

-1-

FLEXIBLE DRIVE TRANSMISSION

The invention is in the field of drive technology and relates to an apparatus for transmitting a rotary movement with at least one drive and at least one power take-off and flexible chain guides arranged between the same.

Rotary drives are known, in which the transmission means, e.g. a rigid shaft, a chord or the like twists, or e.g. in the case of belts, chains, bands and the like is stretched by tensile forces. Each of these drive types has its own intrinsic advantages and disadvantages with respect to the drive transmission and these are also known. The rigid shaft is mainly suitable for force transmission over longer distances and it must be borne in mind that in the case of considerable loading, there can be a disturbing twisting effect. Chords in the sense of flexible shafts or chains are used for transmission over shorter distances. When using e.g. chains a "rigid" transmission is expected and when using chords a flexible transmission, the latter only being suitable for the transmission of small torques due to the high torsion angle on loading.

For example, U.S. Patent 3,718,051 discloses a force transmission system, which transmits a rotary movement between a drive and a power take-off by means of flexible chain guides. The transmission means is constituted by a plurality of loosely connected balls, which are pushed through the chain

channels. In order to obtain a relatively uniform force or power transmission, the chain guide in the pushing drum must be very rigid, because with flexible walling, the balls, which are in
5 contact with one another by their rounded portions are positionally unstable in pushing operation and must be positioned and stabilized by the channel walls. However, in the pulling drum, in which during the "return" of the balls
10 substantially no forces act and certainly no forces resulting from the transmission to the balls, the elasticity or rather the stability of the chain guide is of minor significance. Thus, very narrow limits are applied to the flexibility
15 of the described force transmission by means of balls. A uniform force transmission is not readily possible in the flexible construction.

The object of the invention is to provide an apparatus permitting rotary force transmission
20 by means of flexible connections between the drive and the power take-off in substantially jerk-free and axially synchronous operation by means of a random number of revolutions or over a random sum of angular degrees of both signs, in
25 the case of alternating loading and alternating rotation direction.

~~This object is achieved by the invention as defined in the claims.~~

~~For the discussions of the invention,~~
30 ~~reference is made to the Applicant's Swiss Patent~~

14

- 2a -

According to the present invention there is provided an apparatus for transmitting a rotary movement with at least one drive unit and at least one power take-off unit said units having chain channels therein, and
5 flexible and elastic chain guides arranged between said units, said chain guides and channels together comprising an endless chain casing, wherein the force transmission is by means of an endless chain in said casing, which can operate simultaneously to transmit force in push and pull
10 manner, said chain having a length corresponding essentially to the total length of the chain casing when in the inoperative state, such that in ^{use} ~~using~~ said chain guides respectively expand and contract according to the mode of force transmission of the chain within them while the total
15 chain casing length remains the same.

The chain is preferably a "3D-chain" that is one flexible in any direction and is so referred to herein.

For the discussions of the invention,
reference is made to the Applicant's Swiss Patent

621,183, which describes a chain drive with an endless roller chain. This chain drive is used as a conveying chain, preferably in conjunction with newspaper conveyors, the force action being
5 taken from the chain by means of slots in the rigid chain channel. Certain advantages of this chain drive and in particular the jerk-free, smooth running of the chain in the rigid guidance channels are of assistance in achieving a
10 qualitatively satisfactory realisation of the present invention.

The invention is described in greater detail hereinafter relative to non-limitative embodiments and the attached drawings, wherein
15 show:

Fig 1 a simplified view of a first embodiment of the invention and consequently simultaneously the principle thereof.

20 Figs 2a to c modifications of the invention, as shown in Fig 1.

Fig 3 a basic view of a randomly selectable multiple-axle drive formed from the components of the embodiment according to Fig 1.

25 Fig 4 a power branching by a drive with several power take-offs.

Fig 5 a detail of the embodiment according to Fig 1.

Fig 6 another detail of the embodiment according to Fig 1.

Fig 7 another embodiment of the invention.

