

Particle Size Breakup Device and Its Performance
Estimation Method and Scale Up Method

BACKGROUND

Technical Field

【0001】 Generally, the present invention relates to mixers, and more specifically to a mixer of the so-called rotator-stator type that includes a stator having a plurality of openings or holes and a rotor that is disposed on the inner side of the stator and spaced by a particular gap away from the stator.

Background

【0002】 As shown in Fig. 1, it is general that the mixer of the so-called rotor-stator type generally comprises a mixer unit 4 that includes a stator 2 having a plurality of openings (holes) 1 and a rotor 3 disposed on the inner side of the stator 2 and spaced by a particular gap δ from the stator 2. Such mixer of the rotor-stator type is provided for subjecting a fluid being processed to the emulsification, dispersion, particle size breakup, mixing or any other similar process, by taking advantage of the fact that a high shear stress may be produced in the neighborhood of a gap between the stator 3 rotating at high-speeds and the fixed stator 2. This mixer is used for mixing or preparing the fluid being processed, and has a wide variety of applications in which foods, pharmaceutical medicines, chemical products and others can be manufactured.

【0003】 The mixer of the rotor-stator type may be classed according to the type of the circulation mode for the fluid being processed, that is, one being the externally circulated mixer in which the fluid liquid being processed may be circulated in the direction indicated by the arrow 5a in Fig. 2, and the other being the internally circulated mixer in which a liquid being processed may be circulated in the direction indicated by the arrow 5b in

Fig. 2.

【0004】 For the mixers of the rotor-stator type, many different configurations and circulation modes or systems have been proposed. For example, there is the Japanese patent application No. 2006-506174 in which the mixer of the rotor-stator type and the particle size breakup method are proposed, in which the mixer includes a stator having a plurality of openings (holes) and a rotor disposed on the inner side of the stator and spaced by a specific gap away from the stator. It is disclosed that the mixer may be used widely in the fields in the pharmaceutical medicines, nutrition supplement foods, other foods, chemical products, cosmetics and the like can be manufactured. It is also disclosed that the method disclosed enables the scale up function to be performed in the efficient, simple and easy manner.

【0005】 In addition, for those past years, several indices (theories) have been reported as the performance estimation method for the mixers having the different configurations.

【0006】 Not only for the mixer of the rotor-stator type as described above but also for all other type mixers, it is reported that, when the liquid-liquid dispersion operation in particular occurs, for example, the drop diameter size can be discussed in terms of the magnitude (smallness or greatness) of the values that can be obtained by calculating the average energy dissipation rate (Publications 1 and 2). In those publications 1 and 2, however, the method for calculating the average energy dissipation rates is not disclosed specifically.

【0007】 The publications 3 to 6 report several study cases that may be applied to each individual mixer and in which the results obtained by the respective experiments have been organized into the tabular forms. In those study cases (Publications 3 to 6), however, it is considered that the mixer's particle size breakup function is only affected by the gap between the rotor

and stator and by the openings (holes) on the stator. Only the information that is different for each different type mixer is reported.

【0008】 Several study cases are reported (Publications 7 and 8), in which the particle size breakup mechanism for the mixer of the rotor-stator type was considered and discussed. In those publications 7 and 8, it is suggested that the energy dissipation rates of the turbulent flow will contribute to the particle size breakup effect, and the particle size breakup effect may be affected by the frequency (shear frequency) of the turbulent flow with which the particle size breakup effect is placed under the shear stress of the fluid being processed.

【0009】 For the scale-up method for the mixer of the rotor-stator type, there are several reports (Publication 8) in which the final drop diameter (maximum stable drop diameter) can be obtained during the long-time mixer running period. This, however, is not practical in the actual production sites and is of no utility. Specifically, there are no reports regarding the study cases in which the processing (agitation and mixing) time of the mixer is the object for consideration, and those study cases are not useful enough to estimate the drop diameters that can be obtained during the particular mixer running period. Although it is reported that the drop diameters may be estimated by considering the mixer processing time, yet only the phenomenon (factual action) that is based on the actual measured values (experimental values) is reported. There are no study cases in which such phenomenon is analyzed theoretically.

【0010】 The following publication, which is the document related to the patent application, is cited herein for reference:

Japanese Patent Application No. 2005-506174

【0011】 The following publications, which are not related to the patent application, are cited herein for reference:

(1) David, J. T.; "Drop Sizes of Emulsions Related to Turbulent Energy

- Dissipation Rates”, Chem. Eng. Sci., 40, 839-842 (1985) and David J. T.; “A Physical Interpretation of Drop Sizes in Homogenizers;
- (2) Agitated Tanks, Including the Dispersion of Viscous Oils”, Chem. Eng. Sci., 42, 1671-1676 (1987);
- (3) Calabrese, R. V., M. K. Francis, V. P. Mishra and S. Phongikaroon; “Measurement and Analysis of Drop Size in Batch Rotor-Stator Mixer”, Proc. 10th European Conference on Mixing, pp. 149-156, Delft, the Netherlands (2000);
- (4) Calabrese, R. V., M. K. Francis, V. P. Mishra, G. A. Padron and S. Phongikaroon; “Fluid Dynamic and Emulsification in High Shear Mixers”, Proc. 3rd World Congress on Emulsion, pp. 1-10, Lyon, France (2002);
- (5) Maa, Y. F., and C. Hsu, and C. Hsu; “Liquid-Liquid Emulsification by Rotor/Stator Homogenization”, J. Controlled. Release, 38, 219-228 (1996);
- (6) Barailler, F., M. Heniche and P. A. Tanguy; “CFD Analysis of a Rotor-Stator Mixer with Viscous Fluids”, Chem. Eng. Sci., 61, 2888-2894 (2006);
- (7) Utomo, A. T., M. Baker and A. W. Pacek; “Flow Pattern, Periodicity and Energy Dissipation in a Batch Rotor-Stator Mixer”, Chem. Eng. Res. Des., 86, 1397-1409 (2008);
- (8) Porcelli, J.; “The Science of Rotor-Stator Mixers”, Food Process, 63, 60-66 (2002);
- (9) Urban, K.; “Rotor-Stator and Disc System for Emulsification Processes”, Chem. Eng. Technol., 29, 24-31(2006)

SUMMARY OF THE INVENTION

【0012】 In the patent application cited above, the superiority (performance) of the particular mixer and the value range of the design on which the mixer is based are disclosed, but the theoretical grounds on which the value range of the high-performance mixer design is based are not described. The kinds and configurations of the high performance mixer are not described

specifically.

【0013】 It may be appreciated from the above description that, for those past years, several indices (theories) have been reported as the performance estimation method for the mixers having the different configurations. In most cases, however, those indices can only be applied to the individual mixers having the same configuration. In the actual cases, however, they cannot be applied to the mixers of the various types having the different configurations.

【0014】 As noted above, there are almost no study cases in which the performance estimation method and the scale-up method for those mixers of the rotor-stator type have been defined. There are also no study cases in which those methods can be applied to the mixers of the various types having the different configurations, and the data on the results obtained by the experiments on such study cases have not been organized properly and comprehensively.

【0015】 For the performance estimation method and scale-up method for the mixers of the rotor-stator type according to the prior art, in most cases, the final drop diameters (maximum stable drop diameters) were obtained by using the small scale device for each individual mixer and permitting the device to run for the long time period, and were then estimated. More specifically, in the prior art, there is no estimation method that can be used to estimate the drop diameters that would be obtained by using the large-scale devices (actual production installation) for the mixers of the various types and permitting such large-scale devices to run during the particular time period, or there is no estimation method that can be used to estimate the particular drop diameters obtained during the particular running time or the processing or agitating time required until such particular drop diameters can be obtained.

【0016】 For the above reasons, the performance estimation method and

design (development and fabrication) method for the mixers were actually performed on the trial and error basis by using the actually used processing liquids.

