

DESCRIPTION
AIR CONDITIONING APPARATUS AND AIR CONDITIONING CONTROL
METHOD

Field

5 Embodiments relate to an air conditioning apparatus and an air conditioning control method.

Background

 In recent years, as information technology (IT) has been introduced in various fields, the necessity of a data
10 center that provides customers with a line connecting to a network, maintenance and operation service, and the like has increased.

 Generally, many servers are installed in a server room of a data center and therefore the amount of heat
15 generation is high. In this server room, in order to cause the servers to operate normally, it is necessary to appropriately perform air conditioning on the server room and maintain an environmental condition of a predetermined range.

20 As a technique of cooling the inside of a room in which many computers are installed such as the server room of the data center, there is an air conditioning apparatus configured to suck air from a lower space of the inside of the room and blow the air to an upper space so that the air
25 is sucked into a space above a server rack.

 Using this technique, a temperature gradient of the entire server room is reduced, and a variation in a supply air temperature is reduced, and thus air conditioning control can be efficiently performed.

30 However, when a load of an air conditioning target is an information device as in the server room of the data center, since most of loads are sensible heat loads, it is desirable that setting temperature and humidity are within a predetermined range, and since CO₂ is not generated from
35 a load, air ventilation for preventing an increase in CO₂ density is unnecessary, that is, the server room has a feature different from a state inside a general building. However, in the past, air conditioning control corresponding to this feature is not performed, and thus

there is a problem in that energy may be uselessly expended. For this reason, air conditioning apparatuses for server room management using outdoor air for implementing an air supply condition allowed in American society of heating, refrigerating and air-Conditioning engineers (ASHREA) 2008 have been already proposed. However, it is possible to further save energy using an air supply condition allowed in ASHREA 2011 in which the air supply condition on the server room is recently mitigated.

Brief Description of Drawings

FIG. 1 is an overall diagram illustrating a configuration of an air conditioning apparatus according to an embodiment.

FIG. 2 is a block diagram illustrating a configuration of an air conditioning controller of the air conditioning apparatus according to the same embodiment.

FIG. 3 is a graph illustrating an air condition range of outdoor air classified by the air conditioning controller of the air conditioning apparatus according to the same embodiment on a psychrometric diagram.

FIG. 4 is a graph illustrating an aspect of air conditioning control when an air condition corresponds to a region VI in the air conditioning controller according to the same embodiment on a psychrometric diagram.

FIG. 5 is a diagram for explaining air conditioning control when an air condition corresponds to the region VI in the air conditioning controller according to the same embodiment.

FIG. 6 is a graph for explaining an aspect of air conditioning control when an air condition corresponds to a region VII in the air conditioning controller according to the same embodiment on a psychrometric diagram.

FIG. 7 is a diagram for explaining air conditioning control when an air condition corresponds to the region VII in the air conditioning controller according to the same embodiment.

FIG. 8 is a diagram illustrating a server temperature/humidity condition of ASHRAE 2011.

Description of Embodiments

According to an embodiment, an air conditioning apparatus comprises: an indoor unit that introduces at least one of outdoor air and return air from an air conditioning control target, and discharges the introduced air to the air conditioning control target as supply air; and an air conditioning controller, wherein the indoor unit comprises a first space comprising an outdoor air introducing unit introducing the outdoor air and a cooler and a second space comprising a return air introducing unit introducing the return air from the air conditioning control target, when the air conditioning controller determines that air conditioning control on the air conditioning control target is necessary, the outdoor air is introduced to the first space through the outdoor air introducing unit, and is subjected to at least one of cooling by the cooler and dehumidification in the first space to be processed outdoor air, and the return air is introduced to the second space through the return air introducing unit, and in the second space, the processed outdoor air is mixed with the return air to be discharged to the air conditioning control target as the supply air.

<Configuration of Server Room Management System>

A configuration of a server room management system according to an embodiment will be described with reference to FIG. 1.

A server room management system 1 according to the present embodiment includes a server room 10 such as a data center and an air conditioning apparatus 20 that performs air conditioning of the server room 10. The server room 10 may be configured to be included in the air conditioning apparatus 20. In this case, the server room management system 1 of FIG. 1 corresponds to the air conditioning apparatus 20.

The server room 10 is, for example, an easily constructible container or a predetermined chamber in a building, and divided into two spaces, that is, a cold area 11 containing cold air as a third space and a hot area 14

containing warm air as a fourth space. One or more server racks 12 in which a plurality of servers (not illustrated)

are stored are installed between the cold area 11 and the hot area 14.

In the server room 10 having the above configuration, a plurality of servers are stored in one or more server racks 12 such that an air current is generated in one direction by an operation of a built-in fan as an air blower (not illustrated), or a rack fan (not illustrated) is installed in the server rack 12 such that an air current is generated in one direction near a plurality of servers arranged in the server rack 12. Further, an air current denoted by a reference numeral 13 is generated such that cold air flowing into the cold area 11 enters the server rack 12 by fan operation in the server rack 12, and air warmed by heat generation of a plurality of servers in the server rack 12 flows out to the hot area 14.

Further, the server room 10 may be configured such that a double floor having a plurality of openings is installed, and one or more server racks in which a plurality of servers are stored is installed on the double floor. In this server room 10, due to an operation of a fan installed in the server or a fan installed in the server rack, cold air (supply air) flowing into a space below the double floor serving as the cold area 11 is sucked from the opening of the double floor into the server rack above the opening. Then, the air current 13 is generated such that the sucked cold air is heated by heat generation of a plurality of servers, and flows out to the hot area 14 above the double floor as return air.

