

HYBRID DYNAMIC PRESSURE GAS RADIAL BEARING

TECHNICAL FIELD

The invention relates to a dynamic pressure gas radial bearing, in particular to a hybrid dynamic pressure gas radial bearing, which relates to a rigid characteristic of a high-limit rotating speed of a radial bearing of a groove-type dynamic pressure gas and the flexible characteristic of the high impact resistance and the load capacity of the foil type dynamic pressure gas radial bearing, belongs to the technical field of gas bearings.

BACKGROUND TECHNOLOGY

The gas bearing has the advantages of being high in speed, high in precision, resistant to high temperature and small in friction loss, long in service life and the like. After the rapid development of recent decades, the gas bearing has been widely applied to the fields of high-speed supporting, high-precision supporting and the like. At present, various types of gas bearings have been developed, and the gas bearings are mainly divided into a dynamic pressure type and a static pressure type.

The dynamic pressure gas bearing takes gas as a lubricant, and a gas film is formed between the shaft and the bearing. It is a bearing form that the moving face is not in direct contact with the static face, and it has many advantages such as no pollution, low friction loss, wide temperature range, smooth operation, long use time and high working speed. Due to the small friction loss, and not necessary to use liquid lubricating oil, it is widely used in the field of high-speed rotary applications. In particular, it is often used in ultra-high-speed application field which is usually supported by a rolling bearing and where liquid lubricants are not suitable.

The dynamic pressure gas bearing in accordance with the direction of bearing load is divided into a dynamic pressure gas radial bearing, a dynamic pressure gas thrust bearing and

a dynamic pressure radial thrust combined bearing. The dynamic pressure gas radial bearing is formed by the relatively moving two working faces forming a wedge-shaped space. When they are relatively moved, the gas is driven by its own viscous action and compressed into the wedge-shaped gap, thereby generating dynamic pressure to support the load. Different structural forms of gas dynamic pressure radial bearings due to structural differences, the work process is slightly different. At present, the more common types of dynamic pressure radial bearing structure are a tilting pad type, a groove type and a foil type.

The tilting pad type dynamic pressure gas radial bearing is a kind of excellent dynamic pressure gas bearing with self-adjusting performance, which can work safely in the space of smaller air gap, and is not sensitive to thermal deformation, elastic deformation and so on, and the machining precision is easy to be guaranteed, and it has the outstanding advantages of being capable of automatically tracking the change of the load. At present, it is mainly used in large-scale high-speed rotating machinery and turbine machinery at home and abroad. But its bearing structure is more complex, the installation process is complex, is higher than the general radial bearing requirements, thus its application is limited.

Although the foil type dynamic pressure radial bearing with elastic support, can make the bearing to obtain a certain bearing capacity and alleviate the impact vibration capacity, but the foil bearing is usually made of metal foil sheets, there are still some problems in the material manufacturing technology and processing technology, the damping value of the bearing can not be greatly improved, so that the rigidity of the bearing is not enough, the critical rotating speed of the bearing is low, and the bearing is easy to instability and even stuck during high-speed operation.

The groove type dynamic pressure gas radial bearing has good stability, and the certain stability is achieved even under no load; moreover, at a high speed, the static bearing capacity is larger than that of other forms of bearings. At present, it is mostly used for small-sized high-speed rotating machines, and used as a bearing in precision machinery, such as a gyroscope and a magnetic drum. However, due to the fact that the groove type dynamic pressure gas radial bearing is high in rigidity, so that the impact resistance is not good enough, the

load capacity is not large enough, and high-speed operation under a large load cannot be realized.

How to realize the hybrid dynamic pressure gas radial bearing which has the rigid characteristics of high-limit rotating speed of the groove-type dynamic pressure gas radial bearing, and has the flexible characteristic of the high impact resistance and the load capacity of the foil type dynamic pressure gas radial bearing, is not only a target which is expected to be realized by researchers in the field, but also to achieve the application of dynamic pressure radial bearings in the larger load under the ultra-high-speed field has important value and far-reaching significance.

10 CONTENT OF THE INVENTION

In view of the problems and requirements of the prior art discussed above, it is an object of the present invention to provide hybrid dynamic pressure gas radial bearing which has the rigid characteristics of high-limit rotating speed of the groove-type dynamic pressure gas radial bearing, and has the flexible characteristic of the high impact resistance and the load capacity of the foil type dynamic pressure gas radial bearing, and to achieve the application of dynamic pressure gas radial bearings in the ultra-high-speed areas under larger load.

To achieve the above object, the subject matter of the present invention is as follow:

A hybrid dynamic pressure gas radial bearing, comprises a bearing outer sleeve, an inner sleeve and a foil type elastic part arranged between the bearing outer sleeve and the inner sleeve, the outer circumferential face and the left and right end faces of the bearing inner sleeve are respectively provided with regular groove-type patterns, and the groove-type pattern in one end face is in mirror symmetry with the groove-type pattern in the other end face, and the axial contour lines of the groove-type patterns in the outer circumference face and the radial contour lines of the groove-type patterns in the left and right end faces are in one-to-one correspondence and are connected with each other.

An implementation scheme, the axial high-position lines of the groove-type patterns in

the outer circumferential face of the bearing inner sleeve corresponds to the radial high-position lines of the groove-type patterns in the left and right end faces, and are connected with each other in front of the circumferential chamfer in the end face; and the axial middle-position lines of the groove-type patterns in the outer circumferential face corresponds to the radial middle-position lines of the groove-type patterns in the left and right end faces, and are connected with each other in front of the circumferential chamfer in the end face; and the axial low-position lines of the groove-type patterns in the outer circumferential face corresponds to the radial low-position lines of the groove-type patterns in the left and right end faces, and are connected with each other in front of the circumferential chamfer in the end face.

An implementation scheme, the groove-type pattern is of an impeller shape.

As a preferable scheme, a wear-resistant coating is arranged on the matching face of the foil type elastic part matched with the outer circumferential face of the bearing inner sleeve.

As a preferable scheme, a fit clearance between the foil type elastic part and the bearing inner sleeve is 0.003-0.008 mm.

As a preferable scheme, both ends of the foil type elastic part are fixed on the inner circumferential wall of the bearing outer sleeve.

As a preferable scheme, the number of the foil type elastic parts is multiple, and the foil type elastic parts are uniformly distributed along the inner circumferential wall of the bearing outer sleeve.

