

METHOD FOR CONTROLLING A PUMP ARRANGEMENT

Technical field of the Invention

The present invention relates in general to a method for controlling a pump arrangement comprising a pump and a control unit, the pump comprising a motor, and the control unit being arranged to drive said motor. Especially the present invention relates to a method for controlling a pump arrangement, the motor during operation being associated with an operational parameter from which the torque of the motor may be derived, said operational parameter has a normal value P_N during normal operation of the motor in a first direction,

Background of the Invention and prior art

Upon pumping of a liquid, such as waste water comprising solid matter, by means of for instance a submersible pump, the solid matter will sooner or later have a negative influence on the capacity of the pump to transport the liquid. The solid matter stick fast in the hydraulic unit of the pump and attach slowly to the impeller of the pump as well as to the inner side of the pump house of the pump, and thereby the hydraulic efficiency of the pump is effected negatively and the pump will operate in a strained operation condition due to the increased rotary resistance, increased torque and deteriorated hydraulic properties. Today several known ways of more or less automatic cleaning of a pump when the pump, or more precisely the hydraulic unit of the pump, starts to clog. The strained operation condition is not directly detrimental for the pump, however an increased power consumption and worse pump performance is obtained, which is costly for the plant owner and which may result in negatively attendant effects such as flooded pump station when the accessible capacity of the pump is not enough to empty the pump station.

Known cleaning methods, or methods for controlling a pump arrangement, detects that cleaning is necessary and thereafter perform a predetermined standard cleaning sequence, that at least entail that the motor of the pump is decelerated by having the speed of the motor undergo a long predetermined

ramping down driven by the control unit. It is known that one shall not stop the motor of the pump directly, especially due to requirements of avoiding so-called water hammer in the pipe system downstream the pump, but also due to the high torque and the big momentum that the impeller of the pump have during normal operation. If the motor is stopped directly water hammering will inevitable arise when the kinetic energy and the moment of inertia of the liquid in the pipes downstream the pump cause vibrations that risk to destroy the pipes and other construction elements, thereto it is an imminent risk that the impeller is loosen, the drive shaft of the pump is damaged, etc. Thus, a long and controlled ramping down of the speed of the motor always takes place.

A direct consequence of the absence of intelligence in the cleaning method is that the standard cleaning sequence used, and which is adequate during a strained operation condition as described above, drastically increase the load of the pump when a large and/or hard object enters the hydraulic unit of the pump and is wedged, i.e. when a detrimental operation condition for the pump arrangement has arisen. A detrimental operation condition mean an operation condition that immediately or in a short period of time will entail that the pump and/or the control unit will break. When the control unit, for instance in the form of a frequency converter (VFD), perform said ramping down when a large and/or hard object has become wedged and mechanically brake the impeller, the long and controlled ramping down of the motor force the impeller to rotate and the object is wedged harder/more severe. This entail in its turn that the impeller, drive shaft motor, etc. of the pump or the control unit will become overstrained and damaged.

In order to prevent the pump and/or the control unit to become damaged different safety systems/protective equipment, such as a safety disconnection breaker, fuses, etc., which are arranged to protect the equipment and trig before the equipment is damaged. Common for the above described detrimental operation conditions, i.e. if the safety system triggers and/or the pump arrangement break, service personal

must perform an emergency turnout and take care of the fault/clogging. These turnouts are expensive and thereto an idle pump is costly for the plant owner.

5 Brief description of the object of the Invention

Embodiments of the present invention seek to obviate the above mentioned drawbacks and failings of previously known cleaning methods and at providing an improved method for controlling a pump arrangement. Embodiments of the invention
10 seek to provide an improved method for controlling a pump arrangement according to the initially defined type, which prominently may increase the number of clogging that the pump arrangement may solve by its own.

Embodiments of the present invention also seek to provide
15 a method for controlling a pump arrangement, which may prevent the need for service personnel to perform emergency turnouts.

Brief description of the features of the Invention

According to the invention at least the basic object of
20 the invention will be achieved by the initially defined method having the features defined by the independent claims. Preferable embodiments of the present invention are further defined in the dependent claims.

