

Description

The invention relates to drive means for recording and/or reproducing apparatus.

Recording and/or reproducing apparatus, such as a video tape recorder, has been proposed using a common motor to drive a capstan axle as well as two reel tables in order to make the apparatus more compact.

In such recording and/or reproducing apparatus, the motor causes the capstan axle to rotate clockwise or counterclockwise to cause magnetic tape to travel along a tape travelling path in a forward or reverse direction. The magnetic tape is urged onto the capstan axle by means of a pinch roller so as to travel depending upon the rotation of the capstan axle. The motor also causes two reel tables, i.e. a supply reel table and a take-up reel table selectively to rotate clockwise or counterclockwise via a pendulum gear which is movable between the supply and take-up reel tables. The pendulum gear is engageable with one of the reel tables via corresponding idler gears. With this construction, in order to cause the magnetic tape to travel in the forward direction, for example, the capstan motor rotates counterclockwise, so that the magnetic tape is fed toward the take-up reel table depending upon the rotation of the capstan axle. At this time, the pendulum gear engages the take-up reel table via the respective idler gear so that the take-up reel table rotates clockwise to wind the magnetic tape thereon. On the other hand, in order to cause the magnetic tape to travel in the rearward direction, the capstan motor rotates clockwise, so that the magnetic tape is fed toward the supply reel table depending upon the rotation of the capstan axle. At this time, the pendulum gear engages the supply reel table via the respective idler gear so that the supply reel table rotates counterclockwise to wind the magnetic tape thereon.

In this way, the recording and/or reproducing apparatus may operate in a forward or reverse reproduction mode, in which the recording and/or reproducing apparatus may indicate an image on a screen of a television by reading picture signals by means of a rotary head drum while the magnetic tape travels in the first or second direction.

In a case where the recording and/or reproducing apparatus operates in a FF/REW mode, the magnetic tape is withdrawn into a tape cassette, and thereafter the capstan motor rotates clockwise or counterclockwise so that the supply or take-up reel table rotates to wind the magnetic tape onto the corresponding reel table in the aforementioned manner. In this case, the voltage or electrical current applied to the capstan motor is so controlled that the motor rotates at a higher speed than that in the forward or reverse reproduction mode.

However, in the aforementioned conventional recording and/or reproducing apparatus, although the

rewinding time of the magnetic tape onto the reel table can be shortened, there is a disadvantage in that the switching time from the FF/REW mode to the forward or reverse reproduction mode is relatively long since the magnetic tape is housed only within the tape cassette in the FF/REW mode.

That is, when the operation mode of the recording and/or reproducing apparatus is switched from the FF/REW mode to the forward or reverse reproduction mode, it is required that the magnetic tape is drawn out of the tape cassette so as to form a given tape travelling path. There is a disadvantage in that forming such a tape travelling path is time consuming.

If the magnetic tape is wound onto the reel table at a high speed in the FF/REW mode while it is arranged in the tape loaded condition, i.e. it is arranged in the tape travelling path, then there is a disadvantage in that the rotation speed of the reel table cannot be so high since the surface of the magnetic tape will be damaged, as well as causing undue wear to the magnetic head(s) of the VCR.

Patent Specification EP-A-267 756 discloses a drive arrangement for a tape recorder wherein a pair of reel tables are driven by a driving means by way of first or second gear trains which selectively drive the reel tables according to the operational mode of the tape recorder which has been selected.

According to the invention there is provided a system for supplying driving force to a recording and/or reproducing apparatus which has a first reel table onto which a tape travelling in a first direction is wound, and a second reel table onto which the tape travelling in a second direction, opposite to the first direction, is wound, the system comprising:

driving means to produce driving force;

a first gear train selectively to transmit the driving force produced by the driving means to one of the first and second reel tables in a first operational mode so as to cause the selected first or second reel table to rotate at a first speed;

a second gear train to transmit the driving force produced by the driving means to the first reel table in a second operational mode so as to cause the first reel table to rotate at a second speed higher than the first speed; and

switching means to select one of the first and second operational modes to cause the selected first or second gear train to transmit driving force to the corresponding reel table;

the second gear train transmitting driving force to the first reel table when the tape is positioned in a tape unloaded position different from the tape loaded position;

the tape feeding means including an axle caused to rotate by the driving means, and a roller which is movable between a first position in which the roller makes thrusting contact with the axle via the tape in the tape loaded position, and a second posi-

tion in which the roller is separated from the axle by a predetermined distance, the roller being rotatable depending upon the rotation of the axle and tightly holding the drawn tape between the axle and the roller so as to cause the drawn tape to travel in the first or second direction when the roller is positioned in the first position; and

characterised in that the first gear train transmits driving force to the selected first or second reel table when the roller is positioned in the second position.

Such a system can supply driving force to a recording and/or reproducing apparatus, which can perform the tape forwarding or rewinding in two tape arrangements, one of which is a tape loaded condition in which a tape is arranged along a usual tape travelling path, and the other of which is a tape unloaded condition in which the tape is arranged at a location other than the tape travelling path, or is housed within a tape cassette.

The recording and/or reproducing apparatus can selectively operate in a forward or reverse reproduction mode, a FF/REW mode, or a high-speed FF/REW mode.

The system can include a mechanism to transmit driving force to the recording and/or reproducing apparatus, which can change the decelerating ratio of a gear train arranged between a driving source and a supply or take-up reel table.

A torque limiter can be applied to a gear train arranged between a driving source and a supply or take-up reel table.

The system may further comprise tape feeding means to cause the tape selectively to travel in the first or second direction when tape drawn out of a cassette loaded into the recording and/or reproducing apparatus is positioned in the tape loaded position. The first gear train may transmit the driving force to the selected first or second reel table when the tape drawn out of the cassette is positioned in the tape loaded position. The second gear train may transmit the driving force to the first reel table when the tape is positioned in the tape unloaded position, a position other than the tape loaded position. The tape feeding means may include an axle caused to rotate by the driving means, and a roller which is movable between a first position in which the roller makes thrusting contact with the axle via the tape of the tape loaded position, and a second position in which the roller is separated from the axle by a predetermined distance, the roller being rotatable depending on the rotation of the axle and tightly holding the drawn tape between the axle and the roller so as to cause the drawn tape to travel in the first or second direction when the roller is positioned in the first position. The first gear train may transmit the driving force to the selected first or second reel table when the roller is positioned in the second position.