As a more preferable scheme, clamping grooves used for fixing the foil type elastic part are formed in the inner circumferential wall of the bearing outer sleeve.

As a preferable scheme, the foil type elastic part is subjected to surface heat treatment.

An implementation scheme, the foil type elastic part is composed of a wave foil and a flat foil, the top end of the arc-shaped protrusion of the wave foil is attached to the flat foil, the

transition bottom edge of the wave arch of the wave foil is attached to the inner circumferential wall of the bearing outer sleeve.

Another implementation scheme, the foil type elastic part is composed of a wave foil and a flat foil, the top end of the arc-shaped protrusion of the wave foil is attached to the inner circumferential wall of the bearing outer sleeve, and the transition bottom edge between the wave arches of the wave foil is attached to the flat foil.

As yet another embodiment, the foil type elastic part is composed of two flat foils, wherein a through hole is formed in the flat foil close to the inner circumferential wall of the bearing outer sleeve.

As a preferable scheme, stop rings are arranged at the two ends of a bearing outer sleeve.

As a preferable scheme, through holes and concave holes which are coaxial are formed in the outer circumference of the bearing outer sleeve, and the through holes are located in the concave holes in the same axis.

Compared to the prior art, the present invention offers the following benefits:

The foil type elastic part is arranged between the bearing outer sleeve and the bearing inner sleeve, and the outer circumferential face and the left and right end faces of the bearing inner sleeve are provided with regular groove-type patterns, and the axial contour lines of the groove-type patterns in the outer circumference face and the radial contour lines of the groove-type patterns in the left and right ends are in one-to-one correspondence and are connected with each other, and the groove-type pattern in one end face is in mirror symmetry with the groove-type pattern in the other end face, so that the hybrid dynamic pressure gas radial bearing is obtained, which has the rigidity characteristic of the high-limit rotating speed of the radial bearing of the groove-type dynamic pressure gas, and has the flexible characteristic of the high impact resistance and the load capacity of the foil type dynamic pressure gas radial bearing; compared with an existing simple groove-type dynamic pressure gas ra-

dial bearing, it has the anti-impact capacity and the load capacity which are increased in a multiplied mode at the same rotating speed; and compared with an existing pure foil type dynamic pressure gas radial bearing, it has a limit rotating speed which is increased by times under the same load; through the test, the hybrid dynamic pressure gas radial bearing provided by the invention can be realized a limit rotating speed of 160, 000 rpm to 480,000 rpm under a load of 3-5 kg, but the existing dynamic pressure gas radial bearing can only achieve a load capacity of 1-3 kg, the maximum limit rotating speed can only reach 100,000 rpm to 180,000 rpm. It can be seen that the present invention can realize the application of dynamic pressure gas radial bearings in ultra high-speed areas under larger loads. Compared with the prior art, the present invention has made significant progress, and made the study of the dynamic bearing gas radial bearing technology take a new step.

BRIEF DESCRIPTION OF THE DRAWINGS

FIG. 1 is a schematic diagram of a left-view three-dimensional structure of partial segmentation of a hybrid dynamic pressure gas radial bearing according to the first embodiment of the present invention.

FIG. 2 is an enlarged view of the part A in FIG. 1.

FIG. 3 is a schematic diagram of a right-view three-dimensional structure of partial segmentation of a hybrid dynamic pressure gas radial bearing according to the first embodiment of the present invention.

FIG. 4 is an enlarged view of the part B in FIG. 3.

FIG. 5 is a schematic cross-sectional view of a hybrid dynamic pressure gas radial bearing according to embodiment 1 of the present invention.

FIG. 6 is an enlarged view of the part C in FIG. 5.

FIG. 7 is an enlarged view of the part D in FIG. 6.

FIG. 8 is a schematic cross-sectional view of a hybrid dynamic pressure gas radial bearing according to embodiment 2 of the present invention.

FIG. 9 is a schematic diagram of the wave foil in FIG. 8.

FIG. 10 is a schematic cross-sectional view of a hybrid dynamic pressure gas radial bearing according to embodiment 3 of the present invention.

FIG. 11 is a schematic diagram of a flat foil with through holes in FIG. 10.

FIG. 12 is a schematic cross-sectional view of a hybrid dynamic pressure gas radial bearing according to embodiment 4 of the present invention.

Reference numerals in the figures denote as follows:

1, bearing outer sleeve; 11, stop ring; 12, clamping groove; 13, through hole; 14, concave hole; 2, bearing inner sleeve; 21, groove-type pattern in the outer circumferential face; 211, axial high-position line; 212, axial middle-position line; 213, axial low-position line; 22, groove-type patterns in the left end face; 221, radial high-position line; 222, radial middle-position line; 223, radial low-position line; 23, groove-type pattern in the right end face; 231, radial high-position line; 232, radial middle-position line; 233, radial low-position line; 3, foil type elastic part; 31, wave foil; 311, arc-shaped protrusions; 312, transition bottom edges between the wave arches; 32, flat foils; 33, the flat foil with through holes; 331, the through hole in the flat foil; 34, wear-resistant coating.

DETAILED DESCRIPTION

The subject matter of the present invention will be described in further detail below with reference to several embodiments and the accompanying drawings.

EXAMPLE 1

As shown in FIG. 1 and FIG. 3, the hybrid dynamic pressure gas radial bearing provided by the embodiment of the invention, includes a bearing outer sleeve 1 and a bearing inner sleeve 2, wherein the outer circumferential face of the bearing inner sleeve 2 and the left end face and the right end face are respectively provided with regular groove-type patterns (21, 22 and 23 as shown in the figures, and the groove-type patterns in the embodiment are in the shape of an impeller), and a mirror symmetry is formed between the groove-type patterns 22 in the left end face and the groove-type patterns 23 in the right end face.

With combined reference to Figs. 1 to 4, the axial contour lines of the groove-type patterns 21 in the outer circumferential face of the bearing inner sleeve 2 are connected with the radial contour lines of the groove-type patterns 22 in the left end face and the groove-type patterns 23 in the right end face, and correspond to each other, that is, the axial high-position lines 211 of the groove-type patterns 21 in the outer circumferential face are connected with the radial high-position lines (221 and 231) of the groove-type patterns 22 in the left end face and the groove-type patterns 23 in the right end face in front of the circumferential chamfer in the end face, and correspond to each other; the axial middle-position lines 212 of the groove-type patterns 21 in the outer circumferential face are connected with the radial middle-position lines (222 and 232) of the groove-type patterns 22 in the left end face and the groove-type patterns 23 in the right end face in front of the circumferential chamfer in the end face, and correspond to each other; the axial low-position lines 213 of the groove-type patterns 21 in the outer circumferential face are connected with the radial low-position lines (223 and 233) of the groove-type patterns 22 in the left end face and the groove-type patterns 23 in the right end face in front of the circumferential chamfer in the end face, and correspond to each other.

