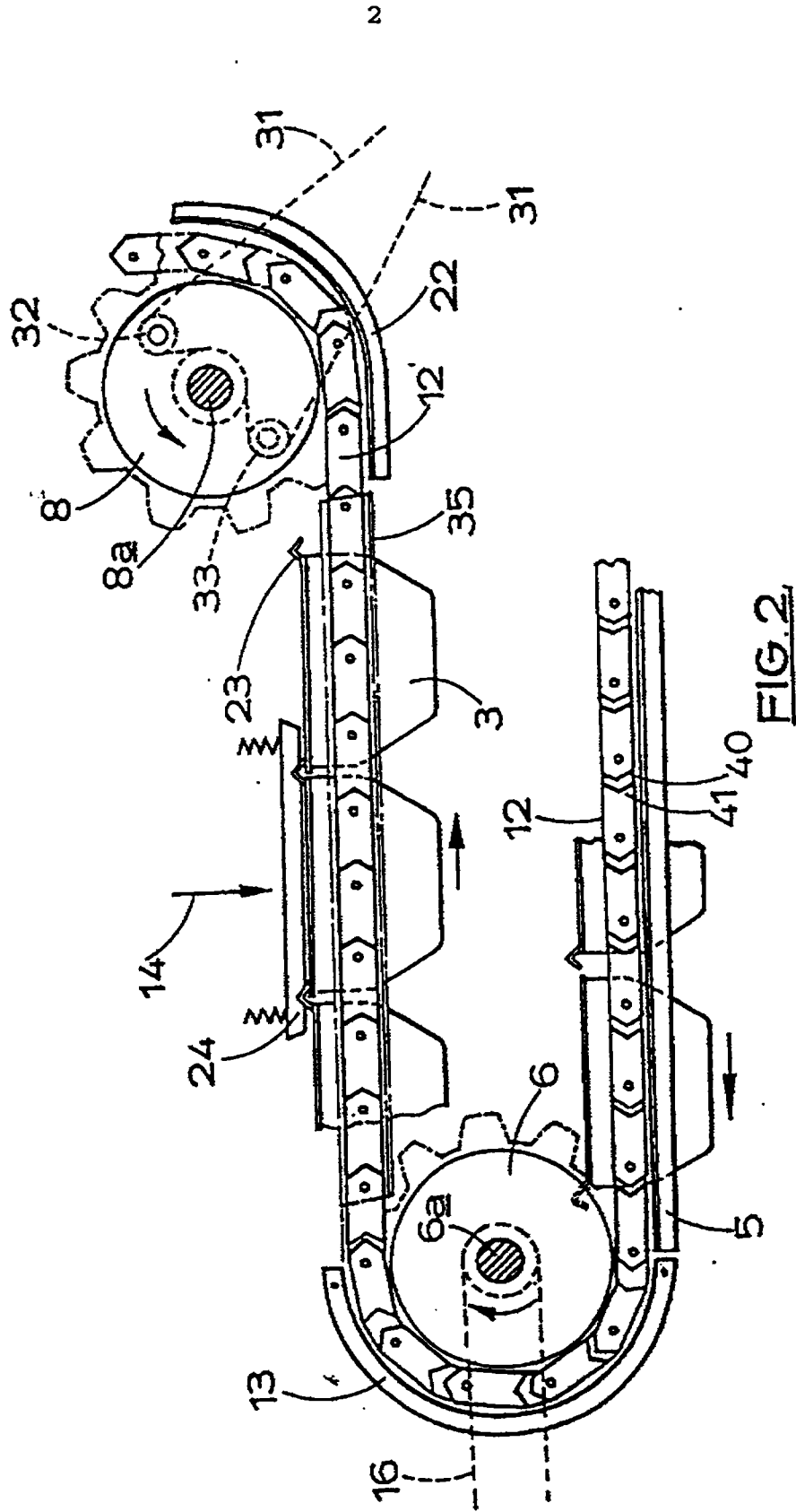
FIG.1

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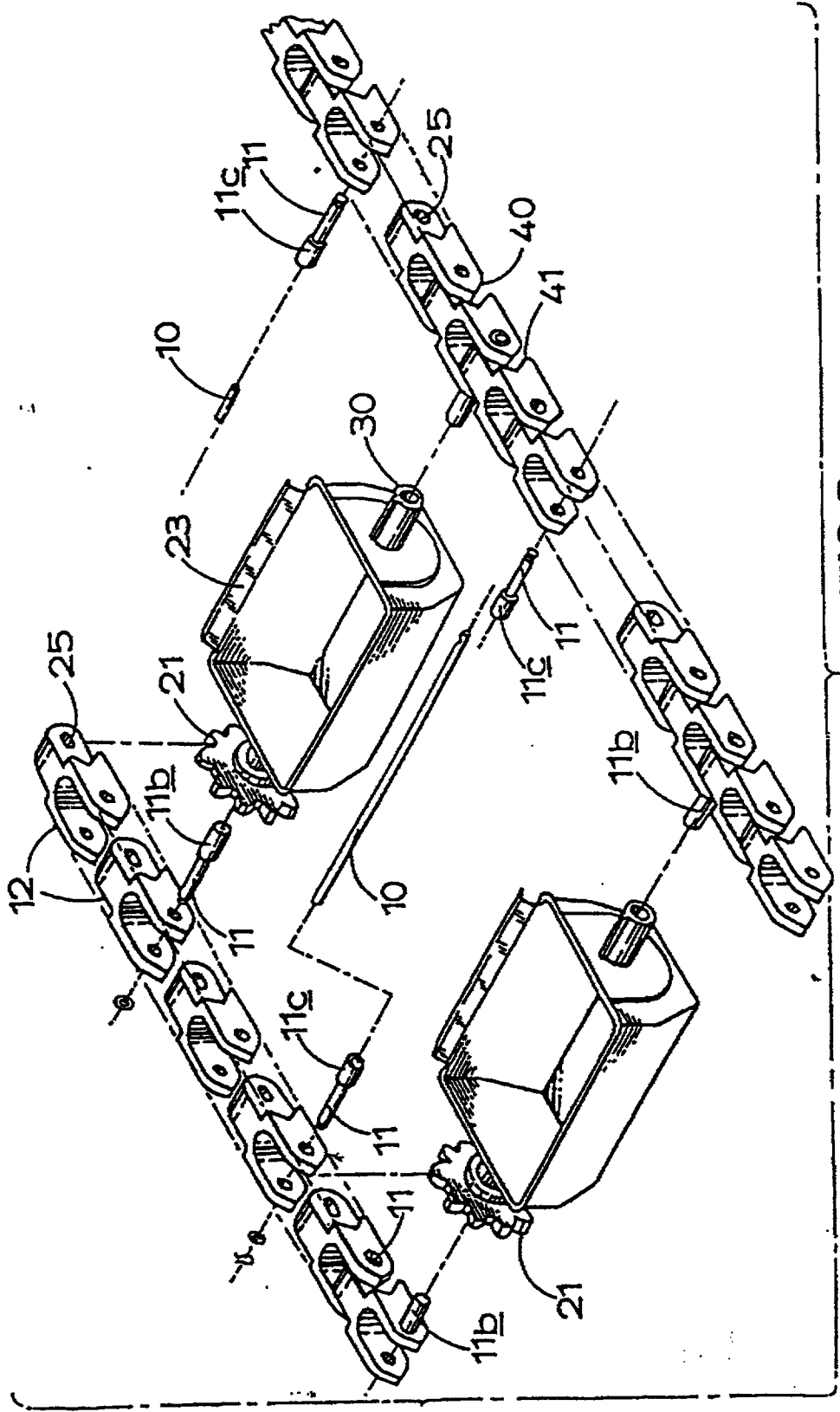


FIG. 3.

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- 1 -

BUCKET ELEVATOR CONVEYORS

REPLACED SHEETS		SG
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☒	Claim(s) : paged ...6..... to paged ...8.....	
☐	Drawing(s) : paged to paged	
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Description

The present invention relates to improvements in bucket elevator conveyor systems used for the gentle transport of loose materials and is particularly concerned with the systems that allow for continuous feeding of the loose material into the carrying bucket without spillage and without damage thereto.

