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PANEL-FORM MICROPHONES

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DESCRIPTION

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TECHNICAL FIELD

The invention relates to microphones and more particularly to microphones comprising panel-form acoustic elements.

BACKGROUND ART

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It is known from GB-A-2262861 to suggest a panel-form loudspeaker comprising:-

a resonant multi-mode radiator element being a unitary sandwich panel formed of two skins of material with a spacing core of transverse cellular construction, wherein
25 the panel is such as to have ratio of bending stiffness (B), in all orientations, to the cube power of panel mass per unit surface area (μ) of at least 10;

a mounting means which supports the panel or attaches

to it a supporting body, in a free undamped manner;

and an electro-mechanical drive means coupled to the panel which serves to excite a multi-modal resonance in the radiator panel in response to an electrical input within a
5 working frequency band for the loudspeaker.

DISCLOSURE OF INVENTION

Embodiments of the present invention use members of nature, structure and configuration achievable generally and/or specifically by implementing teachings of our co-
5 pending PCT application no. (our case P.5711) of even date herewith. Such members thus have capability to sustain and propagate input vibrational energy by bending waves in operative area(s) extending transversely of thickness often but not necessarily to edges of the member(s); are
10 configured with or without anisotropy of bending stiffness to have resonant mode vibration components distributed over said area(s) beneficially for acoustic coupling with ambient air; and have predetermined preferential locations or sites within said area for transducer means,
15 particularly operationally active or moving part(s) thereof effective in relation to acoustic vibrational activity in said area(s) and signals, usually electrical, corresponding to acoustic content of such vibrational activity. Uses are envisaged in co-pending International application No. (our
20 file P.5711) of even date herewith for such members as or in "passive" acoustic devices without transducer means, such as for reverberation or for acoustic filtering or for acoustically "voicing" a space or room; and as or in



"active" acoustic devices with transducer means, such as in a remarkably wide range of sources of sound or loudspeakers when supplied with input signals to be converted to said sound, or in such as microphones when exposed to sound to
5 be converted into other signals.

This invention is particularly concerned with active acoustic devices in the form of microphones.

Members as above are herein called distributed mode acoustic radiators and are intended to be characterised as
10 in the above PCT application and/or otherwise as specifically provided herein.

The invention is a panel-form microphone characterised by a distributed mode acoustic member and a transducer coupled wholly and exclusively to the member to produce a
15 signal in response to resonance of the member due to incident acoustic energy. The member may be mounted in a surrounding frame by means of an interposed resilient support.

The panel-form microphone may be at least two said
20 transducers at spaced locations on the member. The panel-form microphone may comprise a further transducer on the member to produce a signal in response to resonance of the member due to incident acoustic energy, and by means for comparing the signal generated by the said further
25 transducer with that of those generated by the said transducer(s).

The comparison means may comprise a signal receiver and conditioner and signal output means.



The distributed mode acoustic member may be a stiff lightweight panel having a cellular core sandwiched between skins.

The or each transducer may be a piezo-electric device.

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BRIEF DESCRIPTION OF DRAWINGS

The invention is diagrammatically illustrated, by way of example, in the accompanying drawings, in which:-

Figure 1 is a diagram showing a distributed-mode loudspeaker as described and claimed in our co-pending
10 International application No... (our case P.5711);

Figure 2a is a partial section on the line A-A of Figure 1;

Figure 2b is an enlarged cross-section through a distributed mode radiator of the kind shown in Figure 2a
15 and showing two alternative constructions;

Figure 3 is a diagram of an embodiment of distributed-mode microphone according to the present invention, and

Figure 4 is a perspective view of a vibration transducer.

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BEST MODES FOR CARRYING OUT THE INVENTION

Referring to Figure 1 of the drawings, there is shown a panel-form loudspeaker (81) of the kind described and claimed in our co-pending International application (our case P.5711) of even date herewith comprising a rectangular
5 frame (1) carrying a resilient suspension (3) round its inner periphery which supports a distributed mode sound radiating panel (2). A transducer (9) as described in detail with reference to our co-pending International



application (our cases P.5683/4/5) of even date herewith,
is mounted wholly and exclusively on or in the panel (2) at
a predetermined location defined by dimensions x and y , the
position of which location is calculated as described in
5 our co-pending International application (our case P.5711)
of even date herewith, to launch bending waves into the
panel to cause the panel to resonate to radiate an acoustic
output. The transducer (9) is driven by a signal amplifier
(10), e.g. an audio amplifier, connected to the transducer
10 by conductors (28).

