KICKOVER TOOL

(15)

This invention relates to kickover tools.

In the past, many different kickover tools have been devised for landing and retrieving valves from side pocket mandrels. While the tools have taken many different forms, they have generally been characterized by having a telescoping relationship between the body of the tool and the actuator of the tool and by a key on the actuator which engaged a shoulder in the well and which was rendered ineffective after the tool had been kicked over by shearing of a pin.

When pulling a valve, a substantial force may be required and various provisions have been made to keep the kickover arm in the kickover position while pulling a valve from the valve pocket of the mandrel.

Many of these tools have been successful and are in operation today. However, they are all complex and usually fragile at one or more points.

It is an object of this invention to provide a simple rugged kickover tool which is inexpensive to manufacture.

According to the present invention there is provided a kickover tool comprising, an elongate body, an elongate actuator alidable relative to the body, an arm connected to at least one of the body and actuator and movable from

retracted to extended position in response to relative sliding movement of said body and actuator, a locator key carried by said actuator and movable laterally of the actuator between a retracted position and an extended position, spring means urging said key towards extended position, means rendering said spring means ineffective to urge said key towards extended position in response to relative movement of said actuator and body from arm retracted to arm extended position, and said spring means is effective upon said relative movement of said actuator and body to arm extended position to urge said key towards retracted position.

The invention will be described now by way of example only with particular reference to the accompanying drawings. In the drawings:-

Figure 1 is a schematic illustration of a kickover tool in a mandrel and in the kickover position;

Figure 2 is a view, partly in elevation, and partly in section showing a kickover tool;

Figure 3 is a view similar to Figure 2 showing the tool in kicked over position;

Figure 4 is a view taken along the line 4-4 of Figure 2;

Figure 5 is a framentary view in section showing an alternative form of latching means;

Figure 6 is a fragmentary view partially in

cross-section and partially in elevation of a kickover tool having a locator key assembly in accordance with the present invention.

Figure 7 is a view along the line 7-7 of Figure 6 of the key;

Figure 8 is a side view of the key along the line 8-8 of Figure 7;

Figure 9 is a rear view of the key along the line 9-9 of Figure 7;

Figure 10 is a view of the spring utilized with the actuator key of Figures 6 through 9;

Figures 11, 12 and 13 are schematic views illustrating sequentially the operation of the actuator key;

Figure 14 is a fragmentary view partly in section and partly in elevation of a modified form of actuator key assembly;

Figure 15 is a view similar to Figure 14 showing the key fully retracted;

Figure 16 is a fragmentary view in section of a modified form of the kickover tool;

Figure 17 is a view along the lines 17-17 of Figure 16;

Figure 18 is a view along the lines 18-18 of Figure 16;

The arrangements shown in Figures 1 to 5 and Figures

16, 17 and 18 do not come within the scope of the claims of this Application but a description of these Figures is included for explanatory purposes.

Referring first to Figure 1, a conventional side pocket mandrel 10 is illustrated. This mandrel will be made up in a well tubing in the conventional manner to provide for the introduction of gas through the ports 11 into the tubing.

A kickover tool is indicated at 12 in the kicked over position with the control valve indicated

generally at 13 immediately above the valve pocket 14. The tool is run into the well in the conventional manner and raised to engage the stop 15 in a conventional muleshoe or orienting sleeve to position the tool in the kicked over position. When so positioned, the tool may be utilized to land or retrieve a valve from pocket 14. Thereafter, the raising of the tool will cause the tool to engage the top of the bulge of the mandrel 10 to move the tool from the kicked over to the aligned position and permit the tool to be withdrawn from the well.

Referring now to Figures 2, 3 and 4, the kickover tool includes an elongate body 16 having a flat side 16a and which preferably is generally semicircular in transverse section. At the upper end of the body 16, a connector 17 is provided for connecting the body to a means for running the tool such as a wire line or pump down tool string.

An elongate actuator 18 is associated with the body 16. The actuator has a flat side 18a and is preferably generally semicircular in transverse section as shown in Figure 4.