Bucket elevator conveyor systems are known in which buckets, in spaced apart relationship, are rotatably supported between two chains that enables them to negotiate corners on a path in one plane, the buckets being arranged to be inverted at a desired discharge station. In order to achieve continuous filling of the buckets, necessary for efficient operation of the system, the chains, in one type of system, have to be collapsed in order to bring the buckets close together and minimise spillage, adequate chain control to produce this effect is difficult to achieve.

A bucket elevator system of the aforementioned type is described in EP-A-0212858 in which the buckets are carried in swinging links that enable them to pass round a bend of greater radius than the chain whereby touching buckets separate and remain horizontal. The preamble of claim 1 is based on this prior art.

Elevators of this type are used for handling a very wide range of materials that are required to be handled gently during production and/or packaging to avoid damage. Such materials include, for example, peanuts, confectionery, milk powder, tile dust, carbon black, precious metal powders, cereals, pasta, chocolate, semi-liquids and delicate or volatile materials.

We have now devised a bucket elevator conveyor system which overcomes the problems associated with known systems such as those described above. The

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Date: 49 SEP 1997

bucket elevator conveyor system according to the invention comprises a track to route at least one chain, comprising a multiplicity of links connected together by pins, through an endless circuit including a loading station and buckets pivotally mounted on the said chain for movement therewith and to receive material at the loading station and discharge material at the discharge station and means to drive the said chain.

The system is characterised in that: pins pivot in elongated holes formed in the links such that the length of a section of the chain is greater when in tension compared with its length when in compression; means are provided to cause a said section of the chain to be in compression at the loading station; the spacing of the buckets is such that when the chain is in compression adjacent edge portions of the buckets are in contact and when the chain is in tension the buckets can be rotated through 360° about the pivotal mountings.

Preferably the buckets are pivotally suspended between two chains.

Preferably also means are provided for ensuring that when a said section of the chain is in compression it adopts a relatively stiff state. Suitable means for achieving this effect may comprise guide elements surrounding the track on three sides and/or, preferably, adjacent links being provided with interlocking portions that engage when the chain is in compression to reduce the tendency of the chain to buckle and causing it to become rigid in the plane of movement.

The driving means may comprise one or more sprockets driven by an electric motor; the means for bringing a section of the chain into compression at a loading station may comprise a pair of sprockets rotatably mounted at either end of the said loading station and spaced at a distance apart equivalent to the compressed length of the chain in that section. The chain

drive should be on the entry side of the said section and may comprise the appropriate one of said sprockets.

One embodiment of the invention will now be described with reference to the accompanying drawings in which:-

Figure 1 is a diagrammatic side view of a bucket elevator conveyor system according to the present invention;

Figure 2 is a diagrammatic view of the loading station of the circuit of Figure 1; and

Figure 3 is an exploded diagrammatic illustration of a chain and bucket assembly for use with the system of Figures 1 and 2.

Referring to the drawings, a bucket elevator system of the present invention comprises an outer framework generally indicated at 1 having a horizontal first section 1a at one level, a vertical section 1b and a second horizontal section 1c at the second higher level. The said framework houses a circuit of parallel flexible chains 2 consisting of chain links 12 having slotted pivot locations 25 and joined by pivot pins 11 (see Figure 3). The chains move through a loading station 14, in frame section 1a, along a U-shaped guide 35 and in a substantially horizontal path, a vertical section 1b and a discharge station 20 in frame section 1c, in which the chains are again substantially horizontal and at a higher level.

The chains are driven by a pair of sprockets 6, located immediately before the loading station 14, and an electric motor 15 coupled to sprockets 6 by chain 16. The chains are maintained in engagement with sprockets 6 by an encapsulating shroud 13. From sprocket 6 the chains pass over a first pair of control sprockets 8

positioned after the loading station 14 and are maintained in engagement therewith by an encapsulating shroud 22. The chains then move upwardly over guide sprockets 7 and thence horizontally to pass round a terminal sprocket 18 to move horizontally in the opposite direction to a discharge station 20. The chains turn round a second pair of guide sprockets 27 downwardly through chain guide 4 to a second pair of control sprockets 37 and thence horizontally back to drive sprockets 6.

