

Description

This invention relates to an arrangement for lifting and lowering or for pulling loads according to the preamble of claim 1.

Such an arrangement is known from BE 833 841 and comprises two rope-supporting elements which are formed and positioned such that they engage with their flanks the corresponding flanks of the sheave halves. In consequence of this, one sheave half tilts, as it were, with respect to the other sheave half so that the rope is clamped at the opposite side between the two sheave halves.

A drawback of this arrangement consists in that the rope-supporting elements, formed by little wheels, are pressed in the groove and, as a consequence of the different speed at different radial distances, the flanks of the rope-supporting elements rub against the flanks of the sheave-halves causing a considerable wear of the device and reducing its service life.

Another arrangement for lifting and lowering is shown in AT 358 771 and comprises a sheave provided with a one-piece groove.

Slippage of the rope in the groove is not prevented by biasing two sheave halves towards each other but by winding three windings of the rope in the groove and by rollers pressing the windings on the bottom of the groove.

During the winding and unwinding of the rope in the groove, the windings are pushed axially in the groove by the new winding so that, to allow this displacement, the friction of the rope on the bottom of the groove may not be too large so that slippage, especially in the longitudinal direction of the rope, can still occur.

It is clear that such a device can only produce limited wire rope tensions and further that it is subject to serious wear in operation.

In German Patent Specification (DE-PS) 10 32 506, both of the sheave halves are non-spring-loaded pressed together. Their relative adjustment is only possible in one direction during operation. It means that a displacement of one sheave half relatively to the other may be practicable from a larger distance to a smaller one, but not conversely. Consequently, there is a shortcoming in that a thicker rope section following a thinner rope section cannot be received by both sheave halves any longer without an impermissible strong deformation of the rope (pinching, tangling), which shortcoming is absolutely usual in practice.

Another disadvantage is that one sheave half is adjustable by means of a thread. As a result, in a displacement not only lateral forces are exerted on the rope, but also frictional forces in the longitudinal direction thereof, as in a relative displacement the sheave halves are twisted in opposite directions.

It has to be remarked in addition to the mentioned disadvantages that upon inserting the rope, the free terminal rope section has to be threaded by hand, i.e. moved in, almost over the whole periphery of the traction sheave. Therefore the

rope groove should have a section which is as much greater than the section of the rope that inserting be possible without relatively large resistance by frictional forces or hooking in the free rope end.

Another shortcoming of the known construction is the use of only one rope idler pulley in the inlet area of the rope. So, this only idler pulley too effects the application of the rope to the sheave halves. This is associated with an extremely high specific surface pressure which is detrimental to the service life of the rope.

Further, seat grooves are known in which the ropes are pressed down radially with the help of pressure means.

Besides there are the so-called V-shaped grooves wherein clamping of the ropes mostly takes place by counter-weights at the unloaded end of the rope.

The published German Patent Application (DE-OS) 20 54 110 discloses several rope-guiding rollers at the circumference of a traction sheave. This traction sheave is made of only one piece and the rope guide rollers are not fixed but radially freely supported with respect to the traction sheave.

From the UK Patent Specification (GB-PS) 10 35 625 too only such rope rollers at the circumference of traction sheave are known which are spring-loaded. No idler pulleys are present along the entire circumference of the traction sheave. The section of the rope groove is not changeable either.

All of the one-piece traction-sheave embodiments with seat grooves and pressure means in radial direction have the disadvantage that the pressure means cannot revolve in the same direction. Consequently, the efficiency is always impaired. Moreover, high loads are placed on these constructions with secondary forces, which constructions, as a result, are expensive and are provided with a corresponding excess weight.

In the one-piece traction-sheave embodiments with V-shaped rope grooves and radially acting pressure means, in addition to the above-mentioned disadvantages of traction sheaves with seat grooves, there is the decrease of drive capacity of the rope that is associated with the wear of V-shaped rope grooves which even can result in slipping. These circumstances also apply to the use of V-shaped grooves with counterweights at the unloaded rope section. This construction has the additional disadvantage of limited possibilities of use, since generally they can only be used permanently and not temporarily.

Another disadvantage of the arrangements with a traction sheave is a reduction of the wrapping angle of the rope. The fact is that said reduction results in an increase of the necessary clamping forces, as the required deflection of the rope for passing the just entering or leaving tensioned section of the rope cannot take place already in the groove bottom, but only after leaving the traction sheave at the outer diameter thereof.

It is the object of the present invention, in an

arrangement of the type described in the preamble of claim 1, to avoid unnecessary loads in the arrangement, to reduce wear as much as possible, to bring the weight as well as the dimensions to a minimum and to bring the efficiency of the arrangement to a maximum.

