

"An industrial process efficiency method and system "

Cross-Reference to Related Applications

The present application claims priority from Australian Provisional Patent Application No 2006900911 filed on 23 February 2006, the content of which is
5 incorporated herein by reference.

BACKGROUND OF THE INVENTION

1. Field of the Invention

10

This Invention relates to an industrial process efficiency method and system and relates particularly to a method and system for improving the efficiency and performance of any industrial process that uses motor-driven pumps to move liquids, slurries, gases and other fluid or fluid-like material. The invention has particular application in the field of
15 heating, ventilation and air-conditioning systems (HVAC) used, for example, for comfort conditioning for buildings. More specifically, the present invention is directed to methods and systems of improving the overall operating performance and efficiency of movement of fluids such as in HVAC systems.

20 While the invention has broad application throughout all areas of industry, such as in cool rooms, paper processing, water and/or sewage treatment plants, or any other system that incorporates fluid pumping and the like, for simplicity and ease of understanding the invention, it will be described herein in relation to its use in HVAC systems. It will be understood, however, that the invention is not limited to its use in
25 such systems.

2. Description of Related Art

Compression type HVAC systems and direct expansion air conditioners are the most
30 commonly used cooling systems for buildings nowadays. HVAC systems and direct expansion air conditioners operate by absorbing heat from the space being cooled either directly (direct expansion air conditioners) or by circulating a secondary fluid (e.g. water or air). Rejecting the heat that has been absorbed and has been generated by the cooling apparatus is accomplished almost universally by transferring the heat to the
35 environment outside the building or space.

Known systems typically have a motor which drives a compressor drawing low pressure refrigerant gas from an evaporator and discharging it as a higher pressure hot gas into a condenser. In the condenser, the hot gaseous refrigerant is condensed into a high pressure liquid refrigerant which flows through an expansion device that regulates the flow of refrigerant into the evaporator. The low pressure refrigerant absorbs heat of vaporization from the chilled water or air circulating through the evaporator, and low pressure refrigerant vapor is drawn back into the inlet of the compressor and the cycle is continuously repeated.

- 10 Usually such cooling apparatus has some method of regulating cooling capacity for part load operations, such as a modulating scroll or vane apparatus which limits the amount of refrigerant through the compressor, or a variable speed apparatus which controls the rotational speed of the compressor, or both. The chilled water/hot water or chilled air/warmed air is generally circulated through a distribution system for comfort conditioning within the building.

In typical systems, the heat absorbed from the evaporator, along with the heat added by the compressor, is transferred to the external air through the condenser, preferably using cooling towers or the like.

20

When water is being chilled by the evaporator to about 4.4 to 10 °C, it is then delivered by a chilled water pumps to the cooling load, which may include water cooling coils in terminals in which air is cooled and dehumidified.

- 25 In the prior art, several arrangements are used for connecting water chillers into chilled water supply and distribution systems. Further, several arrangements are used for condenser water cooling. However, all systems proposed previously suffer from inefficiencies when loadings vary from optimum design loadings.

- 30 For example, when cooling loads drop during part-load operation, the water volume flow rate of the plant loop (primary chilled water circuit) keeps in constant flow and maintain in full capacity which is recommended by all chiller configuration guidelines due to their awareness that any reduction of water flow rate in the primary water circuit may adversely affect the efficiency of chiller and may further lead to the instable operation of the chiller. In this connection, all prior arts which attempt to operate the primary chilled water pump at reduced speed in accordance with the load which will
- 35

lower the water flow rate of primary chilled water circuit with respect to its original constant flow rate arrangement is not recommended by the chiller's manufacturers.

With regard to the secondary chilled water circuit (also known as building loop), when cooling loads drop during part-load operation, the water volume flow rate reduces in the building loop because control valves have been partially closed. The water pump, therefore, is caused to operate at a loading less than full capacity and therefore at less than maximum efficiency. In prior art, someone may use a variable speed drive to operate the secondary chilled water pump at reduced speed in accordance with the load, however the water flow rate of the secondary chilled water circuit is lowered too. And the extent of energy saving is not significant when compared with our present invention.

SUMMARY OF THE INVENTION

15

It is therefore desirable to provide a method and system to improve performance and efficiency, and thereby save energy significantly in industrial systems such as application in cool rooms, paper processing, water and/or sewage treatment plants, or any other system that incorporates fluid pumping and the like.

20

It is also desirable to avoid instability in chiller /boiler controls or other industrial process operation, and thus provide for stable operation of a chiller/boiler in an industrial system such as HVAC system.

25 It is also desirable to provide a novel, improved industrial liquid circulation system control that can operate without encountering control instability no matter whether the system demand is at maximum or part-load.

It is also desirable to incorporate a plurality of pump means in parallel flow relationship to operate at equal reduced speed or almost equal reduced speed or similar reduced speed in lieu of the original/traditional inefficient pumping arrangement, whereby the operating plurality of pump means providing the optimum or same flow capacity with respect to a percentage of the flow capacity of the original/traditional pumping arrangement.

35

It is also desirable to incorporate a plurality of pump means to operate at equal reduced speed or almost equal reduced speed or similar reduced speed to accomplish a predetermined target (a certain flow rate, a certain pressure differential, a desired chilled water leaving temperature, a desired discharge pressure, etc.) or in response to a loading signals..

It is also desirable to reduce the wear and tear rate of all components of an industrial process including those of industrial liquid circulation systems to extend the service life of the respective components which include the chilled water pumps, boiler pumps, condenser water pumps, general purpose pumps and piping of the industrial liquid circulation system circuit, etc.

It is also desirable to improve the power factor of the system and thereby significantly reduce the demand charge and the associated utility cost.

In accordance with a first broad aspect of the invention there is provided an industrial system having one or more liquid circulation circuits, the system including:

- a plurality of pump means to circulate the liquid through the or each of said circuits,
- motor means driving each pump means,
- load sensing or detecting means to sense operating loads on the system and the circuits, and
- speed control means to vary the speed of said motor means to thereby vary the pumping capacity of each pump means in response to the detected load on the system.

Preferably, the plurality of pump means each operate at equal reduced speed or almost equal reduced speed or similar reduced speed in accordance with the system load, with the motor means driving each pump means, and the speed control means varies the speed of said motor means to thereby vary the pumping capacity of each pump means in response to the detected load on the system or a predetermined target. (I.e. the same flow capacity with respect to the original/traditional pumping arrangement.)

In particular embodiments, the pumps may be used in cool rooms for circulating chilled water, in paper processing systems where the pumps are used to circulate slurries, cooling water, heating water, and the like, in water and/or sewage treatment plants, and

in a wide range of other industries where liquids/fluids are pumped for a variety of purposes.

In accordance with a second aspect of the invention there is provided a refrigeration system having:

one or more chilled water, condenser water, and boiler water circuits,
a plurality of pump means of identical or similar capacity to circulate water through the or each of said circuits. The plurality of pump means to be running at equal reduced speed or almost equal reduced speed or similar reduced speed, motor means driving each pump means,
load detecting means to sense operating load on the system,
speed control means to vary the speed of said motor means the thereby vary the pumping capacity of each pump means in response to the detected load or a predetermined target on the system. (I.e. the same flow capacity with respect to the original/traditional pumping arrangement.)

