

COUPLER

TECHNICAL FIELD

This present invention relates to couplers of the type having a first rotor on one shaft spaced by an air gap from a second rotor on another shaft. The second rotor may have a ferrous-backed electroconductive element and may be located opposite magnets presented by the first rotor. More particularly, the invention relates to adjustment of the air gap.

BACKGROUND TO THE INVENTION

Induction motors are used, for example, to drive fans, blowers, pumps and compressors. It has been recognized that when these motors are operated at full speed they normally have excess capacity as compared to the load requirements, and this excess capacity is compounded when the load is variable. It has also been recognized that if the output of the motors could be adjusted to provide only the needed power, a significant reduction of energy usage would result. Hence, variable speed drives (VSD's) have been developed in the form of electronic devices which match motor speed to that required for a given application. A typical VSD rectifies incoming AC voltage and current into DC, then inverts the DC back to AC at a different voltage and frequency. The output voltage and frequency is determined by the actual power needs and is set automatically by a control system or by an operator.

Heretofore, VSD's have generally been so expensive that they have not been used extensively for energy savings. It has been reported that VSD's require the availability of highly trained maintenance personnel and shorten motor life.

SUMMARY OF THE INVENTION

The present invention aims to provide a mechanical alternative to VSD's which is far more economical, will automatically maintain the speed of the load to a preset speed as the load requirements vary, and will not require modification of the

electric motor or adjustment of the input voltage or frequency. A further object is to provide a coupling which will function in place of VSD's without overheating.

In my prior Patent 5,477,094 there is shown a magnetic coupler in which a magnet rotor unit is straddled by two conductor rotors which are connected together to rotate as a conductor rotor unit on one shaft while the magnet rotor unit is mounted to rotate on a second shaft. The magnet rotor unit has a set of permanent magnets arranged with their opposite poles spaced by air gaps from ferrous-backed electroconductive rings mounted on respective of the conductor rotors. Rotation of one of the two shafts results in rotation of the other shaft by magnetic action without there being any direct mechanical connection between the shafts.

My prior patent also discloses the concept of having two magnet rotors rather than a single magnet rotor unit, with each magnet rotor having a respective set of permanent magnets spaced by an air gap from one of the electroconductive elements presented by the conductor rotors. The two magnet rotors are axially moveable relative to one another and are spring biased apart.

The present invention therefore provides coupler having a shaft having a first rotary axis; a central rotor mounted on the shaft and having slide pins projecting oppositely from the rotor in parallel spaced relation to the axis; and two outer rotors slide-mounted on the slide pins at opposite sides of the central rotor. The swing units are swing-mounted on the central rotor and each has opposite end portions pivotally connected by slide connections to respective of the outer rotors so that axial movement of one of the outer rotors in a selected direction results in an endwise movement of the other one of the outer rotors in a direction opposite to the selected direction.

The outer rotors can each contain a permanent magnet with a pole facing outwardly away from the other said rotor, and the magnet may be backed by a ferrous member. The outer rotors may each contain a set of magnets each having an outwardly facing pole, and the magnets may be backed by a ferrous member. Each slide connection between the opposite end portions of the swing units and the outer rotors has a slot and a roller working in that slot. The swing units maybe swing-mounted on the central rotor at a longitudinal center of the swing units. The central

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rotor may have radially projecting ears at a periphery of the central rotor. The swing units may be swing-mounted at outer ends of the ears on radially extending bolts threaded into the ears. Preferably, the outer rotors have block elements projecting toward the central rotor. Also the swing units may have a retracted position in which the block elements nest with the central rotor adjacent to opposite side edges of the ears, the block elements containing part of the slide connections. The outer rotors may have block elements projecting toward the central rotor and containing part of the slide connections. The swing units may have a retracted position in which the swing units and the block elements are coplanar with the central rotor, and have an extended position at which the swing units extend at acute angles with a plane of the central rotor. The outer rotors may be magnet rotors containing permanent magnets, and in which the magnet rotors are spaced by air gaps from conductor rotors containing electroconductive elements, the conductor rotors being connected together and mounted on a second shaft having a second rotary axis aligned with the first rotary axis.