By forming a mirror symmetry between the groove-type patterns 22 in the left end face and the groove-type patterns 23 in the right end face, and forming one-to-one correspondence and being connected with each other between the axial contour lines of the groove-type patterns 21 in the outer circumferential face of the bearing inner sleeve 2 and the radial contour lines of the groove-type patterns 22 in the left end face and the groove-type patterns 23 in the right end face, so that the pressurized air generated by the groove-type patterns (22 and 23) resembling an impeller in the left and right end faces is continuously conveyed from a center of the shaft along the radial direction into groove-type-like channels formed by the groove-type patterns 21 in the outer circumferential face, and the required air film to provide a stronger support to the high-speed running bearing can be formed, and the air film serves as a lubricant for the dynamic pressure gas radial bearing, then the high-speed stable operation of the hybrid dynamic pressure gas radial bearing under the air floatation state can be ensured, and provides a further guarantee for achieving the high-limit rotating speed.

As shown in FIG. 5, the hybrid dynamic pressure gas radial bearing provided by the embodiment further comprises the foil type elastic part 3 which is disposed between the bearing outer sleeve 1 and the bearing inner sleeve 2. Due to a wedge-shaped space is formed between the foil type elastic part 3 and the outer circumferential face of the bearing inner sleeve 2, when the bearing inner sleeve 2 rotates, the gas is driven and compressed into the wedge-shaped space due to the viscosity action of the gas, the radial dynamic pressure is obviously enhanced, so that the supporting force can be increased in a multiplied mode; meanwhile, due to the fact that the foil type elastic part 3 is additionally arranged, under the elastic action of the foil type elastic part 3, the load capacity of the bearing can be enhanced, and the impact resistance of the bearing and the capability of inhibiting the axial vortex are all improved remarkably. The number of the foil-type elastic members 3 may be plural (three as shown in FIG. 5), and the two ends of each foil type elastic piece 3 are fixed on the inner circumferential wall of the bearing outer sleeve 1.

As shown in FIG. 5 and FIG. 6, the foil type elastic part 3 can be composed of a wave foil 31 and a flat foil 32, the top end of the arc-shaped protrusion 311 of the wave foil 31 is attached to the flat foil 32, the wave arch transition bottom edge 312 of the wave foil 31 is attached to the inner circumferential wall of the bearing outer sleeve 1, and clamping grooves 12 for fixing the two ends of the foil type elastic part 3 are formed in the inner circumferential wall of the bearing outer sleeve 1, the number of the clamping grooves 12 corresponds to the number of the foil type elastic parts 3, are uniformly distributed along the inner circumferential wall of the bearing outer sleeve 1.

As shown in FIG. 7, a wear-resistant coating is provided on a mating face of each foil type elastic part 3 that matched with the outer circumferential face of the bearing inner sleeve 2 (such as the inner face of the flat foil 32 to form the foil type elastic part 3), in order to further reduce the wear of the foil type elastic part 3 by the bearing inner sleeve 2 when the latter is rotating at a high speed, so as to prolong the service life of the bearing.

In addition, when the stop rings 11 are arranged at the two ends of the bearing outer sleeve 1 respectively, a self-sealing effect can be generated between the two end faces of the bearing inner sleeve 2 and the stop ring 11 driven by a high-speed rotating shaft, so that the

dynamic pressure gas continuously generated by the groove type patterns can be completely and hermetically sealed in the whole matching clearance of the bearing, and the lubricating requirements of the high-speed running dynamic-pressure gas radial bearing can be fully guaranteed.

- 5 The foil type elastic part 3 is preferably subjected to surface heat treatment, so that the performance requirements of high-speed operation can be better met; and the matching clearance between the foil type elastic part 3 and the bearing inner sleeve 2 is preferably 0.003-0.008 mm, so that the reliability and the stability of high-speed operation of the bearing can be further ensured.

10 **EXAMPLE 2**

As shown in FIG. 8, the only difference from the hybrid dynamic pressure gas radial bearing in the first embodiment is that the foil type elastic part 3 is composed of a wave foil 31 and a flat foil 32, the top end of the arc-shaped protrusion 311 of the wave foil 31 is attached to the inner circumferential wall of the bearing outer sleeve 1, and the wave-arch transition
15 bottom edge 312 of the wave foil 31 is attached to the flat foil 32.

The structure of the wave foil 31 is shown in FIG. 9.

EXAMPLE 3

As shown in FIG. 10 and FIG.11, the only difference from the hybrid dynamic pressure gas radial bearing in the first embodiment is that the foil type elastic part 3 is composed of a
20 flat foil 32 and a flat foil 33 with through holes 331.

EXAMPLE 4

As shown in FIG. 12, the only difference from the hybrid dynamic pressure gas radial bearing in the first embodiment is that coaxial through holes 13 and concave holes 14 are formed in the outer circumference of the bearing outer sleeve 1, and the through hole 13 is
25 located in the concave hole 14. The through holes 13 and the concave holes 14 are used for conveniently monitoring the running state of the bearing on line (for example: a temperature

sensor, a pressure sensor, a rotating speed sensor and the like).

In addition, the structure of the foil type elastic part 3 is not limited to that of the above embodiment, as long as the matching relation between the foil type elastic part 3 with the outer sleeve and the inner sleeve is ensured to meet the substantive requirements of the invention.

Test results show that the hybrid dynamic pressure gas radial bearing provided by the invention can be realized a limit rotating speed of 160, 000 rpm to 480,000 rpm under a load of 3-5 kg, but the existing dynamic pressure gas radial bearing can only achieve a load capacity of 1-3 kg, the maximum limit rotating speed can only reach 100,000 rpm to 180,000 rpm. It can be seen that the present invention can realize the application of dynamic pressure gas radial bearings in ultra high-speed areas under larger loads. Compared with the prior art, the present invention has made significant progress, and made the study of the dynamic bearing gas radial bearing technology take a new step.

At last, it is noted here that the above disclosure is intended merely to describe the subject matter of the present invention in further detail and is not to be construed as limiting the scope of the invention. Non-substantive improvements and modifications made by those skilled in the art in accordance with the above disclosure all fall within the scope of the present invention.

