

Description

This invention relates to a disc chucking mechanism or unit for supporting a disc loaded on a disc table, such as an optical disc or a magneto-optical disc, for rotation in unison with the disc table.

There is proposed a disc-shaped recording medium, such as an optical disc or a magneto-optical disc, and a disc apparatus for recording and/or reproducing information signals or from the disc-shaped recording medium.

With such disc apparatus the disc-shaped recording medium need to be rotated with the signal recording surface of the disc-shaped recording medium facing a pickup device adapted for recording and/or reading information signals on or from the signal recording surface. To this end, the disc apparatus includes a disc chucking unit adapted for holding the disc-shaped recording medium at a predetermined position and adapted for being rotated by a driving unit.

Referring to Fig. 1, the disc chucking unit includes a disc table 104 mounted on a driving shaft 101a of an electric motor 101 as driving means by means of a stop ring 102 and mounting a disc-shaped recording medium 103 thereon, and a chucking plate 105 mounted in opposition to the disc table 104 and adapted for clamping the disc-shaped recording medium 103 in cooperation with the disc table 104.

The electric motor 101 is mounted on a motor supporting plate 106 and supported in this state within an outer casing 107 of the disc apparatus. The motor supporting plate 106 is supported in a floating manner with respect to a chassis 108 provided within the outer casing 109 by the interposition of a plurality of elastic elements 109 so that vibrations or impacts applied to the disc apparatus will be scarcely propagated to the motor 101.

The stop ring 102 has a shaft 102a fitted to and supported by the driving shaft 101a and a flange 102b disposed at the distal side of the driving shaft 101a. The disc table 104 is fitted to and supported the disc table 104. Thus the lower surface of the flange 102b facing the motor 101 is confronted by a central portion on the upper surface of the disc table 104.

A center ring 110 is interposed between the flange 102b and the disc table 104. This center ring 110 has a central through-hole 110a through which the shaft 102a of the stop ring 102 is fitted so that the center ring 110 is supported for movement along the axis of the driving shaft 101a between the flange 102b and the disc table 104. A centering spring 111 is installed between the center ring 110 and the disc table 104 for thrusting the center ring 110 onto the lower surface of the flange 102b.

The outer periphery 110a of the center ring 110 is substantially conical, that is, it is tapered towards above or towards the distal side of the driving shaft 101a. The outer periphery 110 a is opened to outside

as it surrounds the outer periphery of the flange 102a.

The chucking plate 105 is formed as a disc and supported on the side of the lower surface of a cover member 112 mounted to the outer casing 107 so as to face the disc table 104. As shown by an arrow a in Fig. 1, the cover member 112 is mounted on and supported by the outer casing 107 so as to be movable in a direction towards and away from the disc table 104.

Above the chucking plate 105, there is mounted a supporting member 114 having a disc-shaped flange 113 having a lesser diameter than the chucking plate 105. The supporting member 114 is disposed within a housing section 115 formed integrally with the cover member 112. Below the housing section 115, a holding plate 116 having a through-hole 116a which is of a diameter larger than the supporting member 114 and lesser than the flange 113 and through which is passed the supporting member 114 is mounted for closing the housing section 115. Thus the chucking plate 105 may be rotatable while being movable vertically within the extent of the housing section 115.

A toroidal-shaped permanent magnet 117 is mounted at the center of the chucking plate 105, so that, when the chucking plate 105 and the disc table 104 draw close to each other, they attract each other magnetically.

A ball-shaped projection 118 is formed at the center of the lower surface of the chucking plate 105. A conical recess 102c for receiving the projection 118 is provided at the center on the upper surface of the flange 102b confronted by the chucking plate 105. When the chucking plate 105 is thrust by the disc table 104, the ball-shaped projection 118 fits into the conical recess 102c for centering the chucking plate 105 with respect to the disc table 104.