According to an aspect of the present invention, it is
25 provided a method for controlling a pump arrangement upon clogging of a pump, the pump arrangement comprising a pump and a control unit, the pump comprising a motor, and the control unit being arranged to drive said motor, the motor during operation being associated with an operational parameter from
30 which the torque of the motor may be derived, said operational parameter having a normal value P_N during normal operation of the motor in a first direction, the method comprising the steps of:

- driving the motor in a first direction by means of the control unit, and if a real value P of the operational parameter exceeds a predetermined clogging limit P_I :
- stopping the motor ,
- 5 - driving the motor in a the first direction opposite second direction a predetermined flushing time T_R by means of the control unit, and
- stopping the motor if the absolute value of the real value P of the operational parameter during the flushing time T_R
- 10 exceeds

the absolute value of a first unfastening limit P_{L1} , otherwise stopping the motor after said flushing time T_R and returning to normal operation, the method being characterized in that $P_I \geq 1,05 \cdot P_N$ and $|P_{L1}| \geq 1,1 \cdot P_I$, .

5 Thus, embodiments of the present invention are based on the understanding that by stopping the driving of the motor in the first direction at a lower torque than the torque at which the driving of the motor in the second direction is stopped, i.e. to have a greater torque available for unfastening of the
10 wedged material than the torque that wedged the material, the pump arrangement may be spared and the number of emergency turnouts may be more or less entirely eliminated.

 According to a preferred embodiment of the present invention, the method after the step of stopping the motor if
15 a real value P of the operational parameter exceed a pre-determined clogging limit P_I , where $P_I \geq 1,05 \cdot P_N$, thereto comprises the steps of:

- driving the motor in the first direction during a pre-determined control time T_K by means of the control unit,
- 20 - stopping the motor if the real value P of the operational parameter during the control time T_K exceed a false alarm control limit P_F , where $P_F \leq P_I$.

 Thereby a false alarm function is achieved whereupon unnecessary operation of the pump backwards may be avoided.

25 According to a preferred embodiment of the present invention, the method after the step of stopping the motor if the absolute value of the real value P of the operational parameter exceed the absolute value of a first unfastening limit P_{L1} , where $|P_{L1}| \geq 1,1 \cdot P_I$, thereto comprises the steps
30 of:

- driving the motor in the first direction during a pre-determined flushing time T_R by means of the control unit,
- stopping the motor if the real value P of the operational parameter exceed a second unfastening limit P_{L2} , where $P_{L2} \geq P_I$
35 and $P_{L2} \leq 0,95 \cdot |P_{L1}|$.

 Thereby the pump arrangement tries, when it has failed in the first unfastening attempt backwards, to unfasten the wedged material by means of an unfastening attempt forwards using an available torque that is greater than the available

torque during normal operation forwards but less than the available torque during unfastening backwards.

According to a preferred embodiment of the present invention, the operational parameter is constituted by the power consumption of the motor.

Other advantages and features of the invention are evident from the other dependent claims and from the following detailed description of preferred embodiments.

Brief description of the drawings

A more complete understanding of the above mentioned and other features of the present invention will be evident from the following detailed description of preferred embodiments having reference to the appended drawings, in which:

Fig. 1 is a schematic illustration of a pump station comprising a pump arrangement,

Fig. 2 is a flowchart disclosing a first embodiment of the inventive method,

Fig. 3 is a flowchart disclosing a second embodiment of the inventive method,

Fig. 4 is a flowchart disclosing a third embodiment of the inventive method,

Fig. 5 is a diagram that schematically disclose how the power consumption of the pump is altered over time, during a successful cleaning/unfastening in the second direction,

Fig. 6 is a diagram that schematically disclose how the power consumption of the pump is altered over time, during a successful unfastening in the second direction after several unsuccessful unfastening attempts, and

Fig. 7 is a diagram that schematically discloses how the power consumption of the pump is altered over time, during a false clogging.

Detailed description of preferred embodiments

In figure 1 is shown a pump station, generally designated 1, comprising at least one speed controlled pump 2, usually two submersible pumps, arranged in an active state to pump liquid from a sump 3 comprised in the pump station 1 to a outlet pipe 4 and further away from the pump station 1.

