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Revolving door lock

The present invention relates to a revolving door lock having outer walls laterally defining a passage space.

In the present times of energy saving it is important to seal off entrances of buildings as good as possible so that
5 no heating power disappears from the building. The inner space of the building should also be kept free of draught, which is particularly important in restaurants or hotels, shops and main entrances of commercial properties, hospitals and the like. Until now, this object has been obtained by so-called
10 revolving cross doors, in which three or four door wings define three or four cylinder segments having an angle at the centre of 120° or 90° respectively. With these revolving cross doors, in particular with the four winged revolving door, the passage space is very narrow so that it is for example not
15 possible to enter the building entrance with a trolley or a wheel chair, and the passage with suitcases is difficult. A further disadvantage of these prior art doors is that all persons passing through the revolving cross doors should go equally fast so that old or handicapped people could become
20 pinched due to a faster person which is also in the area of the revolving cross door, or due to a motorized revolving cross door being driven too fast. Motorized revolving cross doors having a lower rotational speed have a reduced capacity, however, and prevent a quick passage through the revolving
25 door lock.

The object of the present invention is to provide a revolving door lock which does not have said disadvantages of the prior art revolving cross doors.

For obtaining said object, the revolving door lock
30 according to the invention is characterized by at least two revolving doors moving relatively synchronized and having two wings rotatable about respective vertical rotating axes, said wings of the different revolving doors have such relative angular position that the passage space is always closed at
35 least on one side.

The revolving door lock according to the invention effects that, as a result, in each position of the revolving doors the inner space of the building is sealed to the outside

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so that no heat energy is lost for nothing and hardly no draught appears. On the other hand, there is no need for three or four wings at the individual revolving doors so that with each revolving door more space is available for the passage
5 with trolleys, wheel chairs, luggage and the like.
Furthermore, the revolving door lock can be passed quickly, in that at each revolving door a free passage is possible since only one of the doors should effect a seal at a time, while the other one can be fully open. As a result, several persons
10 may pass through the revolving door lock in their own speed without interfering with each other.

A maximum passage space at the revolving doors is created if each revolving door comprises two aligned wings on both sides of the respective rotating axis. Then, a proper
15 seal through the revolving door lock is always effected when two revolving doors are provided of which the angles to their own line of symmetry in longitudinal direction have a relative angular off-set of 90°.

A favourable embodiment of the revolving door lock
20 according to the invention is characterized in that the rotating axes of the revolving doors are positioned at a distance which is greater than the sum of the radius of both revolving doors, and preferably greater than 2.25 times the radius.

25 Due to this feature it is possible to create an intermediate space between the revolving doors offering persons passing through the revolving door lock the opportunity to stop between the revolving doors. This is of great importance to the safety as people could panic at the
30 transfer between the revolving doors because they do not know how to walk and could then be hit by the door wings. According to the invention the people are able to reorientate in the intermediate space and only then continue on their way. With a distance of at least ± 0.4 m and preferably 0.75 m between the
35 ends of the door wings there is created enough space for persons, while for example in supermarkets where people pass through the revolving door lock with shopping trolleys a room of at least 1.5 m provides a place to stop. This intermediate space also enables a continued passage between the revolving
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enter the next revolving door just before or just after the passage of a door wing. On the other hand, the feature offers new arrangement possibilities of the revolving doors. It is for instance possible to connect three or more revolving doors to the intermediate or passage space, which are adapted to each other and together effect the sealing of the inner space of the building.

Embodiments of the invention by way of example are elucidated hereafter with reference to the attached drawings.

10 In the drawings:

Fig. 1 is a very schematic plan view of a first embodiment of the revolving door lock and drive,

Fig. 2 shows four different positions of the revolving doors within the revolving door lock of Fig. 1,

15 Fig. 3 shows the revolving door lock of Fig. 1 with the revolving doors in an emergency position,

Fig. 4 shows the revolving door lock of Fig. 1 with the outer revolving doors in the night position,

20 Fig. 5 is a schematic plan view of a second embodiment of the revolving door lock according to the invention illustrating a so-called summer position with interrupted lines,

25 Fig. 6 is a very schematic plan view of a third embodiment of the revolving door lock according to the invention,

Fig. 7 is a substantially simplified block diagram of a synchronizing device of the embodiment of Fig. 5.

In the various figures similar parts are indicated by the same reference numerals.

30 The revolving door lock 1 according to Fig. 1-4 includes four outer walls 2, 3, 4, 5 each forming a part of a cylindrical casing, the outer walls 2 and 3 for example forming an entrance or exit space 6 on the side of the street and the outer walls 4 and 5 defining an entrance or exit space 35 7 on the inner side of a building. The pairs of outer walls 2, 3 and 4, 5, respectively, are diametrically opposed and extend through an angle of 90°. The vertical central shafts 8 and 9 of the space 6, 7 form rotating axes for the revolving doors 10 and 11. Each revolving door 10 and 11 consists of two door 40 wings 26, 27 and 28, 29, respectively, positioned on both



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sides of the rotating axis 8, 9 and each pair being mutually aligned. Of course, the door wings 26, 27 and 28, 29 could be integral. At the upper side of the revolving doors 10 and 11 there is disposed on the respective rotating shafts 8 and 9 and within a gear box a gear 12, 13, respectively, including bevel gearing wheels. Drive shafts 14 and 15 are connected to the gears 12 and 13 and being commonly driven through a bevelled gear transmission 16 by a driving shaft 17 which is rotated by a drive system 18. The drive system 18 is controlled by a control device 21.

In Fig. 2, four positions of the revolving doors 10 and 11 are illustrated. Interrupted lines illustrate the free passage space 22, 23, 24, 25, respectively, through the passage lock. In front of the passage lock there is a movement sensor activating the drive of the revolving doors as soon as a person approaches the passage lock. Contrary to conventional revolving doors in which two revolving doors are positioned perpendicularly and include a common rotating axis, the revolving door lock according to the invention does not offer only a cylindrical segment of 90° as passage space, but a longer area, which better enables the passage with a trolley, in a wheel chair or with suitcases.

The revolving doors preferably move at one revolution per minute in idle operation in which the drive is switched off when there is a rest position without movement sensing. In operation, the number of revolutions per minute is preferably five and after circa 15 seconds without sensing movements there is a switch to the idle operation.

The dotted area 23 according to Fig. 2b illustrates that already in this position the passage space is wide open. In the position of Fig. 2c, the entrance door 10 is closing, wherein, however, the passage is not urged from behind contrary to conventional revolving crosses. In the position of Fig. 2d, the entrance to the interior of the building is released and one can quickly continue its passage and no urge from behind is created.

In Fig. 3 the revolving door lock is shown in a position of the revolving doors 10 and 11 as can be used for example in the case of an emergency. Both revolving doors 10 and 11 are hingedly supported in which both wings 26, 27 and



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28, 29, respectively, of both revolving doors 10 and 11 may be hinged towards each other. All four wings of both revolving doors 10 and 11 are then aligned so that there is created a maximum passage space 6, 7 through the revolving door lock.

5 In the hinged bearing support of the revolving doors 10 and 11, the wings 26, 27 and 28, 29, respectively, are pivotally attached to the respective shafts 8 and 9 so that they can be relatively hinged after they are unlocked.

In Fig. 4, the passage lock is shown in its closed
10 position. Then, both wings 26 and 27 of the outer revolving door 10 are hinged outwardly and are closed to the ends 31 and 32 of both outer walls 2 and 3. The revolving door 11 is then preferably in a position perpendicular to the building wall 33. The passage spaces 6 and 7 of the revolving door lock 1
15 may then be used in this position for entrance purposes during the night or in weekends.

The sealing of the ends 34 and 35, and 36 and 37, respectively, of the revolving doors 10 and 11 to the inner side of the outer walls 2, 3, 4 and 5 of the revolving door
20 lock is effected, for example, by means of vertical brushes attached to the revolving door ends. To the outer ends 31, 32 and 38, 39, respectively, of the outer walls 2, 3, 4 and 5 of the revolving door lock, there could be arranged pneumatic or electric safety pressure strips, for example. In principle,
25 all main and secondary closing edges are secured in accordance with the regulations and the state of the art. In case there is too much force active between the ends of the revolving doors and the ends of the outer walls which may cause a risk of injury, the safety pressure strip is activated and the
30 drive system of the revolving doors is switched off. If the wings are hingedly arranged, then also the wing may hinge back by this force and the driving force is interrupted until the wing is hinged back to its original position.

Fig. 5 shows a second embodiment of the revolving
35 door lock according to the invention in which there is arranged an intermediate part 40, 41, respectively, between the outer walls 2, 4 and 3, 5, respectively, so that there is created an intermediate space 42 between the revolving doors 10 and 11. As a result, the rotating axes 8 and 9 of the
40 revolving doors 10 and 11 are positioned at a distance to each



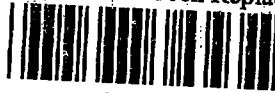
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other which is greater than the sum of the radius R of both revolving doors 10 and 11 of which the radii R are equal in this case. Due to this feature, the paths of both revolving doors 10 and 11 do not overlap as in the first embodiment of Fig. 1-4, and there is even created an intermediate space enabling a person to wait within the intermediate space 42 before continuing its way. In a passage lock having revolving doors the diameter of which being 3 m, the intermediate space between the ends of the door wings may for example be 0.75 m in passages for persons and 1.5 m in passages for persons having shopping trolleys.

Furthermore, each revolving door 10, 11 in this embodiment has its own drive motor 43, 44 which are electronically coupled through a synchronizing device 45 ensuring the synchronized rotation of the revolving doors 10, 11. Such embodiment in which the synchronized rotation of the revolving doors 10, 11 is obtained through a synchronizing device has the important advantage that each drive motor 43, 44 drives a smaller mass resulting in an enhanced safety due to a smaller mass inertia.

A substantially simplified block diagram of the synchronizing device 45 is shown in Fig. 7. The synchronizing device 45 includes a central processing unit 55 controlling the drive motors 43 and 44. Each drive motor 43, 44 comprises a respective so-called incremental angular encoder 56, 57, respectively, which is coupled directly to the shaft of the drive motor and which delivers a certain number of pulses for each revolution of the motor shaft. The drive motors 43, 44 drive the rotating shafts 8, 9 through a reduction gear 58, 59 so that, by means of an angular encoder having a relatively small resolution, nonetheless a great number of pulses per revolution of the revolving door and consequently a high resolution for determining the position of the revolving door is obtained. The counter position of each angular encoder 56, 57, which counter position indicates the rotational position of the respective revolving door 10, 11, is supplied to the processing unit 55.

Furthermore, a position detector 60, 61 for each revolving door is mounted in a fixed position relative to the respective revolving door 10, 11, respectively. As is shown in



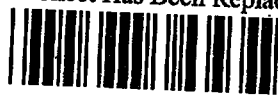
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Fig. 5, the fixed position of the position detector 60 is off-set 90° with respect to the fixed position of the position detector 61. Both position detectors 60, 61 are also connected to the processing unit. Each position detector 60, 61 is able to deliver a signal each 180° or each 360° , dependent on the fact whether one or both door wings comprise for example a magnetic element which is detected by the position detector 60, 61.

Finally, the processing unit 55 is connected to a conventional control unit (not shown) or a revolving door which is adapted a.o. to deliver a rotation command to the processing unit. When the processing unit 55 receives a rotation command, the processing unit activates the drive motors 43, 44 and causes the revolving doors 10, 11 to rotate in a synchronized fashion with a speed adjustable through a control panel 62, by means of the signals of the angular encoders 56, 57 and the position detectors 60, 61. The processing unit 55 resets the counter position of each angular encoder to a starting position, preferably the zero position, each time the respective position detector 60, 61 monitors a door wing 26 or 27 and consequently when a particular rotational position is reached. As a result, the processing unit 55 may obtain the synchronized rotation of the revolving doors 10, 11 by equalizing the counter positions of both angular encoders 56, 57. Maintaining the difference in angular position of 90° between the revolving doors 10, 11 is very simple due to the relative 90° off-set of both position detectors 60, 61.

If the processing unit 55 signalizes a lack of synchronization of the revolving doors 10, 11, the synchronized rotation of the revolving doors is restored by the processing unit by braking the leading revolving door until the revolving doors are synchronized again. Consequently, no unsafe situations will occur as a result of a sudden acceleration of a revolving door.

It will be clear that when there is a lack of synchronization, both the revolving door 10 and the revolving door 11 may be the leading revolving door. In the synchronizing device 45 there is no master-slave situation which is common in synchronizing systems. Both revolving doors



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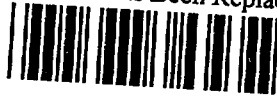
10, 11 can be master and slave for the synchronizing device.

As in normal operation the processing unit 55 receives signals both from the angular encoders 56, 57 and from the position detectors 60, 61, the processing unit 55 can easily check the proper operation of the angular encoders 56, 57 and the position detectors 60, 61. If, after a full revolution, no signal is received from a position detector 60, 61, the processing unit 55 delivers an alarm signal through the control panel 62 in which, for example by means of individual LED's (not shown), the defective position detector is indicated. The processing unit 55 keeps the drive motors 43, 44 in operation and strives to maintain the synchronized rotation by means of the signals of the angular encoders 56, 57. For reasons of safety, the processing unit 55 may select a lower rotational speed, for example.

If no signal from an angular encoder 56, 57 is received, the processing unit 55 also delivers an alarm signal through the control panel 62 in which the defective angular encoder 56, 57 is indicated by a LED. In case of a defective angular encoder 56, 57, the processing unit 55 may strive to maintain a synchronized rotation by means of the position detectors 60, 61. Alternatively, it is possible to stop the drive motor of the defective angular encoder 56, 57 and to release the respective revolving door to allow for manual operation.

If the synchronizing device 45 is connected to the voltage of the supply line for the first time or if the processing unit 55 is reset, the processing unit 55 controls the drive motors 43, 44 such that the revolving doors 10, 11 are rotated to the particular position determined by the respective position detector 60, 61 at a creeping speed to determine the starting positions of the angular encoders 56, 57. Then, the revolving doors 10, 11 are accelerated and decelerated again for a short time, enabling the processing unit 55 to determine from the motor current the mass driven by the drive motors. With this information, the processing unit 55 is able to limit the maximum torque of the revolving doors 10, 11 to a value laid down in safety regulations.

The synchronizing device 45 further includes an emergency stop device 63 directly connected to the drive



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motors 43, 44 and stopping and short-circuiting the drive motors in case of an emergency stop.

By using the synchronizing device 45 including a drive motor 43, 44 for each revolving door 10, 11 it is possible to simply set the revolving doors in certain particular positions. In Fig. 5, for example, interrupted lines indicate a so-called summer position of the revolving door lock 1 in which the revolving doors 10, 11 are positioned individually and aligned with each other along the longitudinal axis of the revolving door lock 1. The doors 10, 11 not only open the passage but also simultaneously form a separation of the entrance and exit paths. This position may also be used for the transport of goods occupying a lot of space.

It is also possible to select a night position through the control panel 62, in which the revolving door 11 is in its summer position indicated by an interrupted line, while the revolving door 10 is perpendicular to the summer position, for example.

Fig. 6 shows a further embodiment in which the revolving door lock 1 comprises three revolving doors, that is a single outer revolving door 46 and two inner revolving doors 47 and 48. Between the revolving doors there is a central passage space 49 defined by the revolving doors 46-48 and by the outer walls 50. Such revolving door lock may for example be used for the entrance and exit of two interior spaces separated by a partition wall 51, and it is also possible that the revolving door 46 having a large capacity is used for both entrance and exit, while the smaller revolving doors 47 and 48 are used as entrance on the one hand and as exit on the other hand. In this embodiment, the vertical planes of symmetry 52, 53 and 54 in the longitudinal directions of the revolving doors 46, 47, 48, respectively, are not parallel to each other. Nonetheless, the angles β of the revolving doors 47 and 58 to their respective longitudinal plane 53, 54 are off-set 90° with respect to the angle α of the revolving door 46 with respect to its longitudinal plane 52. In this way it is ensured that either the revolving door 46 or the revolving doors 47 and 48 are in their closed position thereby preventing draught through the revolving door lock. Of course,

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also other arrangements of a larger or smaller number of revolving doors may be selected within a revolving door lock depending on the respective situation.

5 The invention is not restricted to the embodiments shown in the drawing and described herein before which may be varied in different manners within the scope of the invention defined in the claims.

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1. Revolving door lock having outer walls laterally defining a passage space, characterized by at least two revolving doors moving relatively synchronized and having two wings rotatable about respective vertical rotating axes, said
5 wings of the different revolving doors have such relative angular position that the passage space is always closed at least on one side.

2. Revolving door lock according to claim 1, wherein each revolving door comprises two aligned wings on both sides
10 of the respective rotating axis.

3. Revolving door lock according to claim 1 or 2, wherein two revolving doors are provided, of which the angles to the respective line of symmetry in longitudinal direction have an angular off-set of 90°.

15 4. Revolving door lock according to one of the preceding claims, wherein the rotating axes of the revolving doors are positioned at a distance which is greater than the sum of the radius of both revolving doors.

5. Revolving door lock according to claim 4, wherein
20 the distance between the rotating axes of equally dimensioned revolving doors is at least 2.25 times the radius thereof.

6. Revolving door lock according to claim 1 or 2, wherein three or more revolving doors are provided, being positioned in the outer walls of a larger passage space.

25 7. Revolving door lock according to one of claims 2-6, wherein the wings of the revolving doors are relatively pivotable back and forth about the respective rotating axis.

8. Revolving door lock according to one of the preceding claims, wherein the drive of the revolving doors
30 takes place by means of a single motor being connected to the revolving door shafts by a mechanical coupling.

9. Revolving door lock according to one of claims 1-7, wherein each revolving door is equipped with its own drive motor and the individual drive motors being coupled by an
35 electronic synchronizing device.

10. Revolving door lock according to claim 9, characterized in that the synchronizing device for each



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revolving door comprises an angular encoder delivering a certain number of pulses for each revolution of the revolving door so that a counter position indicates the rotational position of the revolving door, and a position detector for
5 detecting at least one particular revolving position of the revolving door, wherein a processing unit is provided to which the angular encoder and the position detectors are connected and which controls the drive motors, said processing unit resets the counter position of the angular encoder to a
10 starting position when the respective position detector signalizes that the particular rotating position is reached, and said processing device being adapted to allow the revolving doors to move with a synchronized adjustable speed by means of the signals of the angular encoder and position
15 detectors.

11. Revolving door lock according to claim 10, characterized in that the position detector on the one side of the revolving door lock is arranged in a fixed position relative to the respective revolving door, which is rotated
20 90° with respect to the fixed position taken by the position detector(s) on the other side of the revolving door lock relative to the respective revolving door, and the processing unit controlling the drive motors such that the angular encoders indicate the same revolving door position.

25 12. Revolving door lock according to claim 10 or 11, characterized in that, if the revolving doors are not synchronized, the processing unit brakes the leading revolving door to restore the synchronization.

13. Revolving door lock according to claim 10, 11 or
30 12, characterized in that the processing unit monitors the presence of the signals of the angular encoder and the position detectors and delivers an alarm signal if a position detector does not deliver a signal after a revolution of the revolving door, wherein the processing unit preferably
35 maintains a synchronized rotation of the revolving doors at a lower speed by means of the signals of the angular encoders.

14. Revolving door lock according to claim 13, characterized in that the processing unit also delivers an alarm signal if the signal of an angular encoder disappears,
40 wherein preferably at least the respective revolving door is



stopped and released.