Amplifier loading and power requirements can be
entirely normal, similar to conventional cone type
speakers, sensitivity being of the order of 86 - 88dB/watt
under room loaded conditions. Amplifier load impedance is
15 largely resistive at 6 ohms, power handling 20-80 watts.
Where the panel core and/or skins are of metal, they may be
made to act as a heat sink for the transducer to remove
heat from the motor coil of the transducer and thus improve
power handling.

20 Figures 2a and 2b are partial typical cross-sections
through the loudspeaker (81) of Figure 1. Figure 2a shows
that the frame (1), surround (3) and panel (2) are
connected together by respective adhesive-bonded joints
(20). Suitable materials for the frame include lightweight
25 framing, e.g. picture framing of extruded metal e.g.
aluminium alloy or plastics. Suitable surround materials
include resilient materials such as foam rubber and foam
plastics. Suitable adhesives for the joints (20) include



epoxy, acrylic and cyano-acrylate etc. adhesives.

Figure 2b illustrates, to an enlarged scale, that the panel (2) is a rigid lightweight panel having a core (22) e.g. of a rigid plastics foam (97) e.g. cross linked
5 polyvinylchloride or a cellular matrix (98) i.e. a honeycomb matrix of metal foil, plastics or the like, with the cells extending transversely to the plane of the panel, and enclosed by opposed skins (21) e.g. of paper, card, plastics or metal foil or sheet. Where the skins are of
10 plastics, they may be reinforced with fibres e.g. of carbon, glass, Kevlar or the like in a manner known per se to increase their modulus.

Envisaged skin layer materials and reinforcements thus include carbon, glass, Kevlar (RTM), Nomex (RTM) i.e.
15 aramid etc. fibres in various lays and weaves, as well as paper, bonded paper laminates, melamine, and various synthetic plastics films of high modulus, such as Mylar (RTM), Kaptan (RTM), polycarbonate, phenolic, polyester or related plastics, and fibre reinforced plastics, etc. and
20 metal sheet or foil. Investigation of the Vectra grade of liquid crystal polymer thermoplastics shows that they may be useful for the injection moulding of ultra thin skins or shells of smaller size, say up to around 30cm diameter. This material self forms an orientated crystal structure in
25 the direction of injection, a preferred orientation for the good propagation of treble energy from the driving point to the panel perimeter.

Additional such moulding for this and other

thermoplastics allows for the mould tooling to carry location and registration features such as grooves or rings for the accurate location of transducer parts e.g. the motor coil, and the magnet suspension. Additional with
5 some weaker core materials it is calculated that it would be advantageous to increase the skin thickness locally e.g. in an area or annulus up to 150% of the transducer diameter, to reinforce that area and beneficially couple vibration energy into the panel. High frequency response
10 will be improved with the softer foam materials by this means.

Envisaged core layer materials include fabricated honeycombs or corrugations of aluminium alloy sheet or foil, or Kevlar (RTM), Nomex (RTM), plain or bonded papers,
15 and various synthetic plastics films, as well as expanded or foamed plastics or pulp materials, even aerogel metals if of suitably low density. Some suitable core layer materials effectively exhibit usable self-skinning in their manufacture and/or otherwise have enough inherent stiffness
20 for use without lamination between skin layers. A high performance cellular core material is known under the trade name 'Rohacell' which may be suitable as a radiator panel and which is without skins. In practical terms, the aim is for an overall lightness and stiffness suited to a
25 particular purpose, specifically including optimising contributions from core and skin layers and transitions between them.

Several of the preferred formulations for the panel

employ metal and metal alloy skins, or alternatively a carbon fibre reinforcement. Both of these, and also designs with an alloy Aerogel or metal honeycomb core, will have substantial radio frequency screening properties which should be important in several EMC applications. Conventional panel or cone type speakers have no inherent EMC screening capability.

In addition the preferred form of piezo and electro dynamic transducers have negligible electromagnetic radiation or stray magnet fields. Conventional speakers have a large magnetic field, up to 1 metre distant unless specific compensation counter measures are taken.

Where it is important to maintain the screening in an application, electrical connection can be made to the conductive parts of an appropriate DML panel or an electrically conductive foam or similar interface may be used for the edge mounting.