30 The principle of the invention and at the same

16

time a first embodiment thereof is shown in
Fig 1. A is a drive rotating in the clockwise
direction and B is a power take-off rotating in
the same direction. The drive and power take-off
5 are operatively interconnected by means of an
open-link chain, preferably a 3D-chain 6, as is
fundamentally described in the Applicant's Swiss
Patent 538,065. Constructional details will be
considered hereinafter in conjunction with other
10 drawings and the first drawings are merely
intended to illustrate the principle and use.

The drive and the power take-off are in
each case housed in a guide casing 1, 1', in which
are located the sprockets 2, 2' driving or driven by
15 means of the force transmission shafts 3, 3'. The
guide casing 1, 1' is preferably constructed in
symmetrical, two-shell manner, so that the drives
and power take-offs can be randomly assembled in
unit construction-like manner.

20 Chain 6 runs outside the guide casing 1, 1'
through a flexible chain channel 7, which can
e.g. comprise a hose or a flexible tube and can
be bent together with the chain in any random
spatial direction. As a function of the chain
25 size, i.e. the diameter of the chain elements,
it is possible to use radii or curvature within
a wide range, so that apart from random spacings
between the drive and the power take-off, it is
possible to overcome virtually any intermediate
30 topography. The flexible chain channel 7 is, for

example, connected by means of a connecting piece 5 to the guide casing 1, 1'. However, the chain channel can also be fitted into the guide casing in some other way. For the purpose of guiding the open-link chain or 3D-chain 6 in the guide casing 1, 1' is incorporated a chain channel 4, 4' connected in shapewise manner with sprocket 2, 2' and in which the 3D-chain 6 can freely run.

If the length of the complete chain channel is made as close as possible to the nominal length of the endless chain, advantage can be directly taken of the resulting excellent running characteristics, as described in Swiss Patent 621,183. It is obvious that the chain can be operated forwards and backwards in both directions. If the open-link chain is operated clockwise by drive A in the mannershown in this embodiment, the endless chain is loaded in chain guide 7 in pushing direction S and in the other return chain guide 7' in pulling direction Z. The reason for this advantageous and also desired behaviour will now be briefly described.

The total length of the chain channel or chain casing, i.e. the length of chain channels 4, 4' in guide casings 1, 1' of drive A and power take-off B and the chain guide 7, 7' between the same, is defined in the non-operational state. The total length of the chain running therein must essentially correspond to the total length of the chain channel in the non-operative state.

-6-

In dynamic operation with a flexible chain guide, which necessarily must also have a flexibility in the longitudinal direction, these geometrical conditions are influenced in varying manner. There is a relatively limited longitudinal extension in the pushing drum and a roughly identical limited compression in the pulling drum in the running direction, corresponding to the elasticity behaviour of the flexible chain guides, the relative chain speed, the friction behaviour between chain and guide and the torque to be transmitted. Thus, in operation, the chain guides have a varying length in the pushing drum and the pulling drum. However, the nominal length of the chain remains the same and the total length of the chain channel remains substantially the same as the chain length or the length of the non-operative state but, as stated, the proportions between the flexible chain guides transmitting the push and pull forces differ as a function of the operating case.

If the nominal length of the chain is made longer than the total length of the chain casing in the non-operative state, then the chain guide of the pulling drum also starts to expand, because through the pushing of the chain into the flexible chain guide of the pulling drum part of the pulling action is lost. The opposite takes place with a nominal length of the chain, which is less than the total length of the chain

casing. The simultaneous utilization of the pushing and pulling forces during transmission can only take place in the tension-free state within relatively small divergences of the chain length from the chain casing length. The optimum situation is that of identical lengths and in practice substantially identical lengths can be obtained.

The force or power of the transmission from the drive to the power take-off is therefore also substantially halved into pulling and pushing force. A non-force-transmitting zone, such as e.g. with a flexible drive by loosely connected balls in the pulling drum, does not occur with the force transmission according to the invention.

