

Description

A temperature-controlled tank container of the kind specified in the first part of claim 1 is known from DE-C2-2,917,364. It is emphasized as an important feature in said publication that the heat-insulating jacket is mounted on the planar wall surfaces of a cuboid container framework. Large triangular flow areas are formed between these planar wall surfaces and the cylindrical tank. Consequently a major part of the temperature-control medium can flow without contacting the tank, all the more as through-holes for the passage of the temperature-control medium are provided rather far away from the tank in transverse bulkheads.

Furthermore, in the known tank container the space between the tank and the heat-insulating jacket is subdivided by partitions in such a way that the temperature-control medium flows almost exclusively in the axial direction of the tank, so that a laminar flow may develop over large distances. Depending on the arrangement of the partitions, a flow path length is obtained which is no more than two to four times the length of the tank.

These circumstances limit the degree to which the temperature-control medium is utilized for cooling or heating the tank.

Apart from that, because the heat-insulating jacket is disposed along the planar surfaces of the container framework, it requires a considerable amount of insulating and covering material, and this results in a corresponding increase in the tare mass of the entire tank container.

The invention is based on the general object of at least partially eliminating drawbacks as occur in comparable conventional tank containers. In view of the above-described prior art, it is a more specific object of the invention to provide a temperature-controlled tank container which permits a uniform flow around the entire tank including the tank fittings while optimally utilizing the temperature-control medium.

The invention meets with this object by providing a tank container as characterized in claim 1. There, the temperature-control medium is passed as a layer of substantially uniform overall thickness, which in practice amounts to but a few centimetres, along a meandering path extending about the major part of the cylinder circumference in intimate contact with the tank from one end face to the other and thence back to the first end face through a vertex channel which includes the tank fittings. The uniform small distance between the insulating jacket and the tank and the meandering configuration of the flow path also cause continuous turbulences in the temperature-control medium, so that the entire mass of the medium is utilized for effective heat exchange along the tank

surface. Due to the fact that the tank is closely surrounded, less insulating and covering material is required for the insulating jacket, whereby the overall weight of the tank container is reduced.

It is known from DE-U1-7,120,959 to arrange a heat-insulating jacket between rings extending in different radial planes and in concentric relation to the tank, and to make a temperature-control medium flow through the thus formed space. Also there, however, the overall flow of medium around the tank is insufficient.

In the improvement according to claim 2, the invention is applied to a tank which is supported merely via end rings by the end parts of a container framework, and these end rings are used for further subdividing the space between the tank and the insulating jacket and thus for extending the flow path. In this arrangement, claim 3 refers to an advantageous incorporation of the vertex channel in the overall flow system. According to claim 4, an overflow sump surrounding the tank fittings is incorporated in the flow path of the temperature-control medium, while no overflow reaches this flow path. According to claim 5 the partition webs forming the vertex channel may simultaneously be used to tighten the insulating jacket or the inside skin thereof.

By the improvement of claims 6 to 8, a particularly stable structure is obtained, and the measure of claim 9 offers the further advantage that sufficiently large flow cross-sections for the passage of the temperature-control medium are obtained while at the same time the flow of the medium is confined in the circumferential direction.

Claims 10 and 11 are directed to additional or alternative embodiments intended to achieve the necessary distance between the outer and inner skins of the insulating jacket and also to support the jacket on the tank casing.

Advantageous embodiments of the invention will now be described in detail below with reference to the drawings, in which

Figure 1 is a schematic side view of a temperature-controlled tank container,

Figure 2 is a plan view showing the tank container of Figure 1,

Figure 3 is an end view of the tank container of Figures 1 and 2,

Figure 4 is an axial section through a part of the tank and the heat-insulating jacket,

Figure 5 is an axial section similar to Figure 4 through another embodiment of the insulating jacket and its mounting on the tank,

Figure 6 is a further axial section similar to Figure 4 which illustrates another spacer member inserted between the tank and the inner and outer skins of the insulating jacket, and

Figure 7 is a detail which shows a way of

mounting of the inner skin of the insulating jacket.

