

Clock movement having a balance and a hairspring

The present invention relates to a timepiece movement comprising a balance-and-hairspring type oscillator and an escapement, more particularly such a movement the isochronism of which is improved. By "isochronism" is meant the variations of the rate in dependence upon the oscillation amplitude of the balance and in dependence upon the position of the watch.

Whilst the balance of a conventional balance-and-hairspring oscillator is oscillating, the hairspring develops eccentrically owing to the fact that its centre of gravity is not on the axis of the oscillator and is moving. This eccentric development generates large return forces between the pivots of the shaft of the oscillator and the bearings in which they rotate, which forces furthermore vary in dependence upon the oscillation amplitude. These return forces disturb the oscillations of the balance and generate variations in the rate of the oscillator in dependence upon the oscillation amplitude. To overcome this problem, the present applicant has proposed, in its patent EP 1473604, a balance-and-hairspring oscillator whose outer turn of the hairspring has a stiffened portion arranged to make the development of the hairspring concentric.

However, it is known that the concentricity of the development of a hairspring is not the only factor which has an influence on isochronism. Mounted in a movement, the oscillator is disturbed by the escapement which causes a rate loss, particularly in the case of a Swiss lever escapement. In fact, during the unlocking phase, the oscillator is subjected to a resistant torque before the line of centres, and this causes a loss. During the impulse phase, the oscillator is subjected to a drive torque firstly before the line of centres, which causes a gain, then after the line of centres, which causes a loss. Overall, the escapement thus produces a rate loss and this disturbance caused by the escapement is greater for small oscillation amplitudes of the balance than it is for large oscillation amplitudes of the balance.

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The two above-mentioned phenomena, i.e. eccentric development of the hairspring and

rate loss caused by the escapement, are independent or almost independent of the position of the watch. There is added to these two phenomena the effect of gravity, which produces a rate difference between the horizontal position and the vertical position of the watch, and between its different vertical positions.

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The present invention aims to further improve the isochronism of a timepiece movement and to this end proposes a timepiece movement comprising a balance-and-hairspring oscillator and an escapement cooperating with the oscillator, the outer turn of the hairspring comprising a stiffened portion, characterised in that the stiffened portion is
10 arranged to at least partially compensate for the variation in the rate of the movement in dependence upon the oscillation amplitude of the balance caused by the escapement, and in that the hairspring further comprises at least one of the following features:

- a) a distance between the inner end of the hairspring and the centre of rotation of the hairspring lower than 400 μm , for example equal to about 300 μm ,
- 15 b) a Grossmann curve defined by the inner turn of the hairspring,
- c) a stiffened portion defined by the inner turn of the hairspring.

It has been surprisingly noted that by experimenting with the arrangement of the stiffened portion of the outer turn of the hairspring, for example its position, extent or thickness,
20 and by adding thereto one of the above features a), b) and c), the overall isochronism of the movement, taking into account the disturbance caused by the non-concentricity of the hairspring, the disturbance caused by the escapement and the disturbance caused by gravity, could be clearly improved with respect to the oscillator described in patent EP 1473604.

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Advantageously, the stiffened portion of the outer turn is arranged so that the hairspring produces a rate difference, typically a rate gain, caused by the lack of concentricity of the development of the hairspring of at least 2 s/d, or at least 4 s/d, or even at least 6 s/d, or even at least 8 s/d, at an amplitude of 150° with respect to an amplitude of 300°, at least
30 partially compensating for said rate variation caused by the escapement.

In accordance with a first embodiment, the stiffened portion of the outer turn is closer to the outer end of the hairspring than a theoretical stiffened portion which would make the development of the hairspring substantially perfectly concentric, and the thickness and extent of the stiffened portion may be substantially identical to those of said theoretical stiffened portion.

In accordance with a second embodiment, the stiffened portion of the outer turn is thinner than a theoretical stiffened portion which would make the development of the hairspring substantially perfectly concentric, and the position and extent of the stiffened portion may be substantially identical to those of said theoretical stiffened portion.

In accordance with a third embodiment, the stiffened portion of the outer turn is less extended than a theoretical stiffened portion which would make the development of the hairspring substantially perfectly concentric, and the position and thickness of the stiffened portion may be substantially identical to those of said theoretical stiffened portion.

