

Description

This invention relates generally to the improvements in the fabrication of actuator assemblies for use in disk drive systems of computers. More particularly, this invention relates to improved actuator assembly and method of attaching a coil assembly to an actuator in a manner which economically and reliably assures proper alignment of the coil assembly.

In recent years, micro-computer equipment, particularly so called personal and desk top computers, have become extremely popular for a wide variety of business, educational and other uses. Such computers commonly include a main central processor unit having one or more memory storage disks for storage of data. In many modern computers the storage disk or disks sometimes referred to as "hard disks" are provided as part of a Winchester-type disk drive unit having the storage disks supported in a stack on a rotary spindle within a substantially sealed disk drive housing. The disks are rotatably driven in unison by a small spindle motor, and one or more electro-magnetic heads on a movable actuator are displaced by a moving coil, limited rotation DC motor to traverse surfaces of the rotating disks for purposes of reading and writing data. Such Winchester-type disk drive units are generally preferred in comparison with so called floppy-type disk drives due to their higher memory storage capacities and faster operating speeds.

The actuator used in Winchester-type disk drive units resembles the traversing arm of a common phonograph, and is pivotally mounted about a shaft adjacent an edge of the stacked storage disks. Such actuators typically include an arm assembly designed to carry the electro-magnetic heads which are used to read and write data on the disk track, and a counter balance situated opposite the arm assembly with respect to the pivot shaft. A typical example of this type of arrangement can be found in GB-A-2170359.

A coil is mounted to the counter balance portion of the actuator to interact with a magnet assembly, for purposes of controlling the movement and positioning of the electro-magnetic heads. As a current moves through the coil, it reacts with the magnet assembly to move the entire actuator assembly, comprising the actuator and the coil. Since precision control of the armature is necessary for proper operation of the disk drive unit, accurate positioning of the coil with respect to the actuator is critical to satisfactory performance.

The need for precision components in disk drive units often conflicts with the desire by manufacturers to produce products in an efficient and reliable manner. For example, in order to manufac-

ture actuator assemblies to proper specifications and tolerances, the actuator and the coil are typically fabricated separately and then attached to one another prior to installation into the disk drive unit. However, the precision relationship between the actuator and the coil has previously required an unacceptable degree of manual type work during the assembly process. This has resulted in reduced economy of manufacture, higher costs and less than desirable levels of unit reproducibility.

Accordingly, there has been a need for an improved actuator assembly and method for attaching a coil to an actuator in a manner which economically and reliably assures proper alignment of the two elements prior to installation into a disk drive unit. Such an improved actuator assembly should include components easily attached to one another, and which can be properly aligned quickly, locked into position and secured together utilising a minimal number of steps. Further, an improved actuator assembly and related method of fabrication is needed which minimises the possibility of human introduced error during the assembly process which could result in mis-alignment of the coil with respect to the actuator. Moreover, such an improved actuator assembly must be capable of prolonged, reliable use, and accommodate additional elements incorporated into disk drive units. The present invention fulfills these needs and provides other related advantages.

According to one aspect of the present invention, there is provided a process for assembling a coil assembly to a pivotable actuator for use in a disk drive system, the steps being characterised by comprising: providing an actuator having an arm assembly and a counter balance, the counter balance including a recess situated substantially opposite the arm assembly and extending perpendicular to a pivot axis of the actuator, the counter balance recess including a dovetail receiving portion having at least two precision positioning surfaces; providing a coil assembly having a bobbin which supports winding means, the bobbin including a base, a core which extends outwardly from the base to support the winding means, and a dovetail extending from the base opposite the core, the dovetail being dimensioned to fit within the receiving portion of the counter balance recess and including at least two precision positioning surfaces; sliding the base and dovetail of the bobbin into the counter balance recess; applying an outward force to the coil assembly perpendicular to the actuator pivot axis to cause the positioning surfaces of the dovetail to engage the corresponding positioning surfaces of the counter balance recess; and fixing the coil assembly to the actuator with the positioning surfaces engaged, so that upon removal of the outward force applied to the coil

assembly, the positioning surfaces remain engaged.

The counter balance may include a dowel/pin alignment passageway extending to the counter balance substantially parallel to the actuator pivot axis, to the counter balance recess, the base of the bobbin including a dowel/pin recess which is aligned with the alignment passageway when the coil assembly is correctly positioned within the counter balance recess, and including the step of inserting a dowel through the alignment passageway to position an end thereof in the bobbin dowel/pin recess.

The dowel may be inserted is inserted through the alignment passageway into the dowel/pin recess while outward force is applied to the coil assembly.

Preferably the step of fixing the coil assembly to the actuator includes the injection of an adhesive into the receiving portion between the actuator and the bobbin, the positioning surfaces remaining engaged throughout the injection of the adhesive such that none is injected therebetween.

The process may include the step of removing the dowel from the dowel/pin recess and the alignment passageway after the coil assembly has been fixed to the actuator.

Additionally or alternatively the process may include the step of replacing the dowel with a stop pin inserted through the alignment passageway and into the dowel/pin recess.

The dovetail receiving portion may be dimensioned so as to permit sliding entry of the coil assembly without engaging the corresponding precision positioning surfaces of the counter balance recess and the dovetail, until intended to do so through application of the outward force to the coil assembly.

In one embodiment there are at least two precision positioning surfaces of the dovetail receiving portion include an upper planar surface extending generally the length of the counter balance recess, and a lower planar surface spaced from the upper planar surface, the lower planar surface also extending generally the length of the counter balance recess, the upper and lower planar surfaces being inclined opposite to one another with respect to the actuator pivot axis and the direction of force applied to the coil assembly, the upper and lower planar surfaces tending centrally to position the coil assembly within the counter balance recess so that the core extends perpendicularly away from the pivot axis of the actuator.

The at least two precision positioning surfaces of the dovetail may include two planar surfaces dimensioned to bear against the upper and lower planar surfaces of the dovetail receiving portion when the outward force is applied to the coil as-

sembly.

According to another aspect of the present invention, there is provided an actuator assembly for a disk drive system characterised by comprising: an actuator pivotally mounted to a shaft and including an arm assembly and a counter balance, the counter balance including a recess situated substantially opposite the arm assembly and extending perpendicularly to the longitudinal axis of the shaft, the counter balance recess including a dovetail receiving portion having at least two precision positioning surfaces; and a coil assembly including a bobbin which supports winding means, the bobbin having a base, a core which extends outwardly from the base to support the winding means, and a dovetail extending from the base opposite the core, the dovetail being dimensioned to fit within the dovetail receiving portion of the counter balance recess and including at least two precision positioning surfaces, wherein the coil assembly is placed within the counter balance recess and is positioned by means for engagement between the corresponding precision positioning surfaces of the counter balance recess and the dovetail.

The counter balance may include a dowel/pin alignment passageway extending substantially parallel to the shaft.

The actuator assembly may include a position stop pin at least partially positioned within the dowel/pin alignment passageway.

The base of the bobbin may include a dowel/pin recess which is aligned with the dowel/pin alignment passageway of the counter balance when the coil assembly is correctly positioned within the counter balance recess.

The stop pin may have an end which is positioned within the dowel/pin recess of the bobbin and which extends through the dowel/pin alignment passageway.

