

Description

Pump arrangement

5 The invention relates to a pump arrangement, in particular magnetic clutch pump arrangement, having an interior space formed by a pump casing of the pump arrangement, having a containment can which hermetically seals off a chamber surrounded by said
10 containment can with respect to the interior space formed by the pump casing, having an impeller shaft which can be driven in rotation about an axis of rotation, having an impeller which is arranged on one end of the impeller shaft, having an inner rotor
15 arranged on the other end of the impeller shaft, having a drive motor, having a drive shaft which can be driven rotatably about the axis of rotation by the drive motor, and having an outer rotor which is arranged on the drive shaft and which interacts with the inner
20 rotor, wherein the outer rotor has a hub and a first carrier element.

Pump arrangements of said type are widely used and can be found in almost all sectors of industry. Machines of
25 the present type are also used in explosive environments. For different production and conveying installations, in particular in the chemical sector, there are particular guidelines relating to explosion protection. In such installations, use is made, on the
30 one hand, of working machines, for example pumps or turbines, as non-electrical devices, and on the other hand, of power machines, for example drive motors, as electrical devices. Proven safety standards have existed for a long time for electrical devices. Said
35 standards specify the structural measures that must be implemented in order that an electrical device can be used in the various explosive environments. In areas in which the generation of an explosive atmosphere is

possible, ignition sources, that is to say the generation of sparks as a result of friction or impact, friction heat and electrical charging, must be avoided, and possible effects of an explosion must be allowed
5 for by way of preventative and structural measures. Explosion-protected block motors, in particular standard motors of flange-type design, permit only a certain introduction of heat into the motor at the interfaces, in particular flange and shaft, such that
10 the maximum admissible temperatures of the motor are not exceeded.

It has lately become known, in the case of magnetic clutch pump arrangements, that the main introduction of
15 heat into the drive motor takes place through the drive shaft thereof, as the outer magnet carrier of the magnetic clutch is exposed both to the temperature of the media and also to the temperature increase resulting from eddy current losses. The poor heat
20 dissipation from the outer magnet carrier owing to the likewise heated pump casing has the effect that the heat energy is introduced predominantly directly into the drive shaft.

25 In DE 298 14 113 U1, said problem is circumvented by virtue of the outer rotor, referred to as driver, and the drive motor being connected in terms of drive by way of a drive means composed of a material with low thermal conductivity. A disadvantage here is the
30 expensive embodiment with an interposed outer rotor. This is because, aside from the requirement for additional components, not only the motor rolling bearing but also the deep-groove ball bearings which serve for the mounting of the outer rotor have to be
35 serviced. Furthermore, the heat barrier function exists only at the interface to the motor shaft stub. However, since the heat is introduced directly into the inner ring of the deep-groove ball bearings, expansion of the

inner ring and thus bracing of the bearing occur, consequently resulting in a reduction in service life. In the case of an embodiment which acts with coolant, the outer rotor runs in the coolant, giving rise to
5 considerable friction losses, which considerably reduce the efficiency of the pump.

It is the object of the invention to provide a pump arrangement which, in the case of an increased
10 temperature of the medium to be delivered, while simultaneously maintaining the explosion protection of the drive motor, permits a reduction in axial and radial structural space and a simplification of the assembly process.

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The object on which the invention is based is achieved in that the outer rotor has a hollow cylindrical section between the hub and the first carrier element.

20 By virtue of the fact that the hub is arranged not directly on the first carrier element but is connected via the hollow cylindrical section to the drive shaft, the introduction of heat from the outer magnet carrier into the drive shaft, and thus into the drive motor, is
25 reduced.

In one refinement of the invention, the hollow cylindrical section and the hub are of thin-walled form in relation to the first carrier element. The hollow
30 cylindrical section and the hub each have a wall with a certain wall thickness, wherein the wall thickness of the wall of the hollow cylindrical section and the wall thickness of the wall of the hub are smaller than the radius of the drive shaft, and are selected such that,
35 in all situations, reliable torsional and bending fatigue strength is ensured. This leads to a further reduction of the introduction of heat from the outer magnet carrier into the drive shaft of the drive motor.