With the above described disc chucking mechanism, the disc-shaped recording medium 103 is placed on the disc table 104, with the outer periphery 110a of the center ring 110 abutting on the inner periphery of a central chucking opening 103a formed in the disc-shaped recording medium 103. When the cover member 112 is shifted towards the disc table 104, the recording medium 103 is clamped between the chucking plate 105 and the disc table 104, as shown in Fig. 2. Since the outer periphery 110a of the center ring 110 is conical, and the center ring 110 is biased upwards by the centering spring 111, the inner periphery of the chucking opening 103a is thrust by the outer periphery 110a for centering the disc-shaped recording medium 103 relative to the driving shaft 101a.

When the driving shaft 101a is driven into rotation by the electric motor 101, the disc-shaped recording medium 103 is rotated in unison with the disc table 104 and the chucking plate 105.

Meanwhile, with the above described disc chucking unit, when the disc-shaped recording medium 103

is loaded while it is clamped by the disc table 104 and the chucking plate 105, the chucking plate 105 is floatingly supported relative to the chassis 108 by means of the disc-shaped recording medium 103, disc table 104, electric motor 101 and the motor supporting plate 106. Thus the supporting member 114 may be moved within the extent shown by gaps m1, m2 and m3 relative to the chassis 108 under the impact or vibrations propagated from outside.

With such disc chucking unit, for realizing smooth rotation of the disc-shaped recording medium 103, it is necessary to prevent the supporting member 114 from abutting on the cover member 112, even if the supporting member 114 should be moved relative to the chassis 108. Therefore, the gap between the housing section 115 and the supporting member 114 should be larger than is allowed by the floating support between the chassis 108 and the motor supporting plate 106, that is, the gaps m1, m2 and m3 in Fig. 2 should be present between the section 115 and the supporting member 114.

That is, with the disc apparatus making use of the above described disc chucking unit, the central portion of the cover member 112 need to be bulged upwards to provide the supporting member housing section 115, so that the apparatus cannot be reduced in size or thickness.

In addition, the above described disc chucking unit can be assembled or fabricated only by a pains taking operation because of the larger number of component parts making up the disc chucking unit.

FR-A-2 078 456, on which the preamble of appendant claim 1 is based, discloses a table for receiving a disc with the central opening of the disc over a hub. A plurality of elastically movable elements radially protrude from the hub for holding a disc in place on the table. The elastically movable elements may be balls biased outwardly by a piece of rubber, or alternatively may be tongue-shaped spring-pieces formed integrally with the hub.

It is an object of the present invention to provide a disc chucking mechanism or unit which may be assembled and fabricated easily and which contributes to reduction in size and thickness of a disc apparatus.

According to the present invention there is provided a disc chucking mechanism comprising

a disc table adapted to receive a disc having a central opening and to be rotationally driven by rotational driving means for rotating a disc received on the disc table about an axis of rotation,

a fitting member arranged at the centre of the disc table adapted to fit into the central opening of a disc received on the disc table, and

a plurality of ball-shaped members mounted so as to be radially reciprocable on the fitting member, the ball-shaped members being biased outwardly of the fitting member by an elastic element for engaging a disc received on the disc table and thrusting

it towards the disc table,

the fitting member further comprising a plurality of deflectable portions which are elastically radially displaceable for engaging the central opening of a disc received on the disc table for centring the disc with respect to the axis of rotation of the disc table.

With the above described disc chucking unit, when the disc is placed on the disc table, the ball-shaped members reciprocally mounted in a fitting member arranged centrally of the disc table and biased outwards by an elastic element abut on the upper edge of the inner lateral surface of the central opening of the disc, so that the disc is elastically thrust and supported towards the disc table so as to be loaded and retained on the disc table.

The present invention will be further described hereinafter with reference to the following description of an exemplary embodiment and the accompanying drawings, in which:

Fig. 1 is a longitudinal cross-section showing a conventional disc chucking unit and a disc apparatus employing the unit;

Fig. 2 is an enlarged longitudinal cross-section showing the state in which a disc is loaded on the disc chucking unit shown in Fig. 1;

Fig. 3 is an enlarged exploded perspective view showing the construction of a disc chucking unit according to the present invention;

Fig. 4 is an enlarged longitudinal cross-section showing the construction of the disc chucking unit shown in Fig. 3;

Fig. 5 is an enlarged plan view showing the construction of the chucking unit shown in Fig. 3;

Fig. 6 is an exploded perspective view of a disc apparatus constructed by using the disc chucking unit shown in Fig. 3;

Fig. 7 is an enlarged longitudinal cross-section showing a disc being loaded on the disc chucking unit of Fig. 3;

Fig. 8 is an enlarged longitudinal cross-section showing a disc loaded on the disc chucking unit of Fig. 3; and

Fig. 9 is a perspective view showing the disc apparatus of Fig. 8 in its entirety.

