

## DESCRIPTION

### MIXING ELEMENT AND STATIC FLUID MIXER USING THE SAME TECHNICAL FIELD

The present invention relates to an improvement on a mixing  
5 element used in a static fluid mixer which mixes one or more  
kinds of fluids (liquids, gases, solids or mixtures of these)  
without a mechanically movable part.

Static fluid mixers of this kind are used in mixture,  
agitation, extraction, distillation, gas absorption, solution,  
10 diffusion, emulsification, heat exchange, dispersion,  
particulate mixture and the like.

Further, static fluid mixers are used in many fields such  
as chemical industry, paper and pulp industry, petrochemical  
industry, pharmaceutical industry, semiconductor industry,  
15 optical fiber manufacturing industry, energy industry and  
environment-related industry.

For example, a static fluid mixer is used as packing in an  
exhaust gas treatment apparatus of an absorption column method  
in which harmful substances in exhaust gas such as HCl, NH<sub>3</sub>, NO<sub>x</sub>,  
20 SO<sub>x</sub>, SiCl<sub>4</sub>, SiHCl<sub>3</sub>, SiF<sub>4</sub>, CO<sub>2</sub>, Hg and dioxin undergo gas-liquid  
contact, or in a dust removing apparatus which traps and  
collects SiO<sub>2</sub>, microparticles such as soot and dust in exhaust  
gas, and in a distillation apparatus. Further, a static fluid  
mixer is used as an apparatus for removing and collecting

organic chlorine-based chemical compounds, ammonia ( $\text{NH}_4^+$ ) and the like, in which discharged water undergoes diffusion treatment.

#### BACKGROUND ART

Patent application for a mixing element and a static fluid mixer using the element of related art have been made by the inventor, in which two or four right-twisted/left-twisted blades are disposed in a passage tube. The static fluid mixer has an opening at its central part and the edges of right-twisted and left-twisted blades thereof are alternately disposed at right angles to each other with space portions in between. Further, twisting angles of the blades are  $90^\circ$ ,  $180^\circ$ , and  $270^\circ$ . Further, the production method of the static fluid mixer includes a process of dividing a passage tube into several parts in the longitudinal direction and joining two or four blades to an inner wall portion of the passage tube divided and a process of joining together the divided surfaces of the passage tube.

(refer to, for example, Japanese Published Patent Application No. H5-168882)

Next, a production method of a mixing element having blades which are provided inside a cylindrical passage tube to form a

plurality of fluid passages that are continuous with an opening in between is explained. Regarding the mixing element, the passage tube and the blades are produced separately and are joined to produce the mixing element. Twisting angles of the

mixing element are  $90^\circ$ ,  $180^\circ$ ,  $270^\circ$  and  $360^\circ$ . (refer to, for example, Japanese Published Patent Application No. H7-284642) Further, the mixing element is formed of a plurality of spiral blades provided in a passage tube, without the blades provided at the central part of the passage tube, and inner cylindrical tubes are disposed at intervals in the part where the blades are not provided, so as to enhance mechanical strength. The rotation angles of the blades are  $90^\circ$  and  $180^\circ$ , or  $30^\circ$ ,  $45^\circ$  and  $135^\circ$ . (refer to, for example, Japanese Published Patent Application No. 2001-170476)

Further, the mixing element includes: an outer cylindrical tube, blades provided in the outer cylindrical tube, and inner cylindrical tubes provided at intervals so as to provide these blades in the outer cylindrical tube. (refer to, for example, Japanese Published Patent Application No. 2001-187313)

Regarding a mixing element of related art, it is necessary due to the difficulty in terms of production to enlarge the cross-sectional area, that is to say the diameter, of an opening (the central part) as the internal diameter of a passage tube, through which fluid flows, becomes large. For that reason, there is a disadvantage that the effectiveness of mixture and agitation diminishes, because fluid flows in an opening, that is, a short circuit occurs. Further, in order to compensate for the decrease in the effectiveness of mixture and agitation, it is necessary to provide a large number of mixing elements, so that

equipment costs become high.

Furthermore, if a mixing element of a large diameter (internal diameter of 1000mm or more) is produced, production becomes impossible, and there is a disadvantage that mixture and agitation efficiency diminishes greatly. Further, it is impossible for the mixing element to be disposed as packing in a distillation column already installed, because members thereof are large.

Moreover, when a mixing element of a small rotation angle (for example, approximately  $10^\circ$ ) is produced and used, the mixing element can be provided as packing in a distillation column already installed, which enables high performance and high productivity to be obtained.

[Patent Literature 1] Japanese Published Patent Application No. S58-128134

[Patent Literature 2] Japanese Published Patent Application No. H5-168882

[Patent Literature 3] Japanese Published Patent Application No. H7-80279

[Patent Literature 4] Japanese Published Patent Application No. H7-284642

[Patent Literature 5] Japanese Published Patent Application No. 2001-170476

[Patent Literature 6] Japanese Published Patent Application  
No. 2001-187313

[Patent Literature 7] European Patent No. 0678329

[Patent Literature 8] US Patent No. 5,605,400

[Patent Literature 9] US Patent No. 6,431,528

[Non-patent Literature 1] S.J. Chen et al. "STATIC MIXING  
HANDBOOK"

Chemical Research Institute, published in June 1973

[Non-patent Literature 2] Kiichiro Matsumura, Yasushi Morishima  
et al. "STATIC MIXER - FUNDAMENTALS AND APPLICATIONS"

Nikkan Kogyo Shimbun, Ltd., published on September 30, 1981

#### DISCLOSURE OF THE INVENTION

(Problems to be Solved by the Invention)

As regards a mixing element and a static fluid mixer using the mixing element in related art, since mixture and agitation efficiency diminishes as the diameter of a mixing element enlarges, it is necessary to increase an amount of time when fluids are mixed and agitated. Therefore, equipment costs become high. In addition, production and assembly become difficult as a diameter becomes large, and so tool costs become high as well. Further, it is impossible to use as packing in distillation columns already installed, because of the size of members and performance-related aspects. Also, similarly, it

has been impossible to be used as packing in an absorption column already installed for treating large gas volume. Further, a large gas-liquid contact interfacial area, a high-performance liquid distribution function, and a wide operation control range under low pressure loss have been demanded regarding a distillation column using packing, (refer to, for example, Japanese Published Patent Application No. H7-080279) As volume of exhaust gas generated from incinerators, vessels,. power plants and the like to be treated becomes large, such advantages as high performance, less working space, less energy consumption and less cost of an absorption column used in exhaust gas treatment apparatuses are desired.