【0017】 It is, therefore, objects of the present invention to provide a comprehensive performance estimation method that is established so that it can be applied to the mixers of the various types having the various configurations that are likely to be affected mostly by the gap in particular between the rotor and stator or the mixers of the various types having the different circulation systems; to provide the design method that is established by taking the running conditions (processing time) for such mixers into consideration; and to provide the manufacturing method (particle size breakup method) that is established to be used for manufacturing the foods, pharmaceutical medicines and the like by using the above described performance estimation method and design method.

【0018】 In a first aspect of the invention as defined in Claim 1, it is characterized by the fact that the mixer of the rotor-stator type comprises a mixer unit that includes a stator having a plurality of openings (holes) and a rotor that is disposed on the inner side of the stator and spaced by a particular gap away from the stator, wherein when a fluid being processed by the mixer is subjected to the emulsification, diffusion, particle size breakup, mixing or any other similar process, the Equation 1 below is calculated so that a particular drop diameter for the fluid being processed by the mixer can be obtained during a particular mixer running time, and the mixer is thus designed by estimating the particular mixer running time and the drop diameters thus obtained for the fluid being processed.

$$\begin{aligned}\varepsilon_t &= \varepsilon_t \int_{s,h} t_m \\ &= \left[A \pi^4 n_r (D + 2\delta) D^3 h \left(\frac{4\ell}{d} + 1 \right) \left(\frac{N_p}{N_{qd} \pi^2} - 1 \right) \right] \left(\frac{N^4}{V} t_m \right) \\ &= C_h \left(\frac{N^4}{V} t_m \right) \dots \text{the Equation 1}\end{aligned}$$

【0019】 In the Equation 1,

ε_t : Total Energy Dissipation Rate [m²/s³] ε

ε_1 : Local Energy Dissipation Rate in Stator's Opening [m²/s³]

f_{s-h} : Shear Frequency

t_m : Mixing Time [s]

A: Stator's Opening Rate [-]

N_r : Number of Rotor Blades [-]

D: Diameter of Rotor [m]

δ : Gap between Rotor and Stator [m]

h: Height of Stator [m]

l: Thickness of Stator [m]

d: Hole Diameter of Stator [m]

N_p : Number of Powers [-]

N_{pd} : Number of Flow Rates [-]

N: Number of Rotation [1/s]

V: Amount of Liquid [m³]

C_h : Configuration Dependent Term for Gap [m⁵]

【0020】 In a second aspect of the invention as defined in Claim 2, it is characterized by the fact that in the mixer as defined in Claim 1, the stator and the rotor are arranged in such a manner that they can be moving closer to each other or moving farther away from each other in the direction in which the rotary shaft of the rotor extends.

【0021】 In a third aspect of the invention as defined in Claim 3, it is

characterized by the fact that in the mixer defined in Claim 1 or 2, the mixer includes a plurality of stators each having a different circumferential diameter and a rotor disposed on the inner side of each of the plurality of stators and spaced by a particular gap away from each of the stators.

【0022】 In a fourth aspect of the invention as defined in Claim 4, it is characterized by the fact that in the mixer as defined in any one of Claims 1 through 3, the fluid being processed is introduced into the gap between the stators and the rotor disposed on the inner side of each of the stators and spaced by the gap away from each of the stators.

【0023】 In a fifth aspect of the invention as defined in Claim 5, it is characterized by the fact that in the mixer as defined in any one of Claims 1 through 4, the rotor has a plurality of agitating blades extending radially from the center of rotation.

【0024】 In a sixth aspect of the invention as defined in Claim 6, it is characterized by the fact that the method for estimating the performance of the mixer of the rotor-stator type comprising a mixer unit that includes a stator having a plurality of openings (holes) and a rotor disposed on the inner side of the stator and spaced by a particular gap away from the stator is provided, wherein the method for estimating the performance of the mixer includes the steps of: using the Equation 1 below to determine the total energy dissipation rate ε_t , measuring the respective sizes of the rotor and stator and the power requirements and flow rates during the mixer running time period which are included in the Equation 1 below, and estimating the magnitude (smallness or greatness) of the values of the configuration depending term in the gap, which are specific to each of the mixers of the different types.

$$\varepsilon_t = \varepsilon_{t,s,h} t_m$$

$$= \left[A \pi^4 n_r (D + 2\delta) D^3 h \left(\frac{4\ell}{d} + 1 \right) \left(\frac{N_p}{N_{qd} \pi^2} - 1 \right) \right] \left(\frac{N^4}{V} t_m \right)$$

$$= C_h \left(\frac{N^4}{V} t_m \right) \dots \text{the Equation 1}$$

【0025】 In the Equation 1,

ε_t : Total Energy Dissipation Rate [m^2/s^3] ε

ε_1 : Local Energy Dissipation Rate in Stator's Openg [m^2/s^3]

f_{s_h} : Shear Frequency

t_m : Mixing Time [s]

A: Stator's Opening Rate [-]

N_r : Number of Rotor Blades [-]

D: Diameter of Rotor [m]

δ : Gap between Rotor and Stator [m]

h: Height of Stator [m]

l: Thickness of Stator [m]

d: Hole Diameter of Stator [m]

N_p : Number of Powers [-]

N_{pd} : Number of Flow Rates [-]

N: Number of Rotation [1/s]

V: Amount of Liquid [m^3]

C_h : Configuration Dependent Term for Gap [m^5]

【0026】 In a seventh aspect of the invention as defined in Claim 7, it is characterized by the fact that a method for scaling-up or scaling down the mixer of the rotor-stator type comprising a mixer unit that includes a stator having a plurality of openings (holes) and a rotor disposed on the inner side of the stator and spaced by a particular gap away from the stator is provided, wherein the method includes the steps of: using the Equation 1

below to calculate the values of the total energy dissipation rate ε_t on the experimental mixer installation and/or on the pilot plant mixer installation, and matching the values of the total energy dissipation rate: ε_t , thus obtained against the values for the total energy dissipation rate: ε_t obtained on the actual mixer installation that is specifically intended for scaling up or scaling down the mixer so that the former ε_t values can conform with the latter ε_t values..

$$\varepsilon_t = \varepsilon_t f_{s,h} t_m$$

$$= \left[A \pi^4 n_r (D + 2\delta) D^3 h \left(\frac{4\ell}{d} + 1 \right) \left(\frac{N_p}{N_{qd} \pi^2} - 1 \right) \right] \left(\frac{N^4}{V} t_m \right)$$

$$= C_h \left(\frac{N^4}{V} t_m \right) \dots \text{the Equation 1}$$

【0027】 In the Equation 1,

ε_t : Total Energy Dissipation Rate [m²/s³] ε

ε_1 : Local Energy Dissipation Rate in Stator's Opening [m²/s³]

$f_{s,h}$: Shear Frequency

t_m : Mixing Time [s]

A: Stator's Opening Rate [-]

N_r : Number of Rotor Blades [-]

D: Diameter of Rotor [m]

δ : Gap between Rotor and Stator [m]

h: Height of Stator [m]

l: Thickness of Stator [m]

d: Hole Diameter of Stator [m]

N_p : Number of Powers [-]

N_{pd} : Number of Flow Rates [-]

N: Number of Rotation [1/s]

V: Amount of Liquid [m³]

C_h : Configuration Dependent Term for Gap [m^5]

【0028】 In an eighth aspect of the invention as defined in Claim 8, it is characterized by the fact that foods, pharmaceutical medicines or chemical products are manufactured by subjecting the fluid being processed to the emulsification, dispersion, particle size breakup, mixing or any other like process that occurs by using the mixer of the rotor-stator type comprising the mixer unit that includes the stators each having the plurality of openings (holes) and the rotor disposed on the inner side of each of the stators and spaced by the particular gap away from each of the stators, wherein the foods, pharmaceutical medicines or chemical products are manufactured by subjecting the fluid being processed to the emulsification, dispersion, drop breakup, mixing or any other like process by using the Equation 1 below and estimating the mixer running time and the drop diameters that are thus obtained during the mixer running time.

