FAN ARRAY FAN SECTION IN AIR-HANDLING SYSTEMS

The present application is a continuation-in-part application of Patent Application Serial Number 10/806,775, filed March 22, 2004, and entitled FAN ARRAY FAN SECTION IN AIR-HANDLING SYSTEMS. Patent Application Serial Number 10/806,775 is a nonprovisional application claiming the benefit under 35 USC Section 119(e) of U.S. Provisional Patent Application Serial Number 60/554,702, filed March 20, 2004, and entitled FAN ARRAY FAN SECTION IN AIR-HANDLING SYSTEMS. Patent Application Serial Number 10/806,775 is a nonprovisional application claiming the benefit under 35 USC Section 119(e) of U.S. Provisional Patent Application Serial Number 60/456,413, filed March 20, 2003, and entitled FAN ARRAY FAN SECTION IN AIR-HANDLING SYSTEMS. The present application is a continuation-in-part application of PCT Patent Application Serial Number PCT/US2004/008578, filed March 19, 2004, and entitled FAN ARRAY FAN SECTION IN AIR-HANDLING SYSTEMS. The present application is based on and claims priority from these applications, the disclosures of which are hereby expressly incorporated herein by reference.

BACKGROUND OF INVENTION

The present invention is directed to a fan array fan section utilized in an air-handling system.

Air-handling systems (also referred to as an air handler) have traditionally been used to condition buildings or rooms (hereinafter referred to as "structures"). An air-handling system is defined as a structure that includes components designed to work together in order to condition air as part of the primary system for ventilation of structures. The air-handling system may contain components such as cooling coils, heating coils, filters, humidifiers, fans, sound attenuators, controls, and other devices functioning to meet the needs of the structures. The air-handling system may be manufactured in a factory and brought to the structure to be installed or it may be built on site using the necessary devices to meet the functioning needs of the structure. The air-handling compartment 102 of the air-handling system includes the inlet plenum 112 prior to the fan inlet cone 104 and the discharge plenum 110. Within the air-handling compartment 102 is situated the fan unit 100 (shown in FIGS. 1 and 2 as an inlet cone 104, a fan 106, and a motor 108), fan frame, and any appurtenance associated with the function of the fan (e.g. dampers, controls, settling means, and associated cabinetry). Within the fan 106 is a fan wheel (not shown) having at least one blade. The fan wheel has a fan wheel diameter that is measured from one side of the outer periphery of the fan wheel to the opposite side of the outer periphery of the fan wheel. The dimensions of the handling compartment 102 such as height, width, and airway length are determined by consulting fan manufacturers data for the type of fan selected.

FIG. 1 shows an exemplary prior art air-handling system having a single fan unit 100 housed in an air-handling compartment 102. For exemplary purposes, the fan unit 100 is shown having an inlet cone 104, a fan 106, and a motor 108. Larger structures, structures requiring greater air volume, or structures requiring higher or lower temperatures have generally needed a larger fan unit 100 and a generally correspondingly larger air-handling compartment 102.

As shown in FIG. 1, an air-handling compartment 102 is substantially divided into a discharge plenum 110 and an inlet plenum 112. The combined discharge plenum 110 and the inlet plenum 112 can be referred to as the airway path 120. The fan unit 100 may be situated in the discharge plenum 110 as shown), the inlet plenum 112, or partially within the inlet plenum 112 and partially within the discharge plenum 110. The portion of the airway path 120 in which the fan unit 100 is positioned may be generically referred to as the "fan section" (indicated by reference numeral 114). The size of the inlet cone 104, the size of the fan 106, the size the motor 108, and the size of the fan frame (not shown) at least partially determine the length of the airway path 120. Filter banks 122 and/or cooling coils (not shown) may be added to the system either upstream or downstream of the fan units 100.

For example, a first exemplary structure requiring 50,000 cubic feet per minute of air flow at six (6) inches water gage pressure would generally require a prior art air-handling compartment 102 large enough to house a 55 inch impeller, a 100 horsepower motor, and supporting framework. The prior art air-handling compartment 102, in turn would be approximately 92 inches high by 114 to147 inches wide and 106 to 112 inches long. The minimum length of the air-handling compartment 102 and/or airway path 120 would be dictated by published manufacturers data for a given fan type, motor size, and application. Prior art cabinet sizing guides show exemplary rules for configuring an air-handling compartment 102. These rules are based on optimization, regulations, and experimentation.

For example, a second exemplary structure includes a recirculation air handler used in semiconductor and pharmaceutical clean rooms requiring 26,000 cubic feet per minute at two (2) inches water gage pressure. This structure would generally require a prior art air-handling system with a air-handling compartment 102 large enough to house a 44 inch impeller, a 25 horsepower motor, and supporting framework. The prior art air-handling compartment 102, in turn would be approximately 78 inches high by 99 inches wide and 94 to 100 inches long. The minimum length of the air-handling compartment 102 and/or airway path 120 would be dictated by published manufacturers data for a given fan type, motor size and application. Prior art cabinet

sizing guides show exemplary rules for configuring an air-handling compartment 102. These rules are based on optimization, regulations, and experimentation.