(Means for Solving the Problems)

In accordance with a first aspect of the present invention, there is provided a mixing element comprising: a cylindrical passage tube through which fluid flows; spiral first blades of rightward rotation or leftward rotation provided in said passage tube; a first inner cylindrical tube disposed in the axial center portion of said first blades; spiral second blades of rightward rotation or leftward rotation provided in said first inner cylindrical tube; and a second inner cylindrical tube disposed in the axial center portion of said second blades, wherein said first blades and said second blades are formed of a perforated object or a porous object. Embodiments of the present invention may provide, a mixing

element in which high mixture efficiency is obtained, production thereof is facilitated, and production cost is low. Further, a mixing element capable of being applied to a gas-liquid contact apparatus of a large diameter (1m or more) of the distillation column method or absorption column method may be provided by embodiments of the present invention.

(Effect of the Invention)

Embodiments of the present invention may further provide a mixing element where gas-liquid contact time will be reduced with the improvement in mixture and agitation efficiency. Further, since the production is facilitated, production costs may be reduced as well. Further, the production of a distillation column or absorption column having a large diameter may be simplified.

#### BRIEF DESCRIPTION OF DRAWINGS

(BRIEF DESCRIPTION OF DRAWINGS)

[FIG. 1] A perspective view of a 90° rightward rotation type mixing element according to an embodiment of the present invention;

[FIG. 2] Similarly, a bottom view of a mixing element;

[FIG. 3] Similarly, a partly-enlarged perspective view of a mixing element;

[FIG. 4] A perspective view of a mixing element formed of rightward rotation type first blades and leftward rotation type second blades according to an embodiment of the present

invention;

【FIG. 5】 A perspective view of a 90° leftward rotation type mixing element according to an embodiment of the present invention;

【FIG. 6】 Similarly, a perspective view of a mixing element formed of leftward rotation type first blades and rightward rotation type second blades;

【FIG. 7】 An explanatory view showing a cross-section of rightward rotation type mixing according to an embodiment of the present invention;

【FIG. 8】 A perspective view of 15° rightward rotation type mixing elements according to an embodiment of the present invention;

【FIG. 9】 A perspective view of a mixing element wherein four mixing elements made of 15° rightward rotation type blades are stacked, according to an embodiment of the present invention;

【FIG. 10】 Similarly, a perspective view of a mixing element wherein three 30° rightward rotation type mixing elements stacked;

【FIG. 11】 Similarly, a perspective view of a mixing element wherein three 60° rightward rotation type mixing elements are stacked;

【FIG. 12】 Similarly, a perspective view of a mixing element

wherein three 90° rightward rotation type mixing elements are stacked;

【FIG. 13】 A schematic sectional side view of a static fluid mixer using a mixing element according to an embodiment of the present invention;

【FIG. 14】 Similarly, a schematic partly-sectional side view of a static fluid mixer;

【FIG. 15】 Similarly, a schematic partly-sectional side view of a static mixer;

【FIG. 16】 A schematic longitudinal section perspective view of a static fluid mixer according to the present invention;

【FIG. 17】 A schematic partly-longitudinal-sectional view showing an embodiment when a mixing element according to the present invention is applied to a distillation column method gas-liquid contact apparatus; and

【FIG. 18】 Similarly, a schematic partly-longitudinal-sectional view showing an embodiment when a mixing element according to the present invention is applied to an absorption column method gas-liquid contact apparatus.

(DESCRIPTION OF REFERENCE NUMERALS)

1, 10, 20, 29, 38, 47, 54a, 54b, 54c, 54d, 63, 64, 64a, 64b, 64c, 73, 73a, 73b, 73c, 82, 82a, 82b, 82c, 93, 94, 103, 113, 122, 123, 141a, 141b, 141c, 141d, 146a, 146b, 146c, 146d

: MIXING ELEMENT

2,11,21,30,39,48,55,65,74,83,125,134

: PASSAGE TUBE

3,12,22,49,56,66,75,84,96,104,114,126

: RIGHTWARD ROTATION TYPE FIRST BLADES

6,16,43,51,59,69,78,87,98,106,129

: RIGHTWARD ROTATION TYPE SECOND BLADES

31,40,135 : LEFTWARD ROTATION TYPE FIRST BLADES

25,34,116,138 : LEFTWARD ROTATION TYPE SECOND BLADES

5,14,24,33,42,50,58,68,77,86,97,105,115,128,137 : FIRST INNER  
CYLINDRICAL TUBE

8,18,27,36,45,52,61,71,80,89,99,107,117,131 : SECOND INNER  
CYLINDRICAL TUBE

4,7,13,15,17,19,23,26,32,35,41,44,57,60,67,70,76,79,85,88,127,  
130,132,136 : PERFORATIONS

9,28,37,46,53,62,72,81,90,100,108,118,133 : OPENING

91,101,111,121 : STATIC FLUID MIXER

92,102,112,140,145 : CASING

95,110,120,124 : SPACER

109,119 : SPACE PORTION

139: DISTILLATION TOWER

144: ABSORPTION TOWER

142,147: SUPPORTING TOOL

143,148: MANHOLE

(BEST MODE FOR CARRYING OUT THE INVENTION)

Hereinafter, embodiments of the present invention will be explained in detail, referring to the attached drawings. FIG. 1 is a perspective view of a 90° rightward rotation type mixing element according to a first embodiment of the present invention; FIG. 2 is a bottom view of a 90° rightward rotation type mixing element according to the first embodiment; FIG. 3 is a partly-enlarged perspective view of a 90° rightward rotation type mixing element according to a second embodiment; FIG. 4 is a perspective view of a mixing element formed of rightward rotation type first blades and leftward rotation type second

blades, according to a third embodiment of the present invention; similarly, FIG. 5 is a perspective view of a leftward rotation type mixing element according to a fourth embodiment; similarly, FIG. 6 is a perspective view of a mixing element  
5 formed of leftward rotation type first blades and rightward rotation type second blades, according to a fifth embodiment; FIG. 7 is an explanatory view showing a cross-section in the diameter direction of a rightward rotation type mixing element according to the first embodiment of the present invention;  
10 similarly, FIG. 8 is a perspective view of 15° rightward rotation type mixing elements according to a sixth embodiment; FIG. 9 is a perspective view of a mixing element in which four 15° rightward rotation type mixing elements of the sixth embodiment of the present invention are stacked; similarly, FIG.  
15 10 is a perspective view of a mixing element in which three 30° rightward rotation type mixing elements of a seventh embodiment are stacked; similarly, FIG. 11 is a perspective view of a mixing element in which three 60° rightward rotation type mixing elements of an eighth embodiment are stacked; similarly, FIG. 12  
20 is a perspective view of a mixing element in which three 90° rightward rotation type mixing elements of a ninth embodiment are stacked; FIG. 13 is a schematic sectional side view according to a first embodiment of a static fluid mixer using a mixing element of the present invention; FIG. 14 is a schematic