15. Revolving door lock according to one of claims 10-14, characterized in that, upon switching to the voltage of the supply line or upon resetting the processing unit, the processing unit controls the drive motors such that the revolving doors are moved to the rotational position determined by the position detector at a creeping speed, wherein, after reaching this rotational position, the processing unit accelerates and decelerates the revolving doors for a short time to determine the mass driven by the drive motor.

16. Revolving door lock according to one of claims 10-15, characterized in that an emergency stop device is provided being directly connected to the drive motors and short-circuiting them in case of an emergency stop.

17. Revolving door lock according to one of claims 10-16, characterized in that each angular encoder is coupled directly to the shaft of the drive motor and the drive motor is driving the revolving door through a reduction gear.



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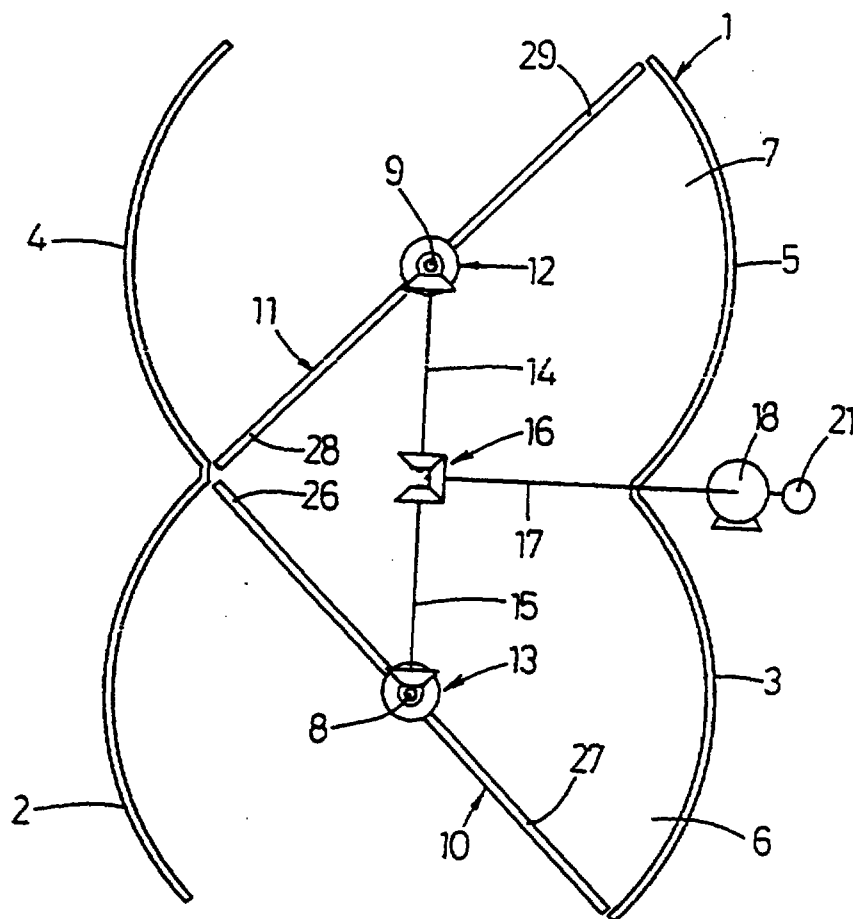


Fig.1

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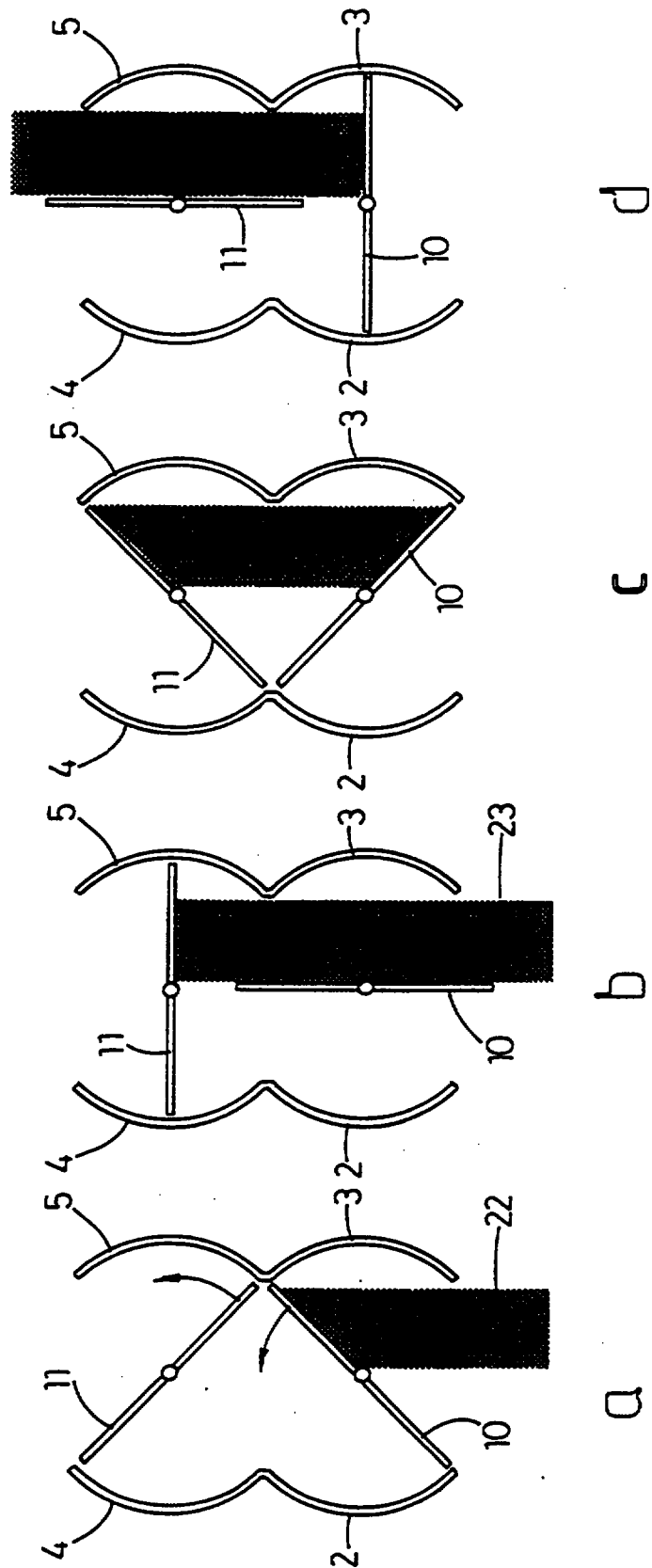


Fig. 2

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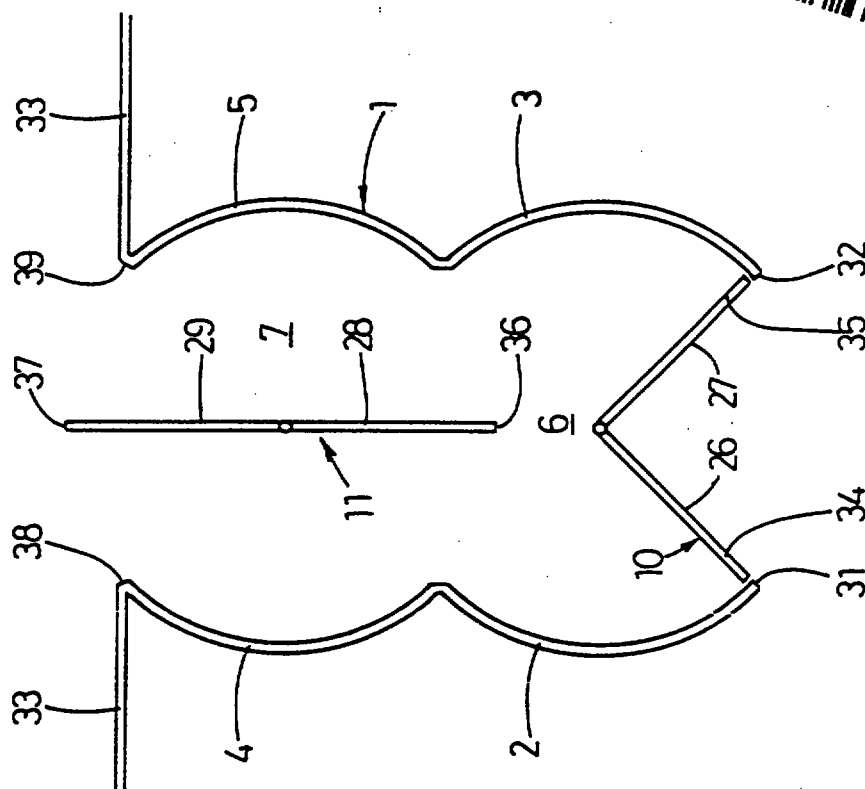


Fig. 4

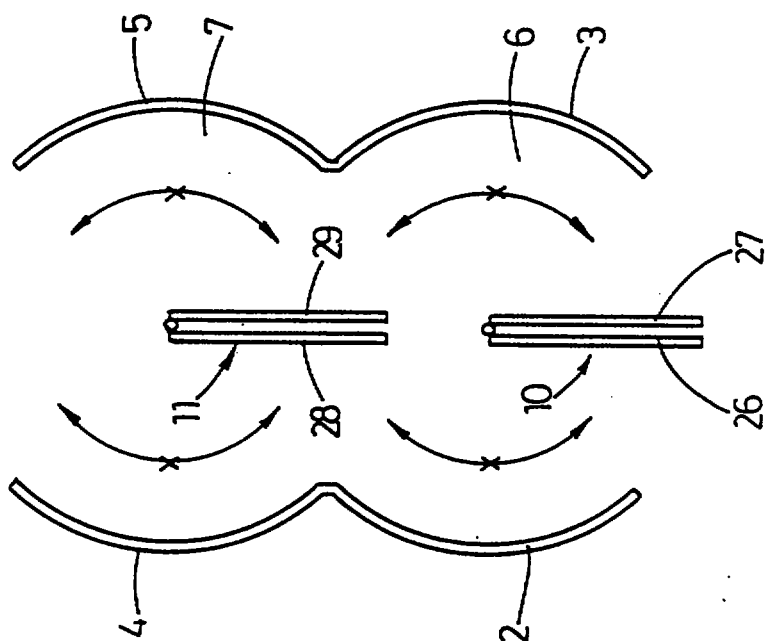


Fig. 3

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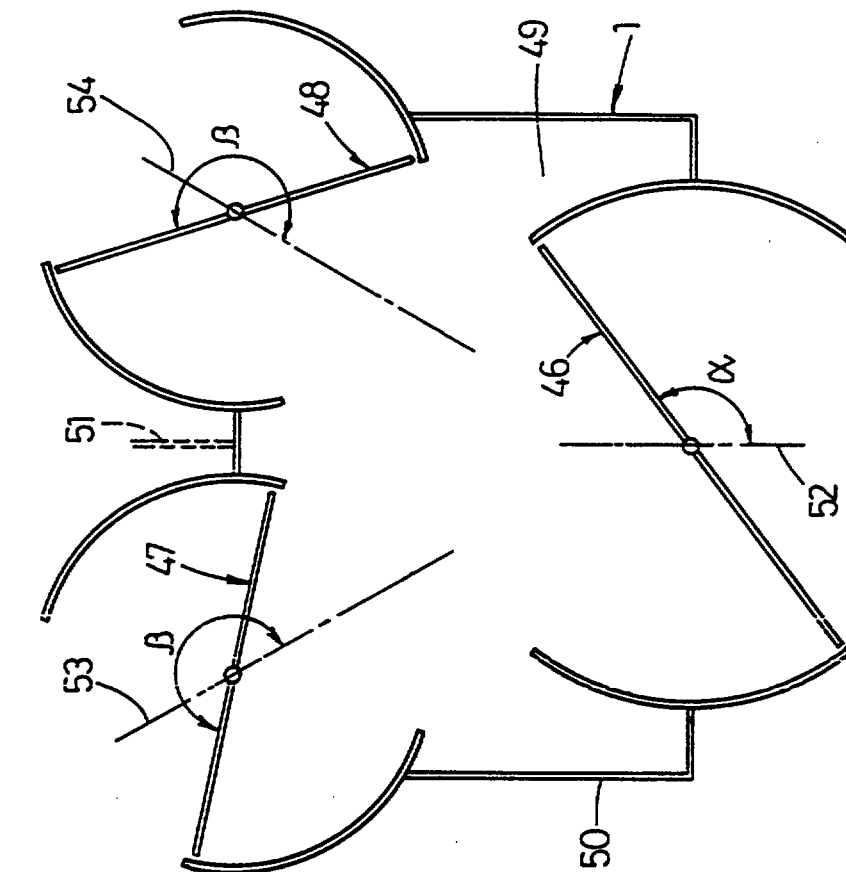


Fig. 6

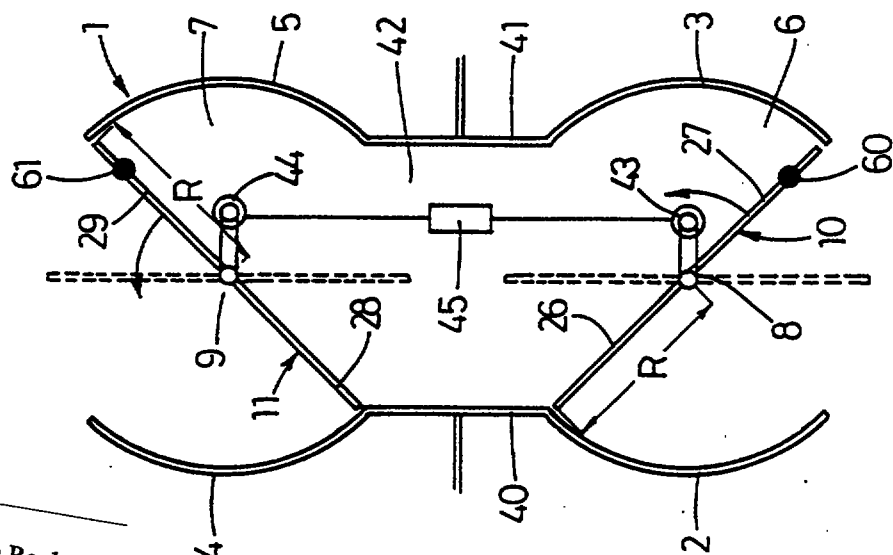


Fig. 5

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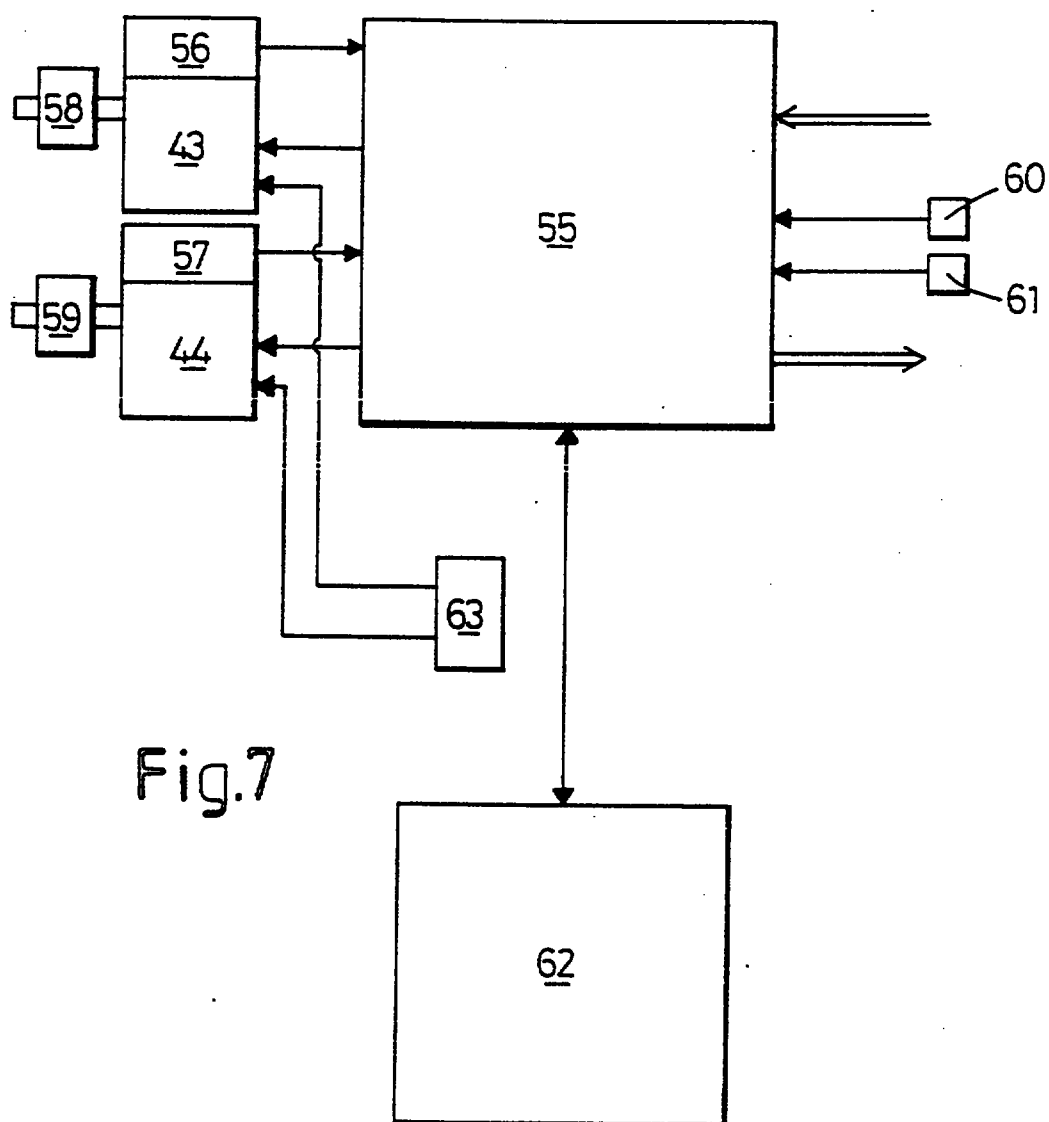
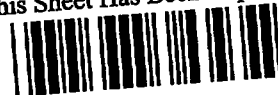
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Fig.7

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Revolving door lock

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In the present times of energy saving it is important to seal off entrances of buildings as good as possible so that
5 no heating power disappears from the building. The inner space of the building should also be kept free of draught, which is particularly important in restaurants or hotels, shops and main entrances of commercial properties, hospitals and the like. Until now, this object has been obtained by so-called
10 revolving cross doors, in which three or four door wings define three or four cylinder segments having an angle at the centre of 120° or 90° respectively. With these revolving cross doors, in particular with the four winged revolving door, the passage space is very narrow so that it is for example not
15 possible to enter the building entrance with a trolley or a wheel chair, and the passage with suitcases is difficult. A further disadvantage of these prior art doors is that all persons passing through the revolving cross doors should go equally fast so that old or handicapped people could become
20 pinched due to a faster person which is also in the area of the revolving cross door, or due to a motorized revolving cross door being driven too fast. Motorized revolving cross doors having a lower rotational speed have a reduced capacity, however, and prevent a quick passage through the revolving
25 door lock.

The object of the present invention is to provide a revolving door lock which does not have said disadvantages of the prior art revolving cross doors.