As shown in Figs. 3 to 5, the disc chucking unit of the present invention is provided with a disc table 3 fitted to a driving shaft 1a of an electric motor 1 of the rotating driving unit and on which is placed a disc 2 as the recording medium.

As shown in Fig. 6, the electric motor 1, secured by set screws 5 to a motor supporting plate 4 constituting a disc apparatus to which the disc chucking unit is applied, is supported within an outer casing 6 of the disc apparatus. For preventing vibrations or impacts applied to the other casing from being propagated to the motor 1, the motor supporting plate 4 is floatingly supported with respect to a chassis 7 provided in the outer casing 6 by the interposition of a plurality of re-

silent elements 8 formed of rubber or otherwise designed as springs.

Although not shown, an optical pickup device or a magnetic head device adapted for recording and/or reading information signals on or from the disc 2 is mounted on the motor supporting plate 4 for movement towards and away from the motor 1.

As shown in Figs. 3 and 4, the disc table 3 is formed unitarily from synthetic resin or the like material, and is constituted by a substantially cylindrical shaft 3a and a substantially disc-shaped plate section 3b on the motor side lower extremity of the shaft 3a. A supporting opening 3c into which fits the driving shaft 1a is formed at the center of the shaft 3a. A disc abutment section 3d in the form of a circumferential rib is provided on the outer upper periphery of the plate section 3b.

On the inner surface of the supporting opening 3c, a supporting metal ring 9 is interposed between the inner surface of the opening 3c and the driving shaft 1a for positively supporting the disc table 3 by the driving shaft 1a. The supporting metal ring 9 is mounted on the disc table 3 such as by so-called insert molding.

On the upper end of the shank 3a, corresponding to the distal end of the driving shaft 1a, a fitting or centering unit 10 is mounted, as shown in Figs. 3 to 5. This centering unit 10 is constituted by a cap 11 fitted into a central chucking opening 2a in the disc 2, an O-ring 12 inserted into the cap 11 as an elastic member, and a plurality of, wherein three, steel balls 13 as ball-shaped members.

The cap 11 is formed unitarily of synthetic resin, and is constituted by a supporting section 11a fitted to the outer periphery of a shaft 3a of the disc table 3, a flange-shaped ceiling plate section 11b provided on the upper peripheral side of the supporting section 11a and a side wall 11c depending from the outer rim of the ceiling plate section 11b towards the disc table 3. The outer peripheral surface of the side wall section 11 is substantially conical so that the diameter thereof is decreased gradually towards the distal end of the driving shaft 1a. The side wall section 11c is so contoured that the diameter of the outer peripheral surface thereof at the lower end thereof towards the disc table 3 as shown by an arrow D_1 in Fig. 5 is slightly lesser than the central opening 2a. An outwardly extending flange section 11d is formed at the lower end of the side wall 11c.

Three pairs of slits 14 are formed for extending from the ceiling plate section 11b to the flange section 11d by way of the side wall section 11c at an angular interval of approximately 120° . The zone delimited between each paired slits 14, 14 is formed as a deflectible portion 15 which may be deformed elastically in a direction towards and away from the supporting section 11a. These deflectible portions 15 are so designed that the diameter D_2 of a circumscribed circle

containing the outer periphery of the deflectible portions 15 at the lower end of the side wall section 11c is slightly larger than the central opening 2a.

The cap 11 is so designed that, when the cap 11 is mounted on the disc table 3, the flange section 11d fits into a circular groove 3e formed on the upper surface of the plate section 3b. Hence, each deflectible portion 15 has its distal side forming a part of the flange 11d fitted into the groove 3e so as to be deflected within the extent that the distal side is moved within the groove 3e.