Thereto the pump station 1 comprises in a conventional way at least one level sensor 5 arranged to determine the liquid level in the pump station 1. It shall be pointed out that the level sensor 5 may be a separate device that is operatively connected to an external control unit 6, operatively connected to said at least one speed controlled pump 2, be built-in in said at least one speed controlled pump 2, etc. Said at least one speed controlled pump 2 is preferably operatively connected to the external control unit 6 in order to admit control of the speed of the pump, alternatively said at least one speed controlled pump 2 may comprise a built-in control unit (not shown). Hereinbelow the term control unit 6 will be used independently of its physical location.

The pump 2 and the control unit 6 together constitute at least a part of the pump arrangement, in which the pump 2 comprises an electrical motor 7 that is arranged to be driven by said control unit 6, and an impeller 8 that is connected with the motor 7 via a drive shaft 9 in a conventional way.

Preferably the impeller 8 is an open impeller, and most preferably an impeller that is axially displaceable in the pump 2, in relation to a suction lid/insert ring at the inlet of the pump, during operation.

By the term "speed controlled" all conceivable ways of altering the speed of the pump, or more precisely the rotational speed/operational speed of the motor 7, are covered. Above all current supply frequency control by means of a frequency converter (Variable Frequency Drive) is concerned, which is built-in in a pump or external, and which constitute an example on said control unit 6, the rotational speed being proportional to the current supply frequency during normal operation. However, also internally or

externally controlled voltage supply control is concerned. Thus, on an overall level of the invention, it is not essential how the operational speed of the pump is controlled, just that the rotational speed of the pump 2 may be
5 controlled/adjusted.

The inventive method is directed to control a pump arrangement comprising a pump 2 having a motor 7 and a control unit 6 arranged to control said motor 7, in order to achieve an efficient cleaning of the pump upon clogging. The pump
10 station 1 shall in this context be seen as a delimited plant to which incoming liquid arrive and from which outgoing liquid is pumped. The pump station shall, as regards the present invention, be regarded independently of the type of liquid and independently of wherefrom the liquid originates and whereto
15 the liquid is pumped. In the case the pump station comprises several pumps 2 suitable alterations between them may take place, however this is not described further by the present application.

Thereto the pump 2 is started and stopped during normal
20 operation in accordance with known methods and is not described herein.

In figure 2 is shown a predetermined embodiment of a method, generally designated 10, for control of a pump arrangement comprising a pump 2 and a control unit 6. It shall
25 be pointed out that the inventive method 10 may be expanded with one or more sub methods, and/or be driven in parallel/sequentially with other control methods.

The inventive method 10 for control of a pump arrangement is in practice a cleaning method for a pump that is entirely
30 or partly clogged, i.e. a foreign material has entered the pump 2 and wedged the impeller 8.

The degree of clogging and/or the type of clogging cause a load on the motor 7 of the pump 2 and indicate an operational condition of the pump arrangement. Thus the motor
35 7 at each individual point of time, when the pump 2 is in an active state and the motor 7 is driven in a first direction by the control unit 6, is associated with a load level that corresponds to an operational condition of the pump arrange-

ment. The pump arrangement also comprises means for, inter-
mittently or continuously, monitoring at least one operational
parameter from which the torque of the motor 7 may be derived,
either by direct measurement or by being derived from the

5 measurement of another operational parameter/ quantity. Said
operational parameter P is preferably constituted by current
consumption or torque, but also other operational parameters
such as power consumption are conceivable. In reality the load
level of the motor 7 will change, thereby changing the torque
10 and the operational/ rotational speed, when the hydraulic unit
of the pump 2 is entirely or partly clogged. A direct effect
of this is that the current consumption, power consumption,
etc. of the pump is changed correspondingly, whereby the
torque of the motor 7 may be derived from for instance the
15 current consumption of the motor. Preferably the real current
consumption of the pump 2, or more precisely of the motor 7,
is monitored when the pump 2 is in the above mentioned active
state, and hereinbelow the invention will be described having
this as a basis. However, it shall be realized that the
20 invention is not delimited to the measurement of the current
consumption as the operational parameter Said operational
parameter has a normal value P_N during normal operation of the
motor 7 in a first direction. By the first direction is meant
that the impeller 8 is driven forwards, i.e. pumps liquid out
25 via the outlet pipe 4.

Now the inventive method 10 will be described in its most
basic form with reference to figure 2.