For obtaining said object, the revolving door lock
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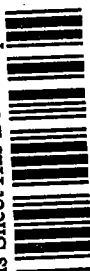
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so that no heat energy is lost for nothing and hardly no draught appears. On the other hand, there is no need for three or four wings at the individual revolving doors so that with each revolving door more space is available for the passage with trolleys, wheel chairs, luggage and the like. Furthermore, the revolving door lock can be passed quickly, in that at each revolving door a free passage is possible since only one of the doors should effect a seal at a time, while the other one can be fully open. As a result, several persons may pass through the revolving door lock in their own speed without interfering with each other.

A maximum passage space at the revolving doors is created if each revolving door comprises two aligned wings on both sides of the respective rotating axis. Then, a proper seal through the revolving door lock is always effected when two revolving doors are provided of which the angles to their own line of symmetry in longitudinal direction have a relative angular off-set of 90° .

A favourable embodiment of the revolving door lock according to the invention is characterized in that the rotating axes of the revolving doors are positioned at a distance which is greater than the sum of the radius of both revolving doors, and preferably greater than 2.25 times the radius.

Due to this feature it is possible to create an intermediate space between the revolving doors offering persons passing through the revolving door lock the opportunity to stop between the revolving doors. This is of great importance to the safety as people could panic at the transfer between the revolving doors because they do not know how to walk and could then be hit by the door wings. According to the invention the people are able to reorientate in the intermediate space and only then continue on their way. With a distance of at least ± 0.4 m and preferably 0.75 m between the ends of the door wings there is created enough space for persons, while for example in supermarkets where people pass through the revolving door lock with shopping trolleys a room of at least 1.5 m provides a place to stop. This intermediate space also enables a continued passage between the revolving doors to both quickly and slowly passing people which may



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enter the next revolving door just before or just after the passage of a door wing. On the other hand, the feature offers new arrangement possibilities of the revolving doors. It is for instance possible to connect three or more revolving doors to the intermediate or passage space, which are adapted to each other and together effect the sealing of the inner space of the building.

Embodiments of the invention by way of example are elucidated hereafter with reference to the attached drawings.

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Fig. 2 shows four different positions of the revolving doors within the revolving door lock of Fig. 1,

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Fig. 7 is a substantially simplified block diagram of a synchronizing device of the embodiment of Fig. 5.

In the various figures similar parts are indicated by the same reference numerals.

30 The revolving door lock 1 according to Fig. 1-4 includes four outer walls 2, 3, 4, 5 each forming a part of a cylindrical casing, the outer walls 2 and 3 for example forming an entrance or exit space 6 on the side of the street and the outer walls 4 and 5 defining an entrance or exit space 7 on the inner side of a building. The pairs of outer walls 2, 3 and 4, 5, respectively, are diametrically opposed and extend through an angle of 90°. The vertical central shafts 8 and 9 of the space 6, 7 form rotating axes for the revolving doors 10 and 11. Each revolving door 10 and 11 consists of two door wings 26, 27 and 28, 29, respectively, positioned on both

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sides of the rotating axis 8, 9 and each pair being mutually aligned. Of course, the door wings 26, 27 and 28, 29 could be integral. At the upper side of the revolving doors 10 and 11 there is disposed on the respective rotating shafts 8 and 9 and within a gear box a gear 12, 13, respectively, including bevel gearing wheels. Drive shafts 14 and 15 are connected to the gears 12 and 13 and being commonly driven through a bevelled gear transmission 16 by a driving shaft 17 which is rotated by a drive system 18. The drive system 18 is controlled by a control device 21.

In Fig. 2, four positions of the revolving doors 10 and 11 are illustrated. Interrupted lines illustrate the free passage space 22, 23, 24, 25, respectively, through the passage lock. In front of the passage lock there is a movement sensor activating the drive of the revolving doors as soon as a person approaches the passage lock. Contrary to conventional revolving doors in which two revolving doors are positioned perpendicularly and include a common rotating axis, the revolving door lock according to the invention does not offer only a cylindrical segment of 90° as passage space, but a longer area, which better enables the passage with a trolley, in a wheel chair or with suitcases.

The revolving doors preferably move at one revolution per minute in idle operation in which the drive is switched off when there is a rest position without movement sensing. In operation, the number of revolutions per minute is preferably five and after circa 15 seconds without sensing movements there is a switch to the idle operation.

The dotted area 23 according to Fig. 2b illustrates that already in this position the passage space is wide open. In the position of Fig. 2c, the entrance door 10 is closing, wherein, however, the passage is not urged from behind contrary to conventional revolving crosses. In the position of Fig. 2d, the entrance to the interior of the building is released and one can quickly continue its passage and no urge from behind is created.

In Fig. 3 the revolving door lock is shown in a position of the revolving doors 10 and 11 as can be used for example in the case of an emergency. Both revolving doors 10 and 11 are hingedly supported in which both wings 26, 27 and

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28, 29, respectively, of both revolving doors 10 and 11 may be hinged towards each other. All four wings of both revolving doors 10 and 11 are then aligned so that there is created a maximum passage space 6, 7 through the revolving door lock.

5 In the hinged bearing support of the revolving doors 10 and 11, the wings 26, 27 and 28, 29, respectively, are pivotally attached to the respective shafts 8 and 9 so that they can be relatively hinged after they are unlocked.

In Fig. 4, the passage lock is shown in its closed
10 position. Then, both wings 26 and 27 of the outer revolving door 10 are hinged outwardly and are closed to the ends 31 and 32 of both outer walls 2 and 3. The revolving door 11 is then preferably in a position perpendicular to the building wall 33. The passage spaces 6 and 7 of the revolving door lock 1
15 may then be used in this position for entrance purposes during the night or in weekends.

The sealing of the ends 34 and 35, and 36 and 37, respectively, of the revolving doors 10 and 11 to the inner side of the outer walls 2, 3, 4 and 5 of the revolving door
20 lock is effected, for example, by means of vertical brushes attached to the revolving door ends. To the outer ends 31, 32 and 38, 39, respectively, of the outer walls 2, 3, 4 and 5 of the revolving door lock, there could be arranged pneumatic or electric safety pressure strips, for example. In principle,
25 all main and secondary closing edges are secured in accordance with the regulations and the state of the art. In case there is too much force active between the ends of the revolving doors and the ends of the outer walls which may cause a risk of injury, the safety pressure strip is activated and the
30 drive system of the revolving doors is switched off. If the wings are hingedly arranged, then also the wing may hinge back by this force and the driving force is interrupted until the wing is hinged back to its original position.

Fig. 5 shows a second embodiment of the revolving
35 door lock according to the invention in which there is arranged an intermediate part 40, 41, respectively, between the outer walls 2, 4 and 3, 5, respectively, so that there is created an intermediate space 42 between the revolving doors 10 and 11. As a result, the rotating axes 8 and 9 of the
40 revolving doors 10 and 11 are positioned at a distance to each

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other which is greater than the sum of the radius R of both revolving doors 10 and 11 of which the radii R are equal in this case. Due to this feature, the paths of both revolving doors 10 and 11 do not overlap as in the first embodiment of Fig. 1-4, and there is even created an intermediate space enabling a person to wait within the intermediate space 42 before continuing its way. In a passage lock having revolving doors the diameter of which being 3 m, the intermediate space between the ends of the door wings may for example be 0.75 m in passages for persons and 1.5 m in passages for persons having shopping trolleys.

Furthermore, each revolving door 10, 11 in this embodiment has its own drive motor 43, 44 which are electronically coupled through a synchronizing device 45 ensuring the synchronized rotation of the revolving doors 10, 11. Such embodiment in which the synchronized rotation of the revolving doors 10, 11 is obtained through a synchronizing device has the important advantage that each drive motor 43, 44 drives a smaller mass resulting in an enhanced safety due to a smaller mass inertia.

A substantially simplified block diagram of the synchronizing device 45 is shown in Fig. 7. The synchronizing device 45 includes a central processing unit 55 controlling the drive motors 43 and 44. Each drive motor 43, 44 comprises a respective so-called incremental angular encoder 56, 57, respectively, which is coupled directly to the shaft of the drive motor and which delivers a certain number of pulses for each revolution of the motor shaft. The drive motors 43, 44 drive the rotating shafts 8, 9 through a reduction gear 58, 59 so that, by means of an angular encoder having a relatively small resolution, nonetheless a great number of pulses per revolution of the revolving door and consequently a high resolution for determining the position of the revolving door is obtained. The counter position of each angular encoder 56, 57, which counter position indicates the rotational position of the respective revolving door 10, 11, is supplied to the processing unit 55.

Furthermore, a position detector 60, 61 for each revolving door is mounted in a fixed position relative to the respective revolving door 10, 11, respectively. As is shown in



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Fig. 5, the fixed position of the position detector 60 is off-set 90° with respect to the fixed position of the position detector 61. Both position detectors 60, 61 are also connected to the processing unit. Each position detector 60, 61 is able to deliver a signal each 180° or each 360°, dependent on the fact whether one or both door wings comprise for example a magnetic element which is detected by the position detector 60, 61.

Finally, the processing unit 55 is connected to a conventional control unit (not shown) or a revolving door which is adapted a.o. to deliver a rotation command to the processing unit. When the processing unit 55 receives a rotation command, the processing unit activates the drive motors 43, 44 and causes the revolving doors 10, 11 to rotate in a synchronized fashion with a speed adjustable through a control panel 62, by means of the signals of the angular encoders 56, 57 and the position detectors 60, 61. The processing unit 55 resets the counter position of each angular encoder to a starting position, preferably the zero position, each time the respective position detector 60, 61 monitors a door wing 26 or 27 and consequently when a particular rotational position is reached. As a result, the processing unit 55 may obtain the synchronized rotation of the revolving doors 10, 11 by equalizing the counter positions of both angular encoders 56, 57. Maintaining the difference in angular position of 90° between the revolving doors 10, 11 is very simple due to the relative 90° off-set of both position detectors 60, 61.

If the processing unit 55 signalizes a lack of synchronization of the revolving doors 10, 11, the synchronized rotation of the revolving doors is restored by the processing unit by braking the leading revolving door until the revolving doors are synchronized again. Consequently, no unsafe situations will occur as a result of a sudden acceleration of a revolving door.

It will be clear that when there is a lack of synchronization, both the revolving door 10 and the revolving door 11 may be the leading revolving door. In the synchronizing device 45 there is no master-slave situation which is common in synchronizing systems. Both revolving doors

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10, 11 can be master and slave for the synchronizing device.

As in normal operation the processing unit 55 receives signals both from the angular encoders 56, 57 and from the position detectors 60, 61, the processing unit 55 can easily check the proper operation of the angular encoders 56, 57 and the position detectors 60, 61. If, after a full revolution, no signal is received from a position detector 60, 61, the processing unit 55 delivers an alarm signal through the control panel 62 in which, for example by means of individual LED's (not shown), the defective position detector is indicated. The processing unit 55 keeps the drive motors 43, 44 in operation and strives to maintain the synchronized rotation by means of the signals of the angular encoders 56, 57. For reasons of safety, the processing unit 55 may select a lower rotational speed, for example.

If no signal from an angular encoder 56, 57 is received, the processing unit 55 also delivers an alarm signal through the control panel 62 in which the defective angular encoder 56, 57 is indicated by a LED. In case of a defective angular encoder 56, 57, the processing unit 55 may strive to maintain a synchronized rotation by means of the position detectors 60, 61. Alternatively, it is possible to stop the drive motor of the defective angular encoder 56, 57 and to release the respective revolving door to allow for manual operation.

If the synchronizing device 45 is connected to the voltage of the supply line for the first time or if the processing unit 55 is reset, the processing unit 55 controls the drive motors 43, 44 such that the revolving doors 10, 11 are rotated to the particular position determined by the respective position detector 60, 61 at a creeping speed to determine the starting positions of the angular encoders 56, 57. Then, the revolving doors 10, 11 are accelerated and decelerated again for a short time, enabling the processing unit 55 to determine from the motor current the mass driven by the drive motors. With this information, the processing unit 55 is able to limit the maximum torque of the revolving doors 10, 11 to a value laid down in safety regulations.

The synchronizing device 45 further includes an emergency stop device 63 directly connected to the drive



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motors 43, 44 and stopping and short-circuiting the drive motors in case of an emergency stop.

By using the synchronizing device 45 including a drive motor 43, 44 for each revolving door 10, 11 it is possible to simply set the revolving doors in certain particular positions. In Fig. 5, for example, interrupted lines indicate a so-called summer position of the revolving door lock 1 in which the revolving doors 10, 11 are positioned individually and aligned with each other along the longitudinal axis of the revolving door lock 1. The doors 10, 11 not only open the passage but also simultaneously form a separation of the entrance and exit paths. This position may also be used for the transport of goods occupying a lot of space.

It is also possible to select a night position through the control panel 62, in which the revolving door 11 is in its summer position indicated by an interrupted line, while the revolving door 10 is perpendicular to the summer position, for example.

Fig. 6 shows a further embodiment in which the revolving door lock 1 comprises three revolving doors, that is a single outer revolving door 46 and two inner revolving doors 47 and 48. Between the revolving doors there is a central passage space 49 defined by the revolving doors 46-48 and by the outer walls 50. Such revolving door lock may for example be used for the entrance and exit of two interior spaces separated by a partition wall 51, and it is also possible that the revolving door 46 having a large capacity is used for both entrance and exit, while the smaller revolving doors 47 and 48 are used as entrance on the one hand and as exit on the other hand. In this embodiment, the vertical planes of symmetry 52, 53 and 54 in the longitudinal directions of the revolving doors 46, 47, 48, respectively, are not parallel to each other. Nonetheless, the angles β of the revolving doors 47 and 48 to their respective longitudinal plane 53, 54 are off-set 90° with respect to the angle α of the revolving door 46 with respect to its longitudinal plane 52. In this way it is ensured that either the revolving door 46 or the revolving doors 47 and 48 are in their closed position thereby preventing draught through the revolving door lock. Of course,

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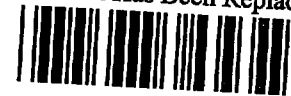
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also other arrangements of a larger or smaller number of revolving doors may be selected within a revolving door lock depending on the respective situation.

- 5 The invention is not restricted to the embodiments shown in the drawing and described herein before which may be varied in different manners within the scope of the invention defined in the claims.

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CLAIMS

1. Revolving door lock having outer walls laterally defining a passage space, characterized by at least two revolving doors moving relatively synchronized and having two wings rotatable about respective vertical rotating axes, said
5 wings of the different revolving doors have such relative angular position that the passage space is always closed at least on one side.
2. Revolving door lock according to claim 1, wherein each revolving door comprises two aligned wings on both sides
10 of the respective rotating axis.
3. Revolving door lock according to claim 1 or 2, wherein two revolving doors are provided, of which the angles to the respective line of symmetry in longitudinal direction have an angular off-set of 90° .
- 15 4. Revolving door lock according to one of the preceding claims, wherein the rotating axes of the revolving doors are positioned at a distance which is greater than the sum of the radius of both revolving doors.
5. Revolving door lock according to claim 4, wherein
20 the distance between the rotating axes of equally dimensioned revolving doors is at least 2.25 times the radius thereof.
6. Revolving door lock according to claim 1 or 2, wherein three or more revolving doors are provided, being positioned in the outer walls of a larger passage space.
- 25 7. Revolving door lock according to one of claims 2-6, wherein the wings of the revolving doors are relatively pivotable back and forth about the respective rotating axis.
8. Revolving door lock according to one of the preceding claims, wherein the drive of the revolving doors
30 takes place by means of a single motor being connected to the revolving door shafts by a mechanical coupling.
9. Revolving door lock according to one of claims 1-7, wherein each revolving door is equipped with its own drive motor and the individual drive motors being coupled by an
35 electronic synchronizing device.
10. Revolving door lock according to claim 9, characterized in that the synchronizing device for each

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revolving door comprises an angular encoder delivering a certain number of pulses for each revolution of the revolving door so that a counter position indicates the rotational position of the revolving door, and a position detector for detecting at least one particular revolving position of the revolving door, wherein a processing unit is provided to which the angular encoder and the position detectors are connected and which controls the drive motors, said processing unit resets the counter position of the angular encoder to a starting position when the respective position detector signalizes that the particular rotating position is reached, and said processing device being adapted to allow the revolving doors to move with a synchronized adjustable speed by means of the signals of the angular encoder and position detectors.

11. Revolving door lock according to claim 10, characterized in that the position detector on the one side of the revolving door lock is arranged in a fixed position relative to the respective revolving door, which is rotated 90° with respect to the fixed position taken by the position detector(s) on the other side of the revolving door lock relative to the respective revolving door, and the processing unit controlling the drive motors such that the angular encoders indicate the same revolving door position.

12. Revolving door lock according to claim 10 or 11, characterized in that, if the revolving doors are not synchronized, the processing unit brakes the leading revolving door to restore the synchronization.

13. Revolving door lock according to claim 10, 11 or 12, characterized in that the processing unit monitors the presence of the signals of the angular encoder and the position detectors and delivers an alarm signal if a position detector does not deliver a signal after a revolution of the revolving door, wherein the processing unit preferably maintains a synchronized rotation of the revolving doors at a lower speed by means of the signals of the angular encoders.

14. Revolving door lock according to claim 13, characterized in that the processing unit also delivers an alarm signal if the signal of an angular encoder disappears, wherein preferably at least the respective revolving door is

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stopped and released.

15. Revolving door lock according to one of claims 10-14, characterized in that, upon switching to the voltage of the supply line or upon resetting the processing unit, the processing unit controls the drive motors such that the revolving doors are moved to the rotational position determined by the position detector at a creeping speed, wherein, after reaching this rotational position, the processing unit accelerates and decelerates the revolving doors for a short time to determine the mass driven by the drive motor.

16. Revolving door lock according to one of claims 10-15, characterized in that an emergency stop device is provided being directly connected to the drive motors and short-circuiting them in case of an emergency stop.

17. Revolving door lock according to one of claims 10-16, characterized in that each angular encoder is coupled directly to the shaft of the drive motor and the drive motor is driving the revolving door through a reduction gear.