These prior art air-handling systems have many problems including the following exemplary problems:

- Because real estate (e.g. structure space) is extremely expensive, the larger size of the air-handling compartment 102 is extremely undesirable.
- The single fan units 100 are expensive to produce and are generally custom produced for each job.
- Single fan units 100 are expensive to operate.
- Single fan units 100 are inefficient in that they only have optimal or peak efficiency over a small portion of their operating range.
- If a single fan unit 100 breaks down, there is no air conditioning at all.
- The low frequency sound of the large fan unit 100 is hard to attenuate.
- The high mass and turbulence of the large fan unit 100 can cause undesirable vibration.

Height restrictions have necessitated the use of air-handling systems built with two fan units 100 arranged horizontally adjacent to each other. It should be noted, however, that a good engineering practice is to design air handler cabinets and discharge plenums 110 to be symmetrical to facilitate more uniform air flow across the width and height of the cabinet. Twin fan units 100 have been utilized where there is a height restriction and the unit is designed with a high aspect ratio to accommodate the desired flow rate. As shown in the Greenheck "Installation Operating and Maintenance Manual," if side-by-side installation was contemplated, there were specific instructions to arrange the fans such that there was at least one fan wheel diameter spacing between the fan wheels and at least one-half a fan wheel diameter between the fan and the walls or ceilings. The Greenheck reference even specifically states that arrangements "with less spacing will experience performance losses." Normally, the air-

handling system and air-handling compartment 102 are designed for a uniform velocity gradient of 500 feet per minute velocity in the direction of air flow. The two fan unit 100 air-handling systems, however, still substantially suffered from the problems of the single unit embodiments. There was no recognition of advantages by increasing the number of fan units 100 from one to two. Further, the two fan unit 100 section exhibits a non-uniform velocity gradient in the region following the fan unit 100 that creates uneven air flow across filters, coils, and sound attenuators.

It should be noted that electrical devices have taken advantage of multiple fan cooling systems. For example, U.S. Patent No. 6,414,845 to Bonet uses a multiple-fan modular cooling component for installation in multiple component-bay electronic devices. Although some of the advantages realized in the Bonet system would be realized in the present system, there are significant differences. For example, the Bonet system is designed to facilitate electronic component cooling by directing the output from each fan to a specific device or area. The Bonet system would not work to direct air flow to all devices in the direction of general air flow. Other patents such as U.S. Patent No. 4,767,262 to Simon and U.S. Patent No. 6,388,880 to El-Ghobashy et al. teach fan arrays for use with electronics.

Even in the computer and machine industries, however, operating fans in parallel is taught against as not providing the desired results except in low system resistance situations where fans operate in near free delivery. For example, Sunon Group has a web page in which they show two axial fans operating in parallel, but specifically state that if "the parallel fans are applied to the higher system resistance that [an] enclosure has, . . . less increase in flow results with parallel fan operation." Similar examples of teaching against using fans in parallel are found in an article accessible from HighBeam Research's library (http://stati.highbeam.com) and an article by Ian McLeod accessible at (http://www.papstplc.com).

BRIEF SUMMARY OF THE INVENTION

The present invention is directed to a fan array fan section in an air-handling system. The fan array fan section includes a plurality of fan units arranged in a fan array. Each fan unit is positioned within a fan unit chamber/cell. Each fan unit chamber/cell has at least one acoustically absorptive insulation surface. The insulation surfaces of the fan unit chambers/cells together form a coplanar silencer. Sound waves from the fan units passing through the insulation surface at least partially dissipate as they pass therethrough. In one preferred embodiments the fan unit chamber/cell is a cell having a frame that supports the insulation surfaces.

The present invention is also directed to a fan array fan section in an air-handling system that includes a plurality of fan units arranged in a fan array and positioned within an air-handling compartment. One preferred embodiment may include an array controller programmed to operate the plurality of fan units at peak efficiency. The plurality of fan units may be arranged in a true array configuration, a spaced pattern array configuration, a checker board array configuration, rows slightly offset array configuration, columns slightly offset array configuration, or a staggered array configuration.

The foregoing and other objectives, features, and advantages of the invention will be more readily understood upon consideration of the following detailed description of the invention, taken in conjunction with the accompanying drawings.

BRIEF DESCRIPTION OF THE SEVERAL VIEWS OF THE DRAWINGS

FIG. 1 is a side view of an exemplary prior art air-handling system having a single large fan unit within an air-handling compartment.

FIG. 2 is a perspective view of an exemplary prior art large fan unit.

FIG. 3 is a side view of an exemplary fan array fan section in an air-handling system of the present invention having a plurality of small fan units within an air-handling compartment.

FIG. 4 is a plan or elevation view of a 4x6 exemplary fan array fan section in an air-handling system of the present invention having a plurality of small fan units within an air-handling compartment.

FIG. 5 is a plan or elevation view of a 5x5 exemplary fan array fan section in an air-handling system of the present invention having a plurality of small fan units within an air-handling compartment.

