

SPECIFICATION **Method of an Apparatus for Attaching Top** **Stops to a Sliding Clasp Fastener Chain**

Description

5 This invention relates to the production or
assemblage of sliding clasp fasteners, and in
particular to the attachment to a fastener chain
(i.e., a continuous length of a pair of coupled
stringers) of top stops which arrest the movement
10 of a slider in its uppermost or closed position in a
completed sliding clasp fastener.

A variety of methods and apparatus have been
suggested and employed for the application of top
stops to the endmost scoops of the successive
15 spaced pairs of opposed scoop groups, or to the
ends of the blank tape spaces adjoining the
endmost scoops, of a fastener chain. According to
the majority of such known methods and
apparatus, the fastener chain is fed along a
20 predetermined path, and its movement is
periodically arrested by some stop means
movable into engagement therewith, for bringing
the successive endmost scoops or the adjoining
ends of the blank tape spaces to a location where
25 top stops are applied. This conventional practice
is objectionable because the positions of the
endmost scoops or of the adjoining tape space
ends for the application of top stops are
determined by the movable stop means and,
30 consequently, because the top stops are not
necessarily attached to the exact desired points
on the fastener chain.

According to one aspect of the invention, there
is provided a method of attaching top stops to a
35 sliding clasp fastener chain including a pair of
continuous stringers, each stringer having
successive spaced groups of scoops mounted on
a continuous stringer tape and interengaged with
opposed complementary groups of scoops on the
40 other stringer tape, which method comprises:
moving the fastener chain longitudinally along a
predetermined path; causing a displaceable
actuating pawl to be engaged by one of the
successive spaced pairs of scoop groups of the
45 fastener chain so as to be displaced therewith
along the path; causing a separator to spread one
of the successive pairs of opposed blank tape
spaces of the fastener chain in response to a
signal generated upon displacement of the
50 actuating pawl over a predetermined distance
with the fastener chain, the pair of blank tape
spaces of the fastener chain in response to
displacement of the actuating pawl over a
predetermined distance with the fastener chain,
55 the pair of blank tape spaces being spread to such
an extent that at least one adjacent pair of scoop
groups are partly disengaged from each other;
subsequently moving the fastener chain along the
path until the disengaged endmost scoops of the
60 partly disengaged pair of scoop groups come into
abutment against stop means disposed adjacent
to a die, whereby the disengaged end portions of
the partly disengaged pair of scoop groups are
positioned upon the die; attaching top stops to

65 the disengaged end portions of the partly
disengaged pair of scoop groups by punch means
coating with the die; and repeating the foregoing
steps to apply top stops to the end portions of the
successive spaced pairs of scoop groups.

70 According to another aspect of the invention,
there is provided an apparatus for attaching top
stops to a sliding clasp fastener chain including a
pair of continuous stringers, each stringer having
successive spaced groups of scoops mounted on
75 a continuous stringer tape and interengaged with
opposed complementary groups of scoops on the
other stringer tape, the apparatus comprising:
means for feeding the fastener chain along a
predetermined path; an actuating pawl disposed
80 on the median line of the fastener chain for
movement both along the path and in a plane
normal to the plane of the fastener chain; means
biasing the actuating pawl toward the fastener
chain whereby the actuating pawl is capable of
85 passing between each of the successive pairs of
opposed blank tape spaces of the fastener chain
so as to be engaged by the succeeding pair of
interengaged scoop groups; switch means
adapted to be actuated when the actuating pawl
90 is displaced a predetermined distance along the
path in engagement with the pair of interengaged
scoop groups; separator means responsive to the
actuation of the switch means for spreading one
of the pairs of opposed blank tape spaces to such
95 an extent that at least one adjacent pair of scoop
groups are partly disengaged from each other; a
die disposed substantially on the path of the
fastener chain; stop means immovably disposed
adjacent to the die, the stop means being adapted
100 to engage and stop the disengaged endmost
scoops of the partly disengaged pair of scoop
groups as the fastener chain is further moved
along the path after the spreading of the pair of
blank tape spaces, whereby the disengaged end
105 portions of the partly disengaged pair of scoop
groups are positioned upon the die; and punch
means coating with the die to attach top stops to
the disengaged end portions of the partly
disengaged pair of scoop groups.