partly-sectional side view according to a second embodiment of a static fluid mixer using a mixing element of the present invention; similarly, FIG. 15 is a schematic partly-sectional side view according to a third embodiment of a static fluid mixer; FIG. 16 is a schematic longitudinally-sectional perspective view of the static fluid mixer according to the embodiment of the present invention shown in FIG. 13; FIG. 17 is a schematic partly-longitudinal sectional view showing an application example when a mixing element of the present invention is applied to a distillation column method gas-liquid contact apparatus; and similarly, FIG. 18 is a schematic partly-sectional side view showing an application example when a mixing element of the present invention is applied to an absorption column method gas-liquid contact apparatus.

(Embodiment 1)

FIG. 1 is a perspective view of a 90° rightward rotation type (in the clockwise direction) mixing element showing a first embodiment according to the present invention, and FIG. 2 is a bottom view of the mixing element. A mixing element 1 has a cylindrical passage tube 2, and a plurality of spiral rightward rotation type first blades 3 provided in the passage tube 2. These first blades 3 are formed of a perforated object having a large number of perforations 4. On the inside of the first blades 3 is disposed a first inner cylindrical tube 5 shaped

like a cylinder. The first inner cylindrical tube 5 is provided at a connecting portion of the first blades 3 by a necessary length in the axial center direction (in the longitudinal direction), and is not disposed in any other places. In the first inner cylindrical tube 5 are provided a plurality of spiral rightward rotation type second blades 6 formed of a perforated object having a large number of perforations 7. On the inside of the second blades 6 is disposed a second inner cylindrical tube 8 shaped like a cylinder to form an opening 9.

10 The second inner cylindrical tube 8 is provided to enhance mechanical strength against twisting stress of the second blades 6. The second inner cylindrical tube 8 is provided at a connecting portion of the second blades 6 by a necessary length, and is not disposed in any other places. Regarding each of the

15 first blades 3, one end thereof is connected to the outer circumferential surface of the first inner cylindrical tube 5 and becomes spirally twisted in a clockwise direction (rightward rotation) toward the inner circumferential surface of the passage tube 2, with another end thereof being connected to the

20 inner circumferential surface of the passage tube 2.

Similarly, regarding each of the second blades 6, one end thereof is connected to the outer circumferential surface of the second inner cylindrical tube 8 and becomes spirally twisted in a clockwise direction (rightward rotation) toward the inner

circumferential surface of the first inner cylindrical tube 5, with another end thereof being connected to the inner circumferential surface of the first inner cylindrical tube 5. Since the second inner cylindrical tube 8 has an opening at its  
5 central part, the second blades 6 are not provided in the axial center portion of the second inner cylindrical tube 8, creating a void. Thus, as shown in FIGS. 1 and 2, in the axial center portion of the second inner cylindrical tube 8 is formed the opening 9 without blades.

10        Rotation angles (twisting angles) of the blades 3 and 6 are not limited to  $90^\circ$  and are preferably in the range of approximately  $5^\circ$  to  $270^\circ$  in accordance to the internal diameter of the mixing element 1, more preferably in the range of approximately  $10^\circ$  to  $90^\circ$ . Further, the number of inner  
15 cylindrical tubes disposed is at least one and can accordingly be increased or decreased such as the third, fourth, fifth and n-th inner cylindrical tubes to be disposed, in accordance with the internal diameter of the mixing element 1 such that the diameter of the opening 9 becomes smallest, for example, 50mm or  
20 less. Similarly, blades can be used accordingly. Also, not limited to 12 or 6, the number of blades 3 and 6 provided can be increased or decreased accordingly.

(Embodiment 2)

FIG. 3 is a partly-enlarged perspective view of a  $90^\circ$

rightward rotation type mixing element showing a second embodiment according to the present invention.

Similarly to the mixing element 1 shown in FIGS. 1 and 2, a mixing element 10 has a cylindrical passage tube 11 and a plurality of spiral rightward rotation type first blades 12 provided in the passage tube 11. The blades 12 are formed of a perforated object having a large number of perforations 13. On the inside of the blades 12 is disposed a first inner cylindrical tube 14 shaped like a cylinder, and to the outer circumferential portion of the inner cylindrical tube 14 is connected one end of each of the blades 12. The inner cylindrical tube 14 is formed of a perforated object having a large number of perforations 15. In the inner cylindrical tube 14 are provided a plurality of spiral rightward rotation type second blades 16 formed of a perforated object having a large number of perforations 17. On the inside of the blades 16 is disposed a second inner cylindrical tube 18 shaped like a cylinder. The inner cylindrical tube 18 is formed of a perforated object having a large number of perforations 19.

By forming the first inner cylindrical tube 14 and the second inner cylindrical tube 18 of a perforated object having the large number of perforations 15 and 19, the mixture effectiveness of fluids flowing in the axial center direction (in the longitudinal direction) in the mixing element 10 is

improved. The shape of the perforations 15 and 19 is accordingly selected for use from a triangle, square, oval, slit and the like, as the need arises. Each ratio of the perforations 15 and 19 is accordingly selected for use in the  
5 range of approximately 5% to 95%.

(Embodiment 3)

FIG. 4 is a perspective view of a mixing element showing a third embodiment according to the present invention. A mixing element 20 has a cylindrical passage tube 21 and a plurality of  
10 spiral rightward rotation type first blades 22 provided in the passage tube 21. The blades 22 are formed of a perforated object having a large number of perforations 23. On the inside of the blades 22 is disposed a first inner cylindrical tube 24 shaped like a cylinder. In the inner cylindrical tube 24 are  
15 provided a plurality of spiral leftward rotation type second blades 25 formed of a perforated object having a large number of perforations 26. On the inside of the blades 25 is disposed a second inner cylindrical tube 27 shaped like a cylinder to form an opening 28.