In preferred embodiments, the efficiency of the HVAC system's water distribution circuit, which includes the chilled water circuit, boiler water circuit, and condenser water circuit, is improved by controlling two or more pumps for each circuit at equal reduced speed or almost equal reduced speed or similar reduced speed in accordance with system load or a predetermined target.

Preferably, embodiments of the invention employ a plurality of variable speed drives to operate the associated pumping components. Cooling system loading is indicated, i.e. by measuring the present power consumption of operating chiller(s)' compressor(s) using a power sensor, or the speed of the operating chiller(s)' compressor(s) using a tachometer or measuring the discharged chilled water/boiler temperature by temperature sensor located in an appropriate location in the water circuit or some other means from which loading can be inferred or pursuing a predetermined target (i.e. operate at the same flow capacity with respect to the original pumping arrangement's operating flow capacity) This strategy sets operation of the plurality of condenser water pumps, chilled water pumps and boiler pumps running at their respective circuit's predetermined equal reduced speed or at almost equal reduced speed or similar reduced speed and can deliver the optimum or same flow capacity with respect to the original/traditional designed pumping arrangements or at a power setting that is a fixed

ratio of the cooling system current power ratio or loading (percent of maximum and subject to limits).

Preferably, the method includes the control and integrating of the plurality of chilled
5 water pumps, boiler water pumps and condenser water pumps operating simultaneously at respective circuit's predetermined equal reduced speed or almost equal reduced speed or similar reduced speed in response to the loading level of the circulating system.

10 The general formula employed in embodiments of this invention for setting the power set point for each device may be expressed as:

$$PR-SP.sub.pd = C * PR.sub.load$$

Where PR-SP.sub.pd is the power ratio (percent of maximum) set point of the
15 respective pumping devices being controlled.

Where PR.sub.load is the current loading/power ratio (percent of maximum) that is being utilized by a circuit or system or a HVAC system/air conditioning compressor(s) or apparatus(s).

20

Where C is a selected constant.

The above equation has a low limit to prevent a power ratio set point being so low as to result in all fluid flow, e.g., all air or water flow, ceasing, and a high limit to ensure the
25 power ratio set point would never rise above undesired flow rates and pressure outputs.

According to a further aspect of the invention particularly adapted for use with HVAC system there is provided a variable capacity, compression type, chilled fluid cooling system comprising:

30 a heat absorbing circuit (also known as chilled water circuit) including:

two or more chilled water pumps
a variable-frequency drive circuit for powering each of the chilled water pumps;
a chiller with evaporator operatively coupled to a numbers of cooling loads,
and a suction line leading back to said chiller;

35 water passing through said chiller in a heat exchange relationship and being cooled;
motor means for driving said chilled water pumps;

means for varying the speed of said motor means;

means for sensing the temperature of water leaving said chiller; and

means for controlling the variable-speed drive circuit in response to the present load on the compressor or a predetermined target so as to regulate operation of the variable

5 speed pumps and simultaneously running at equal reduced speed or almost equal reduced speed or similar reduced speed in response to the load on the compressor or a predetermined target.

Preferably, the chiller includes means for regulating the flow of refrigerant gas through
10 the compressor, and the means for determining a present load on the compressor makes that determination in response to a present state of the gas flow regulating means in the compressor.

In one form, the means for determining a present load on the compressor makes that
15 determination in response to a level of power applied to the compressor motor or the sensed temperature of water leaving said chiller. Alternatively, the means for controlling said means for varying the speed of said variable speed pumps simultaneously at equal reduced speed or at almost equal reduced speed or at similar reduced speed acts responsive to the manual judgment of skilled personnel.

20

Preferably, the means for determining a present load on the compressor and the means for controlling the variable speed drive circuit is configured to regulate the variable speed drive circuit at the predetermined percentage of full power thereby running the chilled water pumps simultaneously at predetermined equal reduced speed or at almost
25 equal reduced speed or at similar reduced speed to save power while system loading is at maximum or below maximum loading.

In one embodiment, the pumps comprise variable speed pumps powered by corresponding variable-speed drive circuits; and further comprising:

30 means for controlling the variable speed drive circuit in response to the present load on the compressor or a predetermined target so as to regulate operation of the chilled water pumps simultaneously at equal reduced speed or at almost equal reduced speed or at similar reduced speed in response to the load on the compressor or a predetermined target.

35

In another form, the variable speed drive circuits are connected to power the chilled water pumps, thereby regulating operation of said two or more pumps simultaneously at equal reduced speed or at almost equal reduced speed or at similar reduced speed responsive to loading on the compressor or a predetermined target.

5

According to a further aspect of the invention there is provided a system comprising:

a heat absorbing circuit (also known as chilled water circuit) including:

three or more chilled water pumps,

a variable-speed drive circuit for powering each of the chilled water pumps;

10 two chillers with evaporators being programmed to one is kept operating and the other remains shutdown/standby in response to a certain degree of part load or two are kept operating in response to a certain degree of full load operatively coupled to a numbers of cooling loads; and

and a suction line leading back to said chillers;

15 water passing through said chillers in a heat exchange relationship and being cooled; motor means for driving said two or more predetermined operating chilled water pumps;

means for varying the speed of said two or more operating motors means;

means for sensing the temperature of water leaving said operating chiller(s);

20 means for controlling the operating variable-speed drive circuits in response to the present load on the operating compressor or a predetermined target so as to regulate operation of the two or more predetermined chilled water pumps simultaneously running at equal reduced speed or almost equal reduced speed or similar reduced speed in response to the load on the operating compressor or a predetermined target.

25

Preferably, in accordance with this embodiment, the chillers include means for regulating the flow of refrigerant gas through the compressors, and the means for determining a present loads on the compressors makes that determination in response to a present state of the gas flow regulating means in the compressors.

30

Preferably, the means for determining a present load on the operating compressor(s) makes that determination in response to a level of power applied to the operating compressor(s)' motor(s) or in response to the sensed temperature of water leaving said operating chiller.

35

Preferably, the means for controlling said means for varying the speed of said two or more predetermined operating chilled water pumps simultaneously at equal reduced speed or at almost equal reduced speed or at similar reduced speed act responsive to the manual judgment of skilled personnel.

5

In one arrangement, the means for determining a present load on the operating compressor and the means for controlling the operating variable-speed drive circuit are configured to regulate the operating variable-speed drive circuits at the predetermined percentage of full power thereby running the two or more operating chilled water
10 pumps simultaneously at predetermined equal reduced speed or at almost equal reduced speed or at similar reduced speed to save power while system loading is at maximum or below maximum loading.

