

## HUMIDITY CONTROLLER APPARATUS

### TECHNICAL FIELD

The present invention generally relates to humidity controller apparatuses, and it  
5 relates more specifically to a humidity controller apparatus provided with a vapor  
compression refrigeration cycle refrigerant circuit.

### BACKGROUND ART

Conventionally, humidity controller apparatuses for regulating the humidity of air  
by making use of an adsorbent material have been known in the prior art. One such  
10 humidity controller apparatus is disclosed in Japanese Patent Application *Kokai*  
Publication No. 1996-189667.

This humidity controller apparatus has an air passage through which a stream of  
outdoor or room air flows and passes. And, a section of the piping of a refrigerant circuit  
which performs a vapor compression refrigeration cycle by the circulation of refrigerant is  
15 disposed in the inside of the air passage. This piping section in the air passage functions  
as an evaporator or as a condenser. In addition, disposed on the periphery of the piping in  
the air passage is a mesh member which is formed of a net-like member in which is  
encapsulated an adsorbent material.

The adsorbent material of the mesh container is cooled by a flow of refrigerant  
20 traveling through the piping section, when the piping section functions as an evaporator.  
By the action of such cooling, moisture present in the room or outdoor air is adsorbed via  
the net-like member. In addition, when the piping section functions as a condenser,  
moisture adsorbed onto the adsorbent material is desorbed when heated by the refrigerant  
flowing through the piping section. As a result, the adsorbent material is regenerated.

25 **PROBLEMS THAT INVENTION INTENDS TO SOLVE**

In the above-described conventional humidity controller apparatus, however, the piping as a means for cooling and heating air and the mesh container as a moisture-adsorbing means are formed and arranged separately. Therefore, in the conventional humidity controller apparatus, the mesh container and the piping are different members, and problems arise. The number of component parts increases. The entire structure of the apparatus becomes complicated. In addition, the humidity controller apparatus grows in size.

Additionally, the contact thermal resistance increases if the adsorbent material is brought into contact with only the piping or the like. This gives rise to another problem that desired cooling and heating effects are not provided.

With the above described problems in mind, the present invention was made. Accordingly, an object of the present invention is to dispose an adsorbent material without separate provision of an adsorption means such as a mesh container or the like.

#### DISCLOSURE OF INVENTION

As illustrated in Figure 1, a first invention is directed to a humidity controller apparatus, provided with an adsorbent material, for regulating the humidity of air by effecting moisture adsorption onto the adsorbent material and moisture desorption from the adsorbent material. In the humidity controller apparatus of the first invention, there is provided a refrigerant circuit (1) which comprises a first heat exchanger (3) and a second heat exchanger (5); which performs a vapor compression refrigeration cycle by the circulation of a refrigerant therethrough; and which alternately effects refrigerant condensation and evaporation in the first and second heat exchangers (3, 5). In addition, the adsorbent material is supported on at least a surface of the first heat exchanger (3).

In the first invention, it is arranged that the adsorbent material is supported on the surface of the first heat exchanger (3). Accordingly, the adsorbent material is cooled by

the evaporation of the refrigerant in the first heat exchanger (3). As a result, heat of adsorption is absorbed.

In addition, the adsorbent material is heated by the condensation of the refrigerant in the first heat exchanger (3). As a result, the adsorbent material is regenerated.

5 A second invention provides a humidity controller apparatus according to the first invention which comprises a moisture absorbing means for switching refrigerant circulation in the refrigerant circuit (1) and air distribution so that refrigerant condensation takes place in the second heat exchanger (5) while, simultaneously, refrigerant evaporation takes place in the first heat exchanger (3), whereby moisture present in a stream of air  
10 flowing through the first heat exchanger (3) is adsorbed by the adsorbent. The humidity controller apparatus of the second invention further comprises a moisture releasing means for switching refrigerant circulation in the refrigerant circuit (1) and air distribution so that refrigerant condensation takes place in the first heat exchanger (3) while, simultaneously, refrigerant evaporation takes place in the second heat exchanger (5), whereby the  
15 adsorbent material is regenerated by the release of moisture therefrom to a stream of air flowing through the first heat exchanger (3).

In the second invention, the refrigerant is evaporated in the first heat exchanger (3) by the moisture absorbing means, and the adsorbent material is cooled. As the result of this, heat of adsorption is absorbed into the refrigerant. In addition, the refrigerant is  
20 condensed in the first heat exchanger (3) by the moisture releasing means, and the adsorbent material is heated. As the result of this, the adsorbent material is regenerated.

A third invention provides a humidity controller apparatus according to the first invention, in which the adsorbent material is supported on a surface of the first heat exchanger (3) and a surface of the second heat exchanger (5) and a dehumidifier means  
25 (80) for switching refrigerant circulation in the refrigerant circuit (1) and air distribution is

provided so that moisture in a stream of air flowing through a refrigerant evaporating heat exchanger (3, 5) is adsorbed by the adsorbent material of the refrigerant evaporating heat exchanger (3, 5) while, simultaneously, the adsorbent material of a refrigerant condensing heat exchanger (5, 3) is regenerated by the release of moisture therefrom to a stream of air  
5 flowing through the refrigerant condensing heat exchanger (5, 3), whereby the stream of air dehumidified by the adsorbent material of the refrigerant evaporating heat exchanger (3, 5) is supplied to an indoor space.

In the third invention, the dehumidifier means (80) switches refrigerant circulation in the refrigerant circuit (1) and air distribution, whereby air dehumidification takes place  
10 in one heat exchanger (i.e., the heat exchanger (3, 5)) and adsorbent-material regeneration takes place in the other heat exchanger (i.e., the heat exchanger (5, 3)). As the result of this, dehumidification is carried out without interruption.

A fourth invention provides a humidity controller apparatus according to the first invention, in which the adsorbent material is supported on a surface of the first heat  
15 exchanger (3) and a surface of the second heat exchanger (5). In addition, the humidity controller apparatus of the fourth invention further comprises a humidifier means (81) for switching refrigerant circulation in the refrigerant circuit (1) and air distribution is provided so that moisture in a stream of air flowing through a refrigerant evaporating heat exchanger (3, 5) is adsorbed by the adsorbent material of the refrigerant evaporating heat  
20 exchanger (3, 5) while, simultaneously, the adsorbent material of a refrigerant condensing heat exchanger (5, 3) is regenerated by the release of moisture therefrom to a stream of air flowing through the refrigerant condensing heat exchanger (5, 3), whereby the stream of air humidified by the adsorbent material of the refrigerant evaporating heat exchanger (5, 3) is supplied to an indoor space.

25 In the fourth invention, the humidifier means (81) switches refrigerant circulation

in the refrigerant circuit (1) and air distribution, air dehumidification takes place in one heat exchanger (i.e., the heat exchanger (3, 5)) while air humidification takes place in the other heat exchanger (i.e., the heat exchanger (5, 3)), whereby the adsorbent material thereof is regenerated. As the result of this, humidification is carried out without interruption.

A fifth invention provides a humidity controller apparatus according to the first invention, wherein the adsorbent material is supported on a surface of the first heat exchanger (3) as well as on a surface of the second heat exchanger (5). Furthermore, a dehumidifier means (80) for switching refrigerant circulation in the refrigerant circuit (1) and air distribution is provided so that moisture in a stream of air flowing through a refrigerant evaporating heat exchanger (3, 5) is adsorbed by the adsorbent material of the refrigerant evaporating heat exchanger (3, 5) while the adsorbent material of a refrigerant condensing heat exchanger (5, 3) is regenerated by the release of moisture therefrom to a stream of air flowing through the refrigerant condensing heat exchanger (5, 3), whereby the stream of air dehumidified by the adsorbent material of the refrigerant evaporating heat exchanger (3, 5) is supplied to an indoor space. In addition, a humidifier means (81) for switching refrigerant circulation in the refrigerant circuit (1) and air distribution is provided so that moisture in a stream of air flowing through a refrigerant evaporating heat exchanger (3, 5) is adsorbed by the adsorbent material of the refrigerant evaporating heat exchanger (3, 5) while the adsorbent material of a refrigerant condensing heat exchanger (5, 3) is regenerated by the release of moisture therefrom to a stream of air flowing through the refrigerant condensing heat exchanger (5, 3), whereby the stream of air humidified by the adsorbent material of the refrigerant evaporating heat exchanger (5, 3) is supplied to an indoor space. Besides, the dehumidifier means (80) and the humidifier means (81) are configured to operate switchably between a dehumidification mode of operation and a

humidification mode of operation.

In the fifth invention, the dehumidifier means (80) switches refrigerant circulation in the refrigerant circuit (1) and air distribution, whereby air dehumidification takes place in one heat exchanger (i.e., the refrigerant evaporating heat exchanger (3, 5)) and  
5 adsorbent-material regeneration takes place in the other heat exchanger (i.e., the refrigerant condensing heat exchanger (5, 3)). As the result of this, dehumidification is carried out without interruption. In addition, when switched to the humidifier means (81), refrigerant circulation in the refrigerant circuit (1) and air distribution are switched. Then, air dehumidification takes place in one heat exchanger (i.e., the heat exchanger (3, 5)) and air  
10 humidification takes place in the other heat exchanger (i.e., the heat exchanger (5, 3)) and the adsorbent material thereof is regenerated. As the result of this, humidification is carried out without interruption.

A sixth invention provides a humidity controller apparatus according to the first invention which comprises a first heat exchange chamber (69) for accommodating the first  
15 heat exchanger (3), and a second heat exchange chamber (73), formed adjacently to the first heat exchange chamber (69), for accommodating the second heat exchanger (5). Furthermore, the humidity controller apparatus of the sixth invention includes a first air inflow passage (63) and a first air outflow passage (65) which are formed along one end surface in a thickness direction in which respective one surfaces of the two heat exchange  
20 chambers (69, 73) continue and which are arranged in a superimposed manner in the thickness direction of both the heat exchange chambers (69, 73). Besides, the humidity controller apparatus includes a second air inflow passage (57) and a second air outflow passage (59) which are formed along another end surface which is an end surface where respective one surfaces of the two heat exchange chambers (69, 73) continue and which is  
25 situated face to face with the one end surface, and which are arranged in a superimposed

manner in the thickness direction of both the heat exchange chambers (69, 73). In addition, the humidity controller apparatus includes opening/closing means (35, ..., 47, ...) for opening and closing openings (31a, ..., 33a, ...) for communication between the first heat exchange chamber (69) and the second heat exchange chamber (73), and each inflow passage (57, 63) and each outflow passage (59, 65).

In the sixth invention, the distribution of air of the first heat exchange chamber (69) and the second heat exchange chamber (73) is switched by opening/closing control by means of the opening/closing means (35, ..., 47, ...). As the result of this, moisture absorption and moisture release are effected in the first and second heat exchange chambers (69, 73).

A seventh invention provides a humidity controller apparatus according to the fifth invention in which the dehumidifier means (80) is configured so as to perform a dehumidification operation in full ventilation mode in which: outdoor air is brought in, moisture in the outdoor air is adsorbed by the adsorbent material of the heat exchanger (3, 5) which becomes an evaporator, and the outdoor air thus converted into a stream of dehumidified air is supplied indoors while, on the other hand, room air is brought in, moisture desorbed from the adsorbent material of the heat exchanger (5, 3) which becomes a condenser is released to the room air whereby the adsorbent material of the heat exchanger (5, 3) is regenerated, and the room air thus converted into a stream of humidified air is discharged outdoors. In addition, the humidifier means (81) is configured so as to perform a humidification operation in full ventilation mode in which: room air is brought in, moisture in the room air is adsorbed by the adsorbent material of the heat exchanger (3, 5) which becomes an evaporator, and the room air thus converted into a stream of dehumidified air is discharged outdoors while, on the other hand, outdoor air is brought in, moisture desorbed from the adsorbent material of the heat exchanger (5, 3) as a

condenser is released to the outdoor air whereby the adsorbent material of the heat exchanger (5, 3) is regenerated, and the outdoor air thus converted into a stream of humidified air is supplied indoors.

In the seventh invention, indoor space dehumidification or humidification is provided simultaneously with air ventilation.

An eighth invention provides a humidity controller apparatus according to the fifth invention in which the dehumidifier means (80) is configured so as to perform a dehumidification operation in circulation mode in which: room air is brought in, moisture in the room air is adsorbed by the adsorbent material of the heat exchanger (3, 5) which becomes an evaporator, and the room air thus converted into a stream of dehumidified air is supplied indoors while, on the other hand, outdoor air is brought in, moisture desorbed from the adsorbent material of the heat exchanger (5, 3) which becomes a condenser is released to the outdoor air whereby the adsorbent material of the heat exchanger (5, 3) is regenerated, and the outdoor air thus converted into a stream of humidified air is discharged outdoors. In addition, the humidifier means (81) is configured so as to perform a humidification operation in circulation mode in which: outdoor air is brought in, moisture in the outdoor air is adsorbed by the adsorbent material of the heat exchanger (3, 5) which becomes an evaporator, and the outdoor air thus converted into a stream of dehumidified air is discharged outdoors while, on the other hand, room air is brought in, moisture desorbed from the adsorbent material of the heat exchanger (5, 3) which becomes a condenser is released to the room air whereby the adsorbent material of the heat exchanger (5, 3) is regenerated, and the room air thus converted into a stream of humidified air is supplied indoors.