As described above, heat generated from the server is cooled by the cold air, and thus the server can normally operate.

The air conditioning apparatus 20 includes a

return air duct 21, an indoor unit 22, an outdoor unit 23, an air supply duct 24, an air conditioning controller 25, an outdoor air temperature sensor 26, an outdoor air humidity sensor 27, and an environmental sensor 28, and the
5 return air duct 21 includes a return air temperature sensor 29 and an exhaust damper 30. In FIG. 1, a signal line to the air conditioning controller 25 and a signal line from the air conditioning controller 25 are indicated by dotted lines.

10 The return air duct 21 is a tube through which air (return air) passes, and connected with the hot area 14 of the server room 10 and the indoor unit 22 of the air conditioning apparatus 20.

 The return air temperature sensor 29 measures the
15 temperature of the return air flowing in from the hot area 14, and transmits the measurement value to the air conditioning controller 25.

 The exhaust damper 30 adjusts the amount of return air by its opening degree when the return air is
20 discharged from the return air duct 21 to the outside.

 The indoor unit 22 includes a first return air introducing damper 221 serving as a return air introducing unit for introducing the return air into a space 22A (described below), an outdoor air introducing damper 222
25 serving as an outdoor air introducing unit, a filter 223, a humidifier 224 serving as a humidifying device, a cooling coil (a cooler) 225 serving as a cold air generating device, an air supply fan (an air blower) 226, a supply air temperature sensor 227 serving as a supply air temperature
30 measuring device, a supply air humidity sensor 228 serving as a supply air humidity measuring device, and a second return air introducing damper 229 serving as a return air introducing unit for introducing the return air into a

space 22B (described below). Here, for the sake of description, a space including the first return air introducing damper 221, the outdoor air introducing damper 222, the filter 223, the humidifier 224, and the cooling coil 225 is referred to as the space 22A as a first space in the indoor unit 22, and a space including the air blower 226, the supply air temperature sensor 227, the supply air humidity sensor 228, and the second return air introducing damper 229 is referred to as the space 22B as a second space in the indoor unit 22.

The first return air introducing damper 221 adjusts the amount of return air to be introduced from the return air duct 21 into the space 22A at a stage prior to the cooling coil 225 in the indoor unit 22 by its opening degree.

The outdoor air introducing damper 222 adjusts the amount of outdoor air to be introduced into the indoor unit 22 by its opening degree.

The filter 223 mainly removes dust from the outdoor air to be introduced when the outdoor air introducing damper 222 is an open state.

The humidifier 224 humidifies the outdoor air to be introduced when the outdoor air introducing damper 222 is in the open state and the return air to be introduced from the return air duct 21 when the first return air introducing damper 221 is in the open state as necessary, and generates cold air.

The cooling coil 225 cools the outdoor air to be introduced when the outdoor air introducing damper 222 is in the open state and the return air to be introduced from the return air duct 21 when the first return air introducing damper 221 is in the open state as necessary, and generates cold air.

When air is humidified by the humidifier 224 as necessary, the air supply fan 226 blasts supply air cooled and generated by the cooling coil 225 as necessary or supply air mixed with the return air flowing in from the second return air introducing damper 229, and sends the supply air to the cold area 11 of the server room 10 from the air supply duct 24. The blast volume is controlled by controlling the speed of revolution of the air supply fan 226. In the present embodiment, the air supply fan 226 causing the supply air to flow into the cold area 11 of the server room 10 is installed, but instead of the air supply fan 226, a return air fan that causes the return air to flow out of the hot area 14 of the server room 10 may be installed at the return air duct 21 side in the air conditioning apparatus 20 as an air blower that causes the supply air to flow into the cold area 11 of the server room 10 and causes the return air to flow out of the hot area 14 of the server room.

The supply air temperature sensor 227 measures the temperature of the supply air to flow into the cold area 11 of the server room 10, and transmits a measurement value to the air conditioning controller 25.

The supply air humidity sensor 228 measures humidity of the supply air to flow into the cold area 11 of the server room 10, and transmits a measurement value to the air conditioning controller 25.

The second return air introducing damper 229 adjusts the amount of return air to be introduced from the return air duct 21 into the space 22B at a stage subsequent to the cooling coil (the cooler) 225 in the indoor unit 22 by its opening degree.

The outdoor unit 23 is connected to the cooling coil 225, and supplies a coolant used to generate cold air

in the cooling coil 225 to the cooling coil 225.

The air supply duct 24 is a tube through which air (here, supply air) passes, and connects the indoor unit 22 with the cold area 11 of the server room 10.

5 The outdoor air temperature sensor 26 measures the outdoor air temperature, and transmits a measurement value to the air conditioning controller 25. The outdoor air humidity sensor 27 measures outdoor air humidity, and transmits a measurement value to the air conditioning
10 controller 25. The environmental sensor 28 measures, for example, smoke or dust of air (outdoor air) outside the indoor unit 22, and transmits a measurement result to the air conditioning controller 25.

 The air conditioning controller 25 includes a
15 supply air target range information storage unit 251, an outdoor air temperature/humidity measurement value acquiring unit 252, an air conditioning control content setting unit 253, and a device controller 254 as illustrated in FIG. 2.