$$\begin{aligned}\varepsilon_t &= \varepsilon_t \int_{s,h} t_m \\ &= \left[A \pi^4 n_r (D + 2\delta) D^3 h \left(\frac{4\ell}{d} + 1 \right) \left(\frac{N_p}{N_{qd} \pi^2} - 1 \right) \right] \left(\frac{N^4}{V} t_m \right) \\ &= C_h \left(\frac{N^4}{V} t_m \right) \dots \text{the Equation 1}\end{aligned}$$

【0029】 In the Equation 1,

ε_t : Total Energy Dissipation Rate [m^2/s^3] ε

ε_1 : Local Energy Dissipation Rate in Stator's Opening [m^2/s^3]

f_{s_h} : Shear Frequency

t_m : Mixing Time [s]

A : Stator's Opening Rate [-]

N_r : Number of Rotor Blades [-]

D : Diameter of Rotor [m]

δ : Gap between Rotor and Stator [m]

h: Height of Stator [m]

l: Thickness of Stator [m]

d: Hole Diameter of Stator [m]

N_p : Number of Powers [-]

N_{pd} : Number of Flow Rates [-]

N: Number of Rotation [1/s]

V: Amount of Liquid [m³]

C_h : Configuration Dependent Term for Gap [m⁵]

【0030】 In a ninth aspect of the invention as defined in Claim 9, it is characterized by the fact that the method for manufacturing foods, pharmaceutical medicines or chemical products by subjecting the fluid being processed to the emulsification, dispersion, particle size breakup, mixing or any other like process that occurs by using the mixer of the rotor-stator type comprising the mixer unit that includes the stator having the plurality of openings (holes) and the rotor disposed inside the stator and spaced by the particular gap away from the stator, wherein the method includes the steps of: using the Equation 1 below, estimating the mixer running time and the drop breakup diameters thus obtained by the calculating step during the mixer running time, and manufacturing the foods, pharmaceutical medicines or chemical products based upon the estimating step.

$$\varepsilon_t = \varepsilon_t f_{s,h} t_m$$

$$\begin{aligned} &= \left[A \pi^4 n_r (D + 2\delta) D^3 h \left(\frac{4\ell}{d} + 1 \right) \left(\frac{N_p}{N_{qd} \pi^2} - 1 \right) \right] \left(\frac{N^4}{V} t_m \right) \\ &= C_h \left(\frac{N^4}{V} t_m \right) \cdots \text{the Equation 1} \end{aligned}$$

【0031】 In the Equation 1,

ε_t : Total Energy Dissipation Rate [m²/s³] ε

ε_1 : Local Energy Dissipation Rate in Stator's Opening [m^2/s^3]

f_{s_h} : Shear Frequency

t_m : Mixing Time [s]

A: Stator's Opening Rate [-]

N_r : Number of Rotor Blades [-]

D: Diameter of Rotor [m]

δ : Gap between Rotor and Stator [m]

h: Height of Stator [m]

l: Thickness of Stator [m]

d: Hole Diameter of Stator [m]

N_p : Number of Powers [-]

N_{pd} : Number of Flow Rates [-]

N: Number of Rotation [1/s]

V: Amount of Liquid [m^3]

C_h : Configuration Dependent Term for Gap [m^5]

【0032】 In the present invention, the index that is called the total energy dissipation rate: ε_t is utilized. The total energy dissipation rate: ε_t that may be applied to each of the mixers of the various types having the different configurations and circulation systems that are offered from each of the manufacturers can be calculated separately from the geometrical sizes, running powers and flow rates that are specific to the rotor and stator. The total energy dissipation rate: ε_t may be expressed by separating the values of the configuration dependent term from the values of the running condition dependent term, both of which terms are specific to the gap for each individual mixer.

【0033】 When the performance for each individual mixer is estimated by the values for the total energy dissipation rate: ε_t , for example, when the mixer performance is estimated by the particle size breakup trend, the magnitude (smallness or greatness) of the values for the configuration dependent term

as measured in the mixer gap can be used.

【0034】 When each individual mixer is to be scaled up or scaled down, the measured values of the total energy dissipation rate: ϵ_t , as coupled with the values of the configuration dependent term and running condition dependent term that are specific to the mixer gap may be used, and each individual mixer can be designed by matching the values of ϵ_t against the respective values of the above terms so that the former ϵ_t value can conform with the latter term values..

【0035】Based upon the discoveries described above, the mixer that provides the higher particle size breakup and emulsification efficiencies than any of the prior art mixers can be developed and designed as the high performance mixer theoretically and experimentally.

【0036】 In the present invention, specifically, the value range for the high performance mixer can be specified in terms of the values of the configuration depending term (factor) in the mixer gap, and can be applied to the performance estimation method for each individual mixer. In order to derive the values for the total energy dissipation rate: ϵ_t , the present invention proposes the equation that allows the value range not covered by any of the prior art (conventional) mixers to be established in terms of the values for the configuration depending terms (factors) in the mixer gap that may be obtained by using the above proposed equation. According to the present invention, the value range that cannot be calculated easily by using the prior art index (theory) or is difficult to be obtained unless it is measured actually can be established.

【 0037 】 According to the method for manufacturing the foods, pharmaceutical medicines or chemical products by subjecting the fluid being processed to the emulsification, dispersion, particle size breakup or mixing process that occurs by using the mixer of the rotor-stator type, the particular mixer running time and the drop diameters thus obtained during

the particular running time for the fluid being processed can be estimated by calculating the total energy dissipation rate: ε_t [m²/s³] using the Equation proposed by the present invention, and the foods, pharmaceutical medicines or chemical products having the desired drop diameters can thus be manufactured.

BRIEF DESCRIPTION OF DRAWINGS

【0038】

Fig. 1 is a perspective view illustrating a mixer unit that is included in the mixer of the so-called rotor-stator type;

Fig. 2 illustrates the mixer of the rotor-stator type that runs in the external circulation mode (the externally circulated mixer) and the mixer of the rotor-stator type that runs in the internal circulation mode (the internally circulated mixer),

Fig. 3 illustrates the mode in which the particle size breakup trend for the drop diameters can be investigated;

Fig. 4 illustrates the mode in which the experimental results on the mixer of the rotor-stator type that runs in the external circulation mode (the externally circulated mixer) can be used to estimate the performance of the mixer of the rotor-stator type that runs in the internal circulation mode (the internally circulated mixer);.

Fig. 5 illustrates the relationship (particle size breakup trend) between the processing (mixing) time and the drop diameter for the small-scale mixers of the rotor-stator type;

Fig. 6 illustrates the relationship (particle size breakup trend) between the total energy dissipation rate ε_t and the drop diameter for the small-scale mixer of the rotor-stator type.

Fig. 7 illustrates the relationships between the processing time and the drop diameter and the relationship between the processing time and the estimated value (theoretical value) for the small-scale mixer of the

rotor-stator type;

Fig. 8 illustrates the relationships between the processing time and the drop diameter and the relationship between the processing time and the estimated value (theoretical value) for another small-scale mixer of the rotor-stator type;

Fig. 9 illustrates the relationships between the processing time and the drop diameter and the relationship between the processing time and the estimated value (theoretical value) for the medium and large-scale mixers of the rotor-stator type;

Fig. 10 is a perspective view illustrating one example of the rotor that is included in the mixer of the rotor-stator type according to the present invention.

Fig. 11 is an exploded perspective view illustrating one example of the multi-staged emulsification mechanism that is employed in the mixer of the rotor-stator type according to the present invention;

Fig. 12 illustrates the direct injection system that is employed in the mixer of the rotor-stator type according to the present invention, in which (a) is a plan view and (b) is a side view; and

Fig. 13 illustrates the relationship (particle size breakup trend) between the total energy dissipation rate ϵ_t and the drop diameter when the nutrition regulated foods that are available on the commercial market are mixed together by using the mixer of the rotor-stator type.