110 The invention will now be described by way of
example, with reference to the accompanying
drawings, wherein:—

Figure 1 is a fragmentary plan view of a sliding
clasp fastener chain to which top stops are to be
115 attached in accordance with this invention;

Figure 2 is a plan view of explanatory nature
showing the fastener chain of Figure 1 with its
pair of constituent stringers partly separated for
the attachment of top stops in accordance with
120 the invention;

Figure 3A is a plan view of a strip of material to
be severed into the blanks of top stops suitable
for use in the practice of the invention;

Figure 3B is a plan view of one of the top stop
125 blanks severed from the strip of Figure 3A;

Figure 3C is a side elevational view of the top
stop formed from the blank of Figure 3B;

Figure 4 is a plan view schematically

illustrating the fastener chain of Figure 1 with the pair of top stops attached thereto;

Figure 5 is an enlarged sectional view taken along the line V—V of Figure 4;

Figure 6 is a fragmentary plan view showing the fastener chain of Figure 1 together with the pair of top stops attached thereto;

Figure 7 is a side elevational view, partly sectioned for clarity, of a preferred form of the apparatus in accordance with the invention;

Figure 8 is an enlarged elevational view, partly sectioned for clarity, of the separator mechanism in the apparatus of Figure 7, the separator mechanism being shown as seen in the direction of the arrows VIII in Figure 7;

Figure 9 is an enlarged top plan view of the stop, die, and part of the separator mechanism in the apparatus of Figure 7, wherein the chain-dot line represents the fastener stringers being separated;

Figure 10 is a partial elevational view, partly in section, showing the separator mechanism in the raised position, the view also showing the die together with the scoops received in its cavities for the application of top stops;

Figure 11 is a view similar to Figure 10 except that the separator mechanism is shown in the lowered position;

Figure 12 is a sectional view taken along the line XII—XII of Figure 9 and showing the stop together with the fastener chain thereon;

Figure 13 is a view similar to Figure 4 except that the pair of top stops are shown attached to the leading ends, instead of to the trailing ends, of each pair of opposed groups of scoops;

Figure 14 is a side elevational view, partly sectioned for clarity, of another preferred form of the apparatus in accordance with the invention, the apparatus being shown adapted for attaching top stops to the leading ends of each pair of opposed scoop groups of the fastener chain as in Figure 13; and

Figure 15 is an enlarged top plan view of the stop, die, and part of the separator mechanism in the apparatus of Figure 14, wherein the chain-dot line represents the fastener stringers being separated.

With reference first and in particular to Figure 1, a sliding clasp fastener chain 20 to which top stops are to be applied is in the form of a pair of coupled continuous stringers 21. Each stringer 21 includes a continuous tape 22 on which there are mounted successive spaced groups of scoops or fastener elements 23 along one longitudinal edge in mating interengagement with the opposed complementary groups of scoops on the other stringer tape. Blank tape spaces 24 are left between the interengaged groups of scoops 23. Purely by way of example each group of scoops 23 are shown to be formed by the undulations or turns of a continuous coupling element of the known helicoidal coil type. These continuous coupling elements are stitched to the stringer tapes 22, with the aid of continuous core cords 25 passing through the coupling elements.

In this specification and in the appended claims the terms "preceding" and "succeeding", and "leading" and "trailing", are used with reference to a predetermined direction in which the fastener chain 20 is fed for the application of top stops thereto. Thus, if the fastener chain 20 is to be fed in the direction indicated by the arrow in Figure 1, for example, then the pair of interengaged groups of scoops designated 23 in this figure precedes the pair of opposed blank tape spaces 24. The scoops at the trailing end of this pair of scoop groups are designated 23'.

According to one aspect of the invention, each pair of opposed blank tape spaces 24 of the fastener chain 20 are spread by the separable halves 26 of a separator finger 27, as shown in Figure 2, thereby partly disengaging the preceding pair of scoop groups 23. Top stops are attached to the scoops 23' at the trailing end of the partly disengaged pair of scoop groups.

Figures 3A, 3B and 3C are illustrations of an example of top stops suitable for use with the invention. The illustrated top stops are formed by cutting an elongated strip 28 of suitable material such as metal into blank 29. Each top stop blank 29 has a pair of laterally spaced, pointed legs 30 at one end and a blunt-ended leg 31 at the other end. For use as a top stop 32, the blank 29 is bent and clinched as shown in Figure 3C.

In Figures 4, 5 and 6 are shown the pair of top stops 32 attached to the trailing ends of the partly disengaged pair of scoop groups 23 in accordance with the preferred method and apparatus of this invention. Although each top stop 32 is shown to embrace only the endmost scoop 23' of one of the scoop groups, the top stop may be adapted to embrace two or more consecutive scoops within the scope of the invention. Further, if desired, the top stops may be applied to the leading end portions 33 of the blank tape spaces 24.

The preferred apparatus of this invention will hereinafter be described in detail with particular reference to Figure 7. The preferred method of the invention will be set forth after the description of the apparatus.