20 The supply air target range information storage unit 251 stores a supply air temperature target range and a supply air humidity target range on the cold area 11 of the server room 10.

 The outdoor air temperature/humidity measurement
25 value acquiring unit 252 acquires the outdoor air temperature measurement value measured by the outdoor air temperature sensor 26 and the outdoor air humidity measurement value measured by the outdoor air humidity sensor 27.

30 The air conditioning control content setting unit 253 acquires the outdoor air temperature measurement value and the outdoor air humidity measurement value acquired from the outdoor air temperature/humidity measurement value

acquiring unit 252, the return air temperature measurement value measured by the return air temperature sensor 29, the supply air temperature measurement value measured by the supply air temperature sensor 227, and the supply air
5 humidity measurement value measured by the supply air humidity sensor 228, and sets air conditioning control content for generating the supply air within the temperature range and the humidity range which are set in advance based on the above measurement values and the
10 supply air temperature target range and the supply air humidity target range stored in the supply air target range information storage unit 251 (the details will be described later). Further, the air conditioning control content setting unit 253 sets exceptional air conditioning control
15 content (which will be described later) as necessary based on the measurement value of the smoke, the dust, or the like measured by the environmental sensor 28.

The device controller 254 controls an operation of each device in the air conditioning apparatus 20 based
20 on the air conditioning control content set by the air conditioning control content setting unit 253.

The air conditioning controller 25 may be configured with a general computer device or the like, and the supply air target range information storage unit 251
25 may be implemented as a storage device such as a memory provided in the computer device. Further, the outdoor air temperature/humidity measurement value acquiring unit 252, the air conditioning control content setting unit 253, and the device controller 254 may be implemented as a function
30 by a central processing unit (CPU) and a storage device provided in the computer device, a control program which is stored in the storage device and executed by the CPU, and an input/output interface.

<Operation of Server Room Management System>

Next, an operation of the server room management system 1 according to the present embodiment will be described.

5 In the present embodiment, the supply air temperature target range and the supply air humidity target range on the cold area 11 of the server room 10 are stored in the supply air target range information storage unit 251 of the air conditioning controller 25 of the server room management system 1 in advance.

 In the present embodiment, as the supply air temperature target range and the supply air humidity target range on the cold area 11 of the server room 10, according to the standard (see FIG. 8) of American society of heating, refrigerating and air-Conditioning engineers (ASHRAE) 2011
15 release, a range corresponding to an A1 class is employed for the temperature and the humidity, a range corresponding to an A3 class is employed for the absolute humidity (corresponding to the dew-point temperature). Specifically,
20 a range of the absolute humidity of 0.019 kg/kg (DA) or less corresponding to the temperature of 15 to 32°C, the relative humidity of 20% to 80%, and the dew-point temperature of 24°C is employed. This information is stored in the supply air target range information storage
25 unit 251. Further, any one of A1 to A4 may be set to the supply air target range information storage unit 251.

 An operation of the air conditioning controller 25 for controlling each device in the state in which the supply air temperature target range and the supply air
30 humidity target range are stored in the supply air target range information storage unit 251 as described above will be described.

 First, the air conditioning control content

setting unit 253 acquires the outdoor air temperature measurement value and the outdoor air humidity measurement value acquired by the outdoor air temperature/humidity measurement value acquiring unit 252.

5 Next, the air conditioning control content setting unit 253 sets the air conditioning control content for generating the supply air within the supply air temperature target range and the supply air humidity target range which are set in advance, based on the supply air
10 temperature target range and the supply air humidity target range stored in the supply air target range information storage unit 251. A process of setting the air conditioning control content which is executed by the air conditioning control content setting unit 253 will be
15 described below in detail.

 First, a region on a psychrometric diagram divided as illustrated in FIG. 3 to which a current outdoor air condition corresponds is determined based on the acquired outdoor air temperature measurement value and the
20 outdoor air humidity measurement value. The psychrometric diagram refers to a diagram in which a status of moist air may be seen from a temperature, absolute/relative humidity, enthalpy (here, specific enthalpy), and the like on a diagram.

25 In the psychrometric diagram of FIG. 3, an air condition of relative humidity 100% (a saturated state) or less represented by an saturation line X100 is divided into regions I to VII representing seven air condition ranges which will be described below based on a value represented
30 by the supply air temperature target range (a range represented by a temperature lower limit line Tl and a temperature upper limit line Th of FIG. 3) and the supply air humidity target range (a range represented by a

relative humidity lower limit line R_l and a relative humidity upper limit line R_h of FIG. 3), an upper limit value of enthalpy represented by an enthalpy upper limit line H_h and a lower limit value of enthalpy represented by an enthalpy lower limit line H_l which correspond to an air condition of a range corresponding to the supply air temperature target range and the supply air humidity target range, an upper limit value of absolute humidity represented by an absolute humidity upper limit line X_h , a value of absolute humidity represented by an absolute humidity lower limit line X_l , a return air temperature (T_r), and an upper limit value of outdoor air cooling enthalpy represented by an outdoor air cooling enthalpy upper limit line H_{ho} (which will be described later). In the present embodiment, an enthalpy upper limit value corresponding to the supply air temperature target range and the supply air humidity target range is assumed to be 50 kJ/kg(DA), and an enthalpy lower limit value is assumed to be 35 kJ/kg(DA).