BEST MODE OF EMBODYING THE INVENTION

【0039】 In describing the present invention below, the total energy dissipation rate ϵ_t which can be derived by calculating the Equation 1 given below and which is proposed by the present invention is used to discuss, compare or estimate the particle size breakup effect (particle size breakup trend) in the mixer of the rotor-stator type.

$$\begin{aligned}\varepsilon_t &= \varepsilon_t f_{s,h} t_m \\ &= \left[A \pi^4 n_r (D + 2\delta) D^3 h \left(\frac{4\ell}{d} + 1 \right) \left(\frac{N_p}{N_{pd} \pi^2} - 1 \right) \right] \left(\frac{N^4}{V} t_m \right) \\ &= C_h \left(\frac{N^4}{V} t_m \right) \dots \text{the Equation 1}\end{aligned}$$

【0040】 In the Equation 1,

ε_t : Total Energy Dissipation Rate [m^2/s^3] ε

ε_1 : Local Energy Dissipation Rate in Stator's Opening [m^2/s^3]

f_{s-h} : Shear Frequency

t_m : Mixing Time [s]

A: Stator's Opening Rate [-]

N_r : Number of Rotor Blades [-]

D: Diameter of Rotor [m]

δ : Gap between Rotor and Stator [m]

h: Height of Stator [m]

l: Thickness of Stator [m]

d: Hole Diameter of Stator [m]

N_p : Number of Powers [-]

N_{pd} : Number of Flow Rates [-]

N: Number of Rotation [1/s]

V: Amount of Liquid [m^3]

C_h : Configuration Dependent Term for Gap [m^5]

【0041】 As described above, the the total energy dissipation rate ε_t may be expressed in terms of the product obtained by multiplying the local energy dissipation rate ε_1 and the shear frequency f_{s-h} for the gap between the rotor and the stator.

【0042】 In the present invention, the mixer's performance can be estimated by measuring the magnitude (smallness or greatness) of the values for the

configuration dependent term: C_h [-] in the gap. Those values are specific to each of the mixers of the various types, and can be obtained by measuring the components included in the Equation 1 for deriving the total energy dissipation rate ε_t . mentioned earlier, such as the power requirements and flow rates during the running time and the respective sizes of the rotor and stator.

【0043】 As it is clear from the Equation 1 proposed by the present invention to allow the value for the total energy dissipation rate ε_t mentioned earlier to be derived, the value for the configuration dependent term: C_h [-] is specific to each of the mixers of the various types, and may vary depending on the gap between the rotor and stator δ [m], the stator's hole diameter d [m], the stator's number of holes n_s [-], the stator's thickness l [m], the number of flow rates N_{qd} [-], and the number of powers N_p [-].

【0044】 Then, the performance of each of the mixers of the various types can be estimated by comparing (estimating) the magnitude (smallness or greatness) of those values.

【0045】 Then, the performance of each of the mixers of the various types may be estimated by comparing (estimating) the values of the configuration depending term: C_h [-] for the stator which is one of the components included in the Equation 1 proposed by the present invention for deriving the total energy dissipation rate ε_t .

【0046】 More specifically, by comparing (estimating) one of the components included in the Equation 1 of the present invention for deriving the value for the total energy dissipation rate ε_t , that is, the configuration depending term: C_h [-] which is specific to each of the mixers of the various types, it is found that the performance of each of the mixers of the various types can be estimated and the high performance mixer can be designed (developed and fabricated) accordingly.

【0047】 By matching the values of the total energy dissipation rate ε_t

obtained on the experimental rotor-stator type mixer installation and/or on the pilot plant installation against the total energy dissipation rate ε_t the values for the total energy dissipation rate ε_t obtained on the actual mixer installation intended for scaling up or scaling down the mixer so that the former ε_t values can conform with the latter ε_t values, it is also found that the mixer can be scaled up or scaled down.

【0048】 By calculating the values for using the Equation 1 proposed by the present invention for deriving the total energy dissipation rate ε_t when the foods (such as dairy products, drinks, etc.), pharmaceutical medicines (such as the quasi-drugs) or chemical products (such as the cosmetics) are to be manufactured by subjecting the fluid being processed to the emulsification, dispersion, particle size breakup, mixing or any other process that occurs by using the mixer of the rotor-stator type, it is also found that those foods, pharmaceutical medicines or chemical products can be manufactured by estimating the mixer running time and the drop diameters obtained during the mixer running time for the fluid being processed.

【0049】 As noted, it has been demonstrated by the embodiment of the present invention that the nutritive components (which correspond to the components such as the fluid foods, the powder milks prepared for babies) which have been manufactured according to the present invention have the good taste feeling, physical properties, quality and the like, and are also excellent from the viewpoint of the hygienic care or workability. Therefore, the present invention should preferably be applied to the manufacture of the foods or pharmaceutical medicines, more particularly it should be applied to the foods. Much more preferably, it should be applied to the nutritive components or dairy products. The most particularly preferred application of the present invention is the nutritive components and dairy products that contain the highly concentrated composition.

【0050】 Change in the total energy dissipation rate ε_t versus the resulting change in the drop diameter (particle size breakup trend):

A liquid that simulates a dairy product is prepared so that it can be used in estimating the drop diameters. This liquid that simulates the dairy product contains the milk protein concentration (MPC, TMP (total milk protein)), rapeseed oil and water. Its composition and ratio are given in Table 1.

Table 1 Composition Ratio of Simulated Liquid for Milk Product

Composition	Milk Product Concentrate (MPC)	8.0%
	Rape Seed Oil	4.5%
	Water	87.5%
	Total.	100%
Ratio	Protein/Water	9.1%
	Oil/Protein	56.3%
	Oil/Water	5.1%
Properties	Density	1028 kg/m ³
	Viscosity	15 mPa·s

【0051】 The mixer performance was estimated on the experimental basis by checking the particle size breakup trend for the drop diameters. As shown in Fig. 3, the externally circulated unit was provided, and the drop diameters were measured on the middle way of the fluid path by using the laser diffraction-type particle size analyzer (SALD-2000 as offered by Shimazu Manufacturing Company).

【0052】 In the present invention, however, it is found that as far as the internally circulated mixer in particular is concerned, it is difficult to grasp the particle size breakup trend for the drop diameters when the particle size breakup trend for the drop diameters is reviewed on the experimental basis and the mixer performance is then estimated. For the internally circulated mixer and the externally circulated mixer, however, they are common in that either of those mixers comprises the mixer unit 4 which includes the stator 2 having the plurality of openings (holes) 1 and the stator which is

disposed on the inner side of the stator 3 and spaced by the particular gap δ away from the stator 2, as shown in Fig. 1. When the performance of the internally circulated mixer was then estimated. This was done by using the results obtained by estimating the externally circulated mixer, under the assumption that the internally circulated mixer comprised the same mixer unit as the externally circulated mixer which included the rotor and stator each having the same dimension (size), configuration and structure as the externally circulated mixer as shown in Fig. 4.

【0053】 Then, the respective performances of the three different mixers were compared. The specifications of those mixers which were used for the comparison are given in Table 2.

Table 2 Summary of Mixer

		Mixer A 100 L			Mixer B 500 L			Mixer C 10 kL		
Stator No.		1	2	3	4	5	6	7		
Rotor Diameter	D [mm]	198	198	198	198	198	198	396		
Stator's Opening Diameter	d [mm]	4	4	4	4	1	4	4		
Stator's Opening Ratio	A [-]	0.11	0.20	0.31	0.26	0.12	0.26	0.18		
Number of Openings	n _s [-]	173	316	500	411	3090	414	1020		
Gap Size	δ [mm]	2	2	2	1	1	1	2		

Number of Rotor's Blades n_r: 6

【0054】 The mixer A has the capacity of 100 liters, the mixer B has the capacity of 500 liters, and the mixer C has the capacity of 10 kilo liters. Those mixers are offered by the same manufacturer, and are available on

the commercial market. Note, however, that for the mixer A, the five different mixers (Stator No. 1 through Stator No. 5), each of which has a different gap size δ and a different number of openings (holes) 1 as shown in Table 2, were studied.