The suspension (3) may damp the edges of the panel (2) to prevent excessive edge movement of the panel. Additionally or alternatively, further damping may be applied, e.g. as patches, bonded to the panel in selected positions to damp excessive movement to distribute resonance equally over the panel. The patches may be of bitumen-based material, as commonly used in conventional loudspeaker enclosures or may be of a resilient or rigid polymeric sheet material. Some materials, notably paper and card, and some cores may be self-damping. Where desired, the damping may be increased in the construction

of the panels by employing resiliently setting, rather than rigid setting adhesives.

Effective said selective damping includes specific application to the panel including its sheet material of means permanently associated therewith. Edges and corners can be particularly significant for dominant and less dispersed low frequency vibration modes of panels hereof. Edge-wise fixing of damping means can usefully lead to a panel with its said sheet material fully framed, though their corners can often be relatively free, say for desired extension to lower frequency operation. Attachment can be by adhesive or self-adhesive materials. Other forms of useful damping, particularly in terms of more subtle effects and/or mid- and higher frequencies can be by way of suitable mass or masses affixed to the sheet material at predetermined effective medial localised positions of said area.

A panel as described above is a good receiver of sound which appears as acoustic vibration over the panel. A preferably lightweight panel structure aids sensitivity and the vibration may be sensed by one and preferably more simple bending transducers e.g. of the piezo variety as described in Figure 4 below. A plurality of transducers and transducer placement positions optimises the quality of coupling from the distributed panel vibrations to the desired electrical output signal. Placement should be in position(s) of high modal density, inboard of the panel, while the panel itself should have the preferred actual or

equivalent geometry for good modal distribution.

Sound energy incident on the panel is converted into free mode vibration. This vibration may be sensed by optical or electrodynamic vibration transducers and the
5 result is a microphone. For non-critical applications a single sensor is effective, placed at an equivalent, optimised driving point.

For superior quality the non reciprocal nature of the transduction principle must be considered. Two factors are
10 pertinent; firstly, some frequency dependent equalisation to reach a flat frequency response, and secondly, the need to capture a broader sampling of the complex vibrations of the acoustic panel. A minimum of three transducers is indicated; they may be inexpensive piezo electric benders
15 with their outputs connected in parallel. Alternatively larger area polymer piezo films may be applied, with suitable geometric pickup patterning to define the vibration integration areas for the required optimisation of sensitivity versus frequency response.

20 For microphone applications it is advantageous that the panel be light to provide the best match between the radiation impedance of the air and the panel. Higher sensitivity is achieved with lower mass panels. For a single transducer the calculations for the theoretical
25 model indicates an optimal location at a panel corner since all vibrational modes are 'voiced' at the corners.

Figure 3 illustrates a distributed mode panel (2) according to the present invention e.g. of the kind shown

in Figures 1 and 2, intended for use as a sound receiver or microphone. Although not shown in the drawing, the panel (2) is mounted in a surrounding frame (1) and is attached to the frame via a resilient suspension (3) in the manner
5 shown in Figures 1 and 2. The frame is suspended on a pair of wires (33), e.g. from a ceiling or on a floor standing frame (not shown).

The panel carries an array of four vibration transducers (63) spaced over the panel and which may be
10 piezo-electric transducers of the kind shown in Figure 4 below which are coupled in parallel to drive a signal receiver and conditioner (65) connected to an output (66).

Figure 4 shows a transducer (9) for a distributed mode panel (2) in the form of a crystalline disc-like piezo
15 bender (27) mounted on a disc (118), e.g. of brass, which is bonded to a face of the panel (2), e.g. by an adhesive bond (20). In operation an acoustic signal applied to the transducer (9) via leads (28) will cause the piezo disc (27) to bend and thus locally resiliently deform the panel
20 (2) to launch bending waves into the panel.

CLAIMS

1. A panel-form microphone characterised by a distributed mode acoustic member and a transducer coupled wholly and exclusively to the member to produce a signal in response
5 to resonance of the member due to incident acoustic energy.
2. A panel-form microphone according to claim 1, characterised in that the member is mounted in a surrounding frame by means of an interposed resilient support.
- 10 3. A panel-form microphone according to claim 1 or claim 2, characterised by at least two said transducers at spaced locations on the member.
4. A panel-form microphone according to any preceding claim, characterised in that the member is a stiff
15 lightweight panel having a cellular core sandwiched between skins.
5. A panel-form microphone according to any preceding claim, characterised in that the or each transducer is a piezo-electric device.

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