Claims

1. A hybrid dynamic pressure gas radial bearing, characterized in that: comprising
a bearing outer sleeve,
an inner sleeve, and
a foil type elastic part arranged between the bearing outer sleeve and the inner
5 sleeve,
wherein an outer circumferential face, a left end face and a right end face of the
bearing inner sleeve are respectively provided with regular groove-type patterns,
the regular groove-type patterns on one of the end faces are in mirror symmetry
with the groove-type pattern on the another end face,
10 axial contour lines of the regular groove-type patterns on the outer circumferential
face, and radial contour lines of the regular groove-type patterns on the left end
face and on the right end face are in one-to-one correspondence and are connected
to each other;
axial high-position lines of the regular groove-type patterns on the outer circum-
15 ferential face of the bearing inner sleeve correspond to radial high-position lines
of the regular groove-type patterns on the left end face and the right end face, and
are connected with each other in front of a circumferential chamfer;
axial middle-position lines of the regular groove-type patterns on the outer circum-
ferential face correspond to radial middle-position lines of the regular groove-type
20 patterns on the left and right end faces, and are connected with each other in front
of the circumferential chamfer; and
axial low-position lines of the regular groove-type patterns on the outer circumfer-
ential face correspond to radial low-position lines of the regular groove-type pat-
terns on the left and right end faces, and are connected with each other in front of
25 the circumferential chamfer.
2. The hybrid dynamic pressure gas radial bearing according to claim 1,
characterized in that
a wear-resistant coating is arranged on matching faces between the foil type elastic

Claims

part and the outer circumferential face of the bearing inner sleeve.

3. The hybrid dynamic pressure gas radial bearing according to claim 1 or 2, characterized in that

5 a fit clearance between the foil type elastic part and the bearing inner sleeve is 0.003-0.008 mm.

4. The hybrid dynamic pressure gas radial bearing according to claim 1, characterized in that

10 two ends of the foil type elastic part are both fixed on an inner circumferential wall of the bearing outer sleeve.

5. The hybrid dynamic pressure gas radial bearing according to claim 1 or 4, characterized in that

15 the foil type elastic parts are multiple pieces, and the foil type elastic parts are uniformly distributed along the inner circumferential wall of the bearing outer sleeve.

6. The hybrid dynamic pressure gas radial bearing according to claim 1 or 4, characterized in that

20 a clamping groove used for fixing the foil type elastic part is formed in the inner circumferential wall of the bearing outer sleeve.

7. The hybrid dynamic pressure gas radial bearing according to claim 1, characterized in that

25 the foil type elastic part is subjected to surface heat treatment.

8. The hybrid dynamic pressure gas radial bearing according to claim 1 or 7, characterized in that

the foil type elastic part is composed of a wave foil and a flat foil;

Claims

a top end of an arc-shaped protrusion of the wave foil is attached to the flat foil;
a transition bottom edge of wave arches of the wave foil is attached to the inner
circumferential wall of the bearing outer sleeve.

- 5 9. The hybrid dynamic pressure gas radial bearing according to claim 1 or 7,
characterized in that
the foil type elastic part is composed of a wave foil and a flat foil,
the top end of the arc-shaped protrusion of the wave foil is attached to the inner
circumferential wall of the bearing outer sleeve, and
10 the transition bottom edge between the wave arches of the wave foil is attached to
the flat foil.
10. The hybrid dynamic pressure gas radial bearing according to claim 1 or 7,
characterized in that
15 the foil type elastic part is composed of two flat foils, through holes being formed
in the flat foils close to the inner circumferential wall of the bearing outer sleeve.
11. The hybrid dynamic pressure gas radial bearing according to claim 1,
characterized in that
20 stop rings are arranged at two ends of the bearing outer sleeve.
12. The hybrid dynamic pressure gas radial bearing according to claim 11,
characterized in that
through holes and concave holes are coaxial and formed on the outer circumfer-
25 ence of the bearing outer sleeve, and
the through holes are located within the concave holes along their common axis.