In the eighth invention, indoor space dehumidification or humidification is provided by the circulation of room air, without air ventilation.

A ninth invention provides a humidity controller apparatus according to the fifth invention in which the dehumidifier means (80) is configured so as to perform a dehumidification operation in air supply mode in which: outdoor air is brought in, moisture in the outdoor air is adsorbed by the adsorbent material of the heat exchanger (3, 5) which becomes an evaporator, and the outdoor air thus converted into a stream of dehumidified air is supplied indoors while, on the other hand, outdoor air is brought in, moisture desorbed from the adsorbent material of the heat exchanger (5, 3) which becomes a condenser is released to the outdoor air whereby the adsorbent material of the heat exchanger (5, 3) is regenerated, and the outdoor air thus converted into a stream of humidified air is discharged outdoors. In addition, the humidifier means (81) is configured so as to perform a humidification operation in air supply mode in which: outdoor air is brought in, moisture in the outdoor air is adsorbed by the adsorbent material of the heat exchanger (3, 5) which becomes an evaporator, and the outdoor air thus converted into a stream of dehumidified air is discharged outdoors while, on the other hand, outdoor air is brought in, moisture desorbed from the adsorbent material of the heat exchanger (5, 3) which becomes a condenser is released to the outdoor air whereby the adsorbent material of the heat exchanger (5, 3) is regenerated, and the outdoor air thus converted into a stream of humidified air is supplied indoors.

In the ninth invention, indoor space dehumidification or humidification is provided by only the taking-in of outdoor air.

Finally, a tenth invention provides a humidity controller apparatus according to the fifth invention in which the dehumidifier means (80) is configured so as to perform a dehumidification operation in air discharge mode in which: room air is brought in, moisture in the room air is adsorbed by the adsorbent material of the heat exchanger (3, 5) which becomes an evaporator, and the room air thus converted into a stream of

dehumidified air is supplied indoors while, on the other hand, room air is brought in, moisture desorbed from the adsorbent material of the heat exchanger (5, 3) which becomes a condenser is released to the room air whereby the adsorbent material of the heat exchanger (5, 3) is regenerated, and the room air thus converted into a stream of humidified air is discharged outdoors. In addition, the humidifier means (81) is configured so as to perform a humidification operation in air discharge mode in which: room air is brought in, moisture in the room air is adsorbed by the adsorbent material of the heat exchanger (3, 5) which becomes an evaporator, and the room air thus converted into a stream of dehumidified air is discharged outdoors while, on the other hand, room air is brought in, moisture desorbed from the adsorbent material of the heat exchanger (5, 3) which becomes a condenser is released to the room air whereby the adsorbent material of the heat exchanger (5, 3) is regenerated, and the room air thus converted into a stream of humidified air is supplied indoors.

In the tenth invention, indoor space dehumidification or humidification is provided by only the discharging of room air.

#### **EFFECTS OF INVENTION**

Accordingly, in accordance with the present invention, it is arranged that adsorbent material is supported on the surface of the first heat exchanger (3) for the integral formation of a heating/cooling means and an adsorption/desorption means. Such an arrangement makes it possible to effect the process of moisture absorption and the process of moisture release with the omission of an adsorbent material container. This reduces the number of component parts, thereby making it possible to provide a simplified structure and, in addition, it becomes possible to provide a downsized humidity controller apparatus.

Furthermore, it is arranged that adsorbent material is supported on at least the

surface of the first heat exchanger (3). This arrangement allows the refrigerant to directly cool or heat the adsorbent material. Consequently, the adsorption/desorption performance of the adsorbent material is brought to a maximum. This not only improves the efficiency of adsorption/desorption but also provides a downsized humidity controller apparatus.

5 Furthermore, in accordance with the second invention, there are provided a moisture absorbing means and a moisture releasing means. This ensures that the process of moisture absorption and the process of moisture release are assuredly performed in the first heat exchanger (3).

10 Furthermore, in accordance with the third or fourth invention, it is arranged that adsorbent material is supported on the surface of the first heat exchanger (3) as well as on the surface of the second heat exchanger (5). Such an arrangement makes it possible to continuously perform a dehumidification operation or a humidification operation with the omission of an adsorbent material container. This reduces the number of component parts, thereby making it possible to provide not only a simplified structure but also a downsized  
15 humidity controller apparatus. Besides, dehumidification or humidification is provided with high efficiency.

Furthermore, in accordance with the fifth invention, the operation is continued by switching between dehumidification and humidification. This reduces the number of component parts, thereby making it possible to provide not only a simplified structure but  
20 also a downsized humidity controller apparatus. Besides, dehumidification or humidification is provided with high efficiency.

In addition, in accordance with the sixth invention, it is arranged that: the first heat exchange chamber (69) and the second heat exchange chamber (73) are adjacently arranged; the inflow passages (57, 63) and the outflow passages (59, 65) are arranged in a  
25 superimposed manner in the thickness direction of the first and second heat exchange

chambers (69, 73). This provides a downsized humidity controller apparatus.

Furthermore, in accordance with the seventh invention, indoor space dehumidification or humidification is provided simultaneously with indoor space air ventilation.

5 In addition, in accordance with the eighth invention, indoor space dehumidification or humidification is provided by the circulation of room air. Humidity conditioning is carried out by inhibiting the variation in room temperature without the taking-in of outdoor air.

Furthermore, in accordance with the ninth invention, indoor space  
10 dehumidification or humidification is provided by only the taking-in of outdoor air.

Lastly, in accordance with the tenth invention, indoor space dehumidification or humidification is provided by only the discharging of room air.

#### **BRIEF DESCRIPTION OF DRAWINGS**

Figure 1 is a circuit diagram showing a refrigerant circuit of a humidity controller  
15 apparatus according to a first embodiment of the present invention;

Figure 2 is a perspective view of a heat exchanger according to the first embodiment of the present invention;

Figure 3 is a plan view of a casing, with the omission of its top surface plate;

Figure 4 is an end elevation view of the casing taken on line A-A of Fig. 3;

20 Figure 5 is an end elevation view of the casing taken on line B-B of Fig. 3;

Figure 6 is a side view of a damper when placed in the closed state;

Figure 7 is a side view of the damper when placed in the open state;

Figure 8 is a side view of a first modification of the damper when placed in the closed state;

25 Figure 9 is a side view of the first damper modification when placed in the open

state;

Figure 10 is a side view of a second modification of the damper when placed in the closed state;

Figure 11 is a side view of the second damper modification when placed in the  
5 open state;

Figure 12 is a plan view of the casing with the omission of its top surface plate, illustrating a first operation during dehumidification;

Figure 13 is a plan view of the casing with the omission of its top surface plate, illustrating a second operation during dehumidification;

10 Figure 14 is a plan view of the casing with the omission of its top surface plate, illustrating a first operation during humidification;

Figure 15 is a plan view of the casing with the omission of its top surface plate, illustrating a second operation during humidification;

Figure 16 is a psychrometric chart representing an air state of the humidity  
15 controller apparatus of the first embodiment and that of a conventional humidity controller apparatus during dehumidification operation;

Figure 17 is a table showing respective data of the humidity controller apparatus of the first embodiment and the conventional humidity controller apparatus during dehumidification operation;

20 Figure 18 shows an end elevation view and a plan view of the casing in a first operation during dehumidification operation in air supply mode in a second embodiment of the present invention;

Figure 19 shows an end elevation view and a plan view of the casing in a second operation during dehumidification operation in air supply mode in the second embodiment;

25 Figure 20 shows an end elevation view and a plan view of the casing in a first

operation during humidification operation in air supply mode in the second embodiment;

Figure 21 shows an end elevation view and a plan view of the casing in a second operation during humidification operation in air supply mode in the second embodiment;

Figure 22 shows an end elevation view and a plan view of the casing in a first  
 5 operation during dehumidification operation in air discharge mode in the second embodiment;

Figure 23 shows an end elevation view and a plan view of the casing in a second operation during dehumidification operation in air discharge mode in the second embodiment;

10 Figure 24 shows an end elevation view and a plan view of the casing in a first operation during humidification operation in air discharge mode in the second embodiment; and

Figure 25 shows an end elevation view and a plan view of the casing in a second operation during humidification operation in air discharge mode in the second embodiment.

## 15 **BEST MODE FOR CARRYING OUT INVENTION**

Hereinafter, embodiments of the present invention will be described in detail with reference to the drawings.

### **EMBODIMENT 1 OF INVENTION**

As shown in Figures 1-5, a humidity controller apparatus of the first embodiment  
 20 effects room-air dehumidification and humidification, and has a hollow, rectangular parallelepiped box-like casing (17). The casing (17) accommodates a refrigerant circuit (1) and other component parts.

Referring to Figure 1, the refrigerant circuit (1) is a closed circuit formed by a sequential connection of a compressor (7), a four-way switching valve (9) which is a flow  
 25 path switching means, a first heat exchanger (3) which is a first heat exchanger, an

expansion valve (11) which is an expansion mechanism, and a second heat exchanger (5) which is a second heat exchanger.

In addition, the refrigerant circuit (1) is charged with a refrigerant. The refrigerant circuit (1) is configured, such that a vapor compression refrigeration cycle is performed by the circulation of the refrigerant therethrough.

One end of the first heat exchanger (3) is connected to the four-way switching valve (9). The other end of the first heat exchanger (3) is connected to one end of the second heat exchanger (5) via the expansion valve (11). The other end of the second heat exchanger (5) is connected to the four-way switching valve (9).

With reference to Figure 2, the first heat exchanger (3) and the second heat exchanger (5) are each formed by a respective fin and tube heat exchanger of the cross fin type. More specifically, each of the first and second heat exchangers (3) and (5) is provided with a large number of aluminum fins (13) each shaped like a rectangular plate and copper heat transfer tubes (15) which penetrate the fins (13).

An adsorbent material is supported by means of dip forming (immersion forming) on the external surface of each fin (13) as well as on the external surface of each heat transfer tube (15).

As the adsorbent material, zeolite, silica gel, activated carbon, materials selected from the organic polymer family having hydrophilic properties or water-absorbing properties, materials selected from the ion-exchange resin family having carboxylic acid group or sulfonic acid group, and functional polymeric materials such as temperature-sensitive polymers may be used.

In the present embodiment, the first heat exchanger (3) and the second heat exchanger (5) are fin and tube heat exchangers of the cross fin type. Alternatively, different types of heat exchangers may be employed. For example, heat exchangers of

the corrugated fin type may be used.

In addition, in the present embodiment, the adsorbent material is supported, by means of dip forming, on the external surface of each of the fins (13) and on the external surface of each of the heat transfer tubes (15). Alternatively, the adsorbent material may  
 5 be supported on the external surfaces by any other techniques as long as its ability to provide adsorption is not reduced.

The four-way switching valve (9) is so configured as to switch between a first state that allows communication between the first and third ports and, at the same time, communication between the second and fourth ports (as indicated in Figure 1(A)), and a  
 10 second state that allows communication between the first and fourth ports and, at the same time, communication between the second and third ports (as indicated in Figure 1(B)). And, the four-way switching valve (9) is switched for establishing a switch between a first operation in which the first heat exchanger (3) functions as a condenser while, simultaneously, the second heat exchanger (5) functions as an evaporator and a second  
 15 operation in which the second heat exchanger (5) functions as a condenser while, simultaneously, the first heat exchanger (3) functions as an evaporator.

Next, with reference to Figures 3-5, the inner structure of the casing (17) is described. Referring to Figure 3, the casing (17) has a front side (which is shown as a lower side in the figure), a backside (upper side), a left-hand side (left side), and a right-  
 20 hand side (right side). In addition, in Figures 4 and 5, the casing (17) has a top surface (which is shown as an upper side in the figure) and a bottom surface (lower side).

The casing (17) is square when viewed in plan and is shaped like a flat box. The casing (17) has a left side plate (17a) which is provided with a first suction opening (19) through which outdoor air *OA* is drawn in and a second suction opening (21) through  
 25 which room air *RA* (return air) is drawn in. The casing (17) further has a right side plate

(17b) which is provided with a first outlet opening (23) through which emission air *EA* is expelled outside the room and a second outlet opening (25) through which humidity-conditioned air *SA* is supplied to an indoor space.

Disposed inside the casing (17) is a partition plate (27) which is a partition member. The partition plate (27) defines an air chamber (29a) and an equipment chamber (29b) inside the casing (17). The partition plate (27) is disposed perpendicularly, i.e., in the direction of the thickness of the casing (17). Referring to Figures 4 and 5, the partition plate (27) is disposed so as to extend from a top plate (17e) of the casing (17) (upper side) to a bottom plate (17f) of the casing (17) (lower side). Furthermore, with reference to Figure 3, the partition plate (27) is disposed so as to extend from a front plate (17c) of the casing (17) (lower side) to a back plate (17d) of the casing (17) (upper side). In addition, in Figure 3, the partition plate (27) is positioned, such that it lies somewhat right relative to the central part of the casing (17).