Terminal sprocket 18 is carried for free rotation about shaft 18a journaled into bearings with sliding mounts and spring loaded at 19 to apply a small tension to the extended chain circuit.

Sprockets 6, 7, 8, 27 and 37 are rotatably carried on common shafts 6a, 7a, 8a, 27a and 37a respectively, each shaft being journaled into bearings secured in frame 1. The sprockets have the same number and pitch of teeth in order that each forwards the chain at the same speed.

Suspended between the chains are buckets 3 carried on inwardly extending end portion 11a of pivot pins 11 that engage in hollow spigots 30 extending outwardly from each side of buckets 3. Gears 21 are non-rotatably secured to spigots 30 on one side of each bucket. Alternatively the male pins can be on the chains and the hollow spigots formed in the bucket trunnions.

The parallel chains 2 are maintained at a constant distance apart by use of tie bars 10 which locate in hollow ends 11c of pivot pins 11 and are positioned between any or all pairs of buckets.

To avoid spillage at the feeding station buckets 3 are provided with overlap member lips 23 located along one side thereof and extending outwardly to extend over an adjacent bucket side edge to provide cover for the space between buckets when they are juxtaposed at the loading station 14.

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Date: 4 9 SEP 1997



The chain links 12 are formed with V-shaped portions 40 at one end that interlock with complimentary shaped portions 41, formed in the opposite end of an adjacent link, when the chain is in compression causing the links to lock into a rigid section. When the chain is under tension portions 40 and 41 separate to allow articulation.

Guide sprockets 8 and 37 are connected by chain 31 passing over the snubs 32 and 33 located either side of drive elements 24 on sprockets 8 and corresponding elements 38 on sprockets 37 whereby sprockets 8 and 37 are caused to rotate at the same speed and maintain correct rotation and correct closed or compressed chain pitch control throughout this section.

In setting up the conveyor system the relative positions of drive sprockets 6 and control sprockets 8 are arranged such that the chain is in a closed rigid and compressed state at the feed station 14 causing the buckets 3 to lie in near touching relationship with the overlap portions 23 of one bucket extending over the opposed edge of an adjacent bucket. To further control the positioning of the buckets 3 in a horizontal position at the feed station 14, and to provide stability to the buckets, a spring-loaded hollow frame 24 contacts the side top edges of the buckets as they pass through the feed station.

In use, with the chains 2 at the feed station 14 in a compressed and rigid condition, buckets 3 are continuously filled with the material to be transferred and drive sprocket 6 propels the chain around control sprocket 8 into the vertical section 1b of frame 1. Terminal sprocket 18 is carried on bearings with sliding mounts and spring-the circuit chain positioned between central sprockets 8 and 37 as it passes round sprocket 6. The chain links slide into the extended state through the upper section of the circuit allowing the buckets 3 to swing freely about the support pivots 11b. Buckets 3 pass around the upper

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Date: 09 SEP 1997



sprockets 7 and 37 and the terminal sprocket 18 to the discharge point where bucket gears 21 engage a toothed rack (not shown) which causes the buckets to rotate through 360° during which rotation material contained in the buckets is discharged. Thereafter the buckets continue around the circuit back to the feed station.

Claims

1. A bucket elevator conveyor system comprising a track to route at least one chain (2), comprising a multiplicity of links (12) connected together by pins (11), through an endless circuit including a loading station (14) and buckets (3) pivotally mounted (11a, 30) on the said chain (2) for movement therewith and to receive material at the loading station (14) and discharge material at the discharge station (20) and means (6,15,16) to drive the said chain (2), characterised in that:

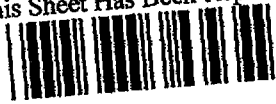
- (i) pins (11) pivot in elongated holes (25) formed in the said links (12) such that the length of a section of the chain (2) is greater when in tension compared with its length when in compression;

- (ii) means (6, 8) are provided to cause a said section of the chain to be in compression at the said loading station (14);

- (iii) the spacing of the said buckets (3) is such that when the chain (2) is in compression adjacent edge portions of the buckets (3) are in contact and when the chain (2) is in tension the buckets (3) can be rotated through 360° about the pivotal mountings (11a, 30).