One advantageous refinement provides that the axial fixing of the outer magnet carrier to the drive shaft is realized by way of a fastening element.

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Here, ideally, the fastening element has a first external thread on one end and has a second external thread on the end situated opposite the first external thread, wherein, between the first external thread and the second external thread, there is situated a spacer section, the outer diameter of which is greater than the outer diameter of the first external thread and of the second external thread.

15 What has proven to be particularly advantageous is a refinement in which the spacer section has, on the side close to the first external thread, a collar of increased outer diameter, whereby the fastening element can be positioned axially in an exact manner and fastened in uncomplicated fashion.

Alternatively, the spacer section may taper off conically at the side close to the first external thread.

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It is expediently provided that, in the hub, there is formed a radial threaded bore into which a screw element is screwed. Thus, when the pump arrangement is at a standstill, the hub abuts against the drive shaft at the point which is abutted against by the hub during operation. A high level of true running accuracy is achieved in this way.

Exemplary embodiments of the invention are illustrated in the drawings and will be described in more detail below. In the drawings:

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figure 1 shows the longitudinal section through a magnetic clutch pump arrangement having an outer rotor according to the invention,

5 figure 2 shows an outer rotor, corresponding to figure 1, in an enlarged illustration, and

figure 3 shows a section along the line III-III from figure 2.

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Figure 1 shows a pump arrangement 1 in the form of a magnetic clutch pump arrangement having a pump part and having an electrical part. The pump part of the pump arrangement 1 has a multi-part pump casing 2 of a
15 centrifugal pump, which pump casing comprises a hydraulics casing 3 designed as a spiral casing, a casing cover 4, a bearing carrier cage 5 and a connecting element 6.

20 The hydraulics casing 3 has an inlet opening 7 for the intake of a delivery medium and has an outlet opening 8 for the discharge of the delivery medium. The casing cover 4 is arranged on that side of the hydraulics casing 3 which is situated opposite the inlet opening
25 7. The bearing carrier cage 5 is fastened to that side of the casing cover 4 which is averted from the hydraulics casing 3. The connecting element 6 is mounted on that side of the bearing carrier cage 5 which is situated opposite the casing cover 4. A drive
30 motor 9, which forms the electrical part, is arranged on the connecting element 6 at the side situated opposite the bearing carrier cage 5.

A containment can 10 is fastened to that side of the casing cover 4 which is averted from the hydraulics casing 3, and said containment can extends at least partially through an interior space 11 delimited by the pump casing 2, in particular by the casing cover 4, by

the bearing carrier cage 5 and by the connecting element 6. The containment can 10 hermetically seals off a chamber 12, which is enclosed by said containment can, with respect to the interior space 11.

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An impeller shaft 13 which is rotatable about an axis of rotation A extends from a flow chamber 14, which is delimited by the hydraulics casing 3 and by the casing cover 4, into the chamber 12 through an opening 15 provided in the casing cover 4.

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An impeller 16 is fastened to a shaft end, situated within the flow chamber 14, of the impeller shaft 13, and an inner rotor 17 arranged within the chamber 12 is arranged on the opposite shaft end, which has two shaft sections 13a, 13b with increasing diameters in each case. The inner rotor 17 is equipped with multiple magnets 18 which are arranged on that side of the inner rotor 17 which faces toward the containment can 10.

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Between the impeller 16 and the inner rotor 17 there is arranged a bearing arrangement 19 which is operatively connected to the impeller shaft 13, which can be driven in rotation about the axis of rotation A.

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The drive motor 9 comprises a drive shaft 20. The drive shaft 20, which can be driven about the axis of rotation A, is arranged substantially coaxially with respect to the impeller shaft 13. The drive shaft 20 extends into the connecting element 6 and possibly at least partially into the bearing carrier cage 5. On the free end of the drive shaft 20 there is arranged an outer rotor 22, which bears multiple magnets 21. The magnets 21 are arranged on that side of the outer rotor 22 which faces toward the containment can 10. The outer rotor 22 extends at least partially over the containment can 10 and interacts with the inner rotor 17 such that the rotating outer rotor 22, by way of

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magnetic forces, sets the inner rotor 17 and thus likewise the impeller shaft 13 and the impeller 16 in rotation.