In one embodiment the arm assembly includes a plurality of head carriers for supporting head assemblies which inter-act with disks for storage and retrieval of information.

The actuator assembly may include a stop plate for limiting movement of the stop pin, thereby defining the full range of movement permitted to the head assemblies.

Preferably the winding means extends peripherally around the core and through the base of the bobbin.

The actuator assembly, in one embodiment, includes securing means for securing the coil assembly in place within the counter balance recess. This securing means may include an adhesive injected into any spacing between the base and the dovetail of the bobbin, and the counter balance recess, such that none is injected between the

facing precision positioning surfaces of the counter balance recess and the dovetail.

The counter balance recess may be dimensioned to permit assembly of the coil assembly to the actuator without engaging the corresponding precision positioning surfaces of the dovetail and the counter balance recess, until intended to do so through application of a pulling force to the coil assembly away from the shaft.

The actuator preferably includes a latch pin fixed to a latch pin arm which extends outwardly from the shaft between the arm assembly and the counter balance, the latch pin being positioned to be locked by locking means of the disk drive system to immobilise the actuator assembly.

The invention is illustrated, merely by way of example, in the accompanying drawings, in which:-

Figure 1 is a top plan view of a hard disk drive unit with a top cover removed and illustrating the operational relationship between an actuator assembly according to the present invention and a stack of disks;

Figure 2 is an enlarged fragmented elevational section taken generally along the line 2 - 2 of Figure 1, illustrating the positioning of an actuator upon a shaft, and the assembly of a coil assembly to an actuator counter balance to position a bobbin which carries windings within a magnet assembly;

Figure 3 is a top plan view of the coil assembly illustrated in Figures 1 and 2;

Figure 4 is a fragmented top plan view of the actuator counter balance which, together with Figure 3, illustrates the manner in which the coil assembly is slidably received within a counter balance recess as one of the initial steps in an assembly process according to the present invention;

Figure 5 is a fragmented elevational section taken generally along the line 5 - 5 of Figure 4, illustrating another step in the assembly process which includes the application of an outward force to the coil assembly to place precision positioning surfaces of both the counter balance recess and the bobbin in direct contact, and to align a dowel/pin alignment passageway with a dowel/pin recess to permit insertion of a dowel therethrough;

Figure 6 is a view similar to Figure 5, illustrating the injection of an adhesive into the counter balance recess between the actuator assembly and the bobbin after the dowel is in place and while the outward force is applied, such that no adhesive is injected between the precision positioning surfaces;

Figure 7 is another elevational section similar to Figures 5 and 6, illustrating the assembly step of removing the dowel after the injected adhe-

sive has been allowed to set; and

Figure 8 is an elevational section similar to Figures 5 to 7, illustrating the assembly step of replacing the dowel with a position stop pin.

As shown in the drawings for the purposes of illustration, the present invention is concerned with an improved actuator assembly, generally designated by the reference numeral 10. The improved actuator assembly 10 is useful in hard disk drive systems, such as the exemplary system 12 somewhat schematically shown in Figure 1. As will become apparent below, the actuator assembly 10 includes components which are capable of being quickly assembled into a precision relationship. This tends to enhance manufacturing economy of the actuator assembly 10 and reduce human introduced errors or defects.

As illustrated in Figure 1, the actuator assembly 10 is pivotally mounted to a pivot shaft 14 within a casing 16 adjacent to a stack of storage disks 18. These disks 18 are clamped onto a shaft 20 of a spindle motor (not shown) by a spacer bushing 22, with the disks being oriented in the stack more or less uniformly spaced from one another. The bushing 22 is, of course, rotatable with the disks 18 and the shaft 20 of a spindle motor which drives the stack of storage disks in unison.

The actuator assembly 10 comprises, broadly, a actuator 24 having an arm assembly 26 which extends towards the stack of storage disks 18, and a counter balance portion 28 situated opposite the arm assembly with respect to the pivot shaft 14. A coil assembly 30 is fixed to the counter balance 28 of the actuator 24, and inter-acts with the magnet assembly 32 to form a precision moving coil, limited rotation DC motor which angularly displaces the actuator assembly 10 about the pivot shaft 14. The arm assembly 26 includes a plurality of head carriers 34 which each support a head assembly 36. These head assemblies, in turn, support electromagnetic heads 38 which are capable of reading and writing data on the respective storage disks 18 as the heads traverse the disks in response to movement of the actuator assembly 10.

A data cable 40 extends from the electromagnetic heads 38 along the respective head assemblies 36 to the arm assembly 26. The data cable 40 is supported by a flexible cable clamp 42 which accommodates the limited movement of the cable 40 as the actuator assembly 10 is pivoted. The cable 40 extends from the flexible cable clamp 42 within the casing 16 around a solenoid 44 to a rigid cable clamp 46, and from there to an appropriate connection (not shown) for connecting the hard disk drive system 12 to appropriate computer hardware.

The solenoid 44 is mounted to the casing 16 by means of a pair of solenoid mounting bolts 48 in a manner whereby it controls the positioning of a pivotal latch pin locking lever 50. This locking lever 50 moves about a lever pivot 52 which is secured to a mounting nut 54. A solenoid actuating shaft 56, which includes a stop plate 58, is connected to the locking lever 50. This arrangement is provided for "parking" the actuator assembly 10, and thereby immobilising the heads 38 when the disk drive system 12 is being moved.

The actuator 24 includes a latch pin arm 60 which extends outwardly from the pivot shaft 14 between the arm assembly 26 and the counter balance 28. This arm 60 supports a latch pin 62 which can be captured by the locking lever 50. In normal operation of the hard disk drive system 12, the locking lever 50 is positioned by the solenoid 44 so as not to engage the latch pin 62, thereby allowing free movement of the electro-magnetic heads 38 across the storage disks 18. When the solenoid 44 is not energised, however, the locking lever 50 is designed automatically to capture the latch pin 62 upon impact of the latch pin against the locking lever. This is accomplished by providing a contour to the leading edge of the locking lever 50 which causes the desired movement of the lever upon impact to capture the latch pin 62.

The magnet assembly 32 inter-acts with the coil assembly 30 to control the positioning of the electro-magnetic heads 38 with respect to the storage disks 18. As shown in Figure 2, the magnet assembly 32 includes a plurality of upper magnets 64 and lower magnets 66 spaced from one another so that the coil assembly 30 is permitted to extend therebetween. The magnets 64, 66 are positioned upon a unitary magnet housing 68, which is secured to the casing 16 by housing mounting bolts 70.

A stop plate 76 is attached by three stop plate mounting bolts 78 to an upper portion 72 of the magnet housing 68. The stop plate 76 provides an elongated opening 80 through which an actuator position stop pin 82 extends. This stop pin 82 is mounted to the counter balance portion 28 of the actuator 24 to move with movement of the actuator assembly 10. The stop plate 76 inter-acts with the stop pin 82 to limit the total range of movement permitted to the actuator assembly 10 about its pivot shaft 14.

