

Coin passage apparatus with hinge means

The present invention relates to coin passage apparatus having hinge means for mounting together two plates defining a coin passageway. It is especially concerned with a hinge mechanism for mounting the flight deck of a coin testing mechanism.

In a coin testing mechanism a passageway for coins is usually formed by two closely spaced plates. A track carried by one of the plates supports the coins on their edges and defines the path which coins follow between the plates. Coin testing devices are arranged at positions alongside the path. Sometimes coins become jammed in the mechanism and it is necessary to provide means for separating the plates sufficiently to allow the jammed coins to fall from the track. In the known arrangements the plates are joined to one another by a simple hinge and a mechanism is provided for swinging the plates apart about the hinge when a coin is jammed. An example of such an arrangement is disclosed in U.S. patent 3,382,962.

The space available for a coin mechanism in a vending machine is often limited, particularly the depth from front to back. It has been found that in some instances there is insufficient depth to allow the plates to be swung apart sufficiently for coins close to the hinge axis to be released.

U.S. patent 4,106,608 discloses a coin flight deck assembly comprising a main plate, and first and second gates pivotally attached to the main plate about a common axis by means of a hinge mechanism. The second gate is disposed between the first gate and the main plate and when both gates are closed the second gate and main plate define between them a coin track. Coins which become jammed on the coin track can be released by operating and actuating mechanism which pivots the second gate about the common hinge axis away from the main plate towards the first gate. Operation of the actuating mechanism also causes a pivotally mounted scavenger blade to swing downwardly in the space between the main plate and second gate to urge any trapped coin close to the common hinge axis towards a discharge passage. The hinge mechanism is specially designed to enable both gates to be dismantled from the main plate, and also from one another, when both gates are in their open positions.

The present invention is concerned with providing coin passage apparatus from which even coins jammed close to the hinge axis can be released without having to provide a special coin scavenging blade.

According to the present invention there is provided coin passage apparatus comprising first and second plates defining a passageway for the movement therealong of coins and hinge

means interconnecting said plates to permit pivotal movement thereof away from each other, characterised by means permitting translatory movement of said plates away from each other at the pivot axis of the hinge means.

Suitably, the hinge means comprises a hinge pin mounted parallel to the plates for movement in a slot which extends generally perpendicular to the plane of one of the plates, the other plate being pivotally mounted about the axis of the hinge pin. The coin passage apparatus may include first biasing means biasing the plates together about the hinge axis, second biasing means biasing the two plates together in the region of the hinge means, and actuating means operable to cause the plates to pivot about the hinge axis against the action of the first biasing means and to move apart at the hinge axis against the action of the second biasing means.

Conveniently the first biasing means may comprise a coil spring disposed around the hinge pin and bearing on the one hand against part of the one plate or a part carried by the said one plate and on the other hand against the other plate. The second biasing means may comprise a leaf spring carried by the one plate and coupled to the hinge pin. The coupling may conveniently be provided by a coil spring disposed around the hinge pin and serving as the first biasing means.

The actuating means may be a lever pivoted on the one plate and carrying a cam surface which co-operates with a cam follower surface on the other plate to force the plates apart when the lever is operated.

An embodiment of the invention will now be described by way of example with reference to the accompanying drawings of which:—

Figure 1 shows two plates of a coin mechanism connected by a hinge mechanism according to the invention;

Figure 2 shows a back view of one of the plates of Figure 1;

Figure 3 shows a horizontal section on the line III—III of Figure 2;

Figure 4 shows a horizontal section on the line III—III with the plates in a different position relative to one another;

Figure 5 shows a horizontal section on the line III with the plates in yet another position, relative to one another;

Figure 6 shows a side elevation of the mechanism of Figures 1 to 5; and

Figure 7 shows a vertical section on the line VII—VII of Figure 5.