FIG. 6 is a plan or elevation view of a 3x4 exemplary fan array fan section in an air-handling system of the present invention having a plurality of small fan units within an air-handling compartment.

FIG. 7 is a plan or elevation view of a 3x3 exemplary fan array fan section in an air-handling system of the present invention having a plurality of small fan units within an air-handling compartment.

FIG. 8 is a plan or elevation view of a 3x1 exemplary fan array fan section in an air-handling system of the present invention having a plurality of small fan units within an air-handling compartment.

FIG. 9 is a plan or elevation view of an alternative exemplary fan array fan section in an air-handling system of the present invention in which a plurality of small fan units are arranged in a spaced pattern array within an air-handling compartment.

FIG. 10 is a plan or elevation view of an alternative exemplary fan array fan section in an air-handling system of the present invention in which a plurality of small fan units are arranged in a checker board array within an air-handling compartment.

FIG. 11 is a plan or elevation view of an alternative exemplary fan array fan section in an air-handling system of the present invention in which a plurality of small fan units are arranged in rows slightly offset array within an air-handling compartment.

FIG. 12 is a plan or elevation view of an alternative exemplary fan array fan section in an air-handling system of the present invention in which a plurality of small fan units are arranged in columns slightly offset array within an air-handling compartment.

FIG. 13 is a plan or elevation view of a 5x5 exemplary fan array fan section in an air-handling system of the present invention running at 52% capacity by turning a portion of the fans on and a portion of the fans off.

FIG. 14 is a plan or elevation view of a 5x5 exemplary fan array fan section in an air-handling system of the present invention running at 32% capacity by turning a portion of the fans on and a portion of the fans off.

FIG. 15 is a side view of an alternative exemplary fan array fan section in an air-handling system of the present invention having a plurality of staggered small fan units within an air-handling compartment.

FIG. 16 is a perspective view of an exemplary fan array using a grid system into which fan units are mounted.

FIG. 17 is a perspective view of an exemplary fan array using a grid system or modular units each of which includes a fan units mounted within its own fan unit chamber.

FIG. 18 is a cross-sectional view of an exemplary insulated grid array system or modular unit system having interior surfaces made from acoustically absorptive material.

FIGS. 19-23 are cross-sectional view of an exemplary insulated grid array system or modular unit system having interior surfaces made from acoustically absorptive material showing sound wave reaction.

FIG. 24 is a wave form diagram illustrating the principle of wave cancellation.

FIG. 25 is a perspective view of an exemplary array of dampeners that may be positioned either in front of or behind the fan units.

DETAILED DESCRIPTION OF THE INVENTION

The present invention is directed to a fan array fan section in an air-handling system. As shown in FIGS. 3-12, the fan array fan section in the air-handling system uses a plurality of individual single fan units 200. In one preferred embodiment, the fan units 200 are arranged in a true array (FIGS. 4-8), but alternative embodiments may include, for example, alternative arrangements such as in a spaced pattern (FIG. 9), a checker board (FIG. 10), rows slightly offset (FIG. 11), or columns slightly offset (FIG. 12). As the present invention could be implemented with true arrays and/or alternative arrays, the term "array" is meant to be comprehensive.

The fan units 200 in the fan array of the present invention may be spaced as little as 20% of a fan wheel diameter. Optimum operating conditions for a closely arranged array may be found at distances as low as 30% to 60% of a fan wheel diameter. By closely spacing the fan units 200, more air may be moved in a smaller space. For example, if the fan wheels of the fan units 200 have a 20 inch fan wheel diameter, only a 4 inch space (20%) is needed between the outer periphery of one fan wheel and the outer periphery of the adjacent fan wheel (or a 2 inch space between the outer periphery of a fan wheel and an the adjacent wall or ceiling).

By using smaller fan units 200 it is possible to support the fan units 200 with less intrusive structure (fan frame). This can be compared to the large fan frame that supports prior art fan units 100 and functions as a base. This large fan frame must be large and sturdy enough to support the entire weight of the prior art fan units 100. Because of their size and position, the known fan frames cause interference with air flow. In the preferred embodiment, therefore, the fan units 200 of the fan array may be supported by a frame that supports the motors 108 with a minimum restriction to air flow.

As mentioned in the Background, others have tried using side-by-side installation of two fan units 100 arranged horizontally adjacent to each other within an air-handling system. As is also mentioned in the Background, fan arrays have been used in electronic and computer assemblies. However, in the air-handling system

industry, it has always been held that there must be significant spacing between the horizontally arranged fan wheels and that arrangements with less spacing will experience performance losses. A single large fan moves all the air in a cabinet. Using two of the same or slightly smaller fans caused the air produced by one fan to interfere with the air produced by the other fan. To alleviate the interference problem, the fans had to be spaced within certain guidelines – generally providing a clear space between the fans of a distance of at least one wheel diameter (and a half a wheel diameter to an adjacent wall). Applying this logic, it would not have made sense to add more fans. And even if additional fans had been added, the spacing would have continued to be at least one wheel diameter between fans. Further, in the air-handling system industry, vertically stacking fan units would have been unthinkable because the means for securing the fan units would not have been conducive to such stacking (they are designed to be positioned on the floor only).

