

DESCRIPTION

COMPONENT MOUNTING APPARATUS

FIELD OF THE INVENTION

5 The present invention relates to a component mounting apparatus for mounting components on a substrate.

BACKGROUND OF THE INVENTION

10 A component mounting apparatus of the prior art is constructed as described hereinafter. That is, the structure comprises a component carrying unit, a chuck provided on the component carrying unit, a component transfer unit for receiving a component held by the chuck, and a head for receiving the component held by the component transfer unit, and that the head comprises a main head body, a vertically-moving means for the main head body, an insertion nail provided at a lower portion
15 of the main head body, an opening / closing means for the insertion nail, and a turning means for moving the insertion nail between forward and backward directions.

20 In the afore-said apparatus of the prior art, the turning means for forward and backward movement of the insertion nail is necessary in order to retract the insertion nail backward after it mounts a component on a substrate.

 Normally, the turning means retracts the insertion nail by a large margin so that the insertion nail does not hit upon the component when it moves upward after making a retractive movement to the back, even if the component being mounted is considerably large.

25 However, in the case of an apparatus having a structure that insertion nail

retracts backward by a large margin, the insertion nail collides with other components in a retractive movement of the insertion nail, if the other components have been mounted already on the substrate, and if there are only limited spaces among them.

5 The foregoing instance is dealt with by replacing the head with one that makes only a small retractive movement to the back. However, this requires again a positioning alignment of the insertion nail especially with respect to the substrate after replacement of the head, thereby impairing workability.

Therefore, the present invention aims at improving the workability.

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SUMMARY OF THE INVENTION

In order to achieve the foregoing object, an apparatus of the present invention is provided with a cam plate, which comprises the afore-said turning means, mounted detachably on the main head body.

15 In other words, a turning locus of an insertion nail toward the back, or a retracting path of the insertion nail after mounting a component, can be altered easily by mounting another cam plate of a different shape, and thereby a density of components mounted on a substrate can be increased. In addition, the invention improves workability, since it requires only a replacement of the cam plate, but no
20 positioning alignment, etc. of the insertion nail with respect to the substrate after the replacement.

BRIEF DESCRIPTION OF THE DRAWINGS

Fig. 1 is a perspective view depicting an exemplary embodiment of the
25 present invention;

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Fig. 2 is a perspective view depicting a component supply unit portion of the same component mounting apparatus;

Fig. 3 is a perspective view depicting a belt portion of the same component mounting apparatus;

5 Fig. 4 is a front view of the same component supply unit portion;

Fig. 5 is a plan view of the same component supply unit portion;

Fig. 6 is a front view of an essential portion of the same component supply unit;

Fig. 7 is another front view of the same portion of the component supply unit;

10 Fig. 8 is another front view of the same portion of the component supply unit;

Fig. 9 is still another front view of the same portion of the component supply unit;

Fig. 10 is still another front view of the same portion of the component supply unit;

15 Fig. 11 is yet another front view of the same portion of the component supply unit;

Fig. 12 is a perspective view depicting a portion around a second cutting blade of the same component mounting apparatus;

20 Fig. 13 is a perspective view depicting a polarity flipper unit portion of the same component mounting apparatus;

Fig. 14 is a plan view depicting a second chuck release portion of the same component mounting apparatus;

Fig. 15 is a perspective view depicting a hold-position correction unit portion of the same component mounting apparatus;

25 Fig. 16 is a perspective view depicting a portion around a third cutting blade

portion of the same component mounting apparatus;

Fig. 17 is a perspective view depicting a component transfer unit portion of the same component mounting apparatus;

Fig. 18 is a side view depicting an essential portion of the same component transfer unit;

Fig. 19 is another side view of the same essential portion of the component transfer unit;

Fig. 20 is a front view depicting an essential portion around the component transfer unit and a head portion of the same component mounting apparatus; and

Fig. 21 is another front view depicting the same portion around the component transfer unit and the head portion.

Fig. 22 is a plan view depicting an essential portion around the component transfer unit and the head portion;

Fig. 23 is another plan view depicting the same essential portion around the component transfer unit and the head portion;

Fig. 24 is still another plan view depicting the same essential portion around the component transfer unit and the head portion;

Fig. 25 is a perspective view depicting the same component transfer unit;

Fig. 26 is a perspective view depicting an essential portion of the same component transfer unit;

Fig. 27 is another perspective view of the same essential portion of the component transfer unit;

Fig. 28 is a perspective view depicting the same component transfer unit and the head portion;

Fig. 29 is a front view depicting the same component transfer unit;

Fig. 30 is a side view depicting a transfer chuck of the same component transfer unit;

Fig. 31 is another front view of the same component transfer unit;

Fig. 32 is another side view of the same transfer chuck of the component transfer unit;

Fig. 33 is still another front view of the same component transfer unit; and

Fig. 34 is still another side view of the same transfer chuck of the component transfer unit.

Fig. 35 is a perspective view depicting a head;

Fig. 36 is a perspective view depicting a main head body of the same head;

Fig. 37 is a perspective view depicting a rotary body of the same head;

Fig. 38 is a perspective view depicting an insertion nail of the same head;

Fig. 39 is an exploded perspective view of the same insertion nail;

Fig. 40 is a plan view depicting the same insertion nail;

Fig. 41 is another plan view depicting the same insertion nail;

Fig. 42 is a front view depicting an operation of the same head;

Fig. 43 is another front view depicting an operation of the same head;

Fig. 44 is another front view depicting an operation of the same head;

Fig. 45 is another front view depicting an operation of the same head;

Fig. 46 is another front view depicting an operation of the same head;

Fig. 47 is still another front view depicting an operation of the same head;

Fig. 48 is yet another front view depicting an operation of the same head;

Fig. 49 is a sectional view depicting a state of insertion by the head;

Fig. 50 is a sectional view depicting another state of insertion by the same

head;

Fig. 51 is a sectional view depicting another state of insertion by the same head;

Fig. 52 is a front view depicting a state of insertion by the same head;

Fig. 53 is a front view depicting another state of insertion by the same head;

5 Fig. 54 is a front view depicting another state of insertion by the same head;

Fig. 55 is a front view depicting another state of insertion by the same head;

Fig. 56 is a front view depicting still another state of insertion by the same head;

10 Fig. 57 is a front view depicting yet another state of insertion by the same head;

Fig. 58 is a front view depicting the same operation;

Fig. 59 is a plan view depicting an example of comparing insertion nails;

Fig. 60 is a plan view depicting another example of comparing insertion nails;

Fig. 61 is a sectional view depicting one example of a catch pin;

15 Fig. 62 is a sectional view depicting another example of a catch pin;

Fig. 63 is a sectional view depicting still another example of a catch pin; and

Fig. 64 is a sectional view depicting yet another example of a catch pin.

Fig. 65 is a longitudinal sectional view depicting an anvil mechanism of the same component mounting apparatus;

20 Fig. 66 is a perspective view depicting a mounting base of the anvil mechanism;

Fig. 67 is a sectional view depicting the mounting base;

Fig. 68 is a sectional view of the anvil mechanism showing an operation thereof;

25 Fig. 69 is another sectional view of the same anvil mechanism showing an

operation thereof;

Fig. 70 is another sectional view of the same anvil mechanism showing an operation thereof;

Fig. 71 is another sectional view of the same anvil mechanism showing an
5 operation thereof;

Fig. 72 is another sectional view of the same anvil mechanism showing an operation thereof;

Fig. 73 is still another sectional view of the same anvil mechanism showing an operation thereof;

10 Fig. 74 is yet another sectional view of the same anvil mechanism showing an operation thereof;

Fig. 75 is a perspective view depicting an essential portion of the same anvil mechanism;

15 Fig. 76 is a plan view depicting the essential portion of the same anvil mechanism;

Fig. 77 is a front view depicting the essential portion of the same anvil mechanism;

Fig. 78 is another front view of the essential portion of the same anvil mechanism;

20 Fig. 79 is still another front view of the essential portion of the same anvil mechanism;

Fig. 80 is a front view depicting a movable blade of another exemplary embodiment;

Fig. 81 is a cross sectional view depicting the same movable blade;

25 Fig. 82 is a perspective view depicting a catch pin portion of the same anvil



mechanism;

Fig. 83 is an exploded perspective view depicting the same catch pin portion;

Fig. 84 is a sectional view depicting the same catch pin portion;

Fig. 85 is a perspective view depicting an operating rod of the same catch pin

5 portion;

Fig. 86 is a plan view of the same operating rod; and

Fig. 87 is another plan view of the same operating rod.

Fig. 88 is a side view depicting another exemplary embodiment of the present invention;

10 Fig. 89 is another side view of the same mechanism;

Fig. 90 is still another side view of the same mechanism;

Fig. 91 is yet another side view of the same mechanism;

Fig. 92 is a perspective view depicting a first lever of the same mechanism;

Fig. 93 is a perspective view depicting a second lever of the same

15 mechanism; and

Fig. 94 is a block diagram of a controller of the same mechanism.

Fig. 95 is a perspective view depicting a transfer mechanism portion for a substrate in the same component mounting apparatus;

Fig. 96 is a perspective view depicting an end portion of a transfer shaft of
20 the same transfer mechanism;

Fig. 97 is a front view depicting a portion of the same transfer mechanism for the substrate;

Fig. 98 is another front view of the same portion of the transfer mechanism for the substrate;

25 Fig. 99 is still another front view of the same portion of the transfer

mechanism for the substrate;

Fig. 100 is a front view depicting a transferring portion of the same transfer mechanism for the substrate;

Fig. 101 is another front view depicting the transferring portion of the same transfer mechanism for the substrate;

Fig. 102 is another front view depicting the transferring portion of the same transfer mechanism for the substrate;

Fig. 103 is another front view depicting the transferring portion of the same transfer mechanism for the substrate;

Fig. 104 is still another front view depicting the transferring portion of the same transfer mechanism for the substrate; and

Fig. 105 is yet another front view depicting the transferring portion of the same transfer mechanism for the substrate.

15 DESCRIPTION OF THE PREFERRED EMBODIMENTS

An exemplary embodiment of the present invention will be described hereinafter by referring to the accompanying figures.

In Fig. 1, a reference numeral 1 represents a main body, and a backside of which is provided with a plurality of component delivery guides 2. Each of the component delivery guides 2 is supplied with a chain of taped components 4, shown in Fig. 2, from a storage unit 3 located under the component delivery guides 2. Each of distal ends of the component delivery guides 2 is linked with a component supply unit 5.

As an exemplified structure of a component delivery unit, an annular belt 6 made of rubber or synthetic resin is run through three pulleys 7, 8 and 9 in a

triangular shape in the main body 1. The belt 6 is embedded internally with dozens of annular metal wires, so as to prevent it from stretching to the utmost.

Also, the belt 6 is provided with ditches and ridges on both of front and back surfaces, so that the ditches and ridges on the back surface engage with ditches and
5 ridges on a peripheral surface of the pulleys 7 through 9.

A motor 10, which normally runs intermittently, drives the pulley 7, and thereby the belt 6 also moves intermittently.

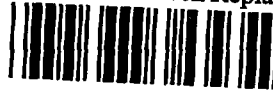
A plurality of chuck retainers 11 are attached to the belt 6 at regular intervals, as shown in Fig. 3.

10 In this instance, the chuck retainers 11 are attached securely in their respective positions on the belt 6 by engaging the ditches and ridges on the surface of the chuck retainers 11 with the ditches and ridges on the front surface of the belt 6. In addition, an upper end and a lower end of each of the chuck retainers 11 are extended toward the backside surface of the belt 6, so that portions of these
15 extended ends also engage with an upper part and a lower part of the ditches and ridges on the backside surface of the belt 6.