Other features and advantages of the present invention will become apparent upon reading the following detailed description with reference to the accompanying drawings, in which:

- Figure 1 shows a hairspring having a stiffened outer turn portion in accordance with the prior art, a collet associated with this hairspring being shown schematically by a dashed line;
- Figure 2 shows an isochronism curve obtained by digitally simulating the movements of the centre of rotation of the hairspring shown in figure 1, the oscillator which this hairspring forms part of being considered to be free, i.e., not subjected to the action of an escapement;
- Figure 3 shows overall isochronism measurement results obtained on a real movement having a hairspring as shown in figure 1;
- Figure 4 shows a hairspring of the type as shown in figure 1 but whose stiffened outer turn portion has been moved;

- Figure 5 shows an isochronism curve obtained by digitally simulating the movements of the centre of rotation of the hairspring shown in figure 4, the oscillator which this hairspring forms part of being considered to be free, i.e., not subjected to the action of an escapement;
- 5 - Figure 6 shows overall isochronism measurement results obtained on a real movement having a hairspring as shown in figure 4;
- Figure 7 shows a hairspring of the type as shown in figure 1 but whose thickness of the stiffened outer turn portion has been modified;
- Figure 8 shows an isochronism curve obtained by digitally simulating the
10 movements of the centre of rotation of the hairspring shown in figure 7, the oscillator which this hairspring forms part of being considered to be free, i.e., not subjected to the action of an escapement;
- Figure 9 shows a hairspring of the type as shown in figure 1 but whose angular extent of the stiffened outer turn portion has been modified;
- 15 - Figure 10 shows an isochronism curve obtained by digitally simulating the movements of the centre of rotation of the hairspring shown in figure 9, the oscillator which this hairspring forms part of being considered to be free, i.e., not subjected to the action of an escapement;
- Figure 11 shows isochronism curves corresponding to different horizontal and
20 vertical positions of a hairspring having a stiffened outer turn portion;
- Figure 12 shows the hairspring the isochronism curves of which are shown in figure 11;
- Figure 13 shows a hairspring having a stiffened outer turn portion and a small-diameter collet, forming an embodiment of the invention;
- 25 - Figure 14 shows isochronism curves corresponding to different horizontal and vertical positions of the hairspring shown in figure 13;
- Figure 15 shows a hairspring having a stiffened outer turn portion, a small-diameter collet and an inner Grossmann curve, forming another embodiment of the invention;
- 30 - Figure 16 shows isochronism curves corresponding to different horizontal and vertical positions of the hairspring shown in figure 15;

- Figure 17 shows a hairspring having a stiffened outer turn portion, a small-diameter collet and a stiffened inner turn portion, forming still another embodiment of the invention;
- Figure 18 shows isochronism curves corresponding to different horizontal and vertical positions of the hairspring shown in figure 17;
- Figure 19 schematically shows a movement in which a hairspring as shown in figure 13, 15 or 17 may be incorporated.

Figure 1 shows a flat hairspring of the type described in patent EP 1473604 for a balance- and-hairspring oscillator of a timepiece movement. This hairspring, designated by reference numeral 1, is in the shape of an Archimedean spiral and is fixed by its inner end 2 to a collet 3 mounted on the shaft of the balance and by its outer end 4 to a stud (not shown) mounted on a fixed part of the movement such as the balance-cock. The spring 1-collet 3 assembly can be formed in a single piece, in a crystalline material such as silicon or diamond, by a micro-etching technique. The outer turn 5 of the hairspring 1 has, locally, a portion 6 which has a greater thickness e than the rest of the strip forming the hairspring. This thickness e which can vary along the portion 6 as shown, stiffens the portion 6 and thus makes it substantially inactive as the hairspring develops. The position and extent of the stiffened portion 6 are selected such that the centre of deformation of the hairspring, substantially corresponding to the centre of gravity of the part of the hairspring other than the stiffened portion 6, is substantially coincident with the centre of rotation O of the hairspring and the collet 3, which coincides with the geometric centre of the hairspring. In so doing, the development of the hairspring is concentric or almost concentric. In practice, the stiffened portion 6 ends before the outer end 4 of the hairspring. This outer end 4, more precisely a terminal part 7 of the outer turn 5 including the stiffened portion 6, is radially offset towards the exterior with respect to the course of the Archimedean spiral to ensure that the penultimate turn 8 remains radially free, i.e., it does not contact any element such as the stud, the outer turn or a regulator pin, during operation of the movement. The spacing between the terminal part 7 and the penultimate turn 8 must be greater than that of a conventional hairspring since the penultimate turn 8, owing to the concentric development of the hairspring, moves radially