As will be apparent from Figures 1 and 2, the tank is composed of a cylindrical shell 10 and bottom members 11, 12 affixed to the two ends thereof, the shell 10 being surrounded by a plurality of reinforcing rings 13 spaced in the axial direction. The bottom members 11, 12 are mounted through end rings 14 on end members 15 of an outer container framework 16 which has a cuboid outline. Such a way of mounting a tank merely by its ends in a framework is known from DE-C2-3,212,696. At the top, the shell 10 is provided with a manhole 17 and tank fittings 18 which are disposed in an overflow sump 19. Walkways are indicated at 20 in Figure 2.

The left-hand part of Figure 3 is an end view of the tank container of Figures 1 and 2 as seen from the left including the bottom member 11, while the right-hand part is a view of the right-hand end as seen in Figures 1 and 2 including the bottom member 12. The bottom member 11 is formed with two holes 21, 22 for connection to a temperature-control medium supply system. Such supply systems are especially common in container ships, and the temperature-control medium is normally cooling air which is available at large rates of flow but small differential pressure. In such a cooling system, the holes 21, 22 are also called "portholes", of which the lower one 21 is an inlet porthole and the upper one 22 is an outlet porthole. The flow of the cooling medium is caused by negative pressure acting on the outlet porthole 22.

The axial section of Figure 4 illustrates a double-T-section reinforcing ring 13 mounted on the tank shell 10 which form the cylindrical part of the tank casing. The outer flange of the reinforcing ring 13 supports a heat-insulating jacket generally referenced 23 and comprising an inner skin 24, an outer skin 25 and insulating material 26 disposed therebetween. The outer skin 25 is supported through a C-section spacer ring 27 which is disposed in the same radial plane as the reinforcing ring 13 and rests on the latter through the intermediate of the inner skin 24. In the regions between the reinforcing rings 13, the inner skin 24 is supported by means of individual corrugations 28 and/or inserted spacer members 29.

The space formed between the tank shell 10 and the inner skin 24 of the insulating jacket 23, which has a height of but a few centimetres as defined by the height of the web of the reinforcing rings 13, is subdivided according to Figure 2 on either side of the upper vertex line by means of partition webs 30 the mutual spacing of which is approximately equal to the width of the overflow sump 19 as measured in the circumferential direction.

These partition webs 30 extend at either end of

the tank shell 10 into the region between the tank bottom members 11, 12 and the inner skin 24 of the insulating jacket 23 to terminate at the end rings 14. In the region between the two partition webs 30, the reinforcing rings 13 are provided with through-holes 31. In this way, a vertex channel 34 is formed along the vertex line of the tank between the tank casing and the insulating jacket, said vertex channel extending in the axial direction from one bottom area to the other. As indicated in Figure 3, the horizontal bottom parts 33 of the overflow sump 19 are provided outside of, and therefore beneath, those locations where the partition webs 30 engage the end walls of the overflow sump 19. Thus, liquid accumulating in the overflow sump is prevented from entering the vertex channel 34.

In the vicinity of the left-hand end, as viewed in Figures 1 and 2, the vertex channel 34 continues into a wedge-shaped compartment 35 defined by two webs 36 which are inserted between the inner skin 24 of the jacket 23 and the tank bottom member 11 and which converge at an angle and enclose the outlet porthole 22. (Figure 3 shows only one of said webs 36). Through this wedge-shaped compartment 35, the vertex channel 34 opens into the outlet porthole 22, which passes through the jacket 23 and is provided with a collar (not illustrated) for connection to a cooling system, for instance of a container ship.

