

Description

The invention relates to a dry-shaving apparatus comprising a housing, a holder for a shear plate, which holder is removable from the housing, and a cutter which is rotatable relative to the shear plate and which comprises a carrier with cutting elements which are movable in a substantially radial direction relative to the carrier and which comprise cutting edges at their radial ends, said ends being in contact with the shear plate during a part of a revolution of the cutter and being clear of other parts of the apparatus during another part of the revolution.

Such a dry-shaving apparatus is known, for example from US-A 3,710,442. During a revolution of the cutter each cutting element will rather abruptly come into contact with the shear plate and briefly afterwards this contact will cease. In general this gives rise to undesired vibrations in the apparatus, which may result in damage to said cutter, to the shear plate and to other parts of the apparatus. Therefore, it is important that the shear plate has zones of such a shape that a most uniform "landing" and "departure" of the cutting elements is assured. Between these two zones the shear plate comprises a portion with hair-entry apertures for the shaving function, which portion should also meet specific requirements with respect to its shape. However, the shear plate is generally constructed as a flexible perforated metal foil which is detachably secured to the holder at a few points. When such a construction is used it is difficult to meet said requirements imposed on the shear plate with respect to its shape.

Moreover, the shear plate can cover a circumferential angle of maximum 180°, because otherwise the holder with the shear plate is no longer detachable from the cutter. This also imposes restraints on the shape of the shear plate.

It is the object of the invention to mitigate said problems and to this end the invention is characterized in that the housing is provided with a guide wall which adjoins the shear plate and with which an end of a cutting element is also in contact during a part of a revolution of the cutter.

Special embodiments are defined in the appended subsidiary claims.

Some embodiments of the invention will now be described in more detail, by way of example, with reference to the Figures.

Figure 1 is a diagrammatic longitudinal sectional view of a dry-shaving apparatus in accordance with an embodiment of the invention.

Figure 2 is a sectional view taken on the line II-II in Figure 1.

Figure 3 is a simplified enlarged-scale sectional view similar to that shown in Figure 2.

Figure 4 is a simplified enlarged-scale sectional view of another embodiment, taken in the same way as that in Figure 2.

The dry-shaving apparatus shown in Figures 1 and 2 comprises a housing 1 with a detachable holder 2 for a shear plate 3 and a cutter 4 which is rotatable relative to the shear plate.

The shear plate 3 comprises a central portion 5 formed with hair entry apertures 6 and also comprises a first peripheral portion 7 and a second peripheral portion 8 which adjoin the central portion 5 and by which the shear plate is attached to the holder 2.

The cutter 4 comprises cutting elements 9 having cutting edges 11 at their radial ends 10. The cutter 4 comprises a shaft 12 which is rotatably supported in the housing and which is partly surrounded by the shear plate 3. During rotation of the cutter 4 hairs which project inwards through the hair-entry apertures 6 can be severed by cooperation between the portion 5 of the shear plate 3 and the cutting edges 11 on the ends 10 of the cutting elements 9 which slide over the inner surface 5' of the portion 5.

The cutter 4 comprises a carrier 13 for the cutting elements 9, which carrier comprises a hub 14 and a cylindrical portion 15 with slots 16. The hub 14 is secured to the shaft 12. The cutting elements 9 are situated partly within the slots 16 and are radially movable over a limited distance relative to the carrier. The hook-shaped ends 17 of the cutting elements are situated between the hub 14 and the cylindrical portion 15 to prevent the cutting elements 9 from falling out the carrier 13. Compression springs 18 are arranged between the hub 14 and the cutting elements 9 to exert outwardly directed radial forces on the cutting elements 9.

The housing 1 accommodates an electric motor 19 for driving the cutter 4, for example in a direction of rotation as indicated by the arrow P (Figure 2). The rotation of the motor 19 is transmitted to the cutter 4 by means of pulleys 20 and 21 on the shaft 12 and on the motor shaft 22 respectively and a drive belt 23.

During a part of the revolution of the cutter 4 the end 10 of a cutting element 9 will be clear of other parts of the apparatus. This situation obtains, for example, for the cutting elements 9, which are situated at a level below the shaft 12 in Figures 1 and 2.

The compression springs 18 will urge such a cutting element 9 as far as possible outwards relative to the carrier 13, causing the hook-shaped ends 17 to engage against the cylindrical portion 15. As the cutter 4 rotates about the axis of rotation 12' the free path which in this situation is followed by the cutting edge 11 on the end 10 of the cutting element may be defined as a cylindrical surface which in a sectional view as shown in Figure 2 may be represented as an arc of circle 24 whose centre is situated on the axis of rotation 12' (Figure 3). When the free path is followed there will be no frictional losses between the ends of the cutting elements and other parts of the apparatus, which is an advantage in particular in view of a minimal power consumption of the apparatus.