Referring to the drawings, these show part of a coin testing mechanism 11. In Figure 1 the parts are shown opened out more than they would be in normal everyday use. The mechanism does however permit the parts to be opened out as far as is illustrated for ease of

cleaning. The mechanism comprises two plates 12 and 13 connected together by a hinge mechanism 14. Rails 15 and 16 carried by the plates 13 and 12 provide a coin track which supports the coins on their edges and define, when the two plates 12 and 13 are closed together, a zig-zag coin path 17 through the mechanism. Coins inserted into the mechanism through a hopper 18 roll under gravity along the track 15, fall off its lower end onto track 16, roll down the track 16 and fall off its lower end into the lower part 40 of the coin mechanism (see Figure 7). Alongside the coin path are located the various coin testing devices, for example inductive sensor 39 as shown in Figure 2 of the kind described in our U.K. patent No. 1,397,083, which determine the authenticity and the denomination of the coins inserted.

The hinge mechanism 14 comprises a hinge pin 19 which passes through two vertically-spaced lugs 20 on the plate 12. The ends of the hinge pin 19 are received in slots 21 in two vertically-spaced lugs 22 on the plate 13. The slots 21 extend in a vertical plane perpendicular to the plane of the plate 13. Thus the hinge pin 19 serves to connect the plates 12 and 13 together and allow rotational movement relative to one another about the hinge axis and translational movement of the plates 12 in a direction perpendicular to the plane of the plate 13.

The plates 12 and 13 are biased together into the closed position by means of a coil spring 23. The central part of the coil spring is formed into a loop 24 which bears against the edge 25 of the plate 12. The ends 26 of the wire of the coil spring 23 extend through holes 27 in the plate 13 and are fixed to a leaf spring 28. The leaf spring 28 is carried on posts 29 on the back of the plate 13. The free end of the leaf spring 28 which carries the ends 26 of the coil spring 23 is spaced from the back of the back plate 13 and acting through the coil spring 23 holds the hinge pin 19 at the bases of the slots 21 in the lugs 22. Thus in the normal condition the two plates are close together at the hinge axis and lie parallel to one another.

For separating the plates an actuating mechanism in the form of a lever 31 is provided. The lever is pivotally mounted on a post 30 carried by the plate 13 so as to rotate about an axis normal to the plane of the plate 13. At one end the lever 31 has an operating handle 32 which depending on the particular coin mechanism in which it is used, may be operated by the person using the vending machine either directly or indirectly through a linkage. The other end of the lever 31 carries a cam surface 33.

A boss 34 on the plate 12 projects across the gap between the plates and through an opening 38 in the plate 13. The boss carries a cam follower surface 35 which is inclined relative to the plane of the plate 12.

The lever 31 is biased by a spring 36 so that

the operating handle 32 is normally in the raised position and the cam portion 33 is situated opposite the part of the boss 34 which has a lower height relative to the plate 12 (the left hand end as viewed in Figure 3). When the operating end of the lever 32 is depressed the cam portion 33 rides up the ram formed by the cam follower surface 35 as it moves to the right as viewed in Figures 3, 4 and 5. This action causes the plates to pivot about the axis of the hinge from the normal closed parallel position of Figure 3 into the position shown in Figure 4 in which the plates are spaced apart at the left hand edges. The relative strengths and configurations of the springs 23 and 28 are such that the force required to produce rotation about the hinge axis is less than is necessary to cause deflection of the leaf spring. When the plates reach the position shown in Figure 4 the free edge of the plate 12 strikes a fixed abutment 38 which may be part of the casing of the coin mechanism. This prevents further rotation of the plate 12 about the hinge axis. Further rotation of the lever 31 about its pivot axis causes the cam follower to ride up the less steeply sloping portion of the cam surface 35. The plate 12 rotates about its point of contact with the abutment 38 causing the hinge pin 19 to move outwardly in the slots 21 and the leaf spring 28 to be deflected into the position shown in Figure 5. Thus the spacing between the plates at the side adjacent the hinge 19 is increased.

It will be understood that when a coin is jammed in the mechanism shown in the drawings it can be released by operating the lever 31 to separate the plates and allow the coins to fall from the track. The mechanism shown in the drawings has the advantage that when a coin is jammed close to the hinge axis separation of the plates sufficient to allow the coin to fall from the track can be achieved even though the amount by which the plates can be separated at the left hand end is limited.