It should be noted that the plenum fan is the preferred fan unit 200 of the present invention. In particular, the APF-121, APF-141, APF-161, and APF-181 plenum fans (particularly the fan wheel and the fan cone) produced by Twin City Fan Companies, Ltd. of Minneapolis, Minnesota, U.S. has been found to work well. The reason that plenum fans work best is that they do not produce points of high velocity such as those produced by axial fans and housed centrifugal fans and large plenum fans. Alternative embodiments use known fan units or fan units yet to be developed that will not produce high velocity gradients in the direction of air flow. Still other embodiments, albeit less efficient, use fan units such as axial fans and/or centrifugal housed fans that have points of high velocity in the direction of air flow.

In the preferred embodiment, each of the fan units 200 in the fan array fan section in the air-handling system is controlled by an array controller 300 (FIGS. 13 and 14). In one preferred embodiment, the array controller 300 may be programmed to operate the fan units 200 at peak efficiency. In this peak efficiency embodiment, rather than running all of the fan units 200 at a reduced efficiency, the array controller 300 turns off certain fan units 200 and runs the remaining fan units 200 at peak efficiency.

In an alternative embodiment, the fan units 200 could all run at the same power level (e.g. efficiency and/or flow rate) of operation.

Another advantage of the present invention is that the array controller 300 (which may be a variable frequency drive (VFD)) used for controlling fan speed and thus flow rate and pressure, could be sized for the actual brake horsepower of the fan array fan section in the air-handling system. Since efficiency of the fan wall array can be optimized over a wide range of flow rates and pressures, the actual operating power consumed by the fan array is substantially less than the actual operating power consumed by the comparable prior art air-handling systems and the array controller's power could be reduced accordingly. The array controller 300 could be sized to the actual power consumption of the fan array where as the controller (which may have been a variable frequency drive) in a traditional design would be sized to the maximum nameplate rating of the motor per Electrical Code requirements. An example of a prior art fan design supplying 50,000 cubic feet per minute of air at 2.5 inches pressure, would require a 50 horsepower motor and 50 horsepower controller. The new invention will preferably use an array of fourteen 2 horsepower motors and a 30 horsepower array controller 300.

This invention solves many of the problems of the prior art air-handling systems including, but not limited to real estate, reduced production costs, reduced operating expenses, increased efficiency, improved air flow uniformity, redundancy, sound attenuation advantages, and reduced vibration.

Controllability

As mentioned, preferably each of the fan units 200 in the fan array fan section in the air-handling system is controlled by an array controller 300 (FIGS. 13 and 14) that may be programmed to operate the fan units 200 at peak efficiency. In this peak efficiency embodiment, rather than running all of the fan units 200 at a reduced efficiency, the array controller 300 is able to turn off certain fan units 200 and run the remaining fan units 200 at peak efficiency. Preferably, the array controller 300 is able to

control fan units 200 individually, in predetermined groupings, and/or as a group as a whole.

For example, in the 5x5 fan array such as that shown in FIGS. 5, 13, and 14, a person desiring to control the array may select desired air volume, a level of air flow, a pattern of air flow, and/or how many fan units 200 to operate. Turning first to air volume, each fan unit 200 in a 5x5 array contributes 4% of the total air. In variable air volume systems, which is what most structures have, only the number of fan units 200 required to meet the demand would operate. A control system (that may include the array controller 300) would be used to take fan units 200 on line (an "ON" fan unit 200) and off line (an "OFF" fan unit 200) individually. This ability to turn fan units 200 on and off could effectively eliminate the need for a variable frequency drive. Similarly, each fan unit 200 in a 5x5 array uses 4% of the total power and produces 4% of the level of air flow. Using a control system to take fan units 200 on line and off line allows a user to control power usage and/or air flow. The pattern of air flow can also be controlled if that would be desirable. For example, depending on the system it is possible to create a pattern of air flow only around the edges of a cabinet or air only at the top. Finally, individual fan units 200 may be taken on line and off line. This controllability may be advantageous if one or more fan units 200 are not working properly, need to be maintained (e.g. needs general service), and/or need to be replaced. The problematic individual fan units 200 may be taken off line while the remainder of the system remains fully functional. Once the individual fan units 200 are ready for use, they may be brought back on line.

A further advantage to taking fan units 200 on and off line occurs when building or structure control systems require low volumes of air at relatively high pressures. In this case, the fan units 200 could be modulated to produce a stable operating point and eliminate the surge effects that sometimes plague structure owners and maintenance staff. The surge effect is where the system pressure is too high for the fan speed at a given volume and the fan unit 200 has a tendency to go into stall.

Examples of controllability are shown in FIGS. 13 and 14. In the fan array fan section in the air-handling system shown in FIG. 13, the array controller 300

alternates "ON" fan units 200 and "OFF" fan units 200 in a first exemplary pattern as shown so that the entire system is set to operate at 52% of the maximum rated air flow but only consumes 32% of full rated power. These numbers are based on exemplary typical fan operations in a structure. FIG. 14 shows the fan array fan section in the air-handling system set to operate at 32% of the maximum rated air flow but only consumes 17% of full rated power. These numbers are based on exemplary typical fan operations in a structure. In this embodiment, the array controller 300 creates a second exemplary pattern of "OFF" fan units 200 and "ON" fan units 200 as shown.