The driving means may produce rotation force which may be selectively transmitted to one of the first and second gear trains. The decelerating ratio of the second gear train is preferably less than that of the first gear train. The first gear train may include a driving gear arranged between the first and second reel tables, a pendulum gear engaging the driving gear, and a first and a second idler gear which engage corresponding gear portions of the first or second reel tables, the pendulum gear being movable for engaging one of the first and second idler gears depending upon the rotation of the driving gear in the first operation mode. In addition, the second gear train may include the driving gear, an intermediate gear engaging the driving gear, and the first idler gear, the intermediate gear engaging the first idler gear in the second operation mode. The system may further comprise means for preventing the pendulum gear from engaging gear portions of either the first or second reel tables in the second operation mode. The first idler gear may include a smaller gear member which is engageable with the intermediate gear, and a larger gear member which is engageable with the gear portion of the first reel table, the diameter of the larger gear member being greater than that of the smaller gear member. The intermediate gear is movable between an upper position in which the intermediate gear disengages from the smaller gear member of the first idler gear, and a lower position in which the intermediate gear engages the smaller gear member.

The invention is diagrammatically illustrated by way of example in the accompanying drawings, in which:-

Figure 1 is a schematic plan view of drive means for recording and/or reproducing apparatus according to the invention;

Figure 2 is a fragmentary plan view of the recording and/or reproducing apparatus of Figure 1;

Figure 3 is a sectional view taken on line III-III of Figure 1;

Figure 4 is a fragmentary exploded perspective view of the recording and/or reproducing apparatus of Figure 1; and

Figures 5(A) to 5(C) are sectional views similar to Figure 3, which indicate a high-speed REW mode, a FF/REW mode and a forward or reverse reproduction mode, respectively.

Recording and/or reproducing apparatus described below is designed to record and reproduce a new and only recently introduced magnetic tape cassette specifically designed for PCM audio recording and/or reproduction. However, the invention is applicable not only to that specific PC audio recording magnetic tape cassette but also to any other magnetic tape cassette which has the same or a similar cassette structure.

The recording and/or reproducing apparatus may selectively operate in forward or reverse reproduction

mode. Throughout the disclosure, by forward and reverse reproduction modes is meant modes in which the recording and/or reproducing apparatus may indicate an image on a television screen by reading picture signals by means of a rotary head drum while the magnetic tape travels in a forward and a reverse direction, respectively.

Because of the rather complicated structure of the preferred embodiments the following disclosure will be made in terms of separate groups of components. Throughout the disclosure, the word "front", "rear", "right" and "left" used with respect to the recording and/or reproducing apparatus mean the bottom and top, and the right and left sides respectively as viewed in Figure 1.

Referring to the drawings and particularly to Figure 1, recording and/or reproducing apparatus 1, such as a video tape recorder, includes an essentially rectangular mechanical chassis 2. A pair of reel tables, i.e. supply and take-up reel tables 3 and 4 are arranged on the upper surface of the mechanical chassis 2 respectively on the front-right and front-left sides of the centre thereof so as to be separated from each other by a predetermined distance. The supply and take-up reel tables 3 and 4 are rotatably supported on the mechanical chassis 2, and have respective reel gears 3a and 4a on the periphery thereof.

A capstan motor 5 is mounted on the lower surface of the mechanical chassis 2 at a location behind the take-up reel table 4. An axle 6 to supply output of the capstan motor 5, which will be referred to as a "capstan axle" hereinafter, passes through an opening formed in the mechanical chassis 2 so as to project upwards from the upper surface thereof.

A pinch roller 7 is rotatably mounted on the upper surface of the mechanical chassis 2 and so arranged as to face the capstan axle 6. The pinch roller 7 is designed tightly to thrust against the capstan axle 6 in order to cause magnetic tape to travel when the recording and/or reproducing apparatus 1 operates in a reproducing mode in which picture signals are reproduced.

A driving pulley 8 is arranged on the lower surface of the mechanical chassis 2 and fixed to the capstan axle 6 so as to rotate depending upon the rotation of the capstan axle 6.

When a magnetic tape cassette 9 is loaded into the recording and/or reproducing apparatus 1 at a predetermined position, a magnetic tape 10 housed within the magnetic tape cassette 9 is designed to be drawn out and to be wound onto a rotary head drum 11 at a predetermined tape-wrap angle. In addition, the magnetic tape 10 is designed to be guided by various tape guides so as to form a given tape travelling path.

Driving Mechanism

(Figures 1 to 5)

A sub-plate 12 is arranged on the lower surface of the mechanical chassis 2 at a location beneath between the supply and take-up reel tables 3 and 4. As can be seen clearly from Figure 3, the sub-plate 12 is fixed to the lower surface of the mechanical chassis 2 so as to form a space therebetween, and supports various gears thereon.

As shown in Figure 1, a driven pulley 13 is also arranged on the lower surface of the mechanical chassis at a location beneath a slightly forward position of the take-up reel table 4, and is rotatably supported on the mechanical chassis 2. As can be seen clearly from Figure 3, a spur gear 14 which has a slightly smaller diameter than that of the driven pulley 13, is arranged on the upper surface of the driven pulley 13. The spur gear 14 is integrally formed with the driven pulley 13. Furthermore, the driven pulley is arranged at generally the same height as that of the driving pulley 8, and the rotation force of the driving pulley 8 is transmitted to the driving pulley 13 via a belt 15.

Switching Mechanism

(Figures 2 to 5)

As shown in Figure 3, a vertically extending shaft 16 is disposed on the left side of the driven pulley 13, and is supported on the upper surface of the sub-plate 12. A mode switching gear 17 is rotatably supported on the shaft 16 and is movable upwards and downwards along the axis of the shaft 16. The mode switching gear 17 is designed always to engage the spur gear 14. In addition, a coil spring 18 is provided around the shaft 16 between the sub-plate 12 and the mode switching gear 17 so that the mode switching gear 17 is biased upwards.