Outside the vertex channel 34, the end rings 14 and the reinforcing rings 13 are formed with further through-holes 37...42 alternatingly provided in the vicinity of the vertex channel (in the circumferential direction on either side thereof) and in the lower bottom area. As shown in Figure 1, the through-hole 37 in the left-hand end ring 14 and the through-holes 39 and 41 in the reinforcing rings 13 are disposed near the vertex channel 34, while the through-holes 38 and 40 in the reinforcing rings 13 and the through-hole 42 (also indicated in Figure 3) in the right-hand end ring 14 are disposed in the vicinity of the tank bottom. As will be apparent from Figure 3, the space formed outside the wedge-shaped compartment 35 and defined by the tank bottom member 11 and the inner skin 24 of the jacket 23 communicates with the lower inlet porthole 21 which, like the outlet porthole 22, passes through the jacket and is provided on the outside thereof with a collar (not illustrated) for connection to the cooling system.

The compartments defined by the various partitions, i.e. the reinforcing rings 13, the end rings 14 and the partition webs 30 and 36, between the tank casing and the insulating jacket are intercommunicated via the through-holes 31 to 32 and 37 to 42 in such a way that the cooling medium entering through the inlet porthole 21 flows initially upwards

along the left-hand container bottom (as viewed in Figure 1), passes the through-hole 37 and flows downwardly in the region between the end ring 14 and the first reinforcing ring 13, passes the through-hole 38 to flow upwardly through the next compartment defined between the two successive reinforcing rings 13, then flows downwards, upwards, and again downwards to pass the through-hole 42 in the right-hand end ring 14, then upwards along the right-hand tank bottom member 12, through the right-hand through-hole 32 (as viewed in Figure 2) formed in the end ring 14 into the vertex channel 34, where it flows leftwards in the axial direction through the overflow sump 19 while surrounding the tank fittings 18 and the manhole 17, and finally flows through the left-hand through-hole 32 (as viewed in Figure 2) of the end ring 14 and through the wedge-shaped compartment 35 to the outlet porthole 22.

Between the inlet porthole 21 and the bottom-side through-hole 42 in the right-hand end ring 14, the cooling medium flow is distributed to the two regions provided to the right and left of the vertical longitudinal plane of the tank, and in each of said regions flows along a meander-like path indicated by arrows in Figures 1.

The way of mounting the insulating jacket 23 illustrated in Figure 5 differs from that of Figure 4 in that the reinforcing ring 13 and the spacer ring 27 of Figure 4 have been combined to form an integral, double-T-section ring 43 which is used as a reinforcing ring for the tank casing. In Figure 5, this ring 43 is shown as composed of two C-section rings which have been preformed with holes and placed back-to-back. In this case, a through-hole 44 for the temperature-control medium (which is illustrated only as a semi-through-hole) may extend across the entire web height and may therefore be limited to a narrower region in the circumferential direction while the cross-section remains the same. This offers the advantage that all of the temperature-control medium is forced to flow as far as possible in the circumferential direction in each compartment defined between two reinforcing rings. In order to use the entire web height for the through-hole 44, the inner skin 24 of the insulating jacket 23 is bent outwards and secured to the outer leg of the ring 43, as illustrated in the left-hand part of Figure 5, whereas other parts thereof are secured to an intermediate portion of the web of the ring 43, as illustrated in the right-hand part of Figure 5.

Figure 6 illustrates a further modification of a spacer member 45 by means of which the outer skin 25 and the inner skin 24 of the jacket 23 can be held in spaced relationship with respect to each other and to the tank shell 10. This spacer member 45 consists of two cup-shaped insulating bodies

46, which may be made from rigid polyurethane foam, wood, or polyethylene, and the bottoms of which engage the inner skin 24 while their edges engage the tank shell 10 and the outer skin 25, respectively. A common bolt 47 is passed through the two cup bottoms and an opening 48 formed in the inner skin 24, which opening may have a substantially larger dimension than the bolt diameter. The two heads of the bolt 47 (such as bolt head and nut) are sunk in the two cup shapes so as not to contact either the tank shell or the outer skin 25. Cup edge and cup bottom of both insulating bodies 46 are crowned so that the cylindrical shape of the tank and the inner and outer skins of the jacket is not disturbed by edges or by the formation of folds.

