

Description

The invention relates to a dry-shaving apparatus comprising a housing having a holder for a shear plate with hair-entry apertures and a cutter which is rotatable about an axis of rotation, which cutter comprises a carrier for cutting elements whose radial ends are formed with cutting edges, the shear plate comprising a central portion which is shaped as a part of a cylindrical surface and which is formed with hair-entry apertures, which central portion partly surrounds the cutter.

Such a dry-shaving apparatus is known, for example, from US-A-3,710,442.

It is the object of the invention to provide a simple cutter which can be manufactured cheaply and which provides an effective shaving action and to this end the invention is characterized in that the cutter comprises coaxial discs which are axially spaced from each other and which are formed with slots which extend in substantially radial directions, and a cutting element comprises axially projecting connecting portions by means of which a cutting element is supported into facing slots in two adjacent discs, the connecting portions being movable inside the slots in substantially radial directions.

Special embodiments are defined in the appended subsidiary Claims.

Embodiments of the invention will now be described in more detail, by way of example, with reference to the accompanying drawings.

Fig. 1 is a schematic longitudinal sectional view of a dry-shaving apparatus in accordance with the invention.

Fig. 2 is a sectional view taken on the line II—II in Fig. 1.

Fig. 3 is a schematic side view of a cutter and a shear plate.

Fig. 4 is a perspective view of another cutter.

Fig. 5 is a schematic view showing a part of the cutter of Fig. 4.

Figs. 6 and 7 are simplified longitudinal sectional views of modifications of the embodiment shown in Fig. 4.

Figs. 8 and 9 are perspective views of modifications of the shear plate.

The dry-shaving apparatus shown in Figs. 1 and 2 comprises a housing 1 with a holder 2 for a shear plate 3 and a cutter 4 which is rotatable relative to the shear plate.

The shear plate 3 comprises a central portion 5 with hair-entry apertures 6 and peripheral portions 7 and 8 by which the shear plate is secured to the holder 2. The central portion 5 partly surrounds the cutter 4.

The cutter 4 comprises a spindle 10 and two discs 11 which are rigidly connected to the spindle. Four cutting elements 12 are arranged between the discs and comprise cutting edges 14 at their ends 13. The spindle 10 is journaled in the holder 2 so as to be rotatable about the axis of rotation 10'.

Preferably, the cutting elements 12 are made of

a sheet material and comprise axially projecting connecting portions 15 which engage in slots 16 in the discs 11. The connecting portions 15 are movable in the slots 16 in substantially radial directions. As a result of this, the cutting elements 12 are also movable relative to the discs 11 over a limited distance in substantially radial directions. This enables the cutting elements to be pressed against the inner side of the shear plate 3 by the centrifugal force produced during rotation of the cutter 4. However, it is alternatively possible to urge the cutting elements 12 against the shear plate in a different way, for example by arranging pressure springs between the spindle 10 and the cutting elements 12.

For driving the cutter 4, for example in a direction of rotation as indicated by the arrow P in Fig. 2, an electric motor 17 is arranged inside the housing 1. The rotation is transmitted from the electric motor 17 to the cutter 4, for example, by means of pulleys 18 and 19, mounted on the spindle 10 and the motor shaft 20 respectively, and a drive belt 21.

Hairs which project inwards through the hair-entry apertures 6 are severed by cooperation of the central portion 5 of the shear plate with the cutting edges 14 of the cutting elements of the rotating cutter.

Fig. 3 is a schematic side view of a cutter 4 and a shear plate 3 used in the embodiment shown in Figs. 1 and 2, only one cutting element 12 and one slot 16 being shown for the sake of simplicity. The axially projecting connecting portion 15 is of rectangular cross-section, the thickness b being adapted to the width B of the slot 16, i.e. the clearance between the slot and the connecting portion being such as to enable the connecting portion to be radially movable inside the slot.

During cutting of a hair a cutting element 12 will generally experience a force K comprising a component K_1 which is directed oppositely to the local direction of rotation P . As a result of this force K_1 the cutting element 12 will be tilted slightly relative to the disc 12, the vertices 22 and 23 of the connecting portions 15 resting against the wall 16' of the slot 16. At the location of these vertices 22 and 23 the walls of the slots 16 will exert forces R_1 and R_2 respectively on the connecting portion. The forces R_1 and R_2 will give rise to frictional forces W_1 and W_2 respectively between the walls 16' of the slot 16 and the connecting portion 15, which forces counteract a displacement of the connecting portion 15 inside the slot 16. Thus, the cutting element 12 is mounted in the discs 11 in a self-locking manner, which means that the cutting element is not pressed inwards by the radial component K_2 of the force K during cutting of a hair. This self-tightening promotes an effective cutting action.

