

"Chair"

The invention relates to a chair which comprises a permanent support element, a movement mechanism and a
5 frame.

Disclosed in DE 1 282 262 A1 is a chair comprising a permanent support element which is configured as a seat shell, the rear region of the seat shell being
10 supported in a lumbar region by a centrally arranged double tube.

Disclosed in EP 0 049 310 B1 is a chair comprising a permanent support element, the rear region thereof
15 being supported and cushioned by arms arranged laterally adjacent to the support element.

Disclosed in EP 1 946 676 A1 is a chair comprising a permanent support element which is configured as a seat
20 shell, the rear region being supported and cushioned in a lumbar region by two supporting arms arranged laterally and symmetrically.

Disclosed in EP 2 110 050 A1 is a chair which comprises
25 a permanent support, which comprises two L-shaped support members, which bear a cover, the support being borne in a transition region and a rear region by a rigid rear element which is incorporated in the support.

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It is the object of the invention to develop a chair comprising a permanent, lightweight support element which has enhanced seating comfort and which is of lightweight construction.

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This object is achieved by the characterizing features of Claim 1 based on the features of the preamble of Claim 1. Advantageous and expedient developments are set forth in the sub-claims.

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In the chair according to the invention, the movement mechanism comprises a torsion element which bears the rear part and controls the inclination of the rear part, the rear part of the support element being
10 exclusively connected above the lumbar support to the torsion element. Due to the support of the rear region of the support element via a torsion element and the articulation of the rear region of the support element in an upper half of the rear region on the torsion
15 element, a lower inherent stability of the rear region of the support element is required, as said support element is borne and stabilized in its upper rear region by the torsion element. This is very advantageous, in particular, as torsional forces are
20 introduced by the user precisely in this upper rear region, in particular via their shoulder blades, when the user seated on the chair leans back to the right or to the left, in order to grasp, for example, a file located to the rear. As a result of the close vicinity
25 between the points of force introduction in the region of the shoulder blades of the user seated on the support element of the chair and the point(s) of articulation of the torsion element in the upper half of the rear region of the support means, stresses are
30 substantially prevented from being formed in the rear element, so that said rear element may be configured as an element of lightweight construction. As a result of this lightweight construction which permits the specific arrangement and attachment of the torsion
35 element, it is also possible to ensure the required

relative movement, which occurs when the loading of the chair is altered by the seated user, by means of elastic deformation of the support element between the rear region and the seat region, without having to
5 implement costly technical measures for this purpose.

According to the invention, in the chair, the rear region can be inclined and/or can be twisted in relation to the seat region from an upright seated
10 position into a reclined seated position, a force which is introduced into the support element by a seated person and which causes bending up of the support element and/or rotation of the rear part in relation to the seat part experiencing a counterforce by the
15 movement mechanism, the movement mechanism being connected to the support element in the rear region above a lumbar support, in particular in a central point of articulation or in particular in two lateral points of articulation, and the movement mechanism
20 being connected in particular fixedly to the support element in the seat region, or being connected in particular via two rotational axes arranged in the seat region. By means of such a connection of the support element, which bears the seated person, to the movement
25 mechanism, the requirement of enabling the seated person to rotate his/her body when sitting upright and when reclining is optimally met. The rear part of the chair can optimally follow the rotation of the person's body, since the movement mechanism is designed in the
30 region of the transition part and in the lower half of the rear part in a manner similar to the person's spine as a rotatable structure which lies in the plane of symmetry.

The invention also provides to equip the torsion element with a torsion rod which extends upwards beyond the lumbar support in a vertical plane of symmetry of the chair. As a result, the torsion rod is optimally
5 oriented relative to asymmetrical loads of the rear region of the support element, which may be produced by a rotation of the upper body of a person seated on the chair.

10 According to the invention, the elastic deformation of the support element between the rear region and the seat region is also achieved, in particular, by eliminating a connection of the support element to the movement mechanism in a lower rear region and in a
15 transition region.

According to the invention, the support element comprises, according to a first variant, two curved, L-shaped support members and a cover, the cover being
20 tensioned between the support members. Such a structure results in a simple manner in a lightweight construction of the support element.

The invention also provides to arrange the movement
25 mechanism between the frame and the support element, the movement mechanism bearing both the rear part of the support element and the seat part of the support element, an inclination of the seat part and an inclination of the rear part being controlled by the
30 movement mechanism, depending on the forces to which the chair is subjected by a seated person, the inclination of the seat part and the inclination of the rear part being controlled in a mutually dependent manner and the inclination of the rear part between an
35 initial position and an end position increasing to a

greater extent than the inclination of the seat part. As a result of such a movement mechanism, a particularly high degree of seating comfort is achieved.

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According to a further variant, the invention provides that the support element is configured as a curved, L-shaped seat shell. A seat shell thus configured is able to be produced in a particularly simple manner in terms of production technology, for example as a one-piece injection-molded part.

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In a simple variant the invention provides to fix the support element in its seat region rigidly to the frame and to support by means of the torsion rod just one cantilever arm, which consists of the transition region and the rear region of the support element. Such a chair which has a high degree of seating comfort has a particularly simple movement mechanism and, therefore, may be produced easily and thus cost-effectively.

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Furthermore, the invention provides a parallel extent of the two rotational axes arranged in the seat region, the rotational axes orthogonally penetrating a vertical plane of symmetry of the chair. By mounting the seat part on the rotational axes, the seat part is stabilized by the movement mechanism and defines the movement profile of the seat part.