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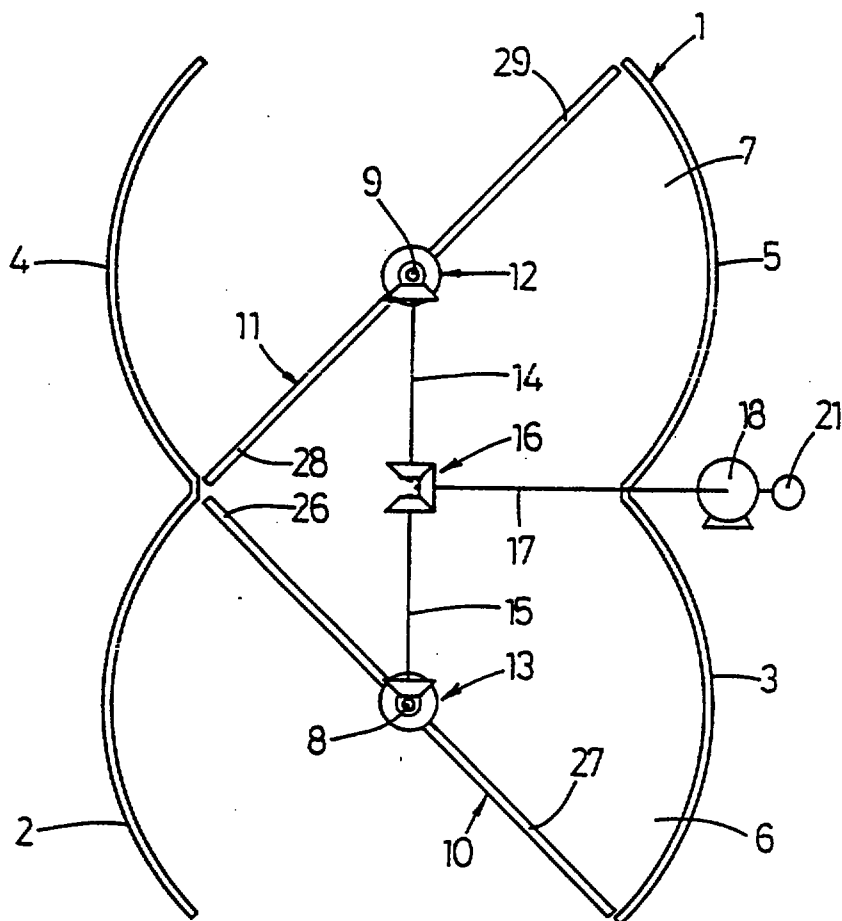


Fig.1

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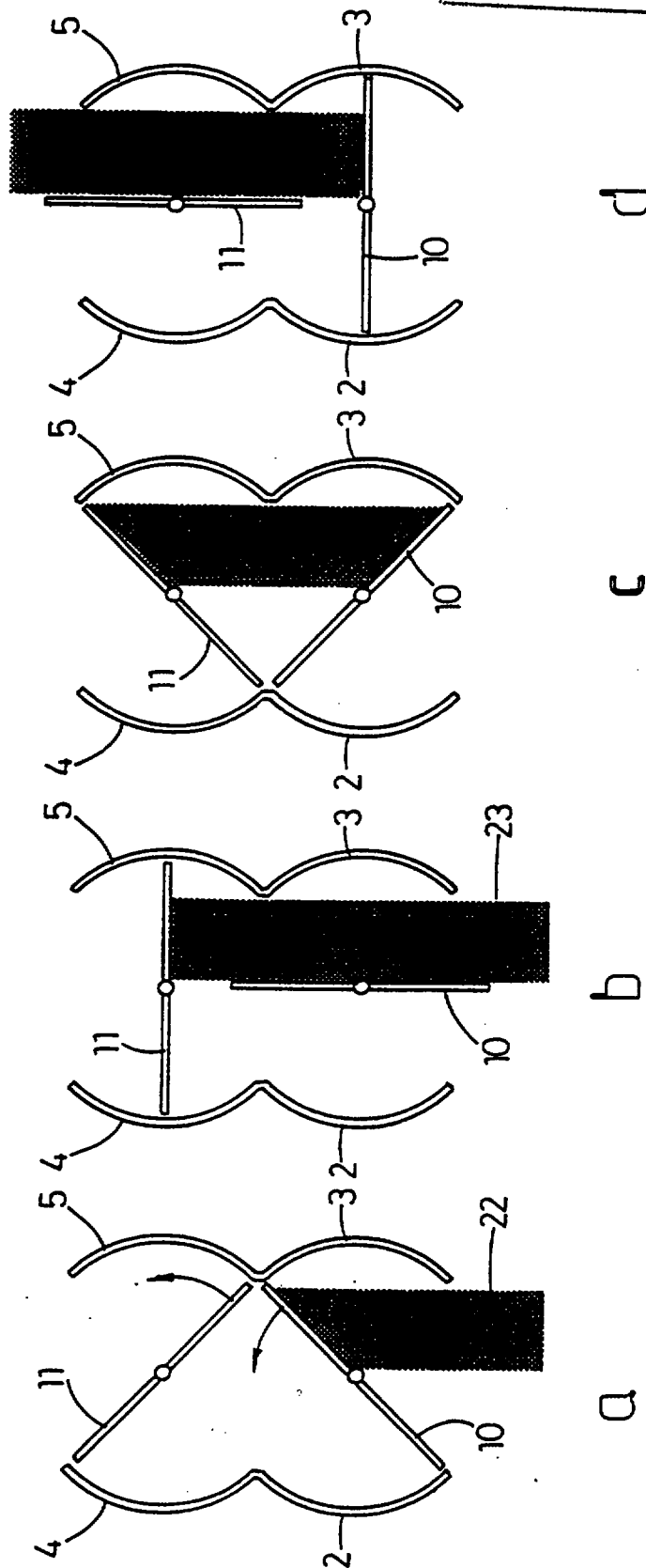


Fig. 2

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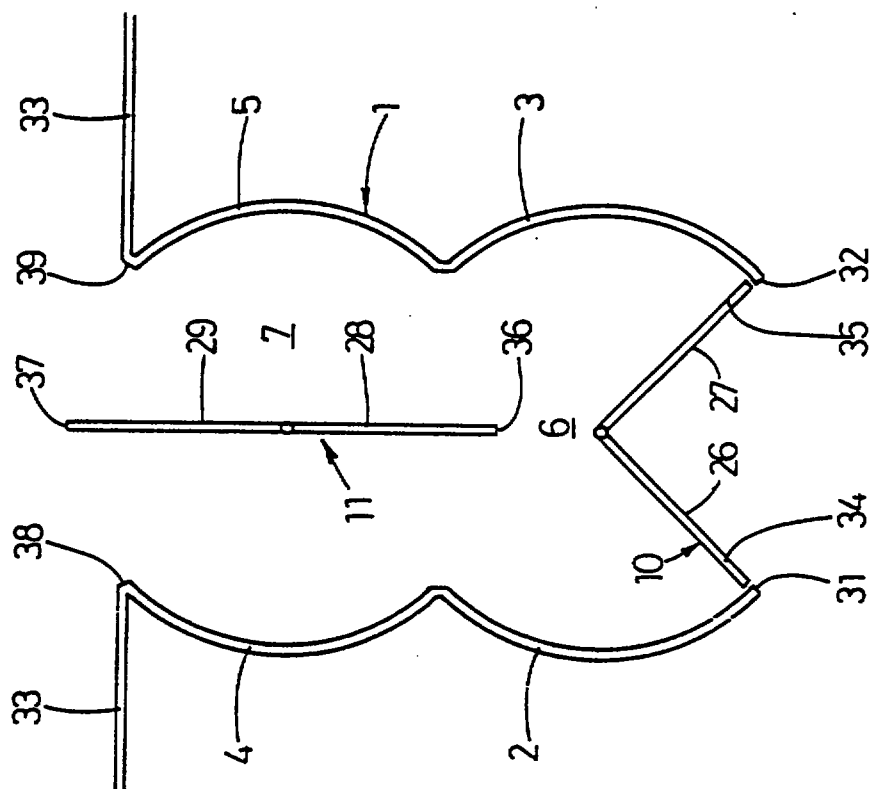


Fig. 4

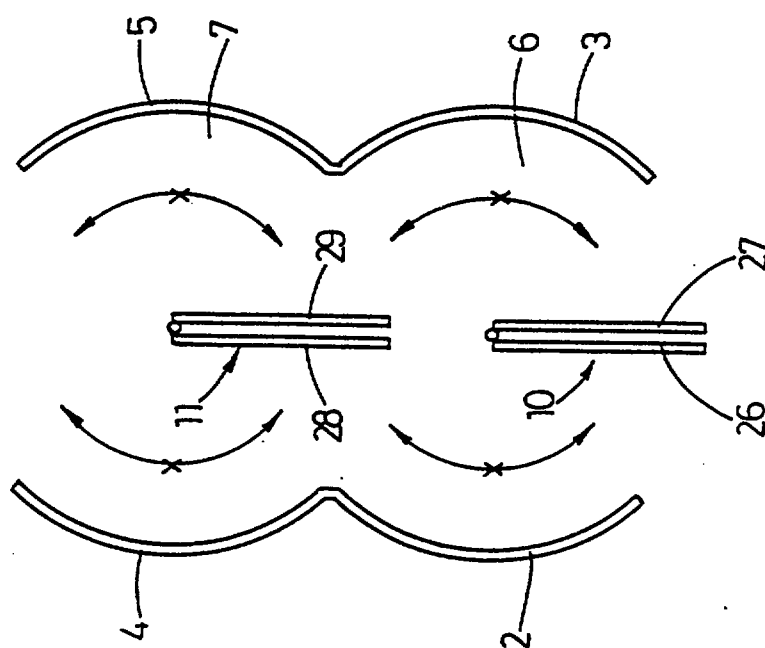


Fig. 3

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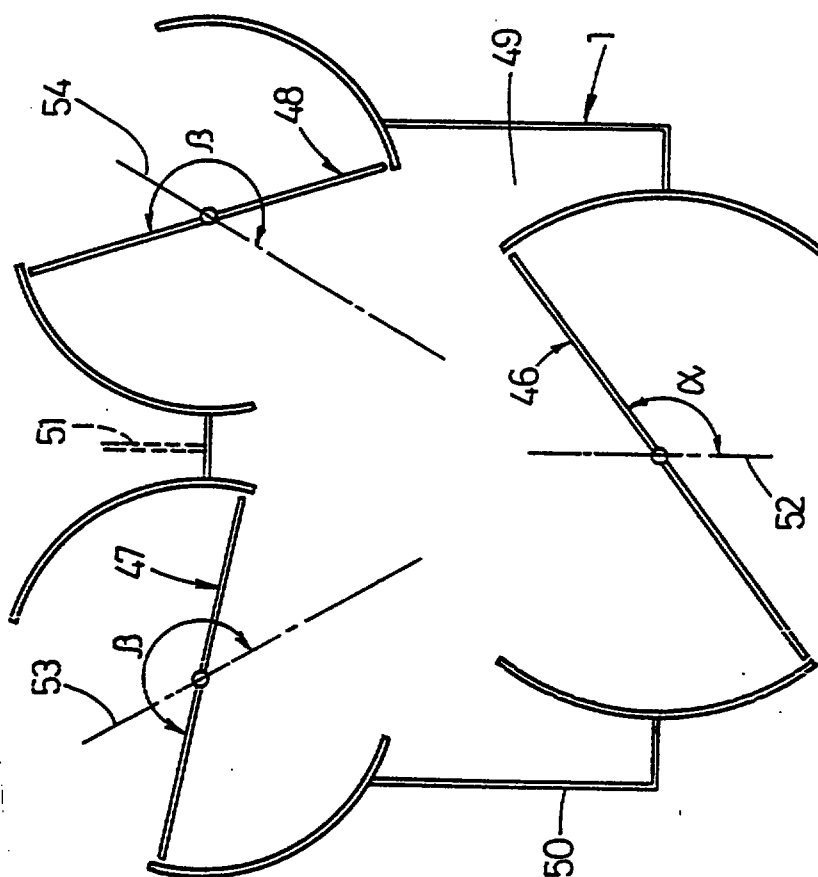
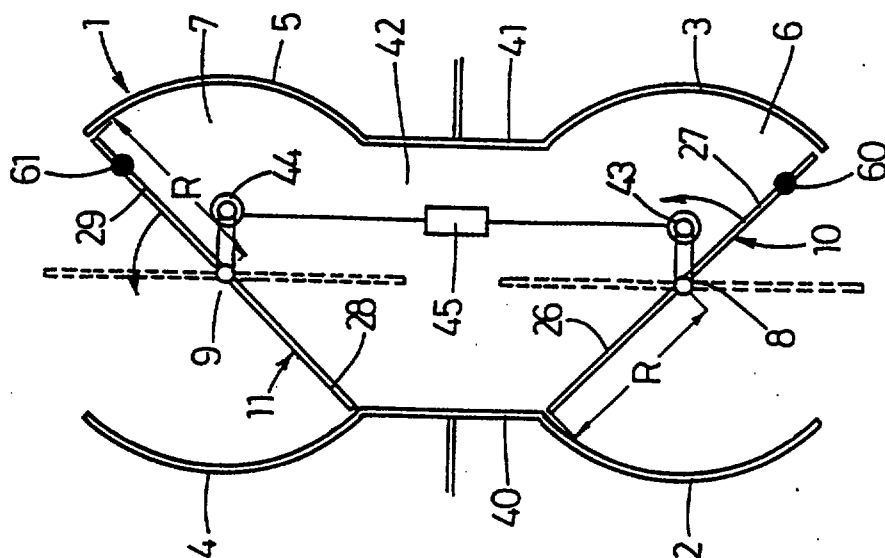


Fig. 6



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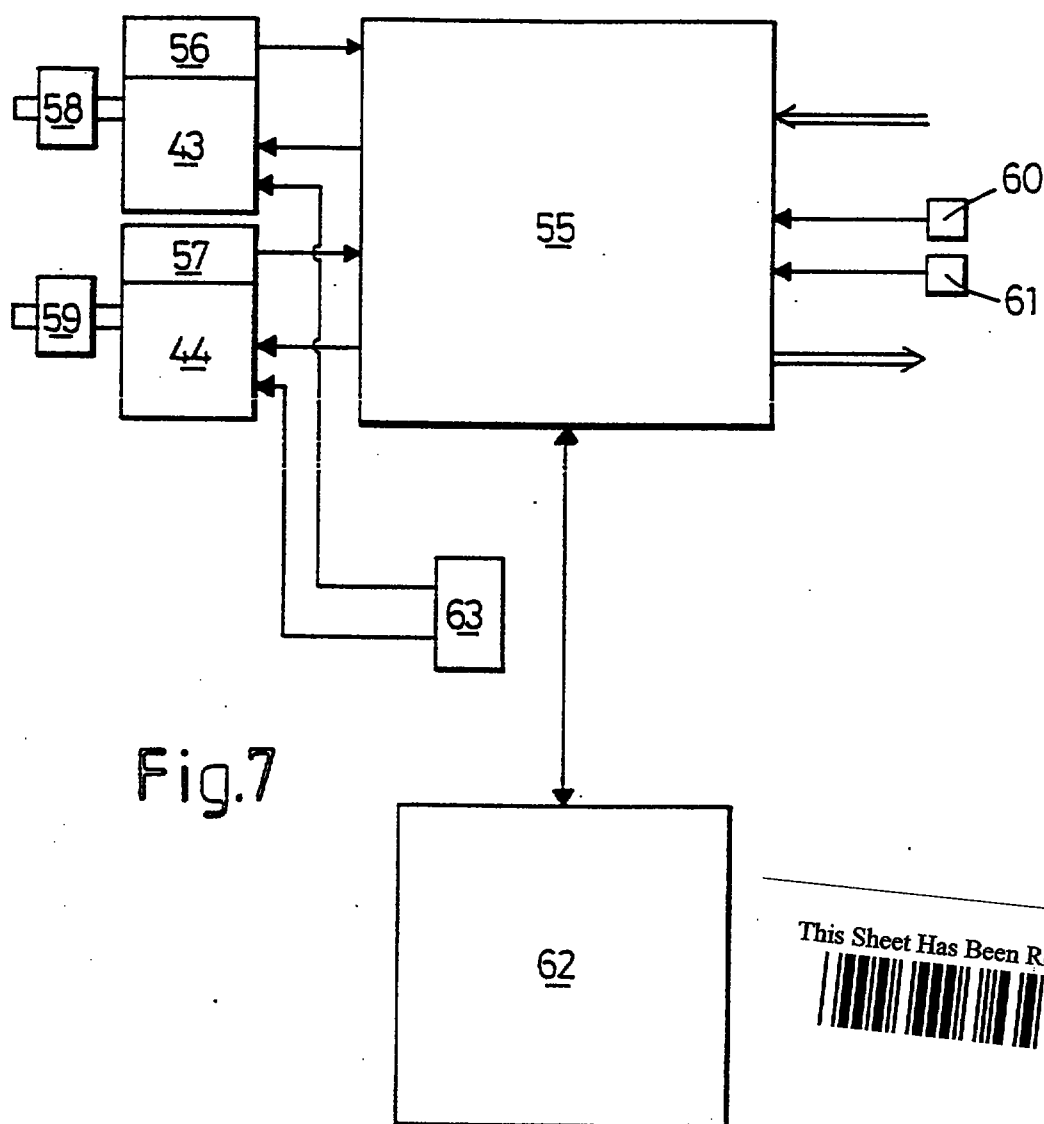


Fig.7

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Revolving door lock

The present invention relates to a revolving door lock having outer walls laterally defining a passage space.

In the present times of energy saving it is important to seal off entrances of buildings as good as possible so that
5 no heating power disappears from the building. The inner space of the building should also be kept free of draught, which is particularly important in restaurants or hotels, shops and main entrances of commercial properties, hospitals and the like. Until now, this object has been obtained by so-called
10 revolving cross doors, in which three or four door wings define three or four cylinder segments having an angle at the centre of 120° or 90° respectively. With these revolving cross doors, in particular with the four winged revolving door, the passage space is very narrow so that it is for example not
15 possible to enter the building entrance with a trolley or a wheel chair, and the passage with suitcases is difficult. A further disadvantage of these prior art doors is that all persons passing through the revolving cross doors should go equally fast so that old or handicapped people could become
20 pinched due to a faster person which is also in the area of the revolving cross door, or due to a motorized revolving cross door being driven too fast. Motorized revolving cross doors having a lower rotational speed have a reduced capacity, however, and prevent a quick passage through the revolving
25 door lock.

The object of the present invention is to provide a revolving door lock which does not have said disadvantages of the prior art revolving cross doors.

For obtaining said object, the revolving door lock
30 according to the invention is characterized by at least two revolving doors moving relatively synchronized and having two wings rotatable about respective vertical rotating axes, said wings of the different revolving doors have such relative angular position that the passage space is always closed at
35 least on one side.

The revolving door lock according to the invention effects that, as a result, in each position of the revolving doors the inner space of the building is sealed to the outside

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so that no heat energy is lost for nothing and hardly no draught appears. On the other hand, there is no need for three or four wings at the individual revolving doors so that with each revolving door more space is available for the passage with trolleys, wheel chairs, luggage and the like.

Furthermore, the revolving door lock can be passed quickly, in that at each revolving door a free passage is possible since only one of the doors should effect a seal at a time, while the other one can be fully open. As a result, several persons may pass through the revolving door lock in their own speed without interfering with each other.

A maximum passage space at the revolving doors is created if each revolving door comprises two aligned wings on both sides of the respective rotating axis. Then, a proper seal through the revolving door lock is always effected when two revolving doors are provided of which the angles to their own line of symmetry in longitudinal direction have a relative angular off-set of 90° .

A favourable embodiment of the revolving door lock according to the invention is characterized in that the rotating axes of the revolving doors are positioned at a distance which is greater than the sum of the radius of both revolving doors, and preferably greater than 2.25 times the radius.

Due to this feature it is possible to create an intermediate space between the revolving doors offering persons passing through the revolving door lock the opportunity to stop between the revolving doors. This is of great importance to the safety as people could panic at the transfer between the revolving doors because they do not know how to walk and could then be hit by the door wings. According to the invention the people are able to reorientate in the intermediate space and only then continue on their way. With a distance of at least ± 0.4 m and preferably 0.75 m between the ends of the door wings there is created enough space for persons, while for example in supermarkets where people pass through the revolving door lock with shopping trolleys a room of at least 1.5 m provides a place to stop. This intermediate space also enables a continued passage between the revolving doors to both quickly and slowly passing people which may



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enter the next revolving door just before or just after the passage of a door wing. On the other hand, the feature offers new arrangement possibilities of the revolving doors. It is for instance possible to connect three or more revolving doors to the intermediate or passage space, which are adapted to each other and together effect the sealing of the inner space of the building.