On assembly, the spacer members 45 can initially be fixed to the inner skin 24, when the two insulating bodies 46 are tightened against each other by means of the bolt 47. The thus equipped inner skin is then stretched around the tank casing, wherein the edges of the radially inner insulating bodies 46 may be fixed to the tank casing such as by an adhesive. Relieving movements of the inner skin 24 are permitted by the play between the bolt 47 and the opening 48. Following the application of the insulating material 26, which may be polyurethane foam, the outer skin 25 is stretched across the outer insulating bodies 46.

When these spacer members 45 are employed, the spacer rings 27 illustrated in Figure 4 may be dispensed with.

As shown in the detailed illustration of Figure 7, the partition webs 30 defining the vertex channel 34 have flanges 49 at their upper ends. When the inner skin 24 of the jacket 23 is stretched over the tank it can be hooked to one of these flanges 49 by means of crimped portions 50. In this case the webs 30 may either extend radially, as assumed in Figure 7, or vertically.

Claims

1. A temperature-controlled tank container comprising
 - a tank mounted in a container framework (16), the casing of said tank consisting of a cylindrical shell (10) with tank fittings (18) provided at the vertex thereof and two bottom members (11, 12),
 - a heat-insulating jacket (23) surrounding the tank casing on all sides and having two portholes (21, 22) formed at one end thereof on top of each other for the entry and exit of a temperature-control medium, and
 - partitions (13, 14, 30, 36) disposed be-

along the left-hand container bottom (as viewed in Figure 1), passes the through-hole 37 and flows downwardly in the region between the end ring 14 and the first reinforcing ring 13, passes the through-hole 38 to flow upwardly through the next compartment defined between the two successive reinforcing rings 13, then flows downwards, upwards, and again downwards to pass the through-hole 42 in the right-hand end ring 14, then upwards along the right-hand tank bottom member 12, through the right-hand through-hole 32 (as viewed in Figure 2) formed in the end ring 14 into the vertex channel 34, where it flows leftwards in the axial direction through the overflow sump 19 while surrounding the tank fittings 18 and the manhole 17, and finally flows through the left-hand through-hole 32 (as viewed in Figure 2) of the end ring 14 and through the wedge-shaped compartment 35 to the outlet porthole 22.

Between the inlet porthole 21 and the bottom-side through-hole 42 in the right-hand end ring 14, the cooling medium flow is distributed to the two regions provided to the right and left of the vertical longitudinal plane of the tank, and in each of said regions flows along a meander-like path indicated by arrows in Figures 1.

The way of mounting the insulating jacket 23 illustrated in Figure 5 differs from that of Figure 4 in that the reinforcing ring 13 and the spacer ring 27 of Figure 4 have been combined to form an integral, double-T-section ring 43 which is used as a reinforcing ring for the tank casing. In Figure 5, this ring 43 is shown as composed of two C-section rings which have been preformed with holes and placed back-to-back. In this case, a through-hole 44 for the temperature-control medium (which is illustrated only as a semi-through-hole) may extend across the entire web height and may therefore be limited to a narrower region in the circumferential direction while the cross-section remains the same. This offers the advantage that all of the temperature-control medium is forced to flow as far as possible in the circumferential direction in each compartment defined between two reinforcing rings. In order to use the entire web height for the through-hole 44, the inner skin 24 of the insulating jacket 23 is bent outwards and secured to the outer leg of the ring 43, as illustrated in the left-hand part of Figure 5, whereas other parts thereof are secured to an intermediate portion of the web of the ring 43, as illustrated in the right-hand part of Figure 5.