This object is realized according to the present invention by the features mentioned in the characteristic part of claim 1.

Some of these features are known per se from GB-A-2 095 202 which shows an arrangement for lifting and lowering loads with a traction sheave provided with a one-piece V-groove.

The rope is pressed into this V-groove by rollers of a chain which are not at a fixed radial distance with respect to the axis of the sheave. The radial distance between the bottom of the groove and the rollers is however larger than the diameter of the rope.

Further the width of the rollers is larger than the width of the groove so that the rollers do not protrude in the groove but press against the rope which is only partly submerged in the groove.

The combinations of the features or characteristics from the preamble and the characteristic part of claim 1 guarantees a simple and non-sophisticated construction, which increases the safety and simplifies the use of the arrangement by requiring no maintenance. In a sense, a combination of clamping-, V-shaped- and seat-rope grooves is created, wherein the properties of the embodiments are completed and the faults are cleared away. Wire-supporting elements are now being provided and distributed over the whole periphery which distribute the surface pressure in the one as well as in the other operating direction of the arrangement and thereby guide the rope more gently.

The spring-loaded relative position of the sheave halves allows to deal with varying rope sections. If there is no rope in the arrangement, the sheave halves touch one another and the section which is formed by the rope groove in connection with the rope-supporting elements is smaller than the section of a rope to be threaded. Upon threading the rope, the rigid sheave halves now serve as abutment. When the rope is introduced, the movable sheave halves are laterally forced off so that the rope obtains a perfect position in the rope groove, not only in any service conditions but also with any occurring rope diameters in the course of a rope.

Because the rope-supporting elements project into the rope groove, the sheave halves guide the rope faultlessly. This means that in view of the desired rope contact ropes of varying thickness always occupy the optimal relative position.

By applying rope-supporting elements over the whole wrapping area it is guaranteed that upon threading a rope in the rope-groove, a drive action is already reached immediately after threading so that the further threading operation can be carried out comparatively smoothly.

A preferred embodiment of the invention consists in the characteristics of claim 3. Therein only

supporting rollers can be provided over the whole wrapping area. Their common distance corresponds to about half the diameter of a supporting roller. It is also conceivable, however, that at least in the inlet range of the slack rope, a shoe in the form of a circular ring segment is applied instead of supporting rollers. Said shoe optionally can extend over the whole circumference of the traction sheave. It can be formed of one or several pieces. Preferably, the material of the shoe is a wear-resistant plastic. Because of the great surface area of such a shoe, the surface pressure of the entering rope can be reduced still further.

The use of shoes is especially advantageous if cross-laid ropes are being utilized, as in this type of ropes the separate rope threads run parallel to the longitudinal axis of the rope.

On the contrary, if equal-laid ropes are being used, whose rope threads run at an angle to the longitudinal axis of the rope, preferably supporting rollers are applied.

Therein a construction according to the features of claim 4 is advantageously being used. A further reduction of surface pressure is associated therewith.

In the embodiment according to the features of claim 5, it is the primary object of the roller with greater diameter to reduce the surface pressure exerted on the rope when the latter is threaded in slack state or when loads are lowered. This is connected with the advantage that the rope experiences a smaller stress and hence, that a longer service life may be expected.

The large supporting roller, however, can take over another function if the winch is suspended fixedly. In this case the point of suspension finds itself above the rotation axis of the traction sheave. Now, the supporting roller not only serves as an insertion supporting element for the rope, but also as tail element for the slack rope.

In order to improve the inlet procedure, especially of the unloaded rope section, in the traction sheave as well as the outlet procedure of the rope from the traction sheave in the case of small outer diameters of said traction sheaves, the features of claim 6 are efficient.

In order to bring the wrapping angle of the rope to a maximum for reducing the axial clamping force, the features of claim 7 are provided.

For allowing the insertion of ropes of different strength and diameter, the features of claim 8 can be applied. The gradual construction can be provided at one groove side or at both.

In order to achieve a faultless penetration of the stressed or slack rope into the rope groove, the features of claim 9 are significant.

For bringing the efficiency and the economy of the arrangement to a maximum it is appropriate that according to claim 10 the rope-supporting rollers be needle bearings, roller bearings or ball bearings, particularly, however, grooved ball bearings. They may be commercially available products. The bearings may also be made at least partly of resilient materials.

In order to optimize mounting and dismount

ing, the features of claim 11 are especially favourable.

In view of the possibility of adjusting the rope-supporting rollers in the optimal mid-position in the rope groove above the rope after the inlet of the latter, it is suited that the rope-supporting rollers are supported axially movable according to claim 12.