The apparatus of Figure 7 broadly comprises a pair of feed rolls 34 and 35 coaxing with a pair of squeeze rolls 36 and 37 to define a horizontal path along which the fastener chain 20 is fed with its groups or scoops 23 directed downwardly, a displaceable actuating pawl 38 disposed on the median line 39 (Figure 1) of the fastener chain traveling along the path, a limit switch 40 to be actuated upon displacement of the actuating pawl, a separator mechanism 41 including the separator finger 27 which responds to the actuation of the limit switch to spread a pair of blank tape spaces 24 of the fastener chain to such an extent that the preceding pair of scoop groups 23 are partly disengaged from each other, a die 42 for receiving the disengaged endmost scoops of the partly disengaged pair of scoop groups, and a punch mechanism 43 movable up and down to

apply the top stops 32 to the disengaged endmost scoops on the die.

Like the above defined terms "preceding" and "succeeding", and "leading" and "trailing", the terms "forward" and "rearward" are also used herein and in the claims appended hereto with reference to the predetermined direction (rightward as viewed in Figure 7) in which the fastener chain 20 is fed along the predetermined path. Thus, in Figure 7, the actuating pawl 38 is disposed rearwardly of the die 42, and the separator mechanism 41 is disposed forwardly of the die.

Of the pair of feed rolls 34 and 35, the roll 34 is a drive roll connected to a suitable drive mechanism (not shown). Further, in this particular embodiment of the invention, the squeeze roll 36 is intended to serve the additional purpose of moving the fastener chain 20 a limited distance in the reverse direction. More will be said later about this reverse rotation of the squeeze roll 36.

A guide block 44 disposed forwardly of the pair of squeeze rolls 36 and 37 defines therein a guideway 45 for the fastener chain 20 so that the latter may travel along the predetermined path without displacement. Further disposed forwardly of the guide block 44 is the actuating pawl 38 which lies on the median line 39 of the fastener chain 20 emerging from the guide block 44. The actuating pawl 38 is formed at and angled upwardly from the right hand end, as viewed in Figure 7, of a first arm 46 extending horizontally under the guide block 44. This first arm 46 together with the actuating pawl 38 is supported by the following means so as to be both movable along the fastener chain path and pivotable about its left hand end in a vertical plane.

The right hand end of the first arm 46 rests by gravity on a horizontally disposed arm support 47 via a roller 48 rotatably mounted on the first arm. The first arm 46 can therefore roll over the arm support 47 for its movement along the fastener chain path. Slidably supported by the frame 49 of the apparatus for up and down motion relative to same, a plunger 50 is biased upwardly by a helical compression spring 51 into abutting engagement with the arm support 47. This arm support is formed at the top end of an upstanding bar 52 which is also slidably supported by the frame 50 for up and down motion relative to same. The bar 52 is connected to an actuating mechanism (not shown) of any known or suitable design capable of moving the bar, and therefore the arm support 47, downwardly against the bias of the compression spring 51.

Normally, this unshown actuating mechanism is held inoperative so that the right hand end of the first arm 46 is urged upwardly under the bias of the compression spring 51. The actuating pawl 38 is thus spring energized to pass upwardly between each pair of opposed blank tape spaces 24 of the fastener chain 20 emerging from the guideway 45, to be engaged by the succeeding pair of interengaged scoop groups 23.

The first arm 46 has its left hand end pivotally

connected to the top end of an upstanding second arm 53. This second arm is itself pivotally mounted at its bottom end on the frame 49. The first arm 46 is thus permitted both to pivot about its left hand end in the vertical plane containing the median line 39 of the fastener chain 20 and to move in the same plane along the fastener chain path in rolling engagement with the arm support 47.

A plunger 54 slidably supported by the frame 49 is biased by a helical compression spring 55 into abutting engagement with the second arm 53, thereby urging the latter leftwardly or rearwardly with respect to the predetermined traveling direction of the fastener chain 20 along the path. The limit switch 40 is mounted on the frame 49 in opposed relationship to the second arm 53. This second arm is therefore to be pivoted in a clockwise direction against the bias of the compression spring 55 when the actuating pawl 38, engaged by each pair of interengaged scoop groups 23 of the fastener chain 20 as above, is displaced therewith forwardly along the path.

It is thus seen that the limit switch 40 is actuated by the second arm 53 each time the actuating pawl 38 is displaced a predetermined distance along the fastener chain path in engagement with one of the successive spaced pairs of interengaged scoop groups 23. The actuation of this limit switch 40 results in the temporary termination of the fastener chain movement along the path, among other operations hereinafter set forth. The stroke S of the second arm 53 can be adjustably varied by an adjusting bolt 56 provided thereto.