As can be clearly seen from Figure 4, a thrust member 19 having generally U-shaped cross-section is provided to thrust the mode switching gear 17 downwards against the force of the coil spring 18. The thrust member 19 generally comprises a pair of arm portions 19a and a base portion 19b. The arm portions 19a of the thrust member 19 are pivotably supported on a pair of supporting lugs 20 which are respectively arranged on the sub-plate 12 at locations adjacent the mode switching gear 17 so as to extend upwards from the upper surface of the sub-plate 12. The base portion 19b extends between the free ends of the arm portions, and has a pair of arm pieces 19c generally at the centre thereof. The arm pieces 19c are separated from each other at a predetermined distance and extend at right angles (as can be seen in Figure 3) from the base portion 19b. Each of the arm pieces 19c has a thrust portion 19c' which extends from the end thereof in an essentially perpen-

dicular direction to the length of the respective arm piece 19c. The ends of the thrust portions 19c' of the arm pieces 19c are designed to thrust the mode switching gear 17 downwards at locations neighbouring the rotation axis of the mode switching gear 17. The thrust member 19 also has a step 19d which is integrally formed with the rearward arm portion 19a. The step 19d extends rearwards from the lower edge of the rearward arm portion 19a. As will be described hereinafter, a slide member 34 is designed to push against the step 19d. The thrust member 19 also has a spring supporting piece 19e which is integrally formed with the step 19d. The spring support piece 19e extends downwards from the left edge of the step 19d, and passes through an opening formed in the sub-plate 12 so as to project from the lower surface thereof.

In addition, the end of the spring support piece 19e is connected to one end of an extension spring 21, the other end of which is connected to a spring support portion 12a extending downwards from the lower surface of the sub-plate 12, so that the extension spring 21 biases the thrust portions 19c' of the arm pieces 19c downwards. In this way, the thrust portions 19c' are designed to thrust the mode switching gear 17 downwards. As will be described hereinafter, when the slide member 34 thrusts the step 19d in the right direction, the thrust member 19 pivots counterclockwise in Figures 5(A) to 5(C) so that the thrust portions 19c' disengage from the mode switching gear 17 to release downward thrusting force thereon. When the mode switching gear 17 is disengaged from the thrust portions 19c', it will move upwardly due to the force of the coil spring 18. In this way, the mode switching gear 17 can move between a lower position in which it is urged toward the sub-plate 12 by means of the thrust member 19, and an upper position in which it is urged upwards by means of the coil spring 18.

A shaft 22 is fixed to the sub-plate 12 generally at the centre thereof. The shaft 22 extends upwards and rotatably supports a driving gear assembly 23.

The driving gear assembly 23 generally comprises a lower spur gear 24, a rotary sleeve member 25 which extends upwards from the centre of the upper surface of the lower spur gear 24, an upper spur gear 26 which is integrally formed on the upper end of the rotary sleeve member 25, and a limiter gear 27 which is rotatable on the rotary sleeve member 25. As can be seen from Figure 3, a compression coil spring 28 is arranged between the limiter gear 27 and the upper spur gear 26. In addition, a felt member (not shown) is arranged between the limiter gear 27 and the lower spur gear 24. When load applied to the driving gear assembly 23 is relatively low, the limiter gear 27 is designed to rotate together with the lower and upper spur gears 24 and 26. As the load becomes greater, the limiter gear 27 is designed to slip on the sleeve

member 25 so as to produce different rotation speeds between the limiter gear 27 and the lower spur gear 24, and between the limiter gear 27 and the upper spur gear 26.

The upper spur gear 26 of the driving gear assembly 23 is relatively thick, and the greater part thereof passes through an opening 2a formed in the mechanical chassis 2 to project from the upper surface of the mechanical chassis 2.

When the mode switching gear 17 is positioned at its lower position, it engages the lower spur gear 24, and when it is positioned at its upper position, it engages the limiter gear 27.

Furthermore, a shaft 29 is fixed to the sub-plate 12 at a location neighbouring the left edge thereof (as viewed in Figure 2). The shaft 29 extends upwards from the sub-plate 12, and passes through an opening 2b formed in the mechanical chassis 2 on the left side of the opening 2a, to project from the upper surface of the mechanical chassis 2.

As can be seen clearly from Figure 4, a thrust member 30 having an essentially U-shaped cross-section is provided for moving an intermediate gear 40 which will be described hereinafter. The thrust member 30 generally comprises a pair of arm portions 30a and a web portion 30b. The arm portions 30a are pivotably supported on a pair of support lugs 31 which are respectively arranged on the sub-plate 12 at the front-left and rear-left sides of the driving gear assembly 23 so as to extend upwards from the upper surface of the sub-plate 12. The web portion 30b extends between the arm portions 30a, and is integrally formed with a thrust piece 30c at a location slightly offset from the middle thereof. The thrust piece 30c extends from the web portion 30b, and has an elongate through opening 30d, through which the support shaft 29 passes.

The thrust member 30 is so biased as to be rotated clockwise (as viewed in Figures 5(A) to 5(C)) by means of a compression coil spring 32 which is disposed between the lower surface of the thrust piece 30c and the sub-plate 12.

As can be seen clearly from Figure 4, the rearward arm portion 30a of the thrust member 30 is provided with a stop piece 30e which extends in a right direction from the lower-right edge of the rearward arm portion 30a. When the thrust member 30 rotates clockwise, the stop piece 30e comes into contact with the upper surface of the sub-plate 12 to prevent the thrust member 30 from rotating excessively. One of the arm portions 30a is formed with a step 30f at the upper-right edge. As will be described hereinafter, the sliding member 34 is designed to thrust against the step 30f.

The top end of the shaft 29 is supported on a support plate 33 which is arranged to be separated from the mechanical chassis 2 by a predetermined distance.

As will be described hereinafter, the intermediate gear 40 is rotatably supported on the shaft 29 at a location above the thrust piece 30c, so as to be movable along the shaft 29 in the upward and downward directions. The intermediate gear 40 is designed to be held at an upper position when the thrust piece 30c rotates clockwise (as viewed in Figures 5(A) to 5(C)) due to the force of the compression coil spring 32, and to be held at a lower position when the thrust piece 30c rotates counterclockwise (as viewed in Figures 5(A) to 5(C)) by means of the sliding member 34.