The equipment chamber (29b) accommodates the compressor (7) and other devices in the refrigerant circuit (1) with the exclusion of the heat exchangers (3, 5). In addition, the equipment chamber (29b) houses a first fan (79) and a second fan (77). The first fan (79) is connected to the first outlet opening (23) while, on the other hand, the second fan (77) is connected to the second outlet opening (25).

The air chamber (29a) of the casing (17) is provided with a first end surface plate (33) which is a partition member, a second end surface plate (31) which is a partition member, and a central dividing plate (67) which is a partition member. The first end surface plate (33), the second end surface plate (31), and the dividing plate (67) are all disposed perpendicularly, i.e., in the direction of the thickness of the casing (17) and, as shown in Figures 4 and 5, these plates are disposed, such that they extend from the top surface plate (17e) to the bottom surface plate (17f) in the casing (17).

As shown in Figure 3, the first end surface plate (33) and the second end surface plate (31) are disposed so as to extend to the partition plate (27) from the left side surface plate (17a) in the casing (17). In addition, the first end surface plate (33) is positioned, such that it slightly overlies the central part of casing (17) in Figure 3. On the other hand, the second end surface plate (31) is positioned, such that it slightly underlies the central part of casing (17), as shown in Figure 3.

The dividing plate (67) is disposed so as to extend between the first end surface plate (33) and the second end surface plate (31), as shown in Figure 3.

And, in the casing (17), a first heat exchange chamber (69) is divisionally formed as a compartment by the first end surface plate (33), the second end surface plate (31), the dividing plate (67), and the partition plate (27). In addition, in the casing (17), a second heat exchange chamber (73) is divisionally formed as a compartment by the first end surface plate (33), the second end surface plate (31), the dividing plate (67), and the left side surface plate (17a) of the casing (17). In other words, the first heat exchange chamber (69) lies on the right-hand side in Figure 3; the second heat exchange chamber (73) lies on the left-hand side in Figure 3; and the first heat exchange chamber (69) and the second heat exchange chamber (73) are formed adjacently parallelly with each other.

The first heat exchanger (3) is disposed in the first heat exchange chamber (69). The second heat exchanger (5) is disposed in the second heat exchange chamber (73).

A horizontal plate (61), serving as a partition member, is disposed between the first end surface plate (33) and the back plate (17d) of the casing (17), such that it defines a first inflow passage (63) and a first outflow passage (65). In addition, a horizontal plate (55), serving as a partition member, is disposed between the second end surface plate (31) and the front plate (17c) of the casing (17), such that it defines a second inflow passage (57) and a second outflow passage (59).

The interior space of the casing (17) is divided vertically (i.e., in the direction of the thickness of the casing (17)) into upper and lower spaces by the horizontal plates (61, 55). And, in Figure 4, the first inflow passage (63) is formed on the top surface side while, on the other hand, the first outflow passage (65) is formed on the bottom surface side. In Figure 5, the second inflow passage (57) is formed on the top surface side while, on the other hand, the second outflow passage (59) is formed on the bottom surface side.

That is, the first inflow passage (63) and the first outflow passage (65) are formed along one end surface in a thickness direction in which respective one surfaces of the first and second heat exchange chambers (69) and (73) continue, and are arranged in a superimposed manner in the thickness direction of the first and second heat exchange chambers (69, 73).

In addition, a second air inflow passage (57) and a second air outflow passage (59) which are formed along another end surface which is an end surface where respective one surfaces of the heat exchange chambers (69, 73) continue and which is situated face to face with the one end surface, and which are arranged in a superimposed manner in the thickness direction of the heat exchange chambers (69, 73).

And, In Figure 3, the first inflow passage (63) and the first outflow passage (65) are arranged symmetrically with the second inflow passage (57) and the second outflow passage (59), in other words, they are arranged in plane symmetry on the basis of a central line that crosses the first and second heat exchange chambers (69) and (73).

In addition, the first inflow passage (63) is in communication with the first suction opening (19). The first outflow passage (65) is in communication with the first fan (79), thereby being in communication with the first outlet opening (23). The second inflow passage (57) is in communication with the second suction opening (21). The second outflow passage (59) is in communication with the second fan (77), thereby being in

communication with the second outlet opening (25).

As shown in Figure 4, the first end surface plate (33) is provided with first to fourth openings (33a, 33b, 33c, 33d). The first to fourth openings (33a, 33b, 33c, 33d) are provided with a first damper (47), a second damper (49), a third damper (51), and a fourth damper (53), respectively. These four openings (33a-33d) are positioned in close proximity to one another in a matrix direction, in other words they are vertically laterally arranged in the form of squares in pairs. The first opening (33a) and the third opening (33c) open to the first heat exchange chamber (69) while, on the other hand, the second opening (33b) and the fourth opening (33d) open to the second heat exchange chamber (73).

The first opening (33a) allows communication between the first inflow passage (63) and the first heat exchange chamber (69). The third opening (33c) allows communication between the first outflow passage (65) and the first heat exchange chamber (69). In addition, the second opening (33b) allows communication between the first inflow passage (63) and the second heat exchange chamber (73). The fourth opening (33d) allows communication between the first outflow passage (65) and the second heat exchange chamber (73).

As shown in Figure 5, the second end surface plate (31) is provided with fifth to eighth openings (31a, 31b, 31c, 31d). The fifth to eighth openings (31a, 31b, 31c, 31d) are provided with a fifth damper (35), a sixth damper (37), a seventh damper (39), and an eighth damper (41), respectively. The four openings (31a, 31b, 31c, 31d) are positioned in close proximity to one another in a matrix direction, in other words they are vertically laterally arranged in the form of squares in pairs. The fifth opening (31a) and the seventh opening (31c) open to the first heat exchange chamber (69) while, on the other hand, the sixth opening (31b) and the eighth opening (31d) open to the second heat exchange

chamber (73).

The fifth opening (31a) allows communication between the second inflow passage (57) and the first heat exchange chamber (69). The seventh opening (31c) allows communication between the second outflow passage (59) and the first heat exchange chamber (69). In addition, the sixth opening (31b) allows communication between the second inflow passage (57) and the second heat exchange chamber (73). The eighth opening (31d) allows communication between the second outflow passage (59) and the second heat exchange chamber (73).

The first to eighth dampers (47-53, 35-41) constitute opening and closing means for opening and closing the openings (33a-33d, 31a-31d). The description is made with a focus on the fifth to eighth dampers (35-41). The damper (35-41) has a vane part (43) shaped like a rectangle and a shaft part (45) arranged centrally in the vane part (43), as shown in Figures 6 and 7. The shaft part (45) supports the vane part (43) on the first end surface plate (33) or on the second end surface plate (31), such that the vane part (43) is rotatable. And, as shown in Figure 7, the damper (35-41) is configured so that the opening (31a-31d) is placed in the open state when the vane part (43) is placed in a horizontal position. The other first to fourth dampers (47-53) are formed so as to have the same structure as that of the fifth to eighth dampers (35-41).

The structure of the dampers (35-41) is not limited to the ones shown in Figures 6 and 7. That is, each damper (47-53, 35-41) may employ a structure shown in Figures 10 and 11 or a structure shown in Figures 8 and 9.

Referring to Figures 8 and 9, there are shown fifth to eighth dampers (35-41) each of which is provided with two vane parts (43). Each damper (35-41) is configured, such that its two vane parts (43) rotate individually upwards and downwards respectively, thereby placing the associated opening (31a-31d) in the open state.

Referring to Figures 10 and 11, there are shown fifth to eighth dampers (35-41) each of which is provided with two vane parts (43). Each damper (35-41) is configured, such that its two vane parts (43) is folded upwards, thereby placing the associated opening (31a-31d) in the open state.

5 In addition, the present humidity controller apparatus is provided with a dehumidifier means (80) and a humidifier means (81). And, it is configured that the dehumidifier means (80) may be switched to function as the humidifier means (81), and vice versa, thereby allowing a switch between a dehumidification mode of operation and a humidification mode of operation.

10 The dehumidifier means (80) switches the circulation of refrigerant in the refrigerant circuit (1) and the distribution of air by the damper (47-53, 35-41) so that moisture present in a stream of air flowing through the first heat exchanger (3) or the second heat exchanger (5), whichever functions as a refrigerant evaporating heat exchanger, is adsorbed by adsorbent material while, on the other hand, adsorbent material is  
15 regenerated by the release of moisture present in the adsorbent material to a stream of air flowing through the second heat exchanger (5) or the first heat exchanger (3), whichever functions as a refrigerant condensing heat exchanger, and the stream of air thus converted into a stream of dehumidified air by the adsorbent material is supplied into the room.

On the other hand, the humidifier means (81) switches the circulation of  
20 refrigerant in the refrigerant circuit (1) and the distribution of air by the damper (47-53, 35-41) so that moisture present in a stream of air flowing through the first heat exchanger (3) or the second heat exchanger (5), whichever functions as a refrigerant evaporating heat exchanger, is adsorbed by adsorbent material while, on the other hand, adsorbent material is regenerated by the release of moisture present in the adsorbent material to a stream of air  
25 flowing through the second heat exchanger (5) or the first heat exchanger (3), whichever

functions as a refrigerant condensing heat exchanger, and the stream of air thus converted into a stream of humidified air by the adsorbent material is supplied into the room.

More specifically, the dehumidifier means (80) performs a dehumidification operation in full ventilation mode. In this dehumidification operation, outdoor air is brought in, moisture in the outdoor air is adsorbed by the adsorbent material of the heat exchanger (3, 5) which becomes an evaporator, and the outdoor air thus converted into a stream of dehumidified air is supplied indoors while, on the other hand, room air is brought in, moisture desorbed from the adsorbent material of the heat exchanger (5, 3) which becomes a condenser is released to the room air whereby the adsorbent material of the heat exchanger (5, 3) is regenerated, and the room air thus converted into a stream of humidified air is discharged outdoors.

The humidifier means (81) performs a humidification operation in full ventilation mode. In this humidification operation, room air is brought in, moisture in the room air is adsorbed by the adsorbent material of the heat exchanger (3, 5) which becomes an evaporator and the room air thus converted into a stream of dehumidified air is discharged outdoors while, on the other hand, outdoor air is brought in, moisture desorbed from the adsorbent material of the heat exchanger (5, 3) which becomes a condenser is released to the outdoor air whereby the adsorbent material of the heat exchanger (5, 3) is regenerated, and the outdoor air thus converted into a stream of humidified air is supplied indoors.

In addition, the dehumidifier means (80) performs a dehumidification operation in circulation mode in which room air is brought in, moisture in the room air is adsorbed by the adsorbent material of the heat exchanger (3, 5) which becomes an evaporator and the room air thus converted into a stream of dehumidified is supplied indoors while, on the other hand, outdoor air is brought in, moisture desorbed from the adsorbent material of the heat exchanger (5, 3) which becomes a condenser is released to the outdoor air whereby

the adsorbent material of the heat exchanger (5, 3) is regenerated, and the outdoor air thus converted into a stream of humidified air is discharged outdoors.

The humidifier means (81) performs a humidification operation in circulation mode in which outdoor air is brought in, moisture in the outdoor air is adsorbed by the  
 5 adsorbent material of the heat exchanger (3, 5) which becomes an evaporator and the outdoor air thus converted into a stream of dehumidified air is discharged outdoors while, on the other hand, room air is brought in, moisture desorbed from the adsorbent material of the heat exchanger (5, 3) which becomes a condenser is released to the room air whereby the adsorbent material of the heat exchanger (5, 3) is regenerated, and the room air thus  
 10 converted into a stream of humidified air is supplied indoors.

## **RUNNING OPERATION**

Next, the running operation of the above-described humidity controller apparatus is described. The humidity controller apparatus takes in a stream of first air and a stream of second air and performs selectively a dehumidification operation or a humidification  
 15 operation. In addition, the humidity controller apparatus continuously performs dehumidification and humidification operations by alternately repeating a first operation and a second operation. Besides, the humidity controller apparatus performs dehumidification and humidification operations in full ventilation mode and, in addition, humidification and dehumidification operations in circulation mode.

## **20 DEHUMIDIFICATION OPERATION IN FULL VENTILATION MODE**

In a dehumidification operation during full ventilation mode by the dehumidifier means (80), outdoor air *OA* is brought in as the first air and is supplied to the room while, on the other hand, room air *RA* is brought in as the second air and is discharged outside the room.

## **25 FIRST OPERATION**

In the first operation in which the first fan (79) and the second fan (77) are driven, the process of moisture adsorption in the second heat exchanger (5) and the process of adsorbent-material regeneration (moisture desorption) in the first heat exchanger (3) are carried out. In other words, in the first operation, moisture present in the outdoor air *OA* is adsorbed in the second heat exchanger (5) and moisture desorbed out of the first heat exchanger (3) is given to the room air *RA*.

