

PUMP CONTROLLER FOR PRECISION PUMPING APPARATUS

Background of the Invention

This invention relates generally to precision pumping apparatus and, more particularly to a pump controller for accurately controlling the amount of fluid
5 dispensed from the precision pumping apparatus.

There are many applications where precise control over the amount and/or rate at which a fluid is dispensed by a pumping apparatus is necessary. In semiconductor processing, for example, it is important to control very precisely the amount and the rate at which photochemicals, such as photoresist, are applied to a semiconductor wafer
10 being processed to manufacture semiconductor devices. The coatings applied to semiconductor wafers during processing typically require a flatness across the surface of the wafer that is measured in angstroms. Many semiconductor processes today have requirements on the order of 30 angstroms or less. The rate at which processing chemicals such as photoresists are applied to the wafer and spun out through
15 centrifugal force to the edges of the wafer has to be controlled in order to ensure that the processing liquid is applied uniformly. It is also critical to control the rate and volume at which photoresist chemicals are applied to the wafer in order to reduce unnecessary waste and consumption. Many of the photochemicals used in the semiconductor industry today are not only toxic, but they are very expensive,
20 frequently costing as much as \$1,000 per liter. Thus, because of the cost of the chemicals as well as the difficulties in handling toxic materials, it is necessary to

ensure that enough of the photoresist is applied to the wafer to satisfy processing requirements while minimizing excessive consumption and waste.

Another important requirement for semiconductor processing is the ability to repeatedly dispense a precisely controlled amount of processing chemical each time since variations in the amount of chemicals can adversely impact consistency from wafer to wafer. In the past, because of the unrepeatability as well as the inability to precisely control the amount of chemical being dispensed, many pumps had to dispense 50% to 100% more liquid than needed in order to ensure a sufficient quantity for processing requirements. This has resulted in waste and increased processing costs.

Conventional pumping apparatus are able to accurately dispense precise amounts of typical fluids. However, these conventional pumping apparatus cannot accurately dispense low viscosity, low dispense rate fluids and the conventional pumping apparatus will either cause a double dispense or a stuttered dispense of the low viscosity fluid. In particular, at the beginning of the dispensing cycle prior to the controlled dispensing of any fluid, a small amount of the low viscosity fluid, e.g., several microliters, may be undesirable ejected onto the wafer's surface resulting in an imprecise amount of fluid being dispensed. The problems of double dispensing and stuttered dispensing of these low viscosity, low flow rate fluids are caused by a variety of factors which are present in a conventional pumping apparatus. For example, pressure may be built up in the dispensing chamber of the pumping apparatus due to the closing of a barrier valve prior to dispensing which may force some fluid into the

dispensing chamber and increases the pressure in the dispensing chamber. The extra fluid and hence the extra pressure in the dispensing chamber may cause the small amount of fluid to be ejected onto the wafer's surface at the start of the dispensing cycle. In addition, the timing of the control valves operation and the dispense system dynamics, such as tubing length, tubing diameter and nozzle size, in a conventional pumping apparatus may also contribute to the problem of the double or stuttered dispense of low viscosity, low dispense rate fluids.

It is desirable to provide low volume, low rate chemical dispensing pumping apparatus capable of precise and repeatable control of the rate and volume of low viscosity chemicals dispensed by the pumping apparatus, and it is to these ends that the present invention is directed.

Summary of the Invention

In accordance with the invention, a low dispense rate precision dispensing pumping apparatus and method is provided which enable precise and repeatable control of dispense rate and volume of low viscosity fluids, and which overcomes the foregoing and other disadvantages of conventional dispensing pumping apparatus and method. The pumping apparatus precisely controls the dispensing amount and/or rate of low viscosity fluids by precisely controlling the operation of several different portions of the pumping apparatus during the dispense cycle. In particular, a pump controller may precisely control the timing of the control valves with respect to each other, the motion of the dispensing motor, and the timing of the control valves with

respect to the movement of the dispensing motor. The pump controller in accordance with the invention accurately controls a pumping apparatus to avoid the double dispense or stuttered dispense problems associated with conventional pumping apparatus.