The method 10 start out from that the pump 2 is in its
active state and the motor 7 is driven in a first direction by
30 the control unit 6. In this connection and during normal
operation said first direction the direction resulting in
liquid being transported by the impeller 8 from the sump 3 via
the outlet pipe 4, i.e. the motor 7 is drive in the forward
direction. Upon start of the pump 2, i.e. starting from an
35 inactive state of the pump 2, the control unit 6 perform a
controlled, for instance linear, ramping up of the real
operational/rotational speed F of the motor 7 from 0 to an
operational speed F_N to be used during normal operation, that

for instance constitute about 75-85% of the so-called maximum rotational speed F_{MAX} of the motor 7. The maximum rotational speed of the motor 7 is the rotational speed that the motor 7 has of the pump 2 should be directly connected to the power mains (i.e. usually a current supply frequency of 50Hz or 60Hz). The normal operational speed F_N may for instance be a constant value or a value changing over time, may for instance be a manually set value or an automatically optimized value based on the momentary energy consumption, etc. This also entail that the normal value P_N of the operational parameter may be constant or changing over time in line with present status of the normal operational speed F_N . It shall also be pointed out that different nature of the pumped liquid entail different load on the pump 2 at unchanged normal operational speed F_N , which entails that the normal value P_N of the operational parameter is also dependent on the load on the pump 2 in the specific application, i.e. different pump stations receives liquid having different characteristics. Thereto the liquid entering one and the same pump station may present different characteristics during different hours of the day.

When the pump 2 is in said active state a real value P of said at least one operational parameter is determined/monitored, and in the described embodiment the real current consumption is determined. The real current/power consumption vary during normal operation about a nominal value of the current consumption due to the fact that solid matter found in the pumped liquid enters, has influence on and is transported through the hydraulic unit of the pump 2 and thereby has a momentary influence on the load level/torque of the motor 7.

During monitoring of said real value P of said at least one operational parameter it can be determined if an externally applied force acts against the motor 7 in such an extent that a detrimental operational condition of the pump arrangement is initiated, which is true if the load level/torque of the motor 7 exceed a detrimental level for the pump arrangement. By detrimental operational condition is meant an operational condition that immediately or within a short time

will result in the pump 2 and/or the control unit 6 will become overworked and break if unchanged operation of the motor 7, alternatively safety systems/protective equipment will trigger. A detrimental operational condition is present if a large and/or hard object enters the hydraulic unit of the pump 2 and is wedged between the impeller 8 and the pump housing or the suction lid/insert ring.

The method 10, when the motor 7 is driven in the first direction, comprises the step of determining if the real value P of the operational parameter exceed a predetermined clogging limit P_I , where P_I is greater than or equal to a factor 1,05 times the normal value P_N of the operational parameter. If $P > P_I$ the motor 7 is stopped otherwise continue normal operation. Preferably the relationship between the operational parameter P_I and the normal value P_N of the operational parameter is: $P_I \geq 1,1 \cdot P_N$, and most preferably $P_I \geq 1,2 \cdot P_N$.

It shall be pointed out that due to the fact that the normal value P_N of the operational parameter may vary during operation also the clogging limit P_I of the operational parameter will vary, however the above given mutual relationship between them remains.

By the expression stopping the motor is meant to perform a change of state from the active state of the pump to an inactive state of the pump 2. The step of stopping the motor 7 preferably include in this connection that the control unit 6 immediately after the determination of the clogging directly break the drive of the motor 7 in the first direction. The feature of directly breaking the drive, is realized by having the operational speed F_N of the motor 7 set equal to zero in the control unit 6, i.e. no ramping down of the rotational speed of the motor 7 takes place, or by having the operational speed F_N of the motor 7 set equal to zero by disengaging the motor 7, i.e. the motor 7 is made completely dead. This entail that the foreign object that entered and wedged the hydraulic unit of the pump 2, is not wedged harder/more severe.

After a clogging is detected and the motor 7 is stopped, the method 10 starts a cleaning sequence. After the step that the motor 7 is stopped a step of driving the motor 7 in a the

first direction opposite second direction a predetermined flushing time T_R by means of the control unit 6 is performed. The term driving the motor 7 in a second direction is meant that the motor 7 is driven in the backwards direction. During the flushing time T_R the pump arrangement tries to flush the object that has become wedged back into the sump 3.