Real Estate

The fan array fan section in the air-handling section 220 of the present invention preferably uses (60% to 80%) less real estate than prior art discharge plenums 120 (with the hundred series number being prior art as shown in FIG. 1 and the two hundred series number being the present invention as shown in FIG. 3) in air-handling systems. Comparing the prior art (FIG. 1) and the present invention (FIG. 3) shows a graphical representation of this shortening of the airway path 120, 220. There are many reasons that using multiple smaller fan units 200 can reduce the length of the airway path 120, 220. For example, reducing the size of the fan unit 100, 200 and motor 108, 208 reduces the length of the discharge plenum 110, 210. Similarly, reducing the size of the inlet cone 104, 204 reduces the length of the inlet plenum 112, 212. The length of the discharge plenum 110, 210 can also be reduced because air from the fan array fan section in the air-handling system of the present invention is substantially uniform whereas the prior art air-handling system has points of higher air velocity and needs time and space to mix so that the flow is uniform by the time it exits the air-handling compartment 102, 202. (This can also be described as the higher static efficiency in that the present invention eliminates the need for settling means downstream from the discharge of a prior art fan system because there is little or no need to transition from high velocity to low velocity.) The fan array fan section in the air-handling system takes in air from the inlet plenum 212 more evenly and efficiently than

the prior art air-handling system so that the length of the inlet plenum 112, 212 may be reduced.

For purposes of comparison, the first exemplary structure set forth in the Background of the Invention (a structure requiring 50,000 cubic feet per minute of air flow at a pressure of six (6) inches water gage) will be used. Using the first exemplary structure, an exemplary embodiment of the present invention could be served by a nominal discharge plenum 210 of 89 inches high by 160 inches wide and 30 to 36 inches long (as compared to 106 to 112 inches long in the prior art embodiments). The discharge plenum 210 would include a 3x4 fan array fan section in the air-handling system such as the one shown in FIG. 6) having 12 fan units 200. The space required for each exemplary fan unit 200 would be a rectangular cube of approximately 24 to 30 inches on a side depending on the array configuration. The airway path 220 is 42 to 48 inches (as compared to 88 to 139 inches in the prior art embodiments).

For purposes of comparison, the second exemplary structure set forth in the Background of the Invention (a structure requiring 26,000 cubic feet per minute of air flow at a pressure of two (2) inches water gage) will be used. Using the second exemplary structure, an exemplary embodiment of the present invention could be served by a nominal discharge plenum 210 of 84 inches high by 84 inches wide, and and 30 to 36 inches long (as compared to 94 to 100 inches long in the prior art embodiments). The discharge plenum would include a 3x3 fan array fan section in the air-handling system (such as the one shown in FIG. 7) having 9 fan units 200. The space required for each exemplary fan unit 200 would be a rectangular cube of approximately 24 to 30 inches on a side depending on the array configuration. The airway path 220 is 42 to 48 inches (as compared to 71 to 95 inches in the prior art embodiments).

Reduced Production Costs

It is generally more cost effective to build the fan array fan section in the air-handling system of the present invention as compared to the single fan unit 100 used in prior art air-handling systems. Part of this cost savings may be due to the fact

that individual fan units 200 of the fan array can be mass-produced. Part of this cost savings may be due to the fact that it is less expensive to manufacture smaller fan units 200. Whereas the prior art single fan units 100 were generally custom built for the particular purpose, the present invention could be implemented on a single type of fan unit 200. In alternative embodiments, there might be several fan units 200 having different sizes and/or powers (both input and output). The different fan units 200 could be used in a single air-handling system or each air-handling system would have only one type of fan unit 200. Even when the smaller fan units 200 are custom made, the cost of producing multiple fan units 200 for a particular project is almost always less that the cost of producing a single large prior art fan unit 100 for the same project. This may be because of the difficulties of producing the larger components and/or the cost of obtaining the larger components necessary for the single large prior art fan unit 100. This cost savings also extends to the cost of producing a smaller air-handling compartment 202.

In one preferred embodiment of the invention, the fan units 200 are modular such that the system is "plug and play." Such modular units may be implemented by including structure for interlocking on the exterior of the fan units 200 themselves. Alternatively, such modular units may be implemented by using separate structure for interlocking the fan units 200. In still another alternative embodiment, such modular units may be implemented by using a grid system into which the fan units 200 may be placed.

Reduced Operating Expenses

The fan array fan section in the air-handling system of the present invention preferably are less expensive to operate than prior art air-handling systems because of greater flexibility of control and fine tuning to the operating requirements of the structure. Also, by using smaller higher speed fan units 200 that require less low frequency noise control and less static resistance to flow.