5 **Brief Description of the Drawings**

Figure 1 is a block diagram illustrating a pumping apparatus including a pump controller in accordance with the invention;

Figure 2 is a block diagram illustrating a two-stage pumping apparatus;

10 Figure 3 is a timing diagram illustrating the conventional sequence for dispensing fluids;

Figure 4 is a timing diagram illustrating a sequence for dispensing fluids in accordance with the invention; and

Figure 5 is a flowchart illustrating a method for controlling a pumping apparatus to dispense low viscosity fluids in accordance with the invention.

Detailed Description of a Preferred Embodiment

The invention is particularly applicable to a pumping apparatus which accurately dispenses precise amounts of low viscosity fluids and it is in this context that the invention will be described. It will be appreciated, however, that the apparatus and method in accordance with the invention has greater utility, such as to accurately dispensing precise amounts of other fluids which may not be low viscosity fluids.

Figure 1 is a block diagram illustrating a pumping apparatus 10 including a pump controller in accordance with the invention. The pumping apparatus 10 may include a two-stage pump 12, a fluid reservoir 14 and a computer 16 which operate together to dispense a precise amount of fluid onto a wafer 18. For purposes of illustration, a low viscosity fluid, which may have a viscosity of less than 5 centipoise (cPs), may be dispensed at a low flow rate of about 0.5 milliliters per second, but the invention is not limited to dispensing low viscosity fluids or low flow rate fluids. The pump 12 is a two-stage pump since the dispensing of the fluid includes a first feed and filtration stage and then a second separate dispensing stage as described below so that the dispense performance does not change over the lifetime of the filter. The operation of the various portions of the pump 12 may be controlled by a software application 20, i.e., a computer program comprising pieces of software code which may be stored in a memory in the computer 16 and may be executed by a processor (not shown) in the computer. The operation of the pump may also be controlled by a software application or pieces of software code which are being executed by a processor located inside the

pump. The location of the processor executing the instructions to control the operation of the pump is not critical to the invention.

The software application 20 may control, for example, the opening and closing of the various control valves in the pump and the movement of the motors or actuators which drive the pump in order to accurately dispense a precise amount of fluid onto the wafer 18. The method implemented by the software application for controlling the pump 12 to dispense low viscosity, low flow rate fluids in accordance with the invention will be described below with reference to Figure 5.

To fill itself with fluid, the pump 12 may draw fluid from the reservoir 14 into a feed chamber as described below. The fluid may then be filtered through a filter and fed into a separate dispensing chamber as described below. From the dispensing chamber, the fluid may be dispensed through a filter 22 onto the wafer 18 in precise amounts even for low viscosity, low rate fluids. The actual cycles of the pump 12 will be described below with reference to Figures 3 and 4. Now, the details of the two-stage pump 12 will be described in order to better understand the invention.

Figure 2 is a block diagram illustrating more details of the two-stage pump 12 with which the invention may be employed. In particular, the two-stage pump 12 may include a feed and filtration stage 30 and a dispensing stage 32. The feed and filtration stage 30 may include a feed chamber 34 which may draw fluid from a fluid supply reservoir through an open inlet valve 36 as more fluid is needed. During the dispensing stages, the inlet valve 36 is closed. To control entry of fluid into and out of

the feed chamber, a feed valve 38 controls whether a vacuum, a positive feed pressure or the atmosphere is applied to a feed diaphragm 40 in the feed chamber. To draw fluid into the feed chamber, a vacuum is applied to the diaphragm 40 so that the diaphragm is pulled against a wall of the feed chamber and pulls fluid into the feed
5 chamber. To push the fluid out of the feed chamber, a feed pressure may be applied to the diaphragm. To remove unwanted air bubbles, a vent valve 42 may be opened as needed.