This is achieved particularly and still more advantageously by the features of claim 13. The shoes too can be supported that way.

In order to follow the right running direction of the rope it is significant, particularly with narrow rope-supporting rollers, that the features of claim 14 be applied.

The invention is more clearly illustrated herein-after by means of the examples represented in the accompanying drawings wherein:

Figure 1 represents a view of an arrangement for lifting and lowering loads, especially for pulling a wire rope, partly in vertical longitudinal section;

Figure 2 represents a section through the arrangement of Figure 1 according to line II—II;

Figure 3 represents an enlargement of the cut-out view III of Figure 1 according to a first embodiment;

Figures 4 and 5 represent the same cut-out view according to two other embodiments;

Figure 6 is a section according to the representation of Figure 2 according to a fourth embodiment;

Figure 7 is a section according to the representation of Figure 2 according to a fifth embodiment, and

Figure 8 is an enlarged representation of a section according to line VIII—VIII of Figure 7.

A two-piece traction sheave 1 with V-shaped rope groove 2 of 20° groove angle (Figures 1, 3 or 5) or with two-stage rope groove 3 (Figure 4) is laterally pressed together in axial direction. Therefore rotating springs 4 are used, which are supported in guiding sheaves 5 and pre-stressed by screws 6. Said screws are countersunk with their heads in an axially non-displaceable sheave half 7 of the traction sheave 1 (Figures 1 and 3 to 5).

For torque transfer, sheave half 7 is screwed and pinned with the rotating housing 8 of a small and special transmission 9 that can be built extremely easily. It forms a whole with said transmission 9.

For taking up the torque, transmission 9 is centred at housing part 10 of arrangement 13, screwed and pinned. It is driven by a braking motor 11, which in its turn is centred and screwed at housing part 12 of arrangement 13 (Figure 1). Motor 11 can also be replaced, e.g., by a non-return crank handle.

The axially movable sheave half 14 of traction sheave 1 (Figures 1 and 3 to 5) is pressed against sheave half 7 by springs 4. Said sheave half 14 is carried along for torque transfer via screws 6 which are led through said sheave half 14. Centring of the latter sheave half occurs with the

necessary clearance for the needed axial reciprocation either on the extended groove base 15 of sheave half 7 (Figure 5) or on the outer diameter of transmission 9 (Figure 1).

Fixedly positioned rope-supporting rollers 20 are mounted on axles 21 over the whole wrapping area of wire rope 16 in traction sheave 1 as well as in the outlet area 17 and the inlet area 18 within housing 19 (Figures 1 to 5). At their turn axles 21 are attached to the one side in housing part 12 and to the other side in housing part 10 of housing 9. Separation as well as centring of both housing parts 10, 12 occurs by means of peripheral faces 22. Both housing parts 10, 12 are connected by eight screws 23 to the common housing 19 of arrangement 13 (Figures 1 and 2).

Rope-supporting rollers 20 in Figure 3 are clamped in between springs 24 for reciprocation on axle 21 whereas in Figures 4 and 5 the rollers 20 are fixedly positioned via spacing tube 25 between the housing parts 10 and 12.

For taking up the anchoring of arrangement 13 two lateral tie blocks 26 are provided (Figures 1 and 2), which are fixedly screwed to housing part 19 by three of the eight connecting screws 23. Said tie blocks 26 can take as much as three anchor bolts 27 and also hold the rope inlet piece 28 and the rope outlet piece 29.

First of all, derailment of wire rope 16 within the arrangement 13 upon threading is prevented by the rope-supporting roller 20. Threading occurs by simply forcing wire rope 16 by hand into the guiding hole of rope-inlet guiding segment 28 (Figure 2) with running motor 11.

Wire rope 16 is then pressed into the V-shaped rope groove 2, 3 of the sheave 1 by the first rope-supporting roller 20 in the inlet area 18, and then carried along automatically by traction sheave 1. It is steered aside in outlet area 17 by suitable guides and past the rope section already entered guided laterally out of housing 19 and lead over rope-outlet guiding segment 29 (Figure 2).

In the inverted direction of pulling the rope-supporting roller 20 in outlet area 17 performs the task of threading the wire rope 16 into rope groove 2, 3 of traction sheave 1, while rope at the segment 28 is permanently unloaded.

For sparing wire rope 16 at the inlet and outlet of rope groove 2, 3 the area 30 at the outer diameter of rope groove 2, 3 is rounded off (Figures 3 to 5). For being able to use wire ropes 16 with varying diameters without increasing the outer diameter of traction sheave 1, a two-stage rope groove 3 is provided in Figure 4 whose outer stage 31 has a greater groove angle for the properly entering of slightly thicker wire ropes 16.