According to another aspect of the invention there is provided a variable capacity,
15 compression type, chilled fluid cooling system comprising:

a heat rejection circuit (also known as condenser water circuit) including:

at least two condenser water pumps

a variable-frequency drive circuit for powering each of the condenser water pumps;

a chiller with condenser operatively coupled to a number of cooling towers

20 and a suction line leading back to said chiller with condenser;

water passing through said chillers with condenser in a heat exchange relationship and being heated;

motor means for driving said two or more operating condenser water pumps;

means for varying the speed of said motors means;

25 means for sensing the temperature of water leaving said chillers; and

means for controlling the variable-speed drive circuits in response to the present load on the compressor or a predetermined target so as to regulate operation of the two or more operating condenser water pumps simultaneously running at equal reduced speed or at almost equal reduced speed or at similar reduced speed in response to the loading

30 on the operating compressor(s) or a predetermined target.

In accordance with a still further aspect of the invention there is provided a method of operating an industrial system having one or more water, or other liquid circulation circuits, comprising the steps of providing a plurality of pump means to circulate water
35 through the or each of said circuits, operating motor means to drive each pump means, sensing operating loads on the system and the circuits, and vary the speed of said motor

means the thereby vary the pumping capacity of each pump means in response to the sensed load on the system or a predetermined target.

According to a further aspect of the invention there is provided a method of operating a
5 variable capacity, compression type cooling system having a plurality of chilled water
pumps, condenser water pumps, boiler pumps, the method comprising the steps of
determining a present load level of the compressor or a pursuing a predetermined target
and regulating operation of the respective plurality of primary chilled water pumps,
secondary chilled water pumps, boiler pumps, condenser water pumps running at
10 respective circuit's predetermined equal reduced speed or at almost equal reduced
speed in response to the present load level of the compressor(s) or pursuing a
predetermined target.

According to another aspect of the invention there is provided an industrial fluid
15 circulating system having one or more fluid circulation circuits, the system including:
a plurality of pump means to circulate the fluid through the or each of said
circuits,
motor means driving each pump means,
load sensing or detecting means to sense operating loads on the system and the
20 circuits, and
speed control means to vary the speed of said motor means to thereby vary the
pumping capacity of each pump means in response to the detected load on the system.

According to a further aspect of the invention there is provided a method of operating
25 an industrial system having one or more water, or other fluid circulation circuits
comprising the steps of:
providing a plurality of pump means to circulate fluid through the or each of said
circuits;
operating motor means to drive each pumps means,
30 sensing operating loads on the system and the circuits; and
varying the speed of said motor means at respective circuit's predetermined equal
reduced speed or at almost equal reduced speed or at similar reduced speed to thereby
vary the pumping capacity of each pump means in response to the sensed load on the
system; or in response to a predetermined target; or in response to manual judgment of
35 skilled personnel.

According to a still further aspect of the invention there is provided an industrial fluid circulating system having a plurality of fluid circulation circuits, the system including:

a variable capacity, compression type, chilled fluid cooling system comprising:
a single circuit chilled water system including:

- 5 at least two chilled water pumps;
- a chiller with evaporator operatively coupled to a numbers of cooling loads
and a suction line leading back to said chiller;
- water passing through said chiller in a heat exchange relationship and being
cooled;
- 10 motor means for driving said chilled water pumps;
- a variable-speed drive circuit for powering each of the motor means to vary the
speed of the chilled water pumps;
- means for sensing the temperature of water leaving said chiller; and
- means for controlling the variable-speed drive circuit in response to the present
- 15 load on a system compressor so as to regulate operation of the said at least two chilled
water pumps running at equal reduced speed or at almost equal reduced speed or at
similar reduced speed simultaneously in response to the load on the compressor.

In order that the invention is more readily understood, embodiments thereof will now
20 be described with reference to the accompanying drawings.

BRIEF DESCRIPTION OF THE DRAWINGS

FIG. 1 is a schematic of a typical prior art HVAC system with a water-cooled
25 condenser

FIG. 2 is a schematic of a typical prior art HVAC system with a plant-through building
loop (also known as "single circuit") chilled/boiler water system.

30 FIG. 3 is a schematic of a typical prior art HVAC system with a plant-building loop
(also known as "primary-secondary water circuit") chilled/boiler water system.

FIG. 4 is a schematic of a typical prior art HVAC system with a condenser water circuit

FIG. 5A is a schematic of a typical HVAC system with a plant-through building loop (also known as “single circuit”) chilled/boiler water system in accordance with one embodiment of the invention,

- 5 FIG. 6A is a schematic of a typical HVAC system with a plant-through building loop (also known as “single circuit”) chilled/boiler water system in accordance with another embodiment of the invention,

- FIG. 7A is a schematic of a typical HVAC system with a plant-building loop (also
10 known as “primary-secondary circuit”) chilled/boiler water system in accordance with another embodiment of the invention,

- FIG. 8A is a schematic of a typical HVAC system with a plant-building loop (also
known as “primary-secondary circuit”) chilled/boiler water system in accordance with
15 another embodiment of the invention,

- FIG. 9A is a schematic of a typical HVAC system with a plant-building loop (also
known as “primary-secondary circuit”) chilled/boiler water system in accordance with
another embodiment of the invention,

20

FIG. 10A is a schematic of a typical HVAC system with a plant-building loop (also
known as “primary-secondary circuit”) chilled/boiler water system in accordance with
another embodiment of the invention,

- 25 FIG. 11A is a schematic of a typical HVAC system with a plant-building loop (also
known as “primary-secondary circuit”) chilled/boiler water system in accordance with
another embodiment of the invention,

- FIG. 12A is a schematic of a typical HVAC system with a condenser water circuit in
30 accordance with another embodiment of the invention,

FIG. 13A is a schematic of a typical HVAC system with a condenser water circuit in
accordance with another embodiment of the invention, and

- 35 FIG. 14A is a schematic of a typical HVAC system with a condenser water circuit in
accordance with another embodiment of the invention,

Prior Art

FIG. 1 illustrates the major components of a typical compression cycle cooling apparatus. In this system, a motor (101) drives the compressor (102), which draws low pressure refrigerant gas from the evaporator (103) through a suction line (104), compresses it, and discharges it as a higher pressure hot gas through a hot gas line (105) into the condenser (106). In the condenser, the hot gaseous refrigerant is condensed into a liquid by rejecting heat to outside air by blowing outside air across the condenser with a fan driven by an electric motor (118) (not shown here) or by rejecting heat to tepid water from a cooling tower (107) through a condenser water circuit (116) which employs condenser water pump(s) (117). The condensed liquid refrigerant flows through an expansion device (109) that regulates the flow of refrigerant into the evaporator (103), which is held at a low pressure by the operation of the compressor. The low pressure environment causes the refrigerant to change state to a gas and as it does so, it absorbs the required heat of vaporization from the chilled water or air circulating through the evaporator, entering at (110) and leaving at (111). The low pressure refrigerant vapor is drawn into the inlet of the compressor and the cycle is continuously repeated. Usually such cooling apparatus has some method of regulating cooling capacity for part load operations such as a modulating scroll or vane apparatus (112) which limits the amount of refrigerant through the compression device, or a variable speed apparatus (113) which controls the rotational speed of the compression device, or both. The chilled water/hot water or chilled air/warmed air is circulated through a distribution system (115) for comfort conditioning, or to provide cooling for certain processes within the building. In this circuit (115), the heat absorbed from the evaporator (103) along with the heat added by the compressor (102) are rejected to the outside air through condenser fan(s) (not shown here) or cooling tower(s) (107).

