

KINETIC MODULAR MACHINE FOR PRODUCING ENERGY FROM FLUID FLOWS.

TECHNICAL FIELD

The present invention concerns the turbines or fluid kinetic floating turbines systems with a single and/or double rotor "open center" (without central shaft and/or hub), in other words the turbines equipped with a floater positioning system, capable to produce electricity from fluid flows, either one or bi directional one, operating at different current speeds.

In particular, the invention involves the turbine, the floating and rotational (around the yaw, roll and pitch axes) system, the machine position with respect to the shore and the flow.

BACKGROUND ART

As known, it is possible to build SintEnergy's turbines for producing electricity from tidal streams. They consist of some kinetic machines with mobile component parts allowing to produce energy full immersed in water and moored to the shore by means of a rope subjected only to a tensile stress driven by a rigid rode (on shore technology). The working principle is quite similar to a kite: the machine is in equilibrium in the water and doesn't change the position during the operation; it is also able to self control its position even when the flow direction and intensity changes, maintaining the rotational plan perpendicular to the flow.

Such a SintEnergy turbine is designed as open center and consists of two counter rotating coaxial rotors, of a single stator (unibody stator solution), two synchronous, independent, built in generators, a central wing (also named positioning wing), installed on the center of the stator, and a floater (buoy).

Each generator consists of two steel rings, one built in to the rotor (rotor steel ring) and the second to the stator (stator steel ring). These rings respectively house the permanent magnets and the coils.

Both rotors, functionally constrained to the stator, are only electrically fully independent. During the operations, when the flows hits the machine, both rotors run inside the stator, together with the generators, producing electricity. The rotation is allowed by a number of balls, at datum gaps among them, along some races made on the flanks of the rotors.

The floater and the central wing allow to partially manage, in an adaptive mode, the machine position with respect to the water surface and the shore.

A disadvantage of the cited conventional technology is the partial transient management related to the startup of the machine and when a change of the flow direction occurs.

Particularly, the above solution, in the author's opinion, doesn't allow to fully manage the machine axes rotations (yaw, roll, pitch) implying any potential fluctuations risky for the "in water" machine stability and equilibrium, as well as the energy production.

More potential troubles can come from:

- the high component elements production costs due to the necessity of complex manufacturing procedures and machines, mainly for the unibody stator manufacturing;
- the stop of the energy production even in case of planned maintenance or unattended failure of just one rotor, due to the fact that both rotors are functionally connected to the stator;
- the use of low aspect central wing, with low aerodynamic efficiency, due to the small central hole;
- the high installation costs, the reduced available shipping ways and manageability of the machine due to the length of the whole anchoring system, as requested to stay off the boundary layer (see S. Barbarelli, G. Florio, M. Amelio, N.M. Scornaienchi, A. Cutrupi, G. Lo Zupone *Transients analysis of a tidal currents self-balancing kinetic turbine with floating stabilizer* Applied Energy 160 (2015) 715–727);
- the high building complexity due to the central deflector, implying a complex device allowing to turn the machine when a flow direction change occurs;
- the machine efficiency is reduced due to turbulences induced by the central wing;
- the low mechanical efficiency during in the operation position (with the swept area in front of the flow) due to the balls bearing, located on the flanks of the rotors: for this reason each rotors weight affect the bearing so it works under cutting loads, increasing the mechanical friction and the cut in speed (the speed at which the machine starts producing energy) and, consequently, reducing the produced energy.

For the above reasons it is necessary to find innovative solutions in order to exceed the said limitations.

DISCLOSURE OF INVENTION

The present invention aims to exceed any previous limitations, already known (closest prior art) in the fluid kinetic machines mainly due to the actual configuration.

The main objective of the present invention, as in the attached claims, is to build a kinetic machine, able to produce energy from fluid flows consisting of a component (module) made of one or more turbines structurally, mechanically and electrically independent, reciprocally connected with screwed systems or pressure/click fixtures, with a built in synchronous generator, in order to reduce the stop of the machine and the loss of energy production when a failure of one or more turbines occurs.

A second objective, depending on the first one, is to build a modular kinetic machine for the energy production from flow currents, assembling a number of component parts, in order to make easier the assembly phases (reducing the number of phases) and off shore maintenance, consequently reducing the risks, the time and the costs of production and management.