Modifications of the design described above are possible within the scope of the appended claims. For example we have found that it is possible to produce separation of the plates at the hinge pin without the use of a fixed abutment 38. By arranging the spring forces of the cantilever spring 28 and the coil spring 23 to balance one another, both plates can be caused to separate substantially parallel to one another by the cam 33. We have found that a cam follower surface 35 which has a single slope is particularly suitable for this arrangement, that is to say the flat area for the final portion of the cam surface is eliminated.

Claims

1. Coin passage apparatus comprising first (12) and second (13) plates defining a passage-way (17) for the movement therealong of coins and hinge means (14) interconnecting said

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Claims

1. Coin passage apparatus comprising first (12) and second (13) plates defining a passageway (17) for the movement therealong of coins and hinge means (14) interconnecting said



plates (12, 13) to permit pivotal movement thereof away from each other, characterised by means (21) permitting translatory movement of said plates away from each other at the pivot axis of the hinge means (14).

2. Apparatus according to Claim 1, characterised in that the hinge means comprises a hinge pin (19) mounted parallel to the plates (12, 13) for movement in a slot (21) which extends generally perpendicular to the plane of one (13) of the plates, the other plate (12) being pivotally mounted about the axis of the hinge pin (19).

3. Apparatus according to Claim 1 or 2, characterised by first biasing means (23) biasing the plates (12, 13) together about the hinge axis, second biasing means (28) biasing the two plates (12, 13) together in the region of the hinge means (14), and actuating means (31) operable to cause the plates (12, 13) to pivot about the hinge axis against the action of the first biasing means (23) and to move apart at the hinge axis against the action of the second biasing means (28).

4. Apparatus according to Claim 3 as appended to Claim 2, characterised in that the first biasing means (23) comprises a coil spring disposed around the hinge pin (19) and bearing, on the one hand, against part of the one plate (13) or a part (28) carried by the said one plate (13) and, on the other hand, against the other plate (12).

5. Apparatus according to Claim 3 or 4 as appended to Claim 2, characterised in that the second biasing means (28) comprises a leaf spring carried by the one plate (13) and coupled to the hinge pin (19).

6. Apparatus according to Claim 5, characterised in that the coupling is provided by a coil spring (23) disposed around the hinge pin (19) and serving as the first biasing means (23).

7. Apparatus according to any one of Claims 3 to 6, characterised in that the actuating means (31) is a lever pivoted on the one plate (13) and carrying a cam surface (33) which co-operates with a cam follower surface (35) on the other plate (12) to force the plates (12, 13) apart when the lever is operated.

8. Apparatus according to any one of Claims 3 to 7 as appended to Claim 2, characterised in that the relative strengths and configuration of the two biasing means (23, 28) are such that the force required at the actuating means (31) to produce rotation about the hinge axis is less than is necessary to cause displacement of the hinge pin (19) away from the said one plate (13).

9. Apparatus according to Claim 8, characterised by a housing having an abutment (38) which is engaged by the edge of the said other plate (12) remote from the hinge means (19) to limit the rotation of the said other plate (12) about the hinge pin (19).

10. Apparatus according to any one of Claims 3 to 7 as appended to Claim 2, charac-

terised in that the relative strengths and configurations of the two biasing means (23, 28) are such that the force required at the actuating means (31) to produce rotation about the hinge pin (19) is equal to the force required to cause displacement of the hinge pin (19) away from the said one plate (13).

Patentansprüche

1. Münzendurchlauf-Vorrichtung, umfassend eine erste Platte (12) und eine zweite Platte (13), die eine Laufbahn (17) für die Bewegung der Münzen längs dieser bilden, sowie eine die Platten (12, 13) miteinander verbindende Scharniereinrichtung (14), die ein Auseinanderschwenken der Platten gestattet, gekennzeichnet durch eine Einrichtung (21), die eine translatorische Auseinanderbewegung der Platten an der Schwenkachse der Scharniereinrichtung (14) gestattet.