For spreading a pair of opposed blank tape spaces 24 of the temporarily stopped fastener chain 20, the separator mechanism 41 is disposed under the fastener chain path and some distance forwardly of the actuating pawl 38. As pictured also in Figures 8 and 9, the separator mechanism 41 includes the separator finger 27 which is aligned with the median line 39 of the fastener chain 20 and which is tapered both upwardly and forwardly. As mentioned, the separator finger 27 has the pair of halves 26 which are movable toward and away from each other in a direction transverse to the fastener chain 20, besides being jointly movable up and down relative to the fastener chain.

The separable halves 26 of the separator finger 27 are each formed on one of the arms 57 of a pair of bell cranks 58 which are pivotally pinned at 59 on an upstanding, hollow guide structure 60. This hollow guide structure 60 is slidably supported by the frame 49 for up and down motion relative to same. Connected to a suitable actuating mechanism (not shown), the hollow guide structure 60 is movable at least between a lowered position of Figure 8 and a raised position of Figure 10.

The other arms 61 of the bell cranks 58 extend away from each other through openings 62 formed in the confronting side walls of the hollow

guide structure 60. These bell crank arms 61 are pivotally pinned at 63 to the ends of a pair of links 64, respectively. The opposite ends of the links 64 are connected by a common pivot pin 65 to a slide 66 slidably received in the hollow 67 of the guide structure 60 and thereby constrained to up and down motion relative to same. A helical compression spring 68 is mounted between guide structure 60 and slide 66 to bias the latter upwardly, so that the separator finger halves 26 are normally held in close contact with each other via the bell cranks 58 and links 64.

Projecting forwardly from the slide 66, through an opening 69 in the guide structure 60, is an arm 70 which is operatively connected to a suitable actuating mechanism (not shown) capable of moving the slide up and down relative to the guide structure. The slide 66 is to be lowered against the bias of the compression spring 68 by the unshown actuating mechanism while the separator finger 27 is in the raised position of Figure 10. The descent of the slide 66 results in the pivotal motion of the pair of bell cranks 58 in the opposite directions indicated by the arrows in Figure 8. The separator finger halves 26 are thus moved away from each other to spread the pair of opposed blank tape spaces 24 and to partly disengage the preceding pair of scoop groups 23 as shown in Figure 2.

The die 42 is also disposed under the path of the fastener chain 20, just rearwardly of the separator mechanism 41. Formed in the horizontally disposed top surface 71 of this die 42 are a pair of transversely spaced, parallel channel-like cavities 72 for receiving the trailing endmost scoops 23' of the partly disengaged pair of scoop groups 23. The die cavities 72 are so shaped and sized in relation to the scoops 23 that the top stops 32 can be bent and clinched around the endmost scoops 23' therein as shown in Figure 5. As clearly seen in Figure 9, the width of the land 73 between the die cavities 72 is equal to the width or thickness of the untapered portion 74 of the separator finger 27 with its separable halves 26 in close contact with each other.

Disposed further rearwardly of the die 42 is a stop 75 having an abutment 76 which lies in a vertical plane transverse to the fastener chain 20 and which substantially bounds the rear extremities of the die cavities 72. This abutment 76 is intended to engage and stop the trailing endmost scoops 23' of the partly disengaged pair of scoop groups 23 as the latter are moved rearwardly past the separator finger 27, with its separable halves 26 closed after having spread the pair of opposed blank tape spaces 24, by the aforementioned reverse rotation of the squeeze roll 36. The endmost scoops 23' are thus properly positioned in the respective die cavities 72.

The stop 75 has a top surface 77 disposed flush with the top surface 71 of the die 42 and substantially on a level with the guideway 45 defined by the guide block 44. Formed in this top surface 77 are a pair of transversely spaced, parallel grooves 78 for relatively closely receiving

the continuous core cords 25 of the fastener chain 20.

As will be noted from Figures 9, 11 and 12, each of the grooves 78 in the stop 75 has a depth and width approximately equal to the thickness T and width W , respectively, of each core cord 25. The grooves 78 are aligned with the die cavities 72. The width D of the land 79 between these grooves, however, is greater than the width of the land 73 between the die cavities 72, to such an extent that those portions of the core cords 25 which immediately succeed the partly disengaged pair of scoop groups 23 are received in the grooves 78 as the endmost scoops 23' of these scoop groups are positioned in the die cavities. The top surface portion of the stop 75, indicated at 87 in Figure 9, is also formed into a depression of the same depth as the grooves 78.

With reference back to Figure 7, the squeeze roll 36 is provided with a reverse feed mechanism 8 for feeding the fastener chain 20 rearwardly a limited distance such that the endmost scoops 23' of the partly disengaged pair of scoop groups 23 are moved back into engagement with the abutment 76 of the stop 75. The reverse feed mechanism 80 comprises a lever 81 connected at one end to the squeeze roll 36 via an electromagnetic clutch or brake (not shown), and an arm 82 pivotally connected to the other end of the lever 81.