Further, guide portions 11a and 11b are integrally formed with the upper and the lower extended ends of the chuck retainers 11, and an upper end and a lower end of a board-shaped guide rail 12, which is positioned at the backside surface of the
20 belt 6 located between the pulleys 7 and 8, between the pulleys 8 and 9 and between the pulleys 9 and 7, engage with the guide portions 11a and 11b.

The above structure allows the belt 6 to move along without trembling. Also, a lower end of the individual chuck retainer 11 is provided with three sets of chuck 13, as shown in Fig. 2 and Fig. 3.

25 These chucks 13 catch individual component pieces cut off by a first cutting



blade 14 from the chain of taped components 4, and transfer them sequentially toward a second cutting blade 15, a polarity flipper unit 16 for the components, a hold-position correction unit 17 for lead terminals, a third cutting blade 18 for cutting the lead terminals, and a component transfer unit 19, along with a movement
5 of the belt 6.

Each individual component piece held by the component transfer unit 19 is handed over to a head 20. The head 20 transfers the component piece along a rail 21, and mounts it on a substrate, which is aligned in position on an X - Y table 22. The substrate will be described later. An anvil mechanism 23 then cuts off extra
10 lengths of the lead terminals, and bends the lead terminals inwards.

The chain of taped components 4 consists of components, each of which comprises an element 24 and at least two lead terminals 25 connected to the element 24, and a taping member 26 on which the components are taped at regular intervals as shown in Fig. 2. The component supply unit 5 supplies the chain of taped
15 components 4 to the chucks in such a manner that the components are positioned with the element 24 side up and the lead terminals 25 side down, and also they are so aligned that a phantom line extended through two lead terminals 25 of a component held by one of the chucks is approximately in line with another phantom line extended through two lead terminals 25 of another component to be held
20 subsequently by another chuck, as shown in Fig. 2.

In other words, the exemplary embodiment of the present invention reduces a space occupied by the component supply unit 5, as shown in Fig. 1, since the component supply unit 5 supplies the chain of taped components 4 linearly toward the chucks 13 on the belt 6 used as the component delivery unit.

25 Although an apparatus of the present exemplary embodiment is provided with

a plurality of component supply units 5 of this kind side by side, as shown in Fig. 1, the apparatus can be small in size of its entirety, when the plurality of component supply units 5 are disposed side by side, since each of the component supply units 5 has a slim shape because of the structure supplying the chain of taped components 4 linearly to the chucks 13. In other words, the apparatus can improve an efficiency of supplying components by arranging a larger number of the component supply units 5 in a limited space, as shown in Fig. 1.

Fig. 4 and Fig. 5 respectively depict portions of the component supply unit 5. The component supply unit 5 comprises: a guide channel 27 for guiding the chain of taped components 4; a feeder 29 linked to perforations 28 in the taping member 26 located in the guide channel 27 for making the taping member 26 to move intermittently in ratchet motion; a first chuck release 30 interlocked with the feeder 29 for freeing the chuck 13; a first cutting blade 14 for cutting the taping member 26 while holding the lead terminals 25 of the chain of taped components 4 with the chuck 13; a cam 31 for driving the feeder 29, the first chuck release 30, and the first cutting blade 14; and a cylinder 32 used as a driving unit for driving the cam 31, and so on.

An element detector 33 fronts on the guide channel 27, as shown in Fig. 5, and it detects whether or not an element 24 is present at a predetermined position on the chain of taped components 4. An output of the element detector 33 controls a magnitude of movement of the cam 31 by means of the cylinder 32, as will be described later.

The cam 31 has two slide openings 34, which connect to their respective pins 35, and it is moved reciprocally by the cylinder 32.

The first chuck release 30 is connected to a chuck opening cam surface 36 of

the cam 31 via a transmission mechanism 37.

The transmission mechanism 37 comprises: a lever 39 having a pin 38 in contact with the chuck opening cam surface 36; a lever 41 supporting the lever 39 with an axle 40; an axle 42 supporting axially the lever 41, and so on.

5 Referring back to the cam 31 for more details, it is provided with a cutting blade open/close cam surface 43 and a cutting blade shift cam surface 44. The cutting blade open/close cam surface 43 is linked with a pin 47 on a lever 46, which is supported axially by an axle 45, thereby enabling the cam 31 to open and close the first cutting blade 14 by means of the lever 46.

10 Also, the cutting blade shift cam surface 44 is linked with a pin 50 on a lever 49, which is supported axially by an axle 48, and a free end of the lever 49 is connected to the afore-cited feeder 29. Further, the free end is also connected to the first cutting blade 14 via a lever 51.

Referring now to Fig. 6 through Fig.11, described hereinafter is a series of
15 operations relating to transfer of the chain of taped components 4, cutting it into individual component pieces, and feeding to the chuck 13.

Fig. 6 shows the cam 31 in its initial position, where it is shifted to a leftmost position by the cylinder 32.

Distinctiveness of this position is that the first cutting blade 14 is set at a side
20 of the component supply unit 5, as shown in Fig. 6.

That is, since the chuck 13 and the first cutting blade 14 are in their positions close to each other, yet the first cutting blade 14 is shifted toward the component supply unit 5, so that the transfer of a component to the second cutting blade 15 by the chuck 13 will not be hindered as the transfer takes
25 place, as shown by Fig. 6.



In the state shown by Fig. 6, the first cutting blade 14 is in its open position, and the chain of taped components 4 advances between the opened first cutting blade 14 in the next step of Fig. 7.

When the cam 31 shifts toward right, as shown in Fig. 7, the lever 41 turns
5 counterclockwise round the axle 42 as an axis, because the pin 38 slides up on the chuck opening cam surface 36 of the cam 31 at a beginning of the rightward movement of the cam 31. This causes the first chuck release 30 to shift toward the chuck 13, as shown in Fig. 5.

The chuck 13 comprises a fixed nail 52, a movable nail 53, which opens and
10 closes with respect to the fixed nail 52, and an elastic body 54 attached to an inside face of the movable nail 53. The movable nail 53 is provided with a hook 55 at a side closer to the component supply unit 5 for making a linkage to the chuck release 30.

Therefore, when the chuck release 30 makes a small stroke of movement, it
15 can open the chuck 13 first, as shown in Fig. 5, because of the hook 55 provided on the movable nail 53 of the chuck 13 at a side closer to the component supply unit 5 for making a linkage to the chuck release 30.

The elastic body 54 is attached to the movable nail 53 with a pin penetrating
through the elastic body 54. Use of the pin for attaching the elastic body 54 to the
20 movable nail 53 by penetrating therethrough can prevent the elastic body 54 from coming off the movable nail 53, even if a stress of deformation is applied repeatedly to the elastic body 54 due to opening and closing of the chuck 13.

The chuck 13 fixes a position of an individual component piece separated
from the chain of taped components 4 with the fixed nail 52, and holds the
25 component piece securely with a moderate elasticity of the elastic body 54 on the

inside face of the movable nail 53.

When the chuck 13 is in its open position, the pin 50 slides up on the cutting blade shift cam surface 44 to drive the feeder 29 via the lever 49, and thereby feeding the chain of taped components 4 into the open chuck 13, as shown in Fig. 7.

5 The chuck 13 is in a standby state for receiving the chain of taped components 4 at the moment, so as not to interfere with the feeding, even if the first cutting blade 14 is shifted toward the chuck 13 via the lever 51 and a holder 56 of the first cutting blade 14.

10 The holder 56 is axially supported by an axle 57, and it shifts the first cutting blade 14 toward the chuck 13, when a free end of it is moved toward left by the lever 51.

15 Fig. 8 shows a state wherein the cam 31 has been shifted further toward right side by the cylinder 32, and the lever 49 has made the feeder 29 to complete the feeding movement at this moment. The pin 38 has nearly slid down on the chuck opening cam surface 36 at this moment. As the lever 41 turns clockwise in this state, the first chuck release 30 shifts toward the right side, and the chuck 13 is also near to its close position.

This means that the lead terminals 25 of the element 24 are being held by the chuck 13 at a leading end of the chain of taped components 4, as shown in Fig. 4.

20 Fig. 9 shows a state wherein the cam 31 has been shifted even further toward the right side by the cylinder 32, and the lead terminals 25 are completely held by the chuck 13.

25 Fig. 10 shows a state wherein the cam 31 has been shifted toward the rightmost end by the cylinder 32. In this state, a left end of the lever 46 supported axially by the axle 45 is moved downward, because the pin 47 is in a position of

completely sliding up on the cutting blade open/close cam surface 43, and thereby the taping member 26 is cut off to separate a component piece, which is then held by the chuck 13.

Finally, the cylinder 32 causes the cam 31 to return to its original position of Fig. 4 by passing through a position shown in Fig. 11.

When the cam 31 returns toward the left side in the position shown in Fig. 11, the pin 38 slides up on the chuck opening cam surface 36. However, if the chuck 13 is opened during this return movement, the component piece held by the chuck 13 slips off. It is therefore important to provide a countermeasure against it.

In this exemplary embodiment, the transmission mechanism 37 is so constituted that the lever 39 turns only clockwise without depressing the lever 41 to the left side, when the pin 38 on the lever 39 slides up on the chuck opening cam surface 36 during the return movement. As a result, the first chuck release 30 does not open the chuck 13, so as not to cause the component piece to fall off.

Also, the first cutting blade 14 does not interfere with a transfer movement of the chuck 13 holding the component piece, because it returns toward the component supply unit 5 after it opens. The feeder 29 also returns to its original position, as needless to mention, in order to become ready for a subsequent transfer movement.

As has been described, the component supply unit 5 of the present exemplary embodiment is able to supply the component to the chuck 13 smoothly, since it is provided with the first chuck release 30 for opening the chuck 13, which enables the component supply unit 5 to make a ganged operation of supplying the component and opening the chuck 13.

Specifically, the foregoing structure is such that the first chuck release 30 is actuated by the chuck open cam surface 36 of the cam 31, which also actuates the

feeder 29 for the chain of taped components 4 provided on the component supply unit 5. Because only this single cam 31 actuates both of the feeder 29 for the chain of taped components 4 on the component supply unit 5 and the first chuck release 30, the structure realizes the ganged operation of advancing the chain of taped components 4 and holding it by the chuck 13, and thereby making a smooth supply of the components to the chuck 13.

The component supply unit 5 is also provided with the first cutting blade 14 between the feeder 29 and the chuck 13. The first cutting blade 14 has such a structure that it is actuated by means of the cutting blade open/close cam surface 43 and the cutting blade shift cam surface 44 of the cam 31. Since the cam 31 also actuates the first cutting blade 14, in addition to the feeder 29 and the first chuck release 30, this structure realizes a ganged operation of the feeder 29 to advance the chain of taped components 4, the first cutting blade 14 to cut the chain into separate component pieces and the chuck 13 to hold the component piece, and thereby resulting in the smooth supply of the components to the chuck 13.

The component supply unit 5 is provided with the element detector 33 for detecting the presence or absence of an element 24 on the chain of taped components 4, as shown in Fig. 4 and Fig. 5. The element detector 33 detects an element 24, or, to be concrete, an absence of element 24 if it does not detect lead terminals 25 under the ordinary operating condition. If the element detector 33 detects an absence of element 24, it actuates the cylinder 32 to return the cam 31 to the position shown in Fig. 4, after shifting it for only a first amount of movement (to the position of Fig. 8).

In other words, the element detector 33 is used to expedite delivery to the chuck 13, of a portion carrying an element 24 in the chain of taped components 4,

by shifting the cam 31 for a second amount of movement (to a position of Fig. 10), without activating the first cutting blade 14 to make a cutting operation of the chain of taped components 4, if no element 24 is detected.

In this case, however, there is a long piece of the taping member 26 left uncut
5 under the chuck 13 and inside of the belt 6 with the component piece held by the chuck 13, and this piece of the taping member 26 is therefore cut into a predetermined length afterwards by the second cutting blade 15.

After the component piece is transferred to the chuck 13 in the above manner, the motor 10 is started and the component piece is carried by the belt 6.