Embodiments of the invention by way of example are elucidated hereafter with reference to the attached drawings.

10 In the drawings:

Fig. 1 is a very schematic plan view of a first embodiment of the revolving door lock and drive,

Fig. 2 shows four different positions of the revolving doors within the revolving door lock of Fig. 1,

15 Fig. 3 shows the revolving door lock of Fig. 1 with the revolving doors in an emergency position,

Fig. 4 shows the revolving door lock of Fig. 1 with the outer revolving doors in the night position,

20 Fig. 5 is a schematic plan view of a second embodiment of the revolving door lock according to the invention illustrating a so-called summer position with interrupted lines,

25 Fig. 6 is a very schematic plan view of a third embodiment of the revolving door lock according to the invention,

Fig. 7 is a substantially simplified block diagram of a synchronizing device of the embodiment of Fig. 5.

In the various figures similar parts are indicated by the same reference numerals.

30 The revolving door lock 1 according to Fig. 1-4 includes four outer walls 2, 3, 4, 5 each forming a part of a cylindrical casing, the outer walls 2 and 3 for example forming an entrance or exit space 6 on the side of the street and the outer walls 4 and 5 defining an entrance or exit space 7 on the inner side of a building. The pairs of outer walls 2, 3 and 4, 5, respectively, are diametrically opposed and extend through an angle of 90°. The vertical central shafts 8 and 9 of the space 6, 7 form rotating axes for the revolving doors 10 and 11. Each revolving door 10 and 11 consists of two door wings 26, 27 and 28, 29, respectively, positioned on both

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sides of the rotating axis 8, 9 and each pair being mutually aligned. Of course, the door wings 26, 27 and 28, 29 could be integral. At the upper side of the revolving doors 10 and 11 there is disposed on the respective rotating shafts 8 and 9 and within a gear box a gear 12, 13, respectively, including bevel gearing wheels. Drive shafts 14 and 15 are connected to the gears 12 and 13 and being commonly driven through a bevelled gear transmission 16 by a driving shaft 17 which is rotated by a drive system 18. The drive system 18 is controlled by a control device 21.

In Fig. 2, four positions of the revolving doors 10 and 11 are illustrated. Interrupted lines illustrate the free passage space 22, 23, 24, 25, respectively, through the passage lock. In front of the passage lock there is a movement sensor activating the drive of the revolving doors as soon as a person approaches the passage lock. Contrary to conventional revolving doors in which two revolving doors are positioned perpendicularly and include a common rotating axis, the revolving door lock according to the invention does not offer only a cylindrical segment of 90° as passage space, but a longer area, which better enables the passage with a trolley, in a wheel chair or with suitcases.

The revolving doors preferably move at one revolution per minute in idle operation in which the drive is switched off when there is a rest position without movement sensing. In operation, the number of revolutions per minute is preferably five and after circa 15 seconds without sensing movements there is a switch to the idle operation.

The dotted area 23 according to Fig. 2b illustrates that already in this position the passage space is wide open. In the position of Fig. 2c, the entrance door 10 is closing, wherein, however, the passage is not urged from behind contrary to conventional revolving crosses. In the position of Fig. 2d, the entrance to the interior of the building is released and one can quickly continue its passage and no urge from behind is created.

In Fig. 3 the revolving door lock is shown in a position of the revolving doors 10 and 11 as can be used for example in the case of an emergency. Both revolving doors 10 and 11 are hingedly supported in which both wings 26, 27 and



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28, 29, respectively, of both revolving doors 10 and 11 may be hinged towards each other. All four wings of both revolving doors 10 and 11 are then aligned so that there is created a maximum passage space 6, 7 through the revolving door lock.

5 In the hinged bearing support of the revolving doors 10 and 11, the wings 26, 27 and 28, 29, respectively, are pivotally attached to the respective shafts 8 and 9 so that they can be relatively hinged after they are unlocked.

10 In Fig. 4, the passage lock is shown in its closed position. Then, both wings 26 and 27 of the outer revolving door 10 are hinged outwardly and are closed to the ends 31 and 32 of both outer walls 2 and 3. The revolving door 11 is then preferably in a position perpendicular to the building wall 33. The passage spaces 6 and 7 of the revolving door lock 1
15 may then be used in this position for entrance purposes during the night or in weekends.

The sealing of the ends 34 and 35, and 36 and 37, respectively, of the revolving doors 10 and 11 to the inner side of the outer walls 2, 3, 4 and 5 of the revolving door
20 lock is effected, for example, by means of vertical brushes attached to the revolving door ends. To the outer ends 31, 32 and 38, 39, respectively, of the outer walls 2, 3, 4 and 5 of the revolving door lock, there could be arranged pneumatic or electric safety pressure strips, for example. In principle,
25 all main and secondary closing edges are secured in accordance with the regulations and the state of the art. In case there is too much force active between the ends of the revolving doors and the ends of the outer walls which may cause a risk of injury, the safety pressure strip is activated and the
30 drive system of the revolving doors is switched off. If the wings are hingedly arranged, then also the wing may hinge back by this force and the driving force is interrupted until the wing is hinged back to its original position.

Fig. 5 shows a second embodiment of the revolving
35 door lock according to the invention in which there is arranged an intermediate part 40, 41, respectively, between the outer walls 2, 4 and 3, 5, respectively, so that there is created an intermediate space 42 between the revolving doors 10 and 11. As a result, the rotating axes 8 and 9 of the
40 revolving doors 10 and 11 are positioned at a distance to each

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other which is greater than the sum of the radius R of both revolving doors 10 and 11 of which the radii R are equal in this case. Due to this feature, the paths of both revolving doors 10 and 11 do not overlap as in the first embodiment of Fig. 1-4, and there is even created an intermediate space enabling a person to wait within the intermediate space 42 before continuing its way. In a passage lock having revolving doors the diameter of which being 3 m, the intermediate space between the ends of the door wings may for example be 0.75 m in passages for persons and 1.5 m in passages for persons having shopping trolleys.

Furthermore, each revolving door 10, 11 in this embodiment has its own drive motor 43, 44 which are electronically coupled through a synchronizing device 45 ensuring the synchronized rotation of the revolving doors 10, 11. Such embodiment in which the synchronized rotation of the revolving doors 10, 11 is obtained through a synchronizing device has the important advantage that each drive motor 43, 44 drives a smaller mass resulting in an enhanced safety due to a smaller mass inertia.

A substantially simplified block diagram of the synchronizing device 45 is shown in Fig. 7. The synchronizing device 45 includes a central processing unit 55 controlling the drive motors 43 and 44. Each drive motor 43, 44 comprises a respective so-called incremental angular encoder 56, 57, respectively, which is coupled directly to the shaft of the drive motor and which delivers a certain number of pulses for each revolution of the motor shaft. The drive motors 43, 44 drive the rotating shafts 8, 9 through a reduction gear 58, 59 so that, by means of an angular encoder having a relatively small resolution, nonetheless a great number of pulses per revolution of the revolving door and consequently a high resolution for determining the position of the revolving door is obtained. The counter position of each angular encoder 56, 57, which counter position indicates the rotational position of the respective revolving door 10, 11, is supplied to the processing unit 55.

Furthermore, a position detector 60, 61 for each revolving door is mounted in a fixed position relative to the respective revolving door 10, 11, respectively. As is shown in

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Fig. 5, the fixed position of the position detector 60 is off-set 90° with respect to the fixed position of the position detector 61. Both position detectors 60, 61 are also connected to the processing unit. Each position detector 60, 61 is able to deliver a signal each 180° or each 360°, dependent on the fact whether one or both door wings comprise for example a magnetic element which is detected by the position detector 60, 61.

Finally, the processing unit 55 is connected to a conventional control unit (not shown) or a revolving door which is adapted a.o. to deliver a rotation command to the processing unit. When the processing unit 55 receives a rotation command, the processing unit activates the drive motors 43, 44 and causes the revolving doors 10, 11 to rotate in a synchronized fashion with a speed adjustable through a control panel 62, by means of the signals of the angular encoders 56, 57 and the position detectors 60, 61. The processing unit 55 resets the counter position of each angular encoder to a starting position, preferably the zero position, each time the respective position detector 60, 61 monitors a door wing 26 or 27 and consequently when a particular rotational position is reached. As a result, the processing unit 55 may obtain the synchronized rotation of the revolving doors 10, 11 by equalizing the counter positions of both angular encoders 56, 57. Maintaining the difference in angular position of 90° between the revolving doors 10, 11 is very simple due to the relative 90° off-set of both position detectors 60, 61.

If the processing unit 55 signalizes a lack of synchronization of the revolving doors 10, 11, the synchronized rotation of the revolving doors is restored by the processing unit by braking the leading revolving door until the revolving doors are synchronized again. Consequently, no unsafe situations will occur as a result of a sudden acceleration of a revolving door.

It will be clear that when there is a lack of synchronization, both the revolving door 10 and the revolving door 11 may be the leading revolving door. In the synchronizing device 45 there is no master-slave situation which is common in synchronizing systems. Both revolving doors



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10, 11 can be master and slave for the synchronizing device.

As in normal operation the processing unit 55 receives signals both from the angular encoders 56, 57 and from the position detectors 60, 61, the processing unit 55 can easily check the proper operation of the angular encoders 56, 57 and the position detectors 60, 61. If, after a full revolution, no signal is received from a position detector 60, 61, the processing unit 55 delivers an alarm signal through the control panel 62 in which, for example by means of individual LED's (not shown), the defective position detector is indicated. The processing unit 55 keeps the drive motors 43, 44 in operation and strives to maintain the synchronized rotation by means of the signals of the angular encoders 56, 57. For reasons of safety, the processing unit 55 may select a lower rotational speed, for example.

If no signal from an angular encoder 56, 57 is received, the processing unit 55 also delivers an alarm signal through the control panel 62 in which the defective angular encoder 56, 57 is indicated by a LED. In case of a defective angular encoder 56, 57, the processing unit 55 may strive to maintain a synchronized rotation by means of the position detectors 60, 61. Alternatively, it is possible to stop the drive motor of the defective angular encoder 56, 57 and to release the respective revolving door to allow for manual operation.

If the synchronizing device 45 is connected to the voltage of the supply line for the first time or if the processing unit 55 is reset, the processing unit 55 controls the drive motors 43, 44 such that the revolving doors 10, 11 are rotated to the particular position determined by the respective position detector 60, 61 at a creeping speed to determine the starting positions of the angular encoders 56, 57. Then, the revolving doors 10, 11 are accelerated and decelerated again for a short time, enabling the processing unit 55 to determine from the motor current the mass driven by the drive motors. With this information, the processing unit 55 is able to limit the maximum torque of the revolving doors 10, 11 to a value laid down in safety regulations.

The synchronizing device 45 further includes an emergency stop device 63 directly connected to the drive



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motors 43, 44 and stopping and short-circuiting the drive motors in case of an emergency stop.

By using the synchronizing device 45 including a drive motor 43, 44 for each revolving door 10, 11 it is possible to simply set the revolving doors in certain particular positions. In Fig. 5, for example, interrupted lines indicate a so-called summer position of the revolving door lock 1 in which the revolving doors 10, 11 are positioned individually and aligned with each other along the longitudinal axis of the revolving door lock 1. The doors 10, 11 not only open the passage but also simultaneously form a separation of the entrance and exit paths. This position may also be used for the transport of goods occupying a lot of space.

It is also possible to select a night position through the control panel 62, in which the revolving door 11 is in its summer position indicated by an interrupted line, while the revolving door 10 is perpendicular to the summer position, for example.

Fig. 6 shows a further embodiment in which the revolving door lock 1 comprises three revolving doors, that is a single outer revolving door 46 and two inner revolving doors 47 and 48. Between the revolving doors there is a central passage space 49 defined by the revolving doors 46-48 and by the outer walls 50. Such revolving door lock may for example be used for the entrance and exit of two interior spaces separated by a partition wall 51, and it is also possible that the revolving door 46 having a large capacity is used for both entrance and exit, while the smaller revolving doors 47 and 48 are used as entrance on the one hand and as exit on the other hand. In this embodiment, the vertical planes of symmetry 52, 53 and 54 in the longitudinal directions of the revolving doors 46, 47, 48, respectively, are not parallel to each other. Nonetheless, the angles β of the revolving doors 47 and 48 to their respective longitudinal plane 53, 54 are off-set 90° with respect to the angle α of the revolving door 46 with respect to its longitudinal plane 52. In this way it is ensured that either the revolving door 46 or the revolving doors 47 and 48 are in their closed position thereby preventing draught through the revolving door lock. Of course,

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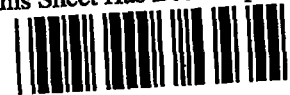
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also other arrangements of a larger or smaller number of revolving doors may be selected within a revolving door lock depending on the respective situation.

The invention is not restricted to the embodiments
5 shown in the drawing and described herein before which may be varied in different manners within the scope of the invention defined in the claims.

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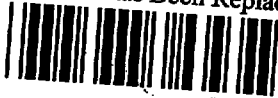
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CLAIMS

1. Revolving door lock having outer walls laterally defining a passage space, characterized by at least two revolving doors moving relatively synchronized and having two wings rotatable about respective vertical rotating axes, said wings of the different revolving doors have such relative angular position that the passage space is always closed at least on one side.

2. Revolving door lock according to claim 1, wherein each revolving door comprises two aligned wings on both sides of the respective rotating axis.

3. Revolving door lock according to claim 1 or 2, wherein two revolving doors are provided, of which the angles to the respective line of symmetry in longitudinal direction have an angular off-set of 90°.

4. Revolving door lock according to one of the preceding claims, wherein the rotating axes of the revolving doors are positioned at a distance which is greater than the sum of the radius of both revolving doors.

5. Revolving door lock according to claim 4, wherein the distance between the rotating axes of equally dimensioned revolving doors is at least 2.25 times the radius thereof.

6. Revolving door lock according to claim 1 or 2, wherein three or more revolving doors are provided, being positioned in the outer walls of a larger passage space.

7. Revolving door lock according to one of claims 2-6, wherein the wings of the revolving doors are relatively pivotable back and forth about the respective rotating axis.

8. Revolving door lock according to one of the preceding claims, wherein the drive of the revolving doors takes place by means of a single motor being connected to the revolving door shafts by a mechanical coupling.

9. Revolving door lock according to one of claims 1-7, wherein each revolving door is equipped with its own drive motor and the individual drive motors being coupled by an electronic synchronizing device.

10. Revolving door lock according to claim 9, characterized in that the synchronizing device for each

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5 revolving door comprises an angular encoder delivering a certain number of pulses for each revolution of the revolving door so that a counter position indicates the rotational position of the revolving door, and a position detector for detecting at least one particular revolving position of the revolving door, wherein a processing unit is provided to which the angular encoder and the position detectors are connected and which controls the drive motors, said processing unit resets the counter position of the angular encoder to a starting position when the respective position detector signalizes that the particular rotating position is reached, and said processing device being adapted to allow the revolving doors to move with a synchronized adjustable speed by means of the signals of the angular encoder and position detectors.

11. Revolving door lock according to claim 10, characterized in that the position detector on the one side of the revolving door lock is arranged in a fixed position relative to the respective revolving door, which is rotated 90° with respect to the fixed position taken by the position detector(s) on the other side of the revolving door lock relative to the respective revolving door, and the processing unit controlling the drive motors such that the angular encoders indicate the same revolving door position.

12. Revolving door lock according to claim 10 or 11, characterized in that, if the revolving doors are not synchronized, the processing unit brakes the leading revolving door to restore the synchronization.

13. Revolving door lock according to claim 10, 11 or 12, characterized in that the processing unit monitors the presence of the signals of the angular encoder and the position detectors and delivers an alarm signal if a position detector does not deliver a signal after a revolution of the revolving door, wherein the processing unit preferably maintains a synchronized rotation of the revolving doors at a lower speed by means of the signals of the angular encoders.

14. Revolving door lock according to claim 13, characterized in that the processing unit also delivers an alarm signal if the signal of an angular encoder disappears, wherein preferably at least the respective revolving door is

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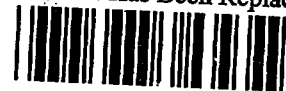
stopped and released.

15. Revolving door lock according to one of claims
10-14, characterized in that, upon switching to the voltage of
the supply line or upon resetting the processing unit, the
5 processing unit controls the drive motors such that the
revolving doors are moved to the rotational position
determined by the position detector at a creeping speed,
wherein, after reaching this rotational position, the
processing unit accelerates and decelerates the revolving
10 doors for a short time to determine the mass driven by the
drive motor.

16. Revolving door lock according to one of claims
10-15, characterized in that an emergency stop device is
provided being directly connected to the drive motors and
15 short-circuiting them in case of an emergency stop.

17. Revolving door lock according to one of claims
10-16, characterized in that each angular encoder is coupled
directly to the shaft of the drive motor and the drive motor
is driving the revolving door through a reduction gear.

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REVOLVING DOOR LOCK

ABSTRACT

A revolving door lock (1) having outer walls (2-5) laterally defining a passage space comprises at least two revolving doors (10, 11) moving relatively synchronized, said revolving doors (10, 11) having two wings (26-29) rotatable about respective vertical rotating axes (8, 9). The wings (26-29) of the individual revolving doors (10, 11) have such angular position relative to each other that the passage space is always closed on at least one side. In one embodiment there are two revolving doors of which the angles to the respective longitudinal line of symmetry are relatively off-set 90° . On the other hand there can be arranged three or more revolving doors being positioned in the outer walls of a larger passage space.