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According to the invention, the counterforce which is produced by the movement mechanism to compensate for the force introduced by a seated person amounts to at least 50% and in particular at least 70% of a required total counterforce, the remaining counterforce being produced by the deforming support element. By this

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means, the support element is greatly relieved of load and can thus be configured to be correspondingly lightweight and flexible.

5 Furthermore, the invention provides to arrange the movement mechanism between the frame and the support element, the movement mechanism comprising a first rocker, a second rocker and a basic body, the second rocker being articulated rotatably on the basic body
10 and rotatably on a front half of the seat part, the first rocker being articulated rotatably on the basic body, and being articulated rotatably on a rear half of the seat part and comprising the torsion element which is connected to the rear region above a lumbar support.
15 By means of such an articulation and a mirror-symmetrical and rigid configuration of each of the two rockers with respect to the plane of symmetry, the inclination behavior and the torsion behavior of the chair can be realized with few structural elements.

20 Alternatively, the invention also provides to equip a movement mechanism, which is arranged between the frame and the support element, with a bearing element which can in particular be elastically deformed, the bearing
25 element being arranged between the frame and the seat part and fixing the seat part, and the movement mechanism comprising an elastically deformable rocker which comprises the torsion element and is connected to the frame and to the rear region above a lumbar
30 support. By means of such a configuration of the chair, the chair has the desired inclination behavior and the desired torsion behavior even without a movement mechanism articulated by rotational joints.

According to the invention, the support element also comprises two spacer rods, the two support members both being held at a predefined distance by the two spacer rods, which connect free ends of the support members, 5 and being held parallel to each other at said defined distance by the movement mechanism. By means of such a construction of the support element as a closed frame, it is possible to tension the cover thereof with high tensioning forces, the support element also being 10 stiffened by the movement mechanism and the articulation thereof on the bearing element.

According to the invention, it is provided to damp the movement mechanism by at least one spring mechanism or 15 a resilient element. By this means, in particular with an adjustable spring mechanism or an exchangeable resilient element, the inclination and/or torsion behavior of the chair can be adjusted to the person using the chair.

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According to the invention, it is provided to connect the seat part to the movement mechanism by a four-point bearing at four points of articulation, and to connect the rear part to the movement mechanism in particular 25 by a two-point bearing, in particular at support members together with an upper half of the rear part, in particular at two points of articulation. By means of a four-point articulation of the seat part and a rigid coupling of the left and right halves of the 30 movement mechanism, which halves are divided by the plane of symmetry, the seat part is fixed to an inclining movement and an undesirable rolling of the seat part about a roll axis lying the plane of symmetry is prevented. At the points at which the forces are 35 introduced by the chair user, a two-point articulation

of the rear part affords optimum support of the rear part by means of the movement mechanism.

Finally, it is provided to configure the rear part with
5 a first buckling device, a lower section of the rear part, which section adjoins the transition part, and an upper section of the rear part, which section adjoins the lower section, being pivotably connected by the first buckling device, the first buckling device being
10 formed below a region of articulation, in which the torsion element is articulated on the rear part, the torsion element comprising a second buckling device, a lower section of the torsion element, which section is articulated on a basic body of the movement mechanism,
15 and an upper section of the torsion element, which section is articulated on the rear element, being pivotably connected by the second buckling device, the second buckling device being formed above the lumbar support. By the chair according to the invention being
20 supplemented in this manner by two buckling devices, the chair can be supplemented by a further function while retaining the described properties thereof. Said additional function is provided in particular for chairs with a high back rest which reaches into the
25 neck region or into the head region of a person seated on the chair. In this case, as a result of the special arrangement of the buckling devices, the upper section of the rear element tips forwards if a person seated in the chair leans back against the rear part, and thus
30 assists the seated person in maintaining an approximately horizontal viewing axis if the person, for example, wishes to continue to keep a monitor in view even when the person is leaning back.

Further details of the invention are disclosed in the drawings, with reference to exemplary embodiments shown schematically.

5 In the drawings:

10 Figures 1 and 2: show a side view and a rear view of a first variant of a chair according to the invention;

15 Figures 3 and 4: show a side view and a rear view of a second variant of a chair according to the invention;

20 Figures 5 and 6: show a side view and a rear view of a third variant of a chair according to the invention;

25 Figures 7 and 8: show a side view and a rear view of a fourth variant of a chair according to the invention;

30 Figures 9 and 10: show a side view and a rear view of a fifth variant of a chair according to the invention;

35 Figures 11 and 12: show a side view and a rear view of a sixth variant of a chair according to the invention;

- Figures 13-15: each show an oblique view of the first, third and fifth variant;
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Figure 16: shows a perspective illustration of the first variant with deconstructed details;
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Figures 17, 18: show further views of the chair shown in figure 16, with the frame removed;
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Figures 19-21: show modifications of the first variant in two side views and a rear view;
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Figures 22-24: show modifications of the second variant in two side views and a rear view;
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Figures 25-27: show modifications of the third variant in two side views and a rear view;
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Figures 28-30: show modifications of the fourth variant in two side views and a rear view;
- Figures 31-33: show modifications of the fifth variant in two side views and a rear view, and

Figures 34-36: show modifications of the sixth variant in two sides views and a rear view.