Increased Efficiency

The fan array fan section in the air-handling system of the present invention preferably is more efficient than prior art air-handling systems because each small fan unit 200 can run at peak efficiency. The system could turn individual fan units 200 on and off to prevent inefficient use of particular fan units 200. It should be noted that an array controller 300 could be used to control the fan units 200. As set forth above, the array controller 300 turns off certain fan units 200 and runs the remaining fan units 200 at peak efficiency.

Redundancy

Multiple fan units 200 add to the redundancy of the system. If a single fan unit 200 breaks down, there will still be cooling. The array controller 300 may take disabled fan units 200 into consideration such that there is no noticeable depreciation in cooling or air flow rate. This feature may also be useful during maintenance as the array controller 300 may turn off fan units 200 that are to be maintained offline with no noticeable depreciation in cooling or air flow rate. A bypass feature, discussed below, uses and enhances the redundancy of the system.

Sound Attenuation Advantages

The high frequency sound of the small fan units 200 is easier to attenuate than the low frequency sound of the large fan unit. Because the fan wall has less low frequency sound energy, shorter less costly sound traps are needed to attenuate the higher frequency sound produced by the plurality of small fan units 200 than the low frequency sound produced by the single large fan unit 100. The plurality of fan units 200 will each operate in a manner such that acoustic waves from each unit will interact to cancel sound at certain frequencies thus creating a quieter operating unit than prior art systems.

Reduced Vibration

The multiple fan units 200 of the present invention have smaller wheels with lower mass and create less force due to residual unbalance thus causing less vibration than the large fan unit. The overall vibration of multiple fan units 200 will transmit less energy to a structure since individual fans will tend to cancel each other due to slight differences in phase. Each fan unit 200 of the multiple fan units 200 manage a smaller percentage of the total air handling requirement and thus produce less turbulence in the air stream and substantially less vibration.

Alternative Embodiments

As mentioned, in one preferred embodiment of the invention, the fan units 200 are modular such that the system is "plug and play." Such modular units may be implemented by including structure for interlocking on the exterior of the fan units 200 themselves. Alternatively, such modular units may be implemented by using separate structure for interlocking the fan units 200. In still another alternative embodiment, such modular units may be implemented by using a grid system into which the fan units 200 may be placed.

FIG. 16 shows an embodiment using an exemplary grid system 230 into which the fan units 200 may be placed. In this embodiment the grid may be positioned and/or built within the air-handling compartment 202. The fan units 200 may then be positioned into the grid openings. One advantage of this configuration is that individual fan units 200 may be easily removed, maintained, and/or replaced. This embodiment uses an exemplary unique motor mount 232 that supports the motor 208 without interfering with air flow therearound. As shown, this exemplary motor mount 232 has a plurality of arms that mount around the fan inlet cone 204. It should be noted that the dimensions of the grid are meant to be exemplary. The grid may be constructed taking into consideration that the fan units 200 in the present invention may be spaced with as little as 20% of a fan wheel diameter between the fan units 200.

FIG. 17 shows an embodiment using either a grid system or modular units 240 using separate structure (not shown) for interlocking the fan units 200. In this

exemplary embodiment, each of the fan units 200 are mounted on a more traditional motor mount 242 within its own fan unit chamber 244. In one preferred embodiment, the fan unit 200 and motor mount 242 are preferably suspended within their own fan unit chamber 244 such that there is an air relief passage 246 therebelow. This air relieve passage 246 tends to improve air flow around the fan units 200.

The fan unit chambers 244 shown in FIG. 17 may include one or more interior surface lined with an acoustically absorptive material or "insulation surface" 248. Similarly, the fan unit cells 244' shown in FIGS. 18 -23 may include one or more interior surface made from an acoustically absorptive material or "insulation surface" 248. Going against conventional industry wisdom that surfaces cannot be placed in close proximity with the fan units 200, the present invention places one or more insulation surfaces 248 at least partially around each fan unit 200 without disrupting air flow. The insulation surfaces 248 may include one or more of the sides, top, bottom, front, or back. Exemplary types of insulation include, but are not limited to traditional insulation board (such as that made from inorganic glass fibers (fiberglass) alone or with a factory-applied foil-scrim-kraft (FSK) facing or a factory-applied all service jacket (ASJ)) or alternative insulation such as open cell foam such as that disclosed in U.S. Patent Application No.10/606,435, which is assigned to the assignee of the present invention, and which the disclosure of which is hereby incorporated by reference herein. Together, the insulation surfaces 248 of the fan unit chambers/cells 244, 244' tend to function as a coplanar silencer. Some of the benefits of using the coplanar silencer include (1) no added airway length for splitters, (2) no pressure drop, and/or (3) relatively low cost. The acoustic advantages of this and other embodiments make the present invention ideal for use in concert halls, lecture halls, performing arts centers, libraries, hospitals, and other applications that are acoustically sensitive.

