

TOOLING FOR FORMING END PANELS OF CANS

This invention relates, in general, to forming end panels for two-piece containers from a blank of metal and relates, in particular, to forming such end panels in double acting presses employing a cushioned reverse draw concept to set the chuck wall.

End panels for two-piece or, for that matter, for three-piece metal containers are well known in the art. It is of critical importance in forming these panels to control metal thickness. With regard to all types of end panels, which have various radiused areas, it is important to avoid thinning of the metal in forming these areas to avoid weakening of the piece. In conventional processes, it is thought to be necessary to compensate for this natural thinning phenomena by using metal with sufficient base weight so that the finished product will still have sufficient strength. Therefore, tooling which enables the base weight of the metal to be reduced without diminishing the effectiveness of the final product is desirable. Also, with regard to pull tab cans such as are used primarily with beverages, the tear line must be held to very precise tolerances and, therefore, the metal thickness of the panel is of even greater importance and control of this dimension during the forming operation is very important.

At high speeds, the shut height of presses normally employed for forming end panels of this type will change and this height can also affect metal thickness. It is, therefore, important to be able to control that shut
5 height, which assists in controlling metal thickness.

In any event, the resulting product which is the subject of the apparatus and method of this application, is not, per se, new. However, it is believed that the tooling for manufacturing it with reduced base weight
10 and without reduced strength is.

It has been found that the above noted objects can be achieved by providing certain unique tooling capable of operation in a double acting press so that the shut height of the press can be effectively controlled and
15 thinning of the metal in the chuck wall or radiused areas can be effectively controlled. It is of critical importance, as noted above, to control the metal thickness and it is believed that utilization of the tooling to be described, in a double acting press, will enable that
20 object to be achieved.

The invention accordingly provides apparatus for forming an end panel from a blank of metal for use in a double acting press having inner and outer slides and a fixed lower platen, comprising: fluid actuated pressure
25 means to be carried by the outer slide of the press and being movable into holding engagement with the metal;

blanking means to be carried by the outer slide of the press and the fixed lower platen thereof; forming means to be carried by the inner slide of the press for forming the panel; die means to be carried by the fixed lower
5 platen for initially forming the chuck wall of the panel in cooperation with the said blanking means; and cushioning means to be carried by the fixed lower platen and resiliently supporting the panel in the chuck wall area during final formation of the chuck wall. The
10 cushioning means permits an effective reverse draw to be achieved so that the critical groove area adjacent the chuck wall can be formed without unnecessarily thinning the metal. Effectively, a slight "bounce" right at the end of the downward stroke of the punch is achieved
15 and this, combined with wiping of the metal about the radius of the die core ring just referred to, permits the critical radius to be formed without any unnecessary thinning of the metal in the critical area.

A preferred form of tooling according to the
20 invention is illustrated in the accompanying drawings.

OF THE DRAWINGS:

FIGURE 1 is an elevational view, partially in section, illustrating the relative positions of the apparatus prior
25 to operation.

FIGURE 2 is a view similar to FIGURE 1 showing the relative positions of the components of the apparatus following blanking.

FIGURE 3 is a view similar to FIGURE 1 showing the relative positions of the elements of the apparatus following wiping.

FIGURE 4 is a view similar to FIGURE 1 showing the relative positions of the components of the apparatus after initial forming of the end panel.

10 FIGURE 5 is a view similar to FIGURE 1 showing the relative positions of the elements of the apparatus following final forming of the end panel.

FIGURE 6 is a view similar to FIGURE 1 showing the relative positions of the elements of the apparatus following retraction of the upper platen and prior to removal of the formed end panel from the press.

FIGURE 7 is an enlarged sectional view showing the critical radius areas of the end panel.