The sliding member 34 is arranged beneath the mechanical chassis 2 at a location behind a line between the rotation axes of the reel tables 3 and 4. The sliding member 34 is supported on the mechanical chassis 2 so as to be slidable in the left and right directions. The right end of the sliding member 34 is integrally formed with a connecting projection 34a which projects downwards from the lower surface thereof. The left end of the sliding member 34 is formed with a step 34b serving as a first thrust portion to thrust the step 30f of the thrust member 30. The sliding member 34 is also integrally formed with a thrust piece 34c which projects downwards from the lower surface thereof, slightly left of the middle. The thrust piece 34c has a second thrust portion 34d which extends from the lower edge thereof so as to face the step 19d of the thrust member 19.

The distance between the first and second thrust portions 34b and 34d is similar to or slightly smaller than the distance between the step 19d when the thrust member 19 thrusts the mode switching gear 17 downwards, and the step 30f when the thrust member 30 thrusts the intermediate gear 40 upwards.

As shown in Figure 2, a cam gear 35 is rotatably supported on the mechanical chassis 2 at a location beneath a position behind the take-up reel table 4. The cam gear 35 is designed to be driven by means of a mode selecting motor 36, mounted on the lower surface of the mechanical chassis 2, via a worm gear 37. The cam gear 35 has a wheel gear portion 35a which engages the worm gear 37 connected to the output shaft of the mode selecting motor 36. The cam gear 35 also has a cam groove 35b on the upper surface thereof.

An oscillating lever 38 is disposed on the lower side of the mechanical chassis 2, and is rotatably supported on a support shaft 39 which projects from the cam gear 35. The oscillating lever 38 has, at the free end thereof, an elongate through opening 38a which extends in the longitudinal direction of the oscillating lever 38. The oscillating lever 38 is integrally formed with a pin 38b which projects downwards from the lower surface of the oscillating lever 38 at a location between the middle of the oscillating lever 38 and the rotation axis thereof.

The pin 38b of the oscillating lever 38 is slidably engaged in the cam groove 35b of the cam gear 35,

and the connecting projection 34a of the sliding member 34 slidably engages the elongate through opening 38a of the oscillating lever 38.

When the cam gear 35 rotates, via the worm gear 37, depending upon the rotation of the mode selecting motor 36, the pin 38b of the oscillating lever 38 moves along the cam groove 35b, so that the oscillating lever 38 oscillates, thereby sliding the sliding member 34 to the right or the left.

The sliding member 34 is movable between a position in which the first thrusting portion 34b thereof thrusts the step 30f of the thrust member 30 in the leftward direction so that the thrust member 30 rotates counterclockwise as viewed in Figure 5(A), and which will be referred to as "position A", and a position in which the second thrusting portion 34d of the sliding member 34 thrusts the step 19d of the thrust member 19 in the rightward direction so that the thrust member 19 rotates counterclockwise and the thrust portions 19c thereof release the mode switching gear 17 from the pushed downwards position as shown in Figure 5(C), which will be referred to as "position C". Between position A and position C, the sliding member 34 can be positioned at a neutral position in which both of the first and second thrust portions 34b and 34d thereof are respectively disengaged from the steps 30f and 19d of the thrust members 30 and 19 as shown in Figure 5(B), this will be referred to as "position B".

As mentioned above, the intermediate gear 40 is rotatably supported by the upper portion of the shaft 29 so as to be movable in the upward and downward directions. The intermediate gear 40 is integrally formed with a sleeve member 41 which extends downwards from the middle of the lower surface thereof. The lower end surface of the sleeve member 41 is supported on the thrust member 30c of the thrust member 30 around the elongate through opening 30d. The intermediate gear 40 is biased downwards by means of a compression coil spring 42 disposed between the upper surface of the intermediate gear 40 and the lower surface of the support plate 33.

The resiliency of the compression coil spring 42 is designed to be less than that of the compression coil spring 32 disposed around the shaft 29 at the lower portion thereof, so that the intermediate gear 40 is held at its upper position when the sliding member 34 is positioned in positions B or C in which the first thrust portion 34b of the sliding member 34 is disengaged from the step 34f of the thrust member 30.

The intermediate gear 40 always engages the upper spur gear 26 of the driving gear assembly 23 when it is positioned at either the upper or lower position.

An oscillating lever 43 is pivotably supported on the shaft 22 of the driving gear assembly 23. The oscillating lever 43 is arranged on the upper side of the upper spur gear 26, and has a shaft 45 which extends downwards from the free end of the oscillating lever

43. The shaft 45 of the oscillating lever 43 rotatably supports a pendulum gear 44 which engages the upper spur gear 26. The lower end of the shaft 45 passes through a generally rectangular opening 2c formed in the mechanical chassis 2, and approximately centred between the reel tables 3 and 4, so as to project downwards from the lower surface of the mechanical chassis 2.

The reel gears 3a and 4a of the reel tables 3 and 4 respectively engage idler gears 46 and 47 which are rotatably supported on the upper surface of the mechanical chassis 2. The pendulum gear 44 can move between the idler gears 46 and 47 to engage one of the idler gears 46 and 47. When the pendulum gear 44 is positioned at a generally intermediate location between the idler gears 46 and 47, the pendulum gear 44 is designed not to engage either of the idler gears 46 or 47.

The idler gear 46 engaging the supply reel gear 3a is integrally formed with a smaller-diameter spur gear 48 which is arranged on the upper surface of the idler gear 46 so as to be coaxial with the idler gear 46. The spur gear 48 is designed to engage the intermediate gear 40 when the intermediate gear 40 is positioned in its lower position.

Oscillation Preventing Mechanism

(Figure 2)

The recording and/or reproducing apparatus 1 is provided with an oscillation preventing mechanism 49 to prevent the pendulum gear 44 from oscillating. The oscillation preventing mechanism 49 generally comprises a cam gear 50, an oscillating link 51 and a slide 52.

The cam gear 50 comprises a spur gear, one surface of which is formed with a cam groove 50a. The cam gear 50 is rotatably supported on the lower surface of the mechanical chassis 2 so that the cam groove 50a faces the lower surface of the mechanical chassis 2. The cam gear 50 is designed to be rotated by means of the mode selecting motor 36 via a gear train (not shown).

The oscillating link 51 is a Bell-crank lever with a shorter arm 51a and a longer arm 51b. The angle of the oscillating link 51 is rotatably supported on the lower surface of the mechanical chassis 2 at a location adjacent the cam gear 50 so that the oscillating link 51 can oscillate along the lower surface of the mechanical chassis 2. The shorter arm 51a of the oscillating link 51 extends from the oscillating axis in a right-rear direction so that the end thereof is disposed between the cam gear 50 and the mechanical chassis 2. The longer arm 51b extends from the oscillating axis in a forward direction.