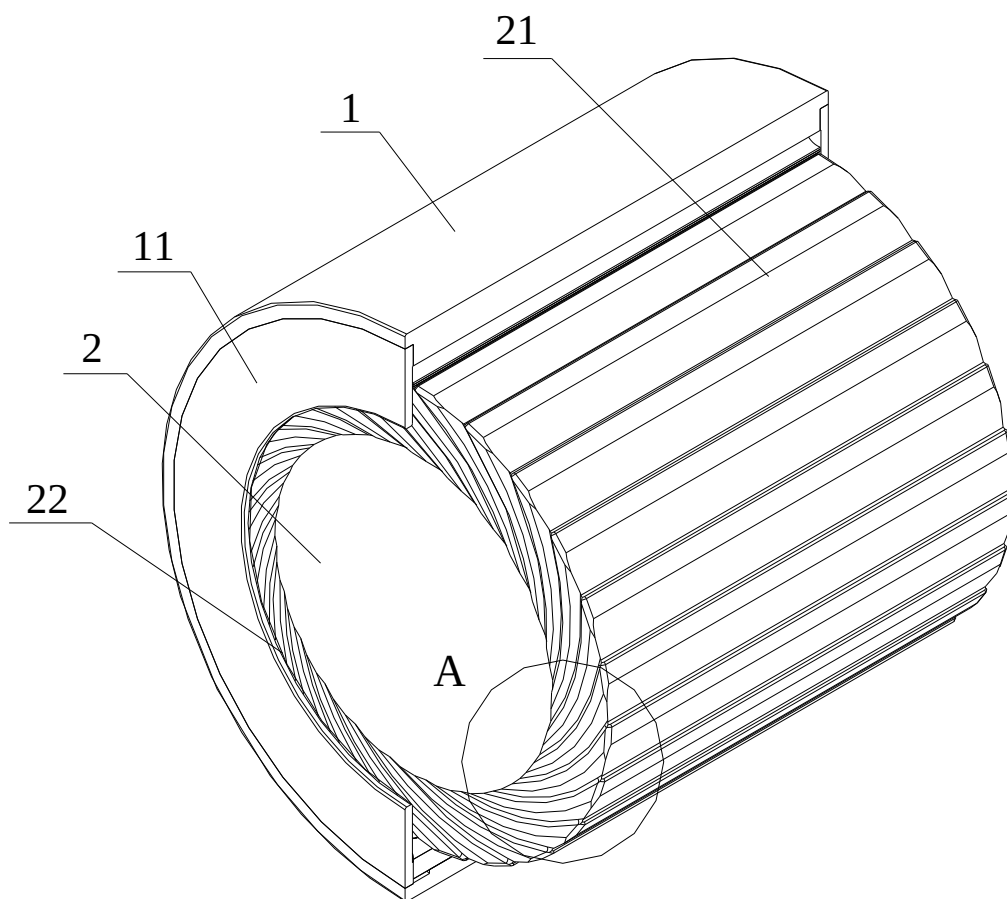


Figure 1

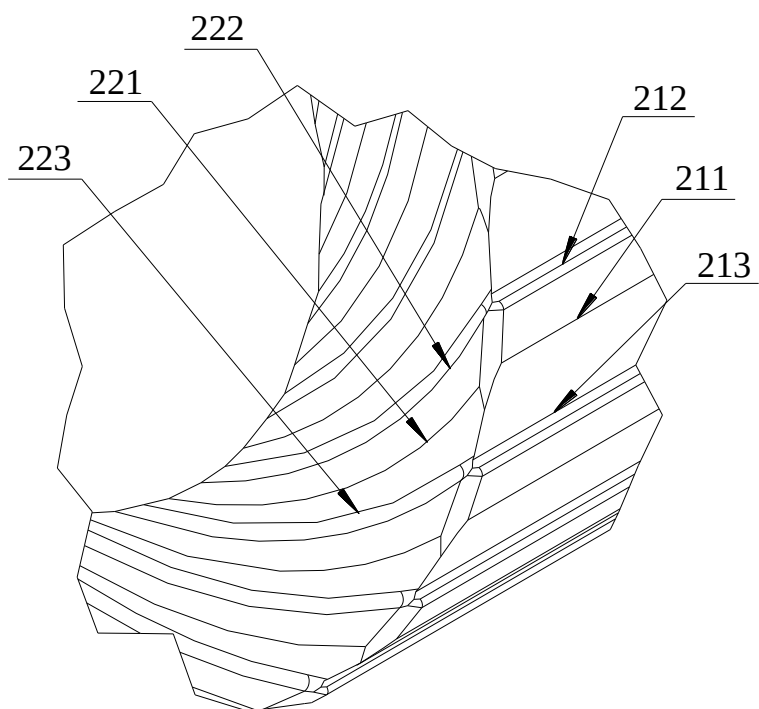


Figure 2

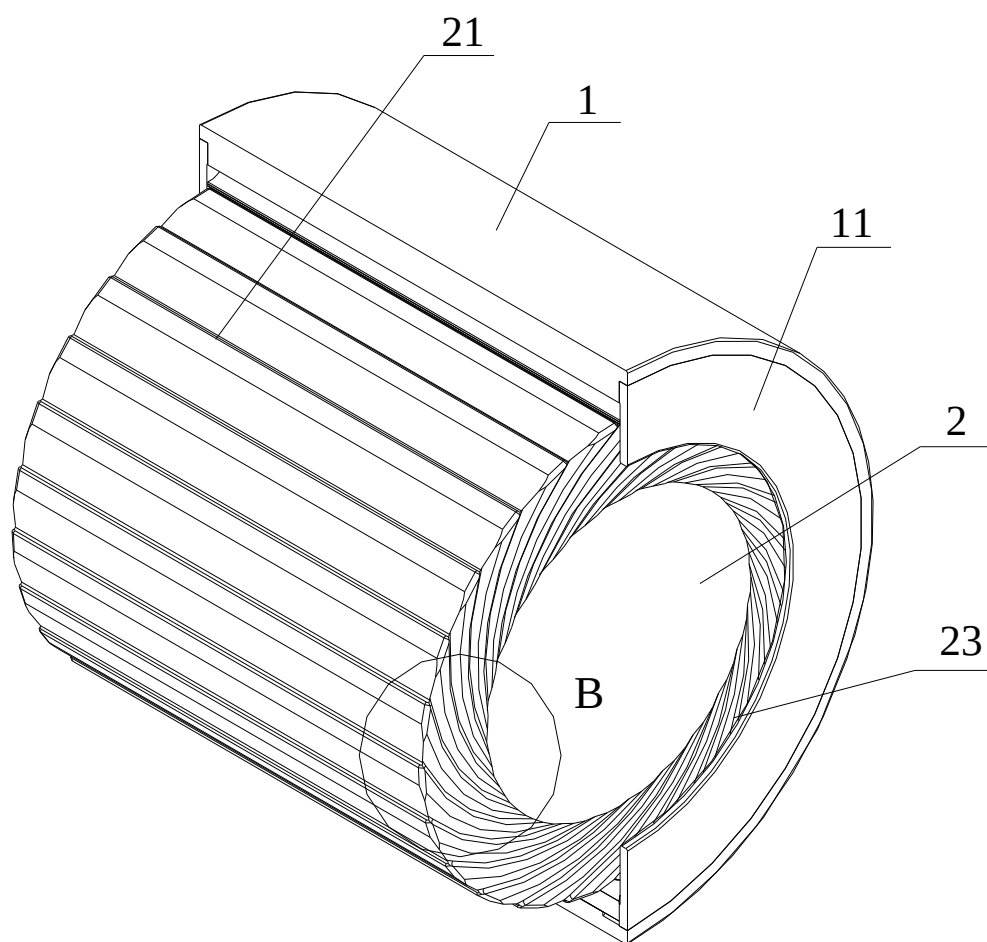