The magnitude of the couple of forces to be exerted on the cutting element by the forces R_1 and R_2 depends only on the magnitude of the force K_1 and on the arm between R_1 and R_2 . This arm is substantially equal to the height h of the connecting portion 15 and is independent of the

cation of the connecting portion inside the slot. Since the forces W_1 and W_2 are proportional to the forces R_1 and R_2 respectively the self-locking effect is also independent of the location of the cutting element 12 relative to the discs 11. This mitigates the drawback of the known construction in which the self-locking effect increases as the cutting element moves in a radially outward direction relative to the support, which may result in substantial forces between the shear plate and the cutting element and hence in damage.

Fig. 4 shows a cutter 4 comprising four discs 24, 25, 26 and 27 on the spindle 10. Between each pair of facing discs four cutting elements 12 are arranged in the same way as in the cutter shown in Figs. 1 and 2. However, for the sake of clarity only one cutting element is shown between each pair of discs. The cutting elements are regularly distributed on the discs, which means that for the discs 24 and 27 the slots 16 for the ends 15 extend along radii from the central axis or axis of rotation 10' of the spindle 10 which enclose an angle α of 90°. Obviously, the discs 25 and 26 should be formed with slots 16 for the cutting elements on opposite sides of the discs 25 and 26. These slots may be disposed along radii which enclose an angle β of 45°. In this way the cutting elements between the pair of discs 25, 26 may be offset from the cutting elements between the pairs of discs 24, 25 and 26, 27.

Preferably, the axial end portions 28 and 29 (Fig. 5) of the cutting elements overlap one another, so that at the location of the discs there is no zone of the shear plate 3 which is not covered by the cutting elements.

By means of the construction shown in Fig. 4 a cutter of arbitrary axial length can be obtained by the use of an arbitrary number of discs without the risk that as a result of an excessive length the cutting elements, which are generally made of a thin sheet material, are bent excessively by the forces K produced during cutting. Such an excessive bending would have an adverse effect on the cutting action. Moreover, long cutting elements, which are comparatively heavy in order to obtain the required rigidity, would also lead to excessive centrifugal forces.

Figs. 6 and 7 are simplified longitudinal sectional views similar to Fig. 1 showing modifications of the cutter in Fig. 4 and also showing the shear plate. In the two modifications the cutting elements are again secured to four discs 24, 25, 26 and 27. In the construction shown in Fig. 6 the cutting elements 12' are positioned between the discs 26 and 25 at a larger diameter than the adjacent cutting elements 12 on the other sides of the discs 25 and 26, so that the distance of the cutting edges 14' of these cutting elements 12' from the axis of rotation 10' is larger than that of the cutting edges 14 of the adjacent cutting elements 12. The shear plate 3 is constructed accordingly with a central portion 3_b having a larger distance from the axis of rotation 10' than the adjoining portions 3_a and 3_c.

In the embodiment shown in Fig. 7 the cutting

edges 14 of the cutting elements 12 between the discs 25 and 26 are disposed on a cylindrical surface, whilst the cutting edges 14'' of the cutting elements 12'' on the other sides of the discs 25 and 26 are situated on conical surfaces.

For the sake of simplicity the cutting elements 12, 12' and 12'' in Figs. 6 and 7 on opposite sides of the discs 25 and 26 are shown in the same plane through the axis of rotation 10'. In reality, however, the cutting elements are arranged offset from each other in the same way as in Fig. 4.

For the shear plate 3 it may also be important in such cases to take steps in order to increase the rigidity. In the constructions shown in Figs. 8 and 9, the central portions 5 of the shear plates 3 are formed with channel-shaped raised portions or ridges 30 and 31 respectively which extend between the peripheral portions 7 and 8. In the construction shown in Fig. 6 the ridges 30 extend transversely of the peripheral portions 7 and 8; in the construction shown in Fig. 7 the ridges 31 have a helical shape. This may lead to a substantial reinforcement of the shear plate, whilst the ridges may also slightly tighten the skin during shaving, which promotes catching of the hairs in the hair-entry apertures 6. Alternatively, the ridges may be provided at the location of the vertices 32 (Fig. 4) at the ends of a cutting element 12, to prevent the shear plate 3 from being damaged in the case of an inclined position of the cutting element with such a vertex 32.