Wire rope 16 is loaded and pressed in the inlet area 18 onto the groove base 15 of traction sheave 1 (Figures 2 to 5). The groove angle of rope groove 2, 3 of 20° has been obtained experimentally and is the most favourable. In the outlet area 17 a low pressure of the rope 16 on the rope-supporting roller 20 is attained, which, however, does not harm said wire rope 16.

The represented cover 32 (Figure 1) has only to

close the opening in housing part 10 of arrangement 13.

In order to reduce the wrapping angle of wire rope 16 for decreasing the axial clamping force to a maximum, according to Figure 5 the bisectrix of rope groove 2 with respect to the perpendicular on the rotation axis of traction sheave 1 is inclined. Herein, groove side 33 of the undisplaceable sheave half 7 is only slightly inclined to the perpendicular, whereas on the contrary, groove side 34 of the axially displaceable sheave half 14 has a considerably stronger inclination.

In the embodiment according to Figure 6, a supporting roller 36 is provided in the area 17 of the slack rope 35, the diameter of said roller being twice as large as that of the other supporting rollers 20. Said large supporting roller 36 primarily performs its function during the threading operation of the slack rope 16 or during a lowering operation, i.e. when the arrangement 13, e.g., is fixed to a platform and rope section 37 is taut. In this case rope section 35 in principle runs between supporting roller 36 and traction sheave 1. The large supporting roller 36 reduces the surface pressure exerted on rope 16. This is associated with the advantage that said rope 16 experiences a smaller load and hence may be expected to have a longer service life.

However, supporting roller 36 can fulfil still another function when the arrangement 13 is fixedly supported and the load is suspended from rope section 37. In this case, the suspension point 38 is above the rotation axis 39 of traction sheave 1. In this case, said supporting roller 36 does not only serve the purpose of inlet supporting element for rope 16, but also of tail element for the slack rope 40 (dash-and-dot route).

In Figure 7 an embodiment is represented according to which a shoe 41 in the form of a circular ring segment is mounted at least in area 17 of the slack rope 35 instead of supporting rollers 20. Said shoe 41 can also extend over almost the entire periphery of traction sheave 1 instead of supporting rollers 20. It can be made of one or of several pieces. Preferably, the material of said shoe 41 is a wear-resistant plastic as has been used, e.g., for making conveying chutes.

Figure 8 shows that said shoe 41 can have suitable outlines 42 at the periphery of rope 16. But the supporting rollers 20 too can be suitably grooved at their periphery in accordance with the section of the rope.

Shoes 41 are advantageously used when cross-laid ropes are being applied. If on the contrary, equal-laid ropes are being used wherein the wire ropes are at an angle with respect to the longitudinal axis of rope 16, supporting rollers 20 are preferably used.

Claims

1. An arrangement for lifting and lowering or for pulling loads by means of a wire rope, which comprises a traction sheave (1) consisting of two sheave halves (7, 14) that are mutually axially

displaceable, each sheave half having at its inner side periphery a groove flank that co-operates with a groove flank of the other sheave half to form a groove (2) for engaging a wire rope (16), spring means (4) for biasing the two sheave halves towards each other, and rope-supporting elements which protrude into the rope groove of the traction sheave, thereby to press a wire rope (16) in contact with the groove of the traction sheave for the transmission of driving force to the rope, the rope-supporting elements being mounted at fixed radial distances with respect to the axis of the traction sheave, characterized in that the rope-supporting elements (20, 36, 41), which engage in the rope groove (2, 3), are associated with said rope groove (2, 3) in the entire wrapping area of the rope (16); that the width of the rope-supporting elements (20), considered in the direction of the axis of the traction sheave (1), is smaller than the width of the groove (2) of the traction sheave for any corresponding radius of the traction sheave; that the radial distance between the bottom of the groove of the sheave and the rope-supporting elements is larger than the diameter of the rope and that the flanks of the two sheaves halves (7, 14) are both V-like shaped, thereby to determine a V-groove (2, 3) for the wire rope.

2. An arrangement according to claim 1, characterized thereby that the V-groove (2, 3) determined by the flanks of the two sheaves halves (7, 14) is a symmetric V-groove (2, 3).

3. Arrangement according to claim 1 or 2, characterized thereby that said rope-supporting elements are formed by rollers (20, 36) and/or by shoes (41) that have the shape of a ring segment.

4. Arrangement according to claim 1, 2 or 3, characterized thereby that said rope-supporting elements (20, 36, 41) show recesses (42) that are adapted to the outline of said rope (16).