At the upper end of the actuator, a vertical slot 19 is provided and an actuator key 21 is supported in position in the slot 19 by a shear pin 22 as shown. A resilient means such as a leaf spring 23 urges the key to extended position as shown in Figure 2 while permitting the key to retract into the slot 19 as the key strikes obstructions in the tubing when lowered in a well. The key illustrated operates in the conventional manner to engage the shoulder in the muleshoe 15 in the tubing arresting upward movement of the actuator to move the tool to kicked over position and in the conventional

manner. The key is released for passing the shoulder by shearing of the pin 22 to permit the key 21 to drop into the slot 19 and disengage the shoulder.

Means are provided for slidably interconnecting the flat sides of the body 16 and the actuator 18 with these flat sides in confronting relationship. This slidable connection may be provided in any desired manner as by the tee 24 extending along an intermediate length of body 16 cooperating with a conforming T-shaped slot 25 in the actuator 18. The slot 25 is longer than the tee 24 as shown by comparison of Figures 2 and 3 to provide for sliding movement of the actuator relative to the body.

An arm, indicated generally at 26, has a slotted section 27 and a support section 28. The arm is pivoted to one of the body and actuator adjacent the lower end of thereof. In the preferred form, the body 16 is provided adjacent its lower end with a slot 16b which receives the slotted section 27 of the arm. The arm is pivoted to the body 16 by a pivot pin 29 in the body which extends through the slot section 27 and supports the arm 26.

The slotted section 27 of the arm has a flat sided slot 31 therein through which the pivot pin 29 extends and the arm is pivotal about the pin 29. A full slot is not necessary and a short slot could be provided for either pin but a full slot is preferred for manufacturing convenience.

The actuator 18 is provided adjacent its lower end with a slot 32 into which the slotted section 27 of arm 26 extends. A pivot pin 33 carried by the actuator 31 extends through the slot 32 in the actuator 18 and through the slot 31 in the arm.

Upon relative reciprocation of the body 16 and the actuator 18, their pivot pins 29 and 33 will cause the arm 26 to rotate between its in-line position shown in Figure 4 and its kick-over position shown in Figure 3.

By reference to Figure 3, it will be noted that with the actuator 18 in its full down position, the slot 31 extends at an angle of approximately 45 degrees to the longitudinal axis of body 16. The slot preferably rotates through an arc of 90 degrees. When an upward pull is exerted through the body 16 on the arm 26, as when a valve is being removed from the valve pocket in a mandrel, the arm will be biased to rotate about pivot pin 29. The rotational force will be applied to pivot pin 33 in the actuator 18 in a direction 45 degrees to the longitudinal axis of the body 16. It results that a substantially equal force is applied to the actuator 18 to move it to in an upward direction while at the same time moving the actuator 18 at the point of pin 33 outwardly from the body in a direction normal to the longitudinal axis of the actuator 18. Thus, a substantial frictional resistance is set up in the T-slot connection resisting return of the arm 26 to its aligned position. It has been found in a model of the tool that a steady force applied to the body 16 in an upward direction will require several hundred pounds to be applied before the arm 26 will rotate to its in-line position. It is believed that the force applied results in bending of the actuator 18 in the space between the T-slot connection and the pivot 33 until the arm 26 is able to rotate a slight distance to an angle less than 45 degrees relative to the longitudinal axis of the body. This upsets the force diagram and provides

a greater force tending to move the actuator 18 upwardly as compared to the force applied laterally of the tool which is inducing the frictional relationship in the T-slot. As these forces become unbalanced, the actuator 18 will be rotated to its in-line position. Thus, in removing a valve from a valve pocket in a mandrel, the relationship of the arm and pivots permits several hundred pounds of force to be applied to remove the valve from the pocket without rotating the arm 26 thus maintaining the valve in alignment with the pocket while it is being removed therefrom. Then after the valve has been removed and the tool moves up hole far enough for the arm 26 to engage the top of the side pocket, a sufficient force may be applied to overcome the frictional resistance and rotate the arm to its in-line position. Of course, the sudden contact of the arm with the side pocket mandrel at the top thereof will set up shock waves which will release the frictional engagement and rotate the arm 26 to its in-line position at a much smaller force than required when a steady pull is applied.

The arm 26 carries at the free end of its support section, the conventional tool carrier 34 which will have secured thereto either a running tool or a pulling tool to which the valve is attached in conventional manner.