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sprockets 7 and 37 and the terminal sprocket 18 to the discharge point where bucket gears 21 engage a toothed rack (not shown) which causes the buckets to rotate through 360° during which rotation material contained in the buckets is discharged. Thereafter the buckets continue around the circuit back to the feed station.

Claims

1. A bucket elevator conveyor system comprising a track to route at least one chain (2), comprising a multiplicity of links (12) connected together by pins (11), through an endless circuit including a loading station (14) and buckets (3) pivotally mounted (11a, 30) on the said chain (2) for movement therewith and to receive material at the loading station (14) and discharge material at the discharge station (20) and means (6,15,16) to drive the said chain (2), characterised in that:

(i) pins (11) pivot in elongated holes (25) formed in the said links (12) such that the length of a section of the chain (2) is greater when in tension compared with its length when in compression;

(ii) means (6, 8) are provided to cause a said section of the chain to be in compression at the said loading station (14);

(iii) the spacing of the said buckets (3) is such that when the chain (2) is in compression adjacent edge portions of the buckets (3) are in contact and when the chain (2) is in tension the buckets (3) can be rotated through 360° about the pivotal mountings (11a, 30).





2. A bucket elevator conveyor according to claim 1 wherein the buckets (3) are pivotally suspended between two chains (2).
3. A bucket elevator conveyor according to claim 1 or claim 2 including means (35, 40, 41) to cause the links (12) to interlock and the chain (2) to adopt a rigid state when in compression.
4. A bucket elevator conveyor according to claim 3 wherein said interlocking means comprises guide elements (35) partially encapsulating the said chain (2).
5. A bucket elevator conveyor according to claim 3 wherein said interlocking means comprises interpression, which portions reduce the tendency of the chains to buckle in the plane of movement.
6. A bucket elevator conveyor according to any one of the preceding claims wherein the means for driving the chain comprises a sprocket (6) member powered by a motor (15).
7. A bucket elevator conveyor according to claim 6 wherein the means for maintaining a section of the chain in compression comprises a pair of sprockets (6, 8) rotatably mounted at the entry and exit ends of said section (14) and spaced apart at a distance equivalent to the compressed length of the chain (2) in that section; the chain drive means (6) being positioned on the entry side of said section.
8. A bucket elevator conveyor according to claim 7 wherein the chain drive comprises the sprocket (6) mounted at the entry end of said section (14).

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Date: 4 9 SEP 1997





9. A bucket elevator conveyor according to claim 7 or claim 8 wherein a third control sprocket (37) is rotatably mounted in the section of the chain circuit approaching the said drive means (6) and is arranged to rotate at the same speed as the sprocket (8) on the exit side of the compressed chain section, all said sprockets (6, 8, 37) having the same number and pitch of teeth whereby the chain is forwarded at the same speed by each said sprocket.
10. A bucket elevator conveyor according to any one of the preceding claims wherein means (24) are provided to stabilise the buckets in a substantially horizontal position when passing through the compressed section of the chain.
11. A bucket elevator conveyor according to claim 10 wherein said stabilising means comprises a spring-loaded hollow frame member (24) dimensioned to contact and bear down on the top edge of the buckets during their passage through the said section.



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Date: 9 SEP 1997



BUCKET ELEVATOR CONVEYORS

ABSTRACT

A bucket elevator conveyor comprises a frame housing (1) incorporating a track to route a pair of parallel chains (2) carrying buckets (3) pivotally mounted (11a, 30) therebetween through an endless circuit including loading (14) and discharge (20) stations. The chains consist of links (12) connected together by pins (11) pivoting in elongated holes (25) such that the chains are shorter in compression than when under tension, the ends of adjacent links (12) including interengaging portions (40, 41) to cause the chains to adopt a relatively stiff state when in compression. The chains (2) are maintained under compression in a section of track at the loading station (14) by two pairs of sprockets (6, 8) positioned at the entry and exit ends of the section and spaced apart to maintain the chains under compression, the sprockets (6) at the entry end being driven by an electric motor drive (15). The spacing of the buckets (3) is such that when the chains (2) are in compression at the loading station (14) the end edge portions of adjacent buckets (3) are in contact, and when the chain is under tension at the discharge station (20) the buckets (3) can be rotated through 360° without colliding.