5. Arrangement according to any of claims 1 to 4, characterized thereby that in the inlet/outlet area (17) of the slack rope (35, 40) a supporting roller (36) is mounted with a diameter that is considerably greater than that of the other supporting rollers (20).

6. Arrangement according to claim 1, characterized thereby that the circumferential area (30) of said rope groove (2, 3) is rounded at the outer diameter of said sheave halves (7, 14).

7. Arrangement according to any of claims 1 to 6, characterized thereby that the bisectrix of said rope groove (2) is sloping with respect to the perpendicular on the axis of rotation (39), wherein especially the groove flank (33) of the axially undisplaceable sheave half (7) is not or only slightly inclined, whereas the groove flank (34) of the axially displaceable sheave half (14) has a stronger inclination.

8. Arrangement according to any of claims 1 to 7, characterized thereby that said rope groove (3) is formed in two stages, the radially outer stage (31) having a greater groove angle than the radially inner stage.

9. Arrangement according to any of claims 1 to

close the opening in housing part 10 of arrangement 13.

In order to reduce the wrapping angle of wire rope 16 for decreasing the axial clamping force to a maximum, according to Figure 5 the bisectrix of rope groove 2 with respect to the perpendicular on the rotation axis of traction sheave 1 is inclined. Herein, groove side 33 of the undisplaceable sheave half 7 is only slightly inclined to the perpendicular, whereas on the contrary, groove side 34 of the axially displaceable sheave half 14 has a considerably stronger inclination.

In the embodiment according to Figure 6, a supporting roller 36 is provided in the area 17 of the slack rope 35, the diameter of said roller being twice as large as that of the other supporting rollers 20. Said large supporting roller 36 primarily performs its function during the threading operation of the slack rope 16 or during a lowering operation, i.e. when the arrangement 13, e.g., is fixed to a platform and rope section 37 is taut. In this case rope section 35 in principle runs between supporting roller 36 and traction sheave 1. The large supporting roller 36 reduces the surface pressure exerted on rope 16. This is associated with the advantage that said rope 16 experiences a smaller load and hence may be expected to have a longer service life.

However, supporting roller 36 can fulfil still another function when the arrangement 13 is fixedly supported and the load is suspended from rope section 37. In this case, the suspension point 38 is above the rotation axis 39 of traction sheave 1. In this case, said supporting roller 36 does not only serve the purpose of inlet supporting element for rope 16, but also of tail element for the slack rope 40 (dash-and-dot route).

In Figure 7 an embodiment is represented according to which a shoe 41 in the form of a circular ring segment is mounted at least in area 17 of the slack rope 35 instead of supporting rollers 20. Said shoe 41 can also extend over almost the entire periphery of traction sheave 1 instead of supporting rollers 20. It can be made of one or of several pieces. Preferably, the material of said shoe 41 is a wear-resistant plastic as has been used, e.g., for making conveying chutes.

Figure 8 shows that said shoe 41 can have suitable outlines 42 at the periphery of rope 16. But the supporting rollers 20 too can be suitably grooved at their periphery in accordance with the section of the rope.

Shoes 41 are advantageously used when cross-laid ropes are being applied. If on the contrary, equal-laid ropes are being used wherein the wire ropes are at an angle with respect to the longitudinal axis of rope 16, supporting rollers 20 are preferably used.

Claims

1. An arrangement for lifting and lowering or for pulling loads by means of a wire rope, which comprises a traction sheave (1) consisting of two sheave halves (7, 14) that are mutually axially

displaceable, each sheave half having at its inside periphery a groove flank that cooperates with a groove flank of the other sheave half to form a groove (2) for engaging a wire rope (16), spring means (4) for biasing the two sheave halves towards each other, and rope-supporting elements which protrude into the rope groove of the traction sheave, thereby to press a wire rope (16) in contact with the groove of the traction sheave for the transmission of driving force to the rope, the rope-supporting elements being mounted at fixed radial distances with respect to the axis of the traction sheave, characterized in that the rope-supporting elements (20, 36, 41), which engage in the rope groove (2, 3), are associated with said rope groove (2, 3) in the entire wrapping area of the rope (16); that the width of the rope-supporting elements (20), considered in the direction of the axis of the traction sheave (1), is smaller than the width of the groove (2) of the traction sheave for any corresponding radius of the traction sheave; that the radial distance between the bottom of the groove of the sheave and the rope-supporting elements is larger than the diameter of the rope and that the flanks of the two sheaves halves (7, 14) are both V-like shaped, thereby to determine a V-groove (2, 3) for the wire rope.