During the flushing time T_R and the driving of the motor 7 in the second direction, the control unit 6 tries to generate a cleaning speed backwards F_{RB} of the motor 7. The absolute value of the cleaning speed backwards F_{RB} is preferably in the range 75-85% of the maximum rotational speed F_{MAX} of the motor 7. During the flushing time T_R the method performs the step of determining if the absolute value of the real value P of the operational parameter exceed the absolute value of the first unfastening limit P_{L1} , where the absolute value of the first unfastening limit P_{L1} of the operational parameter is greater than or equal to a factor 1,1 times the clogging limit P_I of the operational parameter. If $|P| > |P_{L1}|$ stopping the motor 7, which means that the material that has been wedged does not come loose and is not flushed out in the first unfastening attempt backwards. If $|P| < |P_{L1}|$ stopping the motor 7 after said flushing time T_R and then returning to normal operation, which means that the material that has become wedged is flushed back into the sump 3 during the first unfastening attempt backwards. Preferably the relationship between the first unfastening limit P_{L1} of the operational parameter and the clogging limit P_I of the operational parameter is: $|P_{L1}| \geq 2 \cdot P_I$, and most preferably $|P_{L1}| \geq 3 \cdot P_I$.

After the step that the motor 7 is stopped after it is determined that the real value P of the operational parameter exceed the clogging limit P_I , the method preferably comprises also the step of detaining the pump 2 in the inactive state a predetermined waiting time T_V . In other words the pump 2 is kept inactive a waiting time T_V before the first unfastening attempt backwards is initiated, or before a false alarm control that will be described hereinbelow.

After the step that the motor 7 is stopped after the flushing time T_R , the method preferably comprises also the step of detaining the pump 2 in the inactive state a predetermined waiting time T_V . In other words the pump 2 is kept inactive a waiting time T_V before normal operation is resumed.

Reference is now made to figure 3, in which an addition to the method according to figure 2 in the form of a false alarm control is described, other parts of the method remains unamended and are not described hereinbelow.

After the step that the motor 7 is stopped after it is determined that the real value P of the operational parameter exceed the clogging limit P_I , the method comprises the step of driving the motor 7 in the first direction during a pre-determined control time T_K by means of the control unit 6.

During the control time T_K the method perform the step of determining if the real value P of the operational parameter exceed a false alarm control limit P_F , where the false alarm control limit P_F of the operational parameter is less than or equal to the clogging limit P_I of the operational parameter.

The false alarm control is performed one or several times. If $P > P_F$ stopping the motor 7, which means that it is not a false alarm but the clogging is confirmed. During the false alarm control the material that has caused the clogging stop of the motor 7 is sometimes flushed out via the outlet pipe 4.

Preferably the relationship between the false alarm control limit P_F of the operational parameter and the normal value P_N of the operational parameter is: $P_F \geq P_N$. During the control time T_K and during the driving of the motor 7 in the first direction, the control unit 6 tries to generate a false alarm speed F_F of the motor 7 that preferably is equal to the normal operational speed F_N .

After the control time T_K the control unit 6 may continue to drive the motor 7 in the first direction according to normal operation, alternatively the motor 7 may be stopped and the pump 2 is detained in the inactive state a predetermined waiting time T_V before normal operation is resumed.

After the step that the motor 7 is stopped after it is determined that the real value P of the operational parameter

exceed the false alarm control limit P_F , the method preferably comprises also the step of detaining the pump 2 in the inactive state a predetermined waiting time T_V . In other words the pump 2 is kept inactive a waiting time T_V before the first unfastening attempt backwards is initiated.

Reference is now made to figure 4 in which an addition to the method according to figure 2 in the form of an unfastening attempt forward is described, the other parts of the method remains unamended and are not described hereinbelow.

After the step that the motor 7 is stopped after it is determined that the absolute value of real value P of the operational parameter exceed the absolute value of the first unfastening limit P_{L1} , the method comprises the step of driving the motor 7 in the first direction during a predetermined flushing time T_R by means of the control unit 6. During the flushing time T_R and the driving of the motor 7 in the first direction, the control unit 6 tries to generate a cleaning speed forward F_{RF} of the motor 7.