10 There is an extra-length detector 58 positioned at a downstream side of a component supply spot to the chuck 13 on the belt 6 from the component supply unit 5 for detecting the presence or absence of a long piece of the taping member 26, as shown in Fig. 12. This extra-length detector 58 activates the second cutting blade 15 to cut the long piece of taping member 26 into the predetermined length.
15 That is, if the chuck 13 on the belt 6 carries the taping member 26 in a length longer than a predetermined length, the extra-length detector 58 detects it and activates the second cutting blade 15 to cut it into the proper length.

The extra-length detector 58 comprises an optical system having a light-emitting element and a light-receiving element. It determines that a long piece of
20 the taping member 26 is present, if light is cut off by an extra long portion of the taping member 26.

When the extra-length detector 58 detects a long piece of the taping member 26, an elevator 59 lifts the second cutting blade 15 upward from a lower position, and the blade 15 cuts an extra length of the taping member 26 at the lifted position.
25 The second cutting blade 15 moves down to its lower position by the elevator 59

after it completes the cutting operation. Because of this structure, wherein the second cutting blade 15 moves up only when it cuts a long piece of the taping member 26, and moves down after the cutting, the second cutting blade 15 does not interfere with the belt 6 to carry the components.

5 Fig. 13 shows a polarity flipper unit 16 provided at a downstream side of the second cutting blade 15. The polarity flipper unit 16 is freely movable toward and away from the belt 6. That is, the polarity flipper unit 16 is moved toward the belt 6 at a side of the chuck 13, if a need arises that a component be reversed of its polarity prior to mounting it on a substrate by the head 20. The polarity flipper
10 unit 16 reverses polarity of a component by receiving the component from the chuck 13, rotating it, and returning it again to the chuck 13 for the successive carrying movement.

Furthermore, since the polarity flipper unit 16 stays away from the chuck 13 when it is not required to reverse polarity, it does not interfere with carriage of the
15 components.

Incidentally, the chuck 13 needs to be opened for a moment in order to give and take the component piece to and from the polarity flipper unit 16 shown in Fig. 13. A structure to carry out this function will be described by referring to Fig. 14.

As shown in Fig. 14, one end of the chuck 13 opposite to the hook 55 of the
20 movable nail 53 is extended to the backside of the belt 6, and this extended end is linked to a second chuck release 63, so that the chuck 13 is opened by the second chuck release 63.

Fig. 15 shows a hold position correction unit 17 provided at a downstream side of the polarity flipper unit 16. The hold position correction unit 17 carries out
25 correction of a position of the lead terminals 25 held by the chuck 13, after the

polarity flipper unit 16 of Fig. 13 has reversed polarity of the component needing the reversion.

The hold position correction unit 17 comprises: a positioning base 60 for setting thereon a bottom edge of the taping member 26; a holder 61 for holding the taping member 26 by two sides orthogonal to a longitudinal direction thereof; and push bodies 62 for shifting the component by pushing the lead terminals 25 of the component toward the longitudinal direction of the taping member 26.

When making correction, the taping member 26 is set first on the positioning base 60. Then the taping member 26 is centralized by the push bodies 62 at both sides for the correction of a hold position of the lead terminals 25, after opening the chuck 13 with another second chuck release 63 having the same structure as the one shown in Fig. 14 provided in position relative to the chuck 13. The chuck 13 is then closed to hold the lead terminals 25 properly.

Fig. 16 shows a portion around the third cutting blade 18 provided at a downstream side of the hold position correction unit 17 for cutting the lead terminals 25. The third cutting blade 18 is freely movable toward and away from the chuck 13 on the belt 6. That is, the third cutting blade 18 can cut the lead terminals 25 properly, since it cuts the lead terminals 25 only after completion of the correction of hold position with the chuck 13.

The third cutting blade 18 comprises a pair of openable blades 64, of which lower surfaces at their distal portion are tapered. Lengths of the lead terminals 25 are uniformized after the cutting, since they are cut while an upper edge of the taping member 26 of the component piece held by the chuck 13 is kept in contact with the tapered surfaces.

Fig. 17 shows a component transfer unit 19 for transferring a component held

by the chuck 13 to the head 20. The component transfer unit 19 has a structure that includes two grip nails 65 and 66 for holding lower portions of the lead terminals 25, as shown in Fig. 18, and a support nail 67 for supporting upper portions of the lead terminals 25.

5 The grip nail 65 out of the two grip nails is integrated with the support nail 67, and the other grip nail 66 is arranged to be movable with respect to the grip nail 65. Because of the integration of the grip nail 65 with the support nail 67, the structure is simplified, and it realizes reliable supporting of the lead terminals 25 by moving the grip nail 66 as depicted in Fig. 19.

10 Fig. 20 and Fig. 21 show a transfer movement of a component from the component transfer unit 19 to the head 20, wherein portions of the lead terminals 25 located between the grip nails 65 and 66 and the support nail 67 of the component transfer unit 19 are gripped by an insertion nail 68 of the head 20. Because of the structure, in which the insertion nail 68 of the head 20 holds the lead terminals 25
15 supported at both upper and lower portions by gripping center portions of them, the insertion nails 68 can maintain stable holding, and, as a result, the head 20 can carry out reliable mounting of the component on a substrate.

Fig. 22 through Fig. 24 show sequential operation of the component transfer unit 19. Fig. 22 shows an ordinary operation of the component transfer unit 19.
20 In the figure, a transfer chuck 69 first moves toward a side of the chuck 13 as indicated by arrows "A", advances further toward an element 24 held by the chuck 13 as indicated by arrows "B" while maintaining its posture, and catches the element 24 in the same posture. The transfer chuck 69 then moves toward a side of the insertion nail 68 of the head 20 as shown by arrows "C", and returns to the
25 original position by following the arrows "A" after it hands over the element 24 to

the insertion nail of the head 20. The foregoing operation as depicted in Fig. 22 takes place under the ordinary mode. During the ordinary operation shown in Fig. 22, the insertion nail 68 of the head 20 mounts the element 24 on the substrate after it receives the element 24. In case of an error in the mounting, i.e. if the insertion

5 nail 68 failed to insert the lead terminals 25 of the element 24 into mounting holes in the substrate, and caused the element 24 to fall down, as a specific example, the failure is recovered in a manner which is shown in Fig. 23. In a state that the element 24 has fallen down, the transfer chuck 69 has handed over another element 24 to the insertion nail 68 of the head 20, and therefore the new element 24 must be

10 returned over again from the insertion nail 68 of the head 20 to the transfer chuck 69. Specifically, the transfer chuck 69 receives the new element 24 from the insertion nail 68, and stays standing after it moves back toward a direction of arrows "D" to a position distant away from the head 20, as shown in Fig. 23. Since the transfer chuck 69 normally opens when it leaves the insertion nail 68 of

15 the head 20 (in the direction of arrows "A" to be specific), it needs to remain closed in order to move it while holding the element 24 toward such a direction as arrows "D or arrows "G" in Fig. 23 and Fig. 24, by depressing a rotary plate 77, which will be described later, with a manipulator 81 on a driving body 80 constituting an exemplified structure of a special opening/closing means. The insertion nail 68 of

20 the head 20, which has been unloaded in the above step, is actuated manually, for instance, to hold the lead terminals 25 of the previously fallen down element 24, and to remount the element 24. Once the remounting is completed, the transfer chuck 69 is advanced toward arrows "E", so that it hands over the previously withheld element 24 to the head 20, and it completes the recovery operation upon

25 returning to the original position. The transfer chuck 69 again resumes its normal

operation shown in Fig. 22, and repeats the ordinary mounting. Fig. 24 shows another operation that copes with a case, in which the insertion nail 68 of the head 20 fails to mount the element 24 on the substrate, and if the failed element 24 is not reusable. In this instance, the insertion nail 68 needs to obtain again another
5 element 24, which is identical to the failed and non-reusable element 24 among those carried by the belt 6, and to mount it. For this purpose, the transfer chuck 69 receives a succeeding element 24, which it has once handed over to the insertion nail 68 after moving toward arrows "F", and sends it back to the chuck 13 attached to the belt 6 by moving along arrows "G", followed by returning again to the
10 position in front of the insertion nail 68 of the head 20. After the belt 6 rotates to carry away the previously described failed element 24 while maintaining the above state, the transfer chuck 69 moves through the locus shown in Fig. 22 to receive the element 24 from the chuck 13 attached to the belt 6, to pass it over to the insertion nail 68 of the head 20, and to complete the recovery operation for the mounting
15 error. As described above, as shown in Fig. 22 through Fig. 24, the component transfer unit 19 operates in such a manner as to turn arcuately the transfer chuck 69 of its own, or to shift the transfer chuck 69 inside and outside of the arcuate locus while keeping the same posture. In order for the component transfer unit 19 to operate in the above manner, it is so constructed as depicted in Fig. 25 through Fig.
20 34. The component transfer unit 19 is provided with three axles 70, 71 and 72, which are coaxial with one another, as shown in Fig. 25, Fig. 28, Fig. 29, Fig. 31 and Fig. 33. The axle 70 provided at an outermost position of the three is for rotatory moving the component transfer unit 19. The axle 71 provided at an inner side of the axle 70 is for shifting the transfer chuck 69 toward the insertion nail 68
25 of the head 20, toward the chuck 13 on the belt 6, or toward an opposite side thereof.

The axle 72 provided at an innermost side is for opening and closing the transfer chuck 69. They will be described now in more detail below. The axle 70 at the outermost position has a horizontally extended cam attached to an upper portion of it in Fig. 25, although not shown, and a lever is also provided for driving the cam.

5 The lever, in combination with the cam rotatory moves the component transfer unit 19 in the manner as depicted in Fig. 23 and Fig. 24. The axle 71 in the middle position will be described next. The axle 71 in the middle links to a shift lever 73 rotatably at a lower end portion of it, as perspicuously shown in Fig. 31. The shift lever 73 is L-shaped, as shown in Fig. 29, Fig. 31 and Fig. 33, and a center portion
10 of it is pivoted with an axle 74. Accordingly, a lower end of the shift lever 73 shifts the transfer chuck 69 toward the insertion nail 68 of the head 20, as is obvious between Fig. 29 and Fig. 31, when the other end of the shift lever 73 is pushed downwardly at the lower end portion of the axle 71 from a position shown in Fig. 29 to another position in Fig. 31. The axle 72 located at the innermost position
15 will be described next. As shown in Fig. 26 and Fig. 27, a lever 75 is attached at its upper end to a lower end of the axle 72, and a roller 76 is mounted on a lower end of the lever 75. The roller 76 is positioned to be in contact with an upper surface of the rotary plate 77. The rotary plate 77 is securely fixed to the rotary axle 78, and a left end of it, as depicted in Fig. 26, is connected to a lever 79 for
20 sliding the grip nail 66, which composes the transfer chuck 69. In other words, only the grip nail 66 is made to slide with respect to the grip nail 65 and the support nail 67, among the three nails 65, 66 and 67 composing the transfer chuck 69, and this is accomplished by the lever 79. A rotary movement of the lever 79 thus slides the grip nail 66, so as to hold or release the lead terminals 25 of the element
25 24, as has been described. The transfer chuck 69 needs to be operated, i.e. opening

and closing operation, at each of its shifted positions inside and outside of the rotational arc of the transfer chuck 69 as shown in Fig. 22 through Fig. 24. Although the opening and closing operation is made by the axle 72 provided at the innermost position, the roller 76 attached to the lower end of the axle 72 must be
5 capable of pushing down the rotary plate 77 at any shifted positions, both inside and outside of the rotational arc of the transfer chuck 69, in order to open and close the transfer chuck 69 by a vertical movement alone of the axle 72. Conversely, the rotary plate 77 shall be made into a board-like shape in line with the rotary axle 78, as shown in Fig. 26 and Fig. 27, in order for the roller 76 to rotate the rotary axle 78
10 via the rotary plate 77. The transfer chuck 69 can be thus opened and closed only with a vertical movement of the axle 72, by providing the board-shaped rotary plate 77.