FIGURE 5

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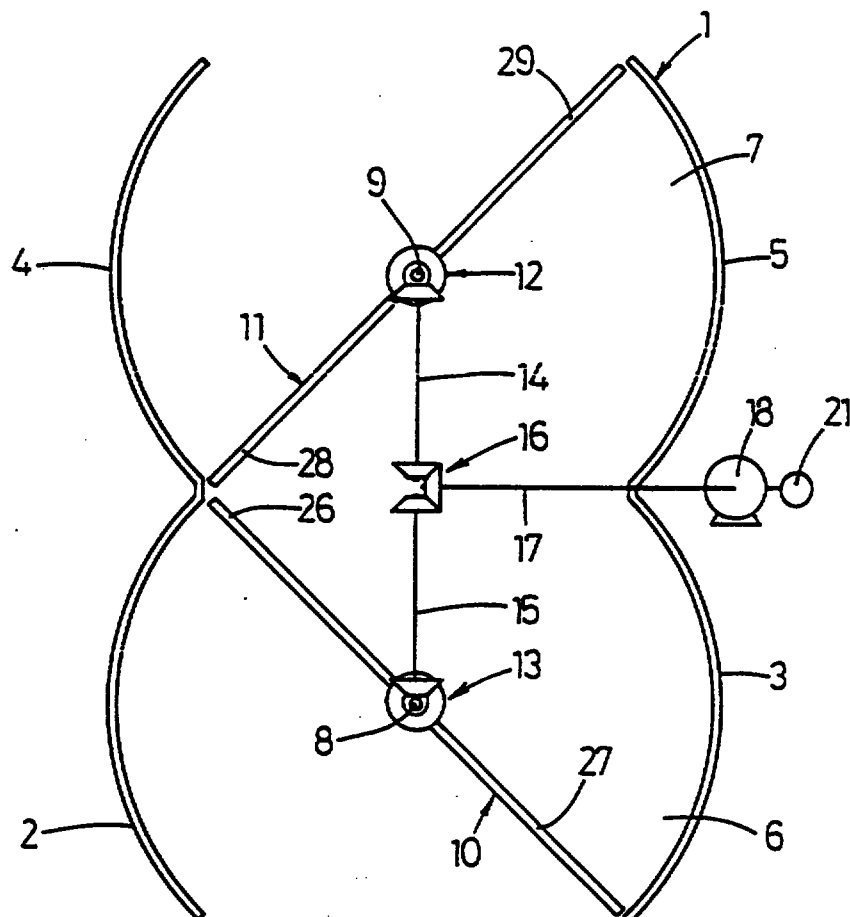


Fig.1

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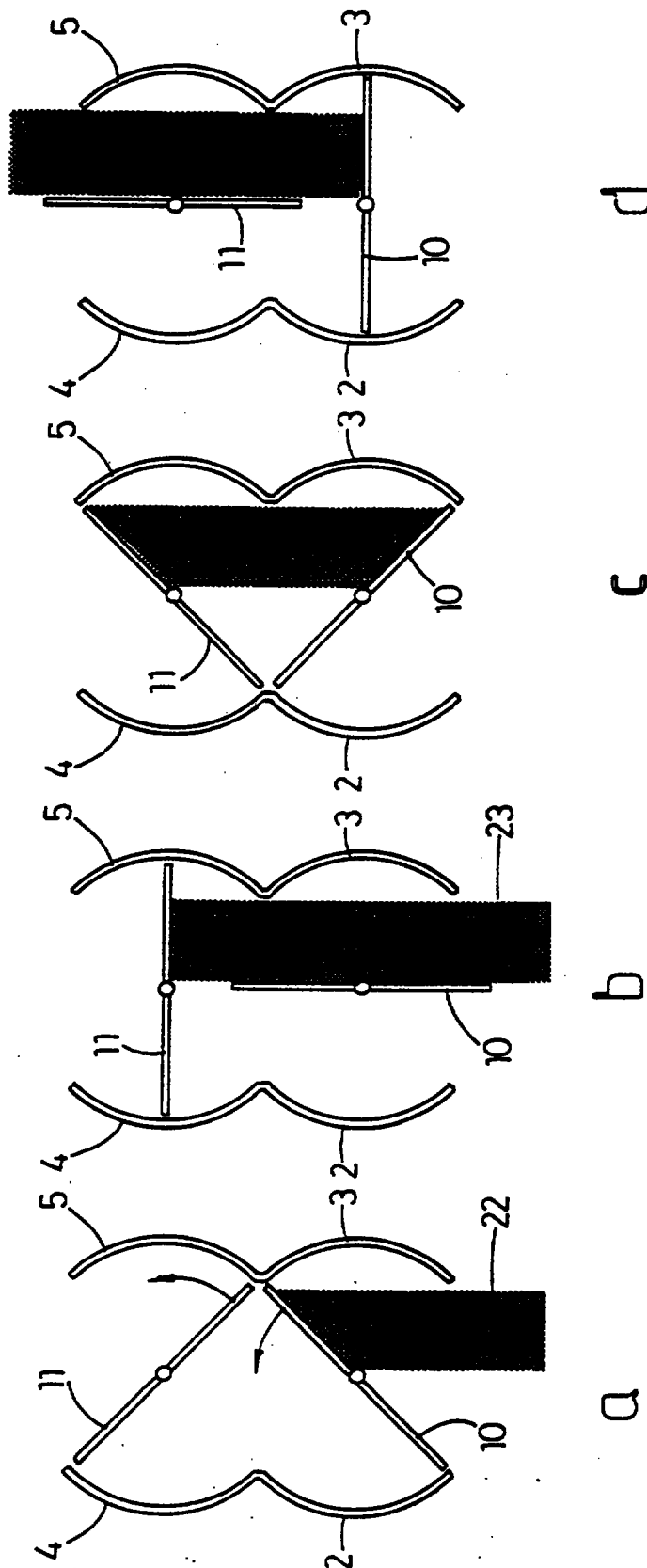


Fig.2

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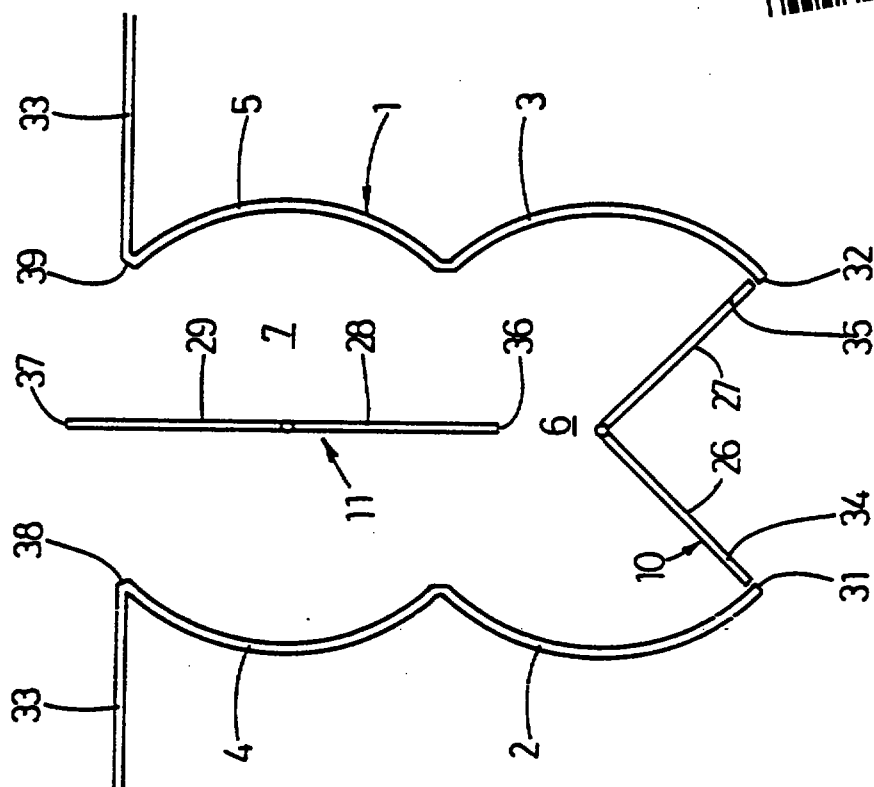


Fig. 4

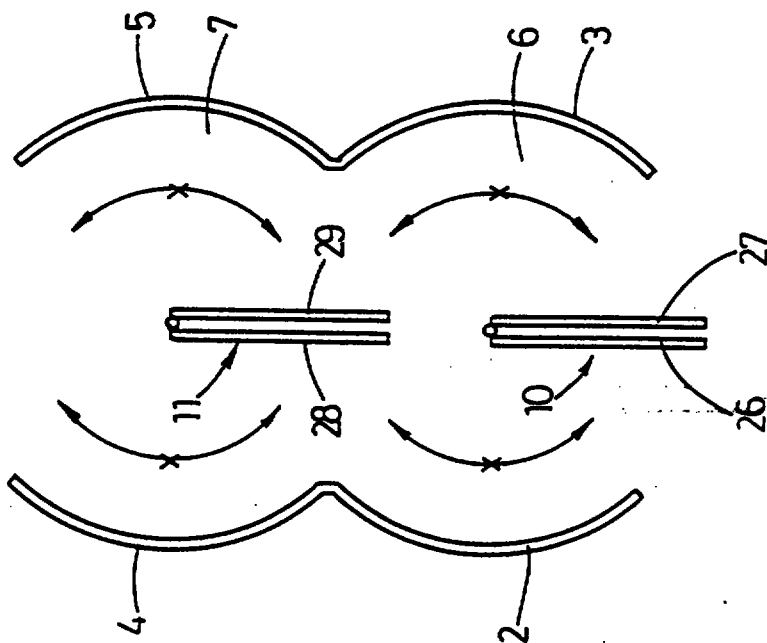


Fig. 3

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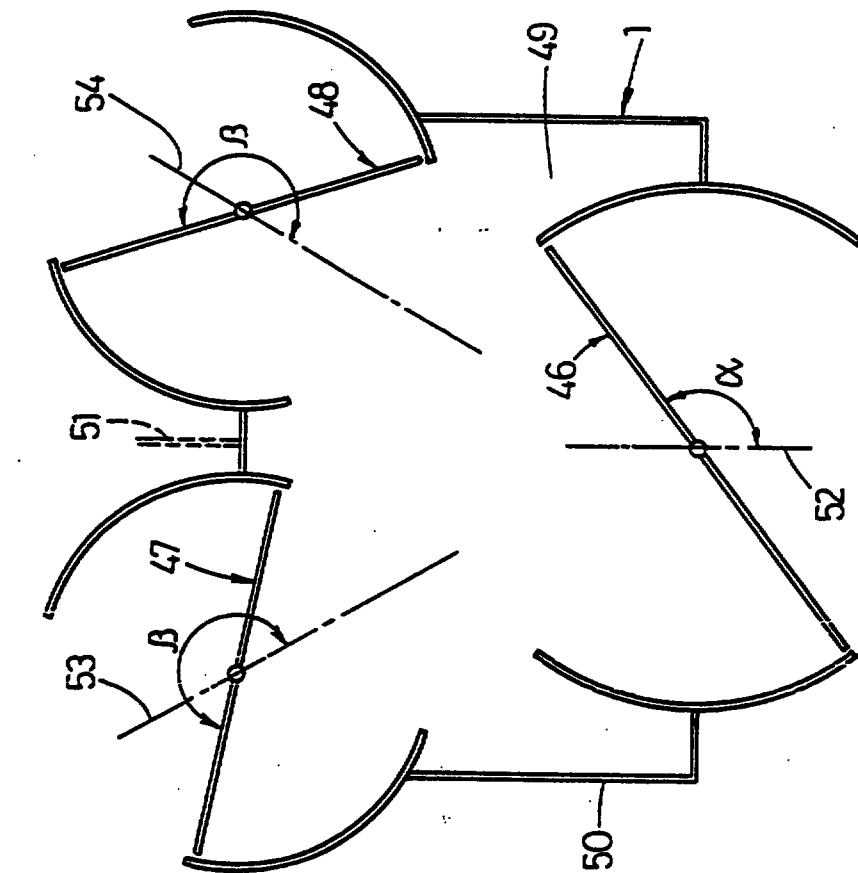


Fig. 6

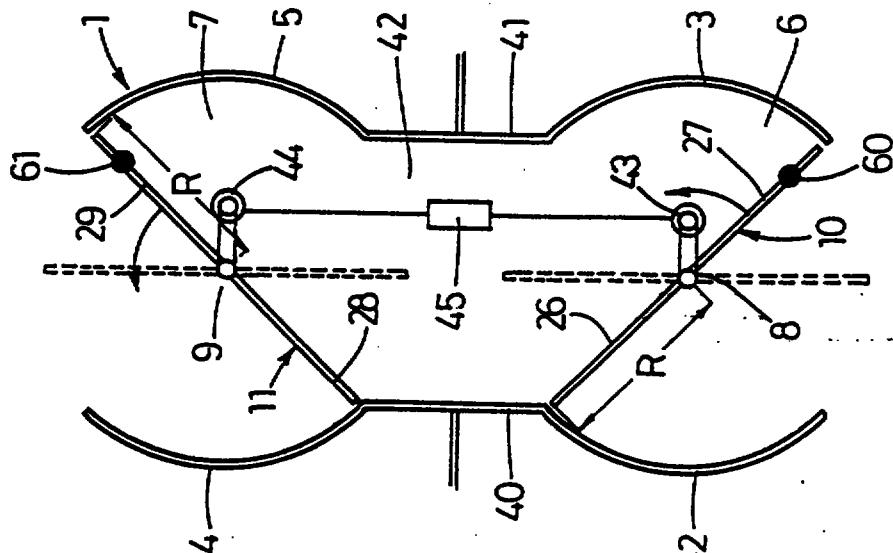


Fig. 5

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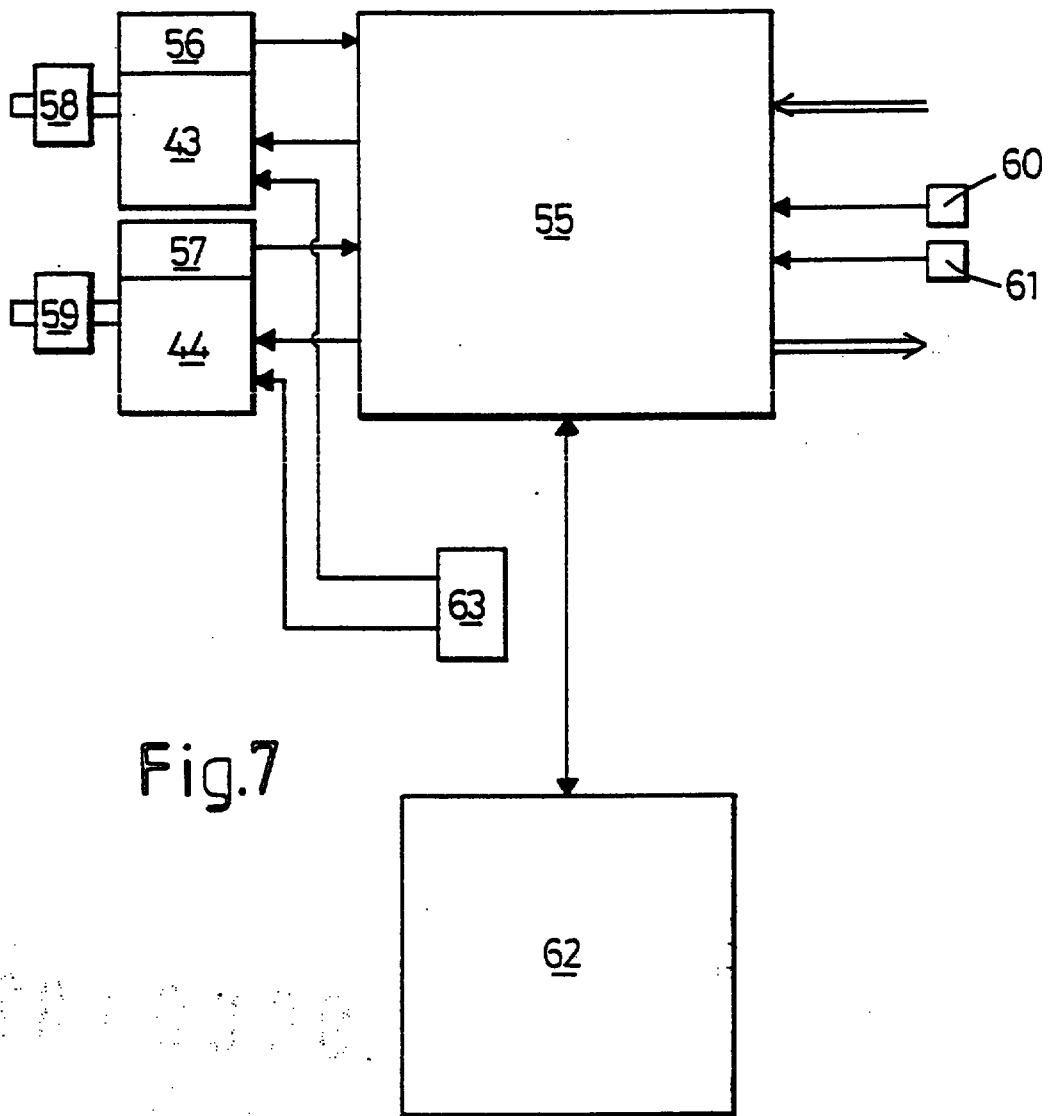



Fig.7

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REVOLVING DOOR LOCK**Description**

The present invention relates to a revolving door lock having outer walls laterally defining a passage space.

In the present times of energy saving it is important to seal off entrances of buildings as good as possible so that no heating power disappears from the building. The inner space of the building should also be kept free of draught, which is particularly important in restaurants or hotels, shops and main entrances of commercial properties, hospitals and the like. Until now, this object has been obtained by so-called revolving cross doors, in which three or four door wings define three or four cylinder segments having an angle at the centre of 120° or 90° respectively. With these revolving cross doors, in particular with the four winged revolving door, the passage space is very narrow so that it is for example not possible to enter the building entrance with a trolley or a wheel chair, and the passage with suitcases is difficult. A further disadvantage of these prior art doors is that all persons passing through the revolving cross doors should go equally fast so that old or handicapped people could become pinched due to a faster person which is also in the area of the revolving cross door, or due to a motorized revolving cross door being driven too fast. Motorized revolving cross doors having a lower rotational speed have a reduced capacity, however, and prevent a quick passage through the revolving door lock.

US-A-0 677 294 discloses a revolving door lock having the features of the preamble of claim 1.

The object of the present invention is to provide a revolving door lock which does not have said disadvantages of the prior art revolving cross doors.

Said object is realised by the characterising features of claim 1.

A revolving door lock according to the invention provides thus an intermediate space which increases safety of a revolving door lock with two revolving doors moving in synchronism with each other, since this space provides a person an opportunity to stop and to reorientate before continuing his passage. In US-A-0 677 294 a person is forced to pass from one continuously rotating re-

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volving door to the other, otherwise he or she will be hit by a door wing. Furthermore, the revolving door lock according to the invention can be passed quickly, in that at each revolving door a free passage is possible since only one of the doors should effect a seal at a time, while the other one can be fully open. As a result, several persons may pass through the revolving door lock in their own speed without interfering with each other.

A favourable embodiment of the revolving door lock according to the invention is characterized in that the rotating axes of the revolving doors are positioned at a distance which is greater than the sum of the radius of both revolving doors, and preferably greater than 2.25 times the radius.

Due to this feature it is possible to create an intermediate space between the revolving doors offering persons passing through the revolving door lock the opportunity to stop between the revolving doors. This is of great importance to the safety as people could panic at the transfer between the revolving doors because they do not know how to walk and could then be hit by the door wings. According to the invention the people are able to reorientate in the intermediate space and only then continue on their way. With a distance of at least ± 0.4 m and preferably 0.75 m between the ends of the door wings there is created enough space for persons, while for example in supermarkets where people pass through the revolving door lock with shopping trolleys a room of at least 1.5 m provides a place to stop. This intermediate space also enables a continued passage between the revolving doors to both quickly and slowly passing people which may enter the next revolving door just before or just after the passage of a door wing. On the other hand, the feature offers new arrangement possibilities of the revolving doors. It is for instance possible to connect three or more revolving doors to the intermediate or passage space, which are adapted to each other and together effect the sealing of the inner space of the building.

Embodiments of the invention by way of example are elucidated hereafter with reference to the attached drawings. In the drawings:

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Fig. 1 is a schematic plan view of a first embodiment of the revolving door lock according to the invention illustrating a so-called summer position with interrupted lines.

Fig. 2 is a very schematic plan view of a second embodiment of the revolving door lock according to the invention.

Fig. 3 is a substantially simplified block diagram of a synchronizing device of the embodiment of Fig. 1.

In the various figures similar parts are indicated by the same reference numerals.

Fig. 1 shows a first embodiment of the revolving door lock according to the invention in which there is arranged an intermediate part 40, 41, respectively, between the outer walls 2, 4 and 3, 5, respectively, so that there is created an intermediate space 42 between the revolving doors 10 and 11. As a result, the rotating axes 8 and 9 of the revolving doors 10 and 11 are positioned at a distance to each other which is greater than the sum of the radius R of both revolving doors 10 and 11 of which the radii R are equal in this case. Due to this feature, the paths of both revolving doors 10 and 11 do not overlap, and there is even created an intermediate space enabling a person to wait within the intermediate space 42 before continuing its way. In a passage lock having revolving doors the diameter of which being 3 m, the intermediate space between the ends of the door wings may for example be 0.75 m in passages for persons and 1.5 m in passages for persons having shopping trolleys.