The chilled water being cooled by the evaporator (103) (about 4.4 to 10 °C) is then delivered by the chilled water pump(s) (114) to the cooling load(s) (108) which include: water cooling coils in AHUs and terminals in which air is cooled and dehumidified. After flowing through the cooling load(s) (108), the chilled water increases in temperature up to (about 15.6 to 18.3 °C) and then returns to the evaporator (103).

In the prior art, there are several arrangements for connecting water chillers into chilled water supply and distribution systems. Fig. 2 illustrates a typical plant-through building loop system (200) (also known as "single circuit system"). The plant-through building loop system include: chilled water, hot water or dual-temperature water systems in which water is transported only by plant (chiller/boiler) pump(s). A plant-through-building loop water system using bypass throttling flow is one of the older chilled/hot water systems that have been adopted in commercial buildings since the use of two-way control valves. For each chiller/boiler (201), a corresponding plant constant-speed water pump (202) is installed. The chilled or hot water is supplied to the coils and terminals (204,205 and 206) through the supply (207) and return (208) mains and branches, and is then returned to the chiller/boiler (201). There is a crossover bridge (209, sometimes called a common pipe) that connects the supply and return mains at junctions (210) and (211). A bypass two-way control valve (212) is often installed on the crossover bridge. A pressure-differential transmitter (213) and pressure relief valve are used to maintain a set pressure differential across the supply and return mains by modulating the opening of the bypass two-way control valve (212) when the system pressure tends to increase during part-load operation. A portion of the water is throttled in the control valve and flows through the bypass crossover (209). It is then combined with water from the return main (208) and returns to the chiller/boiler (201). A constant flow (or approximately constant flow) is maintained in the chiller/boiler (201).

A plant-through-building loop using bypass throttling control cannot save much pumping energy during part-load operation; it sometimes consumes even more energy. The surplus energy which is more than required is dissipated by mechanical means. Plant-through-building loop using bypass throttling control still has applications in small projects and especially in retrofit where space may not be available for a plant-building loop system.

Fig.3 illustrates another prior art chiller arrangement: Plant-building loop (300) water systems. This is also called Primary-Secondary loop (or circuit) water systems, and is the widely adopted water systems for large commercial HVAC installations in the world today. A plant-building loop (300) include: chilled water, hot water, or dual-temperature water system consists of two piping loops, namely, plant loop (primary loop) (301) and building loop (secondary loop) (302).

In plant loop (primary loop) (301), there are chillers (303,304), plant pumps (primary chilled water pumps) (305, 306,), diaphragm expansion tank, corresponding pipes and fittings, and control systems. A constant volume flow rate is maintained in the evaporator of each chiller (303,304). The chilled water volume flow rate in the plant
5 loop (301) will vary when a chiller (303 or 304) and its associated chiller pump (305, 306,) are turned on or off.

In building loop (secondary loop) (302) there are cooling loads (312,313), building pumps (secondary chilled water pumps) (314), two-way control valves (325,326) and
10 control systems, and corresponding pipes, fittings, and accessories.
The water flow in the building loop (302) is varied as the coil load is changed from the design load to part-load.

A short common pipe (321), sometimes also called a bypass, connects these two loops
15 (301 and 302) and combines them into a plant-building loop (300). The common pipe (321) ensures that differences in flows between the primary and secondary water circuits will not affect the operation of either circuit. The common pipe (321) serves as a bypass for both circuits, which is needed to maintain constant flow in the primary circuit.

20 At design load, chilled water leaving the chillers (303,304) at point (322) flows through the junction of the common pipe (321), plant loop (301), and building loop (302) at point (323), is extracted by the building pump (secondary chilled water pump) (314), and is supplied to the cooling loads (312,313). From the cooling loads (312,313),
25 chilled water returns through another junction of the building loop (302) at point (324). There is only a very small amount of bypass chilled water in the common pipe that flows in the direction from point (323) to (324). The chilled water returned from the cooling loads (312,313) is then combined with the bypass water from the common pipe (321) and bypass line (330) and is extracted by the plant pump(s) (305,306) and enters
30 the chiller(s) (303,304) for cooling again.

When the cooling loads (312,313) drop during part-load operation, the water volume flow rate reduces in the building loop (302) because the control valves (325,326) have been partially closed. Chilled water then divides into two flows at the junction (328).
35 One is supplied to the cooling loads (312,313); the remaining water bypasses the building loop by flowing through the bypass line (330) through valve (327) which is

controlled by a differential pressure sensor (329), is extracted by the plant pump(s) (305,306) and returns to the chiller(s) (303,304).

Fig 4 illustrates a prior art condenser water circuit (400) of HVAC system. In this
5 condenser water circuit (400), the heat generated by various cooling apparatus, i.e. compressor (404) and cooling load(s) (406) are absorbed by condenser water at point (407), the condenser water is then forced through the cooling tower (402) by condenser water pump (409), and then back to condenser (401) at point (408) again. In this condenser (401), tepid water from the sump of a cooling tower (402) is circulated
10 through the condenser (401) by a pump (409), and then to spray nozzles or distribution flume which distributes the water over slats or plastic fill that breaks the water up into droplets with a very large surface area such that a fan driven by an electric motor forces air over the water, evaporating a portion of it to provide adiabatic cooling of the water. The cooled water gathers in the sump where the water lost through evaporation is made
15 up by adding water from a water supply. The level of water in the tower sump is maintained by a water level sensor, which operates a valve. Water in the sump is drawn through the condenser (401) to provide continuous rejection of the heat absorbed from the cooling loads through the evaporator (403) and that generated by the cooling apparatus, such as the compressor (404).

20

DETAILED DESCRIPTION OF PREFERRED EMBODIMENTS

Fig. 5A illustrates one embodiment of this invention utilizing variable speed control of two chilled water pumps/boiler pumps operating at predetermined equal reduced speed
25 or at almost equal reduced speed or at similar reduced speed in lieu of the original/traditional pumping arrangement as shown in Fig.2. A chilled water/boiler circuit (500A) (plant-through building loop system, also known as "single circuit system"), in this case, two (chilled water/boiler) pumps (502A and 503A) of identical or similar capacity are deployed to transport the produced chilled water/hot water
30 throughout the water circuit (500A). Said two pumps (502A and 503A) are connected in parallel and operate in predetermined equal reduced speed or almost equal reduced speed or similar reduced speed simultaneously. Said two pumps (502A and 503A) and respective motors are controlled by two variable speed drives (512A and 513A) and the two drives are commanded by a Controller (514A). The Controller (514A), monitors
35 the present loading on the water system (500A) either by measuring the present power consumption of operating chiller's compressor using a power sensor, or the speed of

the operating chiller's (501A) compressor using a tachometer or measuring discharged chilled water/boiler temperature by temperature sensor located in appropriate location in the water circuit or some other means from which loading can be inferred. Based on that loading signal (515A), an algorithm calculates the optimum power loading (subject to limits) for the pumps and sends an output signal (516A) to the variable speed drives (512A and 513A) operating the (chilled water/boiler water) pumps and corresponding motors (502A and 503A) such that the pumps' motors (502A and 503A) were incorporated to run at equal reduced speed or almost equal reduced speed or similar reduced speed simultaneously and operate at the optimum power percentage (or ratio) of maximum power draw based on the present percentage (or ratio) of maximum loading on the water circuit (500A).