2. An arrangement according to claim 1, characterized thereby that the V-groove (2, 3) determined by the flanks of the two sheaves halves (7, 14) is a symmetric V-groove (2, 3).

3. Arrangement according to claim 1 or 2, characterized thereby that said rope-supporting elements are formed by rollers (20, 36) and/or by shoes (41) that have the shape of a ring segment.

4. Arrangement according to claim 1, 2 or 3, characterized thereby that said rope-supporting elements (20, 36, 41) show recesses (42) that are adapted to the outline of said rope (16).

5. Arrangement according to any of claims 1 to 4, characterized thereby that in the inlet/outlet area (17) of the slack rope (35, 40) a supporting roller (36) is mounted with a diameter that is considerably greater than that of the other supporting rollers (20).

6. Arrangement according to claim 1, characterized thereby that the circumferential area (30) of said rope groove (2, 3) is rounded at the outer diameter of said sheave halves (7, 14).

7. Arrangement according to any of claims 1 to 6, characterized thereby that the bisectrix of said rope groove (2) is sloping with respect to the perpendicular on the axis of rotation (39), wherein especially the groove flank (33) of the axially undisplaceable sheave half (7) is not or only slightly inclined, whereas the groove flank (34) of the axially displaceable sheave half (14) has a stronger inclination.

8. Arrangement according to any of claims 1 to 7, characterized thereby that said rope groove (3) is formed in two stages, the radially outer stage (31) having a greater groove angle than the radially inner stage.

9. Arrangement according to any of claims 1 to





5, characterized thereby that in addition to the wrapping area of rope (16) in said rope groove (2, 3), rope-supporting elements (20, 41) are provided in the outlet area (17) and/or in the inlet area (18) of said rope (16).

10. Arrangement according to any of claims 1 to 9, characterized thereby that said rope-supporting rollers (20, 36) are formed by needle bearings, roller bearings or ball bearings, particularly by grooved ball bearings.

11. Arrangement according to claim 1, characterized thereby that it is composed of a two-part housing (10, 12), the separation (22) as well as the centring of said housing in the area of said rope-supporting rollers (20, 36) being such that each roller shaft (21) is mounted in each housing part (10, 12).

12. Arrangement according to any of claims 1 to 11, characterized thereby that said rope-supporting rollers (20, 36) are supported axially movable.

13. Arrangement according to any of claims 1 to 12, characterized thereby that said rope-supporting rollers (20, 36) are axially chucked between springs (24).

14. Arrangement according to any of claims 1 to 13, characterized thereby that said rope-supporting rollers (20) in the outlet and inlet areas (17) of the released rope section (35) are mounted laterally off-centre following the deflection to the passage in the outlet/inlet area (18) at the stressed rope section (37).

Patentansprüche

1. Eine Vorrichtung zum Heben und Senken oder zum Ziehen von Lasten mit Hilfe eines Drahtseiles, zusammengesetzt aus einer Zugseilscheibe (1) bestehend aus zwei Scheibenhälften (7, 14), die wechselweise axial verschiebbar sind, wobei jede Scheibenhälfte an ihrem inneren Umfang eine Rillenflanke aufweist, die mit einer Rillenflanke in der anderen Scheibenhälfte zusammenarbeitet und beide so eine Rille (2) zum Einlegen des Drahtseiles (16) bilden, federnden Mitteln (4) um die zwei Scheibenhälften zueinander zu neigen, und Seilhalteelementen, die in die Seilrille der Zugscheibe ragen und dabei ein Drahtseil (16) gegen die Rille der Zugscheibe pressen, um die Antriebskraft auf das Seil zu übertragen, wobei die Seilhalteelemente in festen radialen Abständen in bezug auf die Achse der Zugscheibe befestigt sind, gekennzeichnet dadurch, daß die Seilhalteelemente (20, 36, 41), die in die Seilrille (2, 3) eingreifen mit dieser Seilrille (2, 3) der gesamten Umlauffläche des Seiles (16) zugeordnet sind; daß die Breite der Seilhalteelemente (20), betrachtet in Richtung auf die Achse der Zugscheibe (1) für jeden entsprechenden Radius der Zugscheibe kleiner ist als die Breite der Rille (2) der Zugscheibe; daß der radiale Abstand zwischen dem Boden der Rille der Scheibe und den Seilhalteelementen größer ist als der Seildurchmesser und daß die Flanken der beiden Scheibenhälften (7, 14) eine V-Form aufweisen und damit eine V-Rille (2, 3) für das Drahtseil bilden.

2. Die Vorrichtung gemäß Anspruch 1, gekennzeichnet dadurch, daß die von den Flanken der beiden Scheibenhälften (7, 14) gebildete V-Rille (2, 3) eine symmetrische V-Rille (2, 3) ist.