Figure 6 illustrates a further modification of a spacer member 45 by means of which the outer skin 25 and the inner skin 24 of the jacket 23 can be held in spaced relationship with respect to each other and to the tank shell 10. This spacer member 45 consists of two cup-shaped insulating bodies

46, which may be made from rigid polyurethane foam, wood, or polyethylene, and the bottoms of which engage the inner skin 24 while their edges engage the tank shell 10 and the outer skin 25, respectively. A common bolt 47 is passed through the two cup bottoms and an opening 48 formed in the inner skin 24, which opening may have a substantially larger dimension than the bolt diameter. The two heads of the bolt 47 (such as bolt head and nut) are sunk in the two cup shapes so as not to contact either the tank shell or the outer skin 25. Cup edge and cup bottom of both insulating bodies 46 are crowned so that the cylindrical shape of the tank and the inner and outer skins of the jacket is not disturbed by edges or by the formation of folds.

On assembly, the spacer members 45 can initially be fixed to the inner skin 24, when the two insulating bodies 46 are tightened against each other by means of the bolt 47. The thus equipped inner skin is then stretched around the tank casing, wherein the edges of the radially inner insulating bodies 46 may be fixed to the tank casing such as by an adhesive. Relieving movements of the inner skin 24 are permitted by the play between the bolt 47 and the opening 48. Following the application of the insulating material 26, which may be polyurethane foam, the outer skin 25 is stretched across the outer insulating bodies 46.

When these spacer members 45 are employed, the spacer rings 27 illustrated in Figure 4 may be dispensed with.

As shown in the detailed illustration of Figure 7, the partition webs 30 defining the vertex channel 34 have flanges 49 at their upper ends. When the inner skin 24 of the jacket 23 is stretched over the tank it can be hooked to one of these flanges 49 by means of crimped portions 50. In this case the webs 30 may either extend radially, as assumed in Figure 7, or vertically.

Claims

1. A temperature-controlled tank container comprising

a tank mounted in a container framework (16), the casing of said tank consisting of a cylindrical shell (10) with tank fittings (18) provided at the vertex thereof and two bottom members (11, 12),

a heat-insulating jacket (23) surrounding the tank casing on all sides and having two portholes (21, 22) formed at one end thereof on top of each other for the entry and exit of a temperature-control medium, and

partitions (13, 14, 30, 36) disposed be-



tween the tank casing and the insulating jacket (23), said partitions defining compartments which by the way of through-holes (31, 32, 37 ... 42) communicate with each other such that the temperature-control medium is forced to flow round the entire tank casing along a path extending twice the length of the tank and surrounding the tank fittings (18),

characterized in

that the insulating jacket (23) surrounds the tank casing with a spacing which is substantially constant throughout, and

that the partitions include a plurality of partition rings (13, 14) surrounding the tank casing in radial planes and two partition webs (30, 36) extending parallel on either side of the vertex line of the tank, said partition rings (13, 14) and webs (30, 36) forming a flow path which leads in one direction through a vertex channel (34) defined by the partition webs (30, 36) and surrounding said tank fittings (18), and in the other direction along a meandering path defined by the partition rings (13, 14) and covering all remaining areas of the tank casing.

2. The container of claim 1, wherein the tank is connected to the end members (15) of the container framework (16) by means of end rings (14) attached to the two bottom members (11, 12) thereof, said end rings defining further partitions for extending the flow path.
3. The container of claim 2, wherein the vertex channel (34) interconnects through-holes (32) formed in the two end rings (14) and at one end terminates in a wedge-shaped compartment (35) which encloses the upper one (22) of said portholes (21, 22).
4. The container of any one of claims 1 to 3, wherein an overflow sump (19) surrounding the tank fittings (18) is incorporated in the vertex channel (34), the horizontal bottom parts (33) of the sump being disposed beneath those positions where the partition webs (30) defining the vertex channel (34) are attached to the sump (19).
5. The container of any one of claims 1 to 4, wherein one of the two partition webs (30) defining the vertex channel (34) is formed with a tangentially extending upper flange (49) for engagement by the insulating jacket (23).
6. The container of any one of claims 1 to 5, wherein the insulating jacket (23) comprises an inner skin (24), an outer skin (25) and insulating material (26) disposed therebetween, the

outer skin (25) being supported relative to the inner skin (24) by means of spacer rings (27) which are disposed in the same radial planes as the partition rings (13).