Figure 3

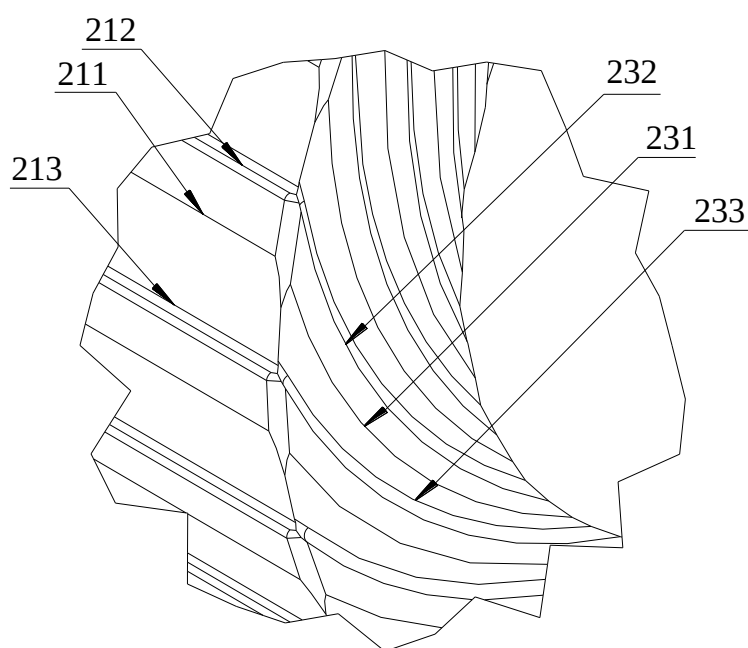
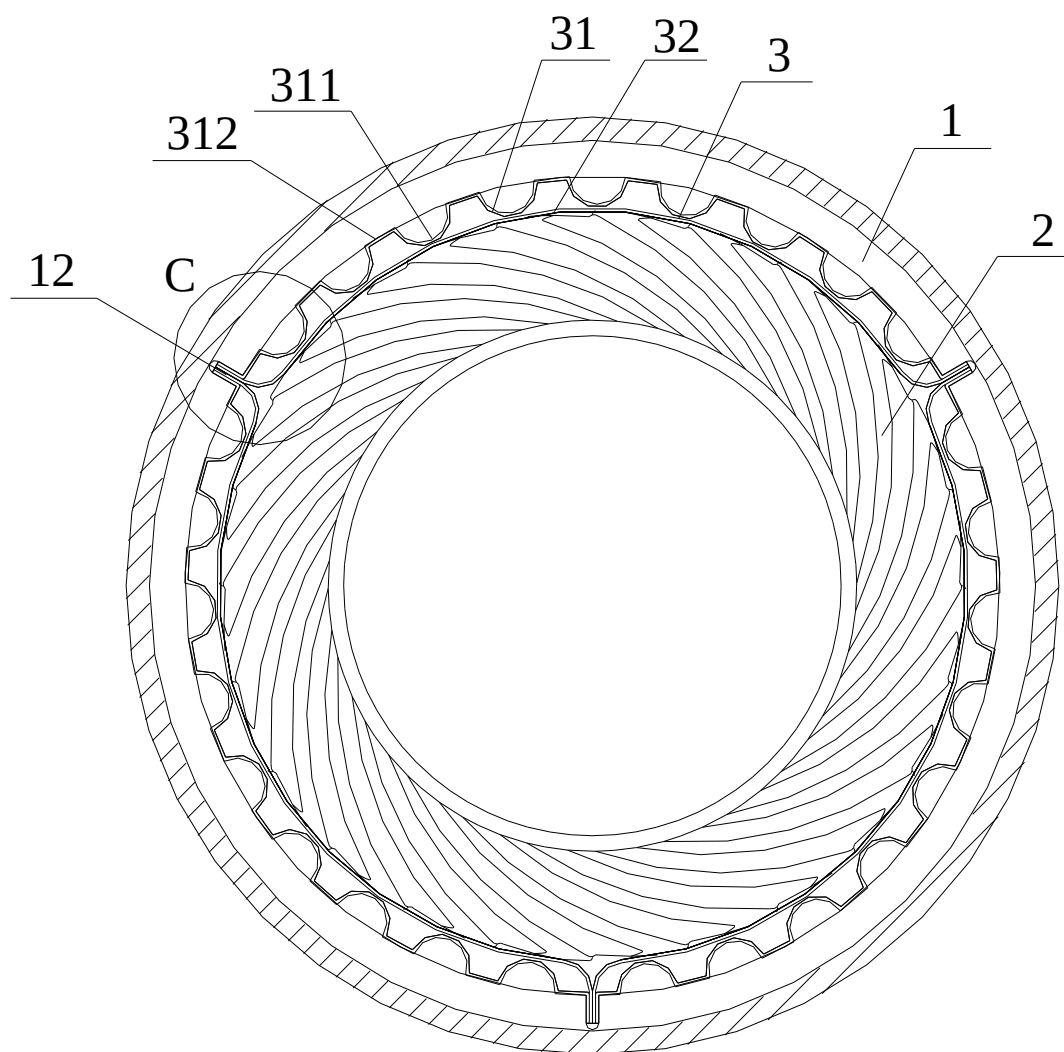
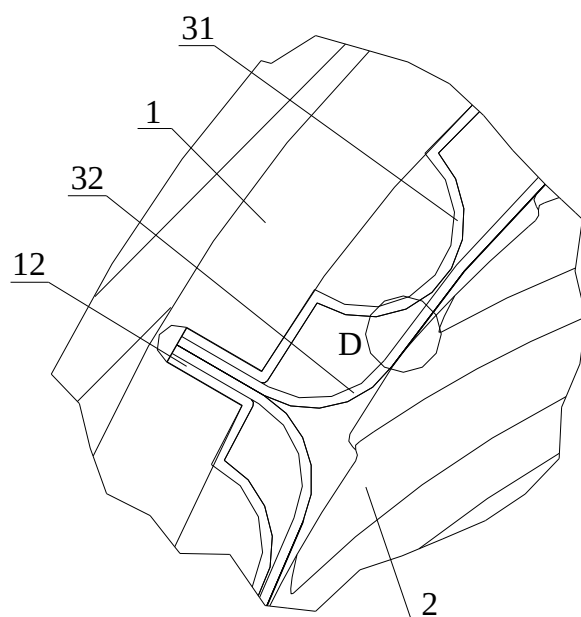