3. Die Vorrichtung gemäß Anspruch 1 oder 2, gekennzeichnet dadurch, daß die erwähnten Seilhalteelemente von Rollen (20, 36) und/oder von Auflagerschuhen (41) in der Form eines Ringsegmentes gebildet werden.

4. Die Vorrichtung gemäß Anspruch 1, 2 oder 3, gekennzeichnet dadurch, daß die erwähnten Seilhalteelemente (20, 36, 41) Auskehlung (42) aufweisen, die der Kontur des erwähnten Seiles (16) angepaßt sind.

5. Die Vorrichtung gemäß einem der Ansprüche 1 bis 4, gekennzeichnet dadurch, daß im Eingangs-/Ausgangsbereich (17) des schlaffen Seiles (35, 40) eine Halterolle (36) befestigt ist, deren Durchmesser erheblich größer ist, als der der anderen Seilhalterollen (20).

6. Die Vorrichtung gemäß Anspruch 1, gekennzeichnet dadurch, daß der Umfangsbereich (30) der erwähnten Seilrille (2, 3) am Außendurchmesser der erwähnten Scheibenhälften (7, 14) abgerundet ist.

7. Die Vorrichtung gemäß einem der Ansprüche 1 bis 6, gekennzeichnet dadurch, daß die Symmetrielinie der erwähnten Seilrille (2) in bezug auf die senkrechte auf der Drehachse (39) geneigt ist, worin besonders die Rillenflanke (33) der axial nicht verschiebbaren Scheibenhälfte (7) nicht oder nur wenig geneigt ist, während die Rillenflanke (34) der axial verschiebbaren Scheibenhälfte (14) eine größere Neigung aufweist.

8. Die Vorrichtung gemäß einem der Ansprüche 1 bis 7, gekennzeichnet dadurch, daß die erwähnte Seilrille (3) in zwei Stufen gebildet wird, wobei die radial außenliegende Stufe (31) einen größeren Rillenkantenwinkel aufweist, als die radial innenliegende Stufe.

9. Die Vorrichtung gemäß einem der Ansprüche 1 bis 5, gekennzeichnet dadurch, daß zusätzlich zum Wickelbereich des Seiles (16) in der erwähnten Seilrille (2, 3) auch der Ausgangsbereich (17) und/oder der Eingangsbereich (18) des erwähnten Seiles (16) mit Seilhalteelementen (20, 41) versehen sind.

10. Die Vorrichtung gemäß einem der Ansprüche 1 bis 9, gekennzeichnet dadurch, daß die erwähnten Seilhalterollen (20, 36) von Nadellagern, Rollenlagern oder Kugellagern, insbesondere von gerillten Kugellagern gebildet werden.

11. Die Vorrichtung gemäß Anspruch 1, gekennzeichnet dadurch, daß sie aus einem zweiteiligen Gehäuse (10, 12) zusammengesetzt ist, wobei die Trennung (22) sowie die Zentrierung des erwähnten Gehäuses im Bereich der erwähnten Seilhalterollen (20, 36) so ausgelegt sind, daß jeder Rollenschachtel (21) in jedem Gehäuseteil (10, 12) befestigt ist.

12. Die Vorrichtung gemäß einem der Ansprüche 1 bis 11, gekennzeichnet dadurch, daß die erwähnten Seilhalterollen (20, 36) axial beweglich gehalten werden.

13. Die Vorrichtung gemäß einem der Ansprüche

che 1 bis 12, gekennzeichnet dadurch, daß die erwähnten Seilhalterollen (20, 36) axial zwischen Federn (24) eingeklemmt sind.

14. Die Vorrichtung gemäß einem der Ansprüche 1 bis 13, gekennzeichnet dadurch, daß die erwähnten Seilhalterollen (20) in den Ausgangs- und Eingangsbereichen (17) des freien Seilabschnittes (35) lateral exzentrisch befestigt sind, folgend der Krümmung des Durchgangs im Ausgangs-/Eingangsbereich (18) des straffen Seilabschnittes (37).

