

"LOCKING DEVICE FOR USE WITH SHAFTS OR TUBES"

TECHNICAL FIELD

The present invention relates to a device which can be selectively locked or clamped to a rotatable component such as a shaft or tube.

BACKGROUND TO THE INVENTION

Our earlier U.K. Patent Application No. 9419372.9 (2 282 209A) describes locking devices which can be used with a rotatable component such as a shaft or a tube. Although such devices achieve a reliable locking function, a collar means of the device tends to tilt when the device is locked in position and hence wobbles when the shaft or tubes revolves. A general object of the present invention is to provide an improved form of locking device which overcomes this defect.

DISCLOSURE OF THE INVENTION

According to one aspect of the invention there is provided a device for locking to a component rotatable about an axis; said device comprising collar means having an end wall with a bore defining a surface for confronting an outer surface of the rotatable component and a region extending axially from the end wall, a ring for disposing



between the axial region of the collar means and the component, the ring having a first inner surface for confronting the outer surface of the component and a second outer surface which is inclined relative to the axis of rotation to converge with respect to the axis of rotation in a direction away from the end wall and several screw-threaded elements received in screw-threaded bores extending through the axial region of the collar means and spaced apart relative to the rotational axis other than at 180° , the threaded bores being disposed in positions non-radial relative to the rotational axis. The arrangement is such that the screw-threaded elements can be tightened to bear into or against the second outer surface of the ring to cause the bore surface of the collar means and the first surface of the ring to act as contact surfaces with respect to the outer surface of the component which exert clamping force on the component at diametrically opposed zones.

According to another aspect of the invention there is provided a device for locking to a hollow component rotatable about an axis; said device comprising collar means having an end wall with an outer surface for confronting an inner surface of the component and a region extending axially from the end wall; a ring for disposing between the axial region of the collar means and the component, the ring having a first outer surface for confronting the inner surface of the component and a second inner surface which is inclined relative to the axis of rotation to diverge with respect to the axis of rotation in a direction away from the end wall and several screw threaded elements received in screw-threaded bores extending through the axial region of the collar means and spaced apart relative to the rotational axis other than at 180° , the threaded bores being disposed in non-radial positions relative to the rotational axis. The



arrangement is such that the screw-threaded elements can be tightened to bear into or against the second inner surface of the ring to cause the outer surface of the ring and the outer surface of the end wall of the collar means to act as contact surfaces with respect to the inner surface of the component which exert clamping force on the component at diametrically opposed zones.

By arranging the screw-threaded elements in bores in non-radial positions on the collar means the tendency for the collar means to wobble as the rotatable component revolves can be prevented.

The threaded bores preferably have axes which intersect at a location offset from the rotational axis in an anti-clockwise sense as viewed in the direction of the rotational axis and from the end wall of the collar means.

Preferably there are two screw-threaded elements with their rotational axes spaced apart around the location where the axes of the associated threaded bores intersect by about 120° . Each screw-threaded element thus has its axis offset from a true radial direction relative to the rotational axis.

The ends of the screw-threaded elements may have plane surfaces normal to their longitudinal axes or surfaces which are set at an angle which generally matches the angle of the second inclined surface of the ring. It is possible to provide shallow depressions, recesses or blind bores in the second surface for receiving the ends of the screw-threaded elements.

The surfaces of the collar means and the ring which confront the component normally lie parallel to the axis of rotation and directly adjacent and alongside one



another to form a continuation of one another prior to the tightening of the screw-threaded elements. Between these contact surfaces there may be radial surfaces of the ring and the collar means which engage on one another and slide against one another as the screw elements are tightened or released.

The rotatable component may be a shaft or spindle with the locking device fitted onto its exterior periphery or a hollow shaft or tube with the locking device fitted inside.

It is not essential for the rotatable component to have a circular cross-section and the component can have a polygonal cross-section.

The invention may be understood more readily and various other aspects and features of the invention may become apparent, from consideration of the following description and accompanying drawings.

BRIEF DESCRIPTION OF DRAWINGS

Embodiments of the invention will now be described by way of examples only, with reference to the accompanying drawing wherein:

Figure 1 is a part sectional side view representation of an assembly employing a locking device constructed in accordance with the invention;

Figure 2 is an end view of the assembly shown in Figure 1 taken in the direction of arrow II of Figure 1;

Figure 3 is a part sectional side view representation of another assembly employing a locking device constructed in accordance with the invention; and



Figure 4 is an end view of the assembly shown in Figure 3 taken in the direction of arrow IV in Figure 3.

DESCRIPTION OF BEST MODES OF CARRYING OUT THE INVENTION

Figures 1 and 3 generally correspond to Figure 1 and 2 of the aforementioned UK patent application 2282209A and use the same reference numerals. As shown in Figures 1 and 2, a shaft 10 carries a rolling-element bearing 11 and a locking device 12. The bearing 11 is composed of an extended inner ring 13 which is to be secured to the shaft 10 with the aid of the device 12 in a non-rotatable manner. The ring 13 has an inner bore surface 9 extending parallel to the outer peripheral surface of the shaft 10. A set of rolling elements, here in the form of balls 14, is disposed in raceways 15 between the inner ring 13 and an outer ring 16. The balls 14 are equi-distantly spaced apart round the raceways 15 with a cage 27 and seals 8 are mounted between the rings 13, 16.

An axial end region 17 of the inner ring 13 is surrounded by an axial region or portion 18 of a collar 19. The collar 19 has an end wall 25 with a bore forming a surface 22 fitted over the shaft 10 with the surface 22 forming a continuation of the inner surface 9 of the ring 13.