As best shown in Figure 2, the pivot shaft 14 includes a threaded lower base 84 which is positioned within a hole 86 provided in the casing 16. A nut 74 is threaded onto the base 84 to hold the pivot shaft 14 securely on the casing 16. A pair of bearings 88 are positioned about the pivot shaft 14, and are spaced from one another by a bearing spacer 90. The actuator 24 includes a shaft cavity

92 which receives the shaft 14, the bearings 88 and the bearing spacer 90. When assembled as illustrated in Figure 2, the actuator 24 is permitted to pivot freely about the shaft 14 upon the bearings 88.

The counter balance 28 of the actuator 24 includes a recess 94 situated generally opposite the arm assembly 26 and extending perpendicularly to the longitudinal axis of the shaft 14. This recess 94 includes a dovetail receiving portion 96 which defines two precision positioning surfaces 98, 100. These positioning surfaces 98, 100 comprise planar surfaces which are inclined opposite to one another and must be accurately machined to precise tolerances to correctly position the coil assembly 30 within the recess 94.

The counter balance 28 further includes a dowel/pin alignment passageway 102 which extends from an upper surface of the counter balance generally parallel to the pivot shaft 14, to the recess 94. The stop pin 82 is positioned within this alignment passageway 102 when the actuator assembly 10 is ready for use. It will be seen, however, that this alignment passageway 102 can be used to help ensure correct positioning of the coil assembly 30 with respect to the actuator 24.

As illustrated best in Figures 2 and 3, the coil assembly 30 includes a bobbin 104 which supports windings 106. The bobbin 104 includes a base 108, a flange-like core 110 which extends outwardly from the base to support the windings 106, and the dovetail 112 extending from the base opposite the core. The dovetail 112 is dimensioned to fit within the dovetail receiving portion 96 of the counter balance recess 94, and includes two precision positioning surfaces 114, 116 which correspond to and inter-act with the positioning surfaces 98, 100 respectively, provided within the recess 94. The bobbin 104 can be manufactured from, for example, brass, aluminium, glass-filled plastics, or other resinous materials. It is presently preferred, however, that the bobbin 104 be constructed of aluminium.

The base 108 includes a dowel/pin recess 118 in an upper surface thereof. This recess is positioned so that it will be aligned with a lower end of the alignment passageway 102 when the positioning surfaces 114, 116 of the bobbin 104 are caused to bear against the positioning surfaces 98, 100 of the counter balance 28. Further, in the assembled configuration of the actuator assembly 10, the lower end of the stop pin 82 is positioned within the recess 118 (as shown in Figure 2).

The windings 106 comprise bondable copper magnet wire, e.g. 34 AWG single film wire with adhesive coating. It is preferred that the single film is of the polyester type with a rated temperature of 180°C or higher. The adhesive coating is prefer-

ably of the epoxy type with an operating temperature of 155°C. The windings 106 extend peripherally about the core 110 through the base 108 of the bobbin.

The windings 106 comprise approximately 275 turns of the wire about the core 110 and the winding resistance is approximately 13.50 ohms. When the bobbin 104 is wound in a clockwise direction and the coil assembly 30 is placed within the magnet assembly 32, a positive voltage applied to the windings will cause the actuator assembly 10 to move towards the inner diameter of the stack of storage disks 18. Conversely, if a negative voltage is applied, the actuator assembly moves to its retracted position.

After the wire has been wound about the core 110, it is preferably heat bonded. Following this heat bonding of the windings 106, it is preferred that the coil temperature is allowed to come down before handling the coil assembly 30, to avoid deformation of the same.

A process according to the present invention for assembling the coil assembly 30 to the actuator 24 will now be described with reference to Figures 3 to 8. First, the coil assembly 30 is caused to slide into the recess 94 in the direction indicated by the arrow 120 between Figures 3 and 4. Since the recess 94 extends all the way through the counter balance of the actuator 24 it is important to be able quickly and accurately to position the base 108 and the dovetail 112 of the bobbin 104 within the recess 94, and then secure it in place.

To accomplish this, the dowel/pin recess 118 is aligned with the alignment passageway 102, and then an outward force is applied to the coil assembly 30 generally perpendicularly away from the pivot shaft 14. This causes the positioning surfaces 114, 116 of the dovetail 112 to engage the positioning surfaces of the recess 94. It should be noted here that the dovetail receiving portion 96 of the recess 94 is dimensioned to permit the sliding of the coil assembly 30 through the recess without engaging the positioning surfaces 98, 100, 114, 116, until intended to do so through the application of the above mentioned outward force. Such force is indicated by the arrow 122 in Figure 5.

As briefly mentioned earlier, the positioning surfaces 98, 100 within the dovetail receiving portion 96 of the recess 94 are planar and inclined opposite to one another with respect to the pivot axis. These surfaces are further inclined opposite to one another with respect to the direction of force (indicated by the arrow 122) applied to the coil assembly 30. The corresponding positioning surfaces 114, 116 of the bobbin 104 are likewise planar and inclined to bear flatly against the corresponding positioning surfaces 98, 100 of the counter balance recess 94 when the outward force

is applied to the coil assembly 30. The inter-action of these surfaces tends to position centrally the coil assembly 30 within the recess 94 so that the core 110 extends perpendicularly away from the pivot axis of the actuator 24.

When the corresponding positioning surfaces 98, 100, 114, 116 are caused to bear against one another through application of the force indicated by the arrow 122, and the dowel/pin recess 118 is aligned directly below the alignment passageway 102, a dowel 124 is inserted through the alignment passageway to position an end thereof within the dowel/pin recess. This has the effect of securely and accurately positioning the coil assembly 30 in its desired configuration with respect to the actuator 24. When so positioned (see Figure 6), the coil assembly 30 can then be fixed to the actuator 24 so that upon removal of the outward force as indicated by the arrow 122, the coil assembly 30 remains properly positioned within the counter balance recess 94, and the precision positioning surfaces engaged.

More particularly, with the dowel 124 in place and the outward force indicated by the arrow 122 applied, an adhesive 126 is injected into the counter balance recess 94 between the actuator 24 and the adjacent portions of the bobbin 104 and windings 106. Such injection of the adhesive 126 is schematically illustrated by the arrows 128 in Figure 6. The outward force is preferably maintained on the coil assembly 30 throughout the application of the adhesive 126 to ensure that the positioning surface 98, 100, 114, 116 remain engaged, such that no adhesive is permitted to seep between corresponding pairs of the precision positioning surfaces. The adhesive is applied to the gaps between the bobbin 104 and the windings 106, and the adjacent portion of the actuator 24 so that any space in between the actuator and the coil assembly 30 is filled with adhesive.

After the adhesive 126 has been allowed to cure and effectively secures the coil assembly 30 in its proper configuration with respect to the actuator 24, the dowel 124 is removed from the dowel/pin recess 118 and the alignment passageway 102 (see Figure 7). The dowel 124 is then replaced by the stop pin 82 (see Figure 8) to complete the actuator assembly 10.

When so assembled, the actuator assembly 10 is mounted to the shaft 14 which, in turn, is mounted to the casing 16.

From the foregoing it will be appreciated that the actuator assembly 10 can be assembled in an economically and highly reliable manner. The inter-action between the positioning surfaces 98, 100 of the recess 94 and the corresponding positioning surfaces 114, 116 of the bobbin 104 ensures that the coil assembly 30 will be properly positioned

with respect to the actuator 24. The particular construction of parts described above facilitates assembly line type construction and minimises those surfaces having dimensions critical for proper assembly and/or operation of the actuator assembly 10.