Referring to Figure 1(A) and Figure 12, during the first operation the second damper (49), the third damper (51), the eighth damper (41), and the fifth damper (35) are placed in the open state while, on the other hand, the first damper (47), the fourth damper (53), the sixth damper (37), and the seventh damper (39) are placed in the closed state. The first heat exchanger (3) is supplied with a stream of room air *RA*. The second heat exchanger (5) is supplied with a stream of outdoor air *OA*.

In addition, the four-way switching valve (9) changes state to a state shown in Figure 1 (A). As a result, in the refrigerant circuit (1) the first heat exchanger (3) functions as a condenser and the second heat exchanger (5) functions as an evaporator.

That is, high-temperature, high-pressure refrigerant expelled out of the compressor (7) flows into the first heat exchanger (3) as a heat carrier for heating. In the first heat exchanger (3), the adsorbent materials supported, respectively, on the external surface of each fin (13) and on the external surface of each heat transfer tube (15) are heated. This heating causes moisture desorption from the adsorbent materials, whereby the adsorbent materials are regenerated.

On the other hand, the refrigerant condensed in the first heat exchanger (3) is decompressed by the expansion valve (11). The post-decompression refrigerant flows into the second heat exchanger (5) as a heat carrier for cooling. In the second heat exchanger (5), heat of adsorption is generated when the adsorbent material supported on

the external surface of each fin (13) and the adsorbent material supported on the external surface of each heat transfer tube (15) adsorb moisture. The refrigerant in the second heat exchanger (5) absorbs the heat of adsorption and then evaporates. The refrigerant evaporated is directed back to the compressor (7) and repeats such a circulation.

5           In addition, an inflow of room air *RA* entering through the second suction opening (21) by drive of the first and second fans (79) and (77) travels through the second inflow passage (57) and flows to the first heat exchange chamber (69) from the fifth opening (31a). In the first heat exchange chamber (69), moisture desorbed from the adsorbent material of the first heat exchanger (3) is released to the room air *RA* and, in this way, the  
10   room air *RA* is humidified. The room air *RA* thus humidified becomes a stream of emission air *EA*. The emission air *EA* exiting the first heat exchange chamber (69) flows through the first outflow passage (65) by way of the third opening (33c). Then, the emission air *EA*, after passing through the first fan (79), is discharged outdoors through the first outlet opening (23).

15           On the other hand, an inflow of outdoor air *OA* entering through the first suction opening (19) travels through the first inflow passage (63) and flows to the second heat exchange chamber (73) from the second opening (33b). In the second heat exchange chamber (73), moisture in the outdoor air *OA* is adsorbed by the adsorbent material of the second heat exchanger (5) and, in this way, the outdoor air *OA* is dehumidified. The  
20   outdoor air *OA* thus dehumidified becomes a stream of humidity conditioned air *SA*. The humidity conditioned air *SA* exiting the second heat exchange chamber (73) flows through the second outflow passage (59) by way of the eighth opening (31d). Then, the humidity conditioned air *SA*, after passing through the second fan (77), is supplied indoors through the second outlet opening (25).

25           Upon completion of the execution of the first operation, the second operation is

carried out.

## SECOND OPERATION

In the second operation in which the first fan (79) and the second fan (77) are driven, the process of adsorption in the first heat exchanger (3) and the process of  
 5 regeneration in the second heat exchanger (5) are carried out. In other words, in the second operation, moisture present in outdoor air *OA* is adsorbed in the first heat exchanger (3) and moisture desorbed out of the second heat exchanger (5) is given to room air *RA*.

Referring to Figure 1(B) and Figure 13, during the second operation the first  
 10 damper (47), the fourth damper (53), the seventh damper (39), and the sixth damper (37) are placed in the open state while, on the other hand, the third damper (51), the second damper (49), the fifth damper (35), and the eighth damper (41) are placed in the closed state. And, the first heat exchanger (3) is supplied with a stream of outdoor air *OA*. The second heat exchanger (5) is supplied with a stream of room air *RA*.

15 In addition, the four-way switching valve (9) changes state to a state shown in Figure 1 (B). As a result, in the refrigerant circuit (1) the second heat exchanger (5) functions as a condenser and the first heat exchanger (3) functions as an evaporator.

That is, high-temperature, high-pressure refrigerant expelled out of the compressor (7) flows into the second heat exchanger (5) as a heat carrier for heating. In  
 20 the second heat exchanger (5), the adsorbent materials supported, respectively, on the external surface of each fin (13) and on the external surface of each heat transfer tube (15) are heated. This heating causes moisture desorption from the adsorbent materials, whereby the adsorbent materials are regenerated.

On the other hand, the refrigerant condensed in the second heat exchanger (5) is  
 25 decompressed by the expansion valve (11). The post-decompression refrigerant flows

into the first heat exchanger (3) as a heat carrier for cooling. In the first heat exchanger (3), heat of adsorption is generated when the adsorbent material supported on the external surface of each fin (13) and the adsorbent material supported on the external surface of each heat transfer tube (15) adsorb moisture. The refrigerant in the first heat exchanger (3) absorbs the heat of adsorption and then evaporates. The refrigerant evaporated is directed back to the compressor (7) and repeats such a circulation.

In addition, an inflow of room air *RA* entering through the second suction opening (21) by drive of the first and second fans (79) and (77) travels through the second inflow passage (57) and flows to the second heat exchange chamber (73) from the sixth opening (31b). In the second heat exchange chamber (73), moisture desorbed from the adsorbent material of the second heat exchanger (5) is released to the room air *RA* and, in this way, the room air *RA* is humidified. The room air *RA* thus humidified becomes a stream of emission air *EA*. The emission air *EA* exiting the second heat exchange chamber (73) flows through the first outflow passage (65) by way of the fourth opening (33d). Then, the emission air *EA*, after passing through the first fan (79), is discharged outdoors from the first outlet opening (23).

On the other hand, an inflow of outdoor air *OA* entering through the first suction opening (19) travels through the first inflow passage (63). The outdoor air *OA* flows into the first heat exchange chamber (69) from the first opening (33a). In the first heat exchange chamber (69), moisture in the outdoor air *OA* is adsorbed by the adsorbent material of the first heat exchanger (3), whereby the outdoor air *OA* is dehumidified. The outdoor air *OA* thus dehumidified becomes a stream of humidity conditioned air *SA*. The humidity conditioned air *SA* exiting the first heat exchange chamber (69) flows through the second outflow passage (59) by way of the seventh opening (31c). Then, the humidity conditioned air *SA*, after passing through the second fan (77), is supplied indoors through

the second outlet opening (25).

After execution of the second operation, the first operation is carried out again. And, indoor space dehumidification is continuously carried out by repetition of the first operation and the second operation.

## 5 HUMIDIFICATION OPERATION IN FULL VENTILATION MODE

In a humidification operation in full ventilation mode by the humidifier means (81), room air *RA* is brought in as the first air and is discharged outside the room while, on the other hand, outdoor air *OA* is brought in as the second air and is supplied into the room.

### FIRST OPERATION

10 In the first operation in which the first fan (79) and the second fan (77) are driven, the process of adsorption in the second heat exchanger (5) and the process of regeneration in the first heat exchanger (3) are carried out. In other words, in the first operation, moisture present in the room air *RA* is adsorbed in the second heat exchanger (5) and moisture desorbed out of the first heat exchanger (3) is given to the outdoor air *OA*.

15 Referring to Figure 1(A) and Figure 14, during the first operation the first damper (47), the fourth damper (53), the seventh damper (39), and the sixth damper (37) are placed in the open state while, on the other hand, the third damper (51), the second damper (49), the fifth damper (35), and the eighth damper (41) are placed in the closed state. The first heat exchanger (3) is supplied with a stream of outdoor air *OA*. The second heat  
20 exchanger (5) is supplied with a stream of room air *RA*.

In addition, the four-way switching valve (9) changes state to a state shown in Figure 1 (A). As a result, in the refrigerant circuit (1) the first heat exchanger (3) functions as a condenser and the second heat exchanger (5) functions as an evaporator.

That is, high-temperature, high-pressure refrigerant expelled out of the  
25 compressor (7) flows into the first heat exchanger (3) as a heat carrier for heating. In the

first heat exchanger (3), the adsorbent materials supported, respectively, on the external surface of each fin (13) and on the external surface of each heat transfer tube (15) are heated. This heating causes moisture desorption from the adsorbent materials, whereby the adsorbent materials are regenerated.

5           On the other hand, the refrigerant condensed in the first heat exchanger (3) is decompressed by the expansion valve (11). The post-decompression refrigerant flows into the second heat exchanger (5) as a heat carrier for cooling. In the second heat exchanger (5), heat of adsorption is generated when the adsorbent material supported on the external surface of each fin (13) and the adsorbent material supported on the external  
10 surface of each heat transfer tube (15) adsorb moisture. The refrigerant in the second heat exchanger (5) absorbs the heat of adsorption and then evaporates. The refrigerant evaporated is directed back to the compressor (7) and repeats such a circulation.

          In addition, an inflow of room air *RA* entering through the second suction opening (21) by drive of the first and second fans (79) and (77) travels through the second inflow  
15 passage (57) and flows into the second heat exchange chamber (73) from the sixth opening (31b). In the second heat exchange chamber (73), moisture in the room air *RA* is adsorbed by the adsorbent material of the second heat exchanger (5), whereby the room air *RA* is dehumidified. The room air *RA* thus dehumidified becomes a stream of emission air *EA*. The emission air *EA* exiting the second heat exchange chamber (73) flows  
20 through the first outflow passage (65) by way of the fourth opening (33d). Then, the emission air *EA*, after passing through the first fan (79), is discharged outdoors through the first outlet opening (23).

          On the other hand, an inflow of outdoor air *OA* entering through the first suction opening (19) travels through the first inflow passage (63) and flows to the first heat  
25 exchange chamber (69) from the first opening (33a). In the first heat exchange chamber

(69), moisture desorbed out of the adsorbent material of the first heat exchanger (3) is released to the outdoor air *OA*, whereby the outdoor air *OA* is humidified. The outdoor air *OA* thus humidified becomes a stream of humidity conditioned air *SA*. The humidity conditioned air *SA* exiting the first heat exchange chamber (69) flows through the second outflow passage (59) by way of the seventh opening (31c). Then, the humidity conditioned air *SA*, after passing through the second fan (77), is supplied indoors through the second outlet opening (25).

Upon completion of the execution of the first operation, the second operation carried out.

## 10 SECOND OPERATION

In the second operation in which the first fan (79) and the second fan (77) are driven, the process of adsorption in the first heat exchanger (3) and the process of regeneration in the second heat exchanger (5) are carried out. In other words, in the second operation, moisture present in room air *RA* is adsorbed in the first heat exchanger (3) and moisture desorbed out of the second heat exchanger (5) is given to outdoor air *OA*.

Referring to Figure 1(B) and Figure 15, during the second operation the second damper (49), the third damper (51), the eighth damper (41), and the fifth damper (35) are placed in the open state while, on the other hand, the fourth damper (53), the first damper (47), the sixth damper (37), and the seventh damper (39) are placed in the closed state. The first heat exchanger (3) is supplied with a stream of room air *RA*. The second heat exchanger (5) is supplied with a stream of outdoor air *OA*.

In addition, the four-way switching valve (9) changes state to a state shown in Figure 1 (B). As a result, in the refrigerant circuit (1) the second heat exchanger (5) functions as a condenser and the first heat exchanger (3) functions as an evaporator.

That is, high-temperature, high-pressure refrigerant expelled out of the

compressor (7) flows to the second heat exchanger (5) as a heat carrier for heating. In the second heat exchanger (5), the adsorbent materials supported, respectively, on the external surface of each fin (13) and on the external surface of each heat transfer tube (15) are heated. This heating causes moisture desorption from the adsorbent materials, whereby  
 5 the adsorbent materials are regenerated.

On the other hand, the refrigerant condensed in the second heat exchanger (5) is decompressed by the expansion valve (11). The post-decompression refrigerant flows to the first heat exchanger (3) as a heat carrier for cooling. In the first heat exchanger (3), heat of adsorption is generated when the adsorbent materials supported, respectively, on  
 10 the external surface of each fin (13) and on the external surface of the heat transfer tube (15) adsorb moisture. The refrigerant in the first heat exchanger (3) absorbs the heat of adsorption and then evaporates. The refrigerant evaporated is directed back to the compressor (7) and repeats such a circulation.

In addition, an inflow of air *RA* entering through the second suction opening (21)  
 15 by drive of the first and second fans (79) and (77) travels through the second inflow passage (57). Then, the room air *RA* flows to the first heat exchange chamber (69) from the fifth opening (31a). In the first heat exchange chamber (69), moisture in the room air *RA* is adsorbed by the adsorbent material of the first heat exchanger (3), whereby the room air *RA* is dehumidified. The room air *RA* thus dehumidified becomes a stream of  
 20 emission air *EA*. The emission air *EA* exiting the first heat exchange chamber (69) flows through the first outflow passage (65) by way of the third opening (33c). Then, after passage through the first fan (79), the emission air *EA* is discharged outdoors through the first outlet opening (23).