Connected to a suitable actuating mechanism (not shown), the arm 82 is to be thrust leftwardly or rearwardly to cause the pivotal motion of the lever 81 in a counterclockwise direction. This counterclockwise rotation of the lever 81 is transmitted to the squeeze roll 36 via the unshown electromagnetic brake, which is then assumed to be in engagement. The stroke of the arm 82 corresponds to the distance the fastener chain 20 is required to traverse in moving the disengaged endmost scoops 23' back into engagement with the abutment 76.

A fastener chain holder plate 83 having an opening 84 formed therein is disposed over the die 42 and stop 75 for up and down motion relative to same. When lowered by an actuating mechanism (not shown), the holder plate 83 lightly holds the stringer tapes 22 on the top surfaces 71 and 77 of the die 42 and stop 75, as shown in Figures 10 and 12. The punch mechanism 43 also disposed over the die 42 for up and down motion relative to same is shown to be of prior art construction.

Although the details of the circuitry for the automatic operation of this apparatus are not specifically illustrated because of their rather common and well known nature, it is understood that the actuation of the limit switch 40 results, either simultaneously or successively, in the termination of the fastener chain movement along the path, the descent of the upstanding bar 52 and fastener chain holder plate 83, the ascent of the hollow guide structure 60, the descent of the slide 66, the leftward or rearward thrust of the arm 82, and the descent of the punch mechanism 43.

It is further understood that these parts and components of the apparatus are to be returned to their initial positions shown in Figure 7 upon lapse of preassigned lengths of time.

5 In the operation of the Figure 7 apparatus, the fastener chain 20 of Figure 1 is first threaded, with the successive pairs of interengaged scoop groups 23 directed downwardly, between the pair of squeeze rolls 36 and 37, through the guideway 45 in the guide block 44, over the die 42, and between the pair of feed rolls 34 and 35. The actuating pawl 38 may be held in abutment against one of the pairs of interengaged scoop groups 23 under the bias of the compression spring 51.

10 The drive roll 34 may now be set in rotation to initiate the travel of the fastener chain 20 forwardly or rightwardly along the path. As one of the pairs of opposed blank tape spaces 24 of the fastener chain 20 emerges from the guideway 45, the actuating pawl 38 projects upwardly through the gap between the pair of blank tape spaces under the bias of the compression spring 51. With the continued travel of the fastener chain 20, the actuating pawl 38 is engaged by the succeeding pair of interengaged scoop groups 23 and is thereby transported forwardly along the path. The upstanding second arm 53 is pivoted in the clockwise direction against the bias of the compression spring 55 with the forward travel of the actuating pawl 38, until the second arm engages and actuates the limit switch 40.

20 As mentioned, the drive roll 34 is set out of rotation upon actuation of the limit switch 40. Simultaneously, the upstanding bar 52 is lowered against the bias of the compression spring 51, with the result that the actuating pawl 38 falls by gravity out of engagement with the fastener chain 20, down to the arm support 47. The disengaged actuating pawl 38 is then moved back to the initial position as the second arm 53 is pivoted in the counterclockwise direction under the bias of the compression spring 55.

25 The hollow guide structure 60 of the separator mechanism 41 is also raised immediately upon actuation of the limit switch 40. It is understood that when the fastener chain 20 is stopped as above, the leading end of the pair of blank tape spaces 24 between which the actuating pawl 38 has been caught is located forwardly of the separator finger 27. Thus, upon ascent of the guide structure 60, the separator finger 27 projects upwardly through the gap between this particular pair of blank tape spaces 24.

30 The fastener chain holder plate 83 is then lowered to lightly hold the pair of stringer tapes 22 against the top surfaces 71 and 77 of the die 42 and stop 75. Then the slide 66 is lowered against the bias of the compression spring 68 within the raised hollow guide structure 60, thereby causing the pair of separator finger halves 26 to move apart from each other. Thus, as shown in Figure 2, the pair of blank tape spaces 24 are spread to such an extent that the preceding pair of interengaged scoop groups 23

are partly disengaged. The slide 66 is then allowed to move upwardly under the bias of the compression spring 68 relative to the guide structure 60, thereby causing the separator finger halves 26 to move back into contact with each other.

70 Then, with the unshown electromagnetic clutch or brake on the squeeze roll 36 engaged, the arm 82 of the reverse feed mechanism 80 is thrust rearwardly to cause the pivotal motion of the lever 81 in the counterclockwise direction. An electromagnetic clutch (not shown) on the drive roll 34 is now assumed to be disengaged, so that the fastener chain 20 is moved rearwardly along the path by the reverse rotation of the squeeze roll 36.