In the case of the rigid chain guide of the open-link chain, as described e.g. in Swiss Patent 621,183, the subdivision of the forces into simultaneously acting pushing and pulling forces does not occur, because the rigid chain guide, unlike the flexible chain guide, in the case of alternating loading and particularly alternating operation, cannot adapt to the peristaltic characteristics of the chain. The resulting characteristics of the chain and chain guide, as result from the teaching according to the invention, lead to a very high synchronism at any random angular position of the drive and power take-off during force transmission under alternating load or in the case of torque changes

in a relatively wide range and in alternating operation.

In the drive according to the invention, it is also possible to frequently replace several expensive mitre gears and universal shafts. Apart from the reduced costs, the costs for servicing such gears and shafts are also avoided.

Figs 2a to 2c show in simple symbolic form modifications to the invention and how the possible uses of the invention can be varied by making simple modifications. The force transmission of drive A in Fig 2a is geared down by a larger sprocket diameter or a large number of teeth on power take-off B, whilst in the reverse case gearing up of the drive is possible. One of the possibilities of flexible drive transmission by means of flexible chain channels is shown in Fig 2b, in which the drive and power take-off are pivoted with respect to one another. Thus, the drive can be installed in virtually all the layers so that difficultly accessible drives can be constructed. Gearing up or gearing down are both possible here. On pivoting by 180° in the manner shown in Fig 2c, a torque reversal can be realised. As in the basic construction according to Fig 1, in all these modifications, force can be transmitted by both chain strands, one transmitting pulling forces and the other pushing forces, which leads to a force balancing effect.

Whereas in the case of the modification

according to Fig 4, force is transmitted parallel from the drive block or power take-off block to the individual components, Fig 3 shows the possibility of a serial force or power transmission to e.g. a drive I and a power take-off II, either in the same rotation direction and/or in opposite rotation directions, as a function of the reciprocal crossing of one or more power take-offs.

10 This makes it possible to solve very varied spatial drive problems. Another advantage of the invention is that the power take-off is movable during power transmission. This is particularly useful in the embodiment with twin hoses as
15 flexible chain guides and which will be discussed hereinafter.

Fig 4 shows a further modification of the invention, in which a multiple drive or power take-off is obtained by combining several drive
20 components A, as is e.g. shown in Fig 1. By means of a common central force transmission shaft 10, Ax drives are operated together, which e.g. drive the two power take-offs B1 and Bx shown in Fig 4 by means of the chains running
25 in the flexible chain channels 7, 7'. In the same way, the power take-offs can be randomly pivoted in spatial manner with respect to the drive block. A power take-off block with several components can just as variably be driven by
30 means of a single drive.

-10-

Fig 5 shows a detail of the embodiment according to Fig 1, which can also be used for various aforementioned modifications, such as gearing up and down, drive or power take-off blocks, rotation direction reversal, etc. The following constructional details relate to the preferred use of a 3D-chain, as is described in principle by the Applicant's Swiss Patent 538,065. What is shown is part of the flexible drive, as represented in Fig 1, namely the drive-side part A. Casing 1 comprises two halves 1a, 1b, in each case with incorporated sprocket casing halves and part of the chain channel 4 which can be particularly clearly seen in Fig 6. Sprocket 2 is peripherally provided with depressions 12 for the partial reception of chain elements 10, which engage with sprocket 2 with depth d. Depth d is selected in such a way that the outside diameter R of sprocket 2 does not reach the radius of curvature R' of chain 6, i.e. $R' - R \approx 0$ always applies and is shown in Fig 6 by the angle $\alpha \approx 180^\circ$. The effect of this measure is described in detail in the Applicant's Swiss Patent application 6620/81-6.

Fig 6 shows a further detail of the construction of a drive according to the invention. In order to achieve vibration free running and a sufficiently long operating life, a low-friction or preferably a friction-free

operation is sought. As can be gathered from Fig 5, a connecting piece 5 is arranged between casing 1 and flexible chain guide 7, the latter being e.g. a high pressure hose, with respect to the chain channel 4 in casing 1, that sprocket 2 can centre itself in the guide casing 1, accompanied by the formation of gaps 15 (Fig 6). The unavoidable friction of the 3D-chain in chain channel 4 is minimized by a lubricant in the closed system.