On the other hand, an inflow of outdoor air *OA* entering through the first suction  
 25 opening (19) travels through the first inflow passage (63). Then, the outdoor air *OA*

flows into the second heat exchange chamber (73) from the second opening (33b). In the second heat exchange chamber (73), moisture desorbed from the adsorbent material of the second heat exchanger (5) is released to the outdoor air *OA*, whereby the outdoor air *OA* is humidified. The outdoor air *OA* thus humidified becomes a stream of humidity

5 conditioned air *SA*. The humidity conditioned air *SA* exiting the second heat exchange chamber (73) flows through the second outflow passage (59) by way of the eighth opening (31d). Then, after passage through the second fan (77), the humidity conditioned air *SA* is supplied indoors through second outlet opening (25).

Upon completion of the execution of the second operation, the first operation is  
10 carried out again. And, indoor space humidification is continuously carried out by repetition of the first operation and the second operation.

#### **DEHUMIDIFICATION OPERATION IN CIRCULATION MODE**

During a dehumidification operation in circulation mode by the dehumidifier means (80), room air *RA* is brought in as the first air and is supplied indoors while, on the  
15 other hand, outdoor air *OA* is brought in as the second air and is discharged outdoors. The circulation of refrigerant in the refrigerant circuit (1) is the same as that in full ventilation mode and the description thereof is omitted accordingly.

#### **FIRST OPERATION**

In the first operation, the process of adsorption in the second heat exchanger (5)  
20 and the process of regeneration (desorption) in the first heat exchanger (3) are carried out. In other words, in the first operation, moisture present in room air *RA* is adsorbed in the second heat exchanger (5) and moisture desorbed out of the first heat exchanger (3) is given to outdoor air *OA*.

During the first operation, the first damper (47), the third damper (51), the sixth  
25 damper (37), and the eighth damper (41) are placed in the open state while, on the other

hand, the second damper (49), the fourth damper (53), the fifth damper (35), and the seventh damper (39) are placed in the closed state. The first heat exchanger (3) is provided with a stream of outdoor air *OA*. The second heat exchanger (5) is provided with a stream of room air *RA*.

5           An inflow of outdoor air *OA* entering through the first suction opening (19) travels through the first inflow passage (63). Then, the outdoor air *OA* flows into the first heat exchange chamber (69) from the first opening (33a). In the first heat exchange chamber (69), moisture desorbed from the adsorbent material of the first heat exchanger (3) is released to the outdoor air *OA*, whereby the outdoor air *OA* is humidified. The outdoor  
10   air *OA* thus humidified becomes a stream of emission air *EA*. The emission air *EA* exiting the first heat exchange chamber (69) flows through the first outflow passage (65) by way of the third opening (33c). Then, after passage through the first fan (79), the emission air *EA* is discharged outdoors through the first outlet opening (23).

          On the other hand, an inflow of room air *RA* entering through the second suction  
15   opening (21) travels through the second inflow passage (57). Then, the room air *RA* flows into the second heat exchange chamber (73) from the sixth opening (31b). In the second heat exchange chamber (73), moisture in the room air *RA* is adsorbed by the adsorbent material of the second heat exchanger (5), whereby the room air *RA* is dehumidified. The room air *RA* thus dehumidified becomes a stream of humidity  
20   conditioned air *SA*. The humidity conditioned air *SA* exiting the second heat exchange chamber (73) flows through the second outflow passage (59) by way of the eighth opening (31d). Then, after passage through the second fan (77), the humidity conditioned air *SA* is supplied indoors through the second outlet opening (25).

          Upon completion of the execution of the first operation, the second operation is  
25   carried out.

## SECOND OPERATION

In the second operation, the process of adsorption in the first heat exchanger (3) and the process of regeneration in the second heat exchanger (5) are carried out. In other words, in the second operation, moisture present in room air *RA* is adsorbed in the first heat exchanger (3) and moisture desorbed out of the second heat exchanger (5) is given to outdoor air *OA*.

During the second operation, the second damper (49), the fourth damper (53), the fifth damper (35), and the seventh damper (39) are placed in the open state while, on the other hand, the first damper (47), the third damper (51), the sixth damper (37), and the eighth damper (41) are placed in the closed state. The first heat exchanger (3) is provided with a stream of room air *RA*. The second heat exchanger (5) is provided with a stream of outdoor air *OA*.

An inflow of outdoor air *OA* entering through the first suction opening (19) travels through the first inflow passage (63). Then, the outdoor air *OA* flows into the second heat exchange chamber (73) from the second opening (33b). In the second heat exchange chamber (73), moisture desorbed from the adsorbent material of the second heat exchanger (5) is released to the outdoor air *OA*, whereby the outdoor air *OA* is humidified. The outdoor air *OA* thus humidified becomes a stream of emission air *EA*. The emission air *EA* exiting the second heat exchange chamber (73) flows through the first outflow passage (65) by way of the fourth opening (33d). Then, after passage through the first fan (79), the emission air *EA* is discharged outdoors through the first outlet opening (23).

On the other hand, an inflow of room air *RA* entering through the second suction opening (21) travels through the second inflow passage (57). Then, the room air *RA* flows into the first heat exchange chamber (69) from the fifth opening (31a). In the first heat exchange chamber (69), moisture in the room air *RA* is adsorbed by the adsorbent

material of the first heat exchanger (3), whereby the room air *RA* is dehumidified. The room air *RA* thus dehumidified becomes a stream of humidity conditioned air *SA*. The humidity conditioned air *SA* exiting the first heat exchange chamber (69) flows through the second outflow passage (59) by way of the seventh opening (31c). Then, after passage  
 5 through the second fan (77), the humidity conditioned air *SA* is supplied indoors through the second outlet opening (25).

After execution of the second operation, the first operation is carried out again. And, indoor space dehumidification is continuously carried out by repetition of the first operation and the second operation.

## 10 HUMIDIFICATION OPERATION IN CIRCULATION MODE

In a humidification operation in circulation mode by the humidifier means (81), outdoor air *OA* is brought in as the first air and is discharged outdoors while, on the other hand, room air *RA* is brought in as the second air and is supplied indoors. The circulation of refrigerant in the refrigerant circuit (1) is the same as that in full ventilation mode and  
 15 the description thereof is omitted accordingly.

### FIRST OPERATION

In the first operation, the process of adsorption in the second heat exchanger (5) and the process of regeneration in the first heat exchanger (3) are carried out. In other words, in the first operation, moisture present in outdoor air *OA* is adsorbed in the second  
 20 heat exchanger (5) and moisture desorbed out of the first heat exchanger (3) is given to room air *RA*.

During the first operation, the second damper (49), the fourth damper (53), the fifth damper (35), and the seventh damper (39) are placed in the open state while, on the other hand, the first damper (47), the third damper (51), the sixth damper (37), and the  
 25 eighth damper (41) are placed in the closed state. The first heat exchanger (3) is supplied

with a stream of room air *RA*. The second heat exchanger (5) is supplied with a stream of outdoor air *OA*.

An inflow of room air *RA* entering through the second suction opening (21) travels through the second inflow passage (57). Then, the room air *RA* flows into the first  
 5 heat exchange chamber (69) from the fifth opening (31a). In the first heat exchange chamber (69), moisture desorbed from the adsorbent material of the first heat exchanger (3) is released to the room air *RA*, whereby the room air *RA* is humidified. The room air *RA* thus humidified exits the first heat exchange chamber (69) and flows through the second outflow passage (59) by way of the seventh opening (31c). Then, after passage  
 10 through the second fan (77), the room air *RA* is supplied indoors through the second outlet opening (25).

On the other hand, an inflow of outdoor air *OA* entering through the first suction opening (19) travels through the first inflow passage (63). Then, the outdoor air *OA* flows into the second heat exchange chamber (73) from the second opening (33b). In the  
 15 second heat exchange chamber (73), moisture in the outdoor air *OA* is adsorbed by the adsorbent material of the second heat exchanger (5), whereby the outdoor air *OA* is dehumidified. The outdoor air *OA* thus dehumidified becomes a stream of emission air *EA*. The emission air *EA* exiting the second heat exchange chamber (73) flows through the first outflow passage (65) by way of the fourth opening (33d). Then, after passage  
 20 through the first fan (79), the emission air *EA* is discharged outdoors through the first outlet opening (23).

Upon completion of the execution of the first operation, the second operation is carried out.

## SECOND OPERATION

25 In the second operation, the process of adsorption in the first heat exchanger (3)

and the process of regeneration in the second heat exchanger (5) are carried out. In other words, in the second operation, moisture present in outdoor air *OA* is adsorbed in the first heat exchanger (3) and moisture desorbed out of the second heat exchanger (5) is given to room air *RA*.

5            During the second operation, the first damper (47), the third damper (51), the sixth damper (37), and the eighth damper (41) are placed in the open state while, on the other hand, the second damper (49), the fourth damper (53), the fifth damper (35), and the seventh damper (39) are placed in the closed state. The second heat exchanger (5) is supplied with a stream of room air *RA*. The first heat exchanger (3) is provided with a  
10    stream of outdoor air *OA*.

          An inflow of room air *RA* entering through the second suction opening (21) travels through the second inflow passage (57). Then, the room air *RA* flows into the second heat exchange chamber (73) from the sixth opening (31b). In the second heat exchange chamber (73), moisture desorbed from the adsorbent material of the second heat  
15    exchanger (5) is released to the room air *RA*, whereby the room air *RA* is humidified. The room air *RA* thus humidified becomes a stream of humidity conditioned air *SA*. The humidity conditioned air *SA* exiting the second heat exchange chamber (73) flows through the second outflow passage (59) by way of the eighth opening (31d). Then, after passage through the second fan (77), the humidity conditioned air *SA* is supplied indoors through  
20    the second outlet opening (25).

          On the other hand, an inflow of outdoor air *OA* entering through the first suction opening (19) travels through the first inflow passage (63). Then, the outdoor air *OA* flows into the first heat exchange chamber (69) from the first opening (33a). In the first heat exchange chamber (69), moisture present in the outdoor air *OA* is adsorbed by the  
25    adsorbent material of the first heat exchanger (3), whereby the outdoor air *OA* is

dehumidified. The outdoor air *OA* thus dehumidified becomes a stream of emission air *EA*. The emission air *EA* exiting the first heat exchange chamber (69) flows through the first outflow passage (65) by way of the third opening (33c). Then, after passage through the first fan (79), the emission air *EA* is discharged outdoors through the first outlet opening (23).

Upon completion of the execution of the second operation, the first operation is carried out again. And, indoor space humidification is continuously carried out by repetition of the first operation and the second operation.

#### PERFORMANCE COMPARISON

Referring to Figure 16, there is shown a psychrometric chart for comparison between the dehumidification operation of a humidity controller apparatus of the present embodiment and the dehumidification operation of a conventional humidity controller apparatus. Both the humidity controller apparatus of the present embodiment and the conventional humidity controller apparatus are capable of dehumidifying a volume of air (about 150 m<sup>3</sup>) per hour.

Referring to Figure 17, there is shown a table for comparison between the data of the dehumidification operation of a humidity controller apparatus of the present embodiment and the data of the dehumidification operation of a conventional humidity controller apparatus. The data shown in the table include information about the inlet temperature of outdoor air *OA* and other pieces of information. In addition, the letters "a"-"h" indicated in Figure 17 correspond to the letters "a"-"h" indicated in Figure 16, respectively.

As is clear from Figures 16 and 17, the amount of dehumidification of the humidity controller apparatus according to the present embodiment is greater than the amount of dehumidification of the conventional humidity controller apparatus. More

specifically, the amount of dehumidification of the humidity controller apparatus according to the present embodiment is twice the amount of dehumidification of the conventional humidity controller apparatus.

#### **EFFECTS OF EMBODIMENT 1**

5           As described above, according to the present embodiment, it is arranged that adsorbent materials are supported on the external surface of each heat transfer tube (15) as well as on the external surface of each fin (13) in the first and second heat exchangers (3) and (5), thereby allowing integral formation of a heating/cooling means and an adsorption/desorption means. Such an arrangement makes it possible to continuously  
10   provide dehumidification and humidification with the omission of an adsorbent-material container. This results in a reduction in the number of component parts, thereby making it possible to provide not only a simplified structure but also a downsized humidity controller apparatus.

          Furthermore, it is arranged that adsorbent materials are supported on the external  
15   surface of each heat transfer tube (15) as well as on the external surface of each fin (13) in the first and second heat exchangers (3) and (5). Such an arrangement makes it possible to allow the refrigerant to directly cool or heat the adsorbent materials. As the result of this, the adsorption/desorption performance of the adsorbent materials is brought to a maximum. This makes it possible to improve the efficiency of adsorption/desorption and  
20   to provide a downsized humidity controller apparatus.

          Furthermore, it is arranged that adsorbent materials are supported on the external surface of the first heat exchanger (3) as well as on the external surface of second heat exchanger (5). Such arrangement makes it possible to continuously perform dehumidification and humidification. As the result of this, it becomes possible to perform  
25   dehumidification and humidification with high efficiency.