75 As clearly depicted in Figure 10, the endmost scoops 23' of the partly disengaged pair of scoop groups 23 travel rearwardly past the separator finger 27, while being guided by its opposite lateral surfaces 85 and the top surfaces 86 of the bell crank arms 57, into the die cavities 72. The rearward travel of the fastener chain 20 terminates as the endmost scoops 23' move into engagement against the abutment 76 of the stop 75. The endmost scoops 23' are now properly positioned in the die cavities 72, and as shown in Figure 12, those portions of the core cords 25 which immediately succeed these endmost scoops are received in the grooves 78 in the stop 75.

80 The punch mechanism 43 is then actuated to apply the top stops 32 to the endmost scoops 23' in the die cavities 72, as will be seen by referring back to Figures 4, 5 and 6. The punch mechanism 43 and fastener chain holder plate 83 are then raised to their initial positions. Thereafter, preferably, the guide structure 60 of the separator mechanism 41 is again raised to such an extent that the endmost scoops 23' together with the top stops 32 thereon are ejected from the die cavities 72 by the top surfaces 86 of the bell crank arms 57. The guide structure 60 is then lowered to the initial position of Figure 8. The arm 82 of the reverse feed mechanism 80 may be returned to the initial position by a spring (not shown) or the like.

85 The drive roll 34 is then set in rotation to resume the feeding of the fastener chain 20 forwardly along the path. The foregoing cycle of operation is thereafter repeated to attach top stops 32 to the trailing ends of the successive spaced pairs of scoop groups 23.

90 In another preferred embodiment shown in Figures 13, 14 and 15, the invention is adapted for applying top stops 32 to the scoops 23' at the leading end of each of the successive spaced pairs of scoop groups 23 of the fastener chain 20. In the apparatus shown in Figure 14, the guide block, the actuating pawl and other means directly associated therewith can be exactly identical in construction and relative arrangement with those in the Figure 7 apparatus. Such parts are therefore identified by the same reference characters as those used to denote the

corresponding parts of the Figure 7 apparatus, and their description will be omitted.

The Figure 14 apparatus differs from that of Figure 7 principally in the relative arrangement of separator mechanism 41a, die 42a and stop 75a, and in an intermittent forward feed mechanism 90 provided to the drive roll 34 instead of the reverse feed mechanism 80 of the preceding embodiment. The separator mechanism 41a is disposed rearwardly of the die 42a, which die in turn is disposed rearwardly of the stop 75a.

Considered individually, the separator mechanism 41a, die 42a and stop 75a are also identical in construction with their respective counterparts of the Figure 7 apparatus, so that any detailed description of their constructions will also be omitted. It should be noted, however, that each of these components is reversed in direction. Thus, for example, the separator finger 27a tapers both upwardly and rearwardly, and the abutment 76a is formed at the rear end of the stop 75a, as will be best understood from Figure 15.

The intermittent forward feed mechanism 90 comprises a lever 91 connected at one end to the drive roll 34 via an overrunning clutch (not shown), and an arm 92 pivotally connected to the other end of the lever 91. The arm 92 is further operatively connected to a suitable actuating mechanism (not shown), thereby to be thrust rightwardly or forwardly to cause the rotation of the drive roll 34 in a clockwise direction via the unshown overrunning clutch. It is of course understood that a separate drive mechanism (not shown) is connected to the drive roll 34 for the usual feeding of the fastener chain 20 along the predetermined path.

In the operation of the Figure 14 apparatus, the actuating pawl 38 is spring biased to project upwardly through the gap between each pair of opposed blank tape spaces 24 of the fastener chain 20 emerging from the guideway 45, just as in the Figure 7 apparatus. Engaged by the succeeding pair of interengaged scoop groups 23, the actuating pawl 38 is displaced forwardly along the path, until the limit switch 40 is actuated by the upstanding second arm 53.

Also in the Figure 7 apparatus, the actuation of the limit switch 40 results in the temporary termination of the fastener chain movement along the path, the disengagement of the actuating pawl 38 from the fastener chain, the spreading of the pair of opposed blank tape spaces 24 by the separator mechanism 41a, and the descent of the fastener chain holder plate 83. In this particular embodiment, however, the spreading of the pair of blank tape spaces 24 is effected in such a way that the succeeding pair of scoop groups 23 are partly disengaged from each other, as will be noted from Figure 13. The separator finger halves 26a are then moved back into contact with each other.