Fig 7 finally shows an embodiment of the invention, in which the flexible chain guidance parts 7, 7' run in a twin hose with a partition 15 between or from and to the drive and power take-off in each case. Endless chain 6 runs in the inner areas of the twin hose provided for the chain channels. When using thin chains, this variant is equivalent to a shaft drive and can be operated as such, without said drive having the often undesired torsional characteristics of a shaft drive. This permits equally large spacings between the drive and power take-off. The freedom to pass the drive anywhere round the ambient space is very advantageous. Large torques can be removed with respect to the power take-off immediately following the starting up of the drive, the transmission behaving in movement-rigid manner and consequently synchronously.

The invention therefore relates to an apparatus for transmitting a rotary movement with at least one drive and at least one power take-off

-12-

with flexible chain guides arranged between the same and which is characterized in that the force transmission takes place by means of an endless chain 6, operable in push and pull manner and

5 corresponding substantially to the complete length of the chain casing in the inoperative state, i.e. the length of the chain channels (4, 4') in guide casings (1, 1') of the drive (A) and power take-off (B) and the flexible chain

10 guides (7, 7') between the same. A serial interconnection of a plurality of power take-offs (BI, BII....BXX) drivable by a drive (A) by means of flexible chain guides (7', 7'', etc) constitutes a first embodiment of the invention and a parallel

15 connection of drives (Ax) driven by a common shaft (10'), with power take-offs (B1...Bx) connected to each drive (A1...Ax) by means of endless chains and flexible chain guides (7, 7') constitutes a second embodiment of the invention.

20 Further embodiments are that the flexible chain guides (7, 7') comprise a hose bendable in all spatial directions, whose inner space is used as the chain channel (4); that the flexible chain guide part (7) running from drive (A) to power

25 take-off (B) and the flexible chain guide parts (7') running from power take-off (B) to drive (A) comprises a twin hose bendable in all spatial directions, whose two inner spaces are used as the chain channel (4). Other obvious embodiments which

30 can be derived from the invention more particularly

(25)
-13-

relate to the construction of the preferably
two-shell drives and/or power take-offs (A, B, C)
and the flexible chain guide parts (7, 7').



26

- 14 -

CLAIMS

1. Apparatus for transmitting a rotary movement with at least one drive unit and at least one power take-off unit said units having chain channels therein, and
5 flexible and elastic chain guides arranged between said units, said chain guides and channels together comprising an endless chain casing, wherein the force transmission is by means of an endless chain in said casing, which can
10 operate simultaneously to transmit force in push and pull manner, said chain having a length corresponding essentially to the total length of the chain casing when in the inoperative state, such that in ^{use} ~~using~~ said chain guides respectively expand and contract according to the mode of force transmission of the chain within them while the total
15 chain casing length remains the same.

2. An apparatus according to claim 1, wherein there is a serial interconnection of a plurality of power ^{units} take-offs, which can be driven by a drive ^{unit} ~~by~~ means of flexible chain guides.

20 3. An apparatus according to claim 1, wherein there is a parallel connection of drives ^{units} ~~drives~~ driven by means of a common shaft, with power take-offs ^{units} ~~connected~~ to each ^{unit} drive ~~by~~ means of endless chains and flexible chain guides.

4. An apparatus according to claims 1, 2 or 3,
25 wherein the flexible chain guide comprises a hose bendable in all spatial directions and whose inner space is used as the chain channel.

(27)



-15-

5. An apparatus according to claims 1, 3 or 4, wherein the flexible chain guide part running from the drive^{unit} to the power take-off^{unit} and the flexible chain guide part running from the power take-off^{unit} to the drive^{unit} comprise a twin hose bendable in all spatial directions and whose two inner spaces are used as chain channels.
6. Apparatus for transmitting a rotary movement substantially as hereinbefore described with reference to and as illustrated in the accompanying drawings.
-