5 In Figure 1 a first variant of a chair 1 is shown in side view. The chair 1 comprises a permanent support element 2, a movement mechanism 3 and a frame 4. The support element 2 comprises a seat region 5, a rear region 6 and a transition region 7. In this case, the
10 seat region 5 forms a seat part 8, the rear region 6 a rear part 9 and the transition region 7 a transition part 10 of the support element 2. The movement mechanism 3 comprises a torsion element 11 and two front levers 12 and 13, the second front lever 12 being
15 concealed by the first front lever 12 in the view of Figure 1. The torsion element 11 is configured as a first rocker 14, which is connected about a rotational axis 15 in the manner of a rotary joint to the seat part 8 and about a rotational axis 16 in the manner of
20 a rotary joint to the frame 4. The two front levers 12, 13 are configured as a second rocker 38 as a single component. Furthermore, the torsion element 11 is connected at two fastening points 17 and 18 to the rear part 9. Viewed in the y-direction, the fastening points
25 17 and 18 are located above a lumbar support 19 of the rear part 9 in an upper half 20 of the rear part 9. A central part M11 of the torsion element 11 is configured as a torsion rod 21, which extends in a vertical plane of symmetry 22 of the chair 1. The plane
30 of symmetry 22 is also denoted as the xy-plane, and is located perpendicular to the drawing plane of Figure 2, which shows the chair 1 from the direction of an arrow II provided in Figure 1. The frame 4 is configured as a fixed frame. According to a variant, not shown, it is
35 provided to design the frame 4 as a swivel frame with

rollers in order to use the chair, for example, as an office chair. Between the support element 2 and the frame 3, a schematically shown resilient element 23 is arranged which holds the support element 2 in the resting position R1 shown in Figure 1, and partially compensates for the loading of the support element 2 by a person seated on the support element 2, and permits a limited alteration of the position of the support element 2 relative to the frame 3 with correspondingly high loads. The above-described movement mechanism 3 is designed so that the inclination of the seat part and the inclination of the rear part are dependent on one another and the inclination of the rear part 9 between an initial position 9-1 shown in Figure 1 and an end position 9-2 indicated by dashed lines, increases to a greater extent than the inclination of the seat part 8 between an initial position 8-1 shown in Figure 1 and an end position 8-2 indicated by dashed lines. A relative movement which takes place between the seat part 8 and the rear part 9, when said parts are moved from their initial positions 8-1 and 9-1 into their end positions 8-2 and 9-2, is compensated by a deformation of the transition part 10, so that in spite of the permanent support element 2 a so-called synchronous mechanism 24 is produced in the chair 1. The torsional properties of the torsion element 11 come to bear when the person seated on the chair 1 does not lean centrally against the rear part 9 in the region of the plane of symmetry 22, but loads the rear part laterally, for example at a loading point 25. Such a loading of the support element 2 leads to a deformation of the support element 2, namely to torsion between the seat part 8 and the rear part 9. This torsion of the support element 2 is limited by the torsion element 11 and, in particular, the torsion rod 21, as said torsion

rod creates a counter force, which is introduced via the fastening points 17 and 18 into the rear part 9, to this end the torsion element 11 being supported on the basic frame 4. Such a limited torsion of the support element 2 is possible, irrespective of the inclination of the rear part 9 of the support element 2, in any inclined position between the initial position 9-1 and the end position 9-2. The torsion rod 21 extends in the y-direction upwards beyond the lumbar support 19. The support element of the first variant shown in Figures 1 and 2, comprises two curved, L-shaped support members 26 and 27 and a cover 28 which is tensioned between the support members 26 and 27. The torsion element 11 and the levers 12 and 13 of the movement mechanism 3 in each case act on the support members 26 and/or 27. In the extension of the torsion rod 21, the torsion element 11 is configured as a pair of antler-like projections 29, which divide the torsion rod 21 towards the upper free ends 26a and 27a of the support members 26 and/or 27.

In Figure 13, the chair 1 is additionally shown in an oblique view obliquely from the rear. It can be seen in this view how the two levers 12 and 13 form the second rocker 38 by means of a bow 37 connecting said levers. The movement mechanism 3 therefore essentially comprises the first rocker 14, the second rocker 38 and a basic body 41. Like the first rocker 14, the second rocker 38 has a rotational axis 39 and a rotational axis 40, on four rotational axes 15, 16 and 39, 40 all the movement mechanism 3 running parallel to one another. The first rocker 14 is not only articulated in the region of the rotational axis 15 on the support element 2 but is also connected to the latter by means of the support members 26, 27 thereof at the fastening

points 17, 18. A free rotatability of the rockers 14, 38 about the rotational axes 16, 40 is damped and restricted by a spring mechanism 42 which is incorporated into the basic body 41.

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In Figures 3 and 4, a second variant of a chair 30 is shown in side view and rear view. Relative to this second variant, reference is initially made to the description of the first variant shown in Figures 1 and 2. Accordingly, similar components are denoted by the reference numerals used in Figures 1 and 2. In contrast to the first variant, the movement mechanism 3 essentially comprises just one torsion element 11 which is rigidly fastened between the seat part 8 of the support element 2 and the frame 4. This torsion element 11 comprises, in the same manner as the torsion element of the first variant, a torsion rod 21 which is divided in the manner of a pair of antler-like projections 29, and is fastened to the free ends 26a and 27a of the support members 26 and 27. In addition to its torsional properties, the torsion element 11 is still configured to be flexible and, as a result, cushions the rear part 9 of the support element 2 between the initial position 9-1 and the end position 9-2, to this end the transition region 7 of the support element 2 being elastically deformed. The movement mechanism also comprises a bearing element 48 with which the two support members 26, 27 are held at a predefined distance from one another.

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Figures 5 and 6 show a third variant of a chair 31. Relative to this third variant, reference is initially made to the description of the first variant shown in Figures 1 and 2. Accordingly, similar components are denoted by the reference numerals used in Figures 1 and 2.