further towards the stud during expansion of the hairspring. The terminal part 7 is in the form of a circular arc with centre C. The angular extent θ of the stiffened portion 6 and its angular position α (defined for example by the angular position of the centre of the stiffened portion 6 with respect to the angular position of the outer end 4) are defined from this centre C. The thickness e is measured along a radius starting from this centre C. In the illustrated example, the hairspring has 14 turns plus a turn portion extending over 30° , the values θ and α are, respectively, 85.9° and 72° and the maximum of the thickness e is $88.7 \mu\text{m}$. The thickness e_0 of the strip forming the hairspring (measured along a radius starting from the centre of rotation O of the hairspring), except for the stiffened portion 6, is $32.2 \mu\text{m}$. The radius R of the collet 3, i.e. the distance between the inner end 2 of the hairspring and the centre of rotation O of the hairspring, is defined as being the radius of the circle (shown in dashed line) of centre O and passing through the middle (at half the thickness e_0) of the inner end 2 of the hairspring. In the example shown, this radius R is equal to $565 \mu\text{m}$.

Figure 2 is an isochronism diagram obtained with the hairspring shown in figure 1 by digital simulation. More precisely, the diagram of figure 2 is obtained by considering the outer end 4 as being fixed and the shaft on which the collet 3 and the balance are fixed as being free (i.e., not mounted in bearings), by calculating, by finite elements, the movement of the centre of rotation O of the hairspring as the balance oscillates, then by interpolating and integrating the movement curve as a function of the oscillation amplitude. Analytical equations linking the movement of the centre of rotation O of the hairspring to the rate as a function of the oscillation amplitude of the balance are proposed, for example, in the book "Traité de construction horlogère" (Treatise on horological construction) by M. Vermot, P. Bovay, D. Prongué and S. Dordor, published by the Presses polytechniques et universitaires romandes (French-speaking Switzerland polytechnic and university presses), 2011. The x-axis of the diagram of figure 2 shows the oscillation amplitude of the balance in degrees with respect to the equilibrium position and the y-axis shows the rate in seconds per day. This diagram thus shows the rate variation of the hairspring caused by the lack of concentricity of the development of the hairspring. This rate variation applies in the same manner in all positions of the

watch. As can be seen in figure 2, the rate difference between an oscillation amplitude of 150° and an oscillation amplitude of 300° with the hairspring shown in figure 1 is in the order of 1 s/d which is excellent. However, this diagram does not take into account the disturbances caused by the escapement nor the disturbances caused by gravity.

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Measurements were taken on twenty movements of identical design, equipped with the hairspring as shown in figure 1 and with a conventional Swiss lever escapement. For each movement, in each of six different positions (VH: vertical, high; VG: vertical, left; VB: vertical, low; VD: vertical, right; HB: horizontal, low; and HH: horizontal, high), the rate of the movement was measured during the relaxing of its mainspring and the measurements were plotted on a graph. By way of example, the graph obtained for one of these movements is shown in figure 3. The y-axis shows the rate in s/d and the x-axis shows the oscillation amplitude of the balance, which decreases progressively between the completely wound state and the unwound state of the mainspring of the movement owing to the reduction in the force of the mainspring. As can be seen, in all positions of the watch the rate decreases progressively as the oscillation amplitude decreases, and there further exists a rate difference between the different vertical positions. For each position of each movement, a curve was interpolated and the rate difference between the oscillation amplitude of 150° and the oscillation amplitude of 300° was determined. The average of the rate differences for all the positions and all the movements was about 6.7 s/d between said amplitudes. In other words, the rate at 150° was, on average, less than the rate at 300° by about 6.7 s/d. This decrease in the rate, or loss at small amplitudes with respect to large amplitudes, is essentially caused by the escapement.

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The present inventor has noted that the decrease in the rate caused by the escapement could, at least in part, be compensated for by modifying the arrangement of the stiffened portion 6, i.e., for example its position α and/or its extent θ and/or its thickness e with respect to the arrangement of figure 1 which gives the turns of the hairspring a perfect, or almost perfect, concentricity.