The free end of the shorter arm 51a of the oscillating link 51 is integrally formed with a pin 51c which projects downwards and engages the cam groove

50a of the cam gear 50. The free end of the longer arm 51b is integrally formed with a connecting pin 51d which projects downwards and engages the slide 52 as will be described hereinafter.

When the cam gear 50 rotates by means of the mode selecting motor 36 via the gear train (not shown), the pin 51c of the shorter arm 51a of the oscillating link 51 moves along the cam groove 50a, so that the connecting pin 51d of the longer arm 51b oscillates in an essentially lateral direction.

The slide 52 is made of a synthetic resin, and is supported on the lower surface of the mechanical chassis 2 at a location adjacent the front edge thereof so as to be slidable in an essentially lateral direction. One end of the slide 52 is formed with a slightly elongate opening 52a which extends in the width direction of the slide 52. The connecting pin 51d of the longer arm 51b of the oscillating link 51 slidably engages the elongate opening 52a. The other end of the slide 52 extends toward the through opening 2c, and is integrally formed with a pair of thrust pieces 53 which project rearwards. Between the thrust pieces 53, the lower end of the shaft 45 of the pendulum gear 44 is disposed.

When the oscillating link 26 oscillates by means of the mode selecting motor 36, the slide 52 slides laterally so that one of the thrust pieces 53 causes the support shaft 45 of the pendulum gear 44 to move to control the oscillating action of the pendulum gear 44.

Operation In Forward and Reverse Reproduction Mode, FF/REW Mode, and High-Speed REW Mode

(Figures 5(A) to 5(C))

In a case where one of the operational modes of the recording and/or reproducing apparatus 1 is selected, the mode selecting motor 36 rotates to cause the cam gears 35 or 50 to rotate so as to set-up the selected operational mode.

When the forward or reverse reproduction mode is selected, the cam gear 35 causes the slide member 34 to move from position B to position C as shown in Figure 5(C). In this position, the second thrusting portion 34d thrusts the step 19d of the thrust member 19 in the rightward direction, so that the thrust portions 19c' move upwards, allowing the mode switching gear 17 to move upwards due to the force of the compression coil spring 18. As a result, the mode selecting gear 17 engages the limiter gear 27 of the driving gear assembly 23.

At this time, in the case of the forward reproduction mode, the capstan motor 5 rotates counterclockwise so that the capstan axle 6 cooperates with the pinch roller 7 to feed the magnetic tape 10 toward the take-up reel table 4. In addition, the rotation force of the capstan motor 5 is transmitted to the upper spur gear 26 of the driving gear assembly 23 via the driving

pulley 8, the belt 15, the driven pulley 13, the spur gear 14, the mode switching gear 17 and the limiter gear 27, so that the upper spur gear 26 rotates counterclockwise. As a result, the oscillating lever 43 moves toward the take-up reel table 4 and engages the take-up idler gear 47 to cause the take-up reel table 4 to rotate clockwise so that the magnetic tape 10 fed by the capstan axle 6 and the pinch roller 7 is wound onto the take-up reel table 4.

On the other hand, in the case of the reverse reproduction mode, the capstan motor 5 rotates reversely or clockwise so that the magnetic tape 10 is fed toward the supply reel table 3. At this time, all of the aforementioned pulleys and gears rotate reversely to the aforementioned forward reproduction mode, so that the pendulum gear 44 moves toward the supply reel table 3, thereby the magnetic tape 10 is wound onto the supply reel table 3.

Furthermore, in the forward or reverse reproduction mode, the thrust pieces 53 of the slide 52 are positioned so as not to thrust the support shaft 45 of the pendulum gear 44, so that the pendulum gear 44 can move in a predetermined direction depending upon the direction of rotation of the driving gear assembly 23.

The recording and/or reproducing apparatus 1 may also selectively operate in a FF/REW mode or a high-speed FF/REW mode. In the FF/REW mode, the magnetic tape 10 is fed to either the take-up or supply reel table under a tape loaded condition, i.e. a condition in which the pinch roller 7 is spaced from the capstan axle 6 while the magnetic tape 10 is being drawn out of the tape cassette 9 to form the tape travelling path. In the high-speed FF/REW mode, the magnetic tape 10 is fed to either the take-up or supply reel table under a tape unloaded condition, i.e. a condition in which the magnetic tape 10 is housed within the tape cassette 9.

When the FF/REW mode is selected, the pinch roller 7 is separated from the capstan axle 6 while the tape travelling path formed by the drawn magnetic tape 10 is not changed. In this condition, the cam gear 35 causes the sliding member 34 to be positioned at the position B as shown Figure 5(B). At this position, neither the first thrust portion 34b nor the second thrust portion 34d thrust the respective step 30f and 19d of the thrust member 30 and 19. Therefore, the mode switching gear 17 moves downwards, and the intermediate gear 40 is held at the upper position.

As a result, the mode switching gear 17 engages the lower spur gear 24 of the driving gear assembly 23, so as to form a first gear train comprising the upper spur gear 26 of the driving gear assembly 23, the pendulum gear 44 and idler gear 46 or 47. In this condition, when the capstan motor 5 rotates, the rotation force thereof is transmitted to the first gear train via the driving pulley 8, the belt 15, the driven pulley 13, the spur gear 14, the mode switching gear and the

lower spur gear 24 of the driving gear assembly 23. In this way, magnetic tape 10 can be wound onto the reel tables 3 or 4 in the FF/REW mode.

When the high-speed REW mode is selected, the magnetic tape 10 is housed within the tape cassette 9. In this condition, the cam gear 35 causes the sliding member 34 to be positioned at the position A as shown in Figure 5(A). At this position, the first thrust portion 34b of the slide member 34 thrusts the step 30f of the thrust member 30 in the left direction, so that the thrust member 30 rotates counterclockwise, thereby the thrust piece 30c thereof moves downwards. As a result, the intermediate gear 40 moves downwards to its lower position due to the force of the compression coil spring 42, so as to engage the spur gear 48 integrally formed on the idler gear 46.