In addition, it is arranged that: the first heat exchange chamber (69) and the second heat exchange chamber (73) are adjacently arranged; the inflow passages (57, 63) and the outflow passages (59, 65) are arranged in a superimposed manner in the thickness direction of the first and second heat exchange chambers (69, 73). Such an arrangement  
 5 provides a downsized humidity controller apparatus.

Besides, the eight dampers (35, ..., 47, ...) are provided, whereby the direction in which air is distributed is switched. This makes it possible to achieve a switch in the distribution direction of air by a simplified structure.

Furthermore, the first inflow and outflow passages (63, 65) are arranged  
 10 symmetrically with the second inflow and outflow passages (57, 59). As a result of such arrangement, the distribution resistance is reduced. This makes it possible to perform dehumidification and other like operation with high efficiency.

In addition, the openings (31a-31d, 33a-33d) are positioned in close proximity to one another in a matrix direction and are opened/closed by the dampers (35, ..., 47, ...),  
 15 respectively. Accordingly, air distribution systems are made simple in construction and the reduction of size is achieved.

Furthermore, the vapor compression refrigeration cycle refrigerant circuit (1) is employed, the adsorption and regeneration of adsorbent materials are carried out with high efficiency.

20 In addition, a dehumidification or humidification operation in full ventilation mode is carried out. Therefore, it becomes possible to perform room dehumidification or humidification while, simultaneously, performing room ventilation.

Finally, a dehumidification or humidification operation in circulation mode is carried out. Therefore, it becomes possible to perform room dehumidification or  
 25 humidification by the circulation of room air. As the result of this, it is possible to control

humidity by inhibiting the variation in room temperature without the taking-in of outdoor air.

## EMBODIMENT 2 OF INVENTION

Next, a second embodiment of the present invention is described in detail with  
5 reference to the drawings.

Referring to Figures 18-25, there is shown a humidity controller apparatus of the present embodiment. In the humidity controller apparatus of the present embodiment, the dehumidifier means (80) and the humidifier means (81) switchably perform a dehumidification or a humidification operation in air supply mode or switchably perform a  
10 dehumidification or a humidification operation in air discharge mode, in addition to the full ventilation mode and circulation mode.

The first and second suction openings (19) and (21) of the present embodiment are each provided with opening/closing means such as dampers (not shown). And, the first suction opening (19) and the second suction opening (21) are placed in the open or closed  
15 state by the opening/closing means, and these openings are configured so as to either permit or prevent the introduction of a stream of outdoor air *OA* or a stream of room air *RA*.

The dehumidifier means (80) performs a dehumidification operation in air supply mode in which outdoor air *OA* is brought in, moisture in the outdoor air *OA* is adsorbed by the adsorbent material of the heat exchanger (3, 5) which becomes an evaporator, and the  
20 outdoor air *OA* thus converted into a stream of dehumidified air is supplied indoors while, on the other hand, outdoor air *OA* is brought in, moisture desorbed from the adsorbent material of the heat exchanger (5, 3) which becomes a condenser is released to the outdoor air *OA* whereby the adsorbent material of the heat exchanger (5, 3) is regenerated, and the outdoor air *OA* converted into a stream of humidified air is discharged outdoors.

25 The humidifier means (81) performs a humidification operation in air supply

mode in which outdoor air *OA* is brought in, moisture in the outdoor air *OA* is adsorbed by the adsorbent material of the heat exchanger (3, 5) which becomes an evaporator, and the outdoor air *OA* thus converted into a stream of dehumidified air is discharged outdoors while, on the other hand, outdoor air *OA* is brought in, moisture desorbed from the  
 5 adsorbent material of the heat exchanger (5, 3) which becomes a condenser is released to the outdoor air *OA* whereby the adsorbent material of the heat exchanger (5, 3) is regenerated, and the outdoor air *OA* thus converted into a stream of humidified air is supplied indoors.

The dehumidifier means (80) performs a dehumidification operation in air  
 10 discharge mode in which room air *RA* is brought in, moisture in the room air *RA* is adsorbed by the adsorbent material of the heat exchanger (3, 5) which becomes an evaporator, and the room air *RA* thus converted into a stream of dehumidified air is supplied indoors while, on the other hand, room air *RA* is brought in, moisture desorbed from the adsorbent material of the heat exchanger (5, 3) which becomes a condenser is  
 15 released to the room air *RA* whereby the adsorbent material of the heat exchanger (5, 3) is regenerated, and the room air *RA* thus converted into a stream of humidified air is discharged outdoors.

The humidifier means (81) performs a humidification operation in air discharge mode in which room air *RA* is brought in, moisture in the room air *RA* is adsorbed by the  
 20 adsorbent material of the heat exchanger (3, 5) which becomes an evaporator, and the room air *RA* thus converted into a stream of dehumidified air is discharged outdoors while, on the other hand, room air *RA* is brought in, moisture desorbed from the adsorbent material of the heat exchanger (5, 3) which becomes a condenser is released to the room air *RA* whereby the adsorbent material of the heat exchanger (5, 3) is regenerated, and the room  
 25 air *RA* thus converted into a stream of humidified air is supplied indoors.

The refrigerant circuit (1) of the present embodiment is identical in configuration and refrigerant circulation with the refrigerant circuit (1) of the first embodiment, and the description thereof is omitted accordingly.

Figures 18-25(b) correspond to Figure 3 of the first embodiment. Figures 18-25(A) correspond to Figure 4 of the first embodiment. Figures 18-25(C) correspond to Figure 5 of the first embodiment. And, in Figures 18-25(A) and (C), when the first to eight dampers (47-41) are placed in the closed state, they are marked with diagonal lines and, when placed in the open state, they have no marks.

#### DEHUMIDIFICATION OPERATION IN AIR SUPPLY MODE

During a dehumidification operation in air supply mode by the dehumidifier means (80), the first suction opening (19) is opened while, on the other hand, the second suction opening (21) is closed (see Figures 18 and 19). Outdoor air *OA* is brought in as the first air and as the second air, wherein a portion of the outdoor air *OA* is supplied indoors while, simultaneously, the remaining portion of the outdoor air *OA* is discharged outdoors.

#### FIRST OPERATION

Referring to Figure 18, in the first operation, the process of adsorption in the second heat exchanger (5) and the process of regeneration (desorption) in the first heat exchanger (3) are carried out. In other words, in the first operation, moisture present in one portion of outdoor air *OA* is adsorbed in the second heat exchanger (5) and moisture desorbed from the first heat exchanger (3) is given to the remaining portion of the outdoor air *OA*.

During the first operation, the first damper (47), the second damper (49), the third damper (51), and the eighth damper (41) are placed in the open state while, on the other hand, the fourth damper (53), the fifth damper (35), the sixth damper (37), and the seventh

damper (39) are placed in the closed state. One portion of outdoor air *OA* is supplied to the first heat exchanger (3). The remaining portion of the outdoor air *OA* is supplied to the second heat exchanger (5).

An inflow of outdoor air *OA* entering through the first suction opening (19) travels  
 5 through the first inflow passage (63), wherein one portion of the outdoor air *OA* flows into the second heat exchange chamber (73) from the second opening (33b). In the second heat exchange chamber (73), moisture in the outdoor air *OA* is adsorbed by the adsorbent material of the second heat exchanger (5), whereby the outdoor air *OA* is dehumidified. The outdoor air *OA* thus dehumidified becomes a stream of humidity conditioned air *SA*.  
 10 The humidity conditioned air *SA* exiting the second heat exchange chamber (73) flows through the second outflow passage (59) by way of the eighth opening (31d). Then, after passage through the second fan (77), the humidity conditioned air *SA* is supplied indoors through the second outlet opening (25).

On the other hand, the remaining portion of the outdoor air *OA* entering through  
 15 the first suction opening (19) flows into the first heat exchange chamber (69) from the first opening (33a). In the first heat exchange chamber (69), moisture desorbed from the adsorbent material of the first heat exchanger (3) is released to the outdoor air *OA*, whereby the outdoor air *OA* is humidified. The outdoor air *OA* thus converted into humidified air becomes a stream of emission air *EA*. The emission air *EA* exiting the first  
 20 heat exchange chamber (69) flows through the first outflow passage (65) by way of the third opening (33c). Then, after passage through the first fan (79), the emission air *EA* is discharged outdoors through the first outlet opening (23).

Upon completion of the execution of the first operation, the second operation is carried out.

## 25 SECOND OPERATION

Referring to Figure 19, in the second operation, the process of adsorption in the first heat exchanger (3) and the process of regeneration in the second heat exchanger (5) are carried out. In other words, in the second operation, moisture present in one portion of outdoor air *OA* is adsorbed in the first heat exchanger (3) and moisture desorbed out of the second heat exchanger (5) is given to the remaining portion of the outdoor air *OA*.

During the second operation, the first damper (47), the second damper (49), the fourth damper (53), and the seventh damper (39) are placed in the open state while, on the other hand, the third damper (51), the fifth damper (35), the sixth damper (37), and the eighth damper (41) are placed in the closed state. One portion of outdoor air *OA* is supplied to the first heat exchanger (3). The remaining portion of the outdoor air *OA* is supplied to the second heat exchanger (5).

An inflow of outdoor air *OA* entering through the first suction opening (19) travels through the first inflow passage (63), wherein one portion of the outdoor air *OA* flows into the first heat exchange chamber (69) from the first opening (33a). In the first heat exchange chamber (69), moisture present in the outdoor air *OA* is adsorbed by the adsorbent material of the first heat exchanger (3), whereby the outdoor air *OA* is dehumidified. The outdoor air *OA* thus converted into dehumidified air becomes a stream of humidity conditioned air *SA*. The humidity conditioned air *SA* exiting the first heat exchange chamber (69) flows through the second outflow passage (59) by way of the seventh opening (31c). Then, after passage through the second fan (77), the humidity conditioned air *SA* is supplied indoors through the second outlet opening (25).

On the other hand, the remaining portion of the outdoor air *OA* entering through the first suction opening (19) flows into the second heat exchange chamber (73) from the second opening (33b). In the second heat exchange chamber (73), moisture desorbed from the adsorbent material of the second heat exchanger (5) is released to the outdoor air

*OA*, whereby the outdoor air *OA* is humidified. The outdoor air *OA* thus converted into humidified air becomes a stream of emission air *EA*. The emission air *EA* exiting the second heat exchange chamber (73) flows through the first outflow passage (65) by way of the fourth opening (33d). Then, after passage through the first fan (79), the emission air

5 *EA* is discharged outdoors through the first outlet opening (23).

After execution of the second operation, the first operation is carried out again. And, indoor space dehumidification is continuously carried out by repetition of the first operation and the second operation.

#### **HUMIDIFICATION OPERATION IN AIR SUPPLY MODE**

10 During a humidification operation in air supply mode by the humidifier means (81), the first suction opening (19) is opened while, on the other hand, the second suction opening (21) is closed (see Figures 20 and 21), as in the dehumidification operation. Outdoor air *OA* is brought in to serve as the first air and as the second air. One portion of the outdoor air *OA* is supplied indoors while, simultaneously, the remaining portion of the

15 outdoor air *OA* is discharged outdoors.

#### **FIRST OPERATION**

Referring to Figure 20, in the first operation, the process of adsorption in the second heat exchanger (5) and the process of regeneration (desorption) in the first heat exchanger (3) are carried out. In other words, in the first operation, moisture present in

20 one portion of outdoor air *OA* is adsorbed in the second heat exchanger (5) and moisture desorbed from the first heat exchanger (3) is given to the remaining portion of the outdoor air *OA*.

During the first operation, the first damper (47), the second damper (49), the fourth damper (53), and the seventh damper (39) are placed in the open state while, on the

25 other hand, the third damper (51), the fifth damper (35), the sixth damper (37), and the

eighth damper (41) are placed in the closed state. One portion of outdoor air *OA* is supplied to the first heat exchanger (3). The remaining portion of the outdoor air *OA* is supplied to the second heat exchanger (5).

An inflow of outdoor air *OA* entering through the first suction opening (19) travels  
 5 through the first inflow passage (63), wherein one portion of the outdoor air *OA* flows into the first heat exchange chamber (69) from the first opening (33a). In the first heat exchange chamber (69), moisture desorbed from the adsorbent material of the first heat exchanger (3) is released to the outdoor air *OA*, whereby the outdoor air *OA* is humidified. The outdoor air *OA* thus converted into humidified air becomes a stream of humidity  
 10 conditioned air *SA*. The humidity conditioned air *SA* exiting the first heat exchange chamber (69) flows through the second outflow passage (59) by way of the seventh opening (31c). Then, after passage through the second fan (77), the humidity conditioned air *SA* is supplied indoors through the second outlet opening (25).

On the other hand, the remaining portion of the outdoor air *OA* entering through  
 15 the first suction opening (19) flows into the second heat exchange chamber (73) from the second opening (33b). In the second heat exchange chamber (73), moisture present in the outdoor air *OA* is adsorbed by the adsorbent material of the second heat exchanger (5), whereby the outdoor air *OA* is dehumidified. The outdoor air *OA* thus converted into dehumidified air becomes a stream of emission air *EA*. The emission air *EA* exiting the  
 20 second heat exchange chamber (73) flows through the first outflow passage (65) by way of the fourth opening (33d). Then, after passage through the first fan (79), the emission air *EA* is discharged outdoors through the first outlet opening (23).