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2. In contrast to the first variant, the support element 2 in the third variant has no support members and no cover but a curved, L-shaped seat shell 32 which is molded, for example, from plastics material. The
5 seat shell 32 is elastically deformable, in particular, in its transition region 7. Analogous to Figure 13, Figure 14 shows an oblique view of the chair 31 shown in Figures 5 and 6. Accordingly, reference is made here to the description of Figure 13. The first rocker 14 is
10 configured comparably to the first rocker of the first variant and is connected to the support element 2 at two points of articulation, but the points of articulation 17, 18 lie on the seat shell 32 instead of on support members.

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Figures 7 and 8 show a fourth variant of a chair 33. Relative to this fourth variant, reference is initially made to the description of the first variant shown in Figures 1 and 2. Accordingly, similar components are
20 denoted by the reference numerals used in Figures 1 and 2. In contrast to the first variant, the support element 2 in the fourth variant has no support members and no cover but a curved, L-shaped seat shell 32, which is molded, for example, from plastics material.
25 The seat shell 32 is elastically deformable, in particular, in its transition region 7. Moreover, the movement mechanism 3 of the fourth variant comprises, in contrast to the movement mechanism of the first variant, only one torsion element 11 which is rigidly
30 fastened between the seat part 8 of the support element 2 and the frame 4. This torsion element 11 comprises, in a similar manner to the torsion element of the first variant, a torsion rod 21 which is split in the manner of a pair of antler-like projections 29 and is fastened
35 to the free ends 26a and 27a of the support members 26

and 27. In addition to its torsional properties, the torsion element 11 is still configured to be flexible and, as a result, cushions the rear part 9 of the support element 2 between the initial position 9-1 and the end position 9-2, to this end the transition region 7 of the support element 2 being elastically deformed.

Figures 9 and 10 show a fifth variant of a chair 34. Relative to this fifth variant, reference is initially made to the description of the first variant shown in Figures 1 and 2. Accordingly, similar components are denoted by the reference numerals used in Figures 1 and 2. In contrast to the first variant, the support element 2 in the fifth variant has no support members and no cover but a curved, L-shaped seat shell 32 which is molded, for example, from plastics material. The seat shell 32 is elastically deformable, in particular, in its transition region 7. Moreover, the seat shell 32 is not attached via a pair of antler-like projections of the torsion element 11 but is centrally fastened in the region of the plane of symmetry 22 to an extension 35 of the torsion rod 21. Analogously to Figure 13 and to Figure 14, Figure 15 shows an oblique view of the chair 34 shown in Figures 9 and 10. Accordingly, reference is made here to the description of Figures 13 and 14. In a departure from the first rocker of the first and third variant, the first rocker 14 does not fork in the manner of a pair of antler-like projections but rather merely has a central point of articulation 18 in which the first rocker 14 is connected to the seat shell 32 of the chair 34.

Figures 11 and 12 show a sixth variant of a chair 36. Relative to this sixth variant, reference is initially made to the description of the first variant shown in

Figures 1 and 2. Accordingly, similar components are denoted by the reference numerals used in Figures 1 and 2. In contrast to the first variant, the support element 2 in the sixth variant comprises no support members and no cover but a curved, L-shaped seat shell 32 which is molded, for example, from plastics material. The seat shell 32 is elastically deformable, in particular, in its transition region 7. Moreover, the movement mechanism 3 of the sixth variant, in contrast to the movement mechanism of the first variant, comprises just one torsion element 11 which is fastened rigidly between the seat part 8 of the support element 2 and the frame 4. The seat shell 32 is not attached via a pair of antler-like projections of the torsion element but is centrally fastened in the region of the plane of symmetry 22 to an extension 35 of the torsion rod 21. In addition to its torsional properties, the torsion element 11 is still configured to be flexible, and as a result cushions the rear part 9 of the support element 2 between the initial position 9-1 and the end position 9-2, to this end the transition region 7 of the support element 2 being elastically deformed.

With reference to the schematic illustrations of Figures 1, 2 and 13, Figure 16 illustrates the chair 1 described as the first variant together with structural details, the frame 4 being shown without rollers provided. A cover 28 is also only indicated as a hatched surface in order to obtain clarity. In addition to the support members 26, 27 and the cover 28, the support element 2 also comprises two spacer rods 43, 44 illustrated schematically. The spacer rods 43, 44 hold the two support members 26, 27 at a defined distance a43, a44 at the free ends 26a, 26b and 27a, 27b thereof

(see Figure 18) and thus ensure a high load-bearing capacity of the cover 28. A high load-bearing capacity of the cover 28 is furthermore ensured by the articulation of the support members 26, 27 on the movement mechanism 3. In the seat region 5, the support member 26 is articulated on the first rocker 14 via a first point of articulation A and on the second rocker 38 via a second point of articulation B, which is concealed. In the seat region 5, the support member 27 is articulated on the first rocker 14 via a first point of articulation C and on the second rocker 38 via a second point of articulation D. Furthermore, the fastening points 17, 18, at which the forking torsion element 11 merges into the support members 26, 27, form two further points of articulation E and F. By means of such a four-point bearing of the seat region 5, during dynamic sitting torsion of the seat region 5 or rolling of the seat region 5 about a roll axis WA, which lies in the plane of symmetry 22 and extends in the x direction (see Figure 2), is reliably avoided and it is ensured that the rear part 9 twists in relation to the seat region 5 with corresponding loading by the seated person. By means of an approximately X-shaped geometry of the torsion element 11 and of the first rocker 14, the movement mechanism 3 confers a high degree of stability on the chair 1 while simultaneously permitting a torsion-like twisting between the upper half 20 of the rear part 9 and the seat part 5, the torsion being made possible by elastic deformation of the torsion element 11, the support members 26, 27 and the cover 28. The torsion takes place in particular in the region of the lumbar support 19 and of the transition part 10. By means of the rotatable articulation of the first rocker 14 on the basic body 41 of the movement mechanism 3, with corresponding