In this way, a second gear train comprising the upper spur gear 26 of the driving gear assembly 23, the intermediate gear 40, the spur gear 48 and the idler gear 46 is formed. In this condition, when the capstan motor 5 rotates clockwise, the rotation force is transmitted to the second gear train, so that the magnetic tape 10 can be wound onto the supply reel table 3 in the high-speed REW mode.

In this case, in order to prevent the oscillating lever 43 from rotating due to the rotation of the driving gear assembly 23 and to prevent the pendulum gear 44 from engaging the idler gear 46, the sliding member 52 is controlled by means of the cam gear 50 so that the thrust piece 53 of the sliding member 52 causes the support shaft 45 of the pendulum gear 44 to be positioned at its neutral position.

As mentioned above, in the second gear train, the spur gear 48 having a smaller diameter than that of the idler gear 46 is integrally formed on the idler gear 46, and the intermediate gear 40 causes the smaller-diameter spur gear 48 to rotate. Therefore, the decelerating ratio of the second gear train used in the high-speed REW mode is smaller than that of the first gear train used in the FF/REW mode. As a result, if the rotation speed of the capstan axle 5 in the FF/REW mode is the same as that in the high-speed REW mode, the supply reel table 3 can rotate at a higher speed in the high-speed REW mode than in the FF/REW mode.

As mentioned above, in the recording and/or reproducing apparatus, forwarding or rewinding of the magnetic tape can be performed under any one of two conditions, i.e. tape loaded and unloaded conditions. In the FF/REW mode, although the travelling speed of the magnetic tape is not so high, mode switching time to the forward or reverse reproduction mode is shorter. On the other hand, in the high-speed FF/REW mode, the time required for forwarding or rewinding magnetic tape is shorter. In addition, these modes can be achieved without providing any complex mechanisms.

In the shown embodiment, although only the

high-speed REW mode is described when the magnetic tape is wound onto the reel table at a higher speed, a high-speed FF mode or both a high-speed FF and a high-speed REW mode can be also formed by means of the aforementioned transmitting mechanism.

Claims

1. A system for supplying driving force to a recording and/or reproducing apparatus which has a first reel table (3) onto which a tape (10) travelling in a first direction is wound, and a second reel table (4) onto which the tape (10) travelling in a second direction, opposite to the first direction, is wound, the system comprising:
 - driving means (5) to produce driving force;
 - a first gear train (23) selectively to transmit the driving force produced by the driving means (5) to one of the first and second reel tables (3, 4) in a first operational mode so as to cause the selected first or second reel table to rotate at a first speed;
 - a second gear train to transmit the driving force produced by the driving means (5) to the first reel table (3) in a second operational mode so as to cause the first reel table to rotate at a second speed higher than the first speed; and
 - switching means (34) to select one of the first and second operational modes to cause the selected first or second gear train to transmit driving force to the corresponding reel table;
 - the second gear train transmitting driving force to the first reel table when the tape is positioned in a tape unloaded position different from the tape loaded position;
 - the tape feeding means including an axle (6) caused to rotate by the driving means, and a roller (7) which is movable between a first position in which the roller makes thrusting contact with the axle via the tape (10) in the tape loaded position, and a second position in which the roller is separated from the axle by a predetermined distance, the roller being rotatable depending upon the rotation of the axle and tightly holding the drawn tape between the axle and the roller so as to cause the drawn tape to travel in the first or second direction when the roller is positioned in the first position; and
 - characterised in that the first gear train transmits driving force to the selected first or second reel table when the roller (7) is positioned in the second position.
2. A system according to claim 1, which further comprises tape feeding means (6, 7) to cause the tape selectively to travel in the first or second direction

when tape, which is drawn out of a cassette loaded in the recording and/or reproducing apparatus, is positioned in a tape loaded position.

3. A system according to claim 2, wherein the first gear train transmits driving force to the selected first or second reel table when the tape drawn out of the cassette is positioned in the tape loaded position.
4. A system according to claim 1, wherein the decelerating ratio of the second gear train is less than that of the first gear train.
5. A system according to claim 1, wherein the first gear train includes a driving gear (26) disposed between the first and second reel tables, a pendulum gear (44) engaging the driving gear, and first and second idler gears (46, 47) which engage corresponding gear portions (3a, 4a) of the first and second reel tables (3, 4), the pendulum gear (44) being movable to engage one of the first and second idler gears depending upon the rotation of the driving gear in the first operational mode, and wherein the second gear train includes the driving gear (26), an intermediate gear (40) engaging the driving gear, and the first idler gear, the intermediate gear (40) engaging the first idler gear (40) in the second operational mode.
6. A system according to claim 5, which further comprises means to prevent the pendulum gear (44) from engaging gear portions (3a, 4a) of either the first or second reel tables (3, 4) in the second operational mode.
7. A system according to claim 5, wherein the first idler gear (46) includes a smaller gear member (48) which is engageable with the intermediate gear (40), and a larger gear member which is engageable with the gear portion (3a) of the first reel table (3), the diameter of the larger gear member being greater than that of the smaller gear member (48).
8. A system according to claim 7, wherein the intermediate gear (40) is movable between an upper position in which the intermediate gear disengages from the smaller gear member (48) of the first idler gear (46), and a lower position in which the intermediate gear engages the smaller gear member.

Patentansprüche

1. System zum Liefern einer Antriebskraft für ein



high-speed REW mode is described when the magnetic tape is wound onto the reel table at a higher speed, a high-speed FF mode or both a high-speed FF and a high-speed REW mode can be also formed by means of the aforementioned transmitting mechanism.

Claims

1. A system for supplying driving force to a recording and/or reproducing apparatus which has a first reel table (3) onto which a tape (10) travelling in a first direction is wound, and a second reel table (4) onto which the tape (10) travelling in a second direction, opposite to the first direction, is wound, the system comprising:
 - driving means (5) to produce driving force;
 - a first gear train (23) selectively to transmit the driving force produced by the driving means (5) to one of the first and second reel tables (3, 4) in a first operational mode so as to cause the selected first or second reel table to rotate at a first speed;
 - a second gear train to transmit the driving force produced by the driving means (5) to the first reel table (3) in a second operational mode so as to cause the first reel table to rotate at a second speed higher than the first speed; and
 - switching means (34) to select one of the first and second operational modes to cause the selected first or second gear train to transmit driving force to the corresponding reel table;
 - the second gear train transmitting driving force to the first reel table when the tape is positioned in a tape unloaded position different from the tape loaded position;
 - the tape feeding means including an axle (6) caused to rotate by the driving means, and a roller (7) which is movable between a first position in which the roller makes thrusting contact with the axle via the tape (10) in the tape loaded position, and a second position in which the roller is separated from the axle by a predetermined distance, the roller being rotatable depending upon the rotation of the axle and tightly holding the drawn tape between the axle and the roller so as to cause the drawn tape to travel in the first or second direction when the roller is positioned in the first position; and
 - characterised in that the first gear train transmits driving force to the selected first or second reel table when the roller (7) is positioned in the second position.
2. A system according to claim 1, which further comprises tape feeding means (6, 7) to cause the tape selectively to travel in the first or second direction

when tape, which is drawn out of a cassette loaded in the recording and/or reproducing apparatus, is positioned in a tape loaded position.