The O ring 12, accommodated within the cap 11, is formed of synthetic rubber or the like material, and is arranged for surrounding the supporting section 11a.

The steel balls 13 are kept in contact with the upper outer periphery of the O ring 12 and supported in this state by being clamped between the O ring 12 and the cap 11. The steel balls are arranged between the deflectible portions 15, 15 at an angular interval of approximately 120° .

The cap 11 is formed with through-holes 16, in register with these steel balls 13, for opening the steel balls to outside. These through-holes 16 are formed as slits extending from the ceiling plate section 11b to the side wall section 11c. Thus the through-holes 16 are extended radially in the ceiling plate section 11b and vertically in the side wall section 11c from the upper side to the lower side thereof. Each through-hole 16 is of a width lesser than the diameter of each steel ball 13 in the ceiling plate section 11b and a width approximately equal to the diameter of the steel ball 13 in the side wall section 11c. On the lower surface of the ceiling plate section 11b, there are provided a pair of inclined sections 16a on both sides of each through-hole 16 which are tapered from each through-hole 16 towards below. The lower side of the inclined section 16a is contiguous to the inner lateral edge of the through-hole 16 in the side wall section 11c.

Each steel ball 13 abuts on the paired inclined sections 16a in the cap 11 so as to be clamped between these inclined sections 16a. Thus the steel balls 13 may be moved along the paired inclined sections 16a radially of the ceiling plate section 11b as shown by arrows A_1 and A_3 in Fig. 3.

Three retention lugs 17a are formed in the flange 11d of the cap 11 in register with the through-holes 16. Each retention lug 17 has its distal end 17a abutted on the associated steel ball 13 to control the movement of the associated steel ball 13 towards the outside of the cap 11. The distal end 17a is formed as a concave spherical surface mating with the spherical surface of the steel ball 13.

Each steel ball 13 is supported and positioned by the O ring 12, the paired inclined sections 16a and the distal end 17a of the retention lug 17. Each steel ball 13 thus positioned is protruded via through-hole 16

towards the outside of the side wall 11c in an amount corresponding to about 30 to 50 percent of its diameter. The steel balls 13 are positioned at this time so that the diameter D_3 (Fig. 5) of a circumscribed circle containing the outwardly protruded portions of the steel balls 13 is larger than the central opening 2a. When moved towards the inside of the cap 11, that is towards the shaft 11a, the steel balls 13 thrust the O ring 12 for deforming the O ring elastically, as a result of which the steel balls 13 are biased in the direction of being protruded out of the cap 11.

Referring to Fig. 6, a cover member 18 of a size to cover the disc 2 is rotatably supported by a pivot shaft 18a on the outer casing 6 of the disc apparatus for pivotal movement towards and away from the disc table 3 as shown by an arrow B in Fig. 6.

When loading the disc 2 in the above described disc chucking unit, the centering unit 10 is fitted into the central opening 2a of the disc 2. At this time, the inner lateral surface of the central opening 2a is abutted by the periphery of the steel balls 13 protruded outwards from the side wall section 11c, as shown in Fig. 7, so that these steel balls 13 are shifted towards the inner side of the cap 11, as shown by an arrow C in Fig. 7, to cause elastic deformation of the O ring 12.

When the disc 2 is moved further to approach to the disc table 3 and placed ultimately thereon, the inner lateral surface of the central opening 2a of the disc 2 is positioned between the peripheral portions of the steel balls 13 protruded out of the side wall section 11c and the disc table 3, while the inner lateral surface of the opening 2a is kept in abutment with the steel balls 13, as shown in Fig. 8. At this time, each steel ball 13 abuts on the upper edge on the inner lateral surface of the central opening 2a. Thus the O ring 12 thrusts the upper edge on the inner lateral surface of the central opening 2a by means of the steel balls 13 in an obliquely downward direction as shown by an arrow D in Fig. 8. That is, the O ring 12 thrusts and supports the disc 12 with respect to the disc table 3, while centering the disc 12 with respect to the driving shaft 1a.