FIGURE 1

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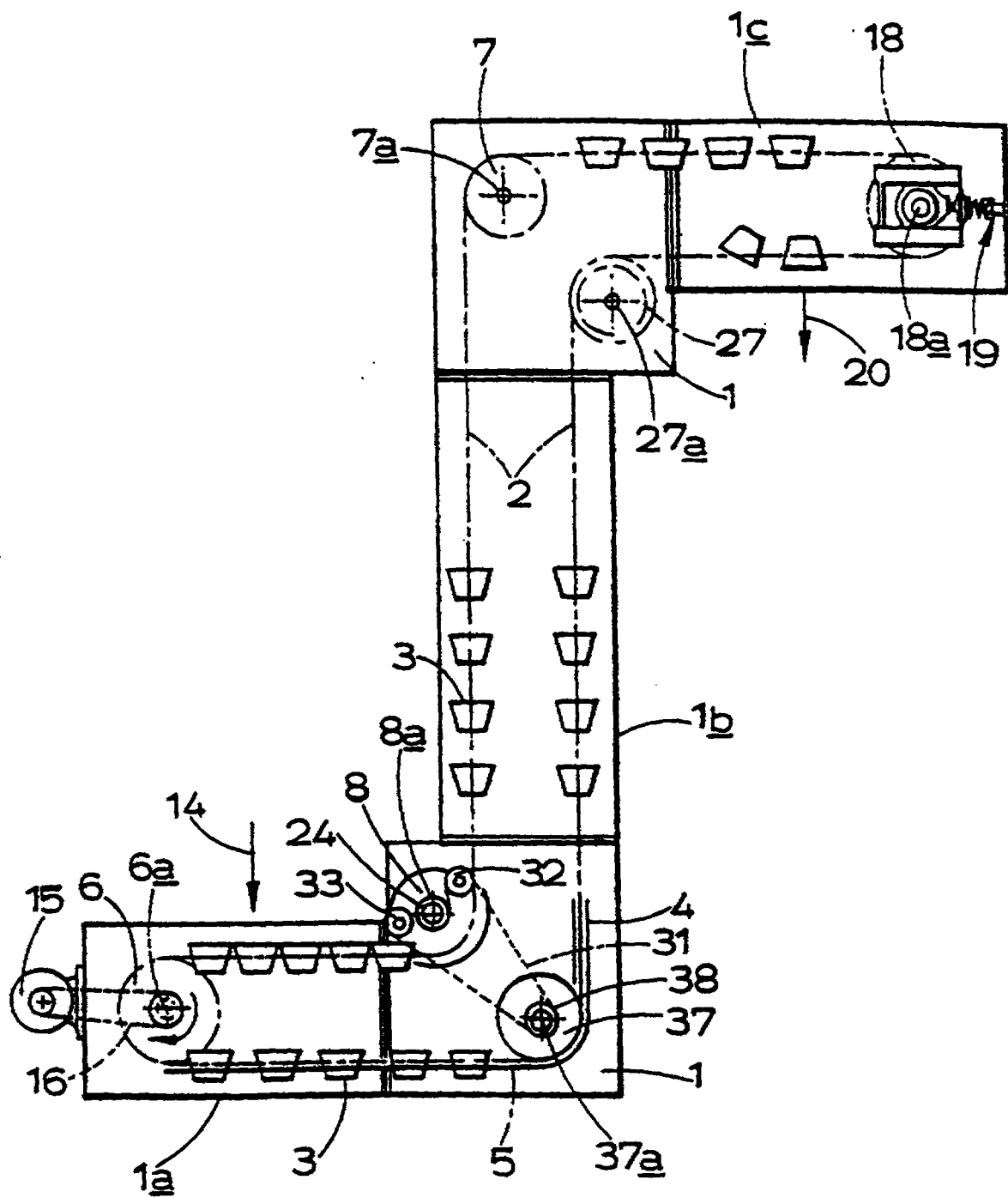
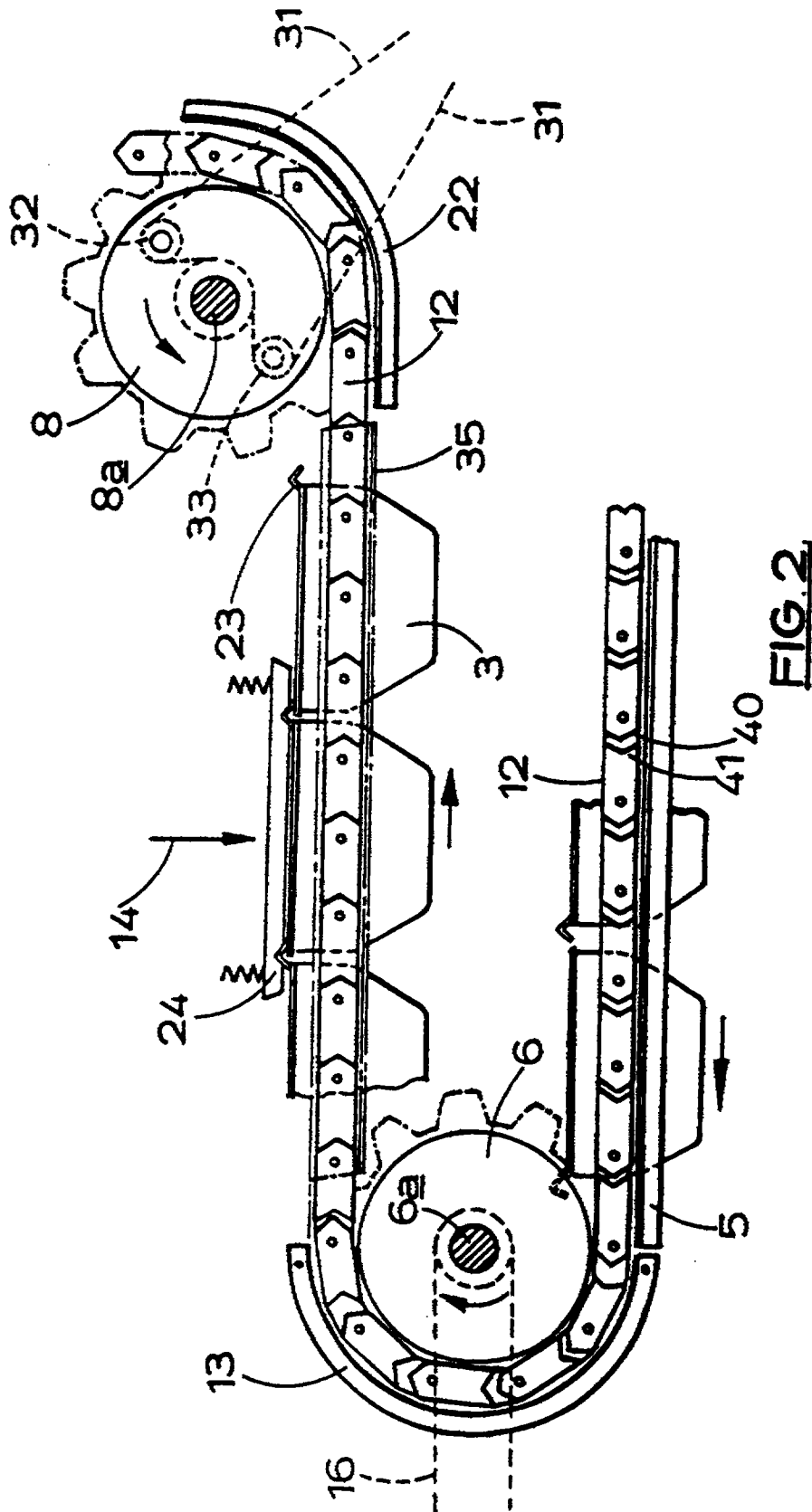


FIG.1.