Claims

1. A process for assembling a coil assembly (30) to a pivotable actuator (24) for use in a disk drive system, the steps being characterised by comprising: providing an actuator (24) having an arm assembly (26) and a counter balance (28), the counter balance including a recess (94) situated substantially opposite the arm assembly (26) and extending perpendicular to a pivot axis of the actuator (24), the counter balance recess including a dovetail receiving portion (96) having at least two precision positioning surfaces (98, 100); providing a coil assembly (30) having a bobbin (104) which supports winding means (106), the bobbin including a base (108), a core (110) which extends outwardly from the base to support the winding means, and a dovetail (112) extending from the base opposite the core, the dovetail being dimensioned to fit within the receiving portion (96) of the counter balance recess (94) and including at least two precision positioning surfaces (114, 116); sliding the base (108) and dovetail (112) of the bobbin (104) into the counter balance recess (94); applying an outward force to the coil assembly (30) perpendicular to the actuator pivot axis to cause the positioning surfaces (114, 116) of the dovetail (112) to engage the corresponding positioning surfaces (98, 100) of the counter balance recess (94); and fixing the coil assembly (30) to the actuator (24) with the positioning surfaces (98, 100, 114, 116) engaged, so that upon removal of the outward force applied to the coil assembly (30), the positioning surfaces remain engaged.
2. A process as claimed in claim 1 characterised in that the counter balance (28) includes a dowel/pin alignment passageway (102) extending to the counter balance substantially parallel to the actuator pivot axis, to the counter balance recess (94), the base (108) of the bobbin (104) including a dowel/pin recess (118) which is aligned with the alignment passageway (102) when the coil assembly (30) is correctly positioned within the counter balance recess (94), and including the step of inserting a dowel (124) through the alignment passageway (102) to position an end thereof in the bobbin dowel/pin recess (118).
3. A process as claimed in claim 2 characterised in that the dowel (124) is inserted through the alignment passageway (102) into the dowel/pin recess (118) while outward force is applied to the coil assembly (30).
4. A process as claimed in claim 2 or 3 characterised in that the step of fixing the coil assembly (30) to the actuator (24) includes the injection of an adhesive into the receiving portion (96) between the actuator (24) and the bobbin (104), the positioning surfaces (98, 100, 114, 116) remaining engaged throughout the injection of the adhesive such that none is injected therebetween.
5. A process as claimed in any of claims 2 to 4 characterised by including the step of removing the dowel (124) from the dowel/pin recess (118) and the alignment passageway (102) after the coil assembly (30) has been fixed to the actuator (24).
6. A process as claimed in claim 5 characterised by including the step of replacing the dowel (124) with a stop pin (82) inserted through the alignment passageway (102) and into the dowel/pin recess (118).
7. A process as claimed in any preceding claim characterised in that the dovetail receiving portion (96) is dimensioned so as to permit sliding entry of the coil assembly (30) without engaging the corresponding precision positioning surfaces (98, 100) of the counter balance recess (94) and the dovetail (112), until intended to do so through application of the outward force to the coil assembly (30).
8. A process as claimed in claim 7 characterised in that the at least two precision positioning surfaces (98, 100) of the dovetail receiving portion (96) include an upper planar surface extending generally the length of the counter balance recess (94), and a lower planar surface spaced from the upper planar surface, the lower planar surface also extending generally the length of the counter balance recess (94), the upper and lower planar surfaces being inclined opposite to one another with respect to the actuator pivot axis and the direction of force applied to the coil assembly (30), the upper and lower planar surfaces tending centrally to position the coil assembly (30) within the counter balance recess (94) so that the core (110) extends perpendicularly away from



with respect to the actuator 24. The particular construction of parts described above facilitates assembly line type construction and minimises those surfaces having dimensions critical for proper assembly and/or operation of the actuator assembly 10.

Claims

1. A process for assembling a coil assembly (30) to a pivotable actuator (24) for use in a disk drive system, the steps being characterised by comprising: providing an actuator (24) having an arm assembly (26) and a counter balance (28), the counter balance including a recess (94) situated substantially opposite the arm assembly (26) and extending perpendicular to a pivot axis of the actuator (24), the counter balance recess including a dovetail receiving portion (96) having at least two precision positioning surfaces (98, 100); providing a coil assembly (30) having a bobbin (104) which supports winding means (106), the bobbin including a base (108), a core (110) which extends outwardly from the base to support the winding means, and a dovetail (112) extending from the base opposite the core, the dovetail being dimensioned to fit within the receiving portion (96) of the counter balance recess (94) and including at least two precision positioning surfaces (114, 116); sliding the base (108) and dovetail (112) of the bobbin (104) into the counter balance recess (94); applying an outward force to the coil assembly (30) perpendicular to the actuator pivot axis to cause the positioning surfaces (114, 116) of the dovetail (112) to engage the corresponding positioning surfaces (98, 100) of the counter balance recess (94); and fixing the coil assembly (30) to the actuator (24) with the positioning surfaces (98, 100, 114, 116) engaged, so that upon removal of the outward force applied to the coil assembly (30), the positioning surfaces remain engaged.
2. A process as claimed in claim 1 characterised in that the counter balance (28) includes a dowel/pin alignment passageway (102) extending to the counter balance substantially parallel to the actuator pivot axis, to the counter balance recess (94), the base (108) of the bobbin (104) including a dowel/pin recess (118) which is aligned with the alignment passageway (102) when the coil assembly (30) is correctly positioned within the counter balance recess (94), and including the step of inserting a dowel (124) through the alignment passageway (102) to position an end thereof in the bobbin dowel/pin recess (118).
3. A process as claimed in claim 2 characterised in that the dowel (124) is inserted through the alignment passageway (102) into the dowel/pin recess (118) while outward force is applied to the coil assembly (30).
4. A process as claimed in claim 2 or 3 characterised in that the step of fixing the coil assembly (30) to the actuator (24) includes the injection of an adhesive into the receiving portion (96) between the actuator (24) and the bobbin (104), the positioning surfaces (98, 100, 114, 116) remaining engaged throughout the injection of the adhesive such that none is injected therebetween.
5. A process as claimed in any of claims 2 to 4 characterised by including the step of removing the dowel (124) from the dowel/pin recess (118) and the alignment passageway (102) after the coil assembly (30) has been fixed to the actuator (24).
6. A process as claimed in claim 5 characterised by including the step of replacing the dowel (124) with a stop pin (82) inserted through the alignment passageway (102) and into the dowel/pin recess (118).
7. A process as claimed in any preceding claim characterised in that the dovetail receiving portion (96) is dimensioned so as to permit sliding entry of the coil assembly (30) without engaging the corresponding precision positioning surfaces (98, 100) of the counter balance recess (94) and the dovetail (112), until intended to do so through application of the outward force to the coil assembly (30).
8. A process as claimed in claim 7 characterised in that the at least two precision positioning surfaces (98, 100) of the dovetail receiving portion (96) include an upper planar surface extending generally the length of the counter balance recess (94), and a lower planar surface spaced from the upper planar surface, the lower planar surface also extending generally the length of the counter balance recess (94), the upper and lower planar surfaces being inclined opposite to one another with respect to the actuator pivot axis and the direction of force applied to the coil assembly (30), the upper and lower planar surfaces tending centrally to position the coil assembly (30) within the counter balance recess (94) so that the core (110) extends perpendicularly away from



the pivot axis of the actuator (24).