loading by the seated person the torsion element 11 permits the torsion-like twisting to be combined with the inclination movement of the rear part 9, which movement is indicated in Figure 2 by the positions 9-1 and 9-2. The seat part 8 is connected in the front half 8a thereof, which is located close to the spacer element 44, to the second rocker 38 and in the rear half 8b thereof, which adjoins the front half 8a at the spacer element 43, to the first rocker 14. By this means, the support members 26, 27 are guided on the basic body 41 by two parallelogram guides P1 and P2 which are arranged in a mirror-inverted manner with respect to the plane of symmetry 22, the parallelogram guides P1, P2 being substantially formed by the two rockers 14 and 38 and operating synchronously.

In Figures 17 and 18 the chair 1 is shown perspectivevly in views from below, the frame, the spacer rods, the cover and the spring mechanism having been omitted from view. Two installation spaces 45, 46 for the spring mechanism (not illustrated) can be seen in the basic body 41. However, the spring mechanism (not illustrated) exclusively influences the inclination behavior of the chair 1. The torsional behavior of the chair 1 is influenced only by the torsion element 11 rather than the spring mechanism. It can be fully seen in Figure 17 how the seat part 8 is suspended on the movement mechanism 3 by means of a four-point bearing 47 via the points of articulation A to D. Furthermore, it can be seen how the rear part 9, which is shown only with the support members 26, 27 and without the cover and spacer rod, is suspended on the movement mechanism 3 by means of a two-point bearing 49 via the points of articulation E and F.

Figure 18 once again denotes the four parallel rotational axes 15, 16, 39 and 40 and the four free ends 26a, 26b and 27a, 27b of the two support members 26, 27.

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With regard to Figures 16 to 18, reference is also made to the description of Figures 1, 2 and 13 with some of the reference numerals mentioned there also being noted in Figures 16 to 18.

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A combining of the inclination movement and torsional movement is possible if the first rocker 14 is of λ -shaped configuration or upside down y-shaped configuration, as shown in the fifth and sixth variant.

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Figures 19 to 36 illustrate modifications of the six variants illustrated in Figures 1 to 12, in two side views and one rear view in each case. With regard to the basic construction and the basic functioning of the modifications shown in Figures 19 to 36, reference is correspondingly made to the description of Figures 1, 2 and 3, 4 and 4, 5 and 5, 6 and 7, 8 and 9, 10 and 11, 12. In contrast to the chairs shown in Figures 1 to 12, in all six chairs shown in Figures 19 to 36, 1a (see Figures 19 to 21), 30a (see Figures 22 to 24), 31a (see Figures 25 to 27), 33a (see Figures 28 to 30), 34a (see Figures 31 to 33) and 36a (see Figures 34 to 36), the rear part 9 is a first buckling device 50 (illustrated symbolically), and the torsion element 11 is a second symbolically illustrated buckling device 51, a lower section 52 of the rear part 9, which section adjoins the transition part 10, and an upper section 53 of the rear part 9, which section adjoins the lower section 52, being pivotably connected by the first buckling

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device 50. In this case, the first buckling device 50 is arranged below a region of articulation 54, in which the torsion element 11 is connected to the rear part 9. The second buckling device 51 formed in the torsion element 11 comprises a lower section 55 of the torsion element 11, which section is connected to a basic body 41 of the movement mechanism 3, and an upper section 56 of the torsion element 11, which section is connected to the rear element 9 in the region of articulation 54, the two sections 55, 56 being connected pivotably by the second buckling device 51. In the region of articulation 54, the upper section 53 of the rear part 9 and the upper section 56 of the torsion element 11 are connected to one another. The second buckling device 51 and therefore also the first buckling device 50 are formed above the lumbar support 19 of the chair 1a, 30a, 31a, 33a, 34a and 36a. In this case, the second buckling device 51 is arranged below the first buckling device 50.

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The first buckling device 50 is preferably configured as a bending zone B50 which permits a type of buckling formation between the lower section 52 and the upper section 53 of the rear part 9 depending on in which position the chair 1a, 30a, 31a, 33a, 34a or 36a is in. In a resting position R1, as shown in Figures 19, 22, 25, 28, 31 and 34, the lower section 52 and the upper section 53 of the rear part 9 have a first angle of aperture α -R1 with respect to one another, the angle of aperture α -R1 lying in the plane of symmetry 22, which has already been described for the individual variants, and being measured between a surface 52a of the lower section 52 in contact with a seated person and a surface 53a of the upper section 53 in contact with a