At this time, each deflectible portion 15 is abutted by the inner lateral surface of the central opening 2a so as to be elastically deflected towards the inside of the cap 11. Thus the deflectible portions 15 thrust the inner lateral surface of the central opening 2a for centering the disc 2 with respect to the driving shaft 1a.

When the disc 2 is positioned and loaded on the disc table 3 and the driving shaft 1a is driven into rotation by the motor 1, the disc 2 is rotated in unison with the disc table 3 to effect recording and/or readout of the information signals with the aid of the optical pickup device or the magnetic head device.

It will be noted that the cover member 18 constituting the disc apparatus making use of the disc chucking unit of the present invention may be designed as a flat plate, as shown in Fig. 9, and a display

unit 19 or an operating unit 20 including switches or pushbuttons may be easily mounted at a central portion on the upper surface of the cover member 18 in register with the disc table 3.

The disc chucking unit according to the present invention is not limited to the above described embodiment. For example, four or more steel balls 13 may be employed. In this case, four or more through-holes 16 may be provided in register with these steel balls 13. The number of the deflectible portions 15 may similarly be four or more, if so desired.

The elastic member of the disc chucking unit of the present invention need not be an O ring, but may be separate elastic elements associated with the ball-shaped members.

It will be seen from the foregoing that, with the disc chucking unit of the present invention, the ball-shaped members reciprocally mounted inside a fitting member arranged at the center of the disc table and biased outwards by the elastic element abut on the upper edge of the inner lateral surface of the central opening of the disc when the disc is placed on the disc table, so that the disc is thrust and supported elastically with respect to the disc table so as to be retained and loaded on the disc table.

Thus it is possible with the disc chucking unit of the present invention to load the disc on the disc table without employing a chucking plate which is mounted in opposition to the disc table.

Claims

1. A disc chucking mechanism comprising
 - a disc table (3) adapted to receive a disc (2) having a central opening (2a) and to be rotationally driven by rotational driving means (1) for rotating a disc (2) received on the disc table (3) about an axis of rotation,
 - a fitting member (11) arranged at the centre of said disc table (3) adapted to fit into the central opening (2a) of a disc (2) received on the disc table (3), and
 - a plurality of ball-shaped members (13) mounted so as to be radially reciprocable on said fitting member (11), said ball-shaped members (13) being biased outwardly of said fitting member (11) by an elastic element (12) for engaging a disc (2) received on said disc table (3) and thrusting it towards said disc table (3),
 - characterised by the fitting member (11) further comprising a plurality of deflectable portions (15) which are elastically radially displaceable for engaging the central opening (2a) of a disc (2) received on the disc table (3) for centring the disc (2) with respect to the axis of rotation of the disc table (3).



towards the outside of the side wall 11c in an amount corresponding to about 30 to 50 percent of its diameter. The steel balls 13 are positioned at this time so that the diameter D_3 (Fig. 5) of a circumscribed circle containing the outwardly protruded portions of the steel balls 13 is larger than the central opening 2a. When moved towards the inside of the cap 11, that is towards the shaft 11a, the steel balls 13 thrust the O ring 12 for deforming the O ring elastically, as a result of which the steel balls 13 are biased in the direction of being protruded out of the cap 11.

Referring to Fig. 6, a cover member 18 of a size to cover the disc 2 is rotatably supported by a pivot shaft 18a on the outer casing 6 of the disc apparatus for pivotal movement towards and away from the disc table 3 as shown by an arrow B in Fig. 6.

When loading the disc 2 in the above described disc chucking unit, the centering unit 10 is fitted into the central opening 2a of the disc 2. At this time, the inner lateral surface of the central opening 2a is abutted by the periphery of the steel balls 13 protruded outwards from the side wall section 11c, as shown in Fig. 7, so that these steel balls 13 are shifted towards the inner side of the cap 11, as shown by an arrow C in Fig. 7, to cause elastic deformation of the O ring 12.