Figure 3 is a sectional view which corresponds to that of Figure 2 and which diagrammatically and to an enlarged scale shows a part of the housing 1, the holder 2 and the shear plate 3. Two cutting elements 9 are shown in broken lines. During rotation of the cutter 4 the end 10 of a cutting element 9 contacts the first peripheral portion 7 of the shear plate 3 at point 25. Subsequently, the cutting edge 11 of this end 10 will follow a constrained path 26, which changes into the free path 24 at point 27.

A part of the constrained path of a cutting edge 11 is defined by the central portion 5 of the shear plate 3 and if this portion of the shear plate is curved in conformity with a circularly cylindrical surface it may be represented as a second arc of circle 28 between points 29 and 30. If the centres of the arcs 24 and 28 coincide in M_1 on the axis of rotation 12' but the radius R_1 of the arc of circle 24 is larger than the radius R_2 of the arc of circle 28, the springs 18 will be compressed over a constant length as a cutting edge 11 follows the part 28 of the constrained path 26. As this part 28 is followed the cutting elements 9 are consequently urged against the inner side of the shear plate with a constant pressure.

The sudden contact between an end 10 of a cutting element 9 and the shear plate 3 at point 25 may be regarded as a collision, producing vibrations which may impair a correct performance of the apparatus and may give rise to damage. These collisions may also give rise to substantial forces between the shear plate and the cutting element, which may also give rise to damage. These large forces may occur in particular if the cutting elements 9 are mounted in the carrier 13 in a self-biasing or self-locking manner, which are techniques commonly used in dry-shaving apparatuses. Therefore, it is essential to provide a most uniform transition from the free path to the constrained path for a cutting element, so as to minimize said collision effects and the adverse consequences thereof.

At the transition from the free path 24 to the constrained path 26 and *vice versa* a cutting element is subjected to a displacement in a radial direction as a result of the different diameters of these paths, which gives rise to additional inertial forces, referred to as Coriolis forces, being exerted on the cutting elements. These Coriolis forces disturb the uniform motion of the cutter and hence the correct performance of the apparatus.

For a uniform transition from the free path to the constrained path and for minimal Coriolis forces it is important that the transitional portions between the points 25 and 29 and between points 27 and 30 have an adequate length.

In the embodiment shown in Figure 3 the housing comprises a guide wall 31 which adjoins the shear plate 3 and with which the end of a cutting element is also in contact during part of a revolution of the cutter. This guide wall 31 forms part of a portion 32 of the housing 1 and defines a part of the transitional portion 26' between the points 27 and 30.

In order to be detachable the shear plate 3 should not enclose the cutter 4 over a circumferential angle of more than 180°. This limits the dimensions of the shear plate in the direction of driving P, which restricts the dimensions of the constrained path defined by the shear plate. However, this restraint is overcome by means of the guide wall 31 which is constructed as part of the portion 32 of the housing 1 which adjoins the detachable holder 2. Moreover, the manufacturing technologies adopted for the housing, which is generally made of a plastics, provide ample possibility of optimizing the shape of the guide wall 31 and hence of the portion of the con-

strained path defined by this guide wall. For example, in the sectional view of Figure 3, the guide wall 31 between point 27 and point 31 where the guide wall adjoins the shear plate 3, may be shaped as an arc of circle extending through points 27 and 33 and having a radius of a magnitude between R_1 and R_2 .

In this embodiment the larger part of the shear plate 3 is available for defining the transitional portion 26' between points 25 and 29 and for the central portion 5 with the hair-entry apertures 6.

Figure 4 shows an embodiment comprising the same parts as the embodiment in Figure 3 and comprising a second guide wall 34 forming part of the portion 35 of the housing. The guide walls 31 and 34 are situated on opposite sides of the cutter and both adjoin the shear plate 3. The guide wall 34 defines a part of the transitional portion 26' between the points 25 and 29 of the constrained path. The advantages described for the guide wall 31, *i.e.* the possibility of constructing the guide wall independently of the shear plate 3 with respect to its shape and dimensions also apply to the guide wall 34. It is also possible to select other types of material for the guide walls than the material of the housing, for example by providing the housing portions 32 and 35 locally with a sheet or foil material. In this way the material for the guide walls 31 and 34 may be chosen so as to be impact-resistant, to produce low frictional losses, and to cause minimal wear.

