

Description

This invention relates to an improved structure for atomisers, particularly an anti-leakage and reinforcement structure for the atomiser head of a perfume bottle.

Generally speaking, when a conventional atomiser head is being pressed upon, the coordination of the various component members of the atomiser head will be actuated so that the pressure of the liquid which is being compressed inside the compactly sealed space will be stronger than the supporting force generated by a spring member located at the lower end of a moveable valve to enable the triangular shaped valve outlet at the top end of the valve body to be separated from the outlet passage to which it is originally kept in close contact, so as to cause the high pressured liquid to be atomised and sprayed out through the outlet passage.

The utilization of a pumping motion for a component member inside the atomiser head enables the perfume to be atomised and sprayed out from the perfume bottle.

In this pumping motion, in addition to the compressed liquid, an appropriate quantity of air being pumped into the air cavity will help form and complete the process of the pumping motion. By practical experience, it proves that difficulty will be encountered if the air can not enter the air cavity i.e. the compression chamber. That is to say, the liquid will be unable to be smoothly sprayed out.

Therefore, it is known to preset an inlet in the atomiser head to speed up the smooth completion of the pumping motion. As indicated in Figs. 1 and 2, a conventional cylinder piston A includes a first portion A1 which has a rather narrow outer diameter of longer length and a second portion A2 of greater diameter. There is a corner angle in a slanting direction set at the turning angle where the two different diameter portions adjoined each other. This is commonly known as a triangular shoulder A3. The first portion A1 is mounted inside a hollow sleeve B. The sleeve B includes an internal ring body B1. An appropriate gap C is set in between the internal ring body B1 and the first portion A1, while the bottom end B2 of the internal ring B1 is disposed in engagement with the triangular shoulder A3. Before the perfume atomiser head is activated, the bottom end B2 of the internal ring B1 is being kept in close engagement with the triangular shoulder A3, so that the interior of the air cavity will be separated from the ambient air. However, once the atomiser head is being pressed down to cause the downward movement of the cylinder piston body A, as indicated Fig. 2, a liquid compression motion end of the internal ring body B1 will also be separated from the triangular shoulder

A3. Thus, the air will enter the air cavity by passing along gap C which is located in between the internal ring B1 and the first portion A1 complete the pumping motion. Structures similar to the type having an air inlet preset in the atomiser head are commonly found in the products made in various countries in the world.

Therefore, it can be referred to as a conventional prior art. However, as shown in the drawing it can be seen that the engagement between the bottom end B2 of the internal ring B1 and the triangular shoulder A3 will be easily broken when the atomiser head is being pressed down. For instance, the bottom end B2 of the internal ring body B1 will be caused to separate from the triangular shoulder A3, if the atomiser head is slightly pressed when the perfume package is being squeezed at the time when the package is being handled and transported. In this way, the liquid contained in the air cavity will be permitted to leak out through the gap C. Consequently, perfume will be found spreading around the atomiser head when the package is being unpacked. This will not only damage the perfume package, but will also seriously ruin the quality of the perfume and spoil its perfect image of a highly luxurious product.

Other atomisers known in the prior art include those disclosed by EP-A-0213476 and FR-A-2349749. EP-A-0213476 discloses a dispenser which comprises, *inter alia* a dose-chamber body which travels back and forth with a pump, the dose-chamber body being located within the stored medium and bounding on one side, the path for the piston which, in turn, surrounds the pump chamber, and forming on the other side, a valve-body of an inlet valve for the pump chamber. On filling of the pump chamber and movement of the piston over a small distance, a very precise dose of the medium is dispensed for each cycle of the pump. This device, however, is particularly designed for dispensing viscous media and other media which do not flow easily or naturally.

FR-A-2349749 discloses a device which is primarily designed to "pre-load" an atomiser or dispenser in a simple, efficient and cost effective manner so that the material is dispensed only after sufficient pressure has built up within a pump chamber in the dispenser, thereby ensuring that the material is dispensed as a fine mist, with no dribbling, regardless of how quickly the user presses the actuator button.

However, despite the various other advantages of these prior art atomisers or dispensers, both types of dispenser run the risk of the contents leaking out of the dispenser due to slight, inadvertent movement of the piston, for example during packaging or transport.