A third objective, depending on the previous one, is to make a number of componentized parts (modules), different and “tailor made” depending on the requirements such the site characteristics or the load.

A fourth objective is to supply a machine with characteristics of structural, chemical/physical/mechanical strength ideal with respect to operating environment, ensured by the shape and kind of made of materials, diversified related to the specific purpose.

A fifth objective is the full and right position control of the machine, also during the transients, achievable by a location of the positioning wing outside the turbine and a suitable design and modelling of the floater.

A sixth objective is to increase the produced energy reducing the mechanical friction over the rotors.

A seventh objective is to optimize the produced energy taking in consideration some comparative Computational Fluid Dynamic (CFD) results.

One more objective is to use, for manufacturing and assembly this kinetic machine, all the assembly strategies, parts, manufacturing and devices already known, ie but not the only ones, screws, holdfast (mechanical and electrical), terminal, joining the parts of each machine or between machines, or for the customer interface.

Based on this invention, the innovative fluid kinetic machine allows to better operate converting the the energy from fluid streams, like the tidal or rivers ones, resulting more efficient and profitable compared to the actual technology state of the art, because it consist of:

- one or more turbines, as well as electrically are also structurally and mechanically independent making the machine fully modular;
- a floating/positioning system implementing the control of roll, pitch and yaw as well as the position with respect of the shore and water surface;
- a central hole free of any encumbrance and designed following the CFD results in order to optimize the energy production, to reduce the wake consequences behind the turbine and also the environmental impact.

BRIEF DESCRIPTION OF DRAWINGS

- figure 1 is an axonometric view displaying a machine as per invention;
- figure 2 is an axonometric view of the external bladed turbine (T1), cross sectioned by a diametral plan perpendicular to the rotational plan;
- figure 3 is an axonometric view of the internal bladed turbine (T2), cross sectioned by a diametral plan perpendicular to the rotational plan;
- figure 4 is a side view of the floating/positioning system (F) of the machine;
- figure 5 shows the more profitable purpose of anchoring of the the invention compared to the previous art;
- figure 6 is a frontal view of the double turbine as assembled;
- figure 7 is an axonometric view of a version of a single turbine machine (M) as per invention;
- figure 8 is an axonometric view of the generator (G1), cross sectioned by a diametral plan perpendicular to the rotational plan, built in the turbine (T1);
- figure 9 is an axonometric view of the generator (G21), cross sectioned by a diametral plan perpendicular to the rotational plan, built in the turbine (T2).

BEST MODE FOR CARRYING OUT THE INVENTION

The following descriptions are the minimum instructions whose an expertise person needs to build the machine, consequently any other improvements/modifications can be introduced without any preconceptions to the subject of the subject of the innovation and without vary the related field of protection as defined on the claims.

The machine of the finding, generically defined as (M) in figure 1, is a floating one, that is to say immersed in water but not anchored to the shore bed, just connected to the shore by a system allowing to balance the drag exerted over the machine itself by the flow.

The machine (M) following the invention consists of two coaxial turbines counter rotating, one with external blades (T1) and the other with internal blades (T2); a floating/positioning system; a connection system between machine and shore (figure 5 shows the working principle).

The turbine (T1), as shown in figure 2, consists of a rotor (R1), a stator (S1) and a synchronous generator (G1).

The rotor (R1), round shaped, is composed of four rings (1a, 1b, 1c, 1d). It rotates inside the stator (S1) housing over the external perimeter the blades (5).

The blades (5), of suitable aerodynamic shape and section and at low aspect ratio (less than two), are characterized by a tapered shape with root chord larger or equal than the tip one. The connection blade-rotor is at section with the longer chord.

The stator (S1) is a case torus shaped assembled with four rings (2a, 2b, 2c, 2d).

The generator (G1), shown in figure 8, consists of a metal support ring (14), housed inside the rotor (R1), where are installed a number of permanent magnets (15) and a metal support ring (16), housed inside the stator (S1), where are installed a number of copper coils (17), in the same quantity as the magnets.

The quantity of the rotor blades, the magnets and coils can vary depending on the design purpose and specifications.