Once the feed chamber 34 is filled with fluid, the inlet valve 36 is shut and the isolation valve 44 and a barrier valve 50 are opened to permit the fluid to flow through
10 a filter 46 into the dispensing stage 32. Once the fluid is in the dispensing stage 32 and to isolate the feed and filtration stage from the dispensing stage, the isolation valve 44 and the barrier valve 50 may be closed. To vent unwanted air from the system or relieve excess pressure, the filter 46 may include a vent valve 48. As the fluid is pushed through the filter 46, unwanted impurities and the like are removed from the
15 fluid. The fluid then flows through a barrier valve 50 into a dispensing chamber 52 in the second or dispensing stage of the pump, and the pump begins a dispense cycle as will now be described.

In the dispensing cycle, once the dispensing chamber is full of fluid and the barrier valve 50 is closed, a purge valve 54 is opened and the fluid in the dispensing
20 chamber 52 is pushed by a dispense diaphragm 56 to eliminate any bubbles in the fluid in the dispensing chamber 52. To push or pull the dispense diaphragm 56, the

dispensing diaphragm may be between the dispensing chamber and a hydraulic fluid chamber 58 filled with hydraulic fluid. The hydraulic fluid may be pressurized or depressurized by a dispensing pump 60 which may include a piston 62, a lead screw 64 and a stepper motor 66. To apply pressure to the fluid in the dispensing chamber 52, the stepper motor is engaged which engages the lead screw and pressurizes the hydraulic fluid. The hydraulic fluid in turn pushes the dispensing diaphragm into the dispensing chamber 52 which pressurizes the fluid in the dispensing chamber 52 or pushes the fluid out of the dispensing chamber 52 if the purge valve 54 or an outlet valve 68 are opened. If the outlet valve 68 is open, then an accurate amount of the fluid is dispensed onto the wafer. Now, the typical process for dispensing fluid will be described.

Figure 3 is a timing diagram illustrating the conventional sequence for controlling a two-stage pump of the type shown in Figure 2 to dispense fluids. As shown at the top of the diagram, the dispensing process may include a sequence of stages, i.e., steps such as a ready stage 70, a dispense stage 72, a suckback stage 74, a fill stage 76, a filter stage 78, a vent stage 80, a purge stage 82, a static purge stage 84. The typical controlling of the motors and valves for each of these different stages will now be described along with the result that occurs as a result of each stage. For example, during the ready stage, the barrier and isolate valves are opened while the outlet valve is shut to bring the system and feed chamber to an equilibrium pressure state so that fluid may be dispensed. As the dispense stage begins, the isolate and barrier valves close, the outlet valve is opened and the motor in the dispensing pump is

started. Due to the relative incompressibility of the fluid being dispensed and the "stiffness" of the pump, the closing of the barrier valve pushes fluid out of the valve as it closes which pressurizes the fluid in the dispensing chamber and may cause the typical double dispense or stuttered dispense problem as described above since the outlet valve is open. The closure of the barrier valve may increase the pressure in the dispensing chamber by a predetermined amount, which may be about 2 - 3 psi. The actual pressure increase, however, depends on the characteristics of the barrier valve being used. In addition, since the motor is started at the same time as the outlet valve is opened, an uneven dispensing of fluid (or stuttered dispensing) may occur since the outlet valve takes more time to open than the starting of the motor and therefore the motor may be initially pushing the fluid through an outlet valve which is not quite completely open. This may cause an initial "spitting" of a small amount of fluid. During the dispensing stage, fluid may be dispensed onto the wafer.

At the end of the dispensing stage and at the beginning of the suckback stage, the motor is stopped and reversed or an external stop/suckback valve (not shown) may be opened to suck any fluid remaining in the nozzle back into the dispensing chamber to ensure that no drips occur at the end of the fluid dispensing. After the fluid has been sucked back into the dispensing chamber, the outlet valve is closed and the motor is stopped. Next, during the fill stage, the inlet valve is opened and a vacuum is applied to the feed diaphragm to draw fluid into the feed chamber from the reservoir. At the beginning of the filter stage, the inlet valve is closed, the isolate valve is opened, the feed motor applies positive pressure to the fluid in the feed chamber, the barrier valve

is opened and the dispense motor is reversed to push fluid through the filter into the dispense chamber. Once the fluid has exited the feed chamber, the isolate valve may be closed.