The axial region 17 of the bearing inner ring 13 is relieved to provide an outer surface 20 which is tapered to extend at an angle to the axis of the shaft 10. The surface 20 converges towards the shaft axis in the direction of the bearing 11. A typical angle for the surface 20 is 3° and generally an angle in the range 1° to 5° is preferred.



Two screws 21 are located in threaded bores 6 in the axial portion 18 of the collar 19.

As shown in Figure 2, the screws 21 and the bores 6 are offset by the same distance 'd' from radial planes R in an anti clockwise direction. The radial planes R are spaced apart by 120° and represent the preferred locations of the screws 21 in accordance with UK patent application No. 2282209A. In contrast to the latter, the longitudinal axes of the screws 21 now intersect at point L spaced from the axis of rotation P and offset therefrom in an anti clockwise sense as viewed in the direction of arrow II, i.e. in the direction of the axis P from the end wall 25 of the collar 19 and the convergence of the surface 20.

The screws 21 can be displaced inwardly of the bores 6 to engage on the outer surface 20 of the ring 13. In a modified arrangement the surface 20 has shallow recesses or depressions or blind bores for locating and receiving the ends 23 of the screws 21. The inner ring 13 of the bearing 11 and the collar 19 can be easily fitted onto the shaft 10 and the screws 21 are then tightened to some predetermined torque. The ends 23 of the screws 21, which can be tapered at the same angle as the surface 20 serve to contact the outer surface 20. When tightened, the ends 23 of the screws 21 press on the outer surface 20 to displace the axial end region 17 of the inner ring 13, and hence the entire inner ring 13, and the axial portion 18 of the collar 19 radially apart. This in turn causes diametrically opposed contact surface zones 24, 26 of the inner bore surface 9 of the ring 13 and of the bore surface 22 in the end wall 25 of the collar 19 to clamp against the shaft 10 radially and uniformly without causing damage to the shaft 10. To release the



locking device 12, the screws 21 are untightened to permit the collar 19 and the bearing 11 to be withdrawn from the shaft 10.

The end wall 25 of the collar 19 and the end of the axial region 17 of the ring 13 abut to form sliding guide surfaces 50 as the screws 21 are tightened or released.

As shown in Figures 3 and 4 another locking device 12 is fitted inside a hollow shaft or tube 120. The locking device is again composed of a collar 190 and a separate ring 170. The collar 190 has an axial region 180 with two angularly offset threaded bores 220 receiving screws 210. An end wall 250 of the collar 190 has an outer peripheral surface 280 which is parallel to and confronts an inner surface 300 of the tube 120. The ring 170 has an outer peripheral surface 240 which is parallel to and likewise confronts the inner surface 300 of the tube 120 and an inner surface 200 which is tapered at an angle to the axis of the tube 120. The surface 200 diverges from the end wall 250 relative to the axis of rotation P. As with the first embodiment, the surface 200 can be inclined at an angle of 1° to 5° and preferably 3° to the axis of the tube 120.

The surface 200 can contain shallow recesses, depressions or blind bores for receiving the ends 230 of the screws 210.

As shown in Figure 4, the screws 210 and the threaded bores 220 are offset by the same distance 'd' from radial planes R, in an anti clockwise direction. The radial planes R are spaced apart by 120° and represent the preferred locations of the screws 210 in accordance with UK patent application 2282209A. In contrast to the latter, the longitudinal axes of the screws 210 now intersect at a point L spaced from the axis of



rotation P and offset therefrom in an anti clockwise sense as viewed in the direction of arrow IV, i.e. in the direction of the axis P and the divergence of the surface 200 from the end wall 250 of the collar 190.

The device 12 is locked to the tube 120 by tightening the screws 210 to cause their ends 230 to displace the axial region 180 of the collar 190 and the ring 170 radially apart. This in turn causes diametrically opposed surface contact zones 290, 291 of the surface 280 of the collar end wall 250 and of the surface 240 of the ring 170 to bear against the inner surface 300 of the tube 120 to lock the device in place.

The ring 170 and the end wall of the collar 190 have radial faces which abut and establish slidable guide surfaces 350 as the screws 210 are tightened or released. The device can be used to provide an abutment or location for some other component such as a bearing.

In contrast to some known forms of locking device or mechanisms, a device in accordance with the invention does not damage the rotatable component and is easily fitted and removed repeatedly. The device also rotates uniformly with the rotatable component without any wobble.

In the Figure 1 arrangement the inner ring 13 is simply extended and easily machined in one operation. There is no need to provide tapped bores in the ring 13. Also since no excessive tensile stresses are created on the inner ring 13 there is no danger of fracture for this reason. There is also no distortion so the raceways for the balls are unaffected.



**Claims**

1. A device (12) for locking to a component (10) rotatable about an axis (P); said device comprising collar means (19) having an end wall (25) with a bore defining a surface (22) for confronting an outer surface of the rotatable component (10) and a region (18) extending axially from the end wall, a ring (13) for disposing between the axial region (18) of the collar means and the component (10), the ring (13) having a first inner surface (9) for confronting the outer surface of the component (10) and a second outer surface (20) which is inclined relative to the axis of rotation (P) to converge with respect to the axis of rotation in a direction away from the end wall (25) and several screw-threaded elements (21) received in screw-threaded bores (6) extending through the axial region (18) of the collar means (19) and spaced apart relative to the rotational axis (P) other than at 180°, the arrangement being such that the screw-threaded elements (21) can be tightened to bear into or against the second outer surface (20) of the ring (13) to cause the bore surface (22) of the collar means (19) and the first surface (9) of the ring (13) to act as contact surfaces with respect to the outer surface of the component (10) which exert clamping force on the component at diametrically opposed zones (24, 26); **characterized in that** the threaded bores (6) are disposed in positions non-radial relative to the rotational axis.