Means are provided for releasably holding the body and actuator together in a first position with the support section of the arm extending substantially in line with the body as shown in Figure 2. This releasable holding means also holds the body and actuator together in a second position with the support section extending transversely of the body as shown in

Figure 3. Any suitable type of means can be utilized for this purpose. As shown in Figures 2 and 3, the actuator 18 may have depressions 35 and 36 therein which cooperate with the detent ball 37 which is urged towards the actuator by spring 38 held in place by a suitable screw or the like. This spring 38 may take any form such a coil spring or a stacked group of Belleville washers.

In an alternate form, a shear pin may be utilized in place of the detent ball 37 as shown in Figure 5 and instead of depression 35, the actuator 18 may have a bore 39 and a counterbore 41. The bore 39 receives a removable plug 42 which, as shown has its inner end positioned below the confronting surfaces 16a on the body 16 and 18a on the actuator 18. A shear pin 43 is substituted for the detent ball 37.

A like bore 44, counterbore 45, and plug 46 are substituted for the depression 36. Figure 5 shows the body and actuator in the run position. As the body and actuator are moved relatively to each other, the shear pin 43 will be sheared and the portion remaining within the body 16 will slide along the confronting surface 18a of the actuator until it reaches the bore 44 at which time the spring 38 will force the remainder of the shear pin 43 into the bore 44 to hold the parts in the kickover position. Then when the tool is pulled up through the top of the side pocket mandrel and the arm engages the top of the mandrel and is forced to rotate clockwise as viewed in Figure 2, the shear pin 43 will again shear and release the body and actuator to permit them to return to the position of alignment as shown in Figure 2. While the shear pin form of holding means can be designed to provide

high shear forces, it does not hold the arm in the in-line position after the shear pin has been sheared and the tool returned to its Figure 2 relationship. The use of the detent as shown in Figure 2 permits the tool to be held in the in-line position.

In Figures 16, 17 and 18 a further form of holding means is illustrated. In this form a friction means is provided between the body 16 and the actuator 18 which resists relative movement between these two parts. The friction means is adjustable so that the operator may provide for as much frictional resistance as desired to adjust the tool for various operating conditions. For instance, in very deep wells the operator might wish to set the holding means to provide for minimum resistance. On the other hand, where a highly deviated well is involved, the operator might wish to provide for maximum resistance to prevent any possibility of the tool falling into a belly down mandrel.

In this form of the tool the actuator 18 is held against the body 16 by a friction bolt 101 which is threadedly secured to the body 16 as shown at 101a. As best shown in Figure 18, the actuator 18 has a slot 102 adjacent the body 16 and a wider slot 103 overlying the slot 102 to provide a shoulder therebetween. The friction bolt 101 will preferably be provided with a suitable friction means, such as washer 104. This washer

will preferably be of brass, plastic, or other like materials, which will not bind or gall with the actuator 18 but will provide a substantial frictional contact therewith. Preferably, the washer should be such that the operator, by changing the tightness of the screw with the body 16,

can adjust the amount of frictional drag between the washer and the actuator 18. After the drag bolt 101 has been adjusted, it may be locked in place by the lock screw 105. The bolt 101 provides a T-shaped member which engages the essentially T-shaped slots 102 and 103 in the actuator.

From the above it will be seen that any desired type of holding means can be utilized to hold the actuator and body in aligned position and, if desired, in kicked over position. The friction drag means of Figure 16 is presently preferred, as it does not require redressing and provides a simple adjustable means for holding the actuator and body in the desired relationship.

Figures 16 and 17 show a slightly different form of kickover arm. The arm 106 is pivoted to the body by pin 107 which permits only rotative movement of the arm 106. The arm is operated by the actuator 18 through the pins 108 and 109 which are welded to the actuator 18, as shown in Figure 1, and confront each other. The two pins 108 and 109 slide within slots 111 and 112 on opposite sides of a web 113. This web of material 113 reinforces and strengthens the section of the arm which is pinned to the actuator 18 by the pair of pins 108 and 109. During operation the arm will not project beyond the outer periphery of the actuator 18.

If desired, the friction means 101 may be spaced a substantial distance from the kickover arm 106 and the friction induced by the forty-five degree angle relationship of the portion of the arm 106 which engages the actuator 18 minimized. On the other hand, the form of tool shown in Figure 16 could have the friction means positioned closely

adjacent to the arm or, in the alternative, the tongue and groove relationship between the body and actuator could additionally be provided in the Figure 16 form of tool at a point adjacent to the kickover arm to give this additionally frictional resistance to movement of the actuator when pulling a tool, if desired.