In the event of VSD (512A or 513A) or chilled water pumps/boiler pumps (502A or 503A) and corresponding motors' failure, an integral bypass switch (not shown here as it is irrelevant to the present invention), can be operated either manually or automatically to allow the operative motor to be operated at full speed. At the same time, an alarm signal will be shown and sent to an appropriate location as required alerting appropriate personnel to take necessary actions.

Fig. 6A illustrates second embodiment of this invention utilizing variable speed control of three chilled water pumps/boiler pumps operating at predetermined equal reduced speed or almost equal reduced speed or similar reduced speed in lieu of the original/traditional designs' pumping arrangement as shown in Fig. 2. A chilled water/boiler circuit (600A) ("plant-through building loop system", also known as "single circuit system"), in this case, three (chilled water/boiler) pumps (602A, 603A, 617A) of identical or similar capacity are deployed to transport the produced chilled water/hot water throughout the water circuit (600A). Said three pumps (602A, 603A, and 617A) are connected in parallel and operate in predetermined equal reduced speed or almost equal reduced speed or similar reduced speed simultaneously. Said three pumps (602A, 603A, 617A) and respective motors are controlled by three variable speed drives (612A, 613A, 618A) and the three drives are commanded by a Controller (614A). The Controller (614A), monitors the present loading on the water system (600A) either by measuring the present power consumption of operating chiller's compressor using a power sensor, or the speed of the operating chiller's (601A) compressor using a tachometer or measuring discharged chilled water/boiler temperature by temperature sensor located in appropriate location in the water circuit or some other means from

- which loading can be inferred. Based on that loading signal (615A), an algorithm calculates the optimum power loading (subject to limits) for the pumps and sends an output signal (616A) to the variable speed drives (612A, 613A, 618A) operating the (chilled water/boiler water) pumps and corresponding motors (602A, 603A, 617A)
- 5 such that the pumps' motors (602A, 603A, 617A) were incorporated to run at equal reduced speed or almost equal reduced speed or similar reduced speed simultaneously and operate at the optimum power percentage (or ratio) of maximum power draw based on the present percentage (or ratio) of maximum loading on the water circuit (600A).
- 10 In the event of VSD (612A, 613A, 618A) or chilled water pumps/boiler pumps (602A, 603A, 617A) and corresponding motors' failure, an integral bypass switch (not shown), can be operated either manually or automatically to allow the operative motor to be operated at predetermined speed. At the same time, an alarm signal will be shown and sent to an appropriate location as required alerting appropriate personnel to take
- 15 necessary actions.

Fig. 7A illustrates third embodiment of this invention utilizing variable speed control of two primary chilled water/boiler pumps operating at predetermined equal reduced speed or almost reduced speed or similar reduced speed in lieu of the

20 original/traditional pumping arrangement as shown in Fig.3 when one chiller is operating. A chilled water/boiler circuit (700A) ("plant-building loop system" also known as "primary-secondary chilled/boiler water circuit"), in this case, in the plant loop (701A), two plant pumps; in the combination of (705A and 706A) or (705A and 707A) or (706A and 707A) of identical or similar capacity are deployed to transport the

25 produced chilled/boiler water through the plant loop water circuit (701A) (also known as "primary loop") if only one chiller/boiler (703A or 704A) is operating at the mean time and the other remains on standby in accordance with the current loading.

The respective two predetermined pumps (the predetermined combination of 705A, 706A, and 707A) are connected in parallel and operate in predetermined equal reduced

30 speed or almost equal reduced speed or similar reduced speed simultaneously. The pumps and the corresponding driving motors are controlled by corresponding variable speed drive (VSD) (730A corresponds to plant pump 705A, 731A corresponds to plant pump 706A, 732A corresponds to plant pump 707A) and the said drives (730A, 731A, 732A) are commanded by a Controller (733A). The Controller (733A), monitors the

35 present loading on the water system either by measuring the present power consumption of operating chiller's (703A or 704A) compressor using a power sensor,

or the speed of the operating chiller's (703A or 704A) compressor using a tachometer or measuring discharged chilled/boiler water temperature by temperature sensor located in appropriate location in the water circuit or some other means from which loading can be inferred. Based on that loading signal (734A), an algorithm calculates the optimum
5 power loading (subject to limits) for the pumps and sends an output signal (735A) to the respective two predetermined variable speed drives (730A,731A ,732A) operating the two corresponding plant pumps (said combination of 705A,706A,707A) such that the two corresponding pumps (said combination of 705A,706A,707A) were incorporated to run at equal reduced speed or almost equal reduced speed or similar
10 reduced speed simultaneously and operate at the optimum power percentage (or ratio)(subject to limits) of maximum power draw based on the present percentage (or ratio) of maximum loading (subject to limits) on the water circuit(700A).

As above, in the event of variable speed drive(s) or plant pump(s) failure, an integral
15 bypass switch (736A, not shown here), can be operated either manually or automatic, will allow said pumps to be operated at full speed. At the same time, an alarm signal will be shown and sent to an appropriate location as required alerting appropriate personnel to take necessary actions.

20 Fig. 8A illustrates a fourth embodiment of this invention utilizing variable speed control of three primary chilled water/boiler pumps operating at predetermined equal reduced speed or at almost equal reduced speed or at similar reduced speed in lieu of the original/traditional designed pumping arrangement as shown in Fig.3 when one chiller (803A or 804A) is operating. A chilled water/boiler circuit (800A) (plant-
25 building loop system) (also known as primary-secondary chilled/boiler water circuit), In this case, in the plant loop (801A), three plant pumps (805A, 806A and 807A) of the same or near the same capacity are deployed to transport the produced chilled/boiler water through the plant loop water circuit (801A) (primary loop) if one chillers/boilers(803A or 804A) is operating at the mean time in accordance with the
30 current loading.

The three pumps (805A, 806A and 807A) are connected in parallel and operate in predetermined equal reduced speed or almost equal reduced speed or similar reduced speed simultaneously in lieu of the original/traditional designed pumping arrangement
35 as shown in Fig.3. The three pumps (805A, 806A and 807A) and corresponding motors are controlled by respective variable speed drives (VSD830A corresponds to plant

pump 805A, VSD831A corresponds to plant pump 806A, VSD 832A corresponds to plant pump 807A) and the said drives (830A, 831A, 832A) were commanded by a Controller (833A). The Controller (833A), monitors the present loading on the water system either by measuring the present power consumption of operating chiller' (803A or 804A) compressor using a power sensor, or the speed of the operating chiller' (803A or 804A) compressor using a tachometer or measuring the discharged chilled water/boiler temperature by temperature sensor located in appropriate location in the water circuit or some other means from which loading can be inferred. Based on that loading signal (834A), an algorithm calculates the optimum power loading (subject to limits) for the pumps and sends an output signal (835A) to the three variable speed drives (830A, 831A, 832A) operating the three corresponding plant pumps (805A, 806A ,807A) such that the three plant pumps (805A, 806A and 807A) were incorporated to run at equal reduced speed or at almost equal reduced speed or at similar reduced speed simultaneously and operate at the optimum power percentage (or ratio)(subject to limits) of maximum power draw based on the present percentage (or ratio) of maximum loading on the water circuit (800A).