9. A process as claimed in claim 8 characterised in that the at least two precision positioning surfaces (114, 116) of the dovetail (112) include two planar surfaces dimensioned to bear against the upper and lower planar surfaces of the dovetail receiving portion (96) when the outward force is applied to the coil assembly (30).
10. An actuator assembly for a disk drive system characterised by comprising: an actuator (24) pivotally mounted to a shaft (14) and including an arm assembly (26) and a counter balance (28), the counter balance including a recess (94) situated substantially opposite the arm assembly and extending perpendicularly to the longitudinal axis of the shaft, the counter balance recess including a dovetail receiving portion (96) having at least two precision positioning surfaces (98, 100); and a coil assembly (30) including a bobbin (104) which supports winding means (106), the bobbin having a base (108), a core (110) which extends outwardly from the base to support the winding means, and a dovetail (112) extending from the base opposite the core, the dovetail being dimensioned to fit within the dovetail receiving portion (96) of the counter balance recess (94) and including at least two precision positioning surfaces (114, 116), wherein the coil assembly (30) is placed within the counter balance recess (94) and is positioned by means for engagement between the corresponding precision positioning surfaces (98, 100, 114, 116) of the counter balance recess (94) and the dovetail (112).
11. An actuator assembly as claimed in claim 10 characterised in that the counter balance (28) includes a dowel/pin alignment passageway (102) extending substantially parallel to the shaft (14).
12. An actuator assembly as claimed in claim 11 characterised by including a position stop pin (82) at least partially positioned within the dowel/pin alignment passageway (102).
13. An actuator assembly as claimed in claim 11 or 12 characterised in that the base (108) of the bobbin (104) includes a dowel/pin recess (118) which is aligned with the dowel/pin alignment passageway (102) of the counter balance (28) when the coil assembly (30) is correctly positioned within the counter balance recess (94).
14. An actuator assembly as claimed in claim 13 when dependent upon claim 12, characterised in that the stop pin (82) has an end which is positioned within the dowel/pin recess (118) of the bobbin (104) and which extends through the dowel/pin alignment passageway (102).
15. An actuator assembly as claimed in any of claims 12, 13 or 14, characterised in that the arm assembly (26) includes a plurality of head carriers (34) for supporting head assemblies (36) which inter-act with disks (18) for storage and retrieval of information.
16. An actuator assembly as claimed in claim 15 characterised by a stop plate (58) for limiting movement of the stop pin (82), thereby defining the full range of movement permitted to the head assemblies (36).
17. An actuator assembly as claimed in any of claims 10 to 16 characterised in that the winding means (106) extends peripherally around the core (110) and through the base (108) of the bobbin (104).
18. An actuator assembly as claimed in any of claims 10 to 17 characterised by including securing means (126) for securing the coil assembly (30) in place within the counter balance recess (94).
19. An actuator assembly as claimed in claim 18 characterised in that the securing means includes an adhesive (126) injected into any spacing between the base (108) and the dovetail (112) of the bobbin (104), and the counter balance recess (94), such that none is injected between the facing precision positioning surfaces (98, 100, 114, 116) of the counter balance recess and the dovetail.
20. An actuator assembly as claimed in any of claims 10 to 19 characterised in that the counter balance recess (94) is dimensioned to permit assembly of the coil assembly (30) to the actuator (24) without engaging the corresponding precision positioning surfaces (114, 116, 98, 100) of the dovetail (112) and the counter balance recess (94), until intended to do so through application of a pulling force to the coil assembly (30) away from the shaft (14).
21. An actuator assembly as claimed in any of claims 10 to 20 characterised in that the actuator (24) includes a latch pin (62) fixed to a latch pin arm (60) which extends outwardly from the shaft (14) between the arm assembly



(26) and the counter balance (28), the latch pin (62) being positioned to be locked by locking means (50) of the disk drive system to immobilise the actuator assembly (10).

Patentansprüche

1. Verfahren zum Zusammenbau einer Wicklungsbaugruppe (30) mit einem schwenkbaren Stellglied (24) zur Verwendung in einem Plattenlaufwerk-System, gekennzeichnet durch die folgenden Schritte: Schaffung eines Stellgliedes (24) mit einer Armbaugruppe (26) und einem Gegengewicht (28), wobei das Gegengewicht eine Ausnehmung (94) einschließt, die im wesentlichen entgegengesetzt zur Armbaugruppe (26) liegt und sich senkrecht zu einer Schwenkachse des Stellgliedes (24) erstreckt, und wobei die Ausnehmung des Gegengewichtes einen Schwalbenschwanz-Aufnahmeabschnitt (96) mit zumindestens zwei Präzisions-Positionieroberflächen (98,100) einschließt; Schaffung einer Spulenbaugruppe (30) mit einem Spulenkörper (104), der Wicklungsteile (106) trägt, wobei der Spulenkörper einen Basisteil (108), einen Kern (110), der sich von dem Basisteil nach außen erstreckt, um die Wicklungsteile zu tragen, und einen Schwalbenschwanz (112) einschließt, der sich von dem Basisteil aus entgegengesetzt zum Kern erstreckt, wobei der Schwalbenschwanz so bemessen ist, daß er in den Aufnahmeabschnitt (96) der Ausnehmung (94) des Gegengewichtes paßt und zumindestens zwei Präzisions-Positionieroberflächen (114,116) einschließt; Einschieben des Basisteils (108) und des Schwalbenschwanzes (112) des Spulenkörpers (104) in die Ausnehmung (94) des Gegengewichtes; Ausüben einer nach außen gerichteten Kraft auf die Wicklungsbaugruppe (30) senkrecht zur Schwenkachse des Stellgliedes derart, daß die Positionieroberflächen (114,116) des Schwalbenschwanzes (112) gegen die entsprechenden Positionieroberflächen (98,100) der Ausnehmung (94) des Gegengewichtes gedrückt werden; und Befestigen der Wicklungsbaugruppe (30) an dem Stellglied (24), während die Positionieroberflächen (98,100, 114,116) miteinander in Eingriff stehen, so daß bei Aufhebung der auf die Wicklungsbaugruppe (30) ausgeübten, nach außen gerichteten Kraft die Positionieroberflächen miteinander in Eingriff bleiben.
2. Verfahren nach Anspruch 1, dadurch gekennzeichnet, daß das Gegengewicht (28) einen Dübel-/Stift-Ausrichtkanal (102) einschließt, der sich zum Gegengewicht
- im wesentlichen parallel zur Schwenkachse des Stellgliedes zur Ausnehmung (94) des Gegengewichtes erstreckt, daß der Basisteil (108) des Spulenkörpers (104) eine Dübel-/Stift-Ausnehmung (118) einschließt, die mit dem Ausrichtkanal (102) ausgerichtet ist, wenn die Wicklungsbaugruppe (30) richtig in der Ausnehmung (94) des Gegengewichtes positioniert ist, wobei das Verfahren den Schritt des Einsetzens eines Dübels (124) durch den Ausrichtkanal (102) derart einschließt, daß ein Ende des Dübels (124) in der Dübel-/Stift-Ausnehmung (118) des Spulenkörpers angeordnet wird.
3. Verfahren nach Anspruch 2, dadurch gekennzeichnet, daß der Dübel (124) durch den Ausrichtkanal (102) in die Dübel-/Stift-Ausnehmung (118) eingesetzt wird, während eine nach außen gerichtete Kraft auf die Wicklungsbaugruppe (3) ausgeübt wird.
4. Verfahren nach Anspruch 2 oder 3, dadurch gekennzeichnet, daß der Schritt der Befestigung der Wicklungsbaugruppe (30) an dem Stellglied (24) die Injektion eines Klebemittels in den Aufnahmeabschnitt (96) zwischen dem Stellglied (24) und dem Spulenkörper (104) einschließt, wobei die Positionieroberflächen (98,100,114,116) während der gesamten Injektion des Klebemittels miteinander in Eingriff bleiben, so daß kein Klebemittel zwischen diese Flächen injiziert wird.
5. Verfahren nach einem der Ansprüche 2 bis 4, dadurch gekennzeichnet, daß es den Schritt der Entfernung des Dübels (124) aus der Dübel-/Stift-Ausnehmung (118) und aus dem Ausrichtkanal (102) einschließt, nachdem die Wicklungsbaugruppe (30) an dem Stellglied (24) befestigt wurde.
6. Verfahren nach Anspruch 5, dadurch gekennzeichnet, daß es den Schritt des Ersetzens des Dübels (124) durch einen Anschlagstift (82) einschließt, der durch den Ausrichtkanal (102) hindurch und in die Dübel-/Stift-Ausnehmung (118) eingesetzt wird.
7. Verfahren nach einem der vorhergehenden Ansprüche, dadurch gekennzeichnet, daß der Schwalbenschwanz-Aufnahmeabschnitt (96) so bemessen ist, daß er ein gleitendes Einschieben der Wicklungsbaugruppe (30) ohne Eingriff der entsprechenden Präzisions-Positionieroberflächen (98,100) der Ausnehmung (94) des Gegengewichtes und des Schwalbenschwanzes