20 Specifically, in the mixing element 20 are provided the first blades 22 of rightward rotation (in the clockwise direction) and the second blades 25 of leftward rotation (in the counterclockwise direction). Thus, regarding fluids of rightward rotation and leftward rotation flowing in the mixing

element 20, mixture efficiency is further improved, because strong shearing stress is generated by swirling currents opposite to each other in the diameter direction in the mixing element 20. In addition, by forming the inner cylindrical tubes 24 and 27 of a perforated object, mixture efficiency is further improved.

(Embodiment 4)

FIG. 5 is a perspective view of a 90° leftward rotation type (in the counterclockwise direction) mixing element showing a fourth embodiment according to the present invention. A mixing element 29 has a cylindrical passage tube 30 and a plurality of spiral leftward rotation type first blades 31 provided in the passage tube 30. The first blades 31 are formed of a perforated object having a large number of perforations 32. On the inside of the first blades 31 is disposed a first inner cylindrical tube 33 shaped like a cylinder. The first inner cylindrical tube 33 is provided at a connecting portion of the first blades 31 by a necessary length in the axial center direction (in the longitudinal direction), and is not disposed in any other places. In the first inner cylindrical tube 33 are provided a plurality of spiral leftward rotation type second blades 34 formed of a perforated object having a large number of perforations 35. On the inside of the second blades 34 is disposed a second inner cylindrical tube 36 shaped like a

cylinder to form an opening 37. Similarly to the one mentioned earlier, the second inner cylindrical tube 36 is provided to enhance mechanical strength against twisting stress of the second blades 34. The second inner cylindrical tube 36 is  
5 provided at a connecting portion of the second blades 34 by a necessary length according to the need, and is not disposed in any other places. Regarding the first blades 31, one end thereof is connected to the outer circumferential surface of the first inner cylindrical tube 33 and becomes spirally twisted in  
10 a counterclockwise direction (leftward rotation) toward the inner circumferential surface of the passage tube 30, with another end thereof being connected to the inner circumferential surface of the passage tube 30.

Similarly, regarding the second blades 34, one end thereof  
15 is connected to the outer circumferential surface of the second inner cylindrical tube 36 and becomes spirally twisted in a counterclockwise direction (leftward rotation) toward the inner circumferential surface of the first inner cylindrical tube 33, with another end thereof being connected to the inner  
20 circumferential surface of the first inner cylindrical tube 33. Since the second inner cylindrical tube 36 has an opening at its central part, the second blades 34 are not provided in the axial center portion of the second inner cylindrical tube 36 to create a void.

Similarly to the one mentioned earlier, rotation angles (twisting angles) of the blades 31 and 34 are not limited to  $90^\circ$  and are preferably in the range of approximately  $5^\circ$  to  $180^\circ$  in accordance with the internal diameter of the mixing element 29, and more preferably in the range of approximately  $10^\circ$  to  $90^\circ$ . Further, the number of inner cylindrical tubes disposed is at least one and can be accordingly increased or decreased, in accordance with the internal diameter of the mixing element 29. Further, the number of blades 31 and 34 provided is not limited to 12 and 6, and can accordingly be increased or decreased in the range where production is possible.

(Embodiment 5)

FIG. 6 is a perspective view of a mixing element showing a fifth embodiment according to the present invention. A mixing element 38 has a cylindrical passage tube 39 and a plurality of spiral leftward rotation type first blades 40 provided in the passage tube 39. The blades 40 are formed of a perforated object having a large number of perforations 41. On the inside of the blades 40 is disposed a first inner cylindrical tube 42 shaped like a cylinder. In the inner cylindrical tube 42 are provided a plurality of spiral rightward rotation type second blades 43 formed of a perforated object having a large number of perforations 44. On the inside of the blades 43 is disposed a second inner cylindrical tube 45 shaped like a cylinder to form

an opening 46.

Similarly to the one mentioned earlier, specifically, in the mixing element 38 are provided the blades 40 of leftward rotation (in the counterclockwise direction) and the blades 43 of rightward rotation (in the clockwise direction). Thus, regarding fluids of rightward rotation and leftward rotation flowing in the mixing element 38, mixture efficiency is further improved, because strong shearing stress is generated by swirling currents opposite to each other in the diameter direction in the mixing element 38. In addition, by forming the inner cylindrical tubes 42 and 45 of a perforated object, mixture efficiency is further improved.

FIG. 7 is an explanatory view relating to dimensions (lengths) regarding the diameters of a passage tube and an inner cylindrical tube in a mixing element according to the present invention. As described in FIGS. 1, 2 and 3, a mixing element 47 includes a passage tube 48, first blades 49, a first inner cylindrical tube 50, second blades 51 and a second inner cylindrical tube 52 to form an opening 53. Regarding the dimensional ratio of a passage tube to an inner cylindrical tube in the mixing element 47, if the diameter of the passage tube 48 is  $\phi D$  and the diameter of the inner cylindrical tube 50 is  $\phi d$ , it is preferable that  $\phi d$  be in the range of approximately 1% to 95% of  $\phi D$ , more preferably 10% to 60%. Further, it is

preferable that the diameter of the opening 53 be small, for example 50mm or less, and be in the range of approximately 5% to 50% of the diameter  $\phi d$  of the first inner cylindrical tube 50, more preferably in the range of approximately 10% to 30%. In addition, the dimensional ratio of a passage tube to an inner cylindrical tube is accordingly selected for use, in accordance with the dimensions of the passage tube. Further, not limited to first and second inner cylindrical tubes, inner cylindrical tubes can be disposed toward the central part of a passage tube in turn, for example, such as the third, fourth, fifth and n-th inner cylindrical tubes and accordingly selected for use, with blades disposed in a similar manner.

(Embodiment 6)

FIG. 8 is a perspective view of 15° rightward rotation type (in the clockwise direction) mixing elements showing a sixth embodiment according to the present invention. A mixing element 54a has a cylindrical passage tube 55 and a plurality of spiral rightward rotation type first blades 56 provided in the passage tube 55. The blades 56 are formed of a perforated object having a large number of perforations 57. On the inside of the blades 56 is disposed a first inner cylindrical tube 58 shaped like a cylinder. The inner cylindrical tube 58 is provided at a connecting portion of the first blades 56 by a necessary length in the axial center direction (in the longitudinal direction),

and not disposed in any other places. In the inner cylindrical tube 58 are provided a plurality of spiral rightward rotation type second blades 59 formed of a perforated object having a large number of perforations 60. On the inside of the blades 59 is disposed a second inner cylindrical tube 61 shaped like a cylinder to form an opening 62. The inner cylindrical tube 61 is provided to enhance mechanical strength against twisting stress of the blades 59. The second inner cylindrical tube 61 is provided at a connecting portion of the second blades 59 by a necessary length according to the need, and is not disposed in any other places. Regarding the first blades 56, one end thereof is connected to the outer circumferential surface of the first inner cylindrical tube 58 and becomes spirally twisted by approximately 15° in a clockwise direction (rightward rotation) toward the inner circumferential surface of the passage tube 55, with another end thereof being connected to the inner circumferential surface of the passage tube 55.