The intermittent forward feed mechanism 90 is then actuated, either upon lapse of a preassigned length of time following the actuation of the limit switch 40 or in response to the operation of the

separator mechanism 41a. With the drive roll 34 rotated by this intermittent forward feed mechanism 90 in the clockwise direction through a limited angle, the fastener chain 20 is fed forwardly until the leading endmost scoops 23" of the partly disengaged pair of scoop groups move into engagement with the abutment 76a of the stop 75a and become thus properly positioned in the die cavities 72a. This condition is pictured in Figure 15. Those portions of the core cords 25 of the fastener chain 20 which immediately precede the endmost scoops 23" are now received in the grooves 78a in the stop 75a. The endmost scoops 23" are of course guided as aforesaid by the separator finger and bell cranks of the separator mechanism 41a in their movement into the die cavities 72a.

The punch mechanism 43 is then actuated to apply the top stops 32 to the endmost scoops 23" in the die cavities 72a, as shown in Figure 13. The punch mechanism 43 and fastener chain holder plate 83 are then raised to their initial positions. The hollow guide structure 60a of the separator mechanism 41a is also raised to an extent necessary to eject from the die cavities 72a the endmost scoops 23" together with the top stops 32 thereon. The guide structure 60a is then lowered to the initial position. Thereafter, with the drive roll 34 set in rotation to resume the feeding of the fastener chain 20, the foregoing cycle of operation is repeated to apply top stops 32 to the leading ends of the successive spaced pairs of scoop groups 23.

As will have been understood from the foregoing, the preferred method of this invention comprises the steps of feeding the fastener chain 20 along the predetermined path; causing the actuating pawl 38 to pass between each pair of opposed blank tape spaces 24 of the fastener chain so as to be engaged by the succeeding pair of interengaged scoop groups 23; causing the separator mechanism 41 or 41a to spread the pair of blank tape spaces in response to the signal generated by the limit switch 40 upon displacement of the actuating pawl with the fastener chain over the predetermined distance along the path, the pair of blank tape spaces being spread to such an extent that at least the preceding or succeeding pair of scoop groups are partly disengaged from each other; further moving the fastener chain along the path, either forwardly or rearwardly, to such an extent that the disengaged endmost scoops 23' or 23" of the partly disengaged pair of scoop groups come into abutment against the stop means 75 or 75a disposed adjacent to the die 42 or 42a, thereby positioning the disengaged end portions of the partly disengaged pair of scoop groups upon the die; attaching the top stops 32 to the disengaged end portions of the pair of scoop groups by the punch mechanism 43 coacting with the die; and repeating the foregoing steps to apply the top stops to the trailing or leading end portions of the successive spaced pairs of scoop groups.



Claims

1. A method of attaching top stops to a sliding clasp fastener chain including a pair of continuous stringers, each stringer having successive spaced groups of scoops mounted on a continuous stringer tape and interengaged with opposed complementary groups of scoops on the other stringer tape, which method comprises: moving the fastener chain longitudinally along a predetermined path; causing a displaceable actuating pawl to be engaged by one of the successive spaced pairs of scoop groups of the fastener chain so as to be displaced therewith along the path; causing a separator to spread one of the successive pairs of opposed blank tape spaces of the fastener chain in response to displacement of the actuating pawl over a predetermined distance with the fastener chain, the pair of blank tape spaces being spread to such an extent that at least one adjacent pair of scoop groups are partly disengaged from each other; subsequently moving the fastener chain along the path until the disengaged endmost scoops of the partly disengaged pair of scoop groups come into abutment against stop means disposed adjacent to a die, whereby the disengaged end portions of the partly disengaged pair of scoop groups are positioned upon the die; attaching top stops to the disengaged end portions of the partly disengaged pair of scoop groups by punch means coacting with the die; and repeating the foregoing steps to apply top stops to the end portions of the successive spaced pairs of scoop groups.
2. A method as claimed in claim 1, wherein the first-mentioned movement of the fastener chain longitudinally along said path is in a predetermined direction; wherein the first-mentioned movement of the fastener chain is stopped when the actuating pawl has been displaced said predetermined distance with the fastener chain; wherein the partly disengaged pair of scoop groups are the preceding pair of scoop groups; wherein said subsequent movement of the fastener chain is in a direction opposite to said predetermined direction; and wherein the repetition of the foregoing steps is such that the end portions to which the top stops are applied are the trailing end portions of the successive spaced pairs of scoop groups.
3. A method as claimed in claim 1, wherein the first-mentioned movement of the fastener chain longitudinally along said path is in a predetermined direction; wherein the first-mentioned movement of the fastener chain is stopped when the actuating pawl has been displaced said predetermined distance with the fastener chain; wherein the partly disengaged pair of scoop groups are the succeeding pair of scoop groups; wherein said subsequent movement of the fastener chain is in said predetermined direction; and wherein the repetition of the foregoing steps is such that the end portions to which the top stops are applied are the leading end portions of the successive spaced pairs of scoop groups.
4. Apparatus for attaching top stops to a sliding clasp fastener chain including a pair of continuous stringers, each stringer having successive spaced groups of scoops mounted on a continuous stringer tape and interengaged with opposed complementary groups of scoops on the other stringer tape, the apparatus comprising: means for feeding the fastener chain along a predetermined path; an actuating pawl disposed on the median line of the fastener chain for movement both along the path and in a plane normal to the plane of the fastener chain; means biasing the actuating pawl toward the fastener chain whereby the actuating pawl is capable of passing between each of the successive pairs of opposed blank tape spaces of the fastener chain so as to be engaged by the succeeding pair of interengaged scoop groups; switch means adapted to be actuated when the actuating pawl is displaced a predetermined distance along the path in engagement with the pair of interengaged scoop groups; separator means responsive to the actuation of the switch means for spreading one of the pairs of opposed blank tape spaces to such an extent that at least one adjacent pair of scoop groups are partly disengaged from each other; a die disposed substantially on the path of the fastener chain; stop means immovably disposed adjacent to the die, the stop means being adapted to engage and stop the disengaged endmost scoops of the partly disengaged pair of scoop groups as the fastener chain is further moved along the path after the spreading of the pair of blank tape spaces, whereby the disengaged end portions of the partly disengaged pair of scoop groups are positioned upon the die; and punch means coacting with the die to attach top stops to the disengaged end portions of the partly disengaged pair of scoop groups.
5. The apparatus according to claim 4, wherein the switch means when actuated is further adapted to cause the feeding means to stop the fastener chain.
6. The apparatus according to claim 5, wherein the separator means is disposed forwardly, and the stop means rearwardly, of the die with respect to a predetermined direction in which the fastener chain is fed along the path, and wherein the apparatus further comprises means for moving the fastener chain in a direction opposite to said predetermined direction in moving the disengaged endmost scoops of the partly disengaged pair of scoop groups into abutment against the stop means.
7. The apparatus according to claim 5, wherein the separator means is disposed rearwardly, and the stop means forwardly, of the die with respect to a predetermined direction in which the fastener chain is fed along the path, and wherein the apparatus further comprises means for moving the fastener chain in said predetermined direction in moving the disengaged endmost scoops of the partly disengaged pair of scoop groups into abutment against the stop means.
8. The apparatus according to claim 4, 5, 6 or