In the operation of the tool as thus far described, the tool is dressed in the Figure 2 relationship and either a pulling or running tool is attached to the tool carrier 34. In the case of the running tool, a valve will also be attached. The tool is run in the conventional manner down to a point below the mandrel and is then raised until the key 21 engages the shoulder 15 in the mandrel. Application of a steady upward pull results in the friction of washer 104 being overcome, the detent ball 37 being released, or the pin 43 shearing permitting the arm 16 to rotate in a counterclockwise direction as viewed in Figure 2 to move the arm to the fully kicked over position shown in Figure 3. After the arm has been rotated and is held in position by friction washer 104 or is detented or latched by the shear pin 43 in the kickover position of Figure 3, the tool is lowered to either land a valve in the pocket of the side pocket mandrel or to engage a valve already in the mandrel. Then an upward pull on the tool either releases the running tool or pulls the valve from the pocket. This action is carried out with a steady force or a jar being applied and the holding action provided by the friction washer, detent or shear pin is augmented by the resistance to rotation of the arm provided by the relationship of the 45-degree slot and the longitudinal axis of the body as

hereinabove explained. While it is preferred to use this relationship of slot and body, at least with the Figure 2 form of the tool, either the washer, the shear pin or detent may be relied upon to hold the arm 26 in the kickover position of Figure 3 until the valve has been released or removed from the pocket. Thereafter, continued upward movement of the tool causes the arm to strike the top of the bulge of the side pocket mandrel and to rotate to the in-line position of Figure 2. This may be effected by supplying sufficient force to overcome the frictional resistance to movement between the body 16 and actuator 18 or the arm may be moved rapidly to strike a sharp blow against the top of the side pocket mandrel to induce vibrations in the several parts of the tool and permit it to readily assume the Figure 3 position.

If desired, where the shear pin system of Figure 5 is utilized, the kickover tool may also have an additional detent such as that provided in Figures 2 and 3 which will cooperate with the shear pin to resist the initial reciprocation of the body and actuator to kickover position and if desired, to assist in holding the arm in kickover position and then when the tool is returned to the in-line position function, to maintain the kickover arm in the aligned position after the pin 43 has been sheared.

While the 45 degree relationship of arm 26 to the axis of the tool will serve as a latch when the tool is in kicked over position, it is apparent that a slightly different angular relationship will also give the same results and thus a relationship of approximately 45 degrees will result in the desired latch relationship.

Figures 6 through 13 show one preferred form of actuator key and means for extending and retracting the key.

The body 50 is the same as body 16 except that a slot 47 extends transversely through the body.

Within the actuator key receiving slot 19 of actuator 18, there is provided a key having special provisions for supporting a resilient means. The key is generally conventional in form with the exception of the means for supporting a spring. The back side of the key adjacent its free end has slots 48 and 49 formed therein leaving an upstanding flange 51 which has a bore 52 therethrough for receiving a retaining pin 66.

At the opposite end of the key a bore 53 is provided for receiving the pivot pin 54 which pivotally supports the key on actuator 18.

Between the slots 48 and 49 and the bore 53, slots 55 and 56 are provided for receiving legs of a spring as will appear hereinafter. A bore 57 extends through the key to receive the ends of a spring as will appear hereinafter.

In Figure 10, a special spring is shown having two coil intermediate sections 58 and 59 with a continuous U-shaped leg 61 connecting the two coil sections. The free ends 62 and 63 of the spring extend in planes parallel to the legs of the U-shaped section 61 and terminate in in turned end portions 64 and 65.

In assembly, the spring is positioned on the back side of the key with the coil sections 58 and 59 on opposite sides of flange 51 and a pin 66 extends through the coils and the flange to fasten the spring in place. The legs 62 and 63 lie within the grooves 55 and 56 on the key and the two bent end

portions 64 and 65 extend into the hole 57 in the key thus fixing the legs 62 and 63 to the key.

In unstressed condition, the U-shaped leg 61 of the spring extends toward the front 67 of the key and forms with the legs 62 and 63 an angle slightly in excess of 200 degrees measured clockwise from leg 61.