Similarly, regarding the second blades 59, one end thereof is connected to the outer circumferential surface of the second inner cylindrical tube 61 and becomes spirally twisted in a clockwise direction (rightward rotation) toward the inner circumferential surface of the first inner cylindrical tube 58, with another end thereof being connected to the inner circumferential surface of the first inner cylindrical tube 58.

Since the second inner cylindrical tube 61 has an opening at its central part, the second blades 59 are not provided in the axial center portion of the second inner cylindrical tube 61 to create a void. Thus, in the axial center portion of the second inner  
5 cylindrical tube 61 is formed the opening 62 without the blades. Mixing elements 54b, 54c, 54d are formed similarly to the mixing element 54a.

As regards the mixing element 54, since the rotation angles of the first blades 56 and the second blades 59 are  
10 approximately 15°, the number of blades 56 and 59 installed can easily be increased, so that mixture efficiency is further improved. Further, production of mixing elements having a large diameter (1000mm or more in diameter) is facilitated, and production costs can be reduced as well. Furthermore, the  
15 mixing element 54 can be disposed in a distillation column and an absorption column already existing, so that assembly and installation work on site and in columns can be made with ease. Regarding a production method of the mixing element 54, each of the passage tube 55, the blades 56, 59, and the inner  
20 cylindrical tubes 58, 61 is produced separately. Produced with a plurality of members which have been divided in at least two parts in the longitudinal direction, the cylindrical passage tube 55 and the inner cylindrical tubes 58, 61 may be formed by connecting the plurality of members divided. Similarly, the

spiral blades 56, 59 may be formed by connecting a plurality of members which have been divided in two or more parts. In addition, the mixing element 54 can be produced with ease by connecting the passage tube 55, the inner cylindrical tubes 58, 61, and the blades 56, 59 by means of welding, bonding, adhesion, locking and so forth.

Regarding a mixing element 63 shown in FIG. 9, the above-described four  $15^\circ$  rightward rotation type mixing elements 54a, 54b, 54c, 54d are stacked and connected such that the rotation angle (twisting angle) of the blades 56 becomes approximately  $60^\circ$ . In other words, by joining adjacent first blades 56, the mixing element 63 having such blades as  $15^\circ + 15^\circ + 15^\circ + 15^\circ = 60^\circ$  is formed.

By disposing a necessary number of mixing elements 54 on top of the other in this manner, a mixing element having an arbitrary rotation angle such as approximately  $180^\circ$ , approximately  $270^\circ$ , and approximately  $360^\circ$  can be produced with ease.

In addition, not necessarily joined together in a predetermined position, edges of blades 56 next to each other may be disposed in an arbitrary position and used. Further, mixing elements are not limited to rightward rotation type blades, and a combination of the rotation direction of blades forming the mixing elements 10, 20, 29, 38 shown in FIGS. 3

through 6 is accordingly selected for use, according to the need.  
(Embodiment 7)

FIG. 10 is a perspective view of a 30° rightward rotation type (in the clockwise direction) mixing element showing a  
5 seventh embodiment according to the present invention. Similarly to the mixing element shown in FIG. 8, a mixing element 64 has a cylindrical passage tube 65 and a plurality of spiral rightward rotation type first blades 66 provided in the passage tube 65. The first blades 66 are formed of a perforated  
10 object having a large number of perforations 67. On the inside of the first blades 66 is disposed a first inner cylindrical tube 68 shaped like a cylinder. The inner cylindrical tube 68 is provided at a connecting portion of the blades 66 by a necessary length in the axial center direction (in the  
15 longitudinal direction), and not disposed in any other places. In the inner cylindrical tube 68 are provided a plurality of spiral rightward rotation type second blades 69 formed of a perforated object having a large number of perforations 70. On the inside of the blades 69 is disposed a second inner  
20 cylindrical tube 71 shaped like a cylinder to form an opening 72. The inner cylindrical tube 71 is provided to enhance mechanical strength against twisting stress of the blades 69. The inner cylindrical tube 71 is provided at a connecting portion of the blades 69 by a necessary length according to the need, and not

disposed in any other places.

The followings are similar to the mixing element shown in FIG. 8 and therefore are not explained in detail.

As regards the mixing element 64 shown in FIG. 10, three  
5 approximately 30° rightward rotation type mixing elements 64a, 64b, 64c are stacked and connected such that the rotation angle of the blades 66 becomes approximately 90°. Similarly to the mixing element 63 shown in FIG. 9, the mixing element 64 having such blades as  $30^{\circ}+30^{\circ}+30^{\circ}=90^{\circ}$  is formed.

10 (Embodiment 8)

FIG. 11 is a perspective view of a 60° rightward rotation type (in the clockwise direction) mixing element showing an eighth embodiment according to the present invention. A mixing element 73 has a cylindrical passage tube 74 and a plurality of  
15 spiral rightward rotation type first blades 75 provided in the passage tube 74. The blades 75 are formed of a perforated object having a large number of perforations 76. On the inside of the blades 75 is disposed a first inner cylindrical tube 77 shaped like a cylinder. The inner cylindrical tube 77 is  
20 provided at a connecting portion of the blades 75 by a necessary length in the axial center direction (in the longitudinal direction), and not disposed in any other places. In the inner cylindrical tube 77 are provided a plurality of spiral rightward rotation type second blades 78 formed of a perforated object

having a large number of perforations 79. On the inside of the blades 78 is disposed a second inner cylindrical tube 80 shaped like a cylinder to form an opening 81. The inner cylindrical tube 80 is provided to enhance mechanical strength against twisting stress of the blades 78. The inner cylindrical tube 80 is provided at a connecting portion of the second blades 78 by a necessary length according to the need, and not disposed in any other places.

The followings are similar to the mixing element shown in FIG. 8 and therefore are not explained in detail.

As regards the mixing element 73 shown in FIG. 11, three approximately 60° rightward rotation type mixing elements 73a, 73b, 73c are stacked and connected such that the rotation angle of the blades 75 becomes approximately 180°. Similarly to the mixing element 63 shown in FIG. 9, the mixing element 73 having such blades as  $60^\circ + 60^\circ + 60^\circ = 180^\circ$  is formed.