7. The container of claim 6, wherein the inner skin (24) is provided with corrugations (28) defining spacer members relative to the tank casing (10...12).
8. The container of claim 6 or 7, wherein the partition and spacer rings are configured as integral rings (43), the inner skin (24) of the insulating jacket (23) being attached to the side faces of said integral rings (43).
9. The container of claim 8, wherein through-holes (44) extend substantially across the full height of the integral rings (43), the inner skin (24) of the insulating jacket (23) being outwardly deformed in the region of said through-holes (44).
10. The container of any one of claims 1 to 9, wherein the insulating jacket (23) comprises an inner skin (24), an outer skin (25) and insulating material (26) disposed therebetween, and wherein spacer members (45) are provided each of which includes two cup-shaped insulating bodies (46) having their bottoms placed on the inner skin (24) and their edges engaging the tank casing (10...12) and the outer skin (25), respectively, said insulating bodies (46) being tightened against each other by means of a common connecting element (47) extending through a hole (48) in the inner skin (24).
11. The container of claim 10, wherein the radially inner insulating body (46) is fixed to the tank casing (10...12) and the connecting element (47) extends with clearance through the hole (48) in the inner skin (24).

Revendications

1. Conteneur-citerne à température contrôlée, comprenant: un réservoir monté dans un bâti de conteneur (16), l'enveloppe dudit réservoir consistant en une coque cylindrique (10) avec des raccords de réservoir (18) prévus à sa partie supérieure et deux organes de fond (11, 12),
une chemise d'isolation thermique (23) entourant l'enveloppe du réservoir sur tous ses côtés et comprenant deux orifices (21, 22) constitués à l'une de ses extrémités et l'un au-dessus de l'autre pour l'entrée et la sortie de



l'agent de contrôle de la température, et des cloisons (13, 14, 30, 36) disposées entre l'enveloppe du réservoir et la chemise isolante (23), lesdites cloisons définissant des compartiments qui communiquent par l'intermédiaire de trous traversants (31, 32, 37 ... 42) les uns avec les autres de manière que l'agent de contrôle de la température soit forcé de passer autour de l'ensemble de l'enveloppe du réservoir le long d'un parcours s'étendant sur deux fois la longueur du réservoir et entourant les raccords de réservoir (18),

caractérisé en ce que

la chemise isolante (23) entoure l'enveloppe du réservoir avec un espacement qui est sensiblement constant sur toute sa longueur, et

les cloisons comprennent plusieurs anneaux de séparation (13, 14) entourant l'enveloppe du réservoir dans des plans radiaux et deux profilés de séparation (30, 36) s'étendant parallèlement de chaque côté de la ligne de sommet du réservoir, lesdits anneaux (13, 14) et profilés (30, 36) de séparation formant un parcours d'écoulement qui passe dans une direction dans une canalisation de sommet (34) définie par les profilés de séparation (30, 36) et entourant lesdits raccords de réservoir (18), et dans l'autre direction le long d'un parcours sinueux défini par les anneaux de séparation (13, 14) et recouvrant toutes les régions restantes de l'enveloppe du réservoir.

2. Conteneur selon la revendication 1, dans lequel le réservoir est relié aux organes d'extrémité (15) du bâti de conteneur (16) au moyen d'anneaux d'extrémité (14) fixés à ses deux organes de fonds (11, 12), lesdits anneaux d'extrémité définissant d'autres cloisons qui prolongent le parcours d'écoulement.
3. Conteneur selon la revendication 2, dans lequel la canalisation de sommet (34) relie des trous traversants (32) formés dans les deux anneaux d'extrémité (14) et se termine à une extrémité par un compartiment (35) en forme de coin qui entoure celui (22) desdits orifices (21, 22) qui est sur le dessus.
4. Conteneur selon l'une quelconque des revendications 1 à 3, dans lequel une cuve de trop-plein (19) entourant les raccords de réservoir (18) est incorporée à la canalisation de sommet (34), les parties de fond horizontales (33) de la cuve étant disposées au-dessous de positions dans lesquelles les profilés de séparation (30) définissant la canalisation de sommet (34) sont fixés à la cuve (19).