The cleaning speed forward F_{RF} is preferably in the range 75-100% of the maximum rotational speed F_{MAX} of the motor 7. During the flushing time T_R the method perform the step of determining if the real value P of the operational parameter exceed a second unfastening limit P_{L2} , where the second unfastening limit P_{L2} is greater than or equal to the clogging limit P_I of the operational parameter and is less than or equal to a factor 0,95 times the absolute value of the first unfastening limit P_{L1} . If $P > P_{L2}$ stopping the motor 7, which means that the material that has become wedged does not come loose and is not flushed out during the first unfastening attempt forwards. If $P < P_{L2}$ and after the flushing time T_R the control unit 6 may continue to drive the motor 7 in the first direction according to normal operation, alternatively the motor 7 may be stopped and the pump 2 being detained in the inactive state a predetermined waiting time T_V before normal operation is resumed. $P < P_{L2}$ entail that the material that has become wedged is flushed out via the outlet pipe 4 during the first unfastening attempt forwards. Preferably the relationship between the first unfastening limit P_{L1} of the operational

parameter and the second unfastening limit P_{L2} of the operational parameter is: $P_{L2} \leq 0,85 * |P_{L1}|$, and most preferably $P_{L2} = 0,8 * |P_{L1}|$.

It shall be pointed out that after the first unfastening attempt backwards yet one or more unfastening attempts backwards may be performed before the first unfastening attempt forwards is performed. Thereto the method 10 may perform several alternations between unfastening attempts backwards and unfastening attempts forwards before service personnel is called to the plant, wherein each unfastening attempt backwards may comprise one or more unfastening attempts and wherein each unfastening attempt forwards may comprise one or more unfastening attempts. For instance the first unfastening limit P_{L1} may increase after each failed unfastening attempt, and for instance the second unfastening limit P_{L2} may increase after each failed unfastening attempt.

The method 10 may also, when the wedged material has become free and before normal operation is resumed, comprise a flushing of the pump 2 by driving the motor 7 in the first direction at the maximum rotational speed F_{MAX} during a flushing time T_R by means of the control unit 6.

Reference is finally made to figures 5-7, which schematically disclose different cleaning sequences by means of an upper graph that disclose the real operational/rotational speed of the pump/motor and how this is changed over time, and a lower graph that disclose the real torque/current consumption of the pump/motor and how this is changed over time.

In figure 5 a clogging is detected whereupon a false alarm control is performed confirming the clogging. Thereafter a first unfastening attempt backwards is performed, which is successful. Thereafter a forward flushing is performed, having an optional subsequent waiting time during which the pump is inactive, before normal operation is resumed.

In figure 6 a clogging is detected whereupon a false alarm control is performed confirming the clogging. Thereafter a first unfastening attempt backwards is performed, which is unsuccessful. A first unfastening attempt forwards, which is

unsuccessful. A second unfastening attempt backwards, which is successful. Thereafter a forward flushing is performed, having an optional subsequent waiting time during which the pump is inactive, before normal operation is resumed.

- 5 In figure 7 a clogging is detected whereupon a false alarm control is performed confirming the false alarm and normal operation is resumed.

Feasible modifications of the Invention

- 10 The invention is not limited only to the embodiments described above and shown in the drawings, which primarily have an illustrative and exemplifying purpose. This patent application is intended to cover all adjustments and variants of the preferred embodiments described herein, thus the
15 present invention is defined by the wording of the appended claims and thus, the equipment may be modified in all kinds of ways within the scope of the appended claims.

- It shall be pointed out that even thus it is not explicitly stated that features from a specific embodiment may
20 be combined with features from another embodiment, the combination shall be considered obvious, if the combination is possible.

- It shall be realized that the waiting time T_V may have different lengths during different phases of the method,
25 however, one and the same reference is used in the description as well as in the claims for sake of clarity. The waiting time T_V is in the range three seconds.

- It shall be realized that the flushing time T_R may have different lengths during different phases of the method,
30 however, one and the same reference is used in the description as well as in the claims for sake of clarity. The flushing time T_{VR} is in the range three seconds.

- Exact values for the limits mentioned in this document are dependent on the specific pump arrangement and its
35 surroundings during operation and are thus not mentioned, instead the mutual relationships between the mentioned limits are the essential in this document.