FIGS. 18 - 23 show an exemplary insulated grid system or modular unit system interior surfaces are made from acoustically absorptive material or "insulation surface" 248. In this embodiment, each fan unit cell 244' preferably has a sturdy frame 250 that supports the insulation surfaces 248. In one preferred embodiment the frame would form only the edges of a cube-shaped fan unit cell 244' and the insulation

surfaces 248 would form the sides (e.g. top, bottom, and/or sides) of the cube-shaped fan unit cell 244'. In alternative preferred embodiments, the frame may include additional structure or braces for support and/or strength. Together, the insulation surfaces 248 of the fan unit cells 244' tend to function as a coplanar silencer. This is shown graphically in FIGS. 19-23 where the coplanar silencer (formed by the insulation surfaces 248) reduces the sound wave reaction as the sound waves travel through the insulation surfaces 248. For example, in FIG. 19, the central fan unit 200a is loudest in its own fan unit cell 244'. As the sound of the fan spreads radially, it at least partially dissipates as it passes through the surrounding insulation surfaces 248. This is shown graphically as the sound wave circles being darkest in the central fan unit cell 244' and lighter in the surrounding fan unit cells 244'. The result is that the sound from the central fan unit 200a that eventually emanates from the system is softer than sound that would emanate from a system without the coplanar silencer. In FIG. 20, the first side fan unit 200b is loudest in its own fan unit cell 244'. As the sound of the fan spreads radially, it at least partially dissipates as it passes through the surrounding insulation surfaces 248. This is shown graphically as the sound wave circles being darkest in the central fan unit cell 244', lighter in the surrounding fan unit cells 244', and still lighter in fan unit cells 244' more distant from the originating fan unit 200b. The result is that the sound from the fan unit 200b that eventually emanates from the system is softer than sound that would emanate from a system without the coplanar silencer. FIG. 21 shows the first side fan unit 200b, a second side fan unit 200c, and their respective sound waves. As shown graphically in FIG. 24, another principle of the present invention is that as the sound waves interact, there is a degree of wave cancellation such that the waves are self-extinguishing. FIG. 24 shows wave A and an opposite wave B that are opposites and therefore interact to form a flat wave A + B. If waves are not exactly opposite, then the combined wave will not be flat, but would have some wave cancellation. This is a basic wave principle of which the present invention is able to avail itself. The result of wave cancellation is that the sound from the fan units 200b and 200c that eventually emanates from the system is softer than sound that would emanate from a system without the coplanar silencer. FIG. 22 emphasizes a first

corner fan unit 200d and its wave pattern. FIG. 23 emphasizes both the first corner fan unit 200d and a second corner fan unit 200b and their respective wave patterns. The analysis of FIGS. 22 and 23 would be similar to that of FIGS. 20 and 21 respectively. It should be noted that in the preferred embodiment, more than two fans might be running simultaneously and all the running fans would have wave patterns. The wave patterns of all the running fans would be able to take advantage of both the dissipation (as they pass though surrounding insulation surfaces 248) and wave cancellation of the coplanar silencer.

Although FIG. 17 shows the discharge plenum 210 positioned within the fan unit chambers 244, alternative embodiments of fan unit chambers 244 could enclose the inlet plenum 212, or at least partially enclose both the inlet plenum 212 and the discharge plenum 210. Still other alternative embodiments of fan unit chambers 244 may have grid or wire surfaces (that increase the safety of the present invention) or be open (that would reduce costs).

Bypass Feature

Multiple fan units enable the array to operate at a range of flow rates from full flow to partial flow where each fan contributes 1/N air flow (where N equals the number of fans). Most direct drive fan systems operate at speeds other than full synchronous motor speed in order to match the heating or cooling requirements of the structure. Speed control is normally maintained using variable frequency drives. Since variable frequency drives are electronic devices, each drive operating within an air handling structure has a certain probability of failure. In a traditional air handling system, if the VFD fails the air handler will either shut down or be operated at full synchronous speed of the motor in what is known as bypass mode. In traditional systems fan units in the air handler have to be throttled back through some mechanical means in order to limit pressure and flow to meet the building requirements. Mechanical throttling in bypass mode on traditional systems creates excessive noise and reduces fan efficiency. The present invention overcomes this problem by allowing for a change in the fan array output by turning certain fans off to meet the design point.

The array can be tailored to meet the flow and pressure requirement without the need for mechanical throttling and subsequent added noise and reduction in efficiency.

Dampeners

FIG. 25 shows an array of dampeners 260 that may be positioned either in front of or behind the fan units 200 to at least partially prevent back drafts. In the shown exemplary embodiment, the dampeners 260 include a plurality of plates, each plate positioned on its own pivot. In the shown exemplary embodiment, the plurality of plates slightly overlap each other. The shown embodiment is constructed such that when air is flowing through the fan units 200, the plates are in the open position and when the air stops, gravity pulls the plates into the closed position. Preferably, each of the dampeners 260 operates independently such that if some of the fan units 200 are ON and some of the fan units 200 are OFF, the dampeners 260 can open or close accordingly. Although shown as a simple mechanical embodiment, alternative embodiments could include structure that is controlled electronically and/or remotely from the dampeners 260.