- Region I (First Air Condition Range):

The region I refers to a range which is the same as the supply air temperature target range, the supply air relative humidity range, and the supply air absolute humidity range which serve as a target, that is, a range satisfying the following Formula (1) on the psychrometric diagram.

(temperature of 15 degree or more and 32 degree or less)

$\cap$ (relative humidity of 20% or more and 80% or less)

$\cap$ (absolute humidity of 0.019 kg/kg(DA) or less) (1)

- Region II (Second Air Condition Range):

The region II refers to a range which is less than the lower limit value of the supply air relative humidity range

serving as a target in a range of the lower limit value of the supply air temperature range or more, is less than the absolute humidity corresponding to the lower limit value of the supply air relative humidity range in a range of less than the lower limit value of the supply air temperature range, and is less than the enthalpy lower limit value in the range (that is, the region I) serving as a target, that is, a range satisfying the following Formula (2) on the psychrometric diagram.

10

$$\{(\text{temperature of 15 degree or more}) \cap (\text{relative humidity of less than 20\%}) \cap (\text{specific enthalpy of less than 35 kJ/kg(DA)})\}$$

$$\cup \{(\text{temperature of less than 15 degree}) \cap (\text{absolute humidity of less than 0.002})\} \quad (2)$$

15

- Region III (Third Air Condition Range):

The region III refers to a range which is the upper limit value of the supply air absolute humidity serving as a target or less, is a value of absolute humidity represented by the absolute humidity lower limit line X1 or more, and is less than the lower limit value of the supply air temperature range serving as a target or the upper limit value of the supply air relative humidity serving as a target or more, that is, a range satisfying the following Formula (3) on an air line.

20

$$(\text{absolute humidity of 0.002 kg/kg(DA) or more and 0.019 kg/kg(DA) or less})$$

$$\cap \{(\text{temperature less than 15 degree}) \cup (\text{relative humidity of 80\% or more})\} \quad (3)$$

30

- Region IV (Fourth Air Condition Range):

The region IV refers to a range which is within an enthalpy range corresponding to the range of the supply air

temperature and the supply air humidity serving as a target, and is less than the lower limit value of the supply air relative humidity range serving as a target or larger than the upper limit value of the supply air temperature range serving as a target, that is, a range satisfying the following Formula (4) on an air line.

$$\begin{aligned} & \text{(specific enthalpy of 35 kJ/kg(DA) or more and} \\ & \text{specific enthalpy of 50 kJ/kg(DA) or less)} \\ 10 \quad & \cap \{ (\text{relative humidity of less than 20\%}) \cup (\text{temperature} \\ & \text{of larger than 32}^\circ\text{C}) \} \quad (4) \end{aligned}$$

- Region VI (Sixth Air Condition Range):

The region VI refers to a range which is less than the upper limit value of the supply air absolute humidity range serving as a target, is larger than the upper limit value of the enthalpy range corresponding to the range of the supply air temperature and the supply air humidity serving as a target, is the return air temperature or less, and is larger than the upper limit value of the supply air temperature range serving as a target, that is, a range satisfying the following Formula (5) on air line.

$$\begin{aligned} & \text{(absolute humidity of 0.019 kg/kg(DA) or less)} \\ & \cap (\text{specific enthalpy of larger than 50 kJ/kg(DA)}) \\ 25 \quad & \cap (\text{return air temperature or less}) \\ & \cap (\text{temperature larger than 32}^\circ\text{C}) \quad (5) \end{aligned}$$

- Region VII (Seventh Air Condition Range):

The region VII refers to a range which is larger than the upper limit value of the supply air absolute humidity serving as a target, and is the upper limit value of outdoor air cooling enthalpy or less, that is, a range satisfying the following Formula (6) on an air line. The

outdoor air cooling enthalpy upper limit line H_{ho} is an enthalpy line corresponding to the return air temperature and the upper limit value of the supply air absolute humidity serving as a target, and is specified as an
 5 enthalpy line intersecting with a crossing point of the return air temperature T_r and the absolute humidity upper limit line X_h on the psychrometric diagram of FIG. 3.

(absolute humidity of larger than 0.019 kg/kg(DA))
 10 $\cap$ (upper limit value of outdoor air cooling enthalpy or less) (6)

- Region V (Fifth Air Condition Range):

The region V refers to a range other than the regions I to VII, but specifically, is a air condition range which
 15 is larger than the upper limit value of the enthalpy range corresponding to the air condition of the range corresponding to the supply air temperature target range and the supply air humidity target range, and is larger than the return air temperature or the upper limit value of
 20 the outdoor air cooling enthalpy as illustrated in FIG. 3.

Further, a condition corresponding to the region VI, the region VII, and a portion above the region I and a portion above the region III corresponds to an outdoor air condition frequently observed in a region of high-
 25 temperature and humidity such as a tropical rain forest climate province.

When a region to which the current outdoor air condition corresponds is decided from among the regions I to VII, the air conditioning control content setting unit
 30 253 sets the air conditioning control content to each region as follows using the outdoor air temperature measurement value and the outdoor air humidity measurement value acquired by the outdoor air temperature/humidity

measurement value acquiring unit 252, the return air temperature measurement value measured by the return air temperature sensor 29, the supply air temperature measurement value measured by the supply air temperature sensor 227, and the supply air humidity measurement value measured by the supply air humidity sensor 228. Here, a sensible heat load by heat generated from the servers in the server room 10 is assumed as a target of air conditioning control, a latent heat load by human breath or the like is assumed not to be generated, and the return air humidity value is the same as the supply air humidity measurement value measured by the supply air humidity sensor 228.