【0055】 The experimental conditions and the values for the total energy dissipation rate that were measured under those experimental conditions are presented in Table 3.

Table 3 Experimental Conditions and Calculated Values

Stator No. (Mixer A)	1	2	3	4	5
Configuration Dependent Term	C_h [m ⁵]	3.52×10^{-3}	8.51×10^{-3}	1.43×10^{-3}	1.54×10^{-2}
Ratio of Configuration Dependent Term	$C_h/C_{h_{stat}}$ [-]	0.23	0.55	0.93	1.00
Total Energy Dissipation Rate	ϵ_t [m ² /s ³]	8.16×10^3	19.8×10^3	33.1×10^3	35.6×10^3
N = 1317 [rpm], V = 0.1 [m ³]					

【0056】 Then, the relationship (particle size breakup trend) between the processing (mixing) time under the running conditions shown in Table 3 and

the resulting drop diameters for the mixer A is presented in Fig. 5. Then, the relationship (particle size breakup trend) between the total energy dissipation rate and the resulting drop diameters for the mixer A is shown in Fig. 6.

【0057】 The relationship (particle size breakup trend) between the processing (mixing) time under the running conditions in Table 3 and the resulting drop diameters (actually measured values) for the Stator Nos. 1, 2, 3 in the mixer A is shown in Fig. 7, together with the relationship between the processing time and the estimated values (theoretical values) for the stators Nos. 1 and 3 being based on the stator No. 2 in Table 3.

【0058】 In addition, the relationship (particle size breakup trend) between the processing (mixing) time under the running conditions in Table 3 and the resulting drop diameters (actually measured values) for the Stator Nos. 4 and 5 in the mixer A is shown in Fig. 8, together with the relationship between the processing time and the estimated values (theoretical values) for Stator No. 5 on the basis of Stator No. 4 in Table 3.

【0059】 In Table 3, when the calculated values for C_h/C_{h_std} which were obtained by normalizing C_h/C_{h_std} with the configuration dependent term C_h for Stator No 4 were compared with the results in Figs. 5 through 8, it is found that the estimated values (theoretical values) and the measured values (actually measured values) will exhibit the similar trends. For all stator numbers, it is also found that the particle size breakup effect (particle size breakup performance will become higher as the values for C_h/C_{h_std} are greater.

【0060】 In addition, for Stator Nos. 3 and 4 whose values for C_h/C_{h_std} are substantially equal, it is found that they will exhibit the substantially similar particle size breakup trends. By estimating the mixer's performance from the C_h/C_{h_std} and total energy dissipation rate ε_t , therefore, it is also found that it is possible not only to grasp the qualitative trend but also

account for (estimate) the quantitative trend.

【0061】 When the results obtained by the experiments were organized properly with the processing (mixing) time being given along the X coordinate, on one hand, it was found that the changes in the drop diameter (particle size breakup trend) could not be expressed (estimated) comprehensively.

【0065】 When the results obtained by the experiments were plotted with the total energy dissipation rate ε_t being given along the X coordinate, on the other hand, it was found that the changes in the drop diameter (particle size breakup trend) could be expressed (estimated) comprehensively.

【0063】 As this is described more specifically, it is found that the drop diameter will also follow the decreasing trend, although the running conditions (number of rotations, mixing time), the mixer's configuration (gap, stator's hole diameter, stator's opening area ratio) may be different..

【0064】 Namely, it can be said that the total energy dissipation rate ε_t , which can be obtained by the Equation proposed by the present invention, serves as the index that allows the performance for the mixer of the stator-rotor type to be estimated by taking the differences in the mixer running conditions and configuration into account.

【0065】 For the mixer B that is the medium-scale mixer and the mixer C that is the large-scale mixer, the relationship (particle size breakup trend) between the total energy dissipation rate ε_t and the drop diameters as proposed by the present invention is shown in Fig. 9. It is found that the drop diameter will depend on the values (magnitude) for the total energy dissipation rate ε_t even though the mixer may have a different scale (size) of 200 through 700 liters. It is also found that the drop diameter will exhibit the similar particle size breakup trend even for the mixers of different scales.

【0066】 It may be apparent from the foregoing description that the mixer of

the rotor-stator type can be scaled up by matching the mixer against the values (sizes) of the total energy dissipation rate ε_t proposed by the present invention so that the mixer can conform with those values, and then by taking the differences in the running condition and configuration into account comprehensively.

【0067】 For the mixer which depends on its opening (hole), the present invention allows the mixer's performance estimation and scale-up operations to be performed by taking the particle size breakup operation effect and emulsification effect into account comprehensively. According to the present invention in one of its specific forms, the theory that is applicable to a wider range of mixers can be implemented and developed on the basis of the performance estimation method and scale-up method whose usages hereto have been restrictive.

【0068】 (Configuration and Design of High Performance Mixer)

In the present invention, the configuration of the high performance mixer can be defined by using the values for the total energy dissipation rate ε_t derived from the Equation of the present invention as the index that may be used to estimate the mixer's performance, and then by using the values for the estimation that results from the above as the reference information. Then, the high performance mixer can be designed from this definition. The general structure of the mixer that may thus be designed is shown in Figs. 10 through 12.

【0069】 (Moving Stator)

When the mixer of the rotor-stator type is used to dissolve (mix) the powdery material or liquid material, and to thereby manufacture the emulsified products, it is possible that the mixer will process the powdery material while the air that has been introduced with the powdery material into the mixer remains not to be separated from the introduced powdery material. If this situation occurs, the liquid thus mixed may contain fine air

bubbles produced by mixing the powdery material. For those years, it has been known that if the mixed liquid is emulsified with such fine air bubbles remaining in the liquid, the particle breakup or emulsification performance (effect) will become worse, as compared with the case in which the mixed liquid is emulsified with no air bubbles remaining in the liquid.

【0070】 In order to prevent such fine air bubbles being produced at the initial stage of dissolving the powdery material, therefore, it is desirable that the mixer is equipped with a moving stator mechanism. In particular, when the powdery material that may tend to produce the air bubbles more easily is processed so as to manufacture the emulsified products, it is more desirable that the mixer should be equipped with the moving stator mechanism. By moving the moving stator away from the rotor at the initial stage of dissolving the powdery material, the powdery material can be diffused quickly into the liquid being mixed or prepared. This can be performed without causing any high energy dissipation. It is preferred that the dissolving, particle size breakup and emulsifying steps for the powdery material should be performed by moving the stator closer to the rotor on the production mode.

【0071】 (Multistage Homogenizer (Multistage Emulsifying Mechanism))

As described above, it is confirmed that the performance (effect) with which the particle size breakup or emulsification operation occurs will become better as the values of the total energy dissipation rate ε_t derived from the Equation 1 proposed by the present invention become larger.

【0072】 Then, it is desirable that the mixer includes several mixing portions formed in the gap between the rotor and the stator.

【0073】 For example, the particle size breakup operation may occur on the preliminary mode in the gap between the first-stage rotor and stator, and may then be performed on the production mode in the gap between the second-stage rotor and stator.

【0074】 (Direct Injection (Direct Injection Adding Mechanism))

As it is clear from the mixer performance estimation which serves as the index for the total energy dissipation rate ϵ_t derived from the Equation 1 proposed by the present invention and from the results obtained by verifying the value for the index, it is confirmed that the particle breakup or emulsification performance (effect) will become better as the value for the above index becomes larger.