The rotation of the rotor (R1) inside the stator (S1) is provided by means of rotating elements (4) of spherical, cylindrical or other suitable shape, order to reduce the mechanical friction between stator and rotor. Such elements, in variable quantity depending on the purpose and design specifications, run along circular races made partially on the rotor, near to the blades root, and partially on the stator (as shown in figure 2); these elements are positioned at the same mutual suitable distance by a spacer (3). As optional configuration, instead of the rotating elements, it is possible, depending on the purpose, it is possible to introduce, or use at the same time, any other devices using other physical phenomena (as example, but not the only one, the magnetic levitation), in order to reduce the friction.

The turbine (T2), as shown in figure 3, consists of a rotor (R2), a stator (S2) and a synchronous generator (G2).

The rotor (R2), round shaped, is composed of four rings (6a, 6b, 6c, 6d). It rotates inside the stator (S2) housing in the internal perimeter the blades (10). It rotates inside the stator (S2) housing in the internal perimeter the blades (10).

The blades (10), of suitable aerodynamic shape and section, are characterized by a tapered shape with root chord smaller or equal than the tip one. The connection blade-rotor is at section with the smaller chord.

The stator (S2) is a case torus shaped assembled with six rings (7a, 7b, 7c, 7d, 7e, 7f).

The generator (G2), shown in figure 9, consists of a metal support ring (18), housed inside the rotor (R2), where are installed a number of permanent magnets (19) and a metal support ring (20), housed inside the stator (S2), where are installed a number of copper coils (21), in the same quantity as the magnets.

The quantity of the rotor blades, the magnets and coils can vary depending on the design purpose and specifications.

The rotation of the rotor (R2) inside the stator (S2) is provided by means of rotating elements (9) of spherical, cylindrical or other suitable shape, order to reduce the mechanical friction between stator and rotor. Such elements, in variable quantity depending on the purpose and design specifications, run along circular races made partially on the rotor, near to the blades root, and partially on the stator (as shown in figure 3); these elements are positioned at the same mutual suitable distance by a spacer (8). As optional configuration, instead of the rotating elements, it is possible, depending on the purpose, it is possible to introduce, or use at the same time, any other devices using other physical phenomena (as example, but not the only one, the magnetic levitation), in order to reduce the friction.

The floating/positioning system (F), as shown in figure 4, consists of a floater (11), a positioning wing (12) and a fixture connecting the machine to the floater (13).

The synergic operation provided by the cited components (11) (12) e (13) allows to control the machine at the datum distance from the the water surface and shore, as per design requirements, as shown in figure 5.

In particular:

- the floater (11), suitably designed and modelled, provides the right depth of the machine and the stability during the transients in order to optimize the energy production (see: S. Barbarelli, G. Florio, M. Amelio, N.M. Scornaienchi, A. Cutrupi, G. Lo Zupone *Transients analysis of a tidal currents self-balancing kinetic turbine with floating stabilizer* Applied Energy 160 (2015) 715–727);

- the positioning wing (12) allows controlling the machine position with regard to the shore (see: Barbarelli S., Amelio M., Castiglione T., Florio G., Scornaienchi N. M., Cutrupi A., Lo Zupone G., *Analysis of the equilibrium conditions of a double rotor turbine prototype designed for the exploitation of the tidal currents*, Energy Conversion and Management, 2014, Vol. 87, pp. 1124-1133 - doi:10.1016/j.egypro.2014.11.1005);
- the connecting fixture machine-floater (13) consists of one or more beams, or any other support structure useful for the purpose, and it is designed for providing the optimal machine depth, at which the maximum suitable flow speed is achieved as required from the design.

The innovative aspects of the present invention, compared to the closest prior art, are:

Modularity

Structural, mechanical and electrical turbine independency, allowing the machine to be fully modular, proves to be the machine itself more profitable in terms of components manufacturing and assembling and also maintenance. Mainly, the modularity concept provides a stop operation time reduction, to pull out one or more fault turbines meanwhile the remaining ones can continue the production even in a lower quantity.