It is, therefore, an object of the present invention to provide a manual, accumulator type atomiser which overcomes the disadvantages described above.

It is a still further object of the present invention to provide an atomiser which has, in addition to the anti-leakage measure which is set in between the bottom end of the conventional internal ring body and the triangular shoulder, a second anti-leakage at the top end of the internal ring body, so as to ensure that the liquid will not leak out under this double assurance.

According to the present invention, there is provided a liquid atomiser of the type including a cylinder piston moveably received within a ring body, the piston having an annular shoulder and air passage means formed on the piston, characterised in that the ring body defines an annular air gap between the ring body and the piston, the annular shoulder is engageable by a lower edge of the ring body for forming a first seal of the air gap and the ring body is provided with a second seal means including an inwardly directed annular flange disposed in sealing and sliding engagement around the periphery of the piston and spaced from the first seal to prevent liquid from entering the air gap and leaking out of the atomiser due to breaking of the first seal caused by inadvertent separation of the lower edge of the ring body from the annular shoulder, whereby, when the flange encounters the air passage means upon movement of the piston relative to the ring body over a predetermined distance, air is permitted to flow through the air passage means and into the air gap during operation of the atomiser.

In order that the invention may be readily understood, embodiments therefore will now be described, by way of example, with reference to the accompanying drawings, in which :

Fig. 1 shows an enlarged partial view of a conventional atomiser in a first position ;

Fig. 2 shows the conventional atomiser of Fig. 1 on a second position ;

Fig. 3 is a longitudinal sectional elevation of an atomiser according to a first embodiment of the present invention ;

Fig. 4 shows an enlarged partial view of the atomiser before the atomiser of Fig. 3 head has been depressed ;

Fig. 5 shows an enlarged partial view of the atomiser of Fig. 3 when the atomiser head begins to be depressed ;

Fig. 6 shows a view similar to that of Fig. 5 but showing the atomiser after the atomiser head has been depressed ;

Fig. 7 is a sectional view taken along the section 7-7 of Fig. 6 ;

Fig. 8 shows a view similar to that of Fig. 1 but showing a second embodiment of the present invention ; and

Fig. 9 is a sectional view taken along the section 9-9 of Fig. 8.

Referring to the longitudinal perspective view and the partial enlarged view of Fig. 3 and Fig. 4 of the present invention, it can be seen that the cylinder piston body 1 includes a ladder-shaped section with four portions of different diameters. Extending respectively downward, in increasing diameters, are the first diameter portion 10, which is joined with the body of the atomiser head, second 11, third 12 and the fourth 13 diameter portions. The turning corner where the third 12 and fourth 13 diameter portions adjoined each other is defined by a triangular shoulder 14 in a slanting direction, which is engaged by the lower end 31 of the sleeve 3 of the internal ring body 30. At its upper end, an anti-leakage in the form of an annular inwardly directed flange disposed and dimensioned for engagement against the third diameter portion 12. An annular groove 321 is preset at an appropriate distance at the upper end 32 of the internal ring body 30 and extend a downward direction, so body 320 will, by means of the groove 321 and the elasticity of the plastic material, be disposed in close contact with the third diameter portion 12 and in sealing engagement with the third diameter portion 12. Referring to Fig. 5, it can be seen that, during the entire stroke process, as long as the atomiser head is not separated from the third diameter portion 12 when it is being pressed downward, the contact which exists between the anti-leakage body 320 located at the upper end 32 of the internal ring body 30 and the third diameter portion 12 will be maintained, even though the lower end 31 of the internal ring body 30 has already been separated from its contact with the triangular shoulder 14. In this way, the excellent liquid sealing effect of the anti-leakage structure provided by the invention will still be maintained because of the close contact which is being maintained between the anti-leakage body 320 located at the upper 32 of the internal ring body 30 and the third diameter portion 12. By the time the atomiser head is pressed downward to cause the piston body 1 to move downward toward the position of the second diameter portion 11 as is shown in Fig. 6 and 7 when sleeve being maintained stationary, the air will then move along the gap between the body 320 and the second diameter portion 11 until the pressing motion is entirely completed.