In assembly, the U-shaped leg 61 is forced to assume the position shown in Figure 6 and bears against the confronting face 50a of the body 50. The actuator 40 has a slot 40a cut in the upper end thereof to cooperate with the spring as will appear from the discussion of operation of the actuator key.

Figures 11, 12 and 13 illustrate the manner in which the spring controls the extension and retraction of the key. In Figure 11, as in Figure 6, the U-shaped leg 61 of the spring is shown to be in contact with the confronting surface 50a of the body 50 when the tool is in running condition with the kickover arm 26 having its support portion 28 extending along the center line of the body. When the key contacts the shoulder 15 in the mandrel, and the body moves upwardly relative to the actuator, the leg 61 extends into the slot 47 in the body and unwinds from the dotted line position shown in Figure 12 to the solid line position where the leg 61 extends into the slot 40a in the top of the actuator and engages the actuator as shown. At this time, due to the upward force being exerted on the tool, the key is held against the shoulder 15 in the side pocket mandrel until the actuator and body have moved to the fully kicked over position of Figure 3. Then, as the operator lowers the tool in the well to land or retrieve a valve, the key is moved away from shoulder 15 and

the spring leg 61 rotates to the position shown in Figure 13 and retracts the key into the actuator 40 and maintains the key in the retracted position until the tool is redressed. Thus, the key will not interfere with withdrawing the tool from the well. As the key and spring are now entirely supported within the actuator by the upper outer corner 68 of the key being held against the top of slot 19 by spring leg 61, the key becomes ineffective during future use and withdrawal of the kickover tool from the well.

While a single spring accomplishing the two functions of urging the key outwardly and subsequently urging the key inwardly has been illustrated, it will be appreciated that two separate springs could be utilized to accomplish these functions. For instance, the slot 40a could be sufficiently large that with the key in the Figure 13 position, the leg 61 would not exert any force on the key. Then, another spring positioned in any desired manner to bear against the key and urge it to retracted position, such as a spring coiled about pin 54 and bearing against the key to urge it in a clockwise direction as viewed in Figure 6 could be utilized to retract the key. Where two springs are used, the force of the retracting spring must, of course, be less than the force of the extending spring so that the retracting spring

only becomes effective upon the extending spring becoming ineffective.

An alternative form of actuating key is shown in Figures 14 and 15. This key differs only from the previously described key in that the key slides in two tracks instead of being rotated about a pivot.

In the alternative form of key, the actuator 70 has a pair of spaced slots 71 and 72 therein. The actuator key 73 has a pair of pins 74 and 75 which reciprocate in the slots 71 and 72 permitting the key to be retracted and extended relative to the actuator 70.

The key may be contoured to receive the spring in the same manner as shown in Figures 7, 8 and 9 and the leg 61 of the key will engage the body in the same manner as before explained.

The only difference in function between the key of Figure 14 and the other key, is that the force exerted by the spring slides the key 73 in the slots 71 and 72 instead of rotating the keys about a pivot pin as shown in Figure 6.

Reference is made to our copending Application No. 8235814 from which the present Application is divided.

CLAIMS

1. A kickover tool comprising, an elongate body, an elongate actuator slidable relative to the body, an arm connected to at least one of the body and actuator and movable from retracted to extended position in response to relative sliding movement of said body and actuator, a locator key carried by said actuator and movable laterally of the actuator between a retracted position and an extended position, spring means urging said key towards extended position, means rendering said spring means ineffective to urge said key towards extended position in response to relative movement of said actuator and body from arm retracted to arm extended position, and said spring means is effective upon said relative movement of said actuator and body to arm extended position to urge said key towards retracted position.

2. The tool of Claim 1 wherein said spring means includes two springs and after the first spring is rendered ineffective the second spring is effective to move the key to retracted position.

3. The tool of Claim 1 wherein said spring means is a single spring having a leg bearing against said body while urging said key towards extended position, said body has a slot therein into which said leg extends to render it ineffective as the body and actuator move towards arm extended position, and said actuator has a shoulder engaged by said leg as the body and actuator move relatively to full arm extended position, said leg when engaging said shoulder rendering said spring effective to urge the key towards retracted position.