2. Vorrichtung nach Anspruch 1, dadurch gekennzeichnet, daß die Scharniereinrichtung einen Scharnierstift (19) umfaßt, der parallel zu den Platten (12, 13) zur Bewegung in einem generell senkrecht zur Ebene einer (13) der Platten verlaufenden Schlitz (21) montiert ist, wobei die andere Platte (12) um die Achse der Scharnierstifts (19) schwenkbar montiert ist.

3. Vorrichtung nach Anspruch 1 oder 2, gekennzeichnet, durch eine erste Vorspanneinrichtung (23), die die Platten (12, 13) um die Scharnierachse zusammenspannt, eine zweite Vorspanneinrichtung (28), die die beiden Platten (12, 13) im Bereich der Scharniereinrichtung (14) zusammenspannt, und eine Betätigungseinrichtung (31), die bei Beaufschlagung bewirkt, daß die Platten (12, 13) gegen die Wirkung der ersten Vorspanneinrichtung (23) um die Scharnierachse schwenken und sich gegen die Wirkung der zweiten Vorspanneinrichtung (28) an der Scharnierachse auseinanderbewegen.

4. Vorrichtung nach Anspruch 3, soweit auf Anspruch 2 rückbezogen, dadurch gekennzeichnet, daß die erste Vorspanneinrichtung (23) eine Schraubenfeder umfaßt, die um den Scharnierstift (19) herum angeordnet ist und einerseits an einem Teil der einen Platte (13) oder einem von dieser einen Platte (13) getragenen Teil (28) und andererseits an der anderen Platte (12) anliegt.

5. Vorrichtung nach Anspruch 3 oder 4 soweit auf Anspruch 2 rückbezogen, dadurch gekennzeichnet, daß die zweite Vorspanneinrichtung (28) eine von der einen Platte (12) getragene und mit dem Scharnierstift (19) gekuppelte Blattfeder umfaßt.

6. Vorrichtung nach Anspruch 5, dadurch gekennzeichnet, daß die Kupplung durch eine Schraubenfeder (23) vermittelt wird, die um den Scharnierstift (19) herum angeordnet ist und als die erste Vorspanneinrichtung (23) dient.

7. Vorrichtung nach einem der Ansprüche 3



bis 6, dadurch gekennzeichnet, daß die Betätigungseinrichtung (31) ein an der einen Platte (13) angelenkter Hebel ist, der eine Nockenfläche (33) trägt, die mit einer Nockenstößelfläche (35) auf der anderen Platte (12) zusammenarbeitet, um die Platten (12, 13) bei Betätigung des Hebels auseinanderzudrücken.

8. Vorrichtung nach einem der Ansprüche 3 bis 7, soweit auf Anspruch 2 rückbezogen, dadurch gekennzeichnet, daß die relativen Kräfte und die Konfiguration der beiden Vorspanneinrichtungen (23, 28) derart ausgelegt sind, daß die Kraft, die die Betätigungseinrichtung (31) benötigt, um eine Drehung um die Scharnierachse zu erzeugen, geringer ist als diejenige Kraft, die benötigt wird, um eine Verschiebung des Scharnierstifts (19) von der einen Platte (13) weg zu bewirken.

9. Vorrichtung nach Anspruch 8, gekennzeichnet durch ein Gehäuse mit einem Anschlag (38), mit dem die von der Scharniereinrichtung (19) entfernte Kante der anderen Platte (12) in Eingriff tritt, um die Drehung dieser anderen Platte (12) um den Scharnierstift (19) zu begrenzen.

10. Vorrichtung nach einem der Ansprüche 3 bis 7, soweit auf Anspruch 2 rückbezogen, dadurch gekennzeichnet, daß die relativen Kräfte und Konfiguration der beiden Vorspanneinrichtungen (23, 28) so getroffen sind, daß die Kraft, die erforderlich ist, damit die Betätigungseinrichtung (31) eine Drehung um den Scharnierstift (19) erzeugt, gleich ist der Kraft, die benötigt wird, um eine Verschiebung des Scharnierstifts (19) von der besagten einen Platte (13) weg zu bewirken.