Claims

1. A dry-shaving apparatus comprising a housing (1) having a holder (2) for a shear plate (3) with hair-entry apertures (6) and a cutter (4) which is rotatable about an axis of rotation (10'), which cutter (4) comprises a carrier for cutting elements (12) whose radial ends (13) are formed with cutting edges (14), the shear plate (3) comprising a central portion (5) which is shaped as a part of a cylindrical surface and which is formed with hair-entry apertures (6), which central portion (5) partly surrounds the cutter (4), characterized in that the cutter comprises coaxial discs (11) which are axially spaced from each other and which are formed with slots (16) which extend in substantially radial directions, and a cutting element (12) comprises axially projecting connecting portions (15) by means of which a cutting element (12) is supported into facing slots (16) in two adjacent discs, the connecting portions (15) being movable inside the slots in substantially radial directions.

2. A dry-shaving apparatus as claimed in Claim 1, characterized in that the axially projecting connecting portions (15) are rectangular in cross-section, the thickness of a connecting portion (15) being adapted to the width of a slot (16).

3. A dry-shaving apparatus as claimed in Claim 1 or 2, characterized in that the cutter (4) comprises more than two coaxial discs (24-27) with cutting elements (12) between every pair of facing discs.

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Fig. 4 shows a cutter 4 comprising four discs 24, 25, 26 and 27 on the spindle 10. Between each pair of facing discs four cutting elements 12 are arranged in the same way as in the cutter shown in Figs. 1 and 2. However, for the sake of clarity only one cutting element is shown between each pair of discs. The cutting elements are regularly distributed on the discs, which means that for the discs 24 and 27 the slots 16 for the ends 15 extend along radii from the central axis or axis of rotation 10' of the spindle 10 which enclose an angle α of 90°. Obviously, the discs 25 and 26 should be formed with slots 16 for the cutting elements on opposite sides of the discs 25 and 26. These slots may be disposed along radii which enclose an angle β of 45°. In this way the cutting elements between the pair of discs 25, 26 may be offset from the cutting elements between the pairs of discs 24, 25 and 26, 27.

Preferably, the axial end portions 28 and 29 (Fig. 5) of the cutting elements overlap one another, so that at the location of the discs there is no zone of the shear plate 3 which is not covered by the cutting elements.

By means of the construction shown in Fig. 4 a cutter of arbitrary axial length can be obtained by the use of an arbitrary number of discs without the risk that as a result of an excessive length the cutting elements, which are generally made of a thin sheet material, are bent excessively by the forces K produced during cutting. Such an excessive bending would have an adverse effect on the cutting action. Moreover, long cutting elements, which are comparatively heavy in order to obtain the required rigidity, would also lead to excessive centrifugal forces.

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In the embodiment shown in Fig. 7 the cutting

edges 14 of the cutting elements 12 between the discs 25 and 26 are disposed on a cylindrical surface, whilst the cutting edges 14'' of the cutting elements 12'' on the other sides of the discs 25 and 26 are situated on conical surfaces.

For the sake of simplicity the cutting elements 12, 12' and 12'' in Figs. 6 and 7 on opposite sides of the discs 25 and 26 are shown in the same plane through the axis of rotation 10'. In reality, however, the cutting elements are arranged offset from each other in the same way as in Fig. 4.

For the shear plate 3 it may also be important in such cases to take steps in order to increase the rigidity. In the constructions shown in Figs. 8 and 9, the central portions 5 of the shear plates 3 are formed with channel-shaped raised portions or ridges 30 and 31 respectively which extend between the peripheral portions 7 and 8. In the construction shown in Fig. 6 the ridges 30 extend transversely of the peripheral portions 7 and 8; in the construction shown in Fig. 7 the ridges 31 have a helical shape. This may lead to a substantial reinforcement of the shear plate, whilst the ridges may also slightly tighten the skin during shaving, which promotes catching of the hairs in the hair-entry apertures 6. Alternatively, the ridges may be provided at the location of the vertices 32 (Fig. 4) at the ends of a cutting element 12, to prevent the shear plate 3 from being damaged in the case of an inclined position of the cutting element with such a vertex 32.