Claims

1. A dry-shaving apparatus comprising a housing (1), a holder (2) for a shear plate (3), which holder (2) is removable from the housing, and a cutter (4) which is rotatable relative to the shear plate (3) and which comprises a carrier (13) with cutting elements (9) which are movable in a substantially radial direction relative to the carrier and which comprise cutting edges (11) at their radial ends (10), said ends being in contact with the shear plate (3) during a part of a revolution of the cutter and being clear of other parts of the apparatus during another part of the revolution, characterized in that the housing (1) is provided with a guide wall (31, 34) which adjoins the shear plate (3) and with which an end of a cutting element is also in contact during a part of a revolution of the cutter.

2. A dry-shaving apparatus as claimed in claim 1, characterized in that on opposite sides of the drivable cutter (4) the housing comprises guide walls (31, 34) which adjoin the shear plate (3).

3. A dry-shaving apparatus as claimed in claim 1 or 2, characterized in that a guide wall (31, 34) is made of another material than the shear plate (3).

Patentansprüche

1. Trockenrasiergerät mit einem Gehäuse (1), einer Halterung (2) für eine Scherplatte (3), wobei die Halterung (2) von dem Gehäuse entfernbar ist, und mit einem Schneidelement (4), das gegenüber der Scherplatte (3) drehbar ist und einen Träger (13) aufweist mit Meißeln (9), die in einer im wesentlichen radialen Richtung gegenüber dem Träger beweglich

A part of the constrained path of a cutting edge 11 is defined by the central portion 5 of the shear plate 3 and if this portion of the shear plate is curved in conformity with a circularly cylindrical surface it may be represented as a second arc of circle 28 between points 29 and 30. If the centres of the arcs 24 and 28 coincide in M_1 on the axis of rotation 12' but the radius R_1 of the arc of circle 24 is larger than the radius R_2 of the arc of circle 28, the springs 18 will be compressed over a constant length as a cutting edge 11 follows the part 28 of the constrained path 26. As this part 28 is followed the cutting elements 9 are consequently urged against the inner side of the shear plate with a constant pressure.

The sudden contact between an end 10 of a cutting element 9 and the shear plate 3 at point 25 may be regarded as a collision, producing vibrations which may impair a correct performance of the apparatus and may give rise to damage. These collisions may also give rise to substantial forces between the shear plate and the cutting element, which may also give rise to damage. These large forces may occur in particular if the cutting elements 9 are mounted in the carrier 13 in a self-biasing or self-locking manner, which are techniques commonly used in dry-shaving apparatuses. Therefore, it is essential to provide a most uniform transition from the free path to the constrained path for a cutting element, so as to minimize said collision effects and the adverse consequences thereof.

At the transition from the free path 24 to the constrained path 26 and *vice versa* a cutting element is subjected to a displacement in a radial direction as a result of the different diameters of these paths, which gives rise to additional inertial forces, referred to as Coriolis forces, being exerted on the cutting elements. These Coriolis forces disturb the uniform motion of the cutter and hence the correct performance of the apparatus.

For a uniform transition from the free path to the constrained path and for minimal Coriolis forces it is important that the transitional portions between the points 25 and 29 and between points 27 and 30 have an adequate length.

In the embodiment shown in Figure 3 the housing comprises a guide wall 31 which adjoins the shear plate 3 and with which the end of a cutting element is also in contact during part of a revolution of the cutter. This guide wall 31 forms part of a portion 32 of the housing 1 and defines a part of the transitional portion 26' between the points 27 and 30.

In order to be detachable the shear plate 3 should not enclose the cutter 4 over a circumferential angle of more than 180°. This limits the dimensions of the shear plate in the direction of driving P, which restricts the dimensions of the constrained path defined by the shear plate. However, this restraint is overcome by means of the guide wall 31 which is constructed as part of the portion 32 of the housing 1 which adjoins the detachable holder 2. Moreover, the manufacturing technologies adopted for the housing, which is generally made of a plastics, provide ample possibility of optimizing the shape of the guide wall 31 and hence of the portion of the con-

strained path defined by this guide wall. For example, in the sectional view of Figure 3, the guide wall 31 between point 27 and point 31 where the guide wall adjoins the shear plate 3, may be shaped as an arc of circle extending through points 27 and 33 and having a radius of a magnitude between R_1 and R_2 .

In this embodiment the larger part of the shear plate 3 is available for defining the transitional portion 26' between points 25 and 29 and for the central portion 5 with the hair-entry apertures 6.