(Embodiment 9)

FIG. 12 is a perspective view of a 90° rightward rotation type (in the clockwise direction) mixing element showing a ninth embodiment according to the present invention. A mixing element 82 has a cylindrical passage tube 83 and a plurality of spiral rightward rotation type first blades 84 provided in the passage tube 83. The blades 84 are formed of a perforated object having a large number of perforations 85. On the inside of the blades

84 is disposed a first inner cylindrical tube 86 shaped like a cylinder. The inner cylindrical tube 86 is provided at a connecting portion of the blades 84 by a necessary length in the axial center direction (in the longitudinal direction), and not disposed in any other places. In the inner cylindrical tube 86 are provided a plurality of spiral rightward rotation type second blades 87 formed of a perforated object having a large number of perforations 88. On the inside of the blades 87 is disposed a second inner cylindrical tube 89 shaped like a cylinder to form an opening 90. The inner cylindrical tube 89 is provided to enhance mechanical strength against twisting stress of the blades 87. The inner cylindrical tube 89 is provided at a connecting portion of the blades 87 by a necessary length according to the need, and not disposed in any other places. The followings are similar to the mixing element shown in FIG. 8 and therefore are not explained in detail.

As regards the mixing element 82 shown in FIG. 12, three approximately 90° rightward rotation type mixing elements 82a, 82b, 82c are stacked and connected such that the rotation angle of the blades 84 becomes approximately 270°. Similarly to the mixing element 63 shown in FIG. 9, the mixing element 82 having such blades as  $90^{\circ}+90^{\circ}+90^{\circ}=270^{\circ}$  is formed.

FIG. 13 is a schematic sectional side view of a static fluid mixer using a mixing element of the present invention, in which a rightward rotation type mixing element according to a first embodiment and a leftward rotation type mixing element

according to a fourth embodiment are vertically connected with a spacer in between. A cylindrical static fluid mixer 91 is formed by alternately disposing a rightward rotation type mixing element 93 and a leftward rotation type mixing element 94, with a spacer 95 of the same diameter as the mixing elements 93, 94 disposed in between, in a cylindrical casing 92. In addition, rightward rotation type second blades 98 are disposed all along rightward rotation type first blades 96. Further, the mixing elements 93, 94 incorporate a first inner cylindrical tube 97 and a second inner cylindrical tube 99 shown in FIGS. 1 and 5, respectively. It is preferable that an opening 100 be small in diameter (50mm or less in diameter). Note that, a static fluid mixer may be formed by alternately disposing the mixing elements 93, 94 in the casing 92 without disposing the cylindrical spacer 95. Further, a static fluid mixer may be formed by joining together the edges of the mixing elements 93, 94.

While two kinds of fluids FA and FB flow in the static fluid mixer 91 constructed as described above, part of the fluids revolves spirally in accordance with the rotation angles of blades, becoming circling currents in the clockwise direction, part of the fluids flows through perforations in blades and then is sheared, and part of the fluids flows through perforations in an inner cylindrical tube and then is sheared; and accordingly

the fluids flow together, and further, are turned over and divided. The two kinds of fluids FA and FB are mixed as revolution, passing, shearing, convergence, turnover and division are thus repeated in this manner.

FIG. 14 is a schematic partly-sectional side view of a static fluid mixer according to a second embodiment shown in FIG. 3 using a mixing element of the present invention. A cylindrical static fluid mixer 101 is formed by disposing a rightward rotation type mixing element 103 and a cylindrical spacer 110 having the same diameter as the mixing element 103 in a cylindrical casing 102. Rightward rotation type first blades 104 provided in the mixing element 103 are similar to the mixing element 93 shown in FIG. 13, whereas rightward rotation type second blades 106 are disposed at a necessary portion in the axial center direction (in the longitudinal direction) in a first inner cylindrical tube 105 with a space portion 109 disposed in between, and a second inner cylindrical tube 107 and an opening 108 are formed to constitute the mixing element 103. By forming the space portion 109, where no second blade 106 is provided, in the first inner cylindrical tube 105 in this manner, mixture efficiency can be improved with the effectiveness of fluids converged to flow together in the diameter direction.

FIG. 15 is a schematic partly-sectional side view of a static fluid mixer according to a third embodiment shown in FIG.

4 using a mixing element of the present invention. A cylindrical static fluid mixer 111 is formed by disposing a rightward rotation type mixing element 113 and a cylindrical spacer 120 having the same diameter as the mixing element 113 in a cylindrical casing 112. Rightward rotation type first blades 114 provided in the mixing element 113 are similar to the mixing element 93 shown in FIG. 13, whereas second blades 116 provided in a first inner cylindrical tube 115 are formed of the leftward rotation type. Further, similarly to the mixing element 103 shown in FIG. 14, with a space portion 119 in between, the leftward rotation type second blades 116 are disposed to form the mixing element 113. Further, similarly to FIG. 13, a second inner cylindrical tube 117 and an opening 118 are formed.

Regarding the static fluid mixer 111 constructed in this manner, mixture efficiency is further improved with the occurrence of circling currents of rightward rotation and leftward rotation.

FIG. 16 is a longitudinally-sectional schematic perspective view, in the axial center direction (in the longitudinal direction), of the static fluid mixer according to an embodiment of the present invention shown in FIG. 13. A cylindrical static fluid mixer 121 is formed by alternately disposing a cylindrical rightward rotation type mixing element 122 and a cylindrical

leftward rotation type mixing element 123, with a cylindrical spacer 124 in between.

The rightward rotation type mixing element 122 has a cylindrical passage tube 125 and a plurality of spiral rightward  
5 rotation type first blades 126 provided in the passage tube 125. The blades 126 are formed of a perforated object having a large number of perforations 127. On the inside of the blades 126 (at the central part), a first inner cylindrical tube 128 having perforations, shaped like a cylinder, is disposed all along the  
10 length of first blades 126. In the inner cylindrical tube 128 are provided a plurality of spiral rightward rotation type second blades 129 formed of a perforated object having a large number of perforations 130. On the inside of the blades 129 (at the central part) is disposed a second inner cylindrical tube  
15 131 shaped like a cylinder to form an opening 133. Similarly to the first inner cylindrical tube 128, the inner cylindrical tube 131 is provided with a large number of perforations 132. One end of the cylindrical spacer 124 which has the same diameter as the mixing element 122 is joined to an edge of the mixing  
20 element 122. It is preferable that the length in the axial center direction (in the longitudinal direction) of the spacer 124 be 0.1 to 10 times the whole length of the mixing element 122. Note that, not limited thereto, the whole length of the spacer 124 can be accordingly selected for use.