Furthermore, each revolving door 10, 11 in this embodiment has its own drive motor 43, 44 which are electronically coupled through a synchronizing device 45 ensuring the synchronized rotation of the revolving doors 10, 11. Such embodiment in which the synchronized rotation of the revolving doors 10, 11 is obtained through a synchronizing device has the important advantage that each drive motor 43, 44 drives a smaller mass resulting in an enhanced safety due to a smaller mass inertia.

A substantially simplified block diagram of the synchronizing device 45 is shown in Fig. 3. The synchronizing device 45 includes a central processing unit 55



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controlling the drive motors 43 and 44. Each drive motor 43, 44 comprises a respective so-called incremental angular encoder 56, 57, respectively, which is coupled directly to the shaft of the drive motor and which delivers a certain number of pulses for each revolution of the motor shaft. The drive motors 43, 44 drive the rotating shafts 8, 9 through a reduction gear 58, 59 so that, by means of an angular encoder having a relatively small resolution, nonetheless a great number of pulses per revolution of the revolving door and consequently a high resolution for determining the position of the revolving door is obtained. The counter position of each angular encoder 56, 57, which counter position indicates the rotational position of the respective revolving door 10, 11, is supplied to the processing unit 55.

Furthermore, a position detector 60, 61 for each revolving door is mounted in a fixed position relative to the respective revolving door 10, 11, respectively. As is shown in Fig. 1, the fixed position of the position detector 60 is off-set 90° with respect to the fixed position of the position detector 61. Both position detectors 60, 61 are also connected to the processing unit. Each position detector 60, 61 is able to deliver a signal each 180° or each 360° , dependent on the fact whether one or both door wings comprise for example a magnetic element which is detected by the position detector 60, 61.

Finally, the processing unit 55 is connected to a conventional control unit (not shown) or a revolving door which is adapted a.o. to deliver a rotation command to the processing unit. When the processing unit 55 receives a rotation command, the processing unit activates the drive motors 43, 44 and causes the revolving doors 10, 11 to rotate in a synchronized fashion with a speed adjustable through a control panel 62, by means of the signals of the angular encoders 56, 57 and the position detectors 60, 61. The processing unit 55 resets the counter position of each angular encoder to a starting position, preferably the zero position, each time the respective position detector 60, 61 monitors a door wing 26 or 27 and consequently when a particular rotational position is reached. As a result, the processing unit 55 may obtain the synchronized rotation of the revolving doors 10, 11 by equalizing the counter positions of both angular encoders 56, 57. Maintaining the difference in



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angular position of 90° between the revolving doors 10, 11 is very simple due to the relative 90° off-set of both position detectors 60, 61.

If the processing unit 55 signalizes a lack of synchronization of the revolving doors 10, 11, the synchronized rotation of the revolving doors is restored by the processing unit by braking the leading revolving door until the revolving doors are synchronized again. Consequently, no unsafe situations will occur as a result of a sudden acceleration of a revolving door.

It will be clear that when there is a lack of synchronization, both the revolving door 10 and the revolving door 11 may be the leading revolving door. In the synchronizing device 45 there is no master-slave situation which is common in synchronizing systems. Both revolving doors 10, 11 can be master and slave for the synchronizing device.

As in normal operation the processing unit 55 receives signals both from the angular encoders 56, 57 and from the position detectors 60, 61, the processing unit 55 can easily check the proper operation of the angular encoders 56, 57 and the position detectors 60, 61. If, after a full revolution, no signal is received from a position detector 60, 61, the processing unit 55 delivers an alarm signal through the control panel 62 in which, for example by means of individual LED's (not shown), the defective position detector is indicated. The processing unit 55 keeps the drive motors 43, 44 in operation and strives to maintain the synchronized rotation by means of the signals of the angular encoders 56, 57. For reasons of safety, the processing unit 55 may select a lower rotational speed, for example.

If no signal from an angular encoder 56, 57 is received, the processing unit 55 also delivers an alarm signal through the control panel 62 in which the defective angular encoder 56, 57 is indicated by a LED. In case of a defective angular encoder 56, 57, the processing unit 55 may strive to maintain a synchronized rotation by means of the position detectors 60, 61. Alternatively, it is possible to stop the drive motor of the defective angular encoder 56, 57 and to release the respective revolving door to allow for manual operation.



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If the synchronizing device 45 is connected to the voltage of the supply line for the first time or if the processing unit 55 is reset, the processing unit 55 controls the drive motors 43, 44 such that the revolving doors 10, 11 are rotated to the particular position determined by the respective position detector 60, 61 at a creeping speed to determine the starting positions of the angular encoders 56, 57. Then, the revolving doors 10, 11 are accelerated and decelerated again for a short time, enabling the processing unit 55 to determine from the motor current the mass driven by the drive motors. With this information, the processing unit 55 is able to limit the maximum torque of the revolving doors 10, 11 to a value laid down in safety regulations.

The synchronizing device 45 further includes an emergency stop device 63 directly connected to the drive motors 43, 44 and stopping and short-circuiting the drive motors in case of an emergency stop.

By using the synchronizing device 45 including a drive motor 43, 44 for each revolving door 10, 11 it is possible to simply set the revolving doors in certain particular positions. In Fig. 5, for example, interrupted lines indicate a so-called summer position of the revolving door lock 1 in which the revolving doors 10, 11 are positioned individually and aligned with each other along the longitudinal axis of the revolving door lock 1. The doors 10, 11 not only open the passage but also simultaneously form a separation of the entrance and exit paths. This position may also be used for the transport of goods occupying a lot of space.

It is also possible to select a night position through the control panel 62, in which the revolving door 11 is in its summer position indicated by an interrupted line, while the revolving door 10 is perpendicular to the summer position, for example.

Fig. 2 shows a second embodiment in which the revolving door lock 1 comprises three revolving doors, that is a single outer revolving door 46 and two inner revolving doors 47 and 48. Between the revolving doors there is a central passage space 49 defined by the revolving doors 46-48 and by the outer walls 50. Such revolving door lock may for example be used for the entrance and exit of two interior spaces separated by a partition wall

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51, and it is also possible that the revolving door 46 having a large capacity is used for both entrance and exit, while the smaller revolving doors 47 and 48 are used as entrance on the one hand and as exit on the other hand. In this embodiment, the vertical planes of symmetry 52, 53 and 54 with respect to the pair of diametrically opposed outer walls of the passage spaces provided by the revolving doors 46, 47, 48, respectively, are not parallel to each other. Nonetheless, the angles B of the revolving doors 47 and 58 to their respective longitudinal plane 53, 54 are off-set 90° with respect to the angle a of the revolving door 46 with respect to its longitudinal plane 52. In this way it is ensured that either the revolving door 46 or the revolving doors 47 and 48 are in their closed position thereby preventing draught through the revolving door lock. Of course, also other arrangements of a larger or smaller number of revolving doors may be selected within a revolving door lock depending on the respective situation.

The invention is not restricted to the embodiments shown in the drawing and described herein before which may be varied in different manners within the scope of the invention defined in the claims.

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Claims

1. Revolving door lock having outer walls (2-5;50) laterally defining a passage space (6,7;49) comprising at least two revolving doors (10,11;46-48) positioned on first and second opposite sides of the passage space (6,7;49), each revolving door (10, 11; 46-48) comprising a pair of diametrically opposed outer walls (2,3;4,5) arranged symmetrically with respect to a longitudinal vertical plane (52-54), and each revolving door (10,11;46-48) having two aligned wings (26,27/28,29) rotatable about a vertical rotating axis (8,9) and defining the diameter of the revolving door, the revolving doors (10,11; 46-48) moving synchronized with each other, whereby the wings of the revolving door or doors on the first side of the passage space (6,7;49) have an angular off-set of 90° relative to the wings of the revolving door or doors on the second side of the passage space (6,7;49) with respect to the longitudinal vertical plane of symmetry (52-54) of the pair of diametrically opposed outer walls of the respective revolving door (10,11;46-48) so that the passage space (6,7;49) is always closed at least on one side, characterized in that the rotating axes (8,9) of two with respect to each other oppositely arranged revolving doors (10,11;46-48) are positioned at a distance with respect to each other which is greater than the sum of the radii of both revolving doors to create an intermediate space (42;49) in the passage space (6,7;49) between the said oppositely arranged revolving doors (10,11;46-48), the distance being the said sum of radii increased by at least substantially 0.4 m.
2. Revolving door lock according to claim 1, wherein the distance between the rotating axes (8,9) of said two oppositely arranged and being equally dimensioned revolving doors (10,11) is at least 2.25 times the radius thereof.
3. Revolving door lock according to claim 1 or 2, wherein three or more revolving doors (46-48) are provided, being positioned in the outer walls (50) of a larger passage space (49).



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4. Revolving door lock according to one of claims 1-3, wherein the wings (26,27/28,29) of the revolving doors (10,11;46-48) are relatively pivotable back and forth about the respective rotating axis (8,9).
5. Revolving door lock according to one of the preceding claims, wherein the drive of the revolving doors (10,11) takes place by means of a single motor (18) being connected to the revolving door shafts (8,9) by a mechanical coupling (12-17).
6. Revolving door lock according to one of claims 1-4, wherein each revolving door (10,11) is equipped with its own drive motor (43,44) and the individual drive motors being coupled by an electronic synchronizing device (45).
7. Revolving door lock according to claim 6, **characterized** in that the synchronizing device (45) for each revolving door (10,11) comprises an angular encoder (56,57) delivering a certain number of pulses for each revolution of the revolving door so that a counter position indicates the rotational position of the revolving door, and a position detector (60,61) for detecting at least one particular revolving position of the revolving door, wherein a processing unit (55) is provided to which the angular encoder (56,57) and the position detectors (60,61) are connected and which controls the drive motors (43,44), said processing unit (55) resets the counter position of the angular encoder (56,57) to a starting position when the respective position detector (60,61) signals that the particular rotating position is reached, and said processing unit (55) being adapted to allow the revolving doors (10,11) to move with a synchronized adjustable speed by means of the signals of the angular encoder (56,57) and position detectors (60,61).
8. Revolving door lock according to claim 7, **characterized** in that the position detector (60) on the one side of the revolving door lock is arranged in a fixed position relative to the respective revolving door

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(10), which is rotated 90° with respect to the fixed position taken by the position detector(s) (61) on the other side of the revolving door lock relative to the respective revolving door (11), and the processing unit (55) controlling the drive motors (43,44) such that the angular encoders (56,57) indicate the same revolving door position.

9. Revolving door lock according to claim 7 or 8, **characterized** in that, if the revolving doors (10,11) are not synchronized, the processing unit (55) brakes the leading revolving door to restore the synchronization.
10. Revolving door lock according to claim 7, 8 or 9, **characterized** in that the processing unit (55) monitors the presence of the signals of the angular encoder (56,57) and the position detectors (60,61) and delivers an alarm signal if a position detector does not deliver a signal after a revolution of the revolving door, wherein the processing unit preferably maintains a synchronized rotation of the revolving doors at a lower speed by means of the signals of the angular encoders.
11. Revolving door lock according to claim 10, **characterized** in that the processing unit (55) also delivers an alarm signal if the signal of an angular encoder (56,57) disappears, wherein preferably at least the respective revolving door (10,11) is stopped and released.
12. Revolving door lock according to one of claims 7-11, **characterized** in that, upon switching to the voltage of the supply line or upon resetting the processing unit (55), the processing unit controls the drive motors (43,44) such that the revolving doors are moved to the rotational position determined by the position detector (60,61) at a creeping speed, wherein, after reaching this rotational position, the processing unit accelerates and decelerates the revolving doors (10,11) for a short time to determine the mass driven by the drive motor.



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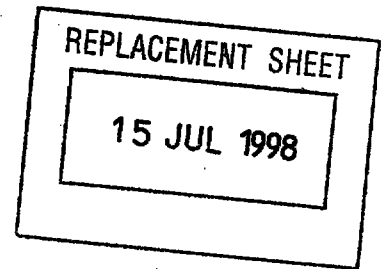
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13. Revolving door lock according to one of claims 7-12, **characterized** in that an emergency stop device is provided being directly connected to the drive motors (43,44) and short-circuiting them in case of an emergency stop.
14. Revolving door lock according to one of claims 7-13, **characterized** in that each angular encoder (56,57) is coupled directly to the shaft of the drive motor (43,44) and the drive motor is driving the revolving door (10,11) through a reduction gear.

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REVOLVING DOOR LOCK**Description**

The present invention relates to a revolving door lock having outer walls laterally defining a passage space.

In the present times of energy saving it is important to seal off entrances of buildings as good as possible so that no heating power disappears from the building. The inner space of the building should also be kept free of draught, which is particularly important in restaurants or hotels, shops and main entrances of commercial properties, hospitals and the like. Until now, this object has been obtained by so-called revolving cross doors, in which three or four door wings define three or four cylinder segments having an angle at the centre of 120° or 90° respectively. With these revolving cross doors, in particular with the four winged revolving door, the passage space is very narrow so that it is for example not possible to enter the building entrance with a trolley or a wheel chair, and the passage with suitcases is difficult. A further disadvantage of these prior art doors is that all persons passing through the revolving cross doors should go equally fast so that old or handicapped people could become pinched due to a faster person which is also in the area of the revolving cross door, or due to a motorized revolving cross door being driven too fast. Motorized revolving cross doors having a lower rotational speed have a reduced capacity, however, and prevent a quick passage through the revolving door lock.

US-A-0 677 294 discloses a revolving door lock having the features of the preamble of claim 1.

The object of the present invention is to provide a revolving door lock which does not have said disadvantages of the prior art revolving cross doors.

Said object is realised by the characterising features of claim 1.

A revolving door lock according to the invention provides thus an intermediate space which increases safety of a revolving door lock with two revolving doors moving in synchronism with each other, since this space provides a person an opportunity to stop and to reorientate before continuing his passage. In US-A-0 677 294 a person is forced to pass from one continuously rotating re-

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volving door to the other, otherwise he or she will be hit by a door wing. Furthermore, the revolving door lock according to the invention can be passed quickly, in that at each revolving door a free passage is possible since only one of the doors should effect a seal at a time, while the other one can be fully open. As a result, several persons may pass through the revolving door lock in their own speed without interfering with each other.

A favourable embodiment of the revolving door lock according to the invention is characterized in that the rotating axes of the revolving doors are positioned at a distance which is greater than the sum of the radius of both revolving doors, and preferably greater than 2.25 times the radius.

Due to this feature it is possible to create an intermediate space between the revolving doors offering persons passing through the revolving door lock the opportunity to stop between the revolving doors. This is of great importance to the safety as people could panic at the transfer between the revolving doors because they do not know how to walk and could then be hit by the door wings. According to the invention the people are able to reorientate in the intermediate space and only then continue on their way. With a distance of at least ± 0.4 m and preferably 0.75 m between the ends of the door wings there is created enough space for persons, while for example in supermarkets where people pass through the revolving door lock with shopping trolleys a room of at least 1.5 m provides a place to stop. This intermediate space also enables a continued passage between the revolving doors to both quickly and slowly passing people which may enter the next revolving door just before or just after the passage of a door wing. On the other hand, the feature offers new arrangement possibilities of the revolving doors. It is for instance possible to connect three or more revolving doors to the intermediate or passage space, which are adapted to each other and together effect the sealing of the inner space of the building.

Embodiments of the invention by way of example are elucidated hereafter with reference to the attached drawings. In the drawings:

Fig. 1 is a schematic plan view of a first embodiment of the revolving door lock according to the invention illustrating a so-called summer position with interrupted lines,

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Fig. 2 is a very schematic plan view of a second embodiment of the revolving door lock according to the invention,

Fig. 3 is a substantially simplified block diagram of a synchronizing device of the embodiment of Fig. 1

In the various figures similar parts are indicated by the same reference numerals.

Fig. 1 shows a first embodiment of the revolving door lock according to the invention in which there is arranged an intermediate part 40, 41, respectively, between the outer walls 2, 4 and 3, 5, respectively, so that there is created an intermediate space 42 between the revolving doors 10 and 11. As a result, the rotating axes 8 and 9 of the revolving doors 10 and 11 are positioned at a distance to each other which is greater than the sum of the radius R of both revolving doors 10 and 11 of which the radii R are equal in this case. Due to this feature, the paths of both revolving doors 10 and 11 do not overlap, and there is even created an intermediate space enabling a person to wait within the intermediate space 42 before continuing its way. In a passage lock having revolving doors the diameter of which being 3 m, the intermediate space between the ends of the door wings may for example be 0.75 m in passages for persons and 1.5 m in passages for persons having shopping trolleys.

Furthermore, each revolving door 10, 11 in this embodiment has its own drive motor 43, 44 which are electronically coupled through a synchronizing device 45 ensuring the synchronized rotation of the revolving doors 10, 11. Such embodiment in which the synchronized rotation of the revolving doors 10, 11 is obtained through a synchronizing device has the important advantage that each drive motor 43, 44 drives a smaller mass resulting in an enhanced safety due to a smaller mass inertia.

A substantially simplified block diagram of the synchronizing device 45 is shown in Fig. 3. The synchronizing device 45 includes a central processing unit 55 controlling the drive motors 43 and 44. Each drive motor 43, 44 comprises a respective so-called incremental angular encoder 56, 57, respectively, which is coupled directly to the shaft of the drive motor and which delivers a certain number of pulses for each revolution of the motor shaft. The drive motors 43, 44 drive the rotating shafts 8, 9 through a reduction gear 58, 59 so that, by means of an angular encoder having a relatively small resolution, nonetheless a great number of pulses per

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revolution of the revolving door and consequently a high resolution for determining the position of the revolving door is obtained. The counter position of each angular encoder 56, 57, which counter position indicates the rotational position of the respective revolving door 10, 11, is supplied to the processing unit 55.

Furthermore, a position detector 60, 61 for each revolving door is mounted in a fixed position relative to the respective revolving door 10, 11, respectively. As is shown in Fig. 1, the fixed position of the position detector 60 is off-set 90° with respect to the fixed position of the position detector 61. Both position detectors 60, 61 are also connected to the processing unit. Each position detector 60, 61 is able to deliver a signal each 180° or each 360° , dependent on the fact whether one or both door wings comprise for example a magnetic element which is detected by the position detector 60, 61.

Finally, the processing unit 55 is connected to a conventional control unit (not shown) or a revolving door which is adapted a.o. to deliver a rotation command to the processing unit. When the processing unit 55 receives a rotation command, the processing unit activates the drive motors 43, 44 and causes the revolving doors 10, 11 to rotate in a synchronized fashion with a speed adjustable through a control panel 62, by means of the signals of the angular encoders 56, 57 and the position detectors 60, 61. The processing unit 55 resets the counter position of each angular encoder to a starting position, preferably the zero position, each time the respective position detector 60, 61 monitors a door wing 26 or 27 and consequently when a particular rotational position is reached. As a result, the processing unit 55 may obtain the synchronized rotation of the revolving doors 10, 11 by equalizing the counter positions of both angular encoders 56, 57. Maintaining the difference in angular position of 90° between the revolving doors 10, 11 is very simple due to the relative 90° off-set of both position detectors 60, 61.