- Air Conditioning Control Content When Air condition corresponds to Region I:

When the current outdoor air condition is within the range of the region I, the outdoor air may be used as the supply air as is. Thus, the air conditioning control content setting unit 253 decides control content such that an outdoor air introduction rate becomes 100% by fully opening the exhaust damper 30, closing the first return air introducing damper 221 and the second return air introducing damper 229, and fully opening the outdoor air introducing damper 222. Further, at this time, the humidifying process by the humidifier 224 and the cooling process of mixed air by the cooling coil 225 are not performed.

- Air Conditioning Control Content When Air condition corresponds to Region II:

When the current outdoor air condition is within the range of the region II, the control content is decided such that the outdoor air temperature is increased by mixing the outdoor air with the return air, and humidifying is

performed. Specifically, the air conditioning control content setting unit 253 decides the control content such that the second return air introducing damper 229 is closed, then the opening degrees of the exhaust damper 30, the
 5 first return air introducing damper 221, and the outdoor air introducing damper 222 are adjusted between 0 to 100% according to the outdoor air introduction rate, and the humidifier 224 performs humidifying of a necessary amount.

At this time, a target value α of the outdoor
 10 air introduction rate is adjusted according to the outdoor air temperature measurement value and the return air temperature measurement value so that the mixed air has a supply air temperature target value which is set in advance within the supply air temperature target range and the
 15 supply air humidity target range stored in the supply air target range information storage unit 251.

An absolute humidity value X of the air in which the outdoor air is mixed with the return air by adjusting the outdoor air introduction rate as described above is
 20 represented by the following Formula (7) using an outdoor air absolute humidity measurement value X_o , a return air absolute humidity measurement value X_r , and an outdoor air introduction rate target value α .

$$25 \quad X = X_o \times \alpha + X_r \times (1 - \alpha) \quad (7)$$

Thus, in order to cause the mixed air to reach the supply air of a temperature and humidity state serving as a target, humidifying of " $X_{s0}-X$ " which is the difference between the absolute humidity value X and the supply air
 30 absolute humidity target value X_{s0} of the mixed air is necessary. A necessary humidifying amount for increasing the humidity value of the difference is " $F_{sx}(X_{s0}-X)$ " using

a supply air current amount F_s , and control content of a valve (not illustrated) for controlling a water quantity of the humidifier 224 so that the necessary humidifying amount is supplied is decided. At this time, the cooling process
 5 of the mixed air by the cooling coil 225 is not performed.

The necessary humidifying amount used to perform humidifying after the outdoor air is mixed with the return air is decided as described above, but when humidifying is performed in advance so that the return air has an absolute
 10 humidity target value $\{X_r + (X - X_{s0})/(1 - \alpha)\}$, humidifying after mixing is unnecessary.

In this case, the outdoor air introduction rate target value α is represented by the following Formula (8) using an outdoor air temperature measurement value T_o , a
 15 return air temperature value Tr_2 after humidifying, and a supply air temperature target value T_{s0} .

$$\alpha = (Tr_2 - T_{s0}) / (Tr_2 - T_o) \times 100(\%) \quad (8)$$

- Air Conditioning Control Content When Air
 20 condition corresponds to Region III:

When the current outdoor air condition is within the range of the region III, the control content is decided such that the outdoor air temperature is increased by mixing the outdoor air with the return air. Specifically,
 25 the air conditioning control content setting unit 253 decides the control content such that the opening degrees of the exhaust damper 30, one of the first return air introducing damper 221 and the second return air introducing damper 229 (the other is closed at this time),
 30 and the outdoor air introducing damper 222 are adjusted between 0 to 100% according to the outdoor air introduction rate.

In this case, the outdoor air introduction rate target value α is represented by the following Formula (9) using the outdoor air temperature measurement value T_o , the return air temperature measurement value T_r , and the supply air temperature target value T_{s0} .

$$\alpha = (T_r - T_{s0}) / (T_r - T_o) \times 100(\%) \quad (9)$$

At this time, the humidifying process by the humidifier 224 and the cooling process by the cooling coil 225 are not performed.

- Air Conditioning Control Content When Air condition corresponds to Region IV:

When the current outdoor air condition is within the range of the region IV, the control content is decided such that the outdoor air is humidified. Specifically, the air conditioning control content setting unit 253 decides the control content such that the outdoor air introduction rate becomes 100% by fully opening the exhaust damper 30, closing the first return air introducing damper 221 and the second return air introducing damper 229, and fully opening the outdoor air introducing damper 222, and the humidifier 224 performs humidifying of a necessary amount.

In this case, when an outdoor air relative humidity measurement value is R_o and a supply air humidity target value is R_{s0} , the necessary humidifying amount is " $R_{s0}-R_o$." At this time, the outdoor air temperature lowers from T_o to T as the humidifying proceeds.

- Air Conditioning Control Content When Air condition corresponds to Region V:

When the current outdoor air condition is within the range of the region V, the control content is decided such that the return air is cooled. Specifically, the air conditioning control content setting unit 253 decides the

control content such that the outdoor air introduction rate becomes 0% by closing the exhaust damper 30 and the second return air introducing damper 229, fully opening the first return air introducing damper 221 and closing the outdoor
 5 air introducing damper 222, and the cooling coil 225 performs the cooling process. At this time, the humidifying process by the humidifier 224 is not performed.