seated person. The resting position R1 is taken up by the chair 1a, 30a, 31a, 33a, 34a or 36a when said chair is unloaded or when a person sits on the chair in such a manner than the person exerts only a small pressure, if any at all, on the rear element 9 of said chair. In a reclined position R2, as shown in Figures 20, 23, 26, 29, 32 and 35, the lower section 52 and the upper section 53 of the rear part 9 have a second angle of aperture α -R2 with respect to one another, the angle of aperture α -R2 likewise lying in the plane of symmetry 22 mentioned. During the transition from the position R1 into the position R2, the two interacting buckling devices 50 and 51 of the rear part 9 and of the torsion element 11 cause a reduction in the angle of aperture from the value α -R1 to the value α -R2. The position which the two sections 52, 53 of the rear part 9 take up with respect to each other therefore changes in the manner of an easily closing flap. The upper section 53 of the rear part 9 moves forwards relative to the lower section 52 of the rear part 9 in the arrow direction x. That is to say, the upper section 53 of the rear part 9 moves relative to the lower section 52 of the rear part 9 during the inclination movement, which the chair 1a, 30a, 31a, 33a, 34a or 36a executes when a person seated on the chair 1a, 30a, 31a, 33a, 34a, or 36a leans back against the rear part 9, and in the process buckles forwards. Said movement, which is opposed to the inclination movement, which is directed in the arrow directions x' and y', stabilizes the neck and the head of the person seated on the chair 1a, 30a, 31a, 33a, 34a or 36a, depending on the shaping and the size of the upper section 53 of the rear part 9, and makes it possible for the person, in an ergonomically desirable manner, to maintain a viewing axis, which has been

adopted in the position seated upright, during the reclining and in the reclined position, since the upper section 53 of the rear part 9, against the surface 53a of which the head of the person seated on the chair bears, inclines to a smaller extent between the position R1 and the position R2 than the lower section 52 of the rear part 9, against the surface 52a of which the upper body of the person seated on the chair leans. In this case, the second buckling device 51 is arranged below the first buckling device 50 in both positions R1 and R2 of the chair 1a, 30a, 31a, 33a, 34a or 36a.

The second buckling device 51 is preferably likewise configured as a bending zone B51 which permits a type of buckling formation between the lower section 55 and the upper section 56 of the torsion rod 11. Where the buckling formation is controlled by the movement mechanism 3, the part thereof is the buckling device 51. In a resting position R1, as shown in Figures 19, 22, 25, 28, 31 and 34, the lower section 55 and the upper section 56 of the torsion rod 11 have a first angle of aperture β -R1 with respect to one another, the angle of aperture β -R1 lying and being measured in the plane of symmetry 22, which has already been described for the individual variants. In a reclined position R2, as shown in Figures 20, 23, 26, 29, 32 and 35, the lower section 55 and the upper section 56 of the torsion element 11 have a second angle of aperture β -R2 with respect to one another, the angle of aperture β -R2 likewise being measured in the plane of symmetry 22. Owing to the fact that the upper section 56 buckles forwards in the x direction during the change from the position R1 into the position R2, the value of the angle of aperture is reduced from β -R1 to β -R2. The

upper section 56 of the torsion element 11 therefore moves forwards relative to the lower section 55 of the torsion element 11 when a person seated on the chair 1a, 30a, 31a, 33a, 34a, or 36a leans back. The two
5 buckling devices 50 and 51 therefore operate synchronously as a consequence of the connection of the upper sections 53 and 56 thereof in the region of articulation 54 - and therefore a closing movement of the movement mechanism 3, to which the buckling device
10 51 and the upper section 56 of the torsion element 11 belong, is transmitted to the rear part 9 of the support element 2. Correspondingly, upon departing from the position R2 into the position R1, an opening movement of the lower and of the upper sections 55, 56
15 of the movement mechanism 3 takes place again and therefore, as a consequence of the coupling, an opening movement of the lower section 52 and of the upper section 53 of the rear part 9 also takes place again.

20 It is provided for the first buckling device 50 to arrange the latter approximately level with the uppermost thoracic vertebra of a person seated in the chair 1a, 30a, 31a, 33a, 34a or 36a in order optimally to support the neck and head of said person in a
25 reclined seating position. Correspondingly, the rear part in the modifications shown in Figures 19 to 36 is dimensioned such that the upper section 53 of the rear part 9 lies level with a neck region or a neck and head region of a person seated on the chair.

30

In the modifications, which are shown in Figures 19 to 21 and 22 to 24, of the first and second variants, the first buckling device 50 is technically formed in the two support members 26, 27 by respective bending zones
35 B50, configured as an elastic region, or alternatively

by respective joint elements. The cover 28 follows the movement predetermined by the support members 26, 27 without additional adaptation.

5 In the modifications, which are shown in Figures 25 to 27, 28 to 30, 31 to 33 and 34 to 36, of the third to sixth variants, the first buckling device 50 is incorporated into the seat shell 32 forming the support element 2, and extends in a horizontal alignment over
10 the rear part 9. The first buckling device 50 is formed by a bending zone B50, which is configured as an elastic region, or alternatively by a joint element.

In the modifications, which are shown in Figures 19 to
15 21, 22 to 24, 25 to 27 and 28 to 30, of the first to fourth variants, the second buckling device 51 is formed in the torsion element 11, or in the first rocker 14 which forms the torsion element 11, below the pair of antler-like projections 29, into which the
20 torsion rod 21 forks in order to merge in the region of articulation 54 into the support members 26, 27. The second buckling device 51 is formed by a bending zone B51, which is configured as an elastic region, or alternatively as a joint element. The pair of antler-
25 like projections 29 forms the upper section 56 of the torsion element 11. The pair of antler-like projections 29 is connected in particular in a planar manner to the support members 26, 37.

30 In the modifications, which are shown in Figures 31 to 33 and 34 to 36, of the fifth and sixth variants, the second buckling device 51 is formed in the torsion element 11, or in the first rocker 14 which forms the torsion element 11, below the central region of
35 articulation 54, in which the rocker 14 is articulated

on the seat shell 32. The second buckling device 51 is formed by a bending zone B51, which is configured as an elastic region, or alternatively by a joint element.