【0075】Then, by delivering (adding) oils, component in its undissolved state, micro component or other similar components directly into the mixing portions (mixer section), the emulsification or dispersion operation can be performed more effectively. Particularly, when those components are delivered (injected) directly into the first-stage stator (the stator located radially inwardly), the emulsification or dispersion operation can be performed much more effectively on the production mode by the second-stage stator (the stator located radially outwardly) that follows the preliminary stage of the first-stage stator.

【0076】 (Configuration of High Performance Mixer)

As described above, it is confirmed that the particle size breakup or emulsification performance (effect) will become better as the values of the total energy dissipation rate ϵ_t derived from the Equation proposed by the present invention are larger. The opening (hole) formed in the stator should desirably have the round shape instead of the comb-like shape.

【0077】 Note, however, that there is a risk that the powdery material might block the passage through the opening (hole) if the stator has the hole diameter of less than 2 mm. In order to permit both the dissolving process and emulsification process for the powdery material to be performed concurrently, it is desirable that the stator should have the hole diameter of between 2 mm and 4 mm.

【0078】 In addition, the opening provided in the stator should desirably

represent 20 % or more of the total opening area on the peripheral wall of the stator.

【0079】 The shear frequency will become higher as the number of holes provided on the stator (opening area ratio) is increased. The problem is, however, that the strength of the opening on the stator will be affected in this case. In the prior art, the opening area ratio of between 18 % and 36 % is generally employed in most cases, but it is desirable that the opening area ratio should be equal to 30 % or more. It is more desirable that it should be equal to between 40 % and 50%.

【0080】 In addition, the rotor should desirably be equipped with several agitating blades extending radially from the center of rotation. Preferably, the rotor should be equipped with six or more agitating blades, or more preferably eight agitating blades.

EMBODIMENT 1

【 0081 】 The following description provides several examples of the preferred embodiment of the present invention by referring to the accompanying drawings, but it should be understood that the present invention is not restricted to those examples of the embodiment described herein and that various modifications or changes may be made without departing from the spirit and scope of the present invention defined in the appended claims.

【0082】 By referring now to Figs. 10 through 12, the summary of the high performance mixer will be provided below. The homogenization index: H.I. which is derived from the Equation 1 proposed by the present invention can be used as the index for estimating the performance of the high performance mixer, and the configuration of the high performance mixer can be defined by using the results that are obtained by verifying the estimation for the performance. Then, the high performance mixer can be designed by using the definition for the configuration of mixer thus obtained.

【0083】The mixer of the rotor-stator type proposed by the present invention may be characterized by the fact that the mixer comprises the mixer unit 14 that includes the plurality of stators each having a plurality of openings (holes) and the rotor disposed on the inner side of each of the stators and spaced by the particular gap away from each said stator. Other structural elements are the same as those of the prior art mixer of the rotor-stator type that has been described above. Note that the following description is only directed to one example of the mixer unit 14 which has the construction and mechanism that characterize the present invention.

【0084】The mixer unit 14 for the mixer of the rotor-stator type according to the present invention includes the rotor 13 and stators 12, 22, each of which is constructed as shown in Figs. 12 and 13.

【0085】Each of the stators 12, 22 is provided with a plurality of round shape openings 11a, 11b like the stator 2 in the prior art mixer unit 14 shown in Fig. 1.

【0086】Of the stators 12, 22, the stator 22 has the circumferential diameter that is greater than the stator 12, and is arranged concentrically on the mixer unit 14 as shown in Fig. 12 (a).

【0087】The rotor 13, on the inner side of which the stators 12, 22 are disposed and spaced by the particular gap away from the rotor 13, is equipped with a plurality of agitating blades extending toward the center of rotation 17 about which the rotor 13 rotates. In the embodiment shown, for example, the rotor 13 is equipped with eight agitating blades 13a, 13b, 13c, 13d, 13e, 13f, 13g, and 13h.

【0088】Each of the agitating blades 13a though 13h has a longitudinal groove 15 formed in the radial center thereof and located between the center of rotation 17 and the outermost end 16. The longitudinal groove 15 has the same diameter for all agitating blades.

【0089】When the mixer unit 14 is formed as shown in 12 (a) and (b), the

stator 12 may be mounted into the longitudinal groove 15 formed in each of the agitating blade 13a through 13h. For each of the agitating blades 13a through 13h, then, this creates a gap δ between the wall 16a of the radial outermost end 16 thereof and the inner peripheral wall 22a of the stator 22. Additionally, a gap is formed between the outermost peripheral face of each of the agitating blades 13a through 13h and the inner peripheral wall 12a of the stator 12, and a gap is formed between the inner peripheral face 15b of the longitudinal groove 15 of each of the agitating blades 13a through 13h and the outer peripheral wall 12b of the stator 12.

【0090】 As it is apparent for the mixer unit 14 in the mixer of the rotor-stator type that has been described above, the rotor is disposed on the inner side of each of the stators 12, 22 having the different circumferential diameter and spaced by the particular gap away from each of the stators.

【0091】 When the rotor 13 is rotated in the direction of an arrow 20 by causing the rotary shaft 17 to rotate about its center of rotation, the two-stage mixing section including the mixing portion located radially inwardly and the mixing portion located radially outwardly is created. This multistage mixing section permits the mixing process to occur with the high performance.

【0092】 In the embodiment shown, the mixing portion located radially inwardly may be created between the outer peripheral surface 15a of the longitudinal groove 15 on each of the agitating blades 13a through 13h and the inner peripheral wall 12a of the stator 12 and between the inner peripheral wall 15 of the longitudinal groove 15 on each of the agitating blades 13a through 13h and the outer peripheral wall 12b of the stator 12. The mixing portion located radially outwardly may be created between the wall 16a of the radially outward end of each of the agitating blades 13a through 13h and the inner peripheral wall 22a of the stator 22.

【0093】 In the mixer of the present invention, each of the stators 12, 22 can

be moving closer to the rotor 13 and moving farther away from the rotor 13 in the direction in that the rotary shaft 17 of the rotor 13 extends. In the embodiment shown, the rotor 13 can be moving in the direction of arrows 22, 23 in Fig. 12 in which the rotary shaft 17 extends.

【0094】 The mixer of the present invention can take the two states as described earlier, that is, one being the state in which the mixer unit 14 can be created by allowing the stator 12 to be fitted in the longitudinal groove formed in each of the agitating blades 13a through 13h when the rotor 13 is moving in the direction of an arrow 22 in Fig. 12 (b), and the other being the state in which the rotor 13 is moving away from the stators 12, 13 as shown by the imaginary lines in Fig. 12 (b).

【0095】 At the initial stage in which the powdery material is dissolved by the mixer, it can be dispersed quickly into the mixing liquid by moving the rotor 13 away from each of the stators as shown by the arrow 23 in Fig. 12 (b). Note that this can be performed without causing the high energy dissipation to occur.

【0096】 The subsequent steps of the dissolution, particle size breakup and emulsification process can be performed efficiently on the production mode by moving the rotor 13 as shown by the arrow 22 in Fig. 12 (b) so that the two-stage mixing section can be created in the radially inward direction and in the radially outward direction as described above, and then by moving the rotor 13 in the direction shown by the arrow 20 in Fig. 12 (b).

【0097】 In the mixer of the present invention, a nozzle 18 extends radially toward the center along the upper ends of the stators 12, 22 constituting the mixer unit 14 as shown in Fig. 12 (a), allowing the fluid being processed to be delivered directly through the nozzle inlet 19 of the nozzle 18 and into the mixing portion (mixer section) as shown by the arrow 21 in Fig. 12.