In addition, even when the current outdoor air condition is within the range of the region V, if an
 10 operator is in the server room 10 and air ventilation is necessary, the outdoor air introduction rate may be set to the lower limit value in advance, and control may be performed such that a predetermined amount of outdoor air is sucked.

15 - Air Conditioning Control Content When Air condition corresponds to Region VI:

When the current outdoor air condition is within the range of the region VI, the control content is decided such that the outdoor air is cooled. In other words, the
 20 control content is decided to be $P_3 \rightarrow P_1$ as illustrated in FIG. 4. Specifically, the air conditioning control content setting unit 253 decides the control content such that the outdoor air introduction rate becomes 100% by closing the first return air introducing damper 221 and the second
 25 return air introducing damper 229 and fully opening the exhaust damper 30 and the outdoor air introducing damper 222, and the cooling coil 225 performs the cooling process on the outdoor air (P_3) as illustrated in FIG. 5. At this time, the humidifying process by the humidifier 224 is not
 30 performed.

- Air Conditioning Control Content When Air condition corresponds to Region VII:

When the current outdoor air condition is within the

range of the region VII, the control content is decided such that the outdoor air is cooled and dehumidified, and the cooled and dehumidified outdoor air is mixed with the return air. In other words, the control content is decided
 5 such that the introduced outdoor air (P_3) is cooled and dehumidified, cooled outdoor air (P_3') is mixed with the return air (P_2), and so the supply air (P_1) has the air condition within the target region I as illustrated in FIG. 6. Specifically, the air conditioning control content
 10 setting unit 253 decides the control content such that the first return air introducing damper 221 is closed, and the exhaust damper 30, the second return air introducing damper 229, and the outdoor air introducing damper 222 are opened, but the opening degrees of the exhaust damper 30, the
 15 second return air introducing damper 229, and the outdoor air introducing damper 222 are adjusted between 0 to 100% according to the outdoor air introduction rate as illustrated in FIG. 7.

In this case, the outdoor air introduction rate
 20 target value α may be obtained by Formula (9) using the cooled outdoor air temperature measurement value as the outdoor air temperature measurement value T_o . At this time, the humidifying process by the humidifier 224 is not performed. The cooled outdoor air temperature may be
 25 estimated from the outdoor air temperature and the cooling performance of the cooling coil 225 or may be measured by a temperature sensor at an outdoor air sending side of the cooling coil 225.

- Exceptional Air Conditioning Control Content:

30 When the environmental sensor 28 detects that the outdoor air contains a lot of smoke or dust, since the filter 223 or the like degrades if the outdoor air is introduced, the control content is decided such that the

return air is cooled regardless of the temperature and humidity condition of the outdoor air. Specifically, the air conditioning control content setting unit 253 decides the control content such that the outdoor air introduction rate becomes 0% by closing the exhaust damper 30 and the second return air introducing damper 229, fully opening the first return air introducing damper 221, and closing the outdoor air introducing damper 222, and the cooling coil 225 performs the cooling process on the introduced return air. At this time, the humidifying process by the humidifier 224 is not performed.

When the air conditioning control content is set in the air conditioning control content setting unit 253 through the above-described process, the device controller 254 generates control signals used to control the opening degrees of the exhaust damper 30, the first return air introducing damper 221, the second return air introducing damper 229, and the outdoor air introducing damper 222, control variables of the humidifier 224 and the cooling coil 225, and the speed of revolution of the air supply fan 226 based on the air conditioning control content, the control signals are transmitted to the respective devices, and thus the supply air having the supply air temperature value and the supply air humidity value within the target range is generated.

Further, in the warm-temperature region or the like, the air conditioning control content may be changed according to the outdoor air condition at every season, and in tropical rain forest climate province or the like, the air conditioning control content may be changed according to the outdoor air condition within one day. In this case, the outdoor air temperature/humidity measurement value acquiring unit 252 acquires each measurement value twice or

more within one day, and the air conditioning control content setting unit 253 sets a necessary air conditioning control content with the acquisition of each measurement value by the outdoor air temperature/humidity measurement value acquiring unit 252. Further, the exceptional air conditioning control is executed as necessary based on the amount of smoke or dust measured by the environmental sensor 28.

As described above, according to the present embodiment, the air conditioning control content is changed according to the outdoor air condition, the outdoor air is used as much as possible, control suitable for server management is performed, and thus air conditioning control of managing the server with the high energy saving effect may be performed.

Further, since the air conditioning control is performed using the outdoor air as much as possible, energy necessary for air conditioning may be reduced, and the cost necessary for operating the server room management system may be reduced.

Furthermore, even when the outdoor air condition is the high temperature and high humidity as in tropical rain forest climate province, the air conditioning control content is changed according to a change in the outdoor air condition within one day, and thus the air conditioning control having the high energy saving effect for server management may be implemented.

In addition, in the air conditioning controller according to the present embodiment, when the air conditioning control content is set in the air conditioning control content setting unit 253, the optimal supply air temperature target value and the supply air humidity target value are calculated within the supply air temperature

target range and the supply air humidity target range stored in the supply air target range information storage unit 251, and the air conditioning control content having the high energy saving effect is set based on the target
 5 values.