If the processing unit 55 signalizes a lack of synchronization of the revolving doors 10, 11, the synchronized rotation of the revolving doors is restored by the processing unit by braking the leading revolving door until the revolving doors are synchronized again. Consequently, no unsafe situations will occur as a result of a sudden acceleration of a revolving door.

It will be clear that when there is a lack of synchronization, both the revolving door 10 and the revolving door 11 may be the leading revolving door. In the syn-

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chronizing device 45 there is no master-slave situation which is common in synchronizing systems. Both revolving doors 10, 11 can be master and slave for the synchronizing device.

As in normal operation the processing unit 55 receives signals both from the angular encoders 56, 57 and from the position detectors 60, 61, the processing unit 55 can easily check the proper operation of the angular encoders 56, 57 and the position detectors 60, 61. If, after a full revolution, no signal is received from a position detector 60, 61, the processing unit 55 delivers an alarm signal through the control panel 62 in which, for example by means of individual LED's (not shown), the defective position detector is indicated. The processing unit 55 keeps the drive motors 43, 44 in operation and strives to maintain the synchronized rotation by means of the signals of the angular encoders 56, 57. For reasons of safety, the processing unit 55 may select a lower rotational speed, for example.

If no signal from an angular encoder 56, 57 is received, the processing unit 55 also delivers an alarm signal through the control panel 62 in which the defective angular encoder 56, 57 is indicated by a LED. In case of a defective angular encoder 56, 57, the processing unit 55 may strive to maintain a synchronized rotation by means of the position detectors 60, 61. Alternatively, it is possible to stop the drive motor of the defective angular encoder 56, 57 and to release the respective revolving door to allow for manual operation.

If the synchronizing device 45 is connected to the voltage of the supply line for the first time or if the processing unit 55 is reset, the processing unit 55 controls the drive motors 43, 44 such that the revolving doors 10, 11 are rotated to the particular position determined by the respective position detector 60, 61 at a creeping speed to determine the starting positions of the angular encoders 56, 57. Then, the revolving doors 10, 11 are accelerated and decelerated again for a short time, enabling the processing unit 55 to determine from the motor current the mass driven by the drive motors. With this information, the processing unit 55 is able to limit the maximum torque of the revolving doors 10, 11 to a value laid down in safety regulations.

The synchronizing device 45 further includes an emergency stop device 63 directly connected to the

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drive motors 43, 44 and stopping and short-circuiting the drive motors in case of an emergency stop.

By using the synchronizing device 45 including a drive motor 43, 44 for each revolving door 10, 11 it is possible to simply set the revolving doors in certain particular positions. In Fig. 5, for example, interrupted lines indicate a so-called summer position of the revolving door lock 1 in which the revolving doors 10, 11 are positioned individually and aligned with each other along the longitudinal axis of the revolving door lock 1. The doors 10, 11 not only open the passage but also simultaneously form a separation of the entrance and exit paths. This position may also be used for the transport of goods occupying a lot of space.

It is also possible to select a night position through the control panel 62, in which the revolving door 11 is in its summer position indicated by an interrupted line, while the revolving door 10 is perpendicular to the summer position, for example.

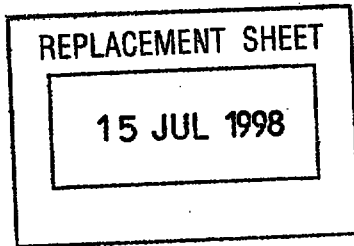
Fig. 2 shows a second embodiment in which the revolving door lock 1 comprises three revolving doors, that is a single outer revolving door 46 and two inner revolving doors 47 and 48. Between the revolving doors there is a central passage space 49 defined by the revolving doors 46-48 and by the outer walls 50. Such revolving door lock may for example be used for the entrance and exit of two interior spaces separated by a partition wall 51, and it is also possible that the revolving door 46 having a large capacity is used for both entrance and exit, while the smaller revolving doors 47 and 48 are used as entrance on the one hand and as exit on the other hand. In this embodiment, the vertical planes of symmetry 52, 53 and 54 with respect to the pair of diametrically opposed outer walls of the passage spaces provided by the revolving doors 46, 47, 48, respectively, are not parallel to each other. Nonetheless, the angles B of the revolving doors 47 and 48 to their respective longitudinal plane 53, 54 are off-set 90° with respect to the angle α of the revolving door 46 with respect to its longitudinal plane 52. In this way it is ensured that either the revolving door 46 or the revolving doors 47 and 48 are in their closed position thereby preventing draught through the revolving door lock. Of course, also other arrangements of a larger or smaller number of revolving doors may be selected within a revolving door lock depending on the respective situation.

The invention is not restricted to the embodiments

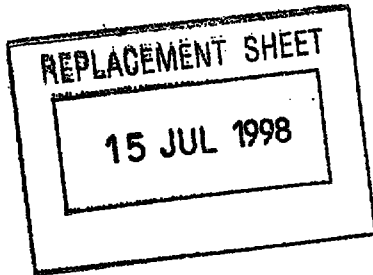
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shown in the drawing and described herein before which
may be varied in different manners within the scope of
the invention defined in the claims.

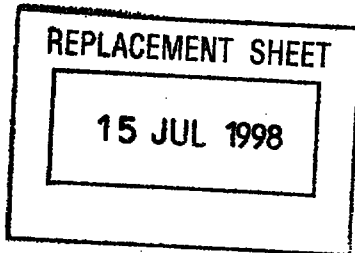


Claims



1. Revolving door lock having outer walls (2-5;50) laterally defining a passage space (6,7;49) comprising at least two revolving doors (10,11;46-48) positioned on first and second opposite sides of the passage space (6,7;49), each revolving door (10, 11; 46-48) comprising a pair of diametrically opposed outer walls (2,3;4,5) arranged symmetrically with respect to a longitudinal vertical plane (52-54), and each revolving door (10,11;46-48) having two aligned wings (26,27/28,29) rotatable about a vertical rotating axis (8,9) and defining the diameter of the revolving door, the revolving doors (10,11; 46-48) moving synchronized with each other, whereby the wings of the revolving door or doors on the first side of the passage space (6,7;49) have an angular off-set of 90° relative to the wings of the revolving door or doors on the second side of the passage space (6,7;49) with respect to the longitudinal vertical plane of symmetry (52-54) of the pair of diametrically opposed outer walls of the respective revolving door (10,11;46-48) so that the passage space (6,7;49) is always closed at least on one side, characterized in that the rotating axes (8,9) of two with respect to each other oppositely arranged revolving doors (10,11;46-48) are positioned at a distance with respect to each other which is greater than the sum of the radii of both revolving doors to create an intermediate space (42;49) in the passage space (6,7;49) between the said oppositely arranged revolving doors (10,11;46-48), the distance being the said sum of radii increased by at least substantially 0.4 m.
2. Revolving door lock according to claim 1, wherein the distance between the rotating axes (8,9) of said two oppositely arranged and being equally dimensioned revolving doors (10,11) is at least 2.25 times the radius thereof.
3. Revolving door lock according to claim 1 or 2, wherein three or more revolving doors (46-48) are provided, being positioned in the outer walls (50) of a larger passage space (49).





4. Revolving door lock according to one of claims 1-3, wherein the wings (26,27/28,29) of the revolving doors (10,11;46-48) are relatively pivotable back and forth about the respective rotating axis (8,9).
5. Revolving door lock according to one of the preceding claims, wherein the drive of the revolving doors (10,11) takes place by means of a single motor (18) being connected to the revolving door shafts (8,9) by a mechanical coupling (12-17).
6. Revolving door lock according to one of claims 1-4, wherein each revolving door (10,11) is equipped with its own drive motor (43,44) and the individual drive motors being coupled by an electronic synchronizing device (45).
7. Revolving door lock according to claim 6, characterized in that the synchronizing device (45) for each revolving door (10,11) comprises an angular encoder (56,57) delivering a certain number of pulses for each revolution of the revolving door so that a counter position indicates the rotational position of the revolving door, and a position detector (60,61) for detecting at least one particular revolving position of the revolving door, wherein a processing unit (55) is provided to which the angular encoder (56,57) and the position detectors (60,61) are connected and which controls the drive motors (43,44), said processing unit (55) resets the counter position of the angular encoder (56,57) to a starting position when the respective position detector (60,61) signals that the particular rotating position is reached, and said processing unit (55) being adapted to allow the revolving doors (10,11) to move with a synchronized adjustable speed by means of the signals of the angular encoder (56,57) and position detectors (60,61).
8. Revolving door lock according to claim 7, characterized in that the position detector (60) on the one side of the revolving door lock is arranged in a fixed position relative to the respective revolving door (10), which is rotated 90° with respect to the fixed position taken by the position detector(s) (61) on the other side of the revolving door lock relative to the respective revolving door (11), and the processing unit (55) controlling the drive motors (43,44) such

that the angular encoders (56,57) indicate the same revolving door position.

9. Revolving door lock according to claim 7 or 8, characterized in that, if the revolving doors (10,11) are not synchronized, the processing unit (55) brakes the leading revolving door to restore the synchronization.

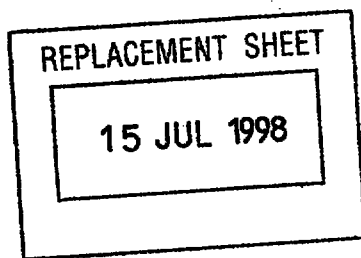
10. Revolving door lock according to claim 7, 8 or 9, characterized in that the processing unit (55) monitors the presence of the signals of the angular encoder (56,57) and the position detectors (60,61) and delivers an alarm signal if a position detector does not deliver a signal after a revolution of the revolving door, wherein the processing unit preferably maintains a synchronized rotation of the revolving doors at a lower speed by means of the signals of the angular encoders.

11. Revolving door lock according to claim 10, characterized in that the processing unit (55) also delivers an alarm signal if the signal of an angular encoder (56,57) disappears, wherein preferably at least the respective revolving door (10,11) is stopped and released.

12. Revolving door lock according to one of claims 7-11, characterized in that, upon switching to the voltage of the supply line or upon resetting the processing unit (55), the processing unit controls the drive motors (43,44) such that the revolving doors are moved to the rotational position determined by the position detector (60,61) at a creeping speed, wherein, after reaching this rotational position, the processing unit accelerates and decelerates the revolving doors (10,11) for a short time to determine the mass driven by the drive motor.

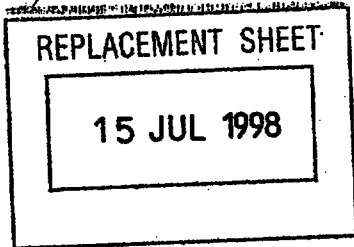
13. Revolving door lock according to one of claims 7-12, characterized in that an emergency stop device is provided being directly connected to the drive motors (43,44) and short-circuiting them in case of an emergency stop.

14. Revolving door lock according to one of claims 7-13, characterized in that each angular encoder



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(56,57) is coupled directly to the shaft of the drive motor (43,44) and the drive motor is driving the revolving door (10,11) through a reduction gear.



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REVOLVING DOOR LOCK

ABSTRACT

A revolving door lock (1) having outer walls (2-5) laterally defining a passage space comprises at least two revolving doors (10, 11) moving relatively synchronized, said revolving doors (10, 11) having two wings (26-29) rotatable about respective vertical rotating axes (8, 9). The wings (26-29) of the individual revolving doorrs (10, 11) have such angular position relative to each other that the passage space is always closed on at least one side. In one embodiment there are two revolving doors of which the angles to the respective longitudinal line of symmetry are relatively off-set 90° . On the other hand there can be arranged three or more revolving doors being positioned in the outer walls of a larger passage space.

FIGURE 1

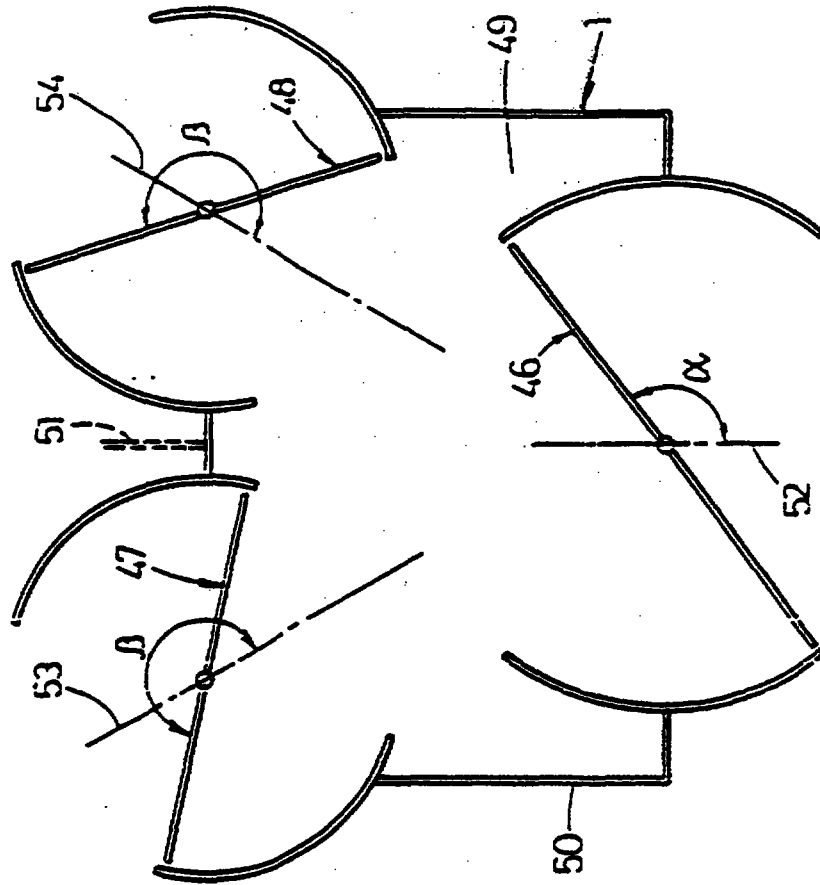


Fig.2

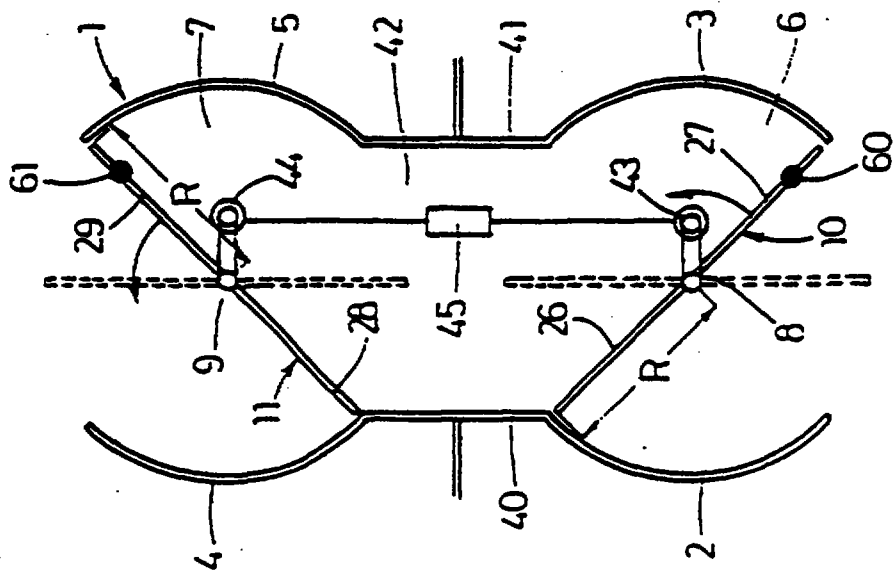


Fig.1

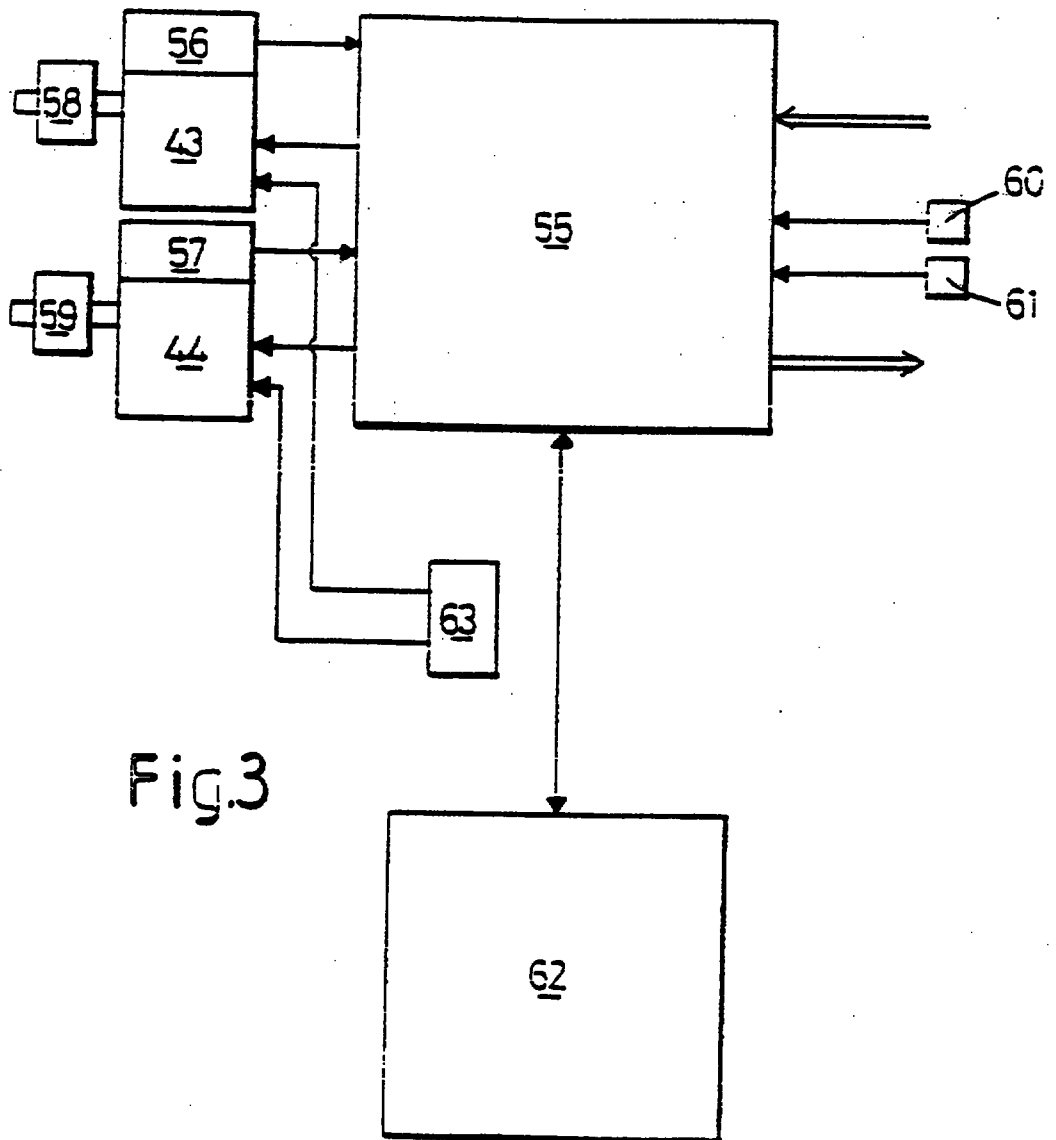


Fig.3