In the event of variable speed drive(s) or plant pump(s) failure, an integral bypass switch (836A, not shown here), can be operated either manually or automatic, will allow said operative pumps to be operated at a predetermined speed. At the same time, an alarm signal will be shown and sent to appropriate location as required alerting appropriate personnel to take necessary actions.

Fig. 9A illustrates a fifth embodiment of this invention utilizing variable speed control of three primary chilled water/boiler pumps operating at predetermined equal reduced speed or almost equal reduced speed or similar reduced speed simultaneously in lieu of the original/traditional designed pumping arrangement as shown in Fig.3 when two chillers (903A and 904A)are operating. A chilled water/boiler circuit (900A) ("plant-building loop system" also known as "primary-secondary chilled/boiler water circuit"), In this case, in the plant loop (901A), three plant pumps (905A, 906A and 907A) of the same or near the same capacity are deployed to transport the produced chilled/boiler water through the plant loop water circuit (901A) (also known as "primary loop") if two chillers/boilers(903A and 904A) are operating at the mean time in accordance with the current loading.

The three pumps (905A, 906A and 907A) are connected in parallel and operate in predetermined equal reduced speed or almost equal reduced speed or similar reduced speed simultaneously. The three pumps (905A, 906A and 907A) and corresponding motors are controlled by respective variable speed drives (VSD930A corresponds to plant pump 905A, VSD931A corresponds to plant pump 906A, VSD 932A corresponds to plant pump 907A) and the said drives (930A, 931A, 932A) were commanded by a Controller (933A). The Controller (933A), monitors the present loading on the water system either by measuring the present power consumption of operating chillers' (903A and 904A) compressor using a power sensor, or the speed of the operating chillers' (903A or 904A) compressor using a tachometer or measuring the discharged chilled water/boiler temperature by temperature sensor located in appropriate location in the water circuit or some other means from which loading can be inferred. Based on that loading signal (934A), an algorithm calculates the optimum power loading (subject to limits) for the pumps and sends an output signal (935A) to the three variable speed drives (930A, 931A, 932A) operating the three corresponding plant pumps (905A, 906A ,907A) such that the three plant pumps (905A, 906A and 907A) were incorporated to run at equal reduced speed or almost equal reduced speed or similar reduced speed simultaneously and operate at the optimum power percentage (or ratio)(subject to limits) of maximum power draw based on the present percentage (or ratio) of maximum loading on the water circuit (900A).

In the event of variable speed drive(s) or plant pump(s) failure, an integral bypass switch (936C, not shown here), can be operated either manually or automatic, will allow said operative pumps to be operated at a predetermined speed. At the same time, an alarm signal will be shown and sent to appropriate location as required alerting appropriate personnel to take necessary actions.

Fig. 10A illustrates sixth embodiment of this invention utilizing variable speed control of four primary chilled water/boiler pumps operating at predetermined equal reduced speed or almost equal reduced speed or similar reduced speed in lieu of the original/traditional pumping arrangement as shown in Fig.3 when two chillers are operating. A chilled water/boiler circuit (1000A) ("plant-building loop system" also known as "primary-secondary chilled water/boiler circuit"), In this case, in the plant loop (1001A), four plant pumps (1005A,1006A,1007A,1037A) of the same or near the same capacity are deployed to transport the produced chilled/boiler water through the

plant loop water circuit (1001A) (primary loop), two chillers/boiler (1003A and 1004A) are operating at the mean time in accordance with the current loading.

The four plant pumps (1005A, 1006A, 1007A, and 1037A) are connected in parallel and operate in predetermined equal reduced speed or almost equal reduced speed or similar reduced speed simultaneously. Said four pumps (1005A,1006A,1007A,1037A) and corresponding motors were controlled by respective variable speed drives (VSD1030A corresponds to plant pump 1005A, VSD1031A corresponds to plant pump 1006A, VSD1032A corresponds to plant pump 1007A, VSD1036A corresponds to plant pump 1037A) and the said drives (1030A,1031A,1032A,1036A) were commanded by a Controller (1033A). The Controller (1033A), monitors the present loading on the water system either by measuring the present power consumption of operating chillers' (1003A and 1004A) compressors using a power sensor, or the speed of the operating chillers' (1003A and 1004A) compressors using a tachometer or measuring discharged chilled/boiler water temperature by temperature sensor located in appropriate location in the water circuit or some other means from which loading can be inferred. Based on that loading signal (1034A), an algorithm calculates the optimum power loading (subject to limits) for the pumps and sends an output signal (1035A) to the respective variable speed drives (1030A,1031A,1032A,1036A) operating the corresponding plant pumps (1005A,1006A,1007A,1037A) such that the four pumps (1005A,1006A,1007A,1037A) were incorporated to run at equal reduced speed or almost equal reduced speed or similar reduced speed simultaneously and operate at the optimum power percentage (or ratio)(subject to limits) of maximum power draw based on the present percentage (or ratio) of maximum loading on the water circuit(1000A).

In the event of variable speed drive(s) or plant pump(s) failure, an integral bypass switch (1036A, not shown here), can be operated either manually or automatic, will allow said operative pumps to be operated at a predetermined speed. At the same time, an alarm signal will be shown and sent to an appropriate location as required alerting appropriate personnel to take necessary actions.

Fig. 11A illustrates seventh embodiment of this invention utilizing variable speed control of two secondary chilled water/boiler pumps operating at predetermined equal reduced speed or almost equal reduced speed or similar reduced speed in lieu of the original/traditional designed pumping arrangement as shown in Fig.3 when chiller(s) is

operating. A chilled/boiler water circuit (1100A) ("plant-building loop system" also known as "primary-secondary chilled/boiler water circuit"), In this case, in the plant loop (1101A) (also known as primary loop), chilled/boiler water leaving the chillers/boilers (1103A, 1104A) at point (1122A) flows through the junction of the
5 common pipe (1121A), plant loop (1101A), and building loop (1102A) at point (1123A), is extracted by the building pumps (also called secondary chilled/boiler water pumps) (1114A and 1115A); and is supplied to the cooling loads (1112A, 1113A). Two building pumps (secondary chilled/boiler water pumps) (1114A and 1115A) of the same or near the same capacity are deployed to transport the produced chilled/boiler
10 water through the building loop water circuit (1102A) (also known as "secondary loop")