When the disc 2 is moved further to approach to the disc table 3 and placed ultimately thereon, the inner lateral surface of the central opening 2a of the disc 2 is positioned between the peripheral portions of the steel balls 13 protruded out of the side wall section 11c and the disc table 3, while the inner lateral surface of the opening 2a is kept in abutment with the steel balls 13, as shown in Fig. 8. At this time, each steel ball 13 abuts on the upper edge on the inner lateral surface of the central opening 2a. Thus the O ring 12 thrusts the upper edge on the inner lateral surface of the central opening 2a by means of the steel balls 13 in an obliquely downward direction as shown by an arrow D in Fig. 8. That is, the O ring 12 thrusts and supports the disc 12 with respect to the disc table 3, while centering the disc 12 with respect to the driving shaft 1a.

At this time, each deflectible portion 15 is abutted by the inner lateral surface of the central opening 2a so as to be elastically deflected towards the inside of the cap 11. Thus the deflectible portions 15 thrust the inner lateral surface of the central opening 2a for centering the disc 2 with respect to the driving shaft 1a.

When the disc 2 is positioned and loaded on the disc table 3 and the driving shaft 1a is driven into rotation by the motor 1, the disc 2 is rotated in unison with the disc table 3 to effect recording and/or readout of the information signals with the aid of the optical pickup device or the magnetic head device.

It will be noted that the cover member 18 constituting the disc apparatus making use of the disc chucking unit of the present invention may be designed as a flat plate, as shown in Fig. 9, and a display

unit 19 or an operating unit 20 including switches or pushbuttons may be easily mounted at a central portion on the upper surface of the cover member 18 in register with the disc table 3.

The disc chucking unit according to the present invention is not limited to the above described embodiment. For example, four or more steel balls 13 may be employed. In this case, four or more through-holes 16 may be provided in register with these steel balls 13. The number of the deflectible portions 15 may similarly be four or more, if so desired.

The elastic member of the disc chucking unit of the present invention need not be an O ring, but may be separate elastic elements associated with the ball-shaped members.

It will be seen from the foregoing that, with the disc chucking unit of the present invention, the ball-shaped members reciprocally mounted inside a fitting member arranged at the center of the disc table and biased outwards by the elastic element abut on the upper edge of the inner lateral surface of the central opening of the disc when the disc is placed on the disc table, so that the disc is thrust and supported elastically with respect to the disc table so as to be retained and loaded on the disc table.

Thus it is possible with the disc chucking unit of the present invention to load the disc on the disc table without employing a chucking plate which is mounted in opposition to the disc table.

Claims

1. A disc chucking mechanism comprising
 - a disc table (3) adapted to receive a disc (2) having a central opening (2a) and to be rotationally driven by rotational driving means (1) for rotating a disc (2) received on the disc table (3) about an axis of rotation,
 - a fitting member (11) arranged at the centre of said disc table (3) adapted to fit into the central opening (2a) of a disc (2) received on the disc table (3), and
 - a plurality of ball-shaped members (13) mounted so as to be radially reciprocable on said fitting member (11), said ball-shaped members (13) being biased outwardly of said fitting member (11) by an elastic element (12) for engaging a disc (2) received on said disc table (3) and thrusting it towards said disc table (3),
 - characterised by the fitting member (11) further comprising a plurality of deflectable portions (15) which are elastically radially displaceable for engaging the central opening (2a) of a disc (2) received on the disc table (3) for centring the disc (2) with respect to the axis of rotation of the disc table (3).



2. A disc chucking mechanism according to claim 1 wherein said ball-shaped members (13) are spaced around said fitting member (11) equian-
gularly and equi-radially with respect to the cen-
tre of said fitting member (11).

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3. A disc chucking mechanism according to claim 2 wherein said elastic element (12) is ring-shaped and arranged within said fitting member (11) co-
axially with the axis of rotation of said disc table
(3) and said fitting member (11).

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4. A disc chucking mechanism according to claim 1, 2 or 3 wherein, when said ball-shaped members (13) are pushed outward of said fitting member (11) by said elastic element (12), said ball-shaped members (13) are arranged in said fitting mem-
ber (11) so that the diameter of a circle formed by
interconnecting the outer periphery of the ball-
shaped members (13) is larger than the diameter
of said central opening (2a) of said disc (2).