Figure 4

**Figure 5****Figure 6**

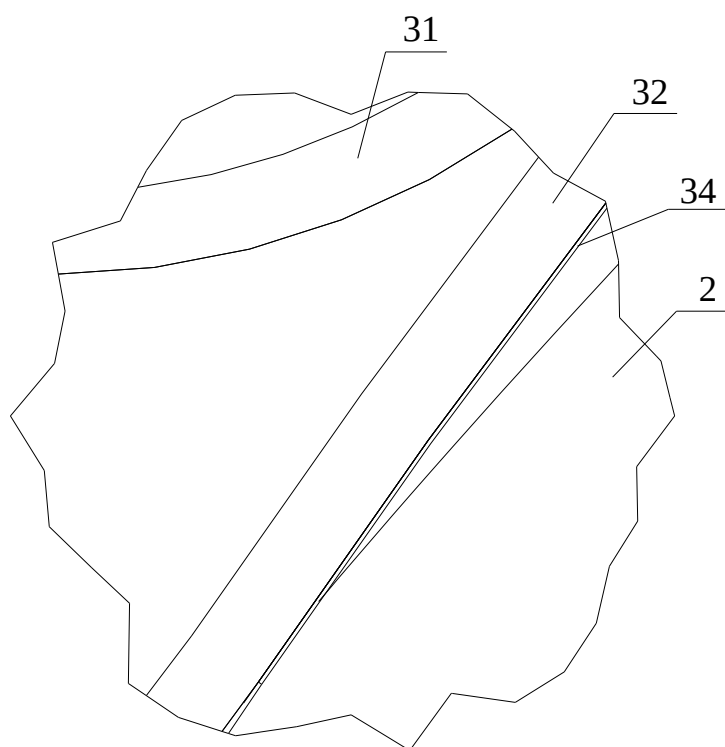


Figure 7

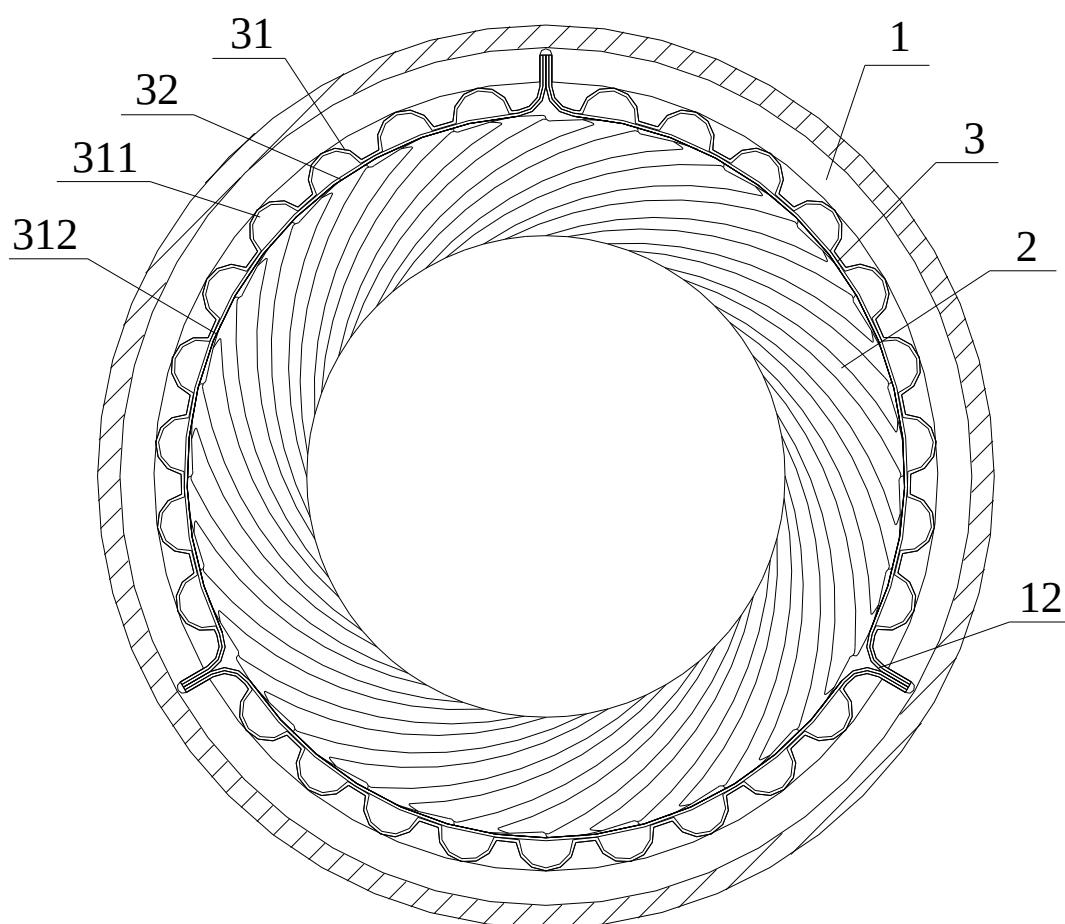


Figure 8

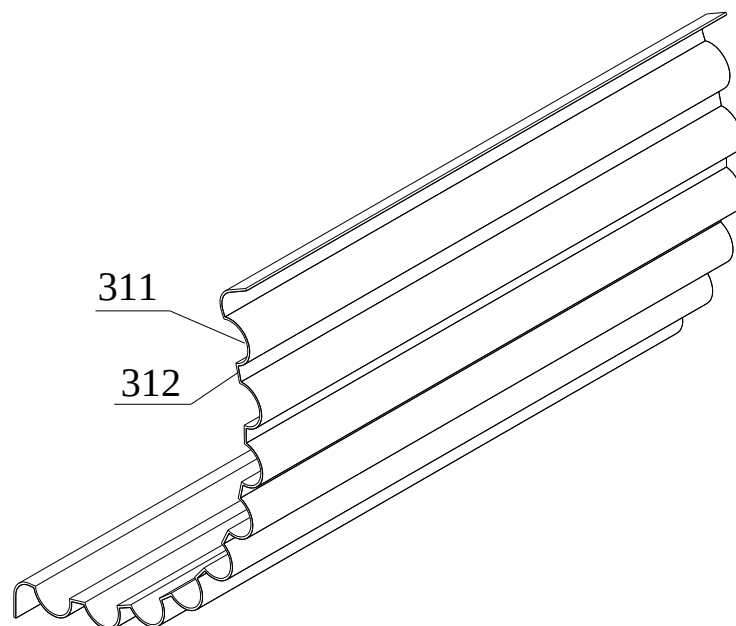