Upon completion of the execution of the second operation, the first operation is carried out again.

## 25 SECOND OPERATION

Referring to Figure 21, in the second operation, the process of adsorption in the first heat exchanger (3) and the process of regeneration in the second heat exchanger (5) are carried out. In other words, in the second operation, moisture present in one portion of outdoor air *OA* is adsorbed in the first heat exchanger (3) and moisture desorbed from the second heat exchanger (5) is given to the remaining portion of the outdoor air *OA*.

During the second operation, the first damper (47), the second damper (49), the third damper (51), and the eighth damper (41) are placed in the open state while, on the other hand, the fourth damper (53), the fifth damper (35), the sixth damper (37), and the seventh damper (39) are placed in the closed state. One portion of outdoor air *OA* is supplied to the first heat exchanger (3). The remaining portion of the outdoor air *OA* is supplied to the second heat exchanger (5).

An inflow of outdoor air *OA* entering through the first suction opening (19) travels through the first inflow passage (63), wherein one portion of the outdoor air *OA* flows into the second heat exchange chamber (73) from the second opening (33b). In the second heat exchange chamber (73), moisture desorbed from the adsorbent material of the second heat exchanger (5) is released to the outdoor air *OA*, whereby the outdoor air *OA* is humidified. The outdoor air *OA* thus converted into humidified air becomes a stream of humidity conditioned air *SA*. The humidity conditioned air *SA* exiting the second heat exchange chamber (73) flows through the second outflow passage (59) by way of the eighth opening (31d). Then, after passage through the second fan (77), the humidity conditioned air *SA* is supplied indoors through the second outlet opening (25).

On the other hand, the remaining portion of the outdoor air *OA* entering through the first suction opening (19) flows into the first heat exchange chamber (69) from the first opening (33a). In the first heat exchange chamber (69), moisture in the outdoor air *OA* is adsorbed by the adsorbent material of the first heat exchanger (3), whereby the outdoor air

*OA* is dehumidified. The outdoor air *OA* thus converted into dehumidified air becomes a stream of emission air *EA*. The emission air *EA* exiting the first heat exchange chamber (69) flows through the first outflow passage (65) by way of the third opening (33c). Then, after passage through the first fan (79), the emission air *EA* is discharged outdoors through the first outlet opening (23).

After execution of the second operation, the first operation is carried out again. And, indoor space humidification is continuously carried out by repetition of the first operation and the second operation.

#### DEHUMIDIFICATION OPERATION IN AIR DISCHARGE MODE

During a dehumidification operation in air discharge mode by the dehumidifier means (80), the second suction opening (21) is opened while, on the other hand, the first suction opening (19) is closed (see Figures 22 and 23). Room air *RA* is brought in to serve as the first air and as the second air. One portion of the room air *RA* is supplied indoors while, simultaneously, the remaining portion of the room air *RA* is discharged outdoors.

#### FIRST OPERATION

Referring to Figure 22, in the first operation, the process of adsorption in the second heat exchanger (5) and the process of regeneration (desorption) in the first heat exchanger (3) are carried out. In other words, in the first operation, moisture present in one portion of room air *RA* is adsorbed in the second heat exchanger (5) and moisture desorbed from the first heat exchanger (3) is given to the remaining portion of the room air *RA*.

During the first operation, the third damper (51), the fifth damper (35), the sixth damper (37), and the eighth damper (41) are placed in the open state while, on the other hand, the first damper (47), the second damper (49), the fourth damper (53), and the

seventh damper (39) are placed in the closed state. And, one portion of room air *RA* is supplied to the first heat exchanger (3). The remaining portion of the room air *RA* is supplied to the second heat exchanger (5).

An inflow of room air *RA* entering through the second suction opening (21)  
 5 travels through the second inflow passage (57), wherein one portion of the room air *RA* flows into the second heat exchange chamber (73) from the sixth opening (31b). In the second heat exchange chamber (73), moisture present in the room air *RA* is adsorbed by the adsorbent material of the second heat exchanger (5), whereby the room air *RA* is dehumidified. The room air *RA* thus converted into dehumidified air becomes a stream of  
 10 humidity conditioned air *SA*. The humidity conditioned air *SA* exiting the second heat exchange chamber (73) flows through the second outflow passage (59) by way of the eighth opening (31d). Then, after passage through the second fan (77), the humidity conditioned air *SA* is supplied indoors through the second outlet opening (25).

On the other hand, the remaining portion of the room air *RA* entering through the  
 15 second suction opening (21) flows into the first heat exchange chamber (69) from the fifth opening (31a). In the first heat exchange chamber (69), moisture desorbed from the adsorbent material of the first heat exchanger (3) is released to the room air *RA*, whereby the room air *RA* is humidified. The room air *RA* thus converted into humidified air becomes a stream of emission air *EA*. The emission air *EA* exiting the first heat exchange  
 20 chamber (69) flows through the first outflow passage (65) by way of the third opening (33c). Then, after passage through the first fan (79), the emission air *EA* is discharged outdoors through the first outlet opening (23).

Upon completion of the execution of the second operation, the first operation is carried out again.

## 25 SECOND OPERATION

Referring to Figure 23, in the second operation, the process of adsorption in the first heat exchanger (3) and the process of regeneration in the second heat exchanger (5) are carried out. In other words, in the second operation, moisture present in one portion of room air *RA* is adsorbed in the first heat exchanger (3) and moisture desorbed from the second heat exchanger (5) is given to the remaining portion of the room air *RA*.

During the second operation, the first suction opening (19) is closed. In addition, the fourth damper (53), the fifth damper (35), the sixth damper (37), and the seventh damper (39) are placed in the open state while, on the other hand, the first damper (47), the second damper (49), the third damper (51), and the eighth damper (41) are placed in the closed state. One portion of room air *RA* is supplied to the first heat exchanger (3). The remaining portion of the room air *RA* is supplied to the second heat exchanger (5).

An inflow of room air *RA* entering through the second suction opening (21) travels through the second inflow passage (57), wherein one portion of the room air *RA* flows into the first heat exchange chamber (69) from the fifth opening (31a). In the first heat exchange chamber (69), moisture in the room air *RA* is adsorbed by the adsorbent material of the first heat exchanger (3), whereby the room air *RA* is dehumidified. The room air *RA* thus converted into dehumidified air becomes a stream of humidity conditioned air *SA*. The humidity conditioned air *SA* exiting the first heat exchange chamber (69) flows through the second outflow passage (59) by way of the seventh opening (31c). Then, after passage through the second fan (77), the humidity conditioned air *SA* is supplied indoors through the second outlet opening (25).

On the other hand, the remaining portion of the room air *RA* entering through the second suction opening (21) flows into the second heat exchange chamber (73) from the sixth opening (31b). In the second heat exchange chamber (73), moisture desorbed from the adsorbent material of the second heat exchanger (5) is released to the room air *RA*,

whereby the room air *RA* is humidified. The room air *RA* thus converted into humidified air becomes a stream of emission air *EA*. The emission air *EA* exiting the second heat exchange chamber (73) flows through the first outflow passage (65) by way of the fourth opening (33d). Then, after passage through the first fan (79), the emission air *EA* is  
 5 discharged outdoors through the first outlet opening (23).

After execution of the second operation, the first operation is carried out again. And, indoor space dehumidification is continuously carried out by repetition of the first operation and the second operation.

#### **HUMIDIFICATION OPERATION IN AIR DISCHARGE MODE**

10 During a humidification operation in air discharge mode by the humidifier means (81), the second suction opening (21) is opened while, on the other hand, the first suction opening (19) is closed (see Figures 24 and 25), as in the dehumidification operation. Room air *RA* is brought in to serve as the first air and as the second air. One portion of the room air *RA* is supplied indoors while, simultaneously, the remaining portion of the  
 15 room air *RA* is discharged outdoors.

#### **FIRST OPERATION**

Referring to Figure 24, in the first operation, the process of adsorption in the second heat exchanger (5) and the process of regeneration (desorption) in the first heat exchanger (3) are carried out. In other words, in the first operation, moisture present in  
 20 one portion of room air *RA* is adsorbed in the second heat exchanger (5) and moisture desorbed from the first heat exchanger (3) is given to the remaining portion of the room air *RA*.

During the first operation, the fourth damper (53), the fifth damper (35), the sixth damper (37), and the seventh damper (39) are placed in the open state while, on the other  
 25 hand, the first damper (47), the second damper (49), the third damper (51), and the eighth

damper (41) are placed in the closed state. One portion of room air *RA* is supplied to the first heat exchanger (3). The remaining portion of the room air *RA* is supplied to the second heat exchanger (5).

An inflow of room air *RA* entering through the second suction opening (21) travels through the second inflow passage (57), wherein one portion of the room air *RA* flows into the first heat exchange chamber (69) from the fifth opening (31a). In the first heat exchange chamber (69), moisture desorbed from the adsorbent material of the first heat exchanger (3) is released to the room air *RA*, whereby the room air *RA* is humidified. The room air *RA* thus converted into humidified air becomes a stream of humidity conditioned air *SA*. The humidity conditioned air *SA* exiting the first heat exchange chamber (69) flows through the second outflow passage (59) by way of the seventh opening (31c). Then, after passage through the second fan (77), the humidity conditioned air *SA* is supplied indoors through the second outlet opening (25).

On the other hand, the remaining portion of the room air *RA* entering through the second suction opening (21) flows into the second heat exchange chamber (73) from the sixth opening (31b). In the second heat exchange chamber (73), moisture present in the room air *RA* is adsorbed by the adsorbent material of the second heat exchanger (5), whereby the room air *RA* is dehumidified. The room air *RA* thus converted into dehumidified air becomes a stream of emission air *EA*. The emission air *EA* exiting the second heat exchange chamber (73) flows through the first outflow passage (65) by way of the fourth opening (33d). Then, after passage through the first fan (79), the emission air *EA* is discharged outdoors through the first outlet opening (23).

Upon completion of the execution of the first operation, the second operation is carried out.

## 25 SECOND OPERATION

Referring now to Figure 25, in the second operation, the process of adsorption in the first heat exchanger (3) and the process of regeneration in the second heat exchanger (5) are carried out. In other words, in the second operation, moisture present in one portion of room air *RA* is adsorbed in the first heat exchanger (3) and moisture desorbed from the second heat exchanger (5) is given to the remaining portion of the room air *RA*.

During the second operation, the third damper (51), the fifth damper (35), the sixth damper (37), and the eighth damper (41) are placed in the open state while, on the other hand, the first damper (47), the second damper (49), the fourth damper (53), and the seventh damper (39) are placed in the closed state. And, one portion of room air *RA* is supplied to the first heat exchanger (3). The remaining portion of the room air *RA* is supplied to the second heat exchanger (5).

An inflow of room air *RA* entering through the second suction opening (21) travels through the second inflow passage (57), wherein one portion of the room air *RA* flows into the second heat exchange chamber (73) from the sixth opening (31b). In the second heat exchange chamber (73), moisture desorbed from the adsorbent material of the second heat exchanger (5) is released to the room air *RA*, whereby the room air *RA* is humidified. The room air *RA* thus converted into humidified air becomes a stream of humidity conditioned air *SA*. The humidity conditioned air *SA* exiting the second heat exchange chamber (73) flows through the second outflow passage (59) by way of the eighth opening (31d). Then, after passage through the second fan (77), the humidity conditioned air *SA* is supplied indoors through the second outlet opening (25).

On the other hand, the remaining portion of the room air *RA* entering through the second suction opening (21) flows into the first heat exchange chamber (69) from the fifth opening (31a). In the first heat exchange chamber (69), moisture in the room air *RA* is adsorbed by the adsorbent material of the first heat exchanger (3), whereby the room air

*RA* is dehumidified. The room air *RA* thus converted into dehumidified air becomes a stream of emission air *EA*. The emission air *EA* exiting the first heat exchange chamber (69) flows through the first outflow passage (65) by way of the third opening (33c). Then, after passage through the first fan (79), the emission air *EA* is discharged outdoors through the first outlet opening (23).

After execution of the second operation, the first operation is carried out again. And, indoor space humidification is continuously carried out by repetition of the first operation and the second operation.

Accordingly, in accordance with the second embodiment, dehumidification or humidification is carried out in air supply mode, thereby making it possible to perform indoor space dehumidification or humidification by only the taking-in of outdoor air *OA*.

In addition, dehumidification or humidification is carried out in air discharge mode, thereby making it possible to perform indoor space dehumidification or humidification by only the discharging of room air *RA*.

## OTHER EMBODIMENTS OF INVENTION

In the foregoing embodiments, in each of the first heat exchanger (3) and the second heat exchanger (5), adsorbent materials are supported on the external surface of each fin (13) as well as on the external surface of each heat transfer tube (15). In the present invention, however, it may be arranged that adsorbent materials are supported on at least either one of the external surface of the fin (13) and the external surface of the heat transfer tube (15).