Throughout this specification and the appended claims, unless the context requires otherwise, it shall be realized that the word "comprise", and variations such as "comprises" or "comprising", will be understood to imply the inclusion of
5 a stated integer or steps or group of integers or steps but not the exclusion of any other integer or step or group of integers or steps.

Claims

1. A method for controlling a pump arrangement upon clogging of a pump, the pump arrangement comprising a pump and a control unit, the pump comprising a motor, and the control unit being arranged to drive said motor, the motor during operation being associated with an operational parameter from which the torque of the motor may be derived, said operational parameter having a normal value P_N during normal operation of the motor in a first direction, the method comprising the steps of:

- driving the motor in a first direction by means of the control unit, and if a real value P of the operational parameter exceeds a predetermined clogging limit P_I :
- stopping the motor,
- driving the motor in a the first direction opposite second direction a predetermined flushing time T_R by means of the control unit, and
- stopping the motor if the absolute value of the real value P of the operational parameter during the flushing time T_R exceeds the absolute value of a first unfastening limit P_{L1} , otherwise stopping the motor after said flushing time T_R and returning to normal operation, the method being characterized in that $P_I \geq 1,05 \cdot P_N$ and $|P_{L1}| \geq 1,1 \cdot P_I$.

2. The method according to claim 1, wherein the relationship between the clogging limit P_I of the operational parameter and the normal value P_N of the operational parameter is: $P_I \geq 1,1 \cdot P_N$, preferably $P_I \geq 1,2 \cdot P_N$.

3. The method according to claim 1 or 2, wherein the relationship between the first unfastening limit P_{L1} of the operational parameter and the clogging limit P_I of the operational parameter is: $|P_{L1}| \geq 2 \cdot P_I$, preferably $|P_{L1}| \geq 3 \cdot P_I$.

4. The method according to any of claims 1-3, wherein the method after the step of stopping the motor if a real value P of the operational parameter exceed a predetermined clogging limit P_I , where $P_I \geq 1,05 \cdot P_N$, thereto comprises the step of:

- detaining the pump in an inactive state a predetermined waiting time T_v .

5. The method according to any of claims 1-4, wherein the method after the step of otherwise stopping the motor after said flushing time T_R , thereto comprises the step of:

- detaining the pump in an inactive state a predetermined waiting time T_V .

6. The method according to any preceding claim, wherein the method after the step of stopping the motor if a real value P of the operational parameter exceeds a predetermined clogging limit P_I , where $P_I \geq 1,05 \cdot P_N$, thereto comprises the steps of:

- driving the motor in the first direction during a predetermined control time T_K by means of the control unit,
- stopping the motor if the real value P of the operational parameter during the control time T_K exceeds a false alarm control limit P_F , where $P_F \leq P_I$.

7. The method according to claim 6, wherein the relationship between the false alarm control limit P_F of the operational parameter and the normal value P_N of the operational parameter is: $P_F \geq P_N$.

8. The method according to any preceding claim, wherein the method after the step of stopping the motor if the absolute value of the real value P of the operational parameter exceeds the absolute value of a first unfastening limit P_{L1} , where $|P_{L1}| \geq 1,1 \cdot P_I$, thereto comprises the steps of:

- driving the motor in the first direction during a predetermined flushing time T_R by means of the control unit,
- stopping the motor if the real value P of the operational parameter exceeds a second unfastening limit P_{L2} , where $P_{L2} \geq P_I$ and $P_{L2} \leq 0,95 \cdot |P_{L1}|$.

9. The method according to claim 8, wherein the relationship between the second unfastening limit P_{L2} and the first unfastening limit P_{L1} is: $P_{L2} \leq 0,85 \cdot |P_{L1}|$, preferably $P_{L2} = 0,8 \cdot |P_{L1}|$.

10. The method according to any preceding claim, wherein the operational parameter is constituted by the power consumption of the motor.

11. The method according to any preceding claim, wherein the sub step of stopping the motor in the step of stopping the motor if a real value P of the operational parameter exceeds a predetermined clogging limit P_1 , where $P_1 \geq 1,05 \cdot P_N$, includes that the control unit immediately after it is determined that the real value P of the operational parameter exceeds the clogging limit P_1 directly breaks the driving of the motor in said first direction.