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It was discovered in particular that a parameter of the stiffened portion 6 having a

particular influence on the isochronism is its position α . By moving the stiffened portion 6 towards the outer end 4 of the hairspring, a rate gain is produced at small oscillation amplitudes with respect to large oscillation amplitudes of the balance. Thus, a rate difference of about 6.7 s/d, but with the opposite sign compared with the average

measured rate difference mentioned above, can be obtained between the amplitudes of 150° and 300° by moving the stiffened portion 6 to the position $\alpha' = 62^\circ$ and by keeping the other characteristics of the stiffened portion 6 (extent, thickness) constant. The rate variation caused by the escapement can thus be substantially fully compensated for. Figure 4 shows the new hairspring obtained, with its stiffened outer turn portion designated by reference numeral 6'. The movement of the stiffened portion 6 of course modifies the development of the hairspring which is no longer as concentric. However, on the one hand, this modification is small – the hairspring still develops in a manner more concentric than a conventional hairspring (i.e., a hairspring without a stiffened portion) – and, on the other hand, this modification contributes to improving the overall isochronism of the movement, the lack of concentricity that is created serving to compensate for another shortcoming. The diagram of figure 5 shows the isochronism curve I4 of the hairspring shown in figure 4, obtained using the same method as for figure 2. It can be seen that the increase in the rate between the amplitude of 300° and the amplitude of 150° is substantially linear and with an inverse slope compared with the slope of the rate variation caused by the escapement. The isochronism curve I1 of the hairspring shown in figure 1 has also been plotted on this figure 5 for comparison purposes. Figure 6 shows the results of measuring the rate of a movement identical to that on which the measurements in figure 3 were taken, but equipped with the hairspring shown in figure 4 instead of that in figure 1. These results show that the rate variation was significantly reduced by moving the stiffened portion to the position α' , in particular in the range of amplitudes from 180° to 300° where the general shape of the graph is flat.

Another parameter of the stiffened portion 6 having an influence on the isochronism is its thickness e . By decreasing the thickness e , a rate gain is produced at small oscillation amplitudes with respect to large oscillation amplitudes of the balance. Thus, for example, a rate difference of about 6.4 s/d, but with the opposite sign compared with the average

measured rate difference mentioned in relation to figure 3, can be obtained between the amplitudes of 150° and 300° by decreasing the maximum of the thickness e of the stiffened portion 6 (and the remaining thickness proportionally) to the value $e' = 44.2 \mu\text{m}$ and by keeping the other characteristics of the stiffened portion (position, extent)
 5 constant. Figure 7 shows the hairspring obtained, with its stiffened outer turn portion designated by reference numeral 6'', and figure 8 shows the isochronism curve I7 corresponding to such a hairspring.

Still another parameter of the stiffened portion having an influence on the isochronism is
 10 its extent θ . By decreasing the extent θ , a rate gain is produced at small oscillation amplitudes with respect to large oscillation amplitudes of the balance. Thus, for example, a rate difference of about 6.9 s/d, but with the opposite sign compared with the average measured rate difference mentioned in relation to figure 3, can be obtained between the amplitudes of 150° and 300° by decreasing the angular extent θ of the stiffened portion to
 15 the value $\theta' = 43.9^\circ$ and by keeping the other characteristics of the stiffened portion (position, thickness or maximum of the thickness) constant. Figure 9 shows the hairspring obtained, with its stiffened outer turn portion designated by reference numeral 6''', and figure 10 shows the isochronism curve I9 corresponding to such a hairspring.

20 In variations, the principles exposed above could, of course, be combined, i.e., at least two of the parameters α , e and θ could be modified.

Referring again to figure 6, it can be noted that the modification brought to the stiffened portion has the effect of compensating for the rate variation caused by the escapement,
 25 but that it does not have any effect, or has little effect, on the rate difference between the different vertical positions of the watch. This applies irrespective of which parameter(s) α , e , θ is(are) chosen to be modified. Figure 11 shows isochronism curves, designated by J1 to J5, of a hairspring the outer turn of which has a stiffened portion arranged to compensate for the rate variation caused by the escapement, as described above. The
 30 curve J1 shows the isochronism of the hairspring in horizontal position, i.e. the rate variations caused by the non-concentric development of the hairspring, and is obtained in

the same manner as the curves in figures 2, 5, 8 and 10. As can be seen, the stiffened portion of the outer turn of the hairspring is arranged so that the hairspring produces a rate gain of 5.3 s/d at the amplitude of 150° with respect to the amplitude of 300°. The curves J2 to J5 show the isochronism of the hairspring in the four vertical positions VG, 5 VH, VB and VD respectively, and are obtained by taking into account both the non-concentric development of the hairspring and the effect of gravity, in other words by adding the rate variations caused by the non-concentric development of the hairspring and by the gravity. In order to determine the rate variation caused by gravity, in a given vertical position, one can calculate by finite elements the movement of the centre of 10 gravity of the hairspring under the effect of the oscillations of the hairspring (the centre of rotation of the hairspring being fixed), then use analytical equations linking this movement and the position of the hairspring to the rate as a function of amplitude. Such analytical equations are proposed, for example, in the aforementioned book "Traité de construction horlogère" (Treatise on horological construction). The static effect of 15 subsidence of the turns caused by gravity is disregarded in the present invention, as is the effect of the unbalance of the balance, which unbalance may be minimised by known means.