BRIEF DESCRIPTION OF DRAWINGS

Figure 1 is a longitudinal sectional view of a first embodiment of the invention shown in a wide air gap position and taken as indicated by line 1-1 in Figure 5;

Figure 2 is a perspective view of the first embodiment without the conductor rotors and showing the air gap adjusting mechanism extended so that the rotors are in a narrow air gap position;

Figure 3 is a plan view corresponding to Figure 2;

Figure 4 is a plan view like Figure 3, but with the air gap adjustment mechanism retracted so that the rotors are in a wide air gap position;

Figure 5 is a transverse sectional view taken as indicated by line 4-4 in Figure 4;

Figure 6 is an end view of the left hand rotor as viewed looking to the right in Figure 1 and with the magnets removed;

Figure 7 is a perspective view showing the barrel cam mechanism and related fork;

Figure 8 is a longitudinal sectional view of a second embodiment shown in a wide air gap position; and

Figure 9 is a perspective view of one of the fan rings.

DETAIL DESCRIPTION OF THE INVENTION

Referring to the drawings, coaxial input and output shafts 20-21 have mounted thereon a conductor rotor unit 22 and a pair of rotors 24-25 which may be magnetic. The conductor rotor unit has two axially spaced conductor rotors 26-27 having respective conductor rings 28-29 facing toward one another and formed from a non-ferrous material with high electrical conductivity such as copper. These conductor rings 28-29 are mounted as by screws on respective backing rings 32-33 which preferably are mild steel. The conductor rotor unit 22 also includes a rotor disc 34 mounted by bolts 35 on a hub 36 and spaced axially from the conductor rotor 28 by a ventilation gap 37. The backing rings 32-33 are connected together and to the disc 34 in axially spaced relation by sets of bolts 38-38' threaded into spacer sleeves 39-39' located outwardly of the orbits of the magnet rotors 24-25. Conductor rotor 27 is separated from the output shaft 21 by an annular clearance space 40. The hub 36 is mounted on input shaft 20 as by a wedge-type coupling or a key connection.

Each of the rotors 24-25 has a non-ferrous mounting disc 42 backed by a ferrous backing disc 43, preferably of mild steel. The mounting discs 42 may be aluminum or a suitable non-magnetic composite, and each is formed with a set of equally spaced rectangular cutouts 44 arranged in a circle and receiving a respective set of permanent magnets 46 seated against the respective backing disc 43. Adjacent magnets have their polarities reversed. The magnets 46 are spaced by air gaps 48-48' from the conductor rings 28-29 of the conductor rotor unit 22.

Preferably the disc 34 is formed with ventilation holes 47 to assist in the circulation of air through the ventilation gap 37 and the air gap 48 for cooling the conductor ring 29. Cooling air for the conductor ring 28 is free to enter the air gap 48 from the clearance space 40. The conductor rotors may also be provided with a

screw-mounted fan ring 49 (Figure 9) presenting multiple blade elements 49a to increase the air flow adjacent the conductor rings 27, 28 for cooling. It is to be understood that providing added ventilation for cooling of the conductor ring 28 by way of the ventilation space 37 and/or ventilation holes 47 or fan blades 49a may not be required for all applications in which case the backing ring 32 would be mounted on the disc 34 or the conductor ring 28 could be mounted directly on the disc 34 which thereupon could serve as the ferrous backing for the conductor ring 28 rather than the backing ring 32.

In accordance with the present invention the rotors 24-25 are mounted so as to rotate in unison with the output shaft 21 and also be axially moveable relative to one another in opposite axial directions for adjustment of the air gaps 48-48'. To this end the rotors 24-25 are preferably slide-mounted by bushings 50 on opposite axial end portions of combination support and guide pins 51. These pins project in opposite axial directions from a fifth rotor 52 which is mounted on the output shaft 21 midway between the conductor rings 28-29. As an alternative arrangement, the rotors 24-25 could be slide-mounted on the output shaft 21 rather than on the pins 51.