At the beginning of the vent stage, the isolate valve is opened, the barrier valve
5 is closed, the vent valve is opened, the dispense motor is stopped and pressure is applied to the feed diaphragm to remove air bubbles from the filter. At the beginning of the purge stage, the isolate valve is closed, the feed pump does not apply pressure or a vacuum to the feed chamber, the vent valve is closed, the purge valve is opened and the dispense pump is moved forward to remove air bubbles from the dispensing chamber.
10 At the beginning of the static purge stage, the dispense motor is stopped but the purge valve remains open to continue the removal of air from the dispensing chamber. At the beginning of the ready stage, the isolate and barrier valves are opened and the purge is closed so that the feed pump and the system reaches ambient pressure and the pump is ready to dispense fluid.

15 As described above, this conventional dispensing process suffers from double dispense or stuttered dispense problems. In particular, the closure of the barrier valve prior to dispensing pushes fluid out of the valve as it closes which pressurizes the fluid in the dispensing chamber. This may cause a small amount of unwanted fluid to dispense onto the wafer since the outlet valve is open. In addition, since the motor is
20 started at the same time as the outlet valve is opened, an uneven dispensing of fluid (or stuttered dispensing) may occur since the outlet valve takes more time to open than the

starting of the motor and therefore the motor may be initially pushing the fluid through an outlet valve which is not quite completely open. A dispensing method in accordance with the invention which solves these problems will now be described.

Figure 4 is a timing diagram illustrating a method for dispensing fluids in accordance with the invention. As with the conventional dispensing process described above, the dispensing process shown in Figure 4 has the same stages, i.e., steps, 70 - 84 as the conventional process. In addition, much of the controlling of the valves and motors is similar to the conventional method above, and only the changes in the controlling of the valves and motors in accordance with the invention will be described here. In particular, in order to prevent the unwanted double dispense or stuttered dispense problems, the method changes the manner of controlling of the valves and motors.

In particular, in accordance with invention, the barrier valve is not closed at the beginning of the dispense stage as it done in the conventional process. Rather, the barrier valve is closed at the beginning of the vent stage and kept closed during the dispense stage. This avoids the sudden rise in pressure in the dispense chamber and, therefore, fluid does not leak out of the outlet valve due to the sudden rise in pressure. Since the barrier valve does not open and close prior to the beginning of the dispense stage, but does close at the beginning of the vent stage, the pressure in the dispense chamber does increase after the vent and purge states and this additional pressure must be released. To release this pressure, during the static purge stage 84, the dispense

motor may be reversed to back out the piston 62 some predetermined distance to compensate for any pressure increase caused by the closure of the barrier valve. As an example, each step of the stepper motor may reduce the pressure by about 0.1 psi. If the closure of the barrier valve increases the pressure by 2 psi, then the motor may be reversed 20 steps to reduce the pressure in the dispense chamber by this amount to compensate for the closure of the barrier valve. The actual pressure decrease, however, depends on the characteristics of the particular stepper motor, lead screw and piston being used. The pressure decrease caused by each step of the motor may be determined by a pressure sensor which is located inside the dispensing chamber. In accordance with the invention, since the outlet valve is not open when the additional pressure is added into the dispensing chamber during the vent stage, no "spitting" of the fluid onto the wafer may occur.

The motor may be further reversed a predetermined additional distance so that the motor may be moved forward just prior to dispensing to adjust the dispense pressure to zero and avoid any backlash which normally occurs when the motor is moved backwards before the dispensing of fluid. In particular, with a piston, lead screw and stepper motor dispense pump, the last motion prior to a dispense operation is normally forward to avoid the fact that, as the piston changes direction, there is some backlash. Thus, the problem of the additional pressure caused by the closure of the barrier valve is avoided.