5 The outer rotor 22, which is illustrated on an enlarged scale in figure 2, comprises a hub 23 with an outer shell surface 24, and a hollow cylindrical section 25 formed on that side of the hub 23 which faces away from the drive motor 9, which hollow cylindrical section has
10 a cell 27 delimited by a wall 26. The outer rotor 22 furthermore comprises a flange-like first carrier element 28, which is formed or arranged on that side of the hollow cylindrical section 25 which faces toward the containment can 10, and a hollow cylindrical second
15 carrier element 29, which is formed or arranged on the first carrier element 28 and which at least partially surrounds the containment can 10 and on which the magnets 21 are arranged. The first and second carrier elements 28, 29 are illustrated as two interconnectable
20 parts, though may also be produced as one part.

The hollow cylindrical section 25 has a wall 25a with a wall thickness S1, and the hub 23 has a wall 23a with a wall thickness S2. The hollow cylindrical section 25
25 and the hub 23 are of thin-walled form in relation to the first carrier element 28. The wall thicknesses S1, S2 are much smaller than the thickness d1 of the first carrier element 28. The wall thickness S1 of the wall 25a of the hollow cylindrical section 25 and the wall
30 thickness S2 of the wall 23a of the hub 23 are selected such that, in all situations, reliable torsional and bending fatigue strength is ensured. The wall thicknesses S1, S2 are furthermore smaller than the radius r of the drive shaft 20. The wall thickness S1
35 of the wall 25a is preferably smaller than the wall thickness S2 of the wall 23a.

A passage bore 30 extends through the hub 23 into the cell 27 of the hollow cylindrical section 25 arranged between the hub 23 and the first carrier element 28, said passage bore forming a hub inner surface 31. An
5 axial groove 32 which extends parallel to the axis of rotation A is provided in the hub inner surface 31. In the drive shaft 20 there is formed a feather key groove 33 which is oriented toward the axial groove 32 and into which a feather key 34 is inserted for the
10 transmission of the motor torque to the hub 23 of the outer rotor 22. The axial fixing of the outer rotor 22 to the drive shaft 20 is realized by way of a fastening element 35.

15 The fastening element 35 has, on one end, a first external thread 37, which can be screwed into a threaded bore 36 formed on the face side of the drive shaft 20 so as to be coaxial with the axis of rotation A, and, on the end situated opposite the first external
20 thread 37, a second external thread 38. Between the first external thread 37 and the second external thread 38 there is formed a spacer section 39, the outer diameter of which is greater than the outer diameter of the first external thread 37 and of the second external
25 thread 38.

The fastening element 35 is screwed by way of the first external thread 37 into the threaded bore 36 until the spacer section 39 abuts against the face side of the
30 drive shaft 20. In the embodiment shown in figures 1 and 2, the spacer section 39 has, on the side close to the first external thread 37, a collar 40 of increased outer diameter, which collar bears against the drive shaft 20. The collar 40 is preferably of hexagonal
35 form, or has at least two wrench flats. Alternatively, the spacer section 39 may taper off conically at the side close to the first external thread 37 and come

into abutment against the conical entry region of the threaded bore 36.

The second external thread 38 extends through an opening 41 in the wall 26, wherein the spacer section 39 of the fastening element 35 is in abutment against the wall 26. The axial fixing of the outer rotor 22 to the drive shaft 20 is realized by way of a threaded nut 42 screwed onto the second external thread 38. In this way, the outer rotor 22 can be positioned axially in an exact manner and fastened in a simple manner. Furthermore, a passage bore 43 extends from one face side of the fastening element 35 to the other in order to minimize the material that transmits the heat from the outer rotor 22 into the drive shaft 20. Alternatively, instead of the passage bore 43, a blind bore may be provided which extends either from the face side close to the first external thread 37 as far as a point close to or in the spacer section 39, or from the face side close to the second external thread 38 as far as the collar 40 or beyond.