【0098】 More specifically, the fluid being processed is delivered as shown by the arrow 21 into the first-stage mixing section located on the inner side,

that is, it is delivered directly through the nozzle inlet 19 and between the outer peripheral surface 15a of each of the longitudinal grooves 13a through 13h and the inner peripheral wall 12a of the stator 12, where the first-stage mixing (preliminary mixing) occurs. Following the first-stage mixing portion, the fluid being processed is then delivered into the second-stage mixing portion located on the outer side, that is, between the radially outside end of the wall 16a of each of the agitating blades 13a through 13h and the inner peripheral wall 22a of the stator 22, where the second-stage mixing occurs on the productive mode.

【0099】 It may be appreciated from the above description that the fluid being processed can be delivered directly into the first-stage and second-stage mixing portions (mixing section), where it can be emulsified and dispersed more effectively.

EMBODIMENT 2

【0100】 The test for the particle size breakup operation took place by using the nutrition prepared foods as offered by MIJI NYUGYO CORP. (MEIBALANCE 1.0 H.P. (Trademark)).

【0101】 The compositions and physical properties of MEIBALANCE 1.0 H.P. are presented in Table 4.

Table 4 Nutrition Conditioned Foods (MEIBALANCE HP 1.0 (Trademark))

Composition (100mL)		
Energy	[kcal]	100
Protein	[g]	5.0
Fat	[g]	2.5
Saccharide	[g]	14.1
Dietary Fiber	[g]	1.2
Ash	[g]	0.7
Water	[g]	84.3
Property Value		
Osmotic Pressure	[mOsm/L]	420
pH (20°C)	[—]	6.7
Viscosity (20°C)	[mPa · s]	10
Specific Gravity (20°C)	[—]	1.078

【0102】 In the current embodiment 2, two types of mixers (one has the capacity of 9 kiloliters and the other has the capacity of 400 liters) were used, and the experiment was conducted by varying the rotating speed of the rotor and the accumulating time. Those two types of mixers are the same as the mixer A which was demonstrated under the embodiment 1 of the present invention.

【0103】 The experimental conditions and the values of the total energy dissipation rate ε_t that were obtained under the experimental conditions are presented in Table 5.

Table 5 Experimental Conditions and Calculated Values (MEIBALACE HP 1.0)

		ε_t
9kL	1050rpm	9.12E+05
	1200rpm	1.53E+06
400L	1500rpm	1.28E+06
	2040rpm	7.29E+06

	Time [min]	d 50 [μm]	Accumulated Time [min]	ε_t [m^2/s^3]
9kL 1050rpm	40	1.013	40	3.65E+07
	5	0.771	45	4.10E+07
	5	0.742	50	4.56E+07
	7	0.691	57	5.20E+07
	15	0.619	72	6.57E+07
9kL 1200rpm	7	13.8	7	1.07E+07
	5	2.37	12	1.84E+07
	8	1.2	20	3.06E+07
	5	0.925	25	3.82E+07
	5	0.807	30	4.59E+07
	5	0.751	35	5.35E+07
	5	0.696	40	6.12E+07
	10	0.642	50	7.65E+07
400L 1500rpm	5.5	5.763	5.5	7.02E+06
	3	2.667	8.5	1.08E+07
	4	1.884	12.5	1.59E+07
	10	1.176	22.5	2.87E+07
400L 2020rpm	5.5	0.68	5.5	4.01E+07
	3	0.617	8.5	6.20E+07
	4	0.593	12.5	9.12E+07
	10	0.527	22.5	1.64E+08

【0104】 The relationship between the values for the total energy dissipation rate ε_t thus obtained and the drop diameters (the resulting particle size breakup trend) is presented in Fig. 3.

【0105】 When the experimental results were organized into the appropriate tabular forms as the values for the total energy dissipation rate ε_t [m^2/s^3] proposed by the present invention being given along the X coordinate, it was found that the change in the drop diameter (the particle size breakup trend) can be expressed (estimated) comprehensively and systematically.

【0106】 The present invention can be applied to the various industrial fields such as the emulsification, dispersion, particle size breakup and other similar processes. Examples of those applications include the

manufacturing fields such as foods, pharmaceutical medicines and chemical products since the present invention provides the excellent effects and functions.

【0107】 The features of the present invention that have been described so far are summarized below:

(1) For the conventional mixers of the rotor-stator type which are available on the commercial market, the performance of those mixers can be estimated by simply operating the mixers using the usual water (water running) instead of using the actual processing liquid. By reviewing the water running operation that is convenient for making such review, the most suitable mixer of the rotor-stator type that can meet each user's requirements for using the mixer can be selected. In this way, the cost of selecting the mixer can be reduced, and the time for the review can also be decreased.

【0108】 (2) By adopting the geometrical size that can maximize the configuration depending term of the total energy dissipation rate ε_t , the performance can be enhanced, designed and manufactured for the inventive and novel mixers of the rotor-stator type, and the performance for the conventional mixers that already exist can be improved as well.

【0109】 (3) For the various mixers of the rotor-stator type that ranges from the small scale to the large scale, the scale-up or scale-down operations efficiency can be achieved by taking the processing (manufacturing) time required for those mixers into consideration.

【0110】 (4) What is required to obtain the particle size effect (drop diameter) that can meet each user's requirements is only to estimate the processing (agitating) time required for that purpose and to operate those mixers with the minimum running (processing) time under such estimation. In this way, the running time required for such mixers can be reduced, and the energy requirements can thus be saved.

【0111】 (5) In accordance with the present invention, the mixers of the rotor-stator type that provide the higher particle size breakup and emulsification effects and higher quality than the typical high performance (high shear type) rotor-stator type mixers that have been offered in the prior art can be manufactured.

【0112】 (6) According to the present invention, the mixers of the rotor-stator type offered by the present invention can provide the higher particle size breakup and emulsification effects and higher quality than the equivalent mixers offered by the prior art, and the processing time can thus be reduced.

【0113】 (7) For the various mixers of the rotor-stator type that range from the small scale to the large scale, the scale-up or scale-down process can be achieved by tanking the processing (manufacturing) time required for those mixer into consideration.

What is claimed is:

1. A mixer of the stator-rotor type comprising a mixer unit that includes a stator having a plurality of openings (holes) and a rotor disposed on the inner side of the stator and spaced by a particular gap away from the stator, the mixer having the construction that is designed by using the Equation 1 given below in such a manner that a particular mixer running time and a particular drop diameter during the particular mixer running time can be obtained when a fluid being processed is subjected to the emulsification, dispersion, particle size breakup or mixing process and by estimating the particular mixer running time and the resulting drop diameter by using the Equation 1:

$$\begin{aligned}\varepsilon_t &= \varepsilon_t f_{s,h} t_m \\ &= \left[A \pi^4 n_r (D + 2\delta) D^3 h \left(\frac{4\ell}{d} + 1 \right) \left(\frac{N_p}{N_{qd} \pi^2} - 1 \right) \right] \left(\frac{N^4}{V} t_m \right) \\ &= C_h \left(\frac{N^4}{V} t_m \right) \dots \text{the Equation 1}\end{aligned}$$

In the Equation 1,

ε_t : Total Energy Dissipation Rate [m²/s³] ε

ε_1 : Local Energy Dissipation Rate in Stator's Opening [m²/s³]

$f_{s,h}$: Shear Frequency

t_m : Mixing Time [s]

A: Stator's Opening Rate [-]

N_r : Number of Rotor Blades [-]

D: Diameter of Rotor [m]

δ : Gap between Rotor and Stator [m]

h: Height of Stator [m]

l: Thickness of Stator [m]

d: Hole Diameter of Stator [m]

N_p : Number of Powers [-]

N_{pd} : Number of Flow Rates [-]

N: Number of Rotation [1/s]

V: Amount of Liquid [m³]

C_h : Configuration Dependent Term for Gap [m⁵]

2. The mixer as defined in Claim 1, wherein the stator and rotor are arranged in such a manner that they can be moving closer to each and moving farther away from each other in the direction in which the rotary shaft of the rotor extends.