One end of the leftward rotation type mixing element 123 is joined to the other end of the spacer 124. The leftward rotation type mixing element 123, whose detailed explanation is omitted, has a cylindrical passage tube 134 and a plurality of spiral leftward rotation type first blades 135 provided in the passage tube 134, similarly to the above-described rightward rotation type mixing element 122. The blades 135 are formed of a perforated object having a large number of perforations 136. On the inside of the blades 135 (at the central part), a first inner cylindrical tube 137 having perforations, shaped like a cylinder, is disposed all along the length of the first blades 135, similarly to FIG. 3. In the inner cylindrical tube 137 are provided a plurality of spiral leftward rotation type second blades 138 formed of a perforated object having a large number of perforations. On the inside of the blades 138 (at the central part) is disposed a second inner cylindrical tube shaped like a cylinder to form an opening 133. Similarly to the first inner cylindrical tube 137, the inner cylindrical tube is provided with a large number of perforations. The other end of the leftward rotation type mixing element 123 is joined to a spacer 124 as described above, and further is joined to a rightward rotation type mixing element 122 as described above with the spacer disposed in between, to form the static fluid mixer 121. The static fluid mixer 121 includes two mixing

elements 122 and one mixing element 123; however, not limited thereto, a static fluid mixer may be formed by using at least one mixing element. The number of mixing elements disposed, rotation angles, rotation directions, and the number of blades disposed are accordingly selected for use.

(Application Example 1)

FIG. 17 is a schematic partly-longitudinally-sectional view showing an application example when a mixing element according to an embodiment of the present invention is applied to a distillation column method gas-liquid contact apparatus. A distillation column 139 has a cylindrical casing 140 and mixing elements 141a, 141b, 141c, 141d disposed in the casing 140. The mixing elements 141a, 141b, 141c, 141d are locked in a predetermined position by means of a mixing element supporting tool 142 provided in the casing 140. A manhole 143 is formed with such a structure and dimensions as allow members of mixing elements and workers to be carried into and out of the casing 140.

In the distillation column 139 thus constructed, an ascending gas (FA) and a descending liquid (FB) in the distillation column 139 flow as countercurrents in the mixing element 141, allowing the gas and the liquid to be agitated and mixed, and so the gas and liquid are sufficiently contacted. By applying the distillation column 139 to flash distillation, steam distillation and so forth, foreign bodies in liquid can be

separated, refined and collected.

By using a mixing element according to the present invention as packing in a distillation column, treatment with a gas velocity in the distillation column which is 1.5 to 5 times the velocity of conventional packing is made possible, so that equipment costs can be reduced. Further, improvement in gas-liquid contact efficiency makes the height of the column decrease, so that operation with low pressure loss is made possible, and the amount of steam supplied decreases. Further, operational management is made easy because of a wide operation control range. Furthermore, by replacing packing of a distillation column already installed, productivity can be improved with ease. Packing replacement work is made easily via a manhole. Further, in comparison with a distillation column using a conventional static mixer, the diameter of an opening of a mixing element can be made smallest (for example, 50mm or less), so that gas-liquid contact efficiency further improves, and production of a mixing element is facilitated to enable production and installation in a small distillation column. Moreover, production of a distillation column of a large diameter (1m or more) is facilitated, enabling large-capacity treatment.

(Application Example 2)

FIG. 18 is a schematic partly-longitudinally-sectional view

showing an application example when a mixing element according to an embodiment of the present invention is applied to an absorption column method gas-liquid contact apparatus. An absorption column 144 has a cylindrical casing 145 and mixing elements 146a, 146b, 146c, 146d disposed in the casing 145. The mixing elements 146a, 146b, 146c, 146d are locked in a predetermined position by means of a mixing element supporting tool 147 provided in the casing 145. A manhole 148 is formed with such a structure and dimensions as allow members of mixing elements to be carried into and out of the casing 145 and workers to go into and out of the casing 145.

In the absorption column 144 thus constructed, a descending gas (FA) and a descending liquid (FB) in the absorption column 144 flow concurrently in the mixing element 146, allowing the gas and the liquid to be agitated and mixed, and so gas and liquid are sufficiently contacted. By applying the absorption column 144 to gas absorption, gas cooling and dust removing operation, foreign bodies in the air can be separated, refined, collected and excluded.

By applying a mixing element according to the present invention to packing of an absorption column, treatment with a gas velocity in the absorption column, which is 1.5 to 10 times the velocity of conventional packing is made possible, so that equipment costs can be reduced. Further, improvement in gas-

liquid contact efficiency makes the height of the column and the diameter of column decrease, so that operation with low pressure loss is made possible, enabling less work space and less energy consumption. Further, operational management is made easy  
5 because of a wide operation control range. Furthermore, by replacing packing of an absorption column already installed, productivity can be improved with ease. Packing replacement work is made easily via a manhole. Further, in comparison with an absorption column using a conventional static mixer, the  
10 diameter of an opening of a mixing element can be made smallest, so that gas-liquid contact efficiency improves, and production of a mixing element of a large diameter (1m or more) is facilitated. Further, an absorption column of a large diameter (1m or more) can be easily produced, so that an absorption  
15 column which treats large wind amounts ( $30000\text{m}^3/\text{Hr}$  or more) can be made with low cost.