- 5 The invention is not limited to the exemplary embodiments shown or described. On the contrary, it comprises developments of the invention which lie within the scope of the protected claims.

List of reference numerals:

1	Chair, 1st variant
1a	Chair, modification of 1
2	Support element
3	Movement mechanism
4	Frame
5	Seat region of 2
6	Rear region of 2
7	Transition region of 2
8	Seat part of 2
8a	Front half of 8
8b	Rear half of 8
8-1	Initial position of 8
8-2	End position of 8
9	Rear part of 2
9-1	Initial position of 9
9-2	End position of 9
10	Transition part of 2
11	Torsion element
12, 13	Lever
14	First rocker
15	First rotational axis of 14
16	Second rotational axis of 14
17, 18	Fastening point
19	Lumbar support
20	Upper half of 9
21	Torsion rod of 11
22	Plane of symmetry and/or xy-plane
23	Resilient element
24	Synchronous mechanism
25	Loading point
26, 27	Support member
28	Cover
29	Pair of antler-like projections

30	Chair, 2nd variant
30a	Chair, modification of 30
31	Chair, 3rd variant
31a	Chair, modification of 31
32	Seat shell
33	Chair, 4th variant
33a	Chair, modification of 33
34	Chair, 5th variant
34a	Chair, modification of 34
35	Extension of 21
36	Chair, 6th variant
36a	Chair, modification of 36
37	Bow between 12 and 13
38	Second rocker
39	Rotational axis of 38
40	Rotational axis of 38
41	Basic body of 3
42	Spring mechanism in 41
43, 44	Spacer rod between 26 and 27
45, 46	Installation space for 42
47	Four-point bearing of 5
48	Bearing element for 5
49	Two-point bearing of 9
50	First buckling device of 9
51	Second buckling device of 11
52	Lower section of 9
52a	Surface of 52
53	Upper section of 9
53a	Surface of 53
54	Region of articulation of 11 on 9
55	Lower section of 11
56	Upper section of 11
A - F	Point of articulation of 2
a43	Distance between 26a and 26b

a44	Distance between 27a and 27b
B50	Bending zone
B51	Bending zone
M11	Central part of 11
P1, P2	Parallelogram guide
R1	Resting position of the chair
R2	Reclined position of the chair
WA	Roll axis

x, y, z	Direction
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α -R1	First angle of aperture between 52 and 53
α -R2	Second angle of aperture between 52 and 53
β -R1	First angle of aperture between 55 and 56
β R2	Second angle of aperture between 55 and 56

Claims:

1. Chair (1, 30, 31, 33, 34, 36; 1a, 30a, 31a, 33a, 34a, 36a) comprising:
 - 5 - a permanent support element (2), a movement mechanism (3) and a frame (4),
 - the support element (2) comprising a seat region (5) which forms a seat part (8),
 - the support element (2) comprising a rear region
10 (6) which forms a rear part (9),
 - the support element (2) comprising a transition region (7) which connects the seat region (5) and the rear region (6),
 - the support element (2) compensating a relative
15 movement between the rear region (6) and the seat region (5), which occurs when the loading of the chair (1, 30, 31, 33, 34, 36; 1a, 30a, 31a, 33a, 34a, 36a) is altered by the seated person, by means of elastic deformation,
 - 20 characterized
 - in that the movement mechanism (3) comprises a torsion element (11),
 - the torsion element (11) bearing the rear part (9) and controlling the inclination of the rear part
25 (9),
 - the rear part (9) of the support element (2) being exclusively connected above a lumbar support (19) to the torsion element (11).
- 30 2. Chair (1, 30, 31, 33, 34, 36; 1a, 30a, 31a, 33a, 34a, 36a) according to claim 1, characterized in that
 - the rear region (6) can be inclined and/or can be twisted in relation to the seat region (5) from a seated position,

- a force which is introduced by a seated person into the support element (2) and causes bending up of the support element and/or rotation of the rear part (9) in relation to the seat part (8) experiencing a counterforce by means of the movement mechanism (3),
- the movement mechanism (3) being connected to the support element (2) in the rear region above a lumbar support (19), in particular in a central point of articulation (18) or in particular into two lateral points of articulation (17, 18), and being connected, in particular fixedly, in the seat region (5), or being connected in particular via two rotational axes (15, 39) arranged in the seat region (5).

3. Chair (1, 30, 31, 33, 34, 36; 1a, 30a, 31a, 33a, 34a, 36a) according to at least one of the preceding claims, characterized in that the torsion element (11) comprises a torsion rod (21) which extends upwards beyond the lumbar support (19) in a vertical plane of symmetry (22) of the chair (1, 30, 31, 33, 34, 36; 1a, 30a, 31a, 33a, 34a, 36a).

4. Chair (1, 30, 31, 33, 34, 36; 1a, 30a, 31a, 33a, 34a, 36a) according to at least one of the preceding claims, characterized in that in order to permit the deformation, a connection of the support element (2) to the movement mechanism (3) is avoided in the rear region (6) of the support element (2) below the lumbar support (19) and in the transition region (7) of the support element (2).

5. Chair, (1, 30; 1a, 30a) according to at least one of the preceding claims, characterized in that the

support element (2) comprises two curved, L-shaped support members (26, 27) and a cover (28), the cover (28) being tensioned between the support members (26, 27).