(112) ermöglicht, bis dieser Eingriff durch die Ausübung der nach außen gerichteten Kraft auf die Wicklungsbaugruppe (30) beabsichtigt ist.

8. Verfahren nach Anspruch 7,
dadurch gekennzeichnet, daß die zumindestens zwei Präzisions-Positionieroberflächen (98,100) des Schwalbenschwanz-Aufnahmeabschnittes (96) eine obere ebene Oberfläche, die sich allgemein über die Länge der Ausnehmung (94) des Gegengewichtes erstreckt, und eine untere ebene Oberfläche in Abstand von der oberen ebenen Oberfläche einschließen, wobei sich die untere ebene Oberfläche ebenfalls allgemein über die Länge der Ausnehmung (94) des Gegengewichtes erstreckt, daß die oberen und unteren ebenen Oberflächen entgegengesetzt zueinander bezüglich der Schwenkachse des Stellgliedes und bezüglich der Richtung der Kraft geneigt sind, die auf die Wicklungsbaugruppe (30) ausgeübt wird, und daß die oberen und unteren ebenen Oberflächen im Sinne einer zentralen Positionierung der Wicklungsbaugruppe (30) in der Ausnehmung (94) des Gegengewichtes wirken, so daß sich der Kern (111) senkrecht von der Schwenkachse des Stellgliedes fort erstreckt.
9. Verfahren nach Anspruch 8,
dadurch gekennzeichnet, daß die zumindestens zwei Präzisions-Positionieroberflächen (114,116) des Schwalbenschwanzes (112) zwei ebene Oberflächen einschließen, die so bemessen sind, daß sie gegen die oberen und unteren ebenen Oberflächen des Schwalbenschwanz-Aufnahmeabschnittes (96) anliegen, wenn die nach außen gerichtete Kraft auf die Wicklungsbaugruppe (30) ausgeübt wird.
10. Stellvorrichtung für ein Plattenlaufwerk-System, dadurch gekennzeichnet, daß sie folgende Teile umfaßt: ein Stellglied (24), das schwenkbar an einer Welle (14) befestigt ist und eine Armbaugruppe (26) und ein Gegengewicht (28) einschließt, wobei das Gegengewicht eine Ausnehmung (94) einschließt, die im wesentlichen entgegengesetzt zur Armbaugruppe angeordnet ist und sich senkrecht zur Längsachse der Welle erstreckt, wobei die Ausnehmung des Gegengewichtes einen Schwalbenschwanz-Aufnahmeabschnitt (96) einschließt, der zumindestens zwei Präzisions-Positionieroberflächen (98,100) aufweist; und eine Wicklungsbaugruppe (30), die einen Spulenkörper (104) einschließt, der Wicklungsteile (106) einschließt, wobei der Spulenkörper einen Basisteil (108), einen Kern (110), der sich von dem Basisteil nach außen hin erstreckt, um die Windungsteile zu halten, und einen Schwalbenschwanz (112) aufweist, der sich von dem Basisteil entgegengesetzt zum Kern erstreckt, wobei der Schwalbenschwanz so bemessen ist, daß er in den Schwalbenschwanz-Aufnahmeabschnitt (96) der Gegengewicht-Ausnehmung (94) paßt und zumindestens zwei Präzisions-Positionieroberflächen (114,116) aufweist, wobei die Wicklungsbaugruppe (30) in die Gegengewicht-Ausnehmung (94) eingesetzt und mit Hilfe von Einrichtungen für einen Eingriff zwischen den entsprechenden Präzisions-Positionieroberflächen (98,100,114,116) der Gegengewicht-Ausnehmung (94) und des Schwalbenschwanzes (112) positioniert wird.
11. Stellvorrichtung nach Anspruch 10, dadurch gekennzeichnet, daß das Gegengewicht (28) einen Dübel-/Stift-Ausrichtkanal (102) einschließt, der sich im wesentlichen parallel zur Welle (14) erstreckt.
12. Stellvorrichtung nach Anspruch 11, dadurch gekennzeichnet, daß sie einen Positionsanschlagstift (82) einschließt, der zumindestens teilweise in dem Dübel-/Stift-Ausrichtkanal (102) angeordnet ist.
13. Stellvorrichtung nach Anspruch 11 oder 12, dadurch gekennzeichnet, daß der Basisteil (108) des Spulenkörpers (104) eine Dübel-/Stift-Ausnehmung (118) einschließt, die mit dem Dübel-/Stift-Ausrichtkanal (102) des Gegengewichtes (28) ausgerichtet ist, wenn die Wicklungsbaugruppe (30) in richtiger Weise in der Gegengewicht-Ausnehmung (94) angeordnet ist.
14. Stellvorrichtung nach Anspruch 13 unter Rückbeziehung auf Anspruch 12, dadurch gekennzeichnet, daß der Anschlagstift (82) ein Ende aufweist, das in der Dübel-/Stift-Ausnehmung (118) des Spulenkörpers (108) angeordnet ist und das sich durch den Dübel-/Stift-Ausrichtkanal (102) erstreckt.
15. Stellvorrichtung nach einem der Ansprüche 12, 13 oder 14, dadurch gekennzeichnet, daß die Armbaugruppe (26) eine Vielzahl von Kopfträgern (34) zur Halterung von Kopfbaugruppen (36) einschließt, die mit Platten (18) zur Speicherung und Wiedergewinnung von Informationen in Wechselwirkung stehen.
16. Stellvorrichtung nach Anspruch 15, gekennzeichnet durch eine Anschlagplatte (58) zur Begrenzung der Bewegung des Anschlags-

stiftes (82), wodurch der volle Bewegungs-
bereich festgelegt wird, den die Kopfbaugruppen
(36) ausführen können.