The foregoing structure can downsize, simplify, and improve an operational reliability of an assembly for turning, shifting, and opening or closing the transfer
15 chuck 69, by providing a coaxial arrangement of the axle 70 defining a turning means for rotatory moving the transfer chuck 69, the axle 71 defining a shifting means for shifting the transfer chuck 69 inside and outside of the rotational arc of the transfer chuck 69, and the axle 72 defining an opening/closing means for opening or closing the transfer chuck 69 after it is shifted by the shifting means.

20 Fig. 35 depicts the head 20. The head 20 comprises a main head body 80 having a side wall 80a resembling the letter L in cross section and a top wall 80b, a vertically-moving means 81 for the main head body 80, an insertion nail 68 provided under the main head body 80, an opening/closing means 82 for the insertion nail 68, and a turning means 83 for moving the insertion nail 68 in a
25 forward-to-backward direction. A cam plate 84, which constitutes a part of the



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turning means 83, is detachably mounted on the main head body 80, as will be described later. The vertically-moving means 81 comprises an external axle 81a fixed to the top wall 80b of the main head body 80, and the cam plate 84 is detachably mounted with a screw 81d on a mount plate 81c attached to a lower end of an internal axle 81b disposed in the external axle 81a.

The turning means 83 has the following structure.

As shown in Fig. 35 and Fig. 36, the turning means 83 has a rotary axle 83a supported axially in through holes "A" provided at two locations in the sidewall 80a of the main head body 80.

10 An opening/closing lever 85, which represents an example of a driving lever shown in Fig. 38, and a spring lever 86 shown in Fig. 35 are integrally fixed to the rotary axle 83a on its right side and left side respectively as viewed in Fig. 35.

Furthermore, a U-shaped rotary body 87 is provided behind the opening/closing lever 85.

15 A right side wall 87a of the rotary body 87 passes through an opening 81e in the side wall 80a of the main head body 80, and protrudes at a side of the main head body 80, as shown in Fig 35. Also, a left sidewall 87b of the rotary body 87 protrudes forwardly at the outside of the main head body 80. The right sidewall 80a and the left sidewall 80b are rotatably supported in this posture by the rotary axle 83a disposed in through holes "B".

20 The rotary body 87 is provided with a lever 87c extending forwardly from an outer surface of the sidewall 87b, as shown in Fig. 37. A front end of the lever 87c stays in contact with a butt section 80c at a lower end of the main head body 80, so as to prevent a further movement of the rotary body 87 beyond that shown in Fig.

25 35.

An upper end of the first insertion nail 68a is fixed to a bottom wall 87d of the rotary body 87, as shown in Fig. 35 and Fig. 38, with a fastener although not shown in the figures.

The first insertion nail 68a and the second insertion nail 68b are engaged together, and the second insertion nail 68b is pivotally supported at a center of it to a center of the first insertion nail 68a with a pin 88. The above engagement of the first and the second insertion nails 68a and 68b can attain a reduction in size, facilitate a precise positional alignment between the first and the second insertion nails 68a and 68b, and thereby improving reliability of the operation.

The first and the second insertion nails 68a and 68b are provided with three grip nails 89 through 91 and 92 through 94 at each of their tip ends, for a total of six nails, as shown in Fig. 39. Even if components have three lead terminals 25, as depicted in Fig. 40 and Fig. 41, these grip nails 89 through 94 are able to hold all of the lead terminals 25 reliably, as shown in Fig. 40 and Fig. 41. Because these lead terminals 25 are held individually between the grip nails 89 through 91 and 92 through 94 of the first and the second insertion nails 68a and 68b, intervening spaces between the lead terminals 25 are not altered, thereby resulting in a smooth mounting of the component.

On the contrary, in the case of another structure, which holds the lead terminals 25 at two sides of an insertion nail 89A with each of other insertion nails 89B and 89C, as shown in Fig. 59 and Fig. 60, an intervening space between the lead terminals 25 varies between "A" and "A + α " depending on thickness of the lead terminals 25. This can result in the lead terminals 25 eccentric with respect to via holes 107 in a substrate 106, so as to carry a risk of mounting failures.

Reference is made again to Fig. 39 through Fig. 41 for the following



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description. The grip nail 92 at one side of the second insertion nail 68b is so constructed that it has a narrower intervening space and larger flexibility than any of the grip nail 93 at the other side and the grip nail 94 in the middle. By providing the second grip nail 68b with the grip nail 92 having the narrower intervening space
5 than the other grip nails 93 and 94, the second grip nail 68b is able to hold a component securely with the flexible grip nail 92, even if terminals of the component are deviated in diameter.

That is, the second grip nail 68b provided with the grip nail 92 having narrower intervening space and larger flexibility is now able to make effective use
10 of this flexible grip nail 92 also to hold the lead terminals 25, whereas only one pair of the grip nails of the first and the second insertion nails 68a and 68b would otherwise hold the lead terminals 25, in the case of a component having a terminal of larger diameter.

On the other hand, a cam follower 95 is provided rotatably on a top end of the
15 second insertion nail 68b, as shown in Fig. 38. This cam follower 95 stays in contact with a cam surface 96 at a left end of the opening/closing lever 85.

In addition, a cam follower 97 at a right end of the opening/closing lever 85 stays in contact with a cam surface 98 of the cam plate 84 as shown in Fig. 38.

In the above structure, a spring 101 and a spring 104 are placed to provide
20 spring tensions between a pin 99 of the spring lever 86 and a pin 100 on the butt section 80c of the main head body 80, and between a pin 102 of the rotary body 87 and a pin 103 on the butt section respectively as shown in Fig. 35. Also, a spring 105 is placed between the lever 87c and an upper portion of the second insertion nail 68b to provide a repulsive force.

25 These components operate in a manner as described below.

First, the transfer chuck 69 of the component transfer unit 19 hands over an element 24 to the insertion nail 68 at an upper position, as shown in Fig. 42.

At this moment, the first and the second insertion nails 68a and 68b need to be opened as shown in Fig. 40. In order to accomplish this, the internal axle 81b is
5 depressed downwardly, which in turn depresses the opening/closing lever 85 via the cam plate 84.

Subsequently, the cam surface 96 of the opening/closing lever 85 shifts backward, and the cam follower 95 on the upper end of the second insertion nail 68b takes its position at a front side of the cam surface 96. This makes the
10 repulsive force of the spring 105 to push the upper portion of the second insertion nail 68b toward right side in Fig. 35 and Fig. 38.

As a result, the first and the second insertion nails 68a and 68b open as shown in Fig. 40, and the element 24 is transferred from the transfer chuck 69.

Next, the internal axle 81b rises in order to close the first and the second
15 insertion nails 68a and 68b. The lead terminals 25 are held among the grip nails 89 through 94, as shown in Fig. 41, and the element 24 is thus held up as shown in Fig. 43.

The external axle 81a and the internal axle 81b are synchronized to move downward, so as to lower the main head body 80 toward a substrate 106, as shown
20 in Fig. 44, thereby inserting the lead terminals 25 into via holes 107 in the substrate 106 as illustrated in Fig. 49.

During this movement, a catch pin 108 rises up in its position underneath the substrate 106, and waits for the lead terminals 25 to come down. When the lead terminals 25 are inserted into the via holes 107, a pusher 109 disposed coaxially in
25 the internal axle 81b is lowered to be in contact with a top of the element 24, so that

the element 24 is caught at the top and bottom ends between the pusher 109 and the catch pin 108.

Fig. 45 depicts the first and the second insertion nails 68a and 68b in their open position as shown in Fig. 40 as the internal axle 81b is depressed. However, the element 24 does not fall down even after it is released from the first and the second insertion nails 68a and 68b, since the element 24 is held now at the top and bottom ends of it by the pusher 109 and the catch pin 108, as depicted in Fig. 49.

With the element 24 in this posture, the insertion nail 68 retracts away from the element 24, as shown in Fig. 50. When the insertion nail 68 completes the retraction, the pusher 109 and the catch pin 108 start moving down until a bottom end of the element 24 finally touches an upper surface of the substrate 106 as shown in Fig. 46 and Fig. 51. The catch pin 108 moves further down, as shown in Fig. 51, while the top surface of the element 24 is kept pushed by the pusher 109. Mounting of the element 24 completes thereafter, when the anvil mechanism 23, although not shown in the figures, cuts and clinches lower ends of the lead terminals 25.

During this operation, the insertion nail 68 moves up while shifting backward as shown in Fig. 46 through Fig. 48.

The insertion nail 68 shifts backward in the following manner.

The internal axle 81b stays at its lower position with respect to the external axle 81a, in order to keep the first and the second insertion nails 68a and 68b open. While keeping the internal axle 81b at the depressed position, only the external axle 81a is lifted.

This movement also provides the opening/closing lever 85 with a lifting force, since the opening/closing lever 85 is attached to the external axle 81a together with

the main head body 80.

However, because the cam plate 84 fixed to the internal axle 81b remains at its lower position, the opening/closing lever 85 turns downwardly as shown by arrow "K" in Fig. 35. And, a rear edge 110 in the back side of the cam surface 96 eventually comes in contact to a butt surface 111 inside the side wall 87b of the rotary body 87 below the through hole "B", and it turns the rotary body 87 backward to a direction of arrow "S" as shown in Fig. 37.

As the rotary body 87 is provided with the insertion nail 68, as described above, the insertion nail 68 moves upwardly while turning backward in a manner to avoid the mounted element 24, as shown in Fig. 46 and Fig. 47.

In the step of Fig. 47, the internal axle 81b starts moving upward, and the insertion nail 68 also moves gradually into its original posture as shown in Fig. 48.

Besides, the pusher 109, which is coaxial with the internal axle 81b, also starts moving upward at the moment shown in Fig. 47.

One of the distinctive features of this exemplary embodiment is that the cam plate 84 is easily removable from the mount plate 81c and replaceable by removing a screw 81d.

Fig. 52 through Fig. 54 show a case wherein there is a large space between elements 24 on a substrate 106, and Fig. 55 through Fig. 57 show another case wherein there is a small space between elements 24 on a substrate 106.

In the case shown in Fig. 52 through Fig. 54, the insertion nail 68 does not strike an elements 24 already mounted behind the insertion nail 68, even if an amount of backward movement (amount of retractive movement) is increased in order for the insertion nail 68 to cope with large elements 24 such as those shown in Fig. 52 through Fig. 54, because the space between the elements 24 is large enough.



However, if the space is reduced between the elements 24 on the substrate 106 in an attempt to increase mounting density on the substrate 106, the insertion nail 68 may strike the element 24 behind it when the insertion nail 68 makes a large retractive movement backward.

5 In this case, it is often likely that smaller size elements 24 such as those shown in Fig. 55 through Fig. 57 are used as compared to the case of Fig. 52 through Fig. 54. It is therefore necessary to reduce the amount of backward movement (amount of retractive movement) of the insertion nail 68.

10 This object is accomplished in the present exemplary embodiment by removing the screw 81d and replacing the cam plate 84. The cam plate 84 shown in Fig. 55 through Fig. 57 has a longer slope 84a on a cam surface contacting the cam follower 97, and a top wall 84b in a higher position, as compared to the other cam plate 84 shown in Fig. 52 through Fig. 54.

15 As described above, since the insertion nail 68 moves backward as the cam follower 97 is pushed relative to the cam surface 84, and eventually turns the rotary body 87, the backward movement of the insertion nail 68 can be altered from a phantom line "Y" to a smaller phantom line "X" in Fig. 58, if the slope 84a of the cam plate 84 is longer and the top wall 84b is higher.

20 This enables the head 20 to mount the element 24, as shown in Fig. 56, even if there is a limited space between the elements 24 on the substrate 106, and thereby increasing the mounting density of the elements 24 on the substrate 106.