Revendications

1. Système en vue de soulever et d'abaisser ou de tirer des charges au moyen d'un cordage métallique, ce système comprenant un réa de traction (1) constitué de deux moitiés de réa (7, 14) qui peuvent être déplacées axialement et mutuellement, chaque moitié de réa ayant, sur sa périphérie intérieure, un flanc de gorge qui coopère avec un flanc de gorge de l'autre moitié de réa pour former une gorge (2) sur laquelle est destiné à venir s'engager un cordage en fil métallique (16), des moyens à ressorts (4) en vue de pousser les deux moitiés de réa l'une vers l'autre, ainsi que des éléments supportant le cordage et ressortant dans la gorge réservée au cordage du réa de traction, pressant ainsi un cordage de fil métallique (16) en contact dans la gorge du réa de traction pour la transmission de la force d'entraînement du cordage, les éléments supportant le cordage étant montés à des distances radiales fixes vis-à-vis de l'axe du réa de traction, caractérisé en ce que les éléments (20, 36, 41) supportant le cordage et venant s'engager dans la gorge (2, 3) réservée à ce dernier sont associés à cette gorge de cordage (2, 3) dans la totalité de la zone d'enroulement du cordage (16), en ce que la largeur des éléments (20) supportant le cordage et vu dans le sens de l'axe du réa de traction (1) est plus courte que la largeur de la gorge (2) du réa de traction pour n'importe quel rayon correspondant du réa de traction; en ce que la distance radiale entre le fond de la gorge du réa et les éléments supportant le cordage est plus grande que le diamètre du cordage et que les flancs des deux moitiés de réa (7, 14) sont tous deux en V, déterminant ainsi une gorge en V (2, 3) pour le cordage en fil métallique.

2. Système selon la revendication 1, caractérisé en ce que la gorge en V (2, 3) déterminée par les flancs des deux moitiés de réa (7, 14) est une gorge en V symétrique (2, 3).

3. Système selon la revendication 1 ou 2, caractérisé en ce que les éléments supportant le cordage sont formés par des rouleaux (20, 36) et/ou par des patins (41) qui ont la forme d'un segment annulaire.

4. Système selon la revendication 1, 2 ou 3, caractérisé en ce que les éléments supportant le cordage (20, 36, 41) présentent des cavités (42) qui sont adaptées au contour du cordage (16).

5. Système selon l'une quelconque des revendications 1 à 4, caractérisé en ce que, dans la zone d'entrée/sortie (17) du cordage en mou (35, 40), un rouleau support (36) est monté avec un diamètre qui est considérablement plus grand que celui des autres rouleaux supports (20).

6. Système selon la revendication 1, caractérisé en ce que la zone circonférentielle (30) de la gorge de cordage (2, 3) est arrondie au diamètre extérieur des moitiés de réa (7, 14).

7. Système selon l'une quelconque des revendications 1 à 6, caractérisé en ce que la bissectrice de la gorge de cordage (2) est inclinée par rapport à la perpendiculaire sur l'axe de rotation (39), le flanc (33) de la gorge de la moitié de réa (7) axialement non déplaçable étant particulièrement non ou seulement légèrement incliné, tandis que le flanc de la gorge (34) de la moitié de réa axialement déplaçable (14) a une plus forte inclinaison.

8. Système selon l'une quelconque des revendications 1 à 7, caractérisé en ce que la gorge de cordage (3) est formée en deux stades, le stade extérieur radial (31) comportant un angle de gorge plus grand que celui du stade intérieur radial.

9. Système selon l'une quelconque des revendications 1 à 5, caractérisé en ce que, outre la zone d'enroulement du cordage (16) dans la gorge de cordage précitée (2, 3), on prévoit des éléments supportant le cordage (20, 41) dans la zone de sortie (17) et/ou dans la zone d'entrée (18) du cordage (16).

10. Système selon l'une quelconque des revendications 1 à 9, caractérisé en ce que les rouleaux (20, 36) supportant le cordage sont formés par des paliers à aiguilles, des paliers à galets ou des paliers à billes, en particulier, des paliers à billes rainurées.

11. Système selon la revendication 1, caractérisé en ce qu'il est constitué d'un logement en deux pièces (10, 12), la séparation (22), du même que le centrage du logement dans la zone des rouleaux (20, 36) supportant le cordage étant tels que chaque arbre de rouleau (21) est monté dans chaque partie de logement (10, 12).

12. Système selon l'une quelconque des revendications 1 à 11, caractérisé en ce que les rouleaux (20, 36) supportant le cordage sont supportés de manière à pouvoir se déplacer axialement.

13. Système selon l'une quelconque des revendications 1 à 12, caractérisé en ce que les rouleaux (20, 36) supportant le cordage sont mandrinés axialement entre des ressorts (24).

14. Système selon l'une quelconque des revendications 1 à 13, caractérisé en ce que les rouleaux (20) supportant le cordage dans les zones de sortie et d'entrée (17) de la section de cordage libérée (35) sont montés latéralement en position décentrée suivant la déflexion par rapport au passage dans la zone de sortie/entrée (18) à la section de cordagesous contrainte (37).