3, The mixer as defined in Claim 1 or 2, wherein the mixer includes a plurality of stators each having a different diameter and a rotor disposed on the inner side of each of the stators and spaced by a particular gap away from said each stator.

4. The mixer as defined in any one of Claims 1 through 3, wherein the fluid being processed is introduced into the gap between each of the stators and the rotor disposed on the inner side of each of the stators and spaced by a particular gap away from said each stator.

5. The mixer as defined in any one of Claims 1 through 4, wherein the rotor includes a plurality of agitating blades extending radially from the center of rotation.

6. A method for estimating the performance of a mixer of the rotor-stator type comprising a mixer unit that includes a stator having a plurality of openings (holes) and a rotor disposed on the inner side of the stator and spaced by a particular gap away from the stator, wherein the method includes the steps of: using the Equation 1 given below to obtain the values for the total energy dissipation rate ϵ_t , measuring the components included in the Equation 1, such as the respective size of the rotor and stator and the power requirements and flow rate during the running time of the

mixer, estimating the magnitude (smallness or greatness) of the values for the configuration dependent term in the gap that is specific to the mixer, and estimating the performance of the mixer:

$$\begin{aligned}\varepsilon_t &= \varepsilon_t f_{s,h} t_m \\ &= \left[A \pi^4 n_r (D + 2\delta) D^3 h \left(\frac{4\ell}{d} + 1 \right) \left(\frac{N_p}{N_{pd} \pi^2} - 1 \right) \right] \left(\frac{N^4}{V} t_m \right) \\ &= C_h \left(\frac{N^4}{V} t_m \right) \dots \text{the Equation 1}\end{aligned}$$

In the Equation 1,

ε_t : Total Energy Dissipation Rate [m^2/s^3]

ε_1 : Local Energy Dissipation Rate in Stator's Opening [m^2/s^3]

$f_{s,h}$: Shear Frequency

t_m : Mixing Time [s]

A: Stator's Opening Rate [-]

N_r : Number of Rotor Blades [-]

D: Diameter of Rotor [m]

δ : Gap between Rotor and Stator [m]

h: Height of Stator [m]

l: Thickness of Stator [m]

d: Hole Diameter of Stator [m]

N_p : Number of Powers [-]

N_{pd} : Number of Flow Rates [-]

N: Number of Rotation [1/s]

V: Amount of Liquid [m^3]

C_h : Configuration Dependent Term for Gap [m^5]

7. A method for scaling up or scaling down a mixer of the rotor-stator type comprising a mixer unit that includes a stator having a plurality of openings (holes) and a rotor disposed on the inner side of the stator and

spaced by a particular gap away from the stator, wherein the method includes the steps of: using the Equation 1 given below to obtain the value for the total energy dissipation rate ε_t on the experimental mixer installation and/or the pilot plant mixer installation, matching the thus obtained values for the total energy dissipation rate ε_t against the values for the total energy dissipation rate ε_t on the actual mixer installation so that the former ε_t values can conform with the latter ε_t values, and scaling up or scaling down the mixer accordingly:

$$\begin{aligned}\varepsilon_t &= \varepsilon_t f_{s,h} t_m \\ &= \left[A \pi^4 n_r (D + 2\delta) D^3 h \left(\frac{4\ell}{d} + 1 \right) \left(\frac{N_p}{N_{pd} \pi^2} - 1 \right) \right] \left(\frac{N^4}{V} t_m \right) \\ &= C_h \left(\frac{N^4}{V} t_m \right) \dots \text{the Equation 1}\end{aligned}$$

In the Equation 1,

ε_t : Total Energy Dissipation Rate [m²/s³]

ε_1 : Local Energy Dissipation Rate in Stator's Opening [m²/s³]

$f_{s,h}$: Shear Frequency

t_m : Mixing Time [s]

A: Stator's Opening Rate [-]

N_r : Number of Rotor Blades [-]

D: Diameter of Rotor [m]

δ : Gap between Rotor and Stator [m]

h: Height of Stator [m]

l: Thickness of Stator [m]

d: Hole Diameter of Stator [m]

N_p : Number of Powers [-]

N_{pd} : Number of Flow Rates [-]

N: Number of Rotation [1/s]

V: Amount of Liquid [m³]

C_h : Configuration Dependent Term for Gap [m⁵]

8. Foods, pharmaceutical medicines or chemical products which are manufactured by subjecting a fluid being processed to the emulsification, dispersion, particle size breakup or mixing process that occurs using the mixer of the stator-rotor type which comprises a mixer unit that includes a stator having a plurality of openings (holes) and a rotor disposed on the inner side of the stator and spaced by a particular gap away from the stator, wherein the foods, pharmaceutical medicines or chemical products are manufactured by using the Equation 1 given below, estimating the mixer running time and the drop diameters thus obtained during the mixer running time for the fluid being processed, and subjecting the fluid being processed to the emulsification, dispersion, particle size breakup or mixing process that occurs using the mixer:

$$\begin{aligned}\varepsilon_t &= \varepsilon_t f_{s,h} t_m \\ &= \left[A \pi^4 n_r (D + 2\delta) D^3 h \left(\frac{4\ell}{d} + 1 \right) \left(\frac{N_p}{N_{qd} \pi^2} - 1 \right) \right] \left(\frac{N^4}{V} t_m \right) \\ &= C_h \left(\frac{N^4}{V} t_m \right) \dots \text{the Equation 1}\end{aligned}$$

In the Equation 1,

ε_t : Total Energy Dissipation Rate [m²/s³]

ε_{s_1} : Local Energy Dissipation Rate in Stator's Opening [m²/s³]

f_{s_h}: Shear Frequency

t_m: Mixing Time [s]

A: Stator's Opening Rate [-]

N_r: Number of Rotor Blades [-]

D: Diameter of Rotor [m]

δ : Gap between Rotor and Stator [m]

h: Height of Stator [m]

l: Thickness of Stator [m]

d: Hole Diameter of Stator [m]

N_p : Number of Powers [-]

N_{pd} : Number of Flow Rates [-]

N: Number of Rotation [1/s]

V: Amount of Liquid [m³]

C_h : Configuration Dependent Term for Gap [m⁵]

9. A method for manufacturing foods, pharmaceutical medicines or chemical products by subjecting a fluid being processed to the emulsification, dispersion, drop breakup or mixing process using a mixer of the stator-rotor type which comprises a mixer unit that includes a stator having a plurality of openings (holes) and a rotor disposed inside the stator and spaced by a particular gap away from the stator, wherein the method includes the steps of: using the Equation 1 given below, and estimating the mixer running time and the drop diameter thus obtained during the mixer running time for the fluid being processed:

$$\varepsilon_t = \varepsilon_l f_{s,h} t_m$$

$$\begin{aligned} &= \left[A \pi^4 n_r (D + 2\delta) D^3 h \left(\frac{4\ell}{d} + 1 \right) \left(\frac{N_p}{N_{qd} \pi^2} - 1 \right) \right] \left(\frac{N^4}{V} t_m \right) \\ &= C_h \left(\frac{N^4}{V} t_m \right) \dots \text{the Equation 1} \end{aligned}$$

In the Equation 1,

ε_t : Total Energy Dissipation Rate [m²/s³]

ε_l : Local Energy Dissipation Rate in Stator's Opening [m²/s³]

$f_{s,h}$: Shear Frequency

t_m : Mixing Time [s]

A: Stator's Opening Rate [-]

N_r : Number of Rotor Blades [-]

D : Diameter of Rotor [m]

δ : Gap between Rotor and Stator [m]

h : Height of Stator [m]

l : Thickness of Stator [m]

d : Hole Diameter of Stator [m]

N_p : Number of Powers [-]

N_{pd} : Number of Flow Rates [-]

N : Number of Rotation [1/s]

V : Amount of Liquid [m³]

C_h : Configuration Dependent Term for Gap [m⁵]