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5. A disc chucking mechanism according to claim 1, 2 or 3 wherein, when said ball-shaped members (13) are placed within the inside of said fitting member (11), said ball-shaped members (13) are arranged within said fitting member (11) so that the diameter of a circle formed by interconnecting the outer periphery of said ball-shaped members (13) is lesser than the diameter of said central opening (2a) of said disc (2).

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6. A disc chucking mechanism according to claim 1, 2 or 3 wherein said ball-shaped members (13) are adapted to thrust said disc (2) placed on said disc table (3) by the sum of a force acting parallel to the surface of said disc table (3) and a force acting along the central axis of rotation of said disc table (3), both of said forces being generated by the force of said elastic element (12) acting through said bolt shaped members (13) on the disc (2).

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Patentansprüche

1. Platteneinklemmechanismus mit

- einem Plattenteller (3) zum Auflegen einer Platte (2) mit einer zentralen Öffnung (2a), der von einem Drehantriebsmittel (1) drehend angetrieben wird, um eine auf dem Plattenteller (3) aufgelegte Platte (2) um eine Drehachse zu drehen,
- einem Einpaßelement (11), das in der Mitte dieses Plattentellers (3) angeordnet ist und in die zentrale Öffnung (2a) einer auf dem Plattenteller (3) aufgelegten Platte (2) paßt, sowie

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- mehreren kugelförmigen Elementen (13), die so angebracht sind, daß sie auf dem Einpaßelement (11) radial hin- und herbewegbar sind, wobei diese kugelförmigen Elemente (13) durch ein elastisches Element (12) zur Außenseite des Einpaßelements (11) hin vorbelastet sind, so daß sie eine auf dem Plattenteller (3) aufgelegte Platte (2) berühren und gegen den Plattenteller (3) drücken,

dadurch gekennzeichnet,

daß das Einpaßelement (11) weiterhin mehrere biegsame Abschnitte (15) umfaßt, die in radialer Richtung elastisch bewegbar sind, so daß sie in die zentrale Öffnung (2a) einer auf dem Plattenteller (3) aufgelegten Platte (2) eingreifen, um die Platte (2) in bezug auf die Drehachse des Plattentellers (3) zu zentrieren.

2. Platteneinklemmechanismus nach Anspruch 1, bei dem die kugelförmigen Elemente (13) in bezug auf die Mitte des Einpaßelements (11) in gleichem Winkelabstand und gleichem radialen Abstand um das Einpaßelement (11) herum angeordnet sind.

3. Platteneinklemmechanismus nach Anspruch 2, bei dem das elastische Element (12) ringförmig ist und im Einpaßelement (11) koaxial mit der Drehachse des Plattentellers (3) und dem Einpaßelement (11) angeordnet ist.

4. Platteneinklemmechanismus nach Anspruch 1, 2 oder 3, bei dem, wenn die kugelförmigen Elemente (13) durch das elastische Element (12) zur Außenseite des Einpaßelements (11) gedrückt werden, diese kugelförmigen Elemente (13) so in dem Einpaßelement (11) angeordnet sind, daß der Durchmesser eines Kreises, der die kugelförmigen Elemente (13) umschreibt, größer ist als der Durchmesser der zentralen Öffnung (2a) der Platte (2).

5. Platteneinklemmechanismus nach Anspruch 1, 2 oder 3, bei dem, wenn sich die kugelförmigen Elemente (13) innerhalb des Einpaßelements (11) befinden, diese kugelförmigen Elemente (13) so in dem Einpaßelement (11) angeordnet sind, daß der Durchmesser eines Kreises, der die kugelförmigen Elemente (13) umschreibt, kleiner ist als der Durchmesser der zentralen Öffnung (2a) der Platte (2).