Figure 4 shows an embodiment comprising the same parts as the embodiment in Figure 3 and comprising a second guide wall 34 forming part of the portion 35 of the housing. The guide walls 31 and 34 are situated on opposite sides of the cutter and both adjoin the shear plate 3. The guide wall 34 defines a part of the transitional portion 26' between the points 25 and 29 of the constrained path. The advantages described for the guide wall 31, *i.e.* the possibility of constructing the guide wall independently of the shear plate 3 with respect to its shape and dimensions also apply to the guide wall 34. It is also possible to select other types of material for the guide walls than the material of the housing, for example by providing the housing portions 32 and 35 locally with a sheet or foil material. In this way the material for the guide walls 31 and 34 may be chosen so as to be impact-resistant, to produce low frictional losses, and to cause minimal wear.

Claims

1. A dry-shaving apparatus comprising a housing (1), a holder (2) for a shear plate (3), which holder (2) is removable from the housing, and a cutter (4) which is rotatable relative to the shear plate (3) and which comprises a carrier (13) with cutting elements (9) which are movable in a substantially radial direction relative to the carrier and which comprise cutting edges (11) at their radial ends (10), said ends being in contact with the shear plate (3) during a part of a revolution of the cutter and being clear of other parts of the apparatus during another part of the revolution, characterized in that the housing (1) is provided with a guide wall (31, 34) which adjoins the shear plate (3) and with which an end of a cutting element is also in contact during a part of a revolution of the cutter.

2. A dry-shaving apparatus as claimed in claim 1, characterized in that on opposite sides of the driveable cutter (4) the housing comprises guide walls (31, 34) which adjoin the shear plate (3).

3. A dry-shaving apparatus as claimed in claim 1 or 2, characterized in that a guide wall (31, 34) is made of another material than the shear plate (3).

Patentansprüche

1. Trockenrasiergerät mit einem Gehäuse (1), einer Halterung (2) für eine Scherplatte (3), wobei die Halterung (2) von dem Gehäuse entfernbar ist, und mit einem Schneidelement (4), das gegenüber der Scherplatte (3) drehbar ist und einen Träger (13) aufweist mit Meißeln (9), die in einer im wesentlichen radialen Richtung gegenüber dem Träger beweglich



sind und die Schneidkanten (11) an ihren radialen Enden (10) aufweisen, wobei diese Enden während eines Teils einer Umdrehung des Schneidelementes die Scherplatte (3) berühren und während eines anderen Teils der Umdrehung andere Teile des Gerätes nicht berühren, dadurch gekennzeichnet, daß das Gehäuse (1) mit einer Führungswand (31, 34) versehen ist, die an die Scherplatte (3) grenzt und die während eines Teils einer Umdrehung des Schneidelementes ebenfalls ein Ende eines Meißels berührt.

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2. Trockenrasiergerät nach Anspruch 1, dadurch gekennzeichnet, daß das Gehäuse an einander gegenüberliegenden Seiten des antreibbaren Schneidelementes (4) an die Scherplatte (3) grenzende Führungswände (31, 34) aufweist.

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3. Trockenrasiergerät nach Anspruch 1 oder 2, dadurch gekennzeichnet, daß eine Führungswand (31, 34) aus einem anderen Material hergestellt ist als die Scherplatte (3).

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Revendications

1. Rasoir à sec comportant un boîtier (1), un support (2) servant à supporter un peigne (3) et pouvant être enlevé du boîtier, et un organe de coupe (4) pouvant tourner par rapport au peigne (3) et comportant un support, (13) muni d'éléments de coupe (9) pouvant être déplacés dans une direction sensiblement radiale par rapport à ce support et présentant des tranchants (11) à leurs extrémités radiales (10), lesdites extrémités étant en contact avec le peigne (3) pendant une partie d'un tour de l'organe de coupe et étant dégagées d'autres parties du rasoir pendant une autre partie du tour, caractérisé en ce que le boîtier (1) présente une paroi de guidage (31, 34) contiguë au peigne (3) et avec laquelle une extrémité d'un élément de coupe est également en contact pendant une partie d'un tour de l'organe de coupe.

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2. Rasoir à sec selon la revendication 1, caractérisé en ce que, sur des faces opposées de l'organe de coupe commandable (4), le boîtier présente des parois de guidage (31, 34) contiguës au peigne (3).

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3. Rasoir à sec selon la revendication 1 ou 2, caractérisé en ce qu'une paroi de guidage (31, 34) est réalisée à partir d'un matériau autre que celui du peigne (3).

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