5

6. Chair (1, 30, 31, 33, 34, 36; 1a, 30a, 31a, 33a, 34a, 36a) according to at least one of the preceding claims, characterized in that the movement mechanism (3) is arranged between the frame (4) and the support
10 element (2) and bears both the rear part (9) of the support element (2) and the seat part (8) of the support element (2), an inclination of the seat part (8) and an inclination of the rear part (9) being controlled by the movement mechanism (3), depending on
15 the forces to which the chair (1, 30, 31, 33, 34, 36; 1a, 30a, 31a, 33a, 34a, 36a) is subjected by a seated person, the inclination of the seat part (8) and the inclination of the rear part (9) being controlled in a mutually dependent manner and the inclination of the
20 rear part (9) between an initial position (9-1) and an end position (9-2) increasing to a greater extent than the inclination of the seat part (8).

7. Chair (31, 33, 34, 36; 31a, 33a, 34a, 36a)
25 according to at least one of the preceding claims, characterized in that the support element (2) is configured as a curved, L-shaped seat shell (32).

8. Chair (30, 33, 36; 30a, 33a, 36a) according to at
30 least one of the preceding claims, characterized in that the support element (2) is fixed in its seat region (5) rigidly to the frame (4).

9. Chair (1, 31, 34; 1a, 31a, 34a) according to claim
35 2, characterized in that the two rotational axes (15,

39) arranged in the seat region extend parallel to each other and orthogonally penetrate a vertical plane of symmetry (22) of the chair (1, 31, 34; 1a, 31a, 34a).

5 10. Chair (1, 30, 31, 33, 34, 36; 1a, 30a, 31a, 33a, 34a, 36a) according to claim 2, characterized in that the movement mechanism (3) to compensate for the force initiated by a seated person produces a counterforce which amounts to at least 50% and in particular leaves
10 70% of a required total counterforce, the remaining counterforce being produced by the deforming support element (2).

11. Chair (1, 31, 34; 1a, 31a, 34a) according to at
15 least one of the preceding claims, characterized in that the movement mechanism is arranged between the frame (4) and the support element (2), the movement mechanism (3) comprising a first rocker (14), a second rocker (38) and a basic body (41), the second rocker
20 (38) being articulated rotatably on the basic body (41) and rotatably on a front half (8a) of the seat part (8), the first rocker (14) being articulated rotatably on the basic body (41) and being articulated rotatably on a rear half (8b) of the seat part (8) and comprising
25 the torsion element (11) which is connected to the rear region (6) above a lumbar support (9).

12. Chair (30, 33, 36) according to at least one of the preceding claims 1 to 11, characterized in that the
30 movement mechanism (3) is arranged between the frame (4) and the support element (2), the movement mechanism (3) comprising an in particular elastically deformable bearing element (48) which is arranged between the frame (4) and the seat part (8), and the movement
35 mechanism (3) comprising an elastically deformable

rocker (14) which comprises the torsion element (11) and is connected to the frame (4) and to the rear region (6) above a lumbar support (9).

5 13. Chair (1, 30; 1a, 30a) according to claim 5, characterized in that the support element also comprises two spacer rods (43, 44), the support members (26, 27) both being held at a defined distance by the two spacer rods (43, 44), which connect free ends (26a,
10 26b; 27a, 27b) of the support members (26, 27), and being held at the defined distance by the movement mechanism (3).

14. Chair (1, 30, 31, 33, 34, 36; 1a, 30a, 31a, 33a,
15 34a, 36a) according to one of the preceding claims, characterized in that the movement mechanism is damped by at least one spring mechanism (42) or a resilient element (23).

20 15. Chair (1, 31, 34; 1a, 31a, 34a) according to one of the preceding claims, characterized in that the seat part (8) is connected to the movement mechanism (3) by a four-point bearing (47) at four points of articulation (A, B, C, D), and in that the rear part is
25 connected to the movement mechanism (3), in particular by a two-point bearing (49), in particular at support members (26, 27) together with an upper half (20) of the rear part (9), in particular at two points of articulation (E, F).

30

16. Chair (1a, 30a, 31a, 33a, 34a, 36a) according to at least one of the preceding claims, characterized in that

- the rear part (9) comprises a first buckling
35 device (50),

- the first buckling device (50) pivotably connecting a lower section (52) of the rear part (9), which section adjoins the transition part (10), and an upper section (53) of the rear part (9), which section adjoins the lower section (52),
5
- the first buckling device (50) being arranged below a region of articulation (54), in which the torsion element (11) is connected to the rear part (9),
- 10 - the torsion element (11) comprising a second buckling device (51),
 - the second buckling device (51) pivotably connecting a lower section (55) of the torsion element (11), which section is articulated on a
15 basic body (41) of the movement mechanism (3), and an upper section (56) of the torsion element (11), which section is articulated on the rear element (9),
 - the second buckling device (51) being formed above
20 the lumbar support (19).

17. Chair according to claim 16, characterized in that, during a movement of the chair (1a, 30a, 31a, 33a, 34a, 36a) from a resting position (R1) into a
25 reclined position (R2), the second buckling device (51), by means of a reduction in an angle of aperture (β -R1, β -R2), as measured in a vertical plane of symmetry (22), between the lower section (55) of the torsion element (11) and the upper section (56) of the
30 torsion element (11) forces a reduction in an angle of aperture (α R1, α -R2), likewise as measured in the vertical plane of symmetry (22), between the lower section (52) of the rear part (9) and the upper section (53) of the rear part (9) such that, during the

movement of the chair (1a, 30a, 31a, 33a, 34a, 36a)
from the resting position (R1) into the reclined
position (R2), the upper section (53) of the rear part
(9) is moved forwards in an arrow direction (x)
5 relative to the lower section (52) of the rear part
(9).