Claims

1. A dry-shaving apparatus comprising a housing (1) having a holder (2) for a shear plate (3) with hair-entry apertures (6) and a cutter (4) which is rotatable about an axis of rotation (10'), which cutter (4) comprises a carrier for cutting elements (12) whose radial ends (13) are formed with cutting edges (14), the shear plate (3) comprising a central portion (5) which is shaped as a part of a cylindrical surface and which is formed with hair-entry apertures (6), which central portion (5) partly surrounds the cutter (4), characterized in that the cutter comprises coaxial discs (11) which are axially spaced from each other and which are formed with slots (16) which extend in substantially radial directions, and a cutting element (12) comprises axially projecting connecting portions (15) by means of which a cutting element (12) is supported into facing slots (16) in two adjacent discs, the connecting portions (15) being movable inside the slots in substantially radial directions.

2. A dry-shaving apparatus as claimed in Claim 1, characterized in that the axially projecting connecting portions (15) are rectangular in cross-section, the thickness of a connecting portion (15) being adapted to the width of a slot (16).

3. A dry-shaving apparatus as claimed in Claim 1 or 2, characterized in that the cutter (4) comprises more than two coaxial discs (24-27) with cutting elements (12) between every pair of facing discs.



4. A dry-shaving apparatus as claimed in Claim 3, characterized in that the cutting elements (12) on opposite sides of a disc are arranged offset from one another and partly overlap each other.

5. A dry-shaving apparatus as claimed in Claim 3 or 4, characterized in that the cutting edges (14) of the cutting elements (12) on one side of a disc have a radial distance from the axis of rotation (10') which differs from the radial distance of the cutting edges (14') of the cutting elements (12') on the other side of the disc from the axis of rotation (10').

6. A dry-shaving apparatus as claimed in Claim 3 or 4, characterized in that the cutting edges (14) of the cutting elements (12) on one side of a disc are situated on a cylindrical surface and the cutting edges (14'') of the cutting elements (12'') on the other side of the disc are situated on a conical surface.

7. A dry-shaving apparatus as claimed in any one of the preceding Claims, characterized in that the shear plate (3) is formed with reinforcement ridges (30, 31).

8. A dry-shaving apparatus as claimed in Claim 7, characterized in that the ridges (31) extend along a helical path relative to the axis of rotation.

Patentansprüche

1. Trockenrasiergerät mit einem Gehäuse (1) mit einer Hlaterung (2) für eine Scherplatte (3) mit Haardurchgangsöffnungen (6) und mit einem um eine Drehungsachse (10) drehbaren Schneidelement (4), das einen Träger für Meißel (12) aufweist, deren radiale Enden (13) Schneidkanten (14) aufweisen, wobei die Scherplatte (3) einen Zentralteil (5) aufweist, der als Teil eines Zylindermantels ausgebildet und mit Haardurchgangsöffnungen (6) versehen ist, wobei dieser Zentralteil (5) das Schneidelement (4) teilweise umgibt, dadurch gekennzeichnet, daß das Schneidelement koaxiale Scheiben (11) aufweist, die axial beabstandet und mit Schlitten (16) versehen sind, die sich in im wesentlichen radialen Richtungen erstrecken, wobei ein Meißel (12) axial herausragende Verbindungsteile (15) aufweist, durch die ein Meißel (12) in einander zugewandten Schlitten (16) in zwei nebeneinanderliegenden Scheiben gehalten wird, wobei die Verbindungsteile (15) in im wesentlichen radialen Richtungen innerhalb der Schlitte beweglich sind.

2. Trockenrasiergerät nach Anspruch 1, dadurch gekennzeichnet, daß die axial hervorragenden Verbindungsteile (15) einen rechteckigen Querschnitt haben, wobei die Dicke an die Breite des Schlittens (16) angepaßt ist.

3. Trockenrasiergerät nach Anspruch 1 oder 2, dadurch gekennzeichnet, daß das Schneidelement (4) zwischen jedem Paar einander gegenüberliegenden Scheiben mehr als nur zwei koaxiale Scheiben (24—27) mit Meißeln (12) aufweist.