(BRIEF DESCRIPTION OF DRAWINGS)

【FIG. 1】 A perspective view of a  $90^\circ$  rightward rotation type mixing element according to an embodiment of the present  
20 invention;

【FIG. 2】 Similarly, a bottom view of a mixing element;

【FIG. 3】 Similarly, a partly-enlarged perspective view of a mixing element;

【FIG. 4】 A perspective view of a mixing element formed of

rightward rotation type first blades and leftward rotation type second blades according to an embodiment of the present invention;

5   【FIG. 5】 A perspective view of a 90° leftward rotation type mixing element according to an embodiment of the present invention;

    【FIG. 6】 Similarly, a perspective view of a mixing element formed of leftward rotation type first blades and rightward rotation type second blades;

10   【FIG. 7】 An explanatory view showing a cross-section of rightward rotation type mixing according to an embodiment of the present invention;

    【FIG. 8】 A perspective view of 15° rightward rotation type mixing elements according to an embodiment of the present  
15   invention;

    【FIG. 9】 A perspective view of a mixing element wherein four mixing elements made of 15° rightward rotation type blades are stacked, according to an embodiment of the present invention;

    【FIG. 10】 Similarly, a perspective view of a mixing element  
20   wherein three 30° rightward rotation type mixing elements stacked;

    【FIG. 11】 Similarly, a perspective view of a mixing element wherein three 60° rightward rotation type mixing elements are

stacked;

【FIG. 12】 Similarly, a perspective view of a mixing element wherein three 90° rightward rotation type mixing elements are stacked;

5 【FIG. 13】 A schematic sectional side view of a static fluid mixer using a mixing element according to an embodiment of the present invention;

【FIG. 14】 Similarly, a schematic partly-sectional side view of a static fluid mixer;

10 【FIG. 15】 Similarly, a schematic partly-sectional side view of a static mixer;

【FIG. 16】 A schematic longitudinal section perspective view of a static fluid mixer according to the present invention;

15 【FIG. 17】 A schematic partly-longitudinal-sectional view showing an embodiment when a mixing element according to the present invention is applied to a distillation column method gas-liquid contact apparatus; and

20 【FIG. 18】 Similarly, a schematic partly-longitudinal-sectional view showing an embodiment when a mixing element according to the present invention is applied to an absorption column method gas-liquid contact apparatus.

(DESCRIPTION OF REFERENCE NUMERALS)

1, 10, 20, 29, 38, 47, 54a, 54b, 54c, 54d, 63, 64, 64a, 64b, 64c, 73, 73a, 73b, 73

c, 82, 82a, 82b, 82c, 93, 94, 103, 113, 122, 123, 141a, 141b, 141c, 141d, 146a, 146b, 146c, 146d

: MIXING ELEMENT

2, 11, 21, 30, 39, 48, 55, 65, 74, 83, 125, 134

5 : PASSAGE TUBE

3, 12, 22, 49, 56, 66, 75, 84, 96, 104, 114, 126

: RIGHTWARD ROTATION TYPE FIRST BLADES

6, 16, 43, 51, 59, 69, 78, 87, 98, 106, 129

: RIGHTWARD ROTATION TYPE SECOND BLADES

10 31, 40, 135 : LEFTWARD ROTATION TYPE FIRST BLADES

25, 34, 116, 138 : LEFTWARD ROTATION TYPE SECOND BLADES

5, 14, 24, 33, 42, 50, 58, 68, 77, 86, 97, 105, 115, 128, 137 : FIRST INNER CYLINDRICAL TUBE

8, 18, 27, 36, 45, 52, 61, 71, 80, 89, 99, 107, 117, 131 : SECOND INNER

15 CYLINDRICAL TUBE

4, 7, 13, 15, 17, 19, 23, 26, 32, 35, 41, 44, 57, 60, 67, 70, 76, 79, 85, 88, 127, 130, 132, 136 : PERFORATIONS

9, 28, 37, 46, 53, 62, 72, 81, 90, 100, 108, 118, 133 : OPENING

91, 101, 111, 121 : STATIC FLUID MIXER

20 92, 102, 112, 140, 145 : CASING

95, 110, 120, 124 : SPACER

109, 119 : SPACE PORTION

139: DISTILLATION TOWER

144: ABSORPTION TOWER

142,147: SUPPORTING TOOL

143,148: MANHOLE

## CLAIMS

1. A mixing element comprising: a cylindrical passage tube through which fluid flows; spiral first blades of rightward rotation or leftward rotation provided in said passage tube; a first inner cylindrical tube disposed in the axial center portion of said first blades; spiral second blades of rightward rotation or leftward rotation provided in said first inner cylindrical tube; and a second inner cylindrical tube disposed in the axial center portion of said second blades, wherein

said first blades and said second blades are formed of a perforated object or a porous object.

2. A mixing element according to claim 1, wherein the rotation angle of said first blades is in the range of approximately 5° to 270°.

3. A mixing element according to any one of claims 1 or 2, wherein said first inner cylindrical tube and said second inner cylindrical tube are formed of a perforated object or a porous object.

4. A mixing element according to any one of claims 1 to 3,

wherein said first inner cylindrical tube is disposed along all or part of the length of said first blades in the axial center direction thereof.

5. A mixing element according to claim 1, wherein the rotation angle of said second blades is in the range of approximately 5° to 270°.

6. A mixing element according to any one of claims 1 to 5, wherein said second inner cylindrical tube is disposed along all or part of the length of said second blades in the axial center direction thereof.

7. A mixing element according to any one of claims 1 to 6, wherein said second inner cylindrical tube has an opening in the axial center portion thereof.

8. A mixing element according to any one of claims 1 to 7, wherein said mixing element is formed of rightward rotation type first blades and leftward rotation type second blades.

9. A mixing element according to any one of claims 1 to 7, wherein said mixing element is formed of leftward rotation type first blades and rightward rotation type second blades.

10. A mixing element according to any one of claims 1 to 9, wherein four of said mixing elements of approximately  $15^\circ$  rightward rotation or leftward rotation are joined together to obtain said mixing element having a rotation angle of approximately  $60^\circ$ .

11. A mixing element according to any one of claims 1 to 9, wherein three of said mixing elements of approximately  $30^\circ$  rightward rotation or leftward rotation are joined together to obtain said mixing element having a rotation angle of approximately  $90^\circ$ .

12. A mixing element according to any one of claims 1 to 9, wherein three of said mixing elements of approximately  $60^\circ$  rightward rotation or leftward rotation are joined together to obtain said mixing element having a rotation angle of approximately  $180^\circ$ .

13. A mixing element according to any one of claims 1 to 9,

wherein three of said mixing elements of approximately  $90^\circ$  rightward rotation or leftward rotation are joined together to obtain said mixing element having a rotation angle of approximately  $270^\circ$ .

14. A static fluid mixer comprising one or more said mixing elements according to any one of claims 1 to 13.

15. A static fluid mixer according to claim 14, wherein said mixing elements of rightward rotation and leftward rotation are alternately disposed in a cylindrical casing with spacers in between.

16. A gas-liquid contact apparatus comprising one or more said mixing elements according to any one of claims 1 to 13 disposed in a distillation column method gas-liquid contact apparatus through which fluids flow as countercurrents.

17. A gas-liquid contact apparatus comprising one or more said mixing elements according to any one of claims 1 to 13 disposed in an absorption column method gas-liquid contact apparatus through which liquids flow concurrently.