Revendications

1. Appareil de guidage de pièces de monnaie comprenant des première (12) et seconde (13) plaques délimitant un couloir (17) le long duquel les pièces se déplacent et des moyens (14) formant charnière reliant entre elles lesdites plaques (12, 13) afin de permettre leur mouvement de pivotement en éloignement l'une de l'autre, caractérisé par des moyens (21) permettant le mouvement de translation desdites plaques en éloignement l'une de l'autre au niveau de l'axe de pivotement des moyens (14) formant charnière.

2. Appareil selon la revendication 1, caractérisé en ce que les moyens formant charnière comprennent un goujon (19) de charnière monté parallèlement aux plaques (12, 13) de façon à pouvoir se déplacer dans une fente (21) qui s'étend dans l'ensemble perpendiculairement au plan d'une (13) des plaques, l'autre plaque étant montée pivotante autour de l'axe du goujon de charnière.

3. Appareil selon la revendication 1 ou 2, caractérisé par des premiers moyens de sollicitation (23) sollicitant les plaques (12, 13) l'une vers l'autre autour de l'axe de charnière,

des seconds moyens de sollicitation (28) sollicitant les deux plaques (12, 13) l'une vers l'autre dans la région des moyens (14) formant charnière et des moyens de commande (31) actionnables pour provoquer le pivotement des plaques (12, 13) autour de l'axe de charnière à l'encontre de l'action des premiers moyens de sollicitation (23) et leur écartement au niveau de l'axe de charnière à l'encontre de l'action des seconds moyens de sollicitation (28).

4. Appareil selon la revendication 3, telle que dépendante de la revendication 2, caractérisé en ce que les premiers moyens de sollicitation (23) comportent un ressort hélicoïdal disposé autour du goujon (19) de charnière et en appui, d'une part, contre une partie de la première plaque (13) ou contre un élément (28) porté par ladite première plaque (13) et, d'autre part, contre l'autre plaque (12).

5. Appareil selon la revendication 3 ou 4, telle que dépendante de la revendication 2, caractérisé en ce que les seconds moyens de sollicitation (28) comprennent une lame-ressort portée par la première plaque (13) et couplée au goujon (19) de charnière.

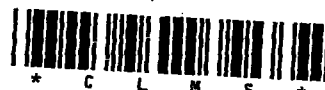
6. Appareil selon la revendication 5, caractérisé en ce que le couplage est effectué par un ressort hélicoïdal (23) disposé autour du goujon (19) de charnière et constituant les premiers moyens de sollicitation (23).

7. Appareil selon l'une quelconque des revendications 3 à 6, caractérisé en ce que les moyens de commande (31) sont constitués par un levier monté pivotant sur la première plaque (13) et portant une surface de came (33) qui coopère avec une surface (35) suiveuse de came formée sur l'autre plaque (12) pour repousser les plaques (12, 13) en écartement l'une de l'autre lorsque le levier est actionné.

8. Appareil selon l'une quelconque des revendications 3 à 7 telles que dépendante de la revendication 2, caractérisé en ce que les forces relatives et les configurations des deux moyens de sollicitation (23, 28) sont telles que la force nécessaire au niveau des moyens de commande (31) pour produire une rotation autour de l'axe de charnière est inférieure à celle qui est nécessaire pour provoquer le déplacement du goujon (19) de charnière en éloignement de ladite première plaque (13).

9. Appareil selon la revendication 8, caractérisé par un boîtier comportant une butée (38) contre laquelle vient en appui le bord de ladite autre plaque (12) éloigné des moyens (19) formant charnière, pour limiter la rotation de ladite autre plaque (12) autour du goujon (19) de charnière.

10. Appareil selon l'une quelconque des revendications 3 à 7, telles que dépendante de la revendication 2, caractérisé en ce que les forces relatives et les configurations des deux moyens de sollicitation (23, 28) sont telles que la force nécessaire au niveau des moyens de commande (31) pour produire une rotation



autour du goujon (19) de charnière est égale à la force nécessaire pour provoquer le déplace-

ment du goujon (19) de charnière en éloignement de ladite première plaque (13).

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