Push-pull means is provided to move the rotors 24-25 axially in unison along the rotary axis of the splined output shaft 21 in opposite directions to vary the width of the air gaps 48-48'. The push-pull means may comprise a first push-pull mechanism extending through the opening 40 for axially moving the rotor 25, and a second push-pull mechanism extending between the magnet rotors for moving the rotor 24 responsive to movement of the rotor 25 by the first mechanism. In the illustrated embodiment the second mechanism includes the fifth rotor 52 and related pins 51.

The fifth rotor 52 can be generally square-shaped in elevation providing four outer edge faces 52a, each of which has a central ear 53 projecting radially therefrom. These ears 53 are formed with threaded radial bores extending toward the shaft 21 from their outer ends to receive shoulder bolts 54 on which bearings 55 are sleeved. The bearings 55 receive center hub portions of swing units 56 each having a pair of swing arms with cam slots 57 formed adjacent their outer ends. These cam slots each

receive a cam follower roller 58 to track therein. Each roller 58 projects outwardly from a mounting stud 59 which is secured in a respective block 60 projecting toward the fifth rotor 52 from the mounting disc 42 of the respective magnet rotor. The blocks 60 may be mounted on the discs 42 by a pair of cap screws 60a. When the rotors 24-25 are retracted the maximum axial distance from the conductor rotors 26-27 as shown in Figure 1, a respective pair of the blocks 61 extends on opposite sides of each of the ears 53 of the fifth rotor 52 so that the swing units 56 will then be coplanar with the fifth rotor 52 as shown in Figures 4-5. This compact arrangement assists in minimizing the length of the coupler.

With the described slotted rocker arm and follower roller arrangement it is apparent that when the rotor 25 is pushed away from the conductor rotor 27 to increase the width of the air gap 48', the swing units 56 will responsively pivot on the center bolts 54 so that their ends will swing toward the fifth rotor 52. During this swinging movement the rollers 58 track in the slots 57 toward their inner end and as a result the rotor 24 is pulled toward the fifth rotor 52, thereby increasing the width of the air gap 48 to the same extent as the width of the air gap 48' is increased by the push on the rotor 25. Likewise, when the rotor 25 is pulled toward the conductor rotor 27 the width of the air gap 48', the swing units 56 will responsively swing on the bolts 54 so that their ends will swing away from the fifth rotor 52, thereby causing the rotor 25 to be pushed toward the conductor rotor 26 and narrow the air gap 48 in correspondence with the narrowing of the air gap 48'.

Pushing and pulling of the rotor 25 to vary the width of the air gaps 48-48' is preferably accomplished by using a barrel cam 61 which has an inner barrel element 62 overlapped by an outer barrel element 63. The inner element 62 is mounted by a bearing unit 64 on the output shaft 21 and the outer element 63 has a neck portion 63a which has clearance with the output shaft 21 and carries a thrust bearing 65 which has its outer race seated in the inner radial end of rotor 25. A bearing cap 66 secured by screws 67 to the disc of the magnet rotor 25 holds the thrust bearing 65 and a seal 68 in position. The inner barrel 62 has a set of cam rollers 70 which project radially outward into curved cam slots 71 in the outer barrel 63. Turning of the outer barrel 63 is prevented by a yoke 72 (Figure 7) having its arms 72a pivotally

connected adjacent their outer ends by rollers extending into holes 73 in the outer barrel from studs 74 mounted in the yoke arms. The yolk 72 has a pair of bottom legs 72b formed with oversized holes 75 receiving cam rollers 76 mounted on studs projecting outwardly from a stationary mounting block 77.

An actuator arm 78 projects outwardly from the inner barrel 62 and is turned in any suitable manner to control the air gaps 48, 48'. Turning of the inner barrel 62 by action of the actuator in one direction causes endwise movement of the outer barrel 63 responsive to movement of the cam rollers 70 in the cam slots 71 which are contoured to give this result. The holes 75 in the yoke legs 72b are sufficiently oversized relative to the rollers 76 to permit the required endwise movement of the outer barrel 63 as the yoke 72 swings responsive to such movement.