The pumps (1114A and 1115A) are connected in parallel and operate in predetermined equal reduced speed or almost equal reduced speed or similar reduced speed
15 simultaneously. The pumps (1114A and 1115A) and corresponding motors were controlled by corresponding variable speed drive (VSD1130A corresponds to building pump 1114A, VSD1131A corresponds to building pump 1115A) and the said drives (1130A and 1131A) were commanded by a Controller (1133A). The Controller (1133A), monitors the present loading on the water system either by measuring the
20 present power consumption of operating chiller(s)' (1103A, 1104A) compressor using a power sensor or the speed of the operating chiller(s)' (1103A or 1104A) compressor using a tachometer and measuring discharged chilled/boiler water temperature by temperature sensor located in appropriate location in the water circuit and the pressure differential of the building loop by pressure differential transmitter located in
25 appropriate location in the building loop, or some other means from which loading can be inferred. Based on these loading signals (1134A), an algorithm calculates the optimum power loading (subject to limits) for the building pumps (1114A and 1115A) and sends an output signal (1135A) to the respective two variable speed drives (1130A and 1131A) operating the two corresponding building pumps (1114A and 1115A) such
30 that the two corresponding building pumps (1114A and 1115A) were incorporated to run at equal reduced speed or almost equal reduced speed or similar reduced speed simultaneously and operate at the optimum power percentage (or ratio)(subject to limits) of maximum power draw based on the present percentage (or ratio) of maximum loading (subject to limits) on the building loop circuit(1102A).

As described above, in the event of variable speed drive(s) or building pump(s) failure, an integral bypass switch (1136A, not shown here), can be operated either manually or automatic, will allow said operative pump to be operated at full speed. At the same time, an alarm signal will be shown and sent to an appropriate location as required
5 alerting appropriate personnel to take necessary actions.

Fig. 12A illustrates eighth embodiment of this invention utilizing variable speed control of two condenser water pumps operating at predetermined equal reduced speed or almost equal reduced speed or similar reduced speed in lieu of the original/traditional
10 designs' pumping arrangement as shown in Fig.4 when one chiller is operating. A condenser water circuit (1200A), In this case, in the condenser water circuit (1200A), two condenser pumps (1206A and 1207A) of the same or near the same capacity are deployed to transport the produced condenser water generated by the chiller (1203A) through the condenser water circuit (1200A) and reject the heat energy through the
15 cooling tower (1202A) to outside atmosphere.

The condenser pumps (1206A and 1207A) are connected in parallel and operate in predetermined equal reduced speed or almost equal reduced speed or similar reduced speed simultaneously. The condenser pumps (1206A and 1207A)) and corresponding
20 motors are controlled by corresponding variable speed drive (VSD1230A corresponds to condenser pump 1206A, VSD1231A corresponds to condenser pump 1207A) and the said drives (1230A, 1231A) are commanded by a Controller (1233A). The Controller (1233A), monitors the present loading on the water system either by measuring the present power consumption of operating chiller's (1203A) compressor
25 using a power sensor, or the speed of the operating chiller's (1203A) compressor using a tachometer or measuring discharged chilled water temperature and the discharged condenser water temperature by temperature sensor located in appropriate location in the water circuit or some other means from which loading can be inferred. Based on these loading signals (1234A), an algorithm calculates the optimum power loading
30 (subject to limits) for the pumps and sends an output signal (1235A) to the respective two variable speed drives (1230A,1231A) operating the two corresponding condenser pumps (1206A,1207A) such that the two corresponding condenser pumps (1206A,1207A) were incorporated to run at equal reduced speed or almost equal reduced speed or similar reduced speed simultaneously and operate at the optimum
35 power percentage (or ratio)(subject to limits) of maximum power draw based on the

present percentage (or ratio) of maximum loading (subject to limits) on the condenser water circuit(1200A).

As described above, in the event of variable speed drive(s) or condenser water pump(s) failure, an integral bypass switch (1236A, not shown here), can be operated either manually or automatic, will allow the operative pump to be operated at full speed. At the same time, an alarm signal will be shown and sent to an appropriate location as required alerting appropriate personnel to take necessary actions.

- 10 Fig. 13A illustrates ninth embodiment of this invention utilizing variable speed control of three condenser water pumps operating at predetermined equal reduced speed or almost equal reduced speed or similar reduced speed in lieu of the original/traditional pumping arrangement as shown in Fig.4 when chiller(s) is operating. A condenser water circuit (1300A), In this case, in the condenser water circuit (1300A), Three
 15 condenser pumps (1306A, 1307A and 1308A) of the same or near the same capacity are deployed to transport the produced condenser water generated by the chiller(s) (1303A,1304A) through the condenser water circuit (1300A) and reject the heat energy through the cooling tower(s) (1302A, 1305A) to outside atmosphere.
- In this case, in the condenser water circuit (1300A), three condenser pumps (1306A,
 20 1307A and 1308A) of the same or near the same capacity are deployed to transport the produced condenser water through the condenser water circuit (1300A) if chiller(s) (1303A,1304A) is operating at the mean time in accordance with the current loading.

- The three condenser pumps (1306A, 1307A and 1308A) are connected in parallel and
 25 operate in predetermined equal reduced speed or almost equal reduced speed or similar reduced speed simultaneously. The three condenser pumps (1306A, 1307A and 1308A) and corresponding motors are controlled by respective variable speed drives (VSD1330A corresponds to condenser pump 1306A, VSD 1331A corresponds to condenser pump 1307A, VSD 1337A corresponds to condenser pump 1308A) and the
 30 said drives (1330A, 1331A, 1337A) are commanded by a Controller (1333A). The Controller (1333A), monitors the present loading on the water system either by measuring the present power consumption of operating chiller(s)' (1303A, 1304A) compressor using a power sensor, or the speed of the operating chiller(s)' (1303A,1304A) compressor using a tachometer or measuring discharged chilled water
 35 temperature and discharged condenser water temperature leaving the chiller by temperature sensor located in appropriate location in the water circuit or some other

means from which loading can be inferred. Based on that loading signal (1334A), an algorithm calculates the optimum power loading (subject to limits) for the pumps and sends an output signal (1335A) to the three variable speed drives (1330A, 1331A, 1337A) operating the three corresponding condenser pumps (1306A, 1307A, 1308A) such that the three condenser pumps (1306A, 1307A and 1308A) were incorporated to run at equal reduced speed or almost equal reduced speed or similar reduced speed simultaneously and operate at the optimum power percentage (or ratio)(subject to limits) of maximum power draw based on the present percentage (or ratio) of maximum loading on the water circuit (1300A).

10

Again, in the event of variable speed drive(s) or condenser pump(s) failure, an integral bypass switch (1336A, not shown here), can be operated either manually or automatic, will allow the operative pumps to be operated at a predetermined speed. At the same time, an alarm signal will be shown and sent to appropriate location as required alerting appropriate personnel to take necessary actions.