4. Trockenrasiergerät nach Anspruch 3, dadurch gekennzeichnet, daß die Meißel (12) an einander gegenüberliegenden Seiten einer Scheibe gegenüber einander versetzt und einander teilweise überlappend angeordnet sind.

5. Trockenrasiergerät nach Anspruch 3 oder 4, dadurch gekennzeichnet, daß die Schneidkanten (14) der Meißel (12) auf einer Seite einer Scheibe einen radialen Abstand von der Drehungsachse (10') aufweisen, der von dem radialen Abstand der Schneidkanten (14') der Meißel (12') auf der anderen Seite der Scheibe von der Drehungsachse (10') abweicht.

6. Trockenrasiergerät nach Anspruch 3 oder 4, dadurch gekennzeichnet, daß die Schneidkanten (14) der Meißel (12) auf einer Seite einer Scheibe auf einem Zylindermantel liegen und die Schneidkanten (14'') der Meißel (12'') auf der anderen Seite der Scheibe auf einem Kegelmantel liegen.

7. Trockenrasiergerät nach einem der vorstehenden Ansprüche, dadurch gekennzeichnet, daß die Scherplatte (3) mit Verstärkungsrippen (30, 31) versehen ist.

8. Trockenrasiergerät nach Anspruch 7, dadurch gekennzeichnet, daß die Rippen (31) gegenüber der Drehungsachse sich schraubenlinienförmig erstrecken.

Revendications

1. Rasoir à sec comportant un boîtier (1) muni d'un support (2) pour un peigne (3) présentant des ouvertures d'entrée de poil (6) et d'un organe de coupe (4) pouvant tourner autour d'un axe de rotation (10') et comportant un support muni d'éléments de coupe (12) dont les extrémités radiales (13) présentent des tranchants (14), peigne (3) comportant une partie centrale (5) réalisée sous la forme d'une partie d'une surface cylindrique et dans laquelle sont pratiquées des ouvertures d'entrée de poil (6), ladite partie centrale (5) entourant partiellement l'organe de coupe (4), caractérisé en ce que l'organe de coupe comporte des disques coaxiaux (11) situés à distance l'un de l'autre dans le sens axial et présentant des rainures (16) s'étendant dans des directions sensiblement radiales, et en ce qu'un élément de coupe (12) présente des parties de liaison (15) faisant saillie dans le sens axial et par lesquelles un élément de coupe (12) est appuyé dans des rainures opposées (16) dans deux disques voisins, les parties de liaison (15) pouvant être déplacées dans les rainures dans des directions sensiblement radiales.

2. Rasoir à sec selon la revendication 1, caractérisé en ce que les parties de liaison (15) faisant saillie dans le sens axial ont une section transversale rectangulaire, l'épaisseur d'une partie de liaison (15) étant adaptée à largeur d'une rainure (16).

3. Rasoir à sec selon la revendication 1 ou 2, caractérisé en ce que l'organe de coupe (4) comporte plus de deux disques coaxiaux (24—27) munis d'éléments de coupe (12) situés entre les disques de chaque paire de disques opposés.

4. Rasoir à sec selon la revendication 3, caractérisé en ce que les éléments de coupe (12) sur des faces opposées d'un disque sont décalés les uns par rapport aux autres et se chevauchent partiellement.



5. Rasoir à sec selon la revendication 3 ou 4, caractérisé en ce que les tranchants (14) des éléments de coupe (12) sur l'une des faces d'un disque sont séparés de l'axe de rotation (10') par une distance radiale différente de la distance radiale qui sépare de l'axe de rotation (10') les tranchants (14') des éléments de coupe (12') sur l'autre face du disque.

6. Rasoir à sec selon la revendication 3 ou 4, caractérisé en ce que les tranchants (14) des éléments de coupe (12) sur l'une des faces d'un

disque sont situés sur une surface cylindrique et les tranchants (14'') des éléments de coupe (12'') sur l'autre face du disque sont situés sur une surface conique.

7. Rasoir à sec selon l'une quelconque des revendications précédentes, caractérisé en ce que le peigne (3) présente des nervures de renforcement (30, 31).

8. Rasoir à sec selon la revendication 7, caractérisé en ce que les nervures (31) s'étendent en hélice par rapport à l'axe de rotation.

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