Fig. 61 through Fig. 64 depict catch pins. A catch pin 108a has a circular recess at a top of it, a catch pin 108b has a conoidal recess, a catch pin 108c has a deep recess, and a catch pin 108d is made out of a cylinder.

25 All of the catch pins 108, 108a through 108d are made to have diameters



greater than a diameter of the via hole 107 perforated in the substrate 106, as shown in Fig. 49. The head 20 is able to mount the element 24 reliably on the substrate 106 by positively holding the top and the bottom of the element 24 with the pusher 109 and the catch pin 108, 108a through 108d, even if the lead terminals 25 of the element 24 are slightly eccentric, because the catch pins 108, 108a through 108d are greater in diameter than the via hole 107 in the substrate 106.

As has been described, the component mounting apparatus of the present exemplary embodiment comprises: the component delivery unit; the chuck 13 provided on the component delivery unit; the component transfer unit 19 for receiving a component held by the chuck 13; and the head 20 for receiving the component held by the component transfer unit 19. The head 20 comprises: the main head body 80; the vertically-moving means 81 for the main head body 80; the insertion nail 68 provided at the bottom portion of the main head body 80; the opening/closing means 82 of the insertion nail 68; and the turning means 83 for moving the insertion nail 68 between forward and backward directions. The cam plate 84, which constitutes the turning means 83, is mounted detachably on the main head body 80, so that a locus of backward movement of the insertion nail 68, i.e. a retractive path of the insertion nail 68 after mounting a component, can be altered readily by mounting another cam plate 84 of different shape of the main head body 80. This can result in an increase of mounting density of components on a substrate, and an improvement of workability, since all that is necessary is to replace only the cam plate 84, without requiring any alignment of the insertion nail 68 with respect to the substrate after the replacement.

Fig. 65 depicts the anvil mechanism 23.

The anvil mechanism 23 comprises: a mounting base 112 having two slant

surfaces 112a opposed to each other; first through third movable blades 113, 114 and 115, and first and second fixed blades 116 and 117, all of which are disposed on the slant surfaces 112a of the mounting base 112 as shown in Fig. 75; the previously described catch pin 108; and a rotating and vertically moving mechanism.

5 The rotating and vertically moving mechanism will be described hereinafter.

A reference numeral 118 in Fig. 65 is a base, which represents a basis of movement, and the base 118 neither rotates nor moves vertically.

A rotary bearing 119 is fixed to the base 118, and a rotary axle 120 is rotatably supported on the inside of the rotary bearing 119. A timing pulley 121 is
10 fixed to a lower end of the rotary axle 120, and a belt is looped up around the timing pulley 121, though the belt is not shown in the figure.

A cylinder 122 is placed inside of the rotary axle 120 in a manner that the cylinder 122 rotates together with the rotary axle 120, but it is freely movable vertically with respect to the rotary axle 120. A cam follower, not shown in the
15 figure, is engaged in a cam follower receptacle 124 at a lower end of the cylinder 122, and the cam follower moves the mounting base 112 together with the cylinder 122.

In other words, the mounting base 112 is fixed on top of the cylinder 122.

The cylinder 122 is also provided with an operating rod 123 in its inside in
20 such a manner that the operating rod 123 rotates together with the cylinder 122, but it is freely movable vertically with respect to the cylinder 122.

Another cam follower, also not shown in the figure, is engaged in a cam follower receptacle 125 at a lower end of the operating rod 123, and the cam follower vertically moves the operating rod 123, which in turn opens and closes the
25 first through third movable blades 113 through 115 shown in Fig. 75, via a coupling

lever 126.

Furthermore, the operating rod 123 is provided in its inside with a cylinder 127 in a manner that the cylinder 127 is rotatable together with the operating rod 123. The cylinder 127 is also vertically movable freely with respect to the
5 operating rod 123, and the vertical movement is produced by a cam follower, which is not shown in the figure, engaged in a cam follower receptacle 128.

The catch pins 108 are mounted on a top end of the cylinder 127 as shown in Fig. 82 and Fig. 83. Specifically, a supporting body 129 is disposed to bridge across an upper end opening of the cylinder 127.

10 The supporting body 129 has a board-like shape, and a supporting pin 130 is inserted horizontally through the board-like supporting body 129. The catch pins 108 are secured in the supporting body 129 with their lower ends butted on the supporting pin 130, so that heights of the catch pins 108 become uniform, and thereby lower ends of the lead terminals 25 are supported uniformly by the catch
15 pins 108.

An upper peripheral portion of the cylinder 127 is formed with a male thread 131, and a notch 132 is provided from the top end of the cylinder 127 vertically downward through the male thread 131 as shown in Fig. 83 and Fig. 84. The supporting pin 130 is engaged in the notch 132. This engagement of the
20 supporting pin 130 in the notch 132 of the cylinder 127 fixes heights of the catch pins 108 with respect to the cylinder 127, and thereby resulting in an uniform support of the lower ends of the lead terminals 25 by the catch pins 108.

In addition, a fitting body 133 is provided under the supporting body 129 to fit into an inside of the cylinder 127, so as to prevent the supporting body 129 from
25 wobbling.

A cap 135 having a female thread around an internal periphery and an opening 134 on top is screwed detachably over the top end of the cylinder 127 in the above arrangement. The cap 135 screw-fitted on the top end not only prevents the catch pins 108 from coming off upwardly, but also makes the catch pins 108
5 replaceable with ease, when necessary, by removing the cap 135.

The mounting base 112 will be described next.

The mounting base 112 is provided with the first through third movable blades 113, 114 and 115, and the first and second fixed blades 116 and 117, all mounted by means of the opposing pair of slant surfaces 112a, as has been
10 described. They are positioned relative to one another as shown in Fig. 75 and Fig. 76.

That is, two of the movable blades, 113 and 114, and one of the fixed blades, 117, are positioned at one side, and the one remaining movable blade 115 and the fixed blade 116 are positioned at the other side, so as to be capable of cutting all
15 three lead terminals 25 at once.

The first and the second fixed blades 116 and 117 are provided with their respective guide surfaces 136 and 137 at sides facing outward in this arrangement, as shown in Fig. 75 and Fig. 76. Cutting edges 138, 139 and 140 are formed beside the guide surfaces 136 and 137, pointing toward their respective counterparts,
20 the first and the second movable blades 113 and 114, as well as the third movable blades 115.

The first through third movable blades 113, 114 and 115, and the first and second fixed blades 116 and 117 are therefore in their open positions at the moments shown in Fig. 75 and Fig. 76.

25 The lead terminals 25 are inserted during the moment of open positions, and

they are cut off when the cutting edges 138 through 140 and the first through third movable blades 113 through 115 are closed, as details of which will be later described.

It is important to avoid undesirable gap from being developed between the first through third movable blades 113 through 115 and the cutting edges 138 through 140 of the first and second fixed blades 116 and 117 in order to cut the lead terminals 25.

In the present exemplary embodiment, the first and the second fixed blades 116 and 117 are loosely set on the slant surfaces 112a of the mounting base 112 in order to achieve the above object. Initially, the first and second movable blades 113 and 114, as a pair, are slid over the slant surface 112a of the mounting base 112 up to positions, which are slightly above anticipated points where the movable blades meet with the cutting edges 138 and 139 of the first fixed blade 116. The first fixed blade 116 is then pushed up along the slant surface 112a of the mounting base 112.

This brings the guide surface 136 of the first fixed blade 116 to get in contact with a lower surface of the first movable blade 113, which is already in the slid up position. The first fixed blade 116 is now secured to the mounting base 112 at this point.

The above procedure assures the first movable blade 113 to slide up along the guide surface 136 of the first fixed blade 116, after it slides up over the slant surface 112a. As a result, no undesirable gap is created between the first movable blade 113 and the cutting edge 138 of the first fixed blade 116, and thereby the lead terminal 25 is cut smoothly.

Also, since the second movable blade 114 slides over the same slant surface



112a of the mounting base 112 as the first movable blade 113, undesirable gap is also avoided between the second movable blade 114 and the second cutting edge 139, when the first fixed blade 116 is properly positioned according to the above procedure. Hence, the lead terminal 25 can be cut smoothly by the second
5 movable blade 114 and the second cutting edge 139.

In the like manner, the third movable blade 115 is slid over the slant surface 112a of the mounting base 112 up to a position slightly above an anticipated point where the movable blade 115 meets with the cutting edge 140 of the second fixed blade 117, when setting a position of the second fixed blade 117. The second fixed
10 blade 117 is then pushed up along the slant surface 112a of the mounting base 112 until the guide surface 137 of the second fixed blade 117 comes in contact with a lower surface of the third movable blade 115. The second fixed blade 117 is then secured to the mounting base 112 at this position after it contacts with the lower surface of the third movable blade 115. In this way, the third movable blade 115
15 reaches the cutting edge 140 after sliding up along the guide surface 137, when the third movable blade 115 is moved thereafter, and therefore no undesirable gap is made between them, thereby cutting the lead terminal 25 smoothly.

Describing next pertains to mounting of the element 24 on the substrate 106.

Fig. 68 depicts a state just prior to the mounting, in which the mounting base
20 112 locates below the substrate 106, and the insertion nail 68 locates above the substrate 106. From this state of Fig. 68, the cam follower receptacles 125 and 128 are raised by their respective cam followers as shown in Fig. 69.

Since an amount of rise of the cylinder 127 is set to be greater than that of the cylinder 122, the catch pins 108 moves up in vicinity of a lower surface of the
25 substrate 106 as shown in Fig. 49.

The first through the third movable blades 113 through 115, and the first and the second fixed blades 116 and 117 are in their open positions, as shown in Fig. 75 and Fig. 76, between the steps depicted in Fig. 68 through Fig. 72, and the catch pins 108 are therefore vertically movable through these openings.

5 The lead terminals 25 are then inserted through the via holes 107 in the substrate 106 by lowering the insertion nail 68 until they contact to the catch pins 108. While maintaining this state, the pusher 109 is lowered from above the element 24 until it touches the element 24 as shown in Fig. 70.

10 This way, the element 24 is held at the top and the bottom between the pusher 109 and the catch pins 108, and the insertion nail 68 retracts outward with the element 24 held in this position, as shown in Fig. 70 and Fig. 71.

15 When the pusher 109 and the catch pins 108 are lowered in a synchronized motion, a bottom end of the element 24 comes to contact with an upper surface of the substrate 106 as shown in Fig. 72, and the catch pins 108 at one side move apart from the bottom ends of the lead terminals 25.

20 When the operating rod 123 is lowered next, as shown in Fig. 73, a point of application of the coupling lever 126 comes down, and a point of action comes up round a fulcrum. This moves the first through the third movable blades 113 through 115 upward, and thereby the first through the third cutting edges 138 through 140 and the first through the third movable blades 113 through 115 cut the lead terminals 25 in a manner as shown in Fig. 77 and Fig. 78. The first through the third movable blades 113 through 115 slide up further beyond that point along the slant surface 112a, and bend the lead terminals 25 at their lower ends, after cut off, toward the substrate 106 as shown in Fig. 79, so as to prevent the element 24
25 from coming off the via holes 107 in the substrate 106.

Fragments 141 of the cut off lead terminals 25, shown in Fig. 78, fall into the cylinder 127 through the opening 134 in the cap 135 shown in Fig. 82 and Fig. 83, and they are collected in a storage vessel provided below a bottom opening of the cylinder 127.

5 The cylinder 122 is then lowered together with the operating rod 123 as shown in Fig. 74. During this lowering step, an amount of down movement of the cylinder 122 is set to be greater than that of the operating rod 123, which has been lowered in the preceding step in Fig. 73, for bringing them into the standby mode of Fig. 68.

10 The first through the third movable blades 113 through 115 and the first and the second fixed blades 116 and 117 are replaceable by taking steps, which will be described hereinafter.