5. Conteneur selon l'une quelconque des revendications 1 à 4, dans lequel l'un des deux profilés de séparation (30) qui définissent la canalisation de sommet (34) comprend une bride supérieure (49) s'étendant tangentiellement et destinée à coopérer avec la chemise isolante (23).
6. Conteneur selon l'une quelconque des revendications 1 à 5, dans lequel la chemise isolante (23) comprend une peau interne (24), une peau externe (25) et un matériau isolant (26) disposé entre elles, la peau externe (25) étant supportée par rapport à la peau interne (24) par des anneaux d'écartement (27) qui sont disposés dans les mêmes plans radiaux que les anneaux de séparation (13).
7. Conteneur selon la revendication 6, dans lequel la peau interne (24) est munie d'ondulations (28) définissant des organes d'écartement par rapport à l'enveloppe (10 ... 12) du réservoir.
8. Conteneur selon la revendication 6 ou 7, dans lequel les anneaux de séparation et d'écartement sont configurés sous forme d'anneaux d'un seul tenant (43), la peau interne (24) de la chemise isolante (23) étant fixée aux faces latérales desdits anneaux d'un seul tenant (43).
9. Conteneur selon la revendication 8, dans lequel des trous traversants (44) s'étendent sensiblement sur la totalité de la hauteur des anneaux d'un seul tenant (43), la peau interne (24) de la chemise isolante (23) étant déformée vers l'extérieur dans la région desdits trous traversants (44).
10. Conteneur selon l'une quelconque des revendications 1 à 9, dans lequel la chemise isolante (23) comprend une peau interne (24), une peau externe (25) et un matériau isolant (26) disposé entre elles, et dans lequel sont prévus des organes d'écartement (45), dont chacun comprend deux corps isolants en forme de cuvette (46) dont les fonds sont placés sur la peau interne (24) et les bords sont en contact avec l'enveloppe (10 ... 12) du réservoir et la peau externe (25), respectivement, lesdits corps isolants (46) étant serrés l'un contre l'autre au moyen d'un élément de connexion (47) passant par un trou (48) de la peau interne (24).
11. Conteneur selon la revendication 10, dans lequel le corps isolant (46) qui est radialement à l'intérieur est fixé à l'enveloppe (10 ... 12) du



réservoir et l'élément de connexion (47) s'étend avec un certain jeu à travers le trou (48) de la peau interne (24).

Ansprüche

1. Temperierbarer Tankcontainer, umfassend
 - einen in einem Containerrahmen (16) montierten Tank, dessen Körper aus einer zylindrischen Schale (10) mit an ihrem Scheitel angeordneten Tankarmaturen (18) und zwei Bodenteilen (11, 12) besteht,
 - einen den Tankkörper allseitig umgebenden Wärmeisolierrmantel (23), an dessen einer Stirnseite zwei Öffnungen (21, 22) zum Ein- und Auslaß eines Temperierungsmediums übereinander angeordnet sind, und
 - zwischen Tankkörper und Isolierrmantel (23) vorgesehene, Kammern bildende Trennwände (13, 14, 30, 36), wobei die Kammern über Durchbrüche (31, 32, 37... 42) derart miteinander verbunden sind, daß eine Umströmung des gesamten Tankkörpers mit dem Temperierungsmedium längs eines die Tanklänge zweimal überstreichenden und die Tankarmaturen (18) umschließenden Weges erzwungen wird,
 - dadurch gekennzeichnet,
 - daß der Isolierrmantel (23) den Tankkörper in einem im wesentlichen überall konstanten Anstand umgibt, und
 - daß die Trennwände mehrere den Tankkörper in Radialebenen umgebende Trennringe (13, 14) sowie zwei zu beiden Seiten der Scheitellinie des Tanks parallele Trennstege (30, 36) umfassen, wobei die Trennringe (13, 14) und Trennstege (30, 36) einen Strömungsweg bilden, der in einer Richtung durch einen zwischen den Trennstegen (30, 36) gebildeten und die Tankarmaturen (18) umgebenden Scheitelkanal (34) führt und in der anderen Richtung einem von den Trennringen (13, 14) definierten und alle übrigen Bereiche des Tankkörpers überstreichenden mäanderförmigen Weg folgt.
2. Container nach Anspruch 1, wobei der Tank mit den Stirnteilen (15) des Containerrahmens (16) über an seinen beiden Bodenteilen (11, 12) angesetzte Stirnringe (14) verbunden ist, die weitere, den Strömungsweg verlängernde Trennwände bilden.
3. Container nach Anspruch 2, wobei der Scheitelkanal (34) in den beiden Stirnringen (14) ausgebildete Durchbrüche (32) verbindet und am einen Ende in einen die obere (22) der