In addition, in the foregoing embodiments, adsorbent materials are supported on both the first heat exchanger (3) and the second heat exchanger (5); however, it may be arranged that adsorbent material is supported on one of the heat exchangers, i.e., only on the first heat exchanger (3). In this case, moisture adsorption and moisture desorption

(adsorbent material regeneration) take place at intervals.

And, also in this case, both a moisture absorbing means and a moisture releasing means are provided.

In the above case, the moisture absorbing means is configured to switch the  
5 circulation of refrigerant in the refrigerant circuit (1) and the distribution of air by the dampers (47-53, 35-41), whereby refrigerant is condensed in the second heat exchanger (5) while, simultaneously, refrigerant is evaporated in the first heat exchanger (3) so that moisture present in a stream of air flowing through the first heat exchanger (3) is adsorbed by the adsorbent material of the first heat exchanger (3).

10 In addition, the moisture releasing means is configured to switch the circulation of refrigerant in the refrigerant circuit (1) and the distribution of air by the dampers (47-53, 35-41), whereby refrigerant is condensed in the first heat exchanger (3) while, simultaneously, refrigerant is evaporated in the second heat exchanger (5) so that moisture is released to a stream of air flowing through the first heat exchanger (3) for the purpose of  
15 adsorbent-material regeneration.

Furthermore, the present invention is not limited to the arrangement of each of the embodiments shown in plan view (see Figure 3). That is, the orientation of arrangement is not limited to that of the embodiments.

In addition, in the above-described embodiments, the first fan (79) is connected to  
20 the first outlet opening (23). Alternatively, the first fan (79) may be provided so as to communicate with the first suction opening (19). In addition, the second fan (77) is connected to the second outlet opening (25). Alternatively, the second fan (77) may be provided so as to communicate with the second suction opening (21). In other words, the first fan (79) and the second fan (77) may be of the draw-through type or the forced draft  
25 type.

Furthermore, the first embodiment is switchable between full ventilation mode and circulation mode and the second embodiment is switchable between full ventilation mode, circulation mode, air supply mode, and air discharge mode. Alternatively, it may be arranged that the dehumidifier means (80) (the humidifier means (81)) of the present invention performs at least any one of full ventilation mode, circulation mode, air supply mode, and air discharge mode.

Finally, it may be arranged in the present invention that either one of the dehumidifier means (80) and the humidifier means (81) is provided.

#### **INDUSTRIAL APPLICABILITY**

10 As has been described above, the humidity controller apparatuses of the present invention are useful in dehumidifying or humidifying an indoor space or the like and are particularly suited for the integral formation of an adsorbent material and a heat exchange means.

## CLAIMS

## WHAT IS CLAIMED IS:

1. A humidity controller apparatus, provided with an adsorbent material, for  
5 regulating the humidity of air by effecting moisture adsorption onto the adsorbent material  
and moisture desorption from the adsorbent material, wherein:

a refrigerant circuit (1) is provided which comprises a first heat exchanger (3) and  
a second heat exchanger (5), which performs a vapor compression refrigeration cycle by  
the circulation of a refrigerant therethrough, and which alternately effects refrigerant  
10 condensation and evaporation in said first and second heat exchangers (3, 5); and  
the adsorbent material is supported on at least a surface of said first heat  
exchanger (3).

2. The humidity controller apparatus of claim 1 comprising:

15 moisture absorbing means for switching refrigerant circulation in said refrigerant  
circuit (1) and air distribution so that refrigerant condensation takes place in said second  
heat exchanger (5) while, simultaneously, refrigerant evaporation takes place in said first  
heat exchanger (3), whereby moisture present in a stream of air flowing through said first  
heat exchanger (3) is adsorbed by the adsorbent material, and

20 moisture releasing means for switching refrigerant circulation in said refrigerant  
circuit (1) and air distribution so that refrigerant condensation takes place in said first heat  
exchanger (3) while, simultaneously, refrigerant evaporation takes place in said second  
heat exchanger (5), whereby the adsorbent material is regenerated by the release of  
moisture therefrom to a stream of air flowing through said first heat exchanger (3).

3. The humidity controller apparatus of claim 1, wherein:

the adsorbent material is supported on a surface of said first heat exchanger (3) and a surface of said second heat exchanger (5); and

dehumidifier means (80) for switching refrigerant circulation in said refrigerant circuit (1) and air distribution is provided so that moisture in a stream of air flowing through a refrigerant evaporating heat exchanger (3, 5) is adsorbed by the adsorbent material of said refrigerant evaporating heat exchanger (3, 5) while, simultaneously, the adsorbent material of a refrigerant condensing heat exchanger (5, 3) is regenerated by the release of moisture therefrom to a stream of air flowing through said refrigerant condensing heat exchanger (5, 3), whereby the stream of air dehumidified by the adsorbent material of said refrigerant evaporating heat exchanger (3, 5) is supplied to an indoor space.

4. The humidity controller apparatus of claim 1, wherein:

the adsorbent material is supported on a surface of said first heat exchanger (3) and a surface of said second heat exchanger (5); and

humidifier means (81) for switching refrigerant circulation in said refrigerant circuit (1) and air distribution is provided so that moisture in a stream of air flowing through a refrigerant evaporating heat exchanger (3, 5) is adsorbed by the adsorbent material of said refrigerant evaporating heat exchanger (3, 5) while, simultaneously, the adsorbent material of a refrigerant condensing heat exchanger (5, 3) is regenerated by the release of moisture therefrom to a stream of air flowing through said refrigerant condensing heat exchanger (5, 3), whereby the stream of air humidified by the adsorbent material of said refrigerant evaporating heat exchanger (5, 3) is supplied to an indoor space.

5. The humidity controller apparatus of claim 1, wherein:

the adsorbent material is supported on a surface of said first heat exchanger (3) as well as on a surface of said second heat exchanger (5);

dehumidifier means (80) for switching refrigerant circulation in said refrigerant circuit (1) and air distribution is provided so that moisture in a stream of air flowing through a refrigerant evaporating heat exchanger (3, 5) is adsorbed by the adsorbent material of said refrigerant evaporating heat exchanger (3, 5) while the adsorbent material of a refrigerant condensing heat exchanger (5, 3) is regenerated by the release of moisture therefrom to a stream of air flowing through said refrigerant condensing heat exchanger (5, 3), whereby the stream of air dehumidified by the adsorbent material of said refrigerant evaporating heat exchanger (3, 5) is supplied to an indoor space;

humidifier means (81) for switching refrigerant circulation in said refrigerant circuit (1) and air distribution is provided so that moisture in a stream of air flowing through a refrigerant evaporating heat exchanger (3, 5) is adsorbed by the adsorbent material of said refrigerant evaporating heat exchanger (3, 5) while the adsorbent material of a refrigerant condensing heat exchanger (5, 3) is regenerated by the release of moisture therefrom to a stream of air flowing through said refrigerant condensing heat exchanger (5, 3), whereby the stream of air humidified by the adsorbent material of said refrigerant evaporating heat exchanger (5, 3) is supplied to an indoor space; and

said dehumidifier means (80) and said humidifier means (81) are configured to operate switchably between a dehumidification mode of operation and a humidification mode of operation.

6. The humidity controller apparatus of claim 1 comprising:

a first heat exchange chamber (69) for accommodating said first heat exchanger (3);

a second heat exchange chamber (73), formed adjacently to said first heat exchange chamber (69), for accommodating said second heat exchanger (5);

a first air inflow passage (63) and a first air outflow passage (65) which are formed along one end surface in a thickness direction in which respective one surfaces of  
 5 said two heat exchange chambers (69, 73) continue and which are arranged in a superimposed manner in the thickness direction of both said heat exchange chambers (69, 73);

a second air inflow passage (57) and a second air outflow passage (59) which are formed along another end surface which is an end surface where respective one surfaces of  
 10 said two heat exchange chambers (69, 73) continue and which is situated face to face with said one end surface, and which are arranged in a superimposed manner in the thickness direction of both said heat exchange chambers (69, 73); and

opening/closing means (35, ..., 47, ...) for opening and closing openings (31a, ..., 33a, ...) for communication between said first heat exchange chamber (69) and said  
 15 second heat exchange chamber (73), and each said inflow passage (57, 63) and each said outflow passage (59, 65).

7. The humidity controller apparatus of claim 5, wherein:

said dehumidifier means (80) is configured so as to perform a dehumidification  
 20 operation in full ventilation mode in which: outdoor air is brought in, moisture in the outdoor air is adsorbed by the adsorbent material of said heat exchanger (3, 5) which becomes an evaporator, and the outdoor air thus converted into a stream of dehumidified air is supplied indoors while, on the other hand, room air is brought in, moisture desorbed from said adsorbent material of said heat exchanger (5, 3) which becomes a condenser is  
 25 released to the room air whereby said adsorbent material of said heat exchanger (5, 3) is

regenerated, and the room air thus converted into a stream of humidified air is discharged outdoors; and

said humidifier means (81) is configured so as to perform a humidification operation in full ventilation mode in which: room air is brought in, moisture in the room air is adsorbed by said adsorbent material of said heat exchanger (3, 5) which becomes an evaporator, and the room air thus converted into a stream of dehumidified air is discharged outdoors while, on the other hand, outdoor air is brought in, moisture desorbed from said adsorbent material of said heat exchanger (5, 3) as a condenser is released to the outdoor air whereby said adsorbent material of said heat exchanger (5, 3) is regenerated, and the outdoor air thus converted into a stream of humidified air is supplied indoors.

8. The humidity controller apparatus of claim 5, wherein:

said dehumidifier means (80) is configured so as to perform a dehumidification operation in circulation mode in which: room air is brought in, moisture in the room air is adsorbed by said adsorbent material of said heat exchanger (3, 5) which becomes an evaporator, and the room air thus converted into a stream of dehumidified air is supplied indoors while, on the other hand, outdoor air is brought in, moisture desorbed from said adsorbent material of said heat exchanger (5, 3) which becomes a condenser is released to the outdoor air whereby said adsorbent material of said heat exchanger (5, 3) is regenerated, and the outdoor air thus converted into a stream of humidified air is discharged outdoors; and

said humidifier means (81) is configured so as to perform a humidification operation in circulation mode in which: outdoor air is brought in, moisture in the outdoor air is adsorbed by said adsorbent material of said heat exchanger (3, 5) which becomes an evaporator, and the outdoor air thus converted into a stream of dehumidified air is

discharged outdoors while, on the other hand, room air is brought in, moisture desorbed from said adsorbent material of said heat exchanger (5, 3) which becomes a condenser is released to the room air whereby said adsorbent material of said heat exchanger (5, 3) is regenerated, and the room air thus converted into a stream of humidified air is supplied  
5 indoors.

9. The humidity controller apparatus of claim 5, wherein:

said dehumidifier means (80) is configured so as to perform a dehumidification operation in air supply mode in which: outdoor air is brought in, moisture in the outdoor  
10 air is adsorbed by said adsorbent material of said heat exchanger (3, 5) which becomes an evaporator, and the outdoor air thus converted into a stream of dehumidified air is supplied indoors while, on the other hand, outdoor air is brought in, moisture desorbed from said adsorbent material of said heat exchanger (5, 3) which becomes a condenser is released to the outdoor air whereby said adsorbent material of said heat exchanger (5, 3) is regenerated,  
15 and the outdoor air thus converted into a stream of humidified air is discharged outdoors; and

said humidifier means (81) is configured so as to perform a humidification operation in air supply mode in which: outdoor air is brought in, moisture in the outdoor air is adsorbed by said adsorbent material of said heat exchanger (3, 5) which becomes an  
20 evaporator, and the outdoor air thus converted into a stream of dehumidified air is discharged outdoors while, on the other hand, outdoor air is brought in, moisture desorbed from said adsorbent material of said heat exchanger (5, 3) which becomes a condenser is released to the outdoor air whereby said adsorbent material of said heat exchanger (5, 3) is regenerated, and the outdoor air thus converted into a stream of humidified air is supplied  
25 indoors.

10. The humidity controller apparatus of claim 5, wherein:

said dehumidifier means (80) is configured so as to perform a dehumidification operation in air discharge mode in which: room air is brought in, moisture in the room air is adsorbed by said adsorbent material of said heat exchanger (3, 5) which becomes an evaporator, and the room air thus converted into a stream of dehumidified air is supplied indoors while, on the other hand, room air is brought in, moisture desorbed from said adsorbent material of said heat exchanger (5, 3) which becomes a condenser is released to the room air whereby said adsorbent material of said heat exchanger (5, 3) is regenerated, and the room air thus converted into a stream of humidified air is discharged outdoors; and

said humidifier means (81) is configured so as to perform a humidification operation in air discharge mode in which: room air is brought in, moisture in the room air is adsorbed by said adsorbent material of said heat exchanger (3, 5) which becomes an evaporator, and the room air thus converted into a stream of dehumidified air is discharged outdoors while, on the other hand, room air is brought in, moisture desorbed from said adsorbent material of said heat exchanger (5, 3) which becomes a condenser is released to the room air whereby said adsorbent material of said heat exchanger (5, 3) is regenerated, and the room air thus converted into a stream of humidified air is supplied indoors.

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