15

Fig. 14A illustrates tenth embodiment of this invention utilizing variable speed control of four condenser water pumps operating at predetermined equal reduced speed or almost equal reduced speed or similar reduced speed in lieu of the original/traditional designed pumping arrangement as shown in Fig. 4 when two chillers are operating. In this case, in the condenser water circuit (1400A), Four condenser pumps (1406A, 1407A, 1408A and 1409A) of the same or near the same capacity are deployed to transport the produced condenser water generated by the chillers (1403A and 1404A) through the condenser water circuit (1400A) and reject the heat energy through the cooling towers (1402A, 1405A, 1439A) to outside atmosphere.

20

25

In this case, in the condenser water circuit (1400A), four condenser pumps (1406A, 1407A, 1408A, 1409A) of the same or near the same capacity are deployed to transport the produced condenser water through the condenser water circuit (1400A) if two chiller (1403A and 1404A) are operating at the mean time in accordance with the current loading.

30

The four condenser pumps (1406A, 1407A, 1408A, and 1409A) are connected in parallel and operate in predetermined equal reduced speed or almost equal reduced speed or similar reduced speed simultaneously. The four condenser pumps (1406A, 1407A, 1408A, 1409A) and corresponding motors are controlled by respective variable

35

speed drives (VSD 1430A corresponds to condenser pump 1406A, VSD 1431A corresponds to condenser pump 1407A, VSD 1437A corresponds to condenser pump 1408A, VSD 1438A corresponds to condenser pump 1409A) and the said drives (1430A, 1431A, 1437A, 1438A) are commanded by a Controller (1433A). The
5 Controller (1433A), monitors the present loading on the water system either by measuring the present power consumption of operating chillers' (1403A and 1404A) compressor using a power sensor, or the speed of the operating chillers' (1403A and 1404A) compressor using a tachometer or measuring discharged chilled water temperature and discharged condenser water temperature leaving the chillers by
10 temperature sensor located in appropriate location in the water circuit or some other means from which loading can be inferred. Based on that loading signal (1434A), an algorithm calculates the optimum power loading (subject to limits) for the pumps and sends an output signal (1435A) to the four variable speed drives (1430A, 1431A, 1437A, 1438A) operating the four corresponding condenser pumps (1406A, 1407A ,
15 1408A, 1409A) such that the four condenser pumps (1406A, 1407A, 1408A, 1409A) were incorporated to run at equal reduced speed or almost equal reduced speed or similar reduced speed simultaneously and operate at the optimum power percentage (or ratio)(subject to limits) of maximum power draw based on the present percentage (or ratio) of maximum loading on the water circuit (1400A).

20

As above, in the event of variable speed drive(s) or condenser pump(s) failure, an integral bypass switch (1436A, not shown here), can be operated either manually or automatic, will allow the operative pumps to be operated at a predetermined speed. At the same time, an alarm signal will be shown and sent to appropriate location as
25 required alerting appropriate personnel to take necessary actions.

It will be appreciated that above preferred embodiments' primary chilled water pumps, secondary chilled water pumps, condenser water pumps, boiler pumps, are shown for illustration, the system can include additional units of said pumping arrangements.
30 Additional loads can also be connected to the system, and the plurality of operating pump means can act responsive to a predetermined target. I.e. a certain flow rate, a certain pressure differential, a desired discharge pressure, etc.

It will be appreciated that the present invention has application in a large range of
35 industrial fields where pumps are used for pumping fluids and where the pumping loads

are at maximum or vary. The features of the invention are then able to be used to improve the system efficiencies.

Examples

5 Example 1.

Development of an embodiment of the invention in “primary chilled water circuit” of a medium size hotel

A medium size hotel with 270 guestrooms is equipped with 3 identical capacity 18 KW primary chilled water pumps. Only one primary chilled water pump is used to circulate
10 the chilled water through the primary chilled water circuit of the HVAC System while the remaining two pumps are in standby mode. The operating pump is allowed to run uncontrolled at its maximum flow.

Constraints

1. Chiller Manufacturer recommends a constant flow of water through the chiller
15 evaporator (the primary chilled water circuit) of the HVAC system.
2. Variable flow in the primary chilled water circuit may cause instable operation or nuisance shutdown of the chillers; therefore, chiller manufacturers specify minimum limit for evaporator water flows.
3. Insufficient flow in the primary chilled water circuit may lead to freeze up in the
20 evaporator and caused severe damage to the chiller (ruptured the evaporator tube)
4. Reduced flow in primary chilled water circuit decreases the overall heat-transfer effectiveness of the evaporator as the convective heat transfer coefficient decreases with a reduction in flow.

Action taken

- 25 1. Three variable frequency drives and one PLC were fitted on the primary chilled water pumps. The system is coordinated with the building management system, enabling easy monitoring of the pump’s operation.
2. Two of the pumps are operated to run at equal reduced speed about 50% of the maximum speed providing the required constant flow capacity of the original systems
30 design requirement. Previously only one pump ran continuously at maximum speed.

Results

The measures resulted in significant energy savings. The total power consumption of the two operating primary chilled water pumps after fitting of our invention fell by nearly 72 percent compared with a theoretical saving of 75 percent ($2 \times 0.5^3 = 0.25$) in a
35 perfect condition. Analysis showed that the energy saved was 113,529 kWh per year, resulting in an annual saving of USD 14,600. In addition, an annual maintenance cost

was saved due to the lower pressure imparted to the system, reduced excessive vibration and loading on the operating pumps and respective pipeline.

Example 2

In a pilot test, three variable frequency drives and one PLC were fitted on the primary chilled water pumps. The system is coordinated with the building management system, enabling easy monitoring of the pump's operation.

Three pumps were run at equal reduced speed of about 34% of the maximum speed providing the required constant flow capacity of the original systems design requirement.

10 **Results**

The measures resulted in significant energy savings. The total power consumption of the three operating primary chilled water pumps after incorporation of the described embodiment of the invention fell by nearly 85 percent ($3 \times 0.34^3 = 0.12$). Analysis showed that the energy saved was 134,028 kWh per year, resulting in a potential annual saving of USD 17,230.

Annual Analysis Results

System Flow Alternatives	Pumping Energy Consumed	Operating Costs, US Currency	Annual Cost Saving, US Currency
Constant Primary	157,680 kWh	\$20,270	NIL
Running Two Pumps at Reducing Speed	44,150 kWh	\$5,670	14,600
Running Three Pumps at Reducing Speed	23,652 kWh	\$3,040	17,230

As the analysis results show, the examples of the inventive system can yield substantial savings when compared with the standard, constant-primary flow design.

20 Example 3

Development of an embodiment of the invention in "secondary chilled water circuit".(under proposed stage)

The said medium size hotel with 270 guestrooms is equipped with two 75 KW secondary chilled water pumps. One secondary chilled water pump is used to circulate the water through the secondary chilled water circuit of the HVAC System while the remaining pump is in standby mode. One operating pump is allowed to run at its maximum flow for 20 hours (00:00 to 14:00 and 18:00 to 24:00) and at 15% reduced speed of its maximum speed for 4 hours (14:00 to 18:00) under low loading condition.

Constraints

1. Reduced flow in secondary chilled water circuit decreases the overall heat-transfer effectiveness as the heat transfer coefficient decreases with a reduction in flow.
2. the original system has installed two variable speed drives to