Öffnungen (21, 22) umschließenden keilförmigen Raum (35) übergeht.

4. Container nach einem der Ansprüche 1 bis 3, wobei in den Scheitelkanal (34) eine die Tankarmaturen (18) umgebende Überlaufwanne (19) einbezogen ist, deren horizontale Bodenteile (33) unterhalb derjenigen Stellen liegen, an denen die den Scheitelkanal (34) bildenden Trennstege (30) an der Wanne (19) ansetzen.
5. Container nach einem der Ansprüche 1 bis 4, wobei einer der beiden den Scheitelkanal (34) bildenden Trennstege (30) einen tangential verlaufenden oberen Flansch (49) zum Einhängen des Isolierrmantels (23) aufweist.
6. Container nach einem der Ansprüche 1 bis 5, wobei der Isolierrmantel (23) eine Innenhaut (24), eine Außenhaut (25) und dazwischen angeordnetes Isoliermaterial (26) umfaßt, und wobei die Außenhaut (25) gegenüber der Innenhaut (24) durch Distanzringe (27) abgestützt ist, die in den gleichen Radialebenen wie die Trennringe (13) angeordnet sind.
7. Container nach Anspruch 6, wobei die Innenhaut (24) mit Ausformungen (28) versehen ist, die Distanzstücke gegenüber dem Tankkörper (10...12) bilden.
8. Container nach Anspruch 6 oder 7, wobei die Trenn- und die Distanzringe als einstückige Ringe (43) ausgebildet sind und die Innenhaut (24) des Isolierrmantels (32) an den Seitenflächen dieser einstückigen Ringe (43) befestigt ist.
9. Container nach Anspruch 8, wobei Durchbrüche (44) im wesentlichen die volle Höhe der einstückigen Ringe (43) einnehmen und die Innenhaut (24) des Isolierrmantels (23) im Bereich dieser Durchbrüche (44) nach außen verformt ist.
10. Container nach einem der Ansprüche 1 bis 9, wobei der Isolierrmantel (23) eine Innenhaut (24), eine Außenhaut (25) und dazwischen angeordnetes Isoliermaterial (26) umfaßt, und wobei Distanzstücke (45) vorgesehen sind, deren jedes zwei mit ihren Böden auf die Innenhaut (24) aufgesetzte und mit ihren Rändern am Tankkörper (10...12) bzw. der Außenhaut (25) anliegende napfförmige Isolierkörper (46) umfaßt, die durch ein gemeinsames, eine Öffnung (48) in der Innenhaut (24) durchsetzendes Verbindungselement (47) gegeneinander festgezogen sind.



11. Container nach Anspruch 10, wobei der radial innere Isolierkörper (46) am Tankkörper (10...12) fixiert ist und das Verbindungselement (47) die Öffnung (48) in der Innenhaut (24) mit Spiel durchsetzt.

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