3. A system according to claim 2, wherein the first gear train transmits driving force to the selected first or second reel table when the tape drawn out of the cassette is positioned in the tape loaded position.
4. A system according to claim 1, wherein the decelerating ratio of the second gear train is less than that of the first gear train.
5. A system according to claim 1, wherein the first gear train includes a driving gear (26) disposed between the first and second reel tables, a pendulum gear (44) engaging the driving gear, and first and second idler gears (46, 47) which engage corresponding gear portions (3a, 4a) of the first and second reel tables (3, 4), the pendulum gear (44) being movable to engage one of the first and second idler gears depending upon the rotation of the driving gear in the first operational mode, and wherein the second gear train includes the driving gear (26), an intermediate gear (40) engaging the driving gear, and the first idler gear, the intermediate gear (40) engaging the first idler gear (40) in the second operational mode.
6. A system according to claim 5, which further comprises means to prevent the pendulum gear (44) from engaging gear portions (3a, 4a) of either the first or second reel tables (3, 4) in the second operational mode.
7. A system according to claim 5, wherein the first idler gear (46) includes a smaller gear member (48) which is engageable with the intermediate gear (40), and a larger gear member which is engageable with the gear portion (3a) of the first reel table (3), the diameter of the larger gear member being greater than that of the smaller gear member (48).
8. A system according to claim 7, wherein the intermediate gear (40) is movable between an upper position in which the intermediate gear disengages from the smaller gear member (48) of the first idler gear (46), and a lower position in which the intermediate gear engages the smaller gear member.

Patentansprüche

1. System zum Liefern einer Antriebskraft für ein



Aufzeichnungs- und/oder Wiedergabegerät, das einen ersten Spulentisch (3) aufweist, auf dem ein Band (10), das in einer ersten Richtung läuft, gewickelt ist, und einen zweiten Spulentisch (4), auf dem das Band (10), das entgegengesetzt zur ersten Richtung in einer zweiten Richtung läuft, gewickelt ist, wobei das System aufweist:

eine Antriebseinrichtung (5), um eine Antriebskraft zu erzeugen;

eine erste Zahnradfolge (23), um wahlweise die Antriebskraft, die durch die Antriebseinrichtung (5) erzeugt wird, auf einen ersten oder zweiten Spulentisch (3, 4) in einem ersten Betriebsmodus zu übertragen, damit der ausgewählte erste oder zweite Spulentisch mit einer ersten Geschwindigkeit sich dreht;

eine zweite Zahnradfolge, um die Antriebskraft, die durch die Antriebseinrichtung (5) erzeugt wurde, auf den ersten Spulentisch (3) in einem zweiten Betriebsmodus zu übertragen, damit der erste Spulentisch mit einer zweiten Geschwindigkeit sich dreht, die höher als die erste Geschwindigkeit ist; und

eine Schalteinrichtung (34), um den ersten oder zweiten Betriebsmodus auszuwählen, damit die ausgewählte erste oder zweite Zahnradfolge die Antriebskraft auf den entsprechenden Spulentisch übertragen kann;

wobei die zweite Zahnradfolge die Antriebskraft an den ersten Spulentisch überträgt, wenn das Band in einer Band-Nichtladeposition sich befindet, die sich von der Bandladeposition unterscheidet;

wobei die Bandzuführeinrichtung eine Achse (6) aufweist, die durch die Antriebseinrichtung gedreht wird, und eine Rolle (7), die zwischen einer ersten Position bewegbar ist, bei der die Rolle einen Schubkontakt mit der Achse über das Band (10) in der Bandladeposition ausführt, und einer zweiten Position, in welcher die Rolle von der Achse um einen vorgegebenen Abstand getrennt wird, wobei die Rolle in Abhängigkeit von der Drehung der Achse drehbar ist und das gezogene Band zwischen der Achse und der Rolle stramm hält, damit das gezogene Band in der ersten oder zweiten Richtung läuft, wenn die Rolle in der ersten Position positioniert ist;

dadurch gekennzeichnet,

daß die erste Zahnradfolge die Antriebskraft auf den ausgewählten ersten oder zweiten Spulentisch überträgt, wenn die Rolle (7) in der zweiten Position positioniert ist.

2. System nach Anspruch 1, welches weiter eine Bandzuführeinrichtung (6, 7) aufweist, damit das Band wahlweise in der ersten oder zweiten Richtung läuft, wenn das Band, das aus der Kassette gezogen wird, die in das Aufzeichnungs- und/oder

Wiedergabegerät geladen ist, in einer Bandladeposition positioniert ist.