From the forementioned description, it is apparent that the main characteristic of the present invention resides in having an anti-leakage body 320 which is provided at the upper end of the internal ring body 30 of the sleeve 3. For the

purposes of matching with the elasticity of the body 320, groove 321 is preset at the top plane of the internal ring body 30 adjacent the position where the body 320 is located. A certain number of ladder-shaped diameter portions 10, 11, 12 are provided on a section of the cylinder piston body 1. During the entire pressing process, an appropriate position will be selected as a direction to arrange for the changes to be provided by the different diameter portions. It is so arranged that the air will be unable to enter if the pressing motion does not reach the fixed position. Consequently, the liquid will not be able to flow out of the container. In view of the defects found in the conventional anti-leakage structure which only a contact maintained between the lower end 31 of the internal ring body 30 and the triangular shoulder 14 as a means of anti-leakage, the present invention provides a second means of anti-leakage, which will overcome such defeats.

A second embodiment of the invention shall be described with reference to Fig. 8 and 9. It can be seen in the embodiments that a certain number of longitudinal grooves 16 of an appropriate length and depth can be provided around the peripheral of the hollow piston body 1. When the grooves 16 are engaged by the anti-leakage body 320, air permitted to enter the air cavity along the gap and through grooves 16 so as to achieve the object of the invention.

In summary, this invention makes use of the changes of the different diameter portions of the cylinder piston body 1 or the provision of the grooves to correlate with the corresponding position and motion of the sleeve 3 with respect to the triangular shoulder 14, whereby air can enter the air cavity only at the time when the pressing process has reached the fixed position. Correspondingly, even if the lower end 31 of the internal ring body 30 happens to be unexpectedly separated from the triangular shoulder 14, the liquid will still be prevented from flowing out because of the second anti-leakage being formed by the anti-leakage body 320 which is located at the upper end 32 of the internal ring body 30.

Claims

1. A liquid atomiser of the type including a cylinder piston (1) moveably received within a ring body (30), the piston (1) having an annular shoulder (14) and air passage means formed on the piston (1), characterised in that the ring body (30) defines an annular air gap between the ring body (30) and the piston (1), the annular shoulder (14) is engageable by a lower edge (31) of the ring body (30) for forming a first seal of the air gap and the ring body (30)

is provided with a second seal means including an inwardly directed annular flange (320) disposed in sealing and sliding engagement around the periphery of the piston (1) and spaced from the first seal to prevent liquid from entering the air gap and leaking out of the atomiser due to breaking of the first seal caused by inadvertent separation of the lower edge of the ring body (30) from the annular shoulder (14), whereby, when the flange (320) encounters the air passage means upon movement of the piston (1) relative to the ring body (30) over a predetermined distance, air is permitted to flow through the air passage means and into the air gap during operation of the atomiser.

2. A liquid atomiser according to Claim 1, which further includes an annular groove (321) formed in the ring body (30) and surrounding the annular flange (320), and wherein the air passage means includes at least one reduced diameter portion (11) on the piston (1).
3. A liquid atomiser according to Claim 2, wherein the air passage means includes three reduced diameter portions on the piston (1).
4. A liquid atomiser according to Claim 1, which further includes an annular groove (321) formed in the ring body (30) and surrounding the annular flange (320), and wherein the air passage means includes a plurality of circumferentially spaced longitudinal grooves (16).

Patentansprüche

1. Ein Flüssigkeitszerstäuber des Typs, der einen Zylinderkolben (1) einschließt, der bewegbar in einem Ringkörper (30) aufgenommen ist, wobei der Kolben (1) eine ringförmige Schulter (14) und Luftdurchlaßmittel, die am Kolben (1) ausgebildet sind, aufweist dadurch gekennzeichnet, daß der Ringkörper (30) einen ringförmigen Luftspalt zwischen dem Ringkörper (30) und dem Kolben (1) definiert, die ringförmige Schulter (14) mit einer Unterkante (31) des Ringkörpers (30) in Eingriff bringbar ist, um eine erste Abdichtung des Luftspaltes zu bilden, und der Ringkörper (30) mit einer zweiten Abdichteinrichtung versehen ist, die einen nach innen gerichteten ringförmigen Flansch (320) einschließt, der in abdichtendem und gleitendem Eingriff um den Umfang des Kolbens (1) herum und mit Abstand von der ersten Abdichtung angeordnet ist, um zu verhindern, daß Flüssigkeit in den Luftspalt eintritt und aufgrund des Aufbrechens der ersten Abdichtung, das bewirkt wird durch unbeabsich-



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In summary, this invention makes use of the changes of the different diameter portions of the cylinder piston body 1 or the provision of the grooves to correlate with the corresponding position and motion of the sleeve 3 with respect to the triangular shoulder 14, whereby air can enter the air cavity only at the time when the pressing process has reached the fixed position. Correspondingly, even if the lower end 31 of the internal ring body 30 happens to be unexpectedly separated from the triangular shoulder 14, the liquid will still be prevented from flowing out because of the second anti-leakage being formed by the anti-leakage body 320 which is located at the upper end 32 of the internal ring body 30.