It can be noted in figure 11 that the rate difference between the vertical positions is of 3.2 20 s/d at an oscillation amplitude of the balance of 250°. In order to decrease this rate difference, the present invention proposes to modify the inner portion of the hairspring, namely the distance between the inner end of the hairspring and the centre of rotation of the hairspring and/or the shape of the inner turn.

25 The hairspring corresponding to the isochronism curves J1 to J5 illustrated in figure 11 is shown in figure 12. It comprises 14 turns. The angular extent and the angular position of its stiffened portion 9 (measured in the same manner as for the hairsprings in figures 1, 4, 7 and 9) are of 60° and 75° respectively. The radius R of its collet, i.e. the distance 30 between the inner end of the hairspring and the centre of rotation of the said hairspring, measured in the same manner as in figure 1, is equal to 565 µm. It has been noted that by decreasing the radius R to a value R', the rate difference between the vertical positions is

reduced. The radius R' is advantageously chosen to be lower than $400\text{ }\mu\text{m}$. Figure 14 shows the isochronism curves of a hairspring (illustrated in figure 13) that is similar to the one in figure 12 but has a collet radius R' equal to $300\text{ }\mu\text{m}$ (and a pitch and a turn thickness adapted accordingly). As is apparent from figure 14, the rate difference
 5 between the vertical positions at an amplitude of 250° is of 1.1 s/d , a value that is much lower than the 3.2 s/d of the hairspring of figure 12. However, to obtain a rate gain between the oscillation amplitudes of 150° and of 300° that is similar to the one of the hairspring of figure 12, the stiffened portion, designated by $9'$, must be adapted. Thus, in figure 13, the angular extent and the angular position of the stiffened portion $9'$ are of 50°
 10 and of 75° respectively.

Another manner of decreasing the rate difference between the vertical positions is to shape the inner turn of the hairspring according to a Grossmann curve or to stiffen an inner turn portion. Such a modification of the inner turn may even be combined with the
 15 decrease of the radius R of the collet to still further reduce the rate difference. Thus, figure 15 shows a hairspring the collet radius R' of which is equal to $300\text{ }\mu\text{m}$ and the inner turn 10 of which is shaped as a Grossmann curve. In figure 16, it can be seen that the rate difference between the vertical positions for this hairspring is of only 0.6 s/d at an oscillation amplitude of 250° . In a similar manner, a hairspring having a stiffened portion
 20 11 on the inner turn as shown in figure 17 (the inner stiffened portion 11 having, like the outer stiffened portion $9'''$, a larger thickness than the remainder of the turns) will enable obtaining a rate difference between the vertical positions of 0.6 s/d at an oscillation amplitude of 250° (figure 18). In the case of the hairspring of figure 15, the stiffened portion $9''$ of the outer turn is arranged so that the hairspring produces a rate gain caused
 25 by the lack of concentricity of the development of the hairspring of 4.2 s/d between the amplitudes of 150° and 300° , to compensate for a rate loss caused by the escapement of the same order of magnitude. In the case of the hairspring of figure 17, the stiffened portion $9'''$ of the outer turn is arranged so that the hairspring produces a rate gain caused by the lack of concentricity of the development of the hairspring of 5.4 s/d between the
 30 amplitudes of 150° and 300° , to compensate for a rate loss caused by the escapement of the same order of magnitude.

Although the combination of a Grossmann curve or of a stiffened inner turn portion with a small collet radius R' is particularly advantageous, it will be noted that the Grossmann curve 10 or the stiffened inner turn portion 11 could also be used with a collet of larger radius R . One could also combine a small collet radius R' , a Grossmann curve and a stiffened inner turn portion. In all cases, the stiffened outer turn portion can be arranged according to any one of the principles exposed in relation to figures 4, 7 and 9 or according to a combination of these principles. Moreover, it is evident that the said principles could be applied to a movement the escapement of which would produce a rate gain instead of a rate loss. Thus, for example, to compensate for such a rate gain the stiffened outer turn portion could be moved away from the outer end of the hairspring or the angular extent of the stiffened outer turn portion could be increased.