2. A device (12) for locking to a hollow component (120) rotatable about an axis (P); said device comprising collar means (190) having an end wall (250) with an outer surface (280) for confronting an inner surface (300) of the component (120) and a region



(180) extending axially from the end wall (250); a ring (170) for disposing between the axial region (180) of the collar means (190) and the component (120), the ring (170) having a first outer surface (240) for confronting the inner surface (300) of the component and a second inner surface (200) which is inclined relative to the axis of rotation to diverge with respect to the axis of rotation in a direction away from the end wall (250) and several screw threaded elements (210) received in screw-threaded bores (220) extending through the axial region (180) of the collar means (190) and spaced apart relative to the rotational axis other than at 180° the arrangement being such that the screw-threaded elements (210) can be tightened to bear into or against the second inner surface (200) of the ring (170) to cause the outer surface (240) of the ring and the outer surface (280) of the end wall (250) of the collar means (190) to act as contact surfaces with respect to the inner surface (300) of the component (120) which exert clamping force on the component at diametrically opposed zones; characterized in that the threaded bores (220) are disposed in non-radial positions relative to the rotational axis (P).

3. A device according to claim 1 or 2 wherein the threaded bores (6, 220) have axes which intersect at a location (L) offset from the rotational axis (P) in an anti-clockwise sense as viewed in the direction of the rotational axis from the end wall (25, 250) of the collar means (19, 190).





4. A device according to claim 1, 2 or 3 wherein the confronting surface (22, 280) of the collar means (19, 190) and the first surface (9, 240) of the ring (13, 170) lie closely adjacent to form a continuation of one another.

5. A device according to claim 4 wherein between the confronting surface (22, 280) of the collar means and the first surface (9, 240) of the ring (13, 170) there are radially abutting guide surfaces (50, 350) of the collar means and of the ring which slide against one another as the screw-threaded elements (21, 210) are tightened or released.

6. A device according to any one of claims 1 to 5 wherein the screw-threaded elements (21, 210) have end surfaces for contacting the second surface (20, 200) of the ring (13, 170) which are inclined relative to the axis of rotation.

7. A device according to any one of claims 1 to 6 wherein there are two screw-threaded elements (21, 210).

8. A device according to claim 7 when appended to claim 3 wherein the threaded bores (6, 220) for receiving the screw-threaded elements (21, 210) are spaced apart by 120° relative to the location (L) at which the axes of their associated bores (6, 220) intersect.





9. A device according to any one of claims 1 to 8 wherein the second surface (20, 200) of the ring (13, 170) has depressions, recesses or blind bores for receiving the ends of the screw-threaded elements (21, 210).
10. A device according to any one of claims 3 to 9 when appended to claim 1, wherein the rotatable component is a solid shaft or spindle (10).
11. A device according to claim 10 wherein the ring (13) is itself another component of, or part of, some other structure (11) which is to be locked with the device.
12. A device according to claim 11 wherein the ring is formed by an extended region of the inner ring of a rolling element bearing (11).
13. A device according to claim 2 or any one of claims 3 to 9 when appended to claim 2, wherein the component is a hollow shaft or tube (120).
14. A device according to any one of the preceding claims, wherein the second surface (20, 200) of the ring is inclined at an angle of 1° to 5° to the axis of rotation (P).
15. An assembly composed of a device according to any one of the preceding claims in combination with the rotatable component (10, 120).



FIG. 1.