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Date: 19 SEP 1997

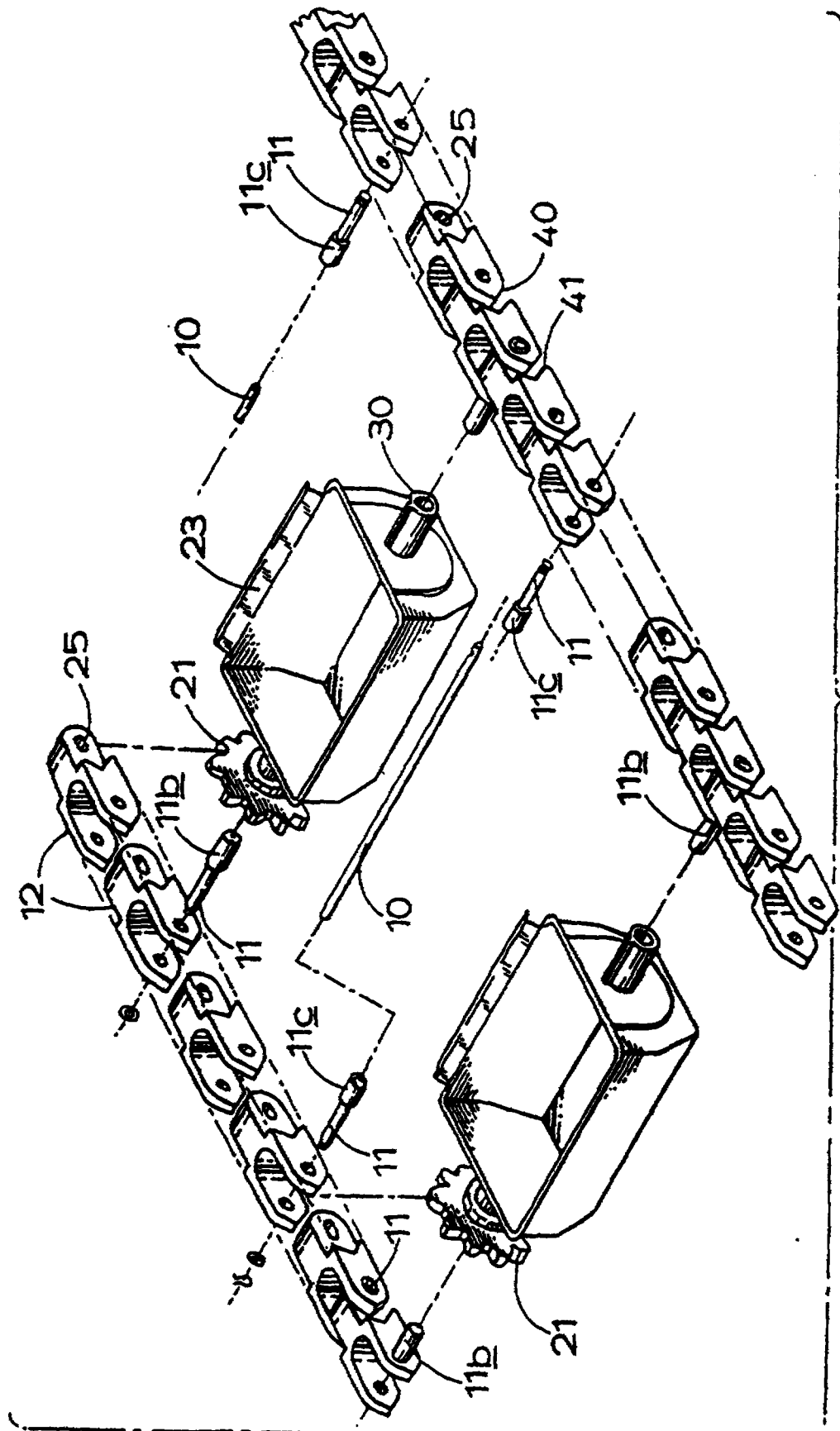


FIG. 3

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Date: 19 SEP 1997