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DESCRIPTION OF THE PREFERRED EMBODIMENT

Referring first to FIGURE 1 of the drawings, it will be understood, as noted above, that the tooling which is the subject of this particular application would be used in a double acting press. No great detail has been illustrated and described herein in connection with the working of a double action press except to point out that

the press includes inner and outer slides which operate independently performing dual actions on each opening and closing or cycle of the press, per. se. The general scheme of operation of a press of this nature is well known to those having ordinary skill in this art and can be seen in Ridgway U.S. Patent 3,902,347, which also discloses the capability of adjustably controlling the timing and movement of the slides by an adjustable shut height.

10 Referring still then to FIGURE 1 for a description of the relevant components of the apparatus, it will be noted that the upper platen of the press carries an inner slide 10 and an outer slide 20 which are reciprocal with respect to lower platen 80. The outer slide 20 has a
15 through central opening 20a for receipt of the tooling carried by the inner slide 10 so that that tooling may reciprocate freely within the opening 20a, as will be described.

Still referring to FIGURE 1 of the drawings and
20 referring primarily to the inner slide 10 and the tooling carried thereby, it will be noted that a punch center post 30 is mounted on the bottom edge of the inner slide 10 and secured thereto by a plurality of screws 30a. The punch center post 30 is elongate with its proximal end
25 secured to slide 10 as just noted and its distal end carrying a punch core 31. The punch core 31 is adjustably secured to the distal end of the punch center post 30 by means of the threaded sleeve assembly 32.

It should also be noted that the punch center post 30 has a through central bore 30b which is in fluid communication with a secondary bore 33 which is, in turn, in fluid communication with an air blow off passage 34 in the punch core 31. All of these passages are also in fluid communication with a bore 10a in the inner slide 10 which leads to a source of air under pressure (not shown). This arrangement is for purposes of assisting removal of the finished end panel and the particular design and operation of the passage 34 in the punch core 31 can be seen in Bulso U.S. Patent 4,343,173. Suffice it to say here that air may be introduced to dislodge the finally formed panel should it stick to the punch core 31.

15 The apparatus just disclosed, namely the punch center post 30 and the punch core 31, are, as noted, secured to the inner slide 10 and move in conjunction therewith as that slide moves toward and away from the bottom platen 80 of the press.

20 Still referring to FIGURE 1 and referring particularly to the outer slide 20, it will be noted that, as already mentioned, the outer slide 20 does have a central opening 20a. Surrounding this opening 20a is an upper cylinder 40 within which a sleeve 50 is disposed. The sleeve 50 is secured to a cap 51 by screws 50a with the cap 51 being, in turn, secured to the outer slide 20 itself by screws 51a.

Also received within the upper cylinder 40, which is formed by the sleeve 50 and the walls of the opening 20a of the outer slide 20, are first and second stacked pistons 41 and 42, respectively.

5 Furthermore, a passage 21 is provided in the outer slide 20 for communication with the upper cylinder 40 for the introduction of fluid under pressure into the cylinder so that it may act upon the pistons 41 and 42, as will be described. Radially directed passages 22, 22
10 are also provided for relief of this fluid. The source of the fluid pressure for passage 21 is not illustrated.

Continuing with reference to FIGURE 1 of the drawings, it will be noted that a pressure sleeve 60 is also carried by the outer slide 20. This pressure sleeve
15 is movable relatively of the slide 20 and is acted upon by the stacked pistons 41, 42 under fluid pressure through the passage 21 into the upper cylinder 40.

Outer slide 20 also includes a punch shell assembly 70 which is mounted on the slide by the retainer 71 which
20 is, in turn, held in place by screws 71a. This punch shell assembly 70 is disposed in surrounding or concentric relationship with the pressure sleeve 60 as is apparent from FIGURE 1 of the drawings.

Still referring to FIGURE 1 of the drawings and
25 paying particular attention to the bottom platen 80 of the press, it will be noted that this platen includes a central recess 81. Fluid passages and air passages 82

and 83 are also provided in the platen 80 for purposes which will be described.

Also mounted on the bottom platen 80 is a cut edge 90 which is mounted in overlying relationship with the
5 central recess 81 and held onto the platen by screws 90a.