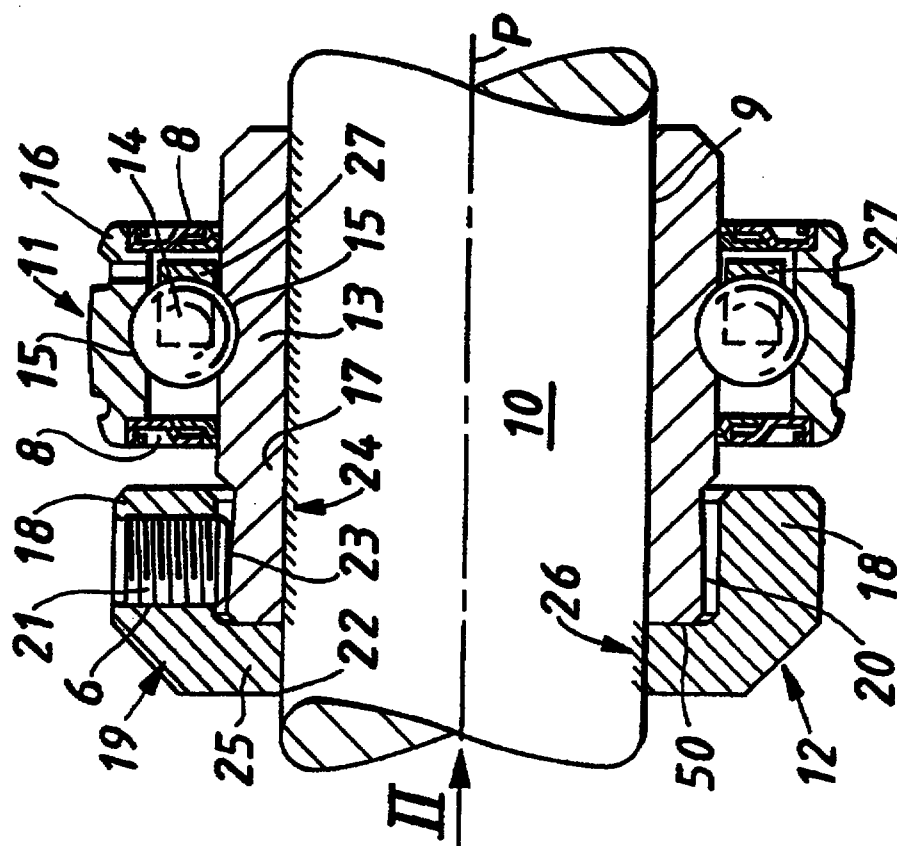
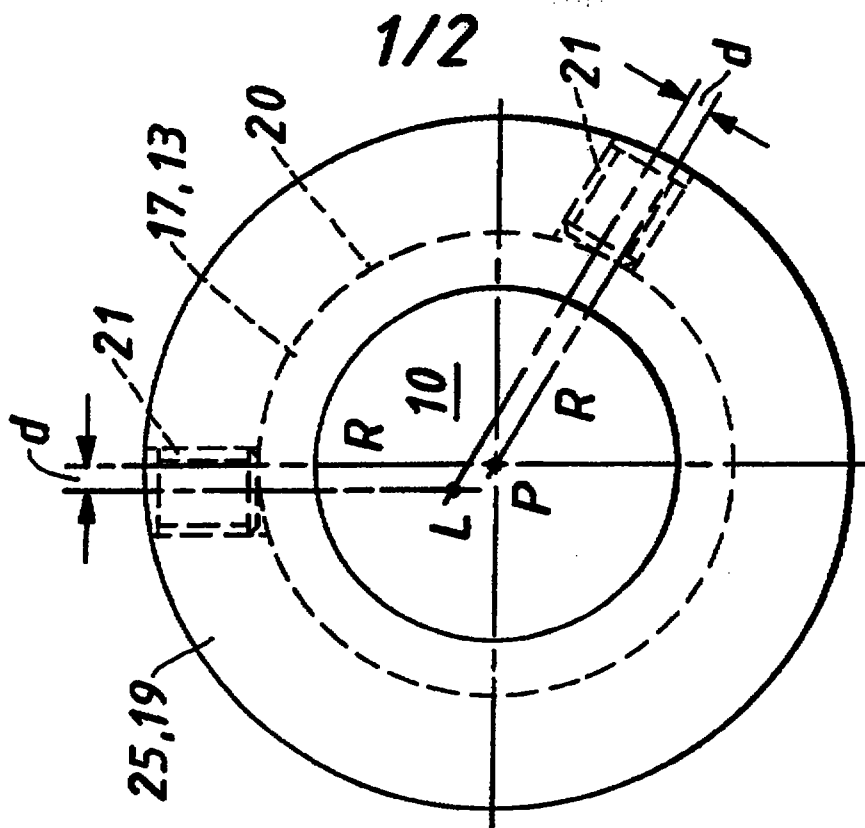


FIG. 2.



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FIG. 3.

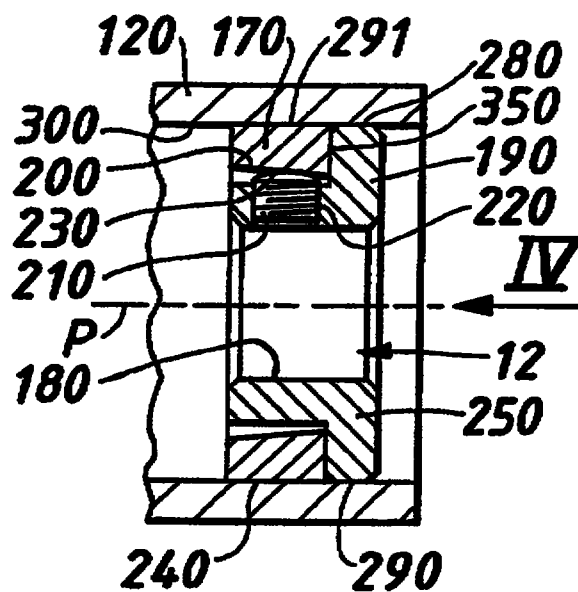
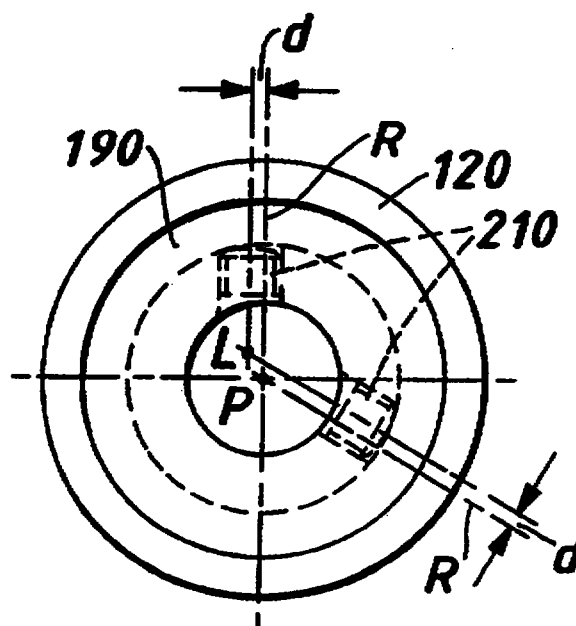


FIG. 4.