Next, a process of calculating the supply air temperature target value and the supply air humidity target value that cause consumption energy to be minimized as the optimal supply air temperature target value and the supply
 10 air humidity target value according to the present embodiment will be described in detail.

First, energy E consumed by the air conditioning apparatus 20 may be expressed by the following Formula (10).

$$\begin{aligned}
 15 \quad E = & \text{gcol}(\text{Fs} \times (\alpha \text{Ho} + (1-\alpha) \text{Hr} - \text{Hs})) \\
 & + \text{gfan}(\text{Fs}) \\
 & + \text{ghum}(\text{Fw}) \qquad (10)
 \end{aligned}$$

where Fs: supply air current amount

Fw: humidification water quantity

20 α : outdoor air introduction rate

Ho: outdoor air specific enthalpy

Hr: return air specific enthalpy

Hs: supply air specific enthalpy

gcol: function representing relation between coil
 25 cooling quantity and cooling coil consumption energy

gfan: function representing relation between supply
 air current amount and fan consumption energy

ghum: function representing relation between
 humidification water quantity and humidifier consumption
 30 energy.

Of these, the supply water current amount Fs, humidification water quantity, the outdoor air introduction rate, the outdoor air specific enthalpy (calculated based

on the outdoor air temperature/humidity measurement value), the return air specific enthalpy (calculated based on the return air temperature measurement value and the supply air absolute humidity measurement value), the supply air
5 specific enthalpy (calculated based on the supply air temperature/humidity measurement value) may be measured or calculated based on the measurement values. Here, a sensible heat load by heat generated from a plurality of servers in the server room 10 is assumed as a target of air
10 conditioning control, a latent heat load by human breath or the like is assumed not to be generated, and the return air humidity value is the same as the supply air humidity measurement value measured by the supply air humidity sensor 228.

15 Further, the function g_{col} representing the relation between the coil cooling quantity and the cooling coil consumption energy, the function g_{fan} representing the relation between the supply air current amount and the fan consumption energy, and the function g_{hum} representing the
20 relation between the humidification water quantity and the humidifier consumption energy are known functions representing a characteristic of a device configuring the air conditioning controller 25. Thus, the energy E consumed by the air conditioning apparatus 20 may be
25 calculated based on these values.

Meanwhile, the air conditioning controller 25 according to the present embodiment decides necessary control variables of the exhaust damper 30, the first return air introducing damper 221, the second return air
30 introducing damper 229, the outdoor air introducing damper 222, the cooling coil 225, and the humidifier 224 by setting the supply air temperature target value and the supply air humidity target value in order to control the

supply air temperature and the supply air humidity by any one of the process of controlling the outdoor air introduction rate based on the opening degrees of the exhaust damper 30, the first return air introducing damper 221, the second return air introducing damper 229, and the outdoor air introducing damper 222, the process of controlling the cooling quantity of the supply air through the cooling coil 225, and the process of controlling the humidification quantity by the humidifier 224 or a combination of a plurality of processes as described above.

Further, the supply air current amount value F_s is calculated by the following Formula (11) using a return air temperature measurement value T_r , a supply air temperature measurement value T_{s0} , an air specific heat C , and a heat generation amount Q of the server.

$$F_s = Q / C / (T_r - T_{s0}) \quad (11)$$

In this case, when the supply air temperature target value, the supply air humidity target value, and the supply air current amount value are given, the necessary consumption energy E may be calculated using Formula (10).

In other words, in the air conditioning control of the server room 10 using the air conditioning controller 25 according to the present embodiment, theoretically, an operation is performed at minimum necessary energy when control is performed by a combination of the supply air temperature value, the supply air humidity value, and the supply air current amount value that cause the consumption energy E calculated by Formula (10) to be minimum.

In this regard, the supply air temperature target value and the supply air humidity target value that cause the consumption energy to be minimum in the outdoor air temperature, the outdoor air humidity, and the server load

of various conditions may be calculated in advance, and an outdoor air condition/air supply condition relation table representing a relation thereof may be generated and stored in the air conditioning control content setting unit 253.

- 5 In this case, the condition of the supply air that causes the consumption energy to be minimum may be calculated based on the condition of the measured outdoor air and the server load.

Here, the server load Q may be calculated using
 10 consumption power of the server or using the following Formula (12). When the server load Q is calculated using the following Formula (12), the air conditioning control content setting unit 253 executes the process of calculating the server load Q before the process of
 15 calculating the air supply condition based on the outdoor air condition/air supply condition relation table.

$$Q = F_s \times C \times (T_r - T_s) \quad (12)$$

where

- 20 Q : air conditioning target load
 F_s : supply air current amount value
 C : air specific heat
 T_r : return air temperature measurement value
 T_s : supply air temperature measurement value.

25 As described above, the air conditioning control content is decided using the supply air temperature target value and the supply air humidity target value which are calculated as described above.

As described above, control is performed while
 30 appropriately changing the supply air temperature setting value and the supply air humidity setting value having the high efficiency according to the change in the outdoor air condition, and thus even when the use environment of the

present system is an environment (region) of high-temperature and humidity, the air conditioning control having the high energy saving effect may be executed.

Further, in the above embodiment, the values specified
5 by the ASHRAE are used as the supply air temperature target range and the supply air humidity target range, but the present invention is not limited to this example, and other values may be used according to a status of a control target or the like. Further, a part or all of each region
10 divided as in FIG. 3 may overlap another region.