15 First, the mounting base 112 is freed from a mount section 143 at a top end of the cylinder 122, in Fig. 65, by removing the screw 142 shown in Fig. 66, and it is rotated for 90 degrees while maintaining its posture.

 The top end of the cylinder 123 is provided with two flanges 144 and 145 around the periphery at a predetermined space as shown in Fig. 85, and the upper flange 144 has two cut-back portions 146 at opposite sides.

20 Therefore, linkage of the coupling levers 126 connecting between the operating rod 123 and the mounting base 112 is disengaged from the cut-back portions 146 by rotating the mounting base 112 (or the operating rod 123) by 90 degrees, when removing the mounting base 112. A connection of the mounting base 112 to the operating rod 123 is made easily by rotating the mounting base 112, when reinstalling it, thereby providing good workability.

25 As shown in Fig. 67, the points of action of the coupling levers 126 disengage

from the first through the third movable blades 113 through 115, when the mounting base 112 is removed from the operating rod 123. The first through the third movable blades 113 through 115, in particular, come off the slant surfaces 112a by their own weights.

5 Fig. 80 and Fig. 81 depicts another exemplary embodiment, in which grooves 147 are formed on top and bottom surfaces of the first through the third movable blades 113 through 115. The first through the third movable blades 113 through 115 and the first and the second fixed blades 116 and 117 produce pulverized cutting dust from cut-off portions of the lead terminals 25, when they cut off the
10 lead terminals 25. The grooves 147 can prevent the cutting dust falling on the upper and/or lower surfaces of the first through the third movable blades 113 through 115 from accumulating on these surfaces, and from impeding the first through the third movable blades 113 through 115 to slide.

 In other words, the first through the third movable blades 113 through 115
15 will fail to operate due to a jam between the first through the third movable blades 113 through 115 and their sliding surfaces, if cutting dust falls and accumulates upon the first through the third movable blades 113 through 115 and the sliding surfaces. However, because the grooves 147 are formed on top and bottom surfaces of the first through the third movable blades 113 through 115, as described
20 above, the cutting dust moves into the grooves 147 as the first through the third movable blades 113 through 115 slide. Consequently, operating failure can be avoided, since the cutting dust does not accumulate on the first through the third movable blades 113 through 115 and the sliding surfaces.

 In addition, a through hole 148 is provided at lower end of the upper and
25 lower grooves 147 for discharging through it the cutting dust moved into the

grooves 147. The through hole 148 can prevent the first through the third movable blades 113 through 115 from failing to operate eventually, if the cutting dust accumulates in the grooves 147 by such an amount that the grooves 147 no longer accept the cutting dust to move in from the first through the third movable blades 113 through 115 and the sliding surfaces.

As has been described, the anvil mechanism 23 comprises: a mounting base 112 having two slant surfaces 112a opposed to each other; fixed blades 116 and 117 mounted on one of the slant surfaces 112a of the mounting base 112; and movable blades 113 through 115 mounted on the other slant surface 112a of the mounting base 112. The fixed blades 116 and 117 are mounted on the mounting base 112 with a fixing means by taking the steps of: sliding up the movable blades 113 through 115, mounted on the slant surfaces 112a of the mounting base 112, over the slant surfaces 112a up to positions, which are slightly above anticipated points where the movable blades 113 through 115 meet with the fixed blades 116 and 117; and, while maintaining the above positions, setting positions of the fixed blades 116 and 117 by sliding them upwardly over the slant surfaces 112a of the mounting base 112 until they come in contact to the movable blades 113 through 115.

In other words, the fixed blades 116 and 117 and the movable blades 113 through 115 come to be in their proper positions with respect to each other, when the fixed blades 116 and 117 are secured to the mounting base 112 with the fixing means after they are set in positions by sliding them upwardly over one slant surface 112a of the mounting base 112 until they come in contact to the movable blades 113 through 115, while the movable blades 113 through 115, mounted on the other slant surface 112a of the mounting base 112, are slid over that slant surface 112a and maintained in their positions above the points where the movable blades

113 through 115 are anticipated to meet with the fixed blades 116 and 117.

Because of the above, the movable blades 113 through 115 never collide with the fixed blades 116 and 117 during the sliding movement. Also, since the gap between the blades is not necessarily too large in order to avoid collisions, they can
5 cut the lead terminals 25 reliably.

Fig. 88 through Fig. 94 depicts still another exemplary embodiment of the present invention.

In this exemplary embodiment, the pusher 109 is so constructed that it is held in a standby position above an element 24 representing a component, before it
10 comes in contact with the element 24 and presses it down toward the substrate 106. The standby position is adjusted according to a height of the element 24 held by the head 20.

As for the structure, in specific, an upper end of the pusher 109 is linked freely movably, as shown in Fig. 88, with a point of action of a first lever 112K
15 shown in Fig. 92 having a dog-legged shape.

Besides, the upper end of the pusher 109 is biased with a downward force by a spring 113K at all times. A fulcrum 114K of the first lever 112K is supported axially by a second lever 115K shown in Fig. 93, and a cam follower 116K, which functions as a point of application of the first lever 112K, engages with a first
20 cam 118K, which is supported axially by an axle 117K. The second lever 115K is also supported axially by the axle 117K, and a cam follower 119K serving as its point of application engages with a second cam 121K, which is rotatably supported by an axle 120K.

The second lever 115K is biased with a downward force by a spring 122K at
25 all times.



The foregoing structure operates in a manner as described hereinafter.

Described first is a case wherein a component (an element 24) to be mounted is large in height, as shown in Fig. 88.

In this instance, a servomotor 124K shown in Fig. 94 rotates the first cam 118K, making good use of an idle time of the pusher 109 before the pusher 109 actually depresses the element 24 toward the substrate 106, so as to position the cam follower 116K in a recess of the first cam 118K. This causes the point of action of the first lever 112K to rotate downwardly round the fulcrum 114K, and the pusher 109 slowly moves down until it touches on the element 24.

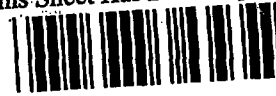
When the pusher 109 touches on the element 24, a contact-sensitive switch (i.e. a detector 123K in Fig. 94), though not shown in the figures, operates to carry out calculation of a height of the element 24 based on a rotational angle of the first cam 118K.

This enables the apparatus to recognize the magnitude of height of the element 24 to be mounted now, and the pusher 109 is held in this position for a standby. Subsequently, the pusher 109 starts from this standby position, when it presses down the element 24 toward the substrate 106 as shown in Fig. 89.

Although not shown in Fig. 88 through Fig. 91, there is the catch pin 108 under the substrate 106. The pusher 109 and the catch pin 108 mount the element 24 by holding it between top and bottom of the element 24 and the lead terminals 25, as shown in Fig. 49 through Fig. 51.

Fig. 90 and Fig. 91 depict another case wherein the element 24 is small in height.

In this case, in the like manner as described above, the pusher 109 is made to touch on the element 24, as shown in Fig. 90, by rotating the first cam 118K during



an idle time of the pusher 109 before the pusher 109 starts a pressing motion.

The cam follower 116K of the first lever 112K stays at a position deeper in the recess of the first cam 118K, in this case, as is obvious by comparing Fig. 90 with Fig.88. This enables the apparatus to recognize the element 24 as being large
5 in height, and the pusher 109 starts the pressing movement hereafter as shown in Fig. 91.

In the real mounting operation, the apparatus carries out a height determination in the same manner as above for every pieces of the element 24, even if 100 pieces of the element 24 are mounted on each substrate 106 for instance, and
10 results are all stored in a memory 125K shown in Fig. 94 by way of a controller 126K.

Accordingly, the apparatus makes use of the idling time to lower the pusher 109 in advance to a height equal to a sum of 1 mm, for example, added to the determined height of each component 24 when mounting on the second and
15 subsequent substrates 106.

The above operation substantially shortens the mounting time required for lowering the pusher 109, since the pusher 109 starts pressing down each of every 100 pieces of the elements 24 from the position of merely 1mm above the element 24 during the mounting operation.

20 A reason for making the pusher 109 to stand by at 1 mm above the element, for example, will be described hereinafter.

The pusher 109 is integrally assembled with the head 20. The reason is therefore to prevent an element 24 from striking the lowering pusher 109, when the component transfer unit 19 transfers the element 24 to the insertion nail 68 of the
25 head 20, as shown in Fig. 21.

As described, the present exemplary embodiment comprises: a component having lead terminals 25 extending downwardly; the head 20 for holding and moving the component to a position where the lead terminals 25 are above the via holes 107 to be inserted in the substrate 106; and the pusher 109 for pressing down the component held by the head 20 toward the substrate 106. The pusher 109 is made to wait at the standby position above the component before it comes in touch on the component and presses it toward the substrate. The standby position is determined according to a height of the component held by the head 20. Thus, the present exemplary embodiment shortens time for the pusher 109 to make pressing operation of the component, and therefore reduces a mounting time, since it adjusts the standby position vertically in advance according to a height of the component to be mounted on the substrate.

Fig. 95 through Fig. 105 depict a transfer means for transferring a substrate 106 onto an X - Y table 22, which is utilized as an example of a positioning table, while guiding it through the rails 21 shown in Fig. 1.

In other words, the transfer means of the present exemplary embodiment comprises: a shaft holder 112N; a transfer shaft 113N provided freely in and out from the shaft holder 112N toward the X - Y table 22; a transfer pin 114N of Fig.97 provided in a direction toward the substrate 106 at one side of the transfer shaft 113N nearer to the X - Y table 22; and so on. The transfer pin 114N is provided slidably on an axle 115N, which is disposed in the direction toward the substrate 106 at one side of the transfer shaft 113N nearer to the X - Y table 22. A spring 116N is connected to the transfer pin 114N as a biasing means. A pin-shifting unit for shifting the transfer pin 114N comprises a lever 117N provided at one side of the shaft holder 112N nearer to the X - Y table 22, and a lever actuator 118N linked to

one end of the lever 117N. The other end of the lever 117N is linked to the transfer pin 114N.

Each of the above-cited elements will now be described in more detail.

The rails 21 shown in Fig. 1 are composed of longitudinally separated
5 portions connected at a right side of the X – Y table 22.

Portions of the rails 21 at the rightmost end shown in Fig. 1 are fixed to interior sides of wall plates 120N as shown in Fig. 95, and that their free ends (at a side nearer to the X – Y table 22) are vertically rotatable around a shaft 119N.

The shaft holder 112N is fixed to an exterior side of one of the wall plates
10 120N, and one end of this shaft holder 112N nearer to the X – Y table 22 is also vertically rotatable around the shaft 119N.

The transfer shaft 113N is housed freely egressably within the shaft holder 112N, and a belt 121N drives the transfer shaft 113N to egress and ingress.

In the foregoing structure, as can be comprehended through the description of
15 Fig.95, the rightmost side portions of the rails 21 shown in Fig. 1 are in such state that their ends nearer to the X – Y table 22 are raised upward with the shaft 119N as an axis, as shown in Fig. 105.

Belts 122N are provided on the rails 21 at both right and left sides as shown in Fig. 95, and these belts 122N are driven to transfer the substrate 106 until the
20 substrate 106 strikes a stopper (not shown) placed at a distal end of the rails 21, as shown in Fig. 100.

An optical sensor 123N is provided at a position immediately before the distal end of the rails 21. Although a motor (not shown) for driving the belt 122N is deenergized when the substrate 106 passes by the optical sensor 123N, the inertia
25 brings the substrate 106 to strike the stopper, and stays in position as described



above.

While keeping the above position, the lever actuator 118N in Fig. 97 shifts a right end of the lever 117N toward left as shown in Fig. 98.