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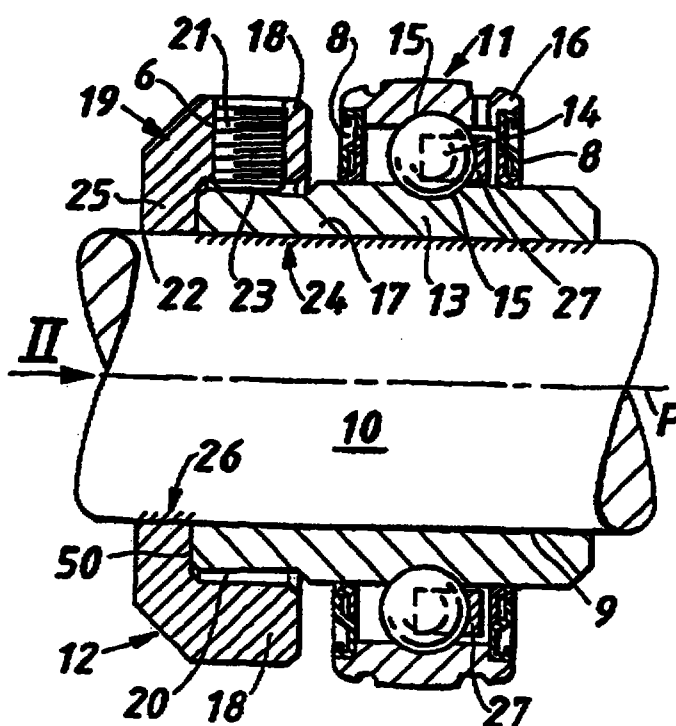
INTERNATIONAL APPLICATION PUBLISHED UNDER THE PATENT COOPERATION TREATY (PCT)

(51) International Patent Classification 6 : F16D 1/08, F16C 35/06	A1	(11) International Publication Number: WO 97/29293 (43) International Publication Date: 14 August 1997 (14.08.97)
(21) International Application Number: PCT/GB97/00336 (22) International Filing Date: 6 February 1997 (06.02.97) (30) Priority Data: 9602340.3 6 February 1996 (06.02.96) GB (71) Applicant (for all designated States except US): NSK RHP EUROPEAN TECHNOLOGY CO. LTD. [GB/GB]; Mere Way, Ruddington Fields Business Park, Ruddington, Nottingham NG11 6JZ (GB). (72) Inventor; and (75) Inventor/Applicant (for US only): SWINLEY, George [GB/GB]; 18 Acacia Drive, Townville, Castleford, West Yorks WF10 3PF (GB). (74) Agent: BARNARD, Eric, E.; Brookes & Martin, High Holborn House, 52/54 High Holborn, London WC1V 6SE (GB).		(81) Designated States: AL, AM, AT, AU, AZ, BA, BB, BG, BR, BY, CA, CH, CN, CU, CZ, DE, DK, EE, ES, FI, GB, GE, HU, IL, IS, JP, KE, KG, KP, KR, KZ, LC, LK, LR, LS, LT, LU, LV, MD, MG, MK, MN, MW, MX, NO, NZ, PL, PT, RO, RU, SD, SE, SG, SI, SK, TJ, TM, TR, TT, UA, UG, US, UZ, VN, ARIPO patent (KE, LS, MW, SD, SZ, UG), Eurasian patent (AM, AZ, BY, KG, KZ, MD, RU, TJ, TM), European patent (AT, BE, CH, DE, DK, ES, FI, FR, GB, GR, IE, IT, LU, MC, NL, PT, SE), OAPI patent (BF, BJ, CF, CG, CI, CM, GA, GN, ML, MR, NE, SN, TD, TG). Published With international search report.

(54) Title: **LOCKING DEVICE FOR USE WITH SHAFTS OR TUBES**

(57) Abstract

A locking device for clamping onto a shaft (10) is composed of a collar (19) and a ring (3, 17) fitted onto the shaft (10). The collar (19) has an end wall (25) and an axial region (18) extending over an outer inclined contact surface (20) of the ring (13, 17). The ring (13, 17) and the collar end wall (25) have inner surfaces (9, 22) confronting the outer surface of the shaft (10) which are separated by sliding radial surfaces (50) of the collar (19) and the ring (13, 17). Screw-threaded elements (21) set in threaded bores (6) in the axial region (18) of the collar (19) can be tightened against the outer surface (20) of the ring (13, 17) to cause contact zones (24, 26) of the inner surfaces (9, 22) to exert clamping force on the shaft (10) at diametrically opposed positions. The screw-threaded elements (21) and the bores (6) adopt non-radial dispositions relative to the axis of rotation (P).



"LOCKING DEVICE FOR USE WITH SHAFTS OR TUBES"

TECHNICAL FIELD

The present invention relates to a device which can be selectively locked or clamped to a rotatable component such as a shaft or tube.