17. Stellvorrichtung nach einem der Ansprüche 10
bis 16,
dadurch gekennzeichnet, daß sich der Wick-
lungsteil (106) in Umfangsrichtung um den
Kern (110) und durch den Basisteil (108) des
Spulenkörpers (104) erstreckt.
18. Stellvorrichtung nach einem der Ansprüche 10
bis 17,
dadurch gekennzeichnet, daß sie Befestigungs-
mittel (126) zur Befestigung der Wicklungsba-
ugruppe an ihrem Platz in der Gegengewicht-
Ausnehmung (94) einschließt.
19. Stellvorrichtung nach Anspruch 18,
dadurch gekennzeichnet, daß die Befesti-
gungsmittel ein Klebemittel (126) einschließen,
das in jeden Zwischenraum zwischen dem Ba-
sisteil (108) und dem Schwalbenschwanz (112)
des Spulenkörpers (104) und der Gegenge-
wicht-Ausnehmung (94) derart injiziert ist, daß
kein Klebemittel zwischen die aufeinander ge-
richteten Präzisions-Positionieroberflächen
(98,100,114,116) der Gegengewicht-Ausneh-
mung und des Schwalbenschwanzes injiziert
wird.
20. Stellvorrichtung nach einem der Ansprüche 10
bis 19,
dadurch gekennzeichnet, daß die Gegenge-
wicht-Ausnehmung (94) so bemessen ist, daß
sie einen Zusammenbau der Wicklungsba-
ugruppe (30) und des Stellgliedes (24) ermög-
licht, ohne daß sich ein Eingriff der entspre-
chenden Präzisions-Positionieroberflächen
(114,116,98,100) des Schwalbenschwanzes
(112 und der Gegengewicht-Ausnehmung (94)
ergibt, bis dieser Eingriff in beabsichtigter Wei-
se durch Ausüben einer Zugkraft auf die Wick-
lungsbaugruppe (30) von der Welle (14) fort
erfolgt.
21. Stellvorrichtung nach einem der Ansprüche 10
bis 20,
dadurch gekennzeichnet, daß das Stellglied
(24) einen Verriegelungsstift (62) einschließt,
der an einem Verriegelungsstiftarm (60) befe-
stigt ist, der sich von der Welle (14) aus nach
außen zwischen die Armbaugruppe (26) und
das Gegengewicht (28) erstreckt, wobei der
Verriegelungsstift (62) so angeordnet ist, daß
er durch Verriegelungseinrichtungen (50) des
Plattenlaufwerk-Systems verriegelbar ist, um
die Stellvorrichtung (10) unbeweglich zu ma-

chen.



Revendications

1. Procédé d'assemblage d'un ensemble d'enrou-
lement (30) à un actionneur pivotant (24) à
utiliser dans un système d'entraînement de
disque, caractérisé en ce qu'il comprend les
étapes consistant à : préparer un actionneur
(24) comprenant un ensemble de bras (26) et
un contrepoids (28), le contrepoids comportant
un évidement (94) placé sensiblement en face
de l'ensemble de bras (26) et s'étendant per-
pendiculairement à l'axe de pivotement de
l'actionneur (24), l'évidement du contrepoids
présentant une partie de réception (96) en
queue d'aronde avec au moins deux surfaces
de positionnement de précision (98, 100) ;
préparer un ensemble de bobine (30) compre-
nant une bobine (104) qui supporte un moyen
d'enroulement (106), la bobine comportant un
socle (108), un noyau (110) qui s'étend vers
l'extérieur du socle pour supporter le moyen
d'enroulement, et une partie de queue d'aron-
de (112) partant du socle à l'opposé du noyau,
la queue d'aronde étant dimensionnée pour
s'adapter à l'intérieur de la partie de réception
(96) de l'évidement (94) du contrepoids et pré-
sentant au moins deux surfaces de positionne-
ment de précision (114, 116) ; faire glisser
l'embase (108) et la queue d'aronde (112) de
la bobine (104) dans l'évidement (94) du
contrepoids ; appliquer à l'ensemble d'enroule-
ment (30) une force orientée vers l'extérieur,
perpendiculaire à l'axe de pivotement de l'ac-
tionneur pour faire venir en prise les surfaces
de positionnement (114, 116) de la queue
d'aronde (112) avec les surfaces de position-
nement correspondantes (98, 100) de l'évide-
ment (94) du contrepoids ; et fixer l'ensemble
d'enroulement (30) à l'actionneur (24) avec les
surfaces de positionnement (98, 100, 114, 116)
en prise, de manière qu'après suppression de
la force, orientée vers l'extérieur, exercée sur
l'ensemble d'enroulement (30), les surfaces de
positionnement restent en prise.
2. Procédé selon la revendication 1, caractérisé
en ce que le contrepoids (28) comporte un
passage d'alignement de goujon/goupille (102)
s'étendant, en direction du contrepoids en
étant sensiblement parallèle à l'axe de pivote-
ment de l'actionneur, jusqu'à l'évidement (94)
du contrepoids, le socle (108) de la bobine
(104) comprenant un évidement (118) de gou-
jon/goupille aligné avec le passage d'aligne-
ment (102) lorsque l'ensemble d'enroulement
est correctement placé à l'intérieur de l'évide-



ment (94) du contrepoids, et comprenant l'étape consistant à introduire un goujon (124) dans le passage d'alignement (102) pour placer une extrémité de ce goujon dans l'évidement (118) de goujon/goupille de la bobine.

3. Procédé selon la revendication 2, caractérisé en ce que le goujon (124) est introduit à travers le passage d'alignement (102) dans l'évidement (118) de goujon/goupille, alors que la force orientée vers l'extérieur est appliquée à l'ensemble d'enroulement (30).
4. Procédé selon la revendication 2 ou 3, caractérisé en ce que l'étape consistant à fixer l'ensemble d'enroulement (30) à l'actionneur (34) prévoit également d'injecter un adhésif dans la partie de réception (96) entre l'actionneur (24) et la bobine (104), les surfaces de positionnement (98, 100, 114, 116) restant en prise pendant toute l'injection de l'adhésif de façon que ce dernier ne soit pas injecté entre elles.
5. Procédé selon l'une quelconque des revendications 2 à 4, caractérisé en ce qu'il comprend l'étape consistant à retirer le goujon (124) de l'évidement (118) de goujon/goupille et du passage d'alignement (102) lorsque l'ensemble d'enroulement (30) a été fixé sur l'actionneur (24).
6. Procédé selon la revendication 5, caractérisé en ce qu'il comprend une étape consistant à remplacer le goujon (124) par une goupille d'arrêt (82) introduite à travers le passage d'alignement (102) et pénétrant dans l'évidement (118) de goujon/goupille.
7. Procédé selon l'une quelconque des revendications précédentes, caractérisé en ce que la partie de réception (96) de queue d'aronde est dimensionnée pour permettre l'introduction par glissement de l'ensemble d'enroulement (30) sans mettre en prise les surfaces de positionnement de précision correspondantes (98, 100) de l'évidement (94) du contrepoids et de la queue d'aronde (112), jusqu'à la venue en prise volontaire de ces surfaces par application à l'ensemble d'enroulement (30) d'une force orientée vers l'extérieur.
8. Procédé selon la revendication 7, caractérisé en ce qu'au moins deux surfaces de positionnement de précision (98, 100) de la partie de réception (96) de la queue d'aronde comprennent une surface plane supérieure s'étendant généralement sur la longueur de l'évidement (94) du contrepoids et une surface plane infé-