5 A left end of the lever 117N is linked to a recess 125N in a holder 124N of the transfer pin 114N as shown in Fig. 97, and the holder 124N is axially supported by the axle 115N to be freely slidable. Therefore, the holder 124N, together with the transfer pin 114N, slides down, if the left end of the lever 117N moves downward as shown in Fig. 98, thereby the transfer pin 114N slips into a through hole 126N in the substrate 106 as shown in Fig. 98.

10 While maintaining the above condition, an end of the shaft holder 112 N moves down about the shaft 119N as an axis, as shown in Fig. 101, so as to make the portions of rails 21 in this position shown in Fig. 95 to be flush with the other portions of the rails 21 shown in Fig. 1 extending to the X - Y table 22. With the rails in the above position, the belt 121 is rotated in order to drive only the transfer
15 shaft 113N to protrude from the shaft holder 112N toward the X - Y table 22 as shown in Fig. 102.

This also transfers the substrate 106 onto the X - Y table 22 as shown in Fig. 99 and Fig. 102, and sets it in position.

As a push lever 127N is also mounted integrally on a tip end of the holder
20 124N as shown in Fig. 96, it pushes out another substrate 106, on which the head 20 has already completed mounting of elements 24 on the X - Y table 22, toward the rails 21 at a collecting end.

Then, the transfer shaft 113N is raised as shown in Fig. 103 with the shaft 119N as an axis, so that the transfer pin 114N is disengaged from the through hole
25 126N in the substrate 106. The substrate 106 on the X - Y table 22 is freed

thereafter from constraint of the transfer pin 114N, and the X – Y table 22 and the head 20 start operating for mounting elements 24.

On the other hand, the raised transfer shaft 113N enters into the shaft holder 112N due to reversed rotation of the belt 121N, as shown in Fig. 104.

5 And then, the lever actuator 118N shifts the right end of the lever 117N toward right in order to raise the transfer pin 114N upward via the holder 124N as shown in Fig. 97 and Fig. 105, and waits for arrival of a subsequent substrate 106.

10 In short, the present exemplary embodiment facilitates reduction in weight, and resultant increase of speediness, since one side of the transfer shaft 113N nearer to the X – Y table 22 includes only the structure comprising the transfer pin 114N and the spring 116N for biasing the transfer pin 114N in a direction toward the substrate 106.

15 In the present exemplary embodiment, as described above, the transfer means for transferring the substrate 106 being guided by the rails 21 onto the X – Y table 22 comprises: the shaft holder 112N; the transfer shaft 113N disposed freely in and out from the shaft holder 112N toward the X – Y table 22; the transfer pin 114N provided at one end of the transfer shaft 113N nearer to the X – Y table 22 in a manner to direct toward the substrate 106; the spring 116N for biasing the transfer pin 114N toward the substrate 106; and the lever 117N provided at the end of the shaft holder 112N nearer to the X – Y table 22 for shifting the transfer pin 114N to a side opposite to the substrate 106 against a biasing force of the spring 116N.

20 Accordingly, since the transfer means has the structure, wherein one side of the transfer shaft nearer to the positioning table has only the transfer pin 114N and the spring 116N for biasing the transfer pin 114N toward the substrate 106, it facilitates reduction in weight, and resultant increase of speediness.

INDUSTRIAL APPLICABILITY

As has been described, a component mounting apparatus of the present invention comprises: a component delivery unit; a chuck provided on the component delivery unit; a component transfer unit for receiving a component held
5 by the chuck; and a head for receiving the component held by the component transfer unit. The head comprises: a main head body; a vertically-moving means for the main head body; an insertion nail provided at a bottom portion of the main head body; an opening/closing means for the insertion nail; and a turning means for
10 moving the insertion nail between forward and backward directions. A cam plate constituting the turning means is mounted detachably on the main head body, so that a locus of backward movement of the insertion nail, i.e. a retractive path for the insertion nails after mounting of a component, can be altered readily by mounting another cam plate of different shape. This can result in an increase of mounting
15 density of components on a substrate. In addition, it requires only a replacement of the cam plate, but no alignment of the insertion nail with respect to the substrate after the replacement, thereby improving workability.

CLAIMS

1 1. A component mounting apparatus comprising:
2 a component delivery unit;
3 a chuck provided on said component delivery unit;
4 a component transfer unit for receiving a component held by said chuck;
5 and
6 a head for receiving the component held by said component transfer unit,
7 said head having a main head body, a vertically-moving means for said main head
8 body, an insertion nail provided at a bottom portion of said main head body, an
9 opening/closing means for said insertion nail, and a turning means for moving said
10 insertion nail between forward and backward directions,
11 wherein said turning means comprises a cam plate attached detachably to
12 said main head body.

1 2. The component mounting apparatus according to claim 1, wherein said
2 vertically-moving means comprises an external axle mounted on top of said main
3 head body, and said cam plate is attached detachably to a mount plate attached to a
4 lower end of an internal axle provided inside of said external axle.

1 3. The component mounting apparatus according to claim 1, further
2 comprising a rotary body engaged to said cam plate to be rotatable between forward
3 and backward directions of said main head body, wherein a first insertion nail
4 constituting said insertion nail is attached to said rotary body, and a second insertion
5 nail also constituting said insertion nail is attached openably to said first insertion





6 nail.

1 4. The component mounting apparatus according to claim 3, wherein an
2 upper portion of said second insertion nail and a portion of a driving lever
3 corresponding to said upper portion make contact with each other via a cam surface.

1 5. The component mounting apparatus according to claim 3, wherein said
2 first and said second insertion nails are each provided with at least three grip nails
3 formed at distal end thereof.

1 6. The component mounting apparatus according to claim 5, wherein one
2 of said grip nails at one side of said second insertion nail has a narrower intervening
3 space and larger flexibility than any of said grip nails at the other side and in the
4 middle.

1 7. The component mounting apparatus according to claim 3, wherein said
2 first insertion nail and said second insertion nail are engaged together, and said
3 second insertion nail is pivotally supported at a center thereof by a center of said
4 first insertion nail.

1 8. The component mounting apparatus according to claim 1, further
2 comprising a pusher for pressing a top of the component held by said insertion nail,
3 and a catch pin for receiving a terminal extending downward from said component,
4 wherein said catch pin receives a lower end of said terminal underneath the
5 substrate whereon said component is being mounted.



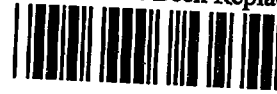
1 9. The component mounting apparatus according to claim 8, wherein said
2 catch pin has a diameter greater than a diameter of a via hole provided in said
3 substrate.

1 10. A component mounting apparatus comprising:
2 a component delivery unit;
3 a chuck provided on said component delivery unit;
4 a component transfer unit for receiving a component held by said chuck,
5 said component having a lead terminal extending downwardly therefrom;
6 a head for receiving said component held by said component transfer unit,
7 and moving said lead terminal above a via hole of a substrate wherein said lead
8 terminal is being inserted; and
9 a pusher for pressing down said component held by said head toward said
10 substrate,

11 said head having a main head body, a vertically-moving means for said
12 main head body, an insertion nail provided at a bottom portion of said main head
13 body, an opening/closing means for said insertion nail, and a turning means for
14 moving said insertion nail between forward and backward directions,

15 wherein said turning means comprises a cam plate attached detachably to
16 said main head body, said pusher is held in a standby position above said
17 component before coming in contact with and pressing down said component
18 toward said substrate, and said standby position is set according to a height of said
19 component held by said head.





1 11. The component mounting apparatus according to claim 10, further
2 comprising a catch pin provided under a via hole of said substrate for supporting a
3 lower end of said lead terminal of said component.

1 12. The component mounting apparatus according to claim 10, wherein
2 setting of said standby position is derived with calculation based on a height of
3 component to be mounted.

1 13. The component mounting apparatus according to claim 12, wherein
2 setting of said standby position is derived by lowering said pusher until it comes in
3 contact with said component held beforehand by said head, and by making
4 calculation according to a magnitude of said lowering movement.

1 14. The component mounting apparatus according to claim 10, wherein
2 said pusher is disposed to be in contact with a point of action of a first lever, a
3 fulcrum of said first lever is supported axially by a second lever, a point of
4 application of said first lever is in contact with a first cam, and a point of
5 application of said second lever is in contact with a second cam.

1 15. The component mounting apparatus according to claim 14, wherein a
2 rotary axle of said first cam is disposed in coaxial with a fulcrum of said second
3 lever.

1 16. A component mounting apparatus comprising:
2 a component delivery unit;



3 a chuck provided on said component delivery unit;
4 a component transfer unit for receiving a component having a lead
5 terminal held by said chuck;
6 a head for receiving said component held by said component transfer unit;
7 a vertically-moving means for said head;
8 a table disposed below said head for supporting a substrate; and
9 an anvil mechanism disposed under said table,
10 said head having a main head body, a vertically-moving means for said
11 main head body, an insertion nail provided at a bottom portion of said main head
12 body, an opening/closing means for said insertion nail, and a turning means for
13 moving said insertion nail between forward and backward directions,
14 wherein said turning means comprises a cam plate attached detachably to
15 said main head body,
16 said anvil mechanism having a mounting base provided with two slant
17 surfaces opposed to each other, a fixed blade mounted on one of said slant surfaces
18 of said mounting base, and a movable blade mounted on another of said slant
19 surfaces of said mounting base,
20 wherein said fixed blade is equipped with a fixing means to said one of
21 slant surfaces of said mounting base after said fixed blade is slid and set in position.

1 17. The component mounting apparatus according to claim 16, further
2 comprising an operating rod disposed under said mounting base, wherein said
3 operating rod links to said movable blade.

1 18. The component mounting apparatus according to claim 17, wherein



2 said operating rod and said movable blade are linked together with a coupling lever
3 supported axially by said mounting base.

1 19. The component mounting apparatus according to claim 17, wherein
2 said operating rod is provided with two flanges on periphery of a top portion thereof
3 at a predetermined space, and one of said flanges at upper side is provided with a
4 cut-back portion.

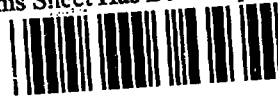
1 20. The component mounting apparatus according to claim 17, wherein
2 said operating rod is formed into a cylindrical shape, and a lead terminal supporter
3 is provided in said cylindrically shaped operating rod in a manner to be in coaxial
4 and vertically movable.

1 21. The component mounting apparatus according to claim 20, wherein
2 said lead terminal supporter comprises a pin disposed in an upper end opening of a
3 cylinder in a manner to protrude upwardly.

1 22. The component mounting apparatus according to claim 21, wherein a
2 supporting body is provided to bridge across the upper end opening of said cylinder,
3 and said pin is mounted on said supporting body.

1 23. The component mounting apparatus according to claim 22, wherein
2 said supporting body has a board-like shape, a supporting pin is inserted
3 horizontally across said board-like supporting body, and said pin is placed with a
4 lower end thereof butting on said supporting pin.





1 24. The component mounting apparatus according to claim 23, wherein an
2 upper peripheral portion of said cylinder is formed with a male thread, a notch is
3 provided from the upper end of said cylinder vertically downward through said
4 male thread, and said supporting pin is engaged in said notch.

1 25. The component mounting apparatus according to claim 24, wherein
2 said supporting pin is engaged in a lower part of said notch, and a projection on said
3 supporting body is engaged in an upper part of said notch.

1 26. The component mounting apparatus according to claim 20, wherein a
2 cap having a female thread on internal periphery and an opening on a top surface is
3 screwed removably over the top end of said cylinder.

1 27. The component mounting apparatus according to claim 16, wherein a
2 groove is formed on at least either one of an upper surface and a lower surface of
3 said movable blade.

1 28. The component mounting apparatus according to claim 27, wherein a
2 groove is formed on an upper surface of said movable blade, and a lower end of
3 said groove is provided with a through hole.