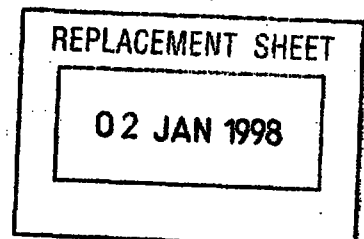
BACKGROUND TO THE INVENTION

Our earlier U.K. Patent Application No. 9419372.9 (2 282 209A) describes locking devices which can be used with a rotatable component such as a shaft or a tube. Although such devices achieve a reliable locking function, a collar means of the device tends to tilt when the device is locked in position and hence wobbles when the shaft or tubes revolves. A general object of the present invention is to provide an improved form of locking device which overcomes this defect.

DISCLOSURE OF THE INVENTION

According to one aspect of the invention there is provided a device for locking to a component rotatable about an axis; said device comprising collar means having an end wall with a bore defining a surface for confronting an outer surface of the rotatable component and a region extending axially from the end wall, a ring for disposing between the axial region of the collar means and the component, the ring having a first

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inner surface for confronting the outer surface of the component and a second outer surface which is inclined relative to the axis of rotation to converge with respect to the axis of rotation in a direction away from the end wall and a plurality, i.e., at least two, screw-threaded elements received in screw-threaded bores extending through the axial region of the collar means and spaced apart relative to the rotational axis other than at 180° , the threaded bores being disposed in positions non-radial relative to the rotational axis. The arrangement is such that the screw-threaded elements can be tightened to bear into or against the second outer surface of the ring to cause the bore surface of the collar means and the first surface of the ring to act as contact surfaces with respect to the outer surface of the component and exert clamping force on the component at diametrically opposed zones.

According to another aspect of the invention there is provided a device for locking to a hollow component rotatable about an axis; said device comprising collar means having an end wall with an outer surface for confronting an inner surface of the component and a region extending axially from the end wall; a ring for disposing between the axial region of the collar means and the component, the ring having a first outer surface for confronting the inner surface of the component and a second inner surface which is inclined relative to the axis of rotation to diverge with respect to the axis of rotation in a direction away from the end wall and a plurality, i.e. at least two, screw threaded elements received in screw-threaded bores extending through the axial region of the collar means and spaced apart relative to the rotational axis other than at 180° , the threaded bores being disposed in non-radial positions relative to the rotational axis. The

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arrangement is such that the screw-threaded elements can be tightened to bear into or against the second inner surface of the ring to cause the outer surface of the ring and the outer surface of the end wall of the collar means to act as contact surfaces with respect to the inner surface of the component and exert clamping force on the component at diametrically opposed zones.

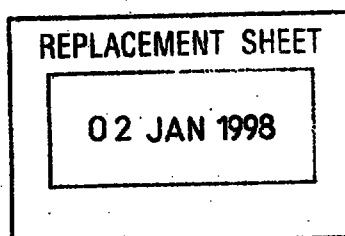
By arranging the screw-threaded elements in bores in non-radial positions on the collar means the tendency for the collar means to wobble as the rotatable component revolves can be prevented.

The threaded bores preferably have axes which intersect at a location which is offset radially from the rotational axis. Preferably, the location is on a radius from the rotational axis and this radius is displaced from a vertical plane in an anti-clockwise sense as viewed in the direction of the rotational axis and from the end wall of the collar means.

Preferably there are just two screw-threaded elements with their rotational axes spaced apart around the location where the axes of the associated threaded bores intersect by about 120° . Each screw-threaded element thus has its axis offset from a true radial direction relative to the rotational axis.

The ends of the screw-threaded elements may have plane surfaces normal to their longitudinal axes or surfaces which are set at an angle which generally matches the angle of the second inclined surface of the ring. It is possible to provide shallow depressions, recesses or blind bores in the second surface for receiving the ends of the screw-threaded elements.

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The surfaces of the collar means and the ring which confront the component normally lie parallel to the axis of rotation and directly adjacent and alongside one another to form a continuation of one another prior to the tightening of the screw-threaded elements. Between these contact surfaces there may be radial surfaces of the ring and the collar means which engage on one another and slide against one another as the screw elements are tightened or released.

The rotatable component may be a shaft or spindle with the locking device fitted onto its exterior periphery or a hollow shaft or tube with the locking device fitted inside.

It is not essential for the rotatable component to have a circular cross-section and the component can have a polygonal cross-section.

The invention may be understood more readily and various other aspects and features of the invention may become apparent, from consideration of the following description and accompanying drawings.

BRIEF DESCRIPTION OF DRAWINGS

Embodiments of the invention will now be described by way of examples only, with reference to the accompanying drawing wherein:

Figure 1 is a part sectional side view representation of an assembly employing a locking device constructed in accordance with the invention;

Figure 2 is an end view of the assembly shown in Figure 1 taken in the direction of arrow II of Figure 1;

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Figure 3 is a part sectional side view representation of another assembly employing a locking device constructed in accordance with the invention; and

Figure 4 is an end view of the assembly shown in Figure 3 taken in the direction of arrow IV in Figure 3.

DESCRIPTION OF BEST MODES OF CARRYING OUT THE INVENTION

Figures 1 and 3 generally correspond to Figure 1 and 2 of the aforementioned UK patent application 2282209A and use the same reference numerals. As shown in Figures 1 and 2, a shaft 10 carries a rolling-element bearing 11 and a locking device 12. The bearing 11 is composed of an extended inner ring 13 which is to be secured to the shaft 10 with the aid of the device 12 in a non-rotatable manner. The ring 13 has an inner bore surface 9 extending parallel to the outer peripheral surface of the shaft 10. A set of rolling elements, here in the form of balls 14, is disposed in raceways 15 between the inner ring 13 and an outer ring 16. The balls 14 are equi-distantly spaced apart round the raceways 15 with a cage 27 and seals 8 are mounted between the rings 13, 16.