Figure 9

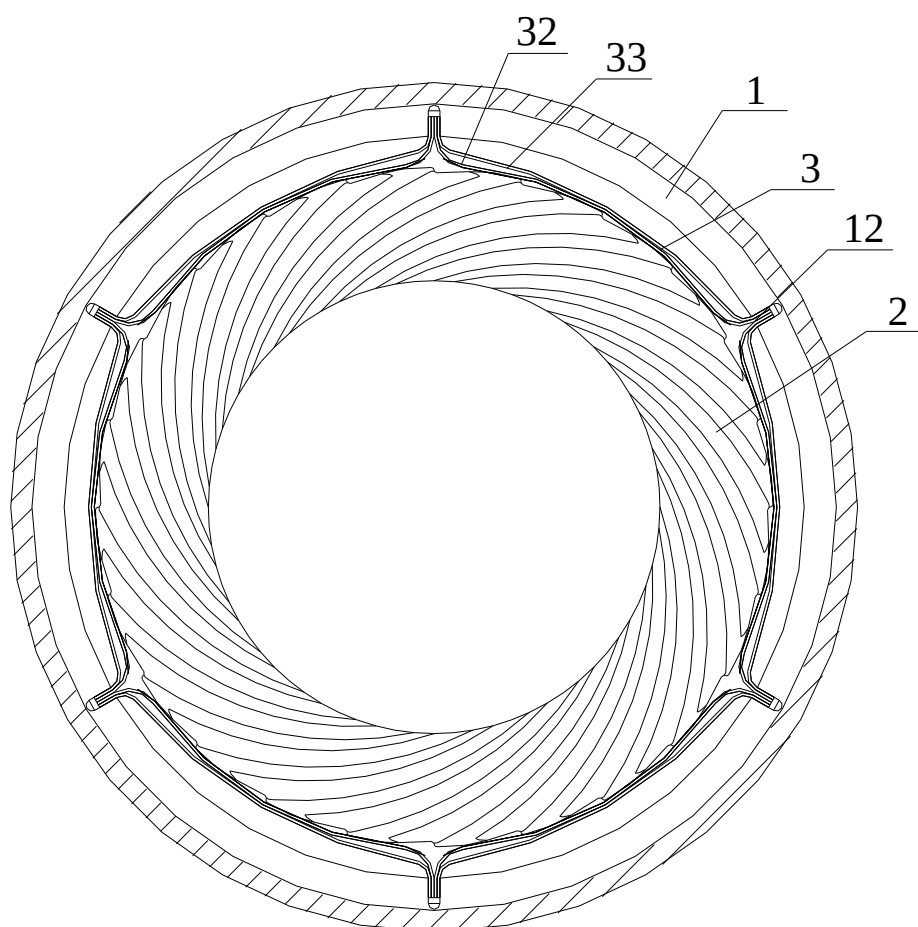


Figure 10

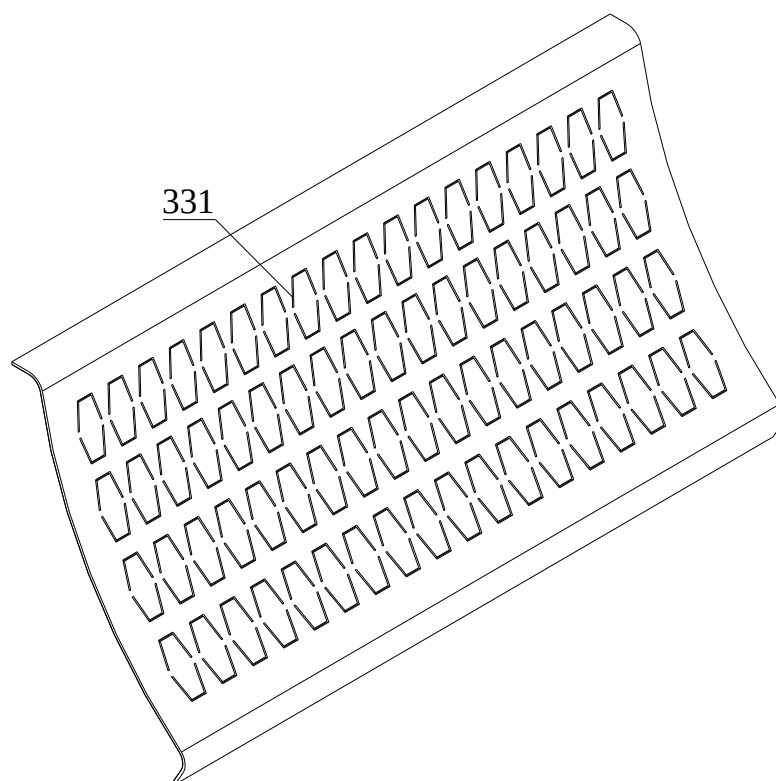


Figure 11

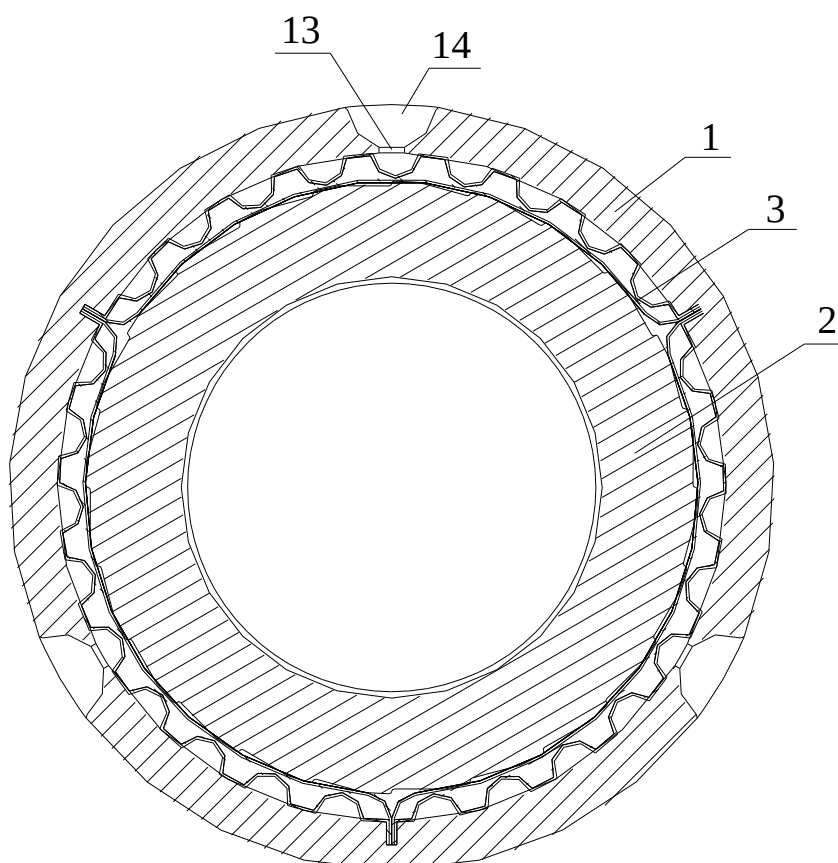


Figure 12