1 29. The component mounting apparatus according to claim 16, wherein
2 said anvil mechanism comprises:

3 a mounting base having two slant surfaces opposed to each other;

4 a fixed blade mounted on one of said slant surfaces of said mounting base;



5 and

6 a movable blade mounted on another of said slant surfaces of said
7 mounting base,

8 wherein said fixed blade is equipped with a fixing means to said one of
9 slant surfaces of said mounting base after said fixed blade is slid and set in position,

10 said movable blade is slid along said another of the slant surfaces up to a
11 position that is above an anticipated point where said movable blade meets with
12 said fixed blade,

13 said fixed blade is then slid up along said one of the slant surfaces until
14 said fixed blade comes in contact to said movable blade, and

15 said fixed blade is fixed on said mounting base in this position.

1 30. A component mounting apparatus comprising:

2 a component delivery unit having an annular shape;

3 a plurality of chucks provided along an outer peripheral surface of said
4 component delivery unit;

5 a plurality of component supply units provided on external side of said
6 plurality of chucks;

7 a chain of taped components supplied by said plurality of component
8 transfer units;

9 a plurality of component transfer units for receiving said components held
10 by said chucks, and

11 a head for receiving said components held by said component transfer
12 units,

13 wherein said chain of taped components consists of components, each



14 having an element and at least two lead terminals connected to said element, taped
15 at regular intervals,

16 said component supply units supply said chain of taped components to
17 said chucks in such a manner that said components are positioned with an element
18 side up and a lead terminal side down, and aligned so that a phantom line extended
19 through two lead terminals of a component held by one of said chucks is
20 approximately in line with another phantom line extended through two lead
21 terminals of another component to be held subsequently by another of said chucks.

1 31. The component mounting apparatus according to claim 30, wherein
2 said plurality of component supply units are disposed side by side.

1 32. The component mounting apparatus according to claim 30, wherein
2 each of said component supply units has a first chuck release for opening said
3 chuck.

1 33. The component mounting apparatus according to claim 32, wherein
2 said first chuck release is actuated by a chuck opening cam surface of a cam
3 provided in said component supply unit for actuating a feeder of said chain of taped
4 components.

1 34. The component mounting apparatus according to claim 33, wherein
2 said component supply unit is provided with a first cutting blade between said
3 feeder and said chuck, and said first cutting blade is actuated by a cutting blade
4 open/close cam surface and a cutting blade shift cam surface of said cam.



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1 35. The component mounting apparatus according to claim 34, wherein
2 said cam is shifted by one or a plurality of driving means for a first amount of
3 movement and a second amount of movement, which is greater than said first
4 amount of movement, and said first amount of movement actuates said first chuck
5 release and said feeder.

1 36. The component mounting apparatus according to claim 35, wherein a
2 shift of said cam for said second amount of movement makes said first cutting blade
3 to close.

1 37. The component mounting apparatus according to claim 36, wherein
2 said component supply unit is provided with an element detector for detecting
3 presence or absence of an element in said chain of taped components, and said
4 driving means shifts said cam for said first amount of movement for a plurality of
5 times, when said element detector detects absence of an element.

1 38. The component mounting apparatus according to claim 33, further
2 comprising a transmission mechanism between said first chuck release and said
3 chuck opening cam surface of said cam, wherein said transmission mechanism
4 actuates said first chuck release only when said cam shifts toward a first direction.

1 39. The component mounting apparatus according to claim 30, wherein
2 said component delivery unit comprises a flexible belt, said belt is formed on a back
3 surface thereof with ditches and ridges for being driven therewith, and said ditches
4 and ridges are used for positioning of said chucks.



* C L W S *

1 40. The component mounting apparatus according to claim 39, wherein
2 said belt is provided with a plurality of chuck retainers at regular intervals, and each
3 of said plurality of chuck retainers is equipped with a chuck.

1 41. The component mounting apparatus according to claim 40, wherein
2 each of said plurality of chuck retainers is formed with a guide piece, and said guide
3 piece engages with a guide rail.

1 42. The component mounting apparatus according to claim 41, wherein at
2 least either one of an upper end and a lower end of each of said plurality of chuck
3 retainers is extended toward a back side of said belt, said guide piece is formed on
4 said extended portion, and said guide piece engages with said guide rail.

1 43. The component mounting apparatus according to claim 42, wherein
2 said chuck is provided on one side thereof adjacent to said component supply unit
3 with a hook for making a linkage to said chuck release.

1 44. The component mounting apparatus according to claim 43, wherein
2 said chuck is extended toward a back side of said belt from an opposite side of said
3 hook, and a second chuck release is linked to said extended portion.

1 45. The component mounting apparatus according to claim 30, wherein
2 said chuck comprises a pair of openable nails, and an elastic body is attached to an
3 inside face of one of said pair of nails..





4 46. The component mounting apparatus according to claim 30, wherein
5 said chuck comprises a fixed nail and a movable nail, which is openable with
6 respect to said fixed nail, and an elastic body is attached to an inside face of said
7 movable nail.

8 47. The component mounting apparatus according to claim 46, wherein
9 said elastic body is attached to nail with a pin penetrating through said elastic body.

1 48. The component mounting apparatus according to claim 30, further
2 comprising an extra-length detector positioned on said component delivery unit at a
3 downstream side of a component supply spot by said component supply unit, for
4 detecting presence or absence of an extra length of a taping member, wherein said
5 extra-length detector actuates said second cutting blade to cut said extra length of
6 taping member into a predetermined length.

1 49. The component mounting apparatus according to claim 48, wherein
2 said second cutting blade cuts said extra length of taping member after said second
3 cutting blade is lifted to an upper position from a lower position, and said second
4 cutting blade returns to said lower position after the cutting.

1 50. The component mounting apparatus according to claim 30, further
2 comprising a polarity flipper unit for components, provided on said component
3 delivery unit at a downstream side of the component supply spot by said component
4 supply unit, said polarity flipper unit being freely movable toward and away with
5 respect to said component delivery unit.



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1 51. The component mounting apparatus according to claim 50, wherein
2 said polarity flipper unit is positioned at a downstream side of said second cutting
3 blade.

1 52. The component mounting apparatus according to claim 30, further
2 comprising a hold position correction unit for a lead terminal of a component held
3 by said chuck, provided on said component delivery unit at a downstream side of
4 the component supply spot by said component supply unit.

1 53. The component mounting apparatus according to claim 52, wherein
2 said hold position correction unit is provided at a downstream side of said polarity
3 flipper unit.

1 54. The component mounting apparatus according to claim 53, wherein
2 said hold position correction unit comprises:

3 a positioning base for setting thereon a bottom edge of said taping
4 member;

5 a holder for holding said taping member by two sides orthogonal to a
6 longitudinal direction thereof; and

7 a push body for shifting a component by pushing a lead terminal of said
8 component toward the longitudinal direction of said taping member.

1 55. The component mounting apparatus according to claim 52, further
2 comprising a third cutting blade provided on said component delivery unit at a
3 downstream side of said hold position correction unit, for cutting said lead terminal,



4 said third cutting blade being freely movable toward and away with respect to said
5 component delivery unit.

1 56. The component mounting apparatus according to claim 55, wherein
2 said third cutting blade comprises a pair of openable blades, lower surfaces at distal
3 end portion of said pair of blades are tapered, and an upper edge of said tapping
4 member comes in contact with said tapered surfaces.

1 57. The component mounting apparatus according to claim 30, having a
2 component transfer unit for transferring said component held by said chucks to said
3 head, wherein said component transfer unit comprises two grip nails for holding a
4 lower portion of said lead terminal, and a support nail for supporting an upper
5 portion of said lead terminal.

1 58. The component mounting apparatus according to claim 57, wherein
2 one of said grip nails and said support nail are integrated, and another of said grip
3 nails is arranged to be freely movable with respect to said one of grip nails
4 integrated with said support nail.

1 59. The component mounting apparatus according to claim 58, wherein
2 said insertion nail on said head grips a portion of said lead terminal located between
3 said grip nails and said support nail.

1 60. A component mounting apparatus comprising:
2 a component delivery unit;



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3 a chuck provided on said component delivery unit;
4 a component transfer unit for receiving a component held by said chuck;
5 and

6 a head for receiving said component held by said component transfer unit,
7 said head having a main head body, a vertically moving means for said
8 main head body, an insertion nail provided at a bottom portion of said main head
9 body, an opening/closing means for said insertion nail, and a turning means for
10 moving said insertion nail between forward and backward directions,

11 wherein said turning means comprises a cam plate attached detachably to
12 said main head body,

13 said component transfer unit having a transfer chuck for receiving said
14 component held by said chuck, a turning means for rotatory moving said transfer
15 chuck, a shifting means for shifting said transfer chuck toward inside and outside of
16 a rotational arc of said transfer chuck, and an opening/closing means for opening or
17 closing said transfer chuck after said transfer chuck is shifted to inside or outside of
18 the rotational arc thereof by said shifting means,

19 wherein a rotary axle of said turning means, a driving axle of said shifting
20 means and an opening/closing axle of said opening/closing means are coaxially
21 disposed.

1 61. The component mounting apparatus according to claim 60, further
2 comprising a special opening/closing means for opening or closing said transfer
3 chuck, in addition to said ordinary opening/closing means operated by said
4 opening/closing axle.



1 62. The component mounting apparatus according to claim 60, wherein
2 said transfer chuck comprises two grip nails for holding a lower portion of said lead
3 terminal, and a support nail for supporting an upper portion of said lead terminal.

1 63. The component mounting apparatus according to claim 62, wherein
2 one of said grip nails and said support nail are integrated, and another of said grip
3 nails is arranged to be freely movable with respect to said one of grip nails
4 integrated with said support nail.

1 64. The component mounting apparatus according to claim 60, further
2 comprising a rotary plate to be rotated by said opening/closing axle, and an
3 opening/closing mechanism for opening and closing said transfer chuck by a rotary
4 movement of said rotary plate, both of said rotary plate and said opening/closing
5 mechanism disposed between said opening/closing axle and said transfer chuck,
6 wherein said opening/closing axle biases said rotary plate for said rotary movement.

1 65. The component mounting apparatus according to claim 60, further
2 comprising a shift lever laid between said driving axle of said shifting means and
3 said transfer chuck, said shift lever being biased by said driving axle for shifting
4 said transfer chuck toward inside and outside of the rotational arc thereof.

1 66. A component mounting apparatus comprising:
2 a component delivery unit;
3 a chuck provided on said component delivery unit;
4 a component transfer unit for receiving a component held by said chuck;



5 a head for receiving said component held by said component transfer unit,
6 and for mounting said component on a substrate positioned on a positioning table;

7 a rail for guiding said substrate on said positioning table; and

8 a transfer means for transferring said substrate to said positioning table
9 while being guided by said rail,

10 said head having a main head body, a vertically-moving means for said
11 main head body, an insertion nail provided at a bottom portion of said main head
12 body, an opening/closing means for said insertion nail, and a turning means for
13 moving said insertion nail between forward and backward directions,

14 wherein said turning means comprises a cam plate attached detachably to
15 said main head body,

16 said transfer means comprising a shaft holder, a transfer shaft provided
17 freely in and out from said shaft holder toward said positioning table, a transfer pin
18 provided at one end of said transfer shaft nearer to said positioning table in a
19 manner to direct toward said substrate, a biasing means for biasing said transfer pin
20 toward said substrate, and a pin shifting unit provided at the end of said shaft holder
21 nearer to said positioning table for shifting said transfer pin to a side opposite to
22 said substrate against a biasing force of said biasing means.

1 67. The component mounting apparatus according to claim 66, wherein
2 said transfer pin is disposed slidably on an axle, which is provided at one end of
3 said transfer shaft nearer to said positioning table in a manner to direct toward said
4 substrate, a spring defining a biasing means is linked to said transfer pin, a pin
5 shifting unit comprises a lever provided at one end of said shaft holder nearer to
6 said positioning table, and a lever actuator linked to one end of said lever, and the



7 other end of said lever is linked to said transfer pin.