An axial end region 17 of the inner ring 13 is surrounded by an axial region or portion 18 of a collar 19. The collar 19 has an end wall 25 with a bore forming a surface 22 fitted over the shaft 10 with the surface 22 forming a continuation of the inner surface 9 of the ring 13.

The axial region 17 of the bearing inner ring 13 is relieved to provide an outer surface 20 which is tapered to extend at an angle to the axis of the shaft 10. The surface

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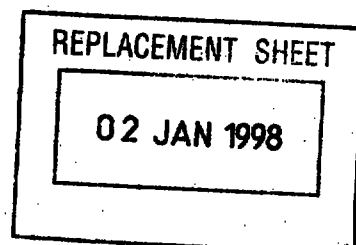
20 converges towards the shaft axis in the direction of the bearing 11. A typical angle for the surface 20 is 3° and generally an angle in the range 1° to 5° is preferred.

Two screws 21 are located in threaded bores 6 in the axial portion 18 of the collar 19.

As shown in Figure 2, the screws 21 and the bores 6 are offset by the same distance 'd' from radial planes R. The radial planes R are spaced apart by 120° and represent the preferred locations of the screws 21 in accordance with UK patent application No. 2282209A. In contrast to the latter, the longitudinal axes of the screws 21 now intersect at point L spaced from the axis of rotation P and offset therefrom in an anti clockwise sense relative to the vertical radial plane R as viewed in the direction of arrow II, i.e. in the direction of the axis P from the end wall 25 of the collar 19 and the convergence of the surface 20.

The screws 21 can be displaced inwardly of the bores 6 to engage on the outer surface 20 of the ring 13. In a modified arrangement the surface 20 has shallow recesses or depressions or blind bores for locating and receiving the ends 23 of the screws 21. The inner ring 13 of the bearing 11 and the collar 19 can be easily fitted onto the shaft 10 and the screws 21 are then tightened to some predetermined torque. The ends 23 of the screws 21, which can be tapered at the same angle as the surface 20 serve to contact the outer surface 20. When tightened, the ends 23 of the screws 21 press on the outer surface 20 to displace the axial end region 17 of the inner ring 13, and hence the entire inner ring 13, and the axial portion 18 of the collar 19 radially apart. This in turn causes diametrically opposed contact surface zones 24, 26 of the inner bore surface 9 of the ring

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13 and of the bore surface 22 in the end wall 25 of the collar 19 to clamp against the shaft 10 radially and uniformly without causing damage to the shaft 10. To release the locking device 12, the screws 21 are untightened to permit the collar 19 and the bearing 11 to be withdrawn from the shaft 10.

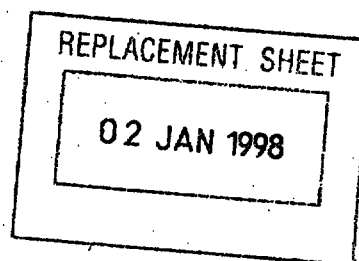
The end wall 25 of the collar 19 and the end of the axial region 17 of the ring 13 abut to form sliding guide surfaces 50 as the screws 21 are tightened or released.

As shown in Figures 3 and 4 another locking device 12 is fitted inside a hollow shaft or tube 120. The locking device is again composed of a collar 190 and a separate ring 170. The collar 190 has an axial region 180 with two angularly offset threaded bores 220 receiving screws 210. An end wall 250 of the collar 190 has an outer peripheral surface 280 which is parallel to and confronts an inner surface 300 of the tube 120. The ring 170 has an outer peripheral surface 240 which is parallel to and likewise confronts the inner surface 300 of the tube 120 and an inner surface 200 which is tapered at an angle to the axis of the tube 120. The surface 200 diverges from the end wall 250 relative to the axis of rotation P. As with the first embodiment, the surface 200 can be inclined at an angle of 1° to 5° and preferably 3° to the axis of the tube 120.

The surface 200 can contain shallow recesses, depressions or blind bores for receiving the ends 230 of the screws 210.

As shown in Figure 4, the screws 210 and the threaded bores 220 are offset by the same distance 'd' from radial planes R. The radial planes R are spaced apart by 120° and represent the preferred locations of the screws 210 in accordance with UK patent application 2282209A. In contrast to the latter, the longitudinal axes of the screws 210

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now intersect at a point L spaced from the axis of rotation P and offset therefrom in an anti clockwise sense relative to the vertical radial plane R as viewed in the direction of arrow IV, i.e. in the direction of the axis P and the divergence of the surface 200 from the end wall 250 of the collar 190.

The device 12 is locked to the tube 120 by tightening the screws 210 to cause their ends 230 to displace the axial region 180 of the collar 190 and the ring 170 radially apart. This in turn causes diametrically opposed surface contact zones 290, 291 of the surface 280 of the collar end wall 250 and of the surface 240 of the ring 170 to bear against the inner surface 300 of the tube 120 to lock the device in place.

The ring 170 and the end wall of the collar 190 have radial faces which abut and establish slidable guide surfaces 350 as the screws 210 are tightened or released. The device can be used to provide an abutment or location for some other component such as a bearing.

In contrast to some known forms of locking device or mechanisms, a device in accordance with the invention does not damage the rotatable component and is easily fitted and removed repeatedly. The device also rotates uniformly with the rotatable component without any wobble.