Endwise movement of the outer barrel 63 acts through the thrust bearing 65 to correspondingly push or pull the magnet rotor 25. As before described, this results in equal endwise motion of the other rotor 24 in the opposite direction by responsive operation of the rocker arm units 57 and follower rollers 59. Thus, selective movement of the actuator arm 78 results in varying the air gaps 48, 48', and thereby varies the output speed of the magnetic coupler. The actuator arm 78, may, for example, be connected by a link 78a to a stationary electric rotary positioner which is controlled by a process controller. If, for example, the load is a pump whose flow output is to be controlled, a measuring device in the output stream feeds the output data to the process controller which then signals the rotary positioner for the required rotary movement of the actuator arm 78 to properly adjust the output speed of the magnet coupler.

Preferably, the output shaft 21, rather than being the actual input shaft of the load, is an add-on shaft section as shown in Figure 1. This add-on section 21 is connected at a necked end portion 21a to the fifth rotor 52 via a round end plate 80 which covers the inner end face of the add-on section 21 and a hub portion 52a of the fifth rotor 52. Sets 82, 83 of bolts connect the end plate 80 to the shaft 21 and fifth rotor hub 52a.

The shaft 21 expands from the necked portion 21a to an intermediate cylindrical portion receiving the bearing 64, and then is formed with an annular

shoulder 21c against which the outer end of the inner race of the bearing 64 is seated. Endwise of the shoulder 21c the output shaft 21 has an outer cylindrical terminal portion 21d receiving a bearing seal 84 and a hub component 86a of coupler 86. The coupler has a complementing adapter hub component 86b with a neck 86c sized to receive the actual input shaft 21' of the load. A wedge-type squeeze unit 87 is sleeved on the coupler neck 86c to force fit the coupler 86 to the shaft 21' responsive to tightening of screws 89. The hub components 86a, 86b of the coupler 86 are secured together by bolts 88, and the coupler is fixed to the shaft section 21 by an annular end plate 90 secured by sets 91, 92 of bolts to the outer end face of the shaft section and to the hub component 86a. A squeeze unit 87 can also be used in conjunction with the hub 36 to secure it to the shaft 20.

The described arrangement incorporating the shaft section 21 and coupler 86 makes it possible to easily install or remove the magnetic coupling of the present invention without moving the load and its related input shaft 21 or the prime mover and its shafter 20.

For some applications of the invention, there is a need to provide the required torque transfer from the input shaft 20 to the output shaft 21 using rotors with a smaller diameter than possible with single pairs of magnet rotors and conductor rotors. As shown in Figure 8, this need may be satisfied by providing a second pair of magnet rotors on the output shaft, extending the conductor rotor unit to present an additional pair of conductor rotors, and connecting one of the magnet rotors in one of the pairs with the corresponding magnet rotor of the other pair of magnet rotors by a push-pull rod passing freely through the fifth rotor and conductor rotor which are located between the two magnet rotors coupled together by the rod.

In the Figure 8 embodiment, corresponding parts to the first described embodiment have been given the same reference numerals. The shaft section 21 has been elongated and is designated 121. The two pairs of rotors have been marked 124-125 and 224-225, and the corresponding fifth rotors have been marked 152 and 252. Spaced by air gaps from the rotors 224 and 225 are conductor rotors 226-227. Conductor rotors 126 and 227 have in common a mild steel ring 232 functioning as a backing for conductor ring element 128 of conductor rotor 126 and also for

conductor ring element 229 of conductor rotor 227. Conductor rings 129 and 228 are backed by ferrous rings 133 and 134, respectively. The latter is connected to hub 136 mounted on shaft 20. The four conductor rotors are held in properly aligned relation by an array of bolts 138 passing through tubular spacers 139-139'.