Furthermore, the above embodiment has been described in connection with the example in which the space of the server room 10 in which a plurality of servers are installed is the air conditioning control target (the air
15 conditioning target space) of the air conditioning apparatus 20, but the present invention is not limited to this example, and a space in which a power supply device or various kinds of devices that generate heat are stored or installed may be the air conditioning control target of the
20 air conditioning apparatus 20. In addition, in the above embodiment, the cold area 11 and the hot area 14 are configured to be structurally separated from each other, but the cold area 11 and the hot area 14 need not be necessarily structurally separated from each other.
25 Instead of separating the cold area 11 and the hot area 14 structurally, a side from which the supply air comes in may be configured as the cold area 11, and a side from which the return air is discharged may be configured as the hot area 14.

30 The exemplary embodiment of the present invention has been described above, but the above embodiment is an example and does not intend to limit the range of the invention. The novel embodiment can be implemented in

various forms, and omissions, replacements, and other alternations can be made in a range not departing from the gist of the present invention. The embodiment or a modification thereof is included in the range or the gist
5 of the invention, and included in the range equivalent to the invention set forth in the accompanying claims.

CLAIMS

1. An air conditioning apparatus comprising:
 - a return air duct that allows return air from an air conditioning control target to flow therethrough;
 - an indoor unit that introduces at least one of outdoor air and the return air and discharges the introduced air to the air conditioning control target as supply air; and
 - an air conditioning controller,
- wherein the indoor unit comprises
 - a first space comprising an outdoor air introducing unit introducing the outdoor air and a cooler and
 - a second space comprising a return air introducing unit introducing the return air from the air conditioning control target,
- when the air conditioning controller determines that air conditioning control on the air conditioning control target is necessary,
 - the outdoor air is introduced to the first space through the outdoor air introducing device, and is subjected to at least one of cooling by the cooler and dehumidification in the first space to be processed outdoor air,
 - the return air is introduced to the second space through the return air introducing unit, and in the second space, the processed outdoor air is mixed with the return air to be discharged to the air conditioning control target as the supply air,
 - the air conditioning controller determines that the air conditioning control on the air conditioning control target is necessary when the air conditioning controller determines that an outdoor air condition corresponds to an

air condition range that is larger than an upper limit value of absolute humidity of a predetermined target range of temperature, absolute humidity, and relative humidity, and is a value of enthalpy corresponding to the upper limit
5 value of the absolute humidity of the target range and a return air temperature or less, and

the target range is calculated so that consumption energy becomes minimized.

10 2. The air conditioning apparatus according to claim 1,
wherein the air conditioning controller determines that the air conditioning control on the air conditioning control target is necessary even when the air conditioning controller determines that an air condition of the outdoor
15 air corresponds to an air condition range that is less than the upper limit value of the absolute humidity of the target range, is larger than an upper limit value of an enthalpy range corresponding to an air condition of a range corresponding to the target range, is the return air
20 temperature or less, and is larger than a temperature upper limit value of the target range, and in this case, instead of the air conditioning control, the outdoor air is introduced from the outdoor air introducing device to the first space, is cooled by the cooler in the first space to
25 be processed outdoor air, and the processed outdoor air is discharged to the air conditioning control target as the supply air.

3. The air conditioning apparatus according to claim 1 or
30 2,

wherein the air conditioning controller acquires measurement values of outdoor air temperature, outdoor air humidity, and return air temperature twice or more within

one day, and determines whether or not air conditioning control is necessary according to acquisition of the measurement values.

5 4. The air conditioning apparatus according to any one of
claims 1 to 3, further comprising a server room, wherein
the air conditioning control target is a space within
the server room, the server room comprises a third space
and a fourth space which are separated from each other, a
10 server is installed between the third space and the fourth
space, and supply air flowing into the third space is
heated by heat generation of the server to flow out through
the fourth space as return air.

15 5. An air conditioning control method comprising:
when an air conditioning controller of an air
conditioning apparatus determines that air conditioning
control on an air conditioning control target is necessary,
the air conditioning apparatus comprising a return air duct
20 which allows return air from an air conditioning control
target to flow therethrough, an indoor unit which
introduces at least one of outdoor air and the return air
and which discharges the introduced air to the air
conditioning control target as supply air, and the air
25 conditioning controller, the indoor unit comprising a first
space comprising an outdoor air introducing unit which
introduces the outdoor air and a cooler, and a second space
comprising a return air introducing unit introducing the
return air from the air conditioning control target,
30 introducing the outdoor air to the first space through
the outdoor air introducing device, and subjecting the
outdoor air to at least one of cooling by the cooler and
dehumidification in the first space to be processed outdoor

air, and

introducing the return air to the second space through the return air introducing unit, and in the second space, mixing the processed outdoor air with the return air to
5 discharge the mixed air to the air conditioning control target as the supply air, wherein

the air conditioning controller determines that the air conditioning control on the air conditioning control target is necessary when the air conditioning controller
10 determines that an outdoor air condition corresponds to an air condition range that is larger than an upper limit value of absolute humidity of a predetermined target range of temperature, absolute humidity, and relative humidity, and is a value of enthalpy corresponding to the upper limit
15 value of the absolute humidity of the target range and a return air temperature or less, and

the target range is calculated so that consumption energy becomes minimized.