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tigte Trennung der Unterkante des Ringkörpers (30) von der ringförmigen Schulter (14) hervorgerufen wird, aus dem Zerstäuber herausleckt, wodurch, wenn der Flansch (320) bei Bewegung des Kolbens (1) relativ zum Ringkörper (30) über eine vorbestimmte Entfernung auf die Luftdurchlaßmittel trifft, ermöglicht wird, daß Luft durch die Luftdurchlaßmittel und in den Luftspalt während des Betriebs des Zerstäubers strömt.

2. Ein Flüssigkeitszerstäuber nach Anspruch 1, der außerdem eine ringförmige Nut (321) einschließt, die im Ringkörper (30) ausgebildet ist und den ringförmigen Flansch (320) umgibt, und wobei die Luftdurchlaßmittel wenigstens einen Abschnitt (11) mit verringertem Durchmesser auf dem Kolben (1) einschließen.
3. Ein Flüssigkeitszerstäuber nach Anspruch 2, wobei die Luftdurchlaßmittel drei Abschnitte mit verringertem Durchmesser auf dem Kolben (1) einschließen.
4. Ein Flüssigkeitszerstäuber nach Anspruch 1, der außerdem eine ringförmige Nut (321) einschließt, die im Ringkörper (30) ausgebildet ist und den ringförmigen Flansch (20) umgibt, und wobei die Luftdurchlaßmittel eine Mehrzahl von über den Umfang mit Abstand angeordneten Längsnuten (16) einschließen.

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que si la collerette (320) rencontre le moyen constituant un passage d'air durant un mouvement du piston (1) par rapport au corps annulaire (30) sur une distance donnée, l'air peut passer à travers le moyen constituant un passage d'air et dans la couche d'air pendant le fonctionnement de l'atomiseur.

2. Atomiseur de liquides selon la revendication 1, qui comprend en outre une gorge annulaire (321) formée dans le corps annulaire (30) et entourant la collerette annulaire (320), et dans lequel le moyen constituant un passage d'air comprend au moins une partie (11) de diamètre réduit sur le piston (1).
3. Atomiseur de liquides selon la revendication 2, dans lequel le moyen constituant un passage d'air comprend trois parties de diamètre réduit sur le piston (1).
4. Atomiseur de liquides selon la revendication 1, qui comprend en outre une gorge annulaire (321) formée dans le corps annulaire (30) et entourant la collerette annulaire (320), et dans lequel le moyen constituant un passage d'air comprend plusieurs gorges longitudinales (16) espacées sur la circonférence.

Revendications

1. Atomiseur de liquides du type comprenant un piston (1) de barillet reçu de façon mobile à l'intérieur d'un corps annulaire (30), le piston (1) ayant un épaulement annulaire (14) et un moyen constituant un passage d'air formé sur le piston (1), caractérisé en ce que le corps annulaire (30) définit une couche d'air annulaire entre le corps annulaire (30) et le piston (1), l'épaulement annulaire (14) pouvant être engagé par un bord inférieur (31) du corps annulaire (30) pour former un premier joint d'étanchéité de la couche d'air, et le corps annulaire (30) est pourvu d'un second moyen formant joint d'étanchéité comprenant une collerette annulaire (320) dirigée vers l'intérieur disposée en s'engageant de façon hermétique et coulissante autour de la périphérie du piston (1), et espacé du premier joint d'étanchéité pour empêcher le liquide de pénétrer dans la couche d'air et de couler en dehors de l'atomiseur à cause du bris du premier joint d'étanchéité provoqué par le fait que le bord inférieur du corps annulaire (30) a été séparé par inadvertance de l'épaulement annulaire (14), de sorte

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