3. System nach Anspruch 2, wobei die erste Zahnradfolge die Antriebskraft auf den ausgewählten ersten oder zweiten Spulentisch überträgt, wenn das Band, das aus der Kassette gezogen ist, in der Bandladeposition positioniert ist.
4. System nach Anspruch 1, wobei das Verzögerungsverhältnis der zweiten Zahnradfolge kleiner ist als das der ersten Zahnradfolge.
5. System nach Anspruch 1, wobei die erste Zahnradfolge ein Antriebszahnrad (26) aufweist, das zwischen dem ersten und zweiten Spulentisch angeordnet ist, ein Pendelzahnrad (44), das in das Antriebszahnrad eingreift, und ein erstes und zweites Zwischenrad (46, 47), die in entsprechende Zahnradteile (3a, 4a) des ersten und zweiten Spulentisches (3, 4) eingreifen, wobei das Pendelzahnrad (44) so bewegbar ist, daß es in das erste oder zweite Zwischenrad in Abhängigkeit von der Drehung des Antriebszahnrads im ersten Betriebsmodus eingreift, und wobei die zweite Zahnradfolge das Antriebszahnrad (26) aufweist, ein Zwischenzahnrad (40), das in das Antriebszahnrad eingreift, und das erste Zwischenrad, wobei das Zwischenzahnrad (40) in das erste Zwischenrad (40) im zweiten Betriebsmodus eingreift.
6. System nach Anspruch 5, welches außerdem eine Einrichtung aufweist, um zu verhindern, daß das Pendelzahnrad (44) in Zahnradteile (3a, 4a) des entweder ersten oder zweiten Spulentisches (3, 4) im zweiten Betriebsmodus eingreift.
7. System nach Anspruch 5, wobei das erste Zwischenrad (46) ein kleineres Zahnrad (48) aufweist, welches in das Zwischenzahnrad (40) eingreifen kann, und ein größeres Zahnrad, das in den Zahnradteil (3a) des ersten Spulentisches (3) eingreifen kann, wobei der Durchmesser des größeren Zahnrads größer als der des kleineren Zahnrads (48) ist.
8. System nach Anspruch 7, wobei das Zwischenzahnrad (40) zwischen einer oberen Position bewegbar ist, in welcher das Zwischenzahnrad sich vom kleineren Zahnrad (48) des ersten Zwischenrads (46) löst, und einer unteren Position, in welcher das Zwischenzahnrad in das kleinere Zahnrad eingreift.



Revendications

1. Dispositif pour l'application d'un effort d'entraînement à un dispositif d'enregistrement et/ou de lecture possédant un premier moyen de bobine (3) sur lequel est enroulée une bande (10) se déplaçant dans une première direction et un second moyen de bobine (4) sur lequel est enroulée la bande (10) se déplaçant dans une seconde direction, opposée à la première, dispositif comprenant :

- un moyen d'entraînement (5) pour produire un effort d'entraînement;
- un premier train d'engrenage (23) pour transmettre, de façon sélective, l'effort d'entraînement produit par le moyen d'entraînement (5) vers un des premier et second moyens de bobine (3, 4) dans un premier mode de fonctionnement de telle façon que le premier ou le second moyen de bobine choisi tourne à une première vitesse;
- un second train d'engrenage pour transmettre l'effort d'entraînement produit par le moyen d'entraînement (5) au premier moyen de bobine (3) dans un second mode de fonctionnement de façon à ce que le premier moyen de bobine (3) tourne à une seconde vitesse supérieure à la première; et
- un moyen de commutation (34) pour choisir un des premier et second modes de fonctionnement pour amener le premier ou le second train d'engrenage choisi à transmettre l'effort d'entraînement au moyen de bobine correspondant;

le second train d'engrenage transmettant l'effort d'entraînement au premier moyen de bobine lorsque la bobine est placée dans une position de bobine non chargée différente de la position de chargement de bobine;

le moyen d'alimentation de bande comprenant un axe (6) entraîné en rotation par le moyen d'entraînement et un galet (7) qui est mobile entre une première position où le galet effectue un contact de poussée avec l'axe via la bande (10) dans la position de chargement de bande, et une seconde position où le galet est séparé de l'axe d'une distance prédéterminée, le galet pouvant tourner selon la rotation de l'axe et maintenant fermement la bande tirée entre l'axe et le galet de façon à provoquer un déplacement de la bande tirée dans la première ou la seconde direction lorsque le galet est placé dans la première position;

dispositif caractérisé en ce que le premier train d'engrenage transmet l'effort d'entraînement au premier ou au second

moyen de bobine choisi lorsque le galet (7) est placé dans la seconde position.

2. Dispositif selon la revendication 1, comprenant, de plus, un moyen d'alimentation de bande (6, 7) pour provoquer un déplacement sélectif de la bande dans la première ou la seconde direction lorsque la bande, qui est extraite d'une cassette chargée dans le dispositif d'enregistrement et/ou de lecture, est placée dans une position de chargement de bande.
3. Dispositif selon la revendication 2, dans lequel le premier train d'engrenage transmet l'effort d'entraînement au premier ou au second moyen de bobine choisi lorsque la bande extraite de la cassette est placée dans la position de chargement de bande.
4. Dispositif selon la revendication 1, dans lequel le rapport de réduction du second train d'engrenage est inférieur à celui du premier train d'engrenage.
5. Dispositif selon la revendication 1, dans lequel le premier train d'engrenage comprend un pignon denté menant (26) placé entre les premier et second moyens de bobine, un pignon denté pendulaire (44) coopérant avec le pignon denté menant, et des premier et second pignons dentés fous (46, 47) qui coopèrent avec des parties dentées correspondantes (3a, 4a) des premier et second moyens de bobine (3, 4), le pignon denté pendulaire (44) pouvant se déplacer pour coopérer avec un des premier et second pignons dentés fous selon la rotation du pignon denté menant dans le premier mode de fonctionnement, et dans lequel le second train d'engrenage comprend le pignon denté menant (26), un pignon denté intermédiaire (40) coopérant avec le pignon denté menant, et le premier pignon denté fou, le pignon denté intermédiaire (40) coopérant avec le premier pignon denté fou (40) dans le second mode de fonctionnement.
6. Dispositif selon la revendication 5, comprenant, de plus, un moyen pour empêcher le pignon denté pendulaire (44) de coopérer avec les parties dentées (3a, 4a) de l'un des premier ou second moyens de bobine (3, 4) dans le second mode de fonctionnement.
7. Dispositif selon la revendication 5, dans lequel le premier pignon denté fou (46) comprend une pièce de pignons dentés plus petite (48) pouvant coopérer avec le pignon denté intermédiaire (3a) du premier moyen de bobine (3), le diamètre de la plus grande pièce de pignon denté étant supé-



rieure à celui de la plus petite pièce de pignon denté (48).

8. Dispositif selon la revendication 7, dans lequel le pignon denté intermédiaire (40) peut se déplacer entre une position supérieure dans laquelle le pignon denté intermédiaire est désolidarisé de la plus petite pièce de pignon denté (48) du premier pignon denté fou (46) et une position inférieure dans laquelle le pignon denté intermédiaire coopère avec la plus petite pièce de pignon denté.

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