- 7, wherein the fastener chain extends horizontally along the path and wherein the apparatus further comprises: a first arm extending substantially horizontally under the path for movement
- 5 therealong and having the actuating pawl formed at one end; an arm support disposed under said one end of the first arm for up and down motion therewith and urged upwardly by the biasing means, said one end of the first arm resting on the
- 10 arm support by gravity and being movable thereover; a second arm disposed substantially uprightly and pivotally supported at its lower end, the second arm having its upper end connected to the other end of the first arm so as to permit the
- 15 pivotal movement of the latter in the plane normal to the plane of the fastener chain, whereby the second arm is pivoted about its lower end with the movement of the first arm along the path; and second biasing means acting on the second arm
- 20 to bias the actuating pawl in a direction opposite to a predetermined direction in which the fastener chain is fed along the path.
9. The apparatus according to claim 8, wherein the switch means is a limit switch adapted to be
- 25 actuated by the second arm.
10. The apparatus according to claim 8, further comprising a roller rotatably mounted on said one end of the first arm for rolling engagement with the arm support.
- 30 11. The apparatus according to claim 4, 5, 6 or 7, wherein the fastener chain extends horizontally along the path, and wherein the separator means comprises: a hollow guide structure disposed under the path for up and down motion relative to
- 35 same; a slide slidably received within the guide structure for up and down motion relative to same; second biasing means yieldably urging the slide upwardly relative to the guide structure; a
- 40 pair of bell cranks pivotally mounted on the guide structure; a pair of links each pivotally connected at one end to the slide and at the other end to one of the arms of one of the bell cranks; and a separator finger having a pair of separable halves formed respectively on the other arms of the bell
- 45 cranks.
12. The apparatus according to claim 11, wherein the die has formed therein a pair of cavities for receiving the disengaged end portions
- 50 of the partly disengaged pair of scoop groups, and wherein the separator finger and said other arms of the bell cranks are adapted to serve as guides in moving the disengaged end portions of the partly disengaged pair of scoop groups into the die cavities.
13. A method substantially as herein described with reference to and as illustrated in Figures 2 to 12 inclusive of the accompanying drawings.
14. A method substantially as herein described with reference to and as illustrated in Figures 13 to
- 60 15 inclusive of the accompanying drawings.
15. Apparatus substantially as herein described with reference to and as illustrated in Figures 2 to 12 inclusive of the accompanying drawings.
16. Apparatus substantially as herein described with reference to and as illustrated in Figures 13 to 15 inclusive of the accompanying drawings.
- 65