Also carried on the bottom platen 80 and within the central recess 81 is a die core 100. This die core 100 is fixed to and mounted on the bottom platen 80 by means of a threaded sleeve 101 and has a through air blow off
10 passage 102 similar to passage 34 of punch core 31 for purposes which will be described below.

Also received on the bottom platen 80 within the recess 81 is a die core ring 110 which rests on a die core ring riser 111 which is, in turn, secured to the bottom
15 platen by screws 111a. The die core ring riser 111 has a fluid passage 111b and the die core ring 110 also has a fluid passage 110a for purposes which will be described.

A pressure sleeve 120 is also received on the bottom platen beneath the cut edge 90 and within the recess 81.
20 Effectively, the pressure sleeve 120 is received within a cylinder formed by the walls of the recess 81 and the die core ring and die core riser 110 and 111 respectively.

Finally, a knock out piston 130 is received interiorly of the die core ring 110 within a cylinder
25 effectively formed by the inner walls of the die core ring 110 and the outer wall of the die core 100.

In use or operation of the improved tooling, it will first be assumed that the components are assembled to the position shown in FIGURE 1, which is the "open" position of the inner and outer slides 10 and 20. The material from which the end panel is to be formed, generally designated by the letter M, is then placed in the press in the position shown in FIGURE 1 of the drawings. The means for feeding or loading the press are not illustrated in detail since such devices are known to those of ordinary skill in the art. An example can be seen in Bulso U.S. Patent 3,980,297.

Closing of the press moves both the inner and outer slides 10 and 20 down toward the bottom platen 80 or from the position of FIGURE 1 to that of FIGURE 2. Contact is first made with the material M by the punch shell assembly 70 to initiate the blanking operation following which the pressure sleeve system is actuated. To activate the pressure sleeve 60, fluid pressure is introduced through passage 21 into upper cylinder 40. This pressure forces the stacked pistons 41 and 42 down and they, in turn, engage sleeve 60 to move it into holding engagement with the material M. This serves to hold the material M during a portion of the blanking operation and during subsequent operations with the operative position of the pressure sleeve 60 being seen in FIGURE 2 of the drawings.

Still referring to FIGURE 2, movement of the press brings the punch shell assembly 70 into contact with the material M and as the distal end of the punch shell assembly 70 passes the die line, it will, in

cooperation with cut edge 90, sever or blank the material M into two pieces, one of which may now be described as the blank B and the other may be described as the scrap S. At the same time, preliminary forming of blank B
5 around the radius 110b of the top of die core ring 110 occurs.

Further downward movement of the inner slide 10 will cause the punch shell assembly 70 to wipe the peripheral edges of the blank B about radius 110b on the projecting
10 end of the die core ring 110, as shown in FIGURE 3 of the drawings. It will be noted at this point that the pressure sleeve 60 does not descend any further since the resistance of the die core ring 110 is such that the pistons 41 and 42 tend to back up, overcoming the fluid
15 pressure from the passage 21. Thus, die core ring 110 is supported on die core riser 111 and at this stage that fixed support will prevent further downward movement of pressure sleeve 60.

As the inner slide 10 continues its downward move-
20 ment punch core 31 passes the top of die core ring 110 or, in other words, passes below the tin line. The chuck wall C is thus formed on the blank B by virtue of the fact that punch shell assembly 70 wipes the outboard edge of the blank B over the end of the fixed die core ring 110 as
25 noted and as can be seen in FIGURE 4.

Upon further downward movement (see FIGURE 5) of the inner slide 10 at this time, it will be noted that pres-

sure sleeve 60 still contacts the metal opposite the top of die core ring 110. As punch core 31 descends, the metal is drawn further down over the radius 110b. At the same time the die core 100, which is fixed, supports the central portion of blank B. The pressure of the periphery of punch core 31 will overcome the pressure acting on the bottom of the knock out piston 130 through bore 110a and 111c, thereby forcing it downward within its cylinder. This provides a cushioning effect and groove G is set against this cushion.