6. Platteneinklemmechanismus nach Anspruch 1, 2 oder 3, bei dem die kugelförmigen Elemente (13) so ausgebildet sind, daß sie gegen eine auf dem Plattenteller (3) aufgelegte Platte (2) mit einer



Kraft drücken, die aus einer Kraft, die parallel zur Oberfläche des Plattentellers (3) wirkt, und einer Kraft, die in Richtung der Mittelachse der Drehung des Plattentellers (3) wirkt, resultiert, wobei diese beiden Kräfte durch die Kraft des elastischen Elements (12), das über die kugelförmigen Elemente (13) auf die Platte (2) einwirkt, erzeugt werden.

Revendications

1. Mécanisme de mandrinage de disque, comprenant :
 - une table à disque (3) adaptée à recevoir un disque (2) ayant une ouverture centrale (2a), et à être entraînée en rotation par des moyens d'entraînement en rotation (1) afin de faire tourner un disque (2) reçu sur la table à disque (3) autour d'un axe de rotation,
 - un élément d'emboîtement (11) agencé au centre de ladite table à disque (3) et adapté à s'emboîter dans l'ouverture centrale (2a) d'un disque (2) reçu sur la table à disque (3), et
 - une pluralité d'éléments (13) en forme de billes montés de manière à pouvoir se déplacer radialement en va-et-vient sur ledit élément d'emboîtement (11), lesdits éléments en forme de billes (13) étant repoussés vers l'extérieur dudit élément d'emboîtement (11) par un élément élastique (12) afin d'engager un disque (2) reçu sur ladite table à disque (3) et repousser celui-ci en direction de ladite table à disque (3), caractérisée par le fait que l'élément d'emboîtement (11) comprend en outre une pluralité de parties fléchissantes (15) qui sont déplaçables élastiquement radialement afin d'engager l'ouverture centrale (2a) d'un disque (2) reçu sur la table à disque (3) afin de centrer le disque (2) par rapport à l'axe de rotation de la table à disque (3).
2. Mécanisme de mandrinage de disque selon la revendication 1, dans lequel lesdits éléments en forme de billes (13) sont espacés autour dudit élément d'emboîtement (11) à angles égaux et sur des rayons égaux par rapport au centre dudit élément d'emboîtement (11).
3. Mécanisme de mandrinage de disque selon la revendication 2, dans lequel ledit élément élastique (12) a la forme d'un anneau et est agencé à l'intérieur dudit élément d'emboîtement (11) coaxialement avec l'axe de rotation de ladite table à disque (3) et dudit élément d'emboîtement (11).
4. Mécanisme de mandrinage de disque selon l'une quelconque des revendications 1, 2 ou 3, dans lequel, lorsque lesdits éléments en forme de billes (13) sont poussés vers l'extérieur dudit élément d'emboîtement (11) par ledit élément élastique (12), lesdits éléments en forme de billes (13) sont agencés dans ledit élément d'emboîtement (11) de telle manière que le diamètre d'un cercle formé en reliant la périphérie extérieure des éléments en forme de billes (13) est supérieur au diamètre de ladite ouverture centrale (2a) dudit disque (2).
5. Mécanisme de mandrinage de disque selon l'une quelconque des revendications 1, 2 ou 3, dans lequel lorsque lesdits éléments en forme de billes (13) sont placés à l'intérieur dudit élément d'emboîtement (11), lesdits éléments en forme de billes (13) sont agencés à l'intérieur dudit élément d'emboîtement de telle manière que le diamètre d'un cercle formé en reliant la périphérie extérieure desdits éléments en forme de billes (13) est inférieur au diamètre de ladite ouverture centrale (2a) dudit disque (2).
6. Mécanisme de mandrinage de disque selon l'une quelconque des revendications 1, 2 ou 3, dans lequel lesdits éléments en forme de billes (13) sont adaptés à repousser ledit disque (2) placé sur ladite table à disque (3) par la somme d'une force agissant parallèlement à la surface de ladite table à disque (3) et d'une force agissant le long de l'axe central de rotation de ladite table à disque (3), lesdites deux forces étant produites par la force dudit élément élastique (12) qui agit par l'intermédiaire desdits éléments en forme de billes (13) sur le disque (2).