rieure espacée de la surface plane supérieure, la surface plane inférieure s'étendant également généralement sur la longueur de l'évidement (94) du contrepoids, les surfaces plane supérieure et inférieure étant inclinées à l'opposé l'une de l'autre par rapport à l'axe de pivotement de l'actionneur et à la direction de la force appliquée à l'ensemble d'enroulement (30), les surfaces planes supérieure et inférieure tendant à positionner centralement l'ensemble d'enroulement (30) à l'intérieur de l'évidement (94) du contrepoids de manière que le noyau (110) s'étende perpendiculairement à l'axe de pivotement de l'actionneur (24) en s'en éloignant.

9. Procédé selon la revendication 8, caractérisé en ce qu'au moins deux surfaces de positionnement de précision (114, 116) de la queue d'aronde (112) sont constituées par deux surfaces planes dimensionnées pour appuyer contre les surfaces planes supérieure et inférieure de la partie (96) de réception de la queue d'aronde lorsque la force orientée vers l'extérieur est appliquée à l'ensemble d'enroulement (30).
10. Ensemble d'actionneur pour système d'entraînement de disque, caractérisé en ce qu'il comprend : un actionneur (24) monté en pivotement sur un arbre (14) et comprenant un ensemble de bras (26) et un contrepoids (28), le contrepoids comprenant un évidement (94) sensiblement placé face à l'ensemble de bras et orienté perpendiculairement à l'axe longitudinal de l'arbre, l'évidement du contrepoids comportant une partie de réception (96) de queue d'aronde présentant au moins deux surfaces de positionnement de précision (98, 100) ; et un ensemble d'enroulement (30) comprenant une bobine (104) qui supporte un moyen d'enroulement (106), la bobine ayant un socle (108), un noyau (110) qui s'étend à l'extérieur du socle pour supporter le moyen d'enroulement, et une queue d'aronde (112) partant du socle dans la direction opposée au noyau, la queue d'aronde étant dimensionnée pour se loger dans une partie de réception (96) de queue d'aronde de l'évidement (94) du contrepoids et présentant au moins deux surfaces de positionnement de précision (114, 116), dans lequel l'ensemble d'enroulement (30) est placé à l'intérieur de l'évidement (94) du contrepoids et positionné au moyen de la prise entre les surfaces de positionnement de précision correspondantes (98, 100, 114, 116) de l'évidement (94) du contrepoids et de la queue d'aronde (112).



11. Ensemble d'actionneur selon la revendication 10, caractérisé en ce que le contrepoids (28) comprend un passage d'alignement (102) pour goujon/goupille dont l'orientation est sensiblement parallèle à l'arbre (14). 5
12. Ensemble d'actionneur selon la revendication 11, caractérisé en ce qu'il comprend une goupille d'arrêt de position (82) au moins partiellement placée dans le passage d'alignement (102) de goujon/goupille. 10
13. Ensemble d'actionneur selon la revendication 11 ou 12, caractérisé en ce que le socle (108) de la bobine (104) comprend un évidement (118) de goupille/goujon qui est aligné avec le passage d'alignement (102) de goujon/goupille du contrepoids (28) lorsque l'ensemble d'enroulement (30) est correctement placé à l'intérieur de l'évidement (94) du contrepoids. 15 20
14. Ensemble d'actionneur selon la revendication 13, lorsqu'elle est dépendante de la revendication 12, caractérisé en ce que la goupille d'arrêt (82) a une extrémité qui est placée dans l'évidement (118) de goupille/goujon de la bobine (104) et qui traverse le passage d'alignement (102) de goujon/goupille. 25
15. Ensemble d'actionneur selon l'une quelconque des revendications 12, 13 ou 14, caractérisé en ce que l'ensemble de bras (26) comprend une pluralité de supports de têtes (34) pour supporter des ensembles de têtes (36) qui coopèrent avec des disques (18) pour la mise en mémoire et la recherche d'informations. 30 35
16. Ensemble d'actionneur selon la revendication 15, caractérisé par une plaque de butée (58) pour limiter le mouvement de la goupille d'arrêt (82), afin de définir ainsi la course totale du mouvement que les ensembles de têtes (36) sont autorisés à effectuer. 40
17. Ensemble d'actionneur selon l'une quelconque des revendications 10 à 16, caractérisé en ce que le moyen d'enroulement (106) s'étend autour de la périphérie du noyau (110) et traverse le socle (108) de la bobine (104). 45 50
18. Ensemble d'actionneur selon l'une quelconque des revendications 10 à 17, caractérisé en ce qu'il comprend un moyen de fixation (126) pour fixer l'ensemble d'enroulement (30) en place à l'intérieur de l'évidement (94) du contrepoids. 55
19. Ensemble d'actionneur selon la revendication 18, caractérisé en ce que le moyen de fixation comprend un adhésif (126) injecté dans un espace quelconque entre le socle (108) et la queue d'aronde (112) de la bobine (104), et l'évidement (94) du contrepoids, de manière qu'il n'y ait pas d'adhésif injecté entre les surfaces de positionnement de précision (98, 100, 114, 116) en vis-à-vis de l'évidement du contrepoids et de la queue d'aronde.
20. Ensemble d'actionneur selon l'une quelconque des revendications 10 à 19, caractérisé en ce que l'évidement (94) du contrepoids est dimensionné pour permettre d'assembler l'ensemble d'enroulement (30) à l'actionneur (24) sans mettre en prise les surfaces de positionnement de précision correspondantes (114, 116, 98, 100) de la queue d'aronde (112) et de l'évidement (94) du contrepoids, jusqu'à ce que cette mise en prise soit effectuée volontairement en appliquant à l'ensemble d'enroulement (30) une force de tirage l'éloignant de l'arbre (14).
21. Ensemble d'actionneur selon l'une quelconque des revendications 10 à 20, caractérisé en ce que l'actionneur (24) comprend une goupille de verrouillage (62) fixée à un bras de goupille de verrouillage (60) qui part de l'arbre (14) vers l'extérieur entre l'ensemble de bras (26) et le contrepoids (28), la goupille de verrouillage (62) étant placée pour être verrouillée par un moyen de verrouillage (50) du système d'entraînement de disque pour immobiliser l'ensemble d'actionneur (10).