Figure 3 shows that, in the hub 23, there is formed a radial threaded bore 44 into which a screw element 45, in particular a grub screw, is screwed. That end of the screw element 45 which faces toward the drive shaft 20 is preferably of frustoconical form. The threaded bore 44 is always arranged at an angle α of approximately 35° to approximately 55° , and preferably at an angle α of 40° to 50° , and preferably at an angle α of approximately 45° , with respect to the axial groove 32 in the direction of rotation of the driven drive shaft 20, indicated here by the arrow M. If required, further threaded bores 44 (not illustrated) are provided in the hub 23 along its axial extent.

List of reference designations

1	Pump arrangement	28	First carrier
2	Pump casing		element
3	Hydraulics casing	29	Second carrier
4	Casing cover		element
5	Bearing carrier cage	30	Passage bore
6	Connecting element	31	Hub inner surface
7	Inlet opening	32	Axial groove
8	Outlet opening	33	Feather key groove
9	Drive motor	34	Feather key
10	Containment can	35	Fastening element
11	Interior space	36	Threaded bore
12	Chamber	37	First external
13	Impeller shaft		thread
13a	Shaft section	38	Second external
13b	Shaft section		thread
14	Flow chamber	39	Spacer section
15	Opening	40	Collar
16	Impeller	41	Opening
17	Inner rotor	42	Threaded nut
18	Magnet	43	Passage bore
19	Bearing arrangement	44	Threaded bore
20	Drive shaft	45	Screw element
21	Magnet		
22	Outer rotor	A	Axis of rotation
23	Hub	S1	Wall thickness of
23a	Wall		hollow cylindrical
24	Outer shell surface		section
25	Hollow cylindrical	S2	Wall thickness of
	section		hub
25a	Wall	r	Radius of drive
26	Wall		shaft
27	Cell		

Patent claims

Pump arrangement

5 1. A pump arrangement, in particular magnetic clutch
pump arrangement, having an interior space formed
by a pump casing of the pump arrangement, having a
containment can which hermetically seals off a
chamber surrounded by said containment can with
10 respect to the interior space formed by the pump
casing, having an impeller shaft which can be
driven in rotation about an axis of rotation,
having an impeller which is arranged on one end of
the impeller shaft, having an inner rotor arranged
15 on the other end of the impeller shaft, having a
drive motor, having a drive shaft which can be
driven rotatably about the axis of rotation by the
drive motor, and having an outer rotor which is
arranged on the drive shaft and which interacts
20 with the inner rotor, wherein the outer rotor has
a hub and a first carrier element,

characterized in that

25 the outer rotor (22) has a hollow cylindrical
section (25) between the hub (23) and the first
carrier element (28).

30 2. The pump arrangement as claimed in claim 1,
characterized in that the hollow cylindrical
section (25) and the hub (23) are of thin-walled
form in relation to the first carrier element
(28).

35 3. The pump arrangement as claimed in claim 1 or 2,
characterized in that the axial fixing of the
outer rotor (22) to the drive shaft (20) is
realized by way of a fastening element (35).

4. The pump arrangement as claimed in claim 3,
characterized in that the fastening element (35)
has a first external thread (37) on one end and
5 has a second external thread (38) on the end
situated opposite the first external thread (37),
wherein, between the first external thread (37)
and the second external thread (38), there is
situated a spacer section (39), the outer diameter
10 of which is greater than the outer diameter of the
first external thread (37) and of the second
external thread (38).
5. The pump arrangement as claimed in claim 4,
15 characterized in that the spacer section (39) has,
on the side close to the first external thread
(37), a collar (40) of increased outer diameter.
6. The pump arrangement as claimed in claim 4,
20 characterized in that the spacer section (39)
tapers off conically at the side close to the
first external thread (37).
7. The pump arrangement as claimed in one of claims 1
25 to 6, characterized in that, in the hub (23),
there is formed a radial threaded bore (44) into
which a screw element (45) is screwed.