Floating/positioning system

The floating/positioning system (F) covers the control of the roll, pitch and yaw of the machine (M), thanks to the configuration and the innovative features of the system (F). Its configuration, as per invention, is specifically favourable because, as shown in figure 1, the machine (M) acts as a pendulum hinged at roll axis A' and pitch axis B' (depending on the considered oscillation plan); meanwhile the rotational control around the yaw axis C is performed by the floater (11) shape and by the fluid dynamics performances of the positioning wing (12). In particular:

- the floater volume (11), due to the volume of the moved fluid by the immersed machine, provides to avoid the machine sinks; meanwhile, the shape, due to specific fluid dynamic requirements as shown in figures only for example purpose, allows to maintain the right machine position during the small oscillations around the axes A' and B';
- the positioning wing (12), located outside the turbine, can be designed at higher aspect ratios (more than one), in order to operate with higher positioning angles and

linking machine-shore fixture shorter, as shown in figure 5 (cfr. S. Barbarelli, G. Florio, M. Amelio, N.M. Scornaienchi, A. Cutrupi, G. Lo Zupone *Transients analysis of a tidal currents self-balancing kinetic turbine with floating stabilizer* Applied Energy 160 (2015) 715–727);

- the length “l” of the element (13), measured from the rotational axis A of the turbine, fixes the oscillation period of the machine, once known mass and momentum of inertia of the machine itself.

Central hole

The optimal sizing of the central hole is based on a comparative analysis performed by CFD, referring to a conventional turbine “wind like”, with central hub, a turbine open center single rotor, as per invention, and a double rotor turbine, as per invention, demonstrate that the last one is profitable in terms of energy production.

It is in fact known that the performances, in terms of energy production, depend, at same other factors, on the Power Coefficient C_p and the swept area S .

The results show that, by increasing the central hole diameter D_i , as shown in figure 6, maintaining the external diameter D_e of the turbine (T1), an open center turbine provides a C_p higher than a “wind like” (with central hub). Anyway the energy production is lower, in the open center case, due to the fact that the swept area reduces when the central hole diameter increases.

A good settlement, between the advantages taken from the conventional technology and the ones offered from the open center solution, following the obtained results, is provided adopting the open center solution combined with the double rotor configuration, taking the advantages of suitable energy production of the innovative technology (as shown in Giacomo Lo Zupone, Mario Amelio, Silvio Barbarelli, Gaetano Florio, Nino Michele Scornaienchi, Antonino Cutrupi *LCOE evaluation for a tidal kinetic self balancing turbine: Case study and comparison* Applied Energy 185 (2017) 1292–1302), on the current invention is based.

Indeed, this solution allows to achieve, despite a lower swept area, an increase of C_p and energy production compared to the ones of the conventional technology, with more economic advantages (as shown in Giacomo Lo Zupone, Mario Amelio, Silvio Barbarelli, Gaetano Florio, Nino Michele Scornaienchi, Antonino Cutrupi *LCOE evaluation for a tidal kinetic self balancing turbine: Case study and comparison* Applied Energy 185 (2017) 1292–1302).

It is also known that the central hole solution reduces the fauna impact and, as demonstrated by the CFD results, the wake phenomena, behind the machine, are significantly reduced.

Figure 7 depicts one more example of practical execution of the invention involving the principle just displayed. A single turbine configuration is more performing, in terms of Power Coefficient C_p , compared to the double rotor configuration previously considered, and so it is a profitable solution for low cost purposes and/or small users.

Such a solution is mostly profitable for some purposes requiring a number of installed turbines on the same anchoring structure, including floating/positioning systems custom designed.

SEQUENCE LISTING PART OF THE DESCRIPTION:

M	machine
T1	turbine: external bladed
T2	turbine: internal bladed
F	floating/positioning system
A	turbine rotational axis
B	turbine pitch axis
C	yaw axis
A'	machine roll axis
B'	machine pitch axis
1	machine - floater lenght fixture
R1	rotor 1
1a	ring 1
1b	ring 2
1c	ring 3
1d	ring 4
S1	stator 1
2a	ring 1
2b	ring 2
2c	ring 3
2d	ring 4
3	balls spacer
4	balls
5	blades
G1	external bladed turbine generator
14	generator rotor ring
15	magnet
16	generator stator ring
17	coil
R2	rotor 2
6a	ring 1
6b	ring 2
6c	ring 3

6d	ring 4
S2	stator 2
7a	ring 1
7b	ring 2
7c	ring 3
7d	ring 4
7e	ring 5
7f	ring 6
8	balls spacer
9	balls
10	blades
G2	internal bladed turbine generator
18	generator rotor ring
19	magnet
20	generator stator ring
21	coil
11	floater
12	positioning wing
13	fixture link machine-floater
22	prior art
23	new finding subject of present invention
24	anchoring element length of prior art
25	anchoring element length of new finding
26	anchoring base
27	boundary layer
28	shore
Di	internal turbine diameter
De	external turbine diameter