Two fifth rotors 152 and 252 are fixed on the shaft 121 so as to be midway between conductor elements 129, 132 and 228, 229. They have the same general configuration as fifth rotor 52 and each has a set of four guide rods 51 supporting the respective pair of rotors 124-125 and 224-225. In addition, the fifth wheel 152 has four clearance openings 153 spaced midway between its guide rods 51 for free passage of push-pull rods 300. These rods also pass freely through openings 153' in the rotor 124. At their inner end the push-pull rods are threaded into the rotor 225 and at their outer end they pass through the rotor 125 and are held in fixed relation thereto by a pair of snap rings 301.

It will apparent that endwise movement of the rotor 125 will be duplicated by the rotor 225 by way of the push-pull rods 300. This endwise movement is duplicated in the reverse direction by rotors 124 and 224 by way of the action of the swing units 56 and related parts as previously described. Although not preferred, the rotors 124 and 224 can be coupled together by push-pull rods rather than the rotors 125 and 225 being coupled together.

From the foregoing it will be appreciated that, although specific embodiments of the invention have been described herein for purposes of illustration, various modifications may be made without deviating from the spirit and scope of the invention. Accordingly, the invention is not limited except as by the appended claims.

CLAIMS:

1. A coupler comprising:
a shaft having a first rotary axis;
a central rotor mounted on said shaft and having slide pins projecting oppositely from said rotor in parallel spaced relation to said axis;
two outer rotors slide-mounted on said slide pins at opposite sides of said central rotor; and swing units swing-mounted on said central rotor and each having opposite end portions pivotally connected by slide connections to respective of said outer rotors so that axial movement of one of said outer rotors in a selected direction results in an endwise movement of the other one of said outer rotors in a direction opposite to said selected direction.
2. A coupler according to claim 1, in which said outer rotors each contain a permanent magnet with a pole facing outwardly away from the other said rotor.
3. A coupler according to claim 2, in which said magnet is backed by a ferrous member.
4. A coupler according to claim 1, in which said outer rotors each contain a set of magnets each having an outwardly facing pole.
5. A coupler according to claim 4, in which said magnets are backed by a ferrous member.
6. A coupler according to any one of claims 1 to 5, in which each said slide connections between said opposite end portions of said swing units and said outer rotors comprises a slot and a roller working in said slot.

7. A coupler according to any one of claims 1 to 6, in which said swing units are swing-mounted on said central rotor at a longitudinal center of said swing units.
8. A coupler according to any one of claims 1 to 7, in which said central rotor has radially projecting ears at a periphery of said central rotor, and said swing units are swing-mounted at outer ends of said ears on radially extending bolts threaded into said ears.
9. A coupler according to claim 8, in which said outer rotors have block elements projecting toward said central rotor, said swing units having a retracted position in which said block elements nest with said central rotor adjacent to opposite side edges of said ears, said block elements containing part of said slide connections.
10. A coupler according to any one of claims 1 to 8, in which said outer rotors have block elements projecting toward said central rotor and containing part of said slide connections.
11. A coupler according to claim 10, in which said swing units have a retracted position in which said swing units and said block elements are coplanar with said central rotor, and have an extended position at which said swing units extend at acute angles with a plane of said central rotor.
12. A coupler according to claim 1, in which said outer rotors are magnet rotors containing permanent magnets, and in which said magnet rotors are spaced by air gaps from conductor rotors containing electroconductive elements, said conductor rotors being connected together and mounted on a second shaft having a second rotary axis aligned with said first rotary axis.

ABSTRACT

COUPLER

A coupler comprising a shaft having a first rotary axis; a central rotor mounted on the shaft and having slide pins projecting oppositely from the rotor in parallel spaced relation to the axis; and two outer rotors slide-mounted on the slide pins at opposite sides of the central rotor. The swing units are swing-mounted on the central rotor and each has opposite end portions pivotally connected by slide connections to respective of the outer rotors so that axial movement of one of the outer rotors in a selected direction results in an endwise movement of the other one of the outer rotors in a direction opposite to the selected direction.

Fig. 1