Next, during the beginning of the dispense stage 72, the timing of the outlet valve and the start of the motor are changed to avoid the stuttering dispense problem. In particular, the valve is a mechanical device that requires a finite period of time to open. The motor, on the other hand, may start more quickly than the outlet valve may

5 open. Therefore, starting the motor and opening the outlet valve simultaneously will cause a rise in pressure of the dispense fluid which in turn causes the stuttered dispensing. To avoid this problem, the outlet valve is opened and then, some predetermined period of time, T, later, the dispense motor is started so that the outlet valve is completely open when the motor is started which achieves a good dispense.

10 The predetermined period of time depends on the characteristics of the outlet valve and dispense motor being used, but, if the outlet valve takes approximately 50 ms to open, then the predetermined period of time may be, for example, between 50 and 75 mS and preferably approximately 75 mS. This predetermined period of time may also be referred to as a delay. Thus, in accordance with the invention, the dispense motor is no

15 longer pushing fluid through a partially open outlet valve so that an accurate, controlled amount of fluid may be dispensed onto the wafer. Thus, in accordance with the invention, the problems caused by the closure of the barrier valve and the simultaneously opening of the outlet valve and starting of the dispense motor are avoided to provide more accurate dispensing of fluids, such as low viscosity fluids.

20 As described above, the valves and motors in the pumping apparatus are controlled by a software application so that the above changes in the dispensing process may be applied to any two-stage pumping apparatus since no hardware

changes are needed. Thus, for example, if the tubing, tubing length, nozzle height or nozzle diameter is changed, the process in accordance with the invention may be easily adapted. Now, the method for controlling the dispense process in accordance with the invention will be described.

5 Figure 5 is a flowchart illustrating a method 100 for controlling the dispensing of low viscosity fluids from a pumping apparatus in accordance with the invention. At step 102, the barrier valve is closed at the end of the filtering stage which increases the pressure in the dispense chamber. In step 104, during the static purge stage, the dispense motor is reversed a predetermined distance to compensate for the pressure
10 increase caused by the closure of the barrier valve. Next, in step 106, the motor may be reversed an additional distance so that, in step 108, when the motor is moved forward to eliminate backlash, the pressure of the dispense chamber remains at zero. In step 108, the pump is now ready for dispensing. In step 110, the outlet valve is opened. Next, in step 112, the dispense motor is started some predetermined period of
15 time later and fluid is dispensed in step 114. The method is then completed.

While the foregoing has been with reference to a particular embodiment of the invention, it will be appreciated by those skilled in the art that changes in this embodiment may be made without departing from the principles and spirit of the invention.

WHAT IS CLAIMED IS:

1. A pump for dispensing fluid, comprising:
 - a multistage pump having a feed chamber and a dispensation chamber therein
 - 5 connected through a series of valves and motors configured to draw fluid within the respective chambers and to dispense the fluid from the pump;
 - a fluid reservoir for providing fluid to the feed chamber; and
 - a pump controller for controlling the operation of the series of valves and motors in the pump so that fluid is passed between the feed chamber and the
 - 10 dispensation chamber and dispensed wherein a precise amount of fluid is dispensed without a double dispense or a sputtered dispense.
2. The pump of Claim 1, wherein the multistage pump further comprises
 - a barrier valve disposed in the feed chamber being controlled by the pump
 - 15 controller so that the barrier valve is closed so that an increased pressure results within the dispensation chamber;
 - a dispensation motor disposed in the dispensation chamber configured to operate in a forward and a reverse direction being controlled by the pump controller so that the dispensation motor is operated in the reverse direction to compensate for the
 - 20 pressure in the dispensation chamber such that upon the dispensation motor being operated in the forward direction the pressure in the dispensation chamber results in a zero pressure; and
 - an outlet valve disposed in the dispensation chamber being controlled by the pump controller so that the outlet valve is opened so as to dispense the fluid from the
 - 25 dispensation chamber upon the dispensation motor being operated in the forward direction.
3. The pump of Claim 1, wherein the multistage pump comprises:



a fluid drawing means for drawing fluid from the fluid reservoir and supplying the fluid to the multistage pump;

a filtering means for filtering impurities from the fluid; and

a dispensing means for providing the filtered fluid to the object, wherein the
5 filtering means is disposed between the drawing means and the dispensing means,