At this point, the blank B has received the configuration of FIGURE 5 of the drawings, which is essentially its final configuration at this stage in the overall manufacturing process.

With reference to FIGURE 7 of the drawings, it will be seen how an end panel of this type can thus be formed without undue thinning of the metal.

Thus, the critical radius areas R-1, R-2, R-3, R-4 and R-5 are maintained at their desired thickness and the base weight can be reduced. For example, common practice has been to allow about 0.002 inches for thinning in these areas while the present method has been found satisfactory with an allowance of between 0.0004 and 0.0005 inches. When the large numbers of pieces normally produced by tooling of this type are considered, this represents a significant material savings.

After the panel is thus formed and the press bottoms out, both the inner and outer slides 10 and 20 are retracted from the position of FIGURE 5 to the position of FIGURE 6. Once contact with the finally formed can end panel E is released by the punch core 31, the knock out piston 130 moves back upward toward the die line under fluid pressure transmitted from the passage 82 of the bottom platen through the passage 111c of the die core ring riser 111 and through the passage 110a of the die core ring 110. This raises the can end panel E to the die line from which position it may be removed from the press and transferred to the next station or to the next press for further operations thereon such as, for example, scoring the tear line and attaching the rivet, etc.

During removal, if the panel E should stick to punch core 31, actuation of air through passage 34 will dislodge it. Similarly, if the panel should resist removal from die core 100, actuation of air through passages 102 and 111b will dislodge it.

This application is divided out of application 8412230 (2141052) which has similar description, but different claims.

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CLAIMS:

1. Apparatus for forming an end panel from a blank of metal for use in a double acting press having inner and outer slides and a fixed lower platen, comprising: fluid actuated pressure means to be carried
5 by the outer slide of the press and being movable into holding engagement with the metal; blanking means to be carried by the outer slide of the press and the fixed lower platen thereof; forming means to be carried by the inner slide of the press for forming the panel; die means
10 to be carried by the fixed lower platen for initially forming the chuck wall of the panel in cooperation with the said blanking means; and cushioning means to be carried by the fixed lower platen and resiliently supporting the panel in the chuck wall area during final formation of the
15 chuck wall.
2. The apparatus of Claim 1 wherein said cushioning means include a piston received in the fixed platen in opposed relationship with said forming means; and the piston being fluid actuated.
- 20 3. The apparatus of Claim 1 or Claim 2 wherein said forming means include a punch core carried by the inner slide; and said die means comprising a die core ring carried by said fixed platen in opposed relationship thereto.
- 25 4. The apparatus of Claim 1 wherein said die means include a die core fixed to said fixed lower platen; and a die core ring carried by said fixed lower platen in



encircling relationship with said die core; said cushioning means being disposed between said die core and said die core ring.

5 5. The apparatus of any of Claims 1 to 4 wherein said forming means and said die means include panel removal means.

6. Apparatus for forming an end panel from a blank of metal for use in a double acting press having inner and outer slides and a fixed lower platen, comprising:
10 fluid actuated pressure means to be carried by the outer slide of the press and being movable into holding engagement with the metal; a blanking member to be carried by the outer slide and a cut edge on the fixed lower platen thereof; forming means to be carried by the inner slide of the press
15 for forming the panel; die means including a die core ring to be carried by the fixed lower platen for initially forming the chuck wall of the panel in cooperation with the said blanking means; and cushioning means to be carried by the fixed lower platen and resiliently supporting the
20 panel in the chuck wall area during final formation of the chuck wall, the cushioning means including a movable pressure sleeve disposed in encircling relationship with said die core ring and in opposed relation to said blanking member for cooperation with said blanking member
25 during a portion of its travel to hold the metal, and said blanking member being movable into the space between said cut edge and said die core ring to engage a peripheral edge portion of the metal during initial forming

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of the chuck wall.

7. Apparatus for forming an end panel from a blank of metal substantially as hereinbefore described with reference to the accompanying drawings.

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