It should be noted that FIG. 4 shows a 4x6 fan array fan section in the air-handling system having twenty-four fan units 200, FIG. 5 shows a 5x5 fan array fan section in the air-handling system having twenty-five fan units 200, FIG. 6 shows a 3x4 fan array fan section in the air-handling system having twelve fan units 200, FIG. 7 shows a 3x3 fan array fan section in the air-handling system having nine fan units 200, and FIG. 8 shows a 3x1 fan array fan section in the air-handling system having three fan units 200. It should be noted that the array may be of any size or dimension of more than two fan units 200. It should be noted that although the fan units 200 may be arranged in a single plane (as shown in FIG. 3), an alternative array configuration could contain a plurality of fan units 200 that are arranged in a staggered configuration (as shown in FIG. 15) in multiple planes. It should be noted that cooling coils (not shown) could be added to the system either upstream or downstream of the fan units 200. It

should be noted that, although shown upstream from the fan units 200, the filter bank 122, 222 could be downstream.

It should be noted that an alternative embodiment would use a horizontally arranged fan array. In other words, the embodiments shown in FIGS. 3-15 could be used horizontally or vertically or in any direction perpendicular to the direction of air flow. For example, if a vertical portion of air duct is functioning as the air-handling compartment 202, the fan array may be arranged horizontally. This embodiment would be particularly practical in an air handling compartment for a return air shaft.

It should be noted that the fan section 214 may be any portion of the airway path 220 in which the fan units 200 are positioned. For example, the fan units 200 may be situated in the discharge plenum 210 (as shown), the inlet plenum 212, or partially within the inlet plenum 212 and partially within the discharge plenum 210. It should also be noted that the air-handling compartment 202 may be a section of air duct.

It should be noted that many of the features and properties associated with the fan unit chambers 244 (FIG. 17) would be identical to or similar to properties of the fan unit cells 244' (FIGS. 18 -23).

The terms and expressions that have been employed in the foregoing specification are used as terms of description and not of limitation, and are not intended to exclude equivalents of the features shown and described or portions of them. The scope of the invention is defined and limited only by the claims that follow.

What is claimed is:

1. A fan array fan section in air-handling compartment which is part of an air-handling for conditioning the air of a structure, said fan array fan section comprising: (a) a plurality of fans units; (b) said plurality of fan units arranged in a fan array; (c) each of said plurality of fans units positioned within a fan unit chamber/cell; (d) each fan unit chamber/cell having at least one acoustically absorptive insulation surfaces; (e) said insulation surface of said fan unit chamber/cells together form a co-planar silencer; wherein (f) each of said plurality of fan units is suspended within a fan unit chamber/cell such that there is an air relief passage which is isolated from the remainder of said fan unit chamber/cells so that a portion of the air drawn by said fan unit passed through said air relief passage without passing through said fan unit.

2. A fan array section in an air-handling compartment which is part of an air-handling system for conditioning the air of a structure, said fan array fan section comprising: (a) a plurality of fans units; (b) said plurality of fan units arranged in a fan array; (c) each of said plurality of fans units positioned within a fan unit chamber/cell which substantially surrounds said fan unit; (d) each fan unit chamber/cell having at least one acoustically absorptive insulation surfaces; (e) said each fan unit chamber/cell being arranged within an air-handling compartment such that the air from all of said fan units is mixed before it enters the structure; wherein; and (f) each of said plurality of fan units is suspended within a fan unit chamber/cell such that there is an air passageway therebelow.

3. A fan array fan section in air-handling system comprising: (a) at least three fan units; (b) each of said at least three fan units positioned within a fan unit chamber/cell which substantially surrounds said fan unit; (c) each fan unit's chamber/cell having at least one insulating surface; (d) said at least three fan units arranged in a fan array; (e) wherein said fan unit chamber/cell is cube-shaped; (f) said cube having a frame which forms four edges of said cube; and (g) said insulation surfaces are independent of said frame and from the portion of said cube not formed by said frame.

4. The fan array section in a air-handling system of claim 3, said insulation surface of said fan unit chamber/cell together forming a co-planar silencer

5. The fan array section in a air-handling system of claim 3, wherein each said insulation surface is acoustically absorptive material.

6. The fan array section in a air-handling system of claim 3, wherein sound waves from said fan units passing through said insulation surfaces are at least partially dissipated as they pass therethrough.

7. The fan array fan section in air-handling system of claim 3, wherein each of said at least three fan units is suspended within a fan unit chamber/cell such that there is an air relief passage therebelow.

8. The fan array fan section in air-handling system of claim 3, wherein said at least three fan units are a plurality of fan units arranged in a fan array configuration selected from the group consisting of: (a) a true array configuration; (b) a spaced pattern array configuration; (c) a checker board array configuration; (d) rows slightly offset array configuration; (e) columns slightly offset array configuration; and (f) a staggered array configuration.

9. The fan array fan section in an air-handling system of claim 3, wherein said at least three fan units are plenum fans including at least two vertically arranged fan units.

10. The fan array fan section in an air-handling system of claim 3, wherein each of said at least three fan units is mounted in a grid system.

11. The fan array fan section in an air-handling system of claim 3, said fan unit chamber/cells being modular units for in a modular unit system.