4. The pump of Claim 3, wherein the fluid drawing means comprises a feed diaphragm disposed within the feed chamber and configured to move between a first drawing position and a second purging position in accordance with a drawing force
10 such that upon the feed diaphragm moving from the second purging position to the first drawing position, the fluid is drawn into the feed chamber via an inlet valve and upon the feed diaphragm moving from the first drawing position to the second purging position, the fluid is provided to the dispensation chamber via a feed valve.

15 5. The pump of Claim 4, wherein the drawing force is either a vacuum force, a positive feed pressure force or an atmospheric force.

6. The pump of Claim 4, further comprising a vent valve configured to remove air bubbles from the fluid.
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7. The pump of Claim 3, wherein the filtering means comprises a filter for removing impurities from the fluid and a vent valve for removing air bubbles from the fluid or for relieving excess pressure from the multistage pump.

25 8. The pump of Claim 3, wherein the dispensing means comprises a dispense diaphragm disposed within the dispensation chamber and configured to move between a first drawing position and a second purging position in accordance with a drawing force such that upon the dispense diaphragm moving from the second purging position to the first drawing position, the fluid is drawn into the dispensation chamber via a feed



valve and upon the dispense diaphragm moving from the first drawing position to the second purging position, the fluid is provided to the object via an outlet valve.

9. The pump of Claim 8, wherein the dispensing means further comprises a hydraulic fluid chamber configured to pressurize a hydraulic fluid resident within the hydraulic fluid chamber so that the dispense diaphragm is moved between the first and second positions when the hydraulic fluid is pressurized and so that the dispense diaphragm is moved between the second and first positions when the hydraulic fluid is depressurized.

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10. A process for controlling a multistage pump to dispense a fluid, the multistage pump having a feed chamber, a dispensation chamber and a filter disposed therebetween, the process comprising:

- 15 a ready stage for bringing the feed chamber to an equilibrium pressure state, wherein upon opening an isolation valve disposed in the feed chamber, closing an outlet valve disposed in the dispensation chamber and opening a barrier valve disposed in the feed chamber, the feed chamber is brought to an equilibrium pressure state;

- 20 a dispense stage for dispensing the fluid onto an object, wherein a dispensation pump disposed in the dispensation chamber is activated to dispense the fluid onto an object upon closing the isolation valve and opening the outlet valve such that the dispensation pump is activated subsequent to the outlet valve being opened so as to eliminate stuttered dispensing of the fluid; and

- 25 a suckback stage for eliminating excess fluid that flows out of the dispensation chamber, wherein operation of the dispensation pump is reversed to suck excess fluid back into the dispensation chamber and wherein the outlet valve is closed after the excess fluid is sucked back into the dispensation chamber.

11. The process of Claim 10 further comprising a fill stage for drawing fluid into the feed chamber, wherein upon opening the inlet valve and applying a vacuum to the a



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feed diaphragm disposed within the feed chamber the fluid is drawn into the feed chamber.

12. The process of Claim 10, wherein excess fluid spitting is eliminated from the
5 dispensation chamber.

13. A method for controlling the opening and closing of a plurality of valves and
the activating and deactivating of a plurality of motors in a multistage pump having a
feed chamber, a dispensation chamber and a filter disposed therebetween, the method
10 comprising the steps of:

closing a barrier valve in the feed chamber so as to increase the pressure within
the dispensation chamber;

reversing the operation of a dispensation motor in the dispensation chamber so
as to compensate for the pressure increase within the dispensation chamber;

15 further reversing the operation of the dispensation motor so that when the
dispensation motor is forward activated to compensate for backlash, the pressure of the
dispensation chamber remains at a zero pressure;

opening an outlet valve in the dispensation chamber; and

operating the dispensation motor subsequent to opening the outlet valve so that
20 fluid is dispensed from the dispensation chamber.