The hairsprings described above are each intended to form part of an oscillator of a timepiece movement of the type of the movement 12 shown in the form of a block-diagram in figure 19. Besides the oscillator, designated by 16, the movement 12 comprises, in a conventional manner, a drive member 13 such as a barrel, a gear train 14, an escapement 15 and a display 17.

CLAIMS

1. Timepiece movement comprising a balance-and-hairspring oscillator (16) and an escapement (15) cooperating with the oscillator (16), the outer turn of the hairspring comprising a stiffened portion (9'; 9"; 9'''), characterised in that the stiffened portion (9'; 9"; 9''') is arranged to at least partially compensate for the variation in the rate of the movement in dependence upon the oscillation amplitude of the balance caused by the escapement, and in that the hairspring further comprises at least one of the following features:
 - a) a distance (R') between the inner end of the hairspring and the centre of rotation of the hairspring lower than 400 μm ,
 - b) a Grossmann curve (10) defined by the inner turn of the hairspring,
 - c) a stiffened portion (11) defined by the inner turn of the hairspring.
2. Timepiece movement as claimed in claim 1, characterised in that the stiffened portion (9'; 9"; 9''') of the outer turn is arranged so that the hairspring produces a rate difference caused by the lack of concentricity of the development of the hairspring of at least 2 s/d at an amplitude of 150° with respect to an amplitude of 300°, at least partially compensating for said rate variation caused by the escapement.
3. Timepiece movement as claimed in claim 2, characterised in that the stiffened portion (9'; 9"; 9''') of the outer turn is arranged so that the hairspring produces a rate difference caused by the lack of concentricity of the development of the hairspring of at least 4 s/d at an amplitude of 150° with respect to an amplitude of 300°, at least partially compensating for said rate variation caused by the escapement.
4. Timepiece movement as claimed in claim 3, characterised in that the stiffened portion (9'; 9"; 9''') of the outer turn is arranged so that the hairspring produces a rate difference caused by the lack of concentricity of the development of the

hairspring of at least 6 s/d at an amplitude of 150° with respect to an amplitude of 300°, at least partially compensating for said rate variation caused by the escapement.

- 5 5. Timepiece movement as claimed in claim 4, characterised in that the stiffened portion (9'; 9"; 9''') of the outer turn is arranged so that the hairspring produces a rate difference caused by the lack of concentricity of the development of the hairspring of at least 8 s/d at an amplitude of 150° with respect to an amplitude of 300°, at least partially compensating for said rate variation caused by the
10 escapement.
6. Timepiece movement according to any one of claims 1 to 5, characterised in that said rate difference is a rate gain.
- 15 7. Timepiece movement as claimed in any one of claims 1 to 6, characterised in that the stiffened portion (6') of the outer turn is closer to the outer end (4) of the hairspring than a theoretical stiffened portion (6) which would make the development of the hairspring substantially perfectly concentric.
- 20 8. Timepiece movement as claimed in any one of claims 1 to 7, characterised in that the stiffened portion (6'') of the outer turn is thinner than a theoretical stiffened portion (6) which would make the development of the hairspring substantially perfectly concentric.
- 25 9. Timepiece movement as claimed in any one of claims 1 to 8, characterised in that the stiffened portion (6''') of the outer turn is less extended than a theoretical stiffened portion (6) which would make the development of the hairspring substantially perfectly concentric.
- 30 10. Timepiece movement as claimed in claim 7, characterised in that the thickness (e) and extent (θ) of the stiffened portion (6') of the outer turn are substantially

identical to those of said theoretical stiffened portion (6).

11. Timepiece movement as claimed in claim 8, characterised in that the position (α) and extent (θ) of the stiffened portion (6'') of the outer turn are substantially identical to those of said theoretical stiffened portion (6).
12. Timepiece movement as claimed in claim 9, characterised in that the position (α) and thickness (e) of the stiffened portion (6''') of the outer turn are substantially identical to those of said theoretical stiffened portion (6).
13. Timepiece movement according to any one of claims 1 to 12, characterised in that the hairspring comprises the feature a).
14. Timepiece movement according to claim 13, characterised in that said distance (R') is of about 300 μm .
15. Timepiece movement according to claim 13 or 14, characterised in that the hairspring further comprises the feature b).
16. Timepiece movement according to any one of claims 13 to 15, characterised in that the hairspring further comprises the feature c).