In the Figure 1 arrangement the inner ring 13 is simply extended and easily machined in one operation. There is no need to provide tapped bores in the ring 13. Also since no excessive tensile stresses are created on the inner ring 13 there is no danger of fracture for this reason. There is also no distortion so the raceways for the balls are unaffected.

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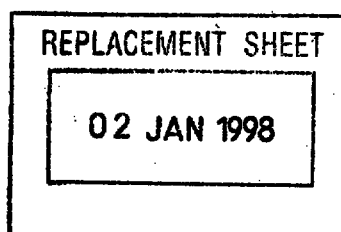
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Claims

1. A device (12) for locking to a component (10) rotatable about an axis (P); said device comprising collar means (19) having an end wall (25) with a bore defining a surface (22) for confronting an outer surface of the rotatable component (10) and an axial region (18) extending axially from the end wall, a ring (13) for disposing between the axial region (18) of the collar means and the component (10), the ring (13) having a first inner surface (9) for confronting the outer surface of the component (10) and a second outer surface (20) which is inclined relative to the axis of rotation (P) to converge with respect to the axis of rotation in a direction away from the end wall (25) and at least two screw-threaded elements (21) received in screw-threaded bores (6) extending through the axial region (18) of the collar means (19) and spaced apart relative to the rotational axis (P) other than at 180°, the arrangement being such that the screw-threaded elements (21) can be tightened to bear into or against the second outer surface (20) of the ring (13) to cause the bore surface (22) of the collar means (19) and the first surface (9) of the ring (13) to act as contact surfaces with respect to the outer surface of the component (10) and exert clamping force on the component at diametrically opposed zones (24, 26); characterized in that the threaded bores (6) are disposed in positions non-radial relative to the rotational axis.

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2. A device (12) for locking to a hollow component (120) rotatable about an axis (P); said device comprising collar means (190) having an end wall (250) with an outer surface (280) for confronting an inner surface (300) of the component (120) and an axial region (180) extending axially from the end wall (250); a ring (170) for disposing between the axial region (180) of the collar means (190) and the component (120), the ring (170) having a first outer surface (240) for confronting the inner surface (300) of the component and a second inner surface (200) which is inclined relative to the axis of rotation to diverge with respect to the axis of rotation in a direction away from the end wall (250) and at least screw threaded elements (210) received in screw-threaded bores (220) extending through the axial region (180) of the collar means (190) and spaced apart relative to the rotational axis other than at 180° the arrangement being such that the screw-threaded elements (210) can be tightened to bear into or against the second inner surface (200) of the ring (170) to cause the outer surface (240) of the ring and the outer surface (280) of the end wall (250) of the collar means (190) to act as contact surfaces with respect to the inner surface (300) of the component (120) and exert clamping force on the component at diametrically opposed zones; characterized in that the threaded bores (220) are disposed in non-radial positions relative to the rotational axis (P).

3. A device according to claim 1 or 2 wherein the threaded bores (6, 220) have axes which intersect at a location (L) which is offset from the rotational axis (P) and disposed on a radius from the rotational axis, said radius being displaced in an anti-clockwise

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sense relative to a vertical plane as viewed in the direction of the rotational axis from the end wall (25, 250) of the collar means (19, 190).

4. A device according to claim 1, 2 or 3 wherein the confronting surface (22, 280) of the collar means (19, 190) and the first surface (9, 240) of the ring (13, 170) lie closely adjacent to form a continuation of one another.

5. A device according to claim 4 wherein between the confronting surface (22, 280) of the collar means and the first surface (9, 240) of the ring (13, 170) there are radially abutting guide surfaces (50, 350) of the collar means and of the ring which slide against one another as the screw-threaded elements (21, 210) are tightened or released.

6. A device according to any one of claims 1 to 5 wherein the screw-threaded elements (21, 210) have end surfaces for contacting the second surface (20, 200) of the ring (13, 170) which are inclined relative to the axis of rotation.

7. A device according to any one of claims 1 to 6 wherein there are two screw-threaded elements (21, 210).

8. A device according to claim 7 when appended to claim 3 wherein the threaded bores (6, 220) for receiving the screw-threaded elements (21, 210) are spaced apart by

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120° relative to the location (L) at which the axes of their associated bores (6, 220) intersect.

9. A device according to any one of claims 1 to 8 wherein the second surface (20, 200) of the ring (13, 170) has depressions, recesses or blind bores for receiving the ends of the screw-threaded elements (21, 210).

10. A device according to any one of claims 3 to 9 when appended to claim 1, wherein the rotatable component is a solid shaft or spindle (10).

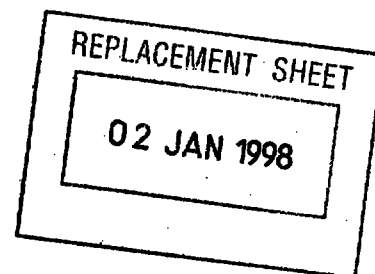
11. A device according to claim 10 wherein the ring (13) is itself another component of, or part of, some other structure (11) which is to be locked with the device.

12. A device according to claim 11 wherein the ring is formed by an extended region of the inner ring of a rolling element bearing (11).

13. A device according to claim 2 or any one of claims 3 to 9 when appended to claim 2, wherein the component is a hollow shaft or tube (120).

14. A device according to any one of the preceding claims, wherein the second surface (20, 200) of the ring is inclined at an angle of 1° to 5° to the axis of rotation (P).

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15. An assembly composed of a device according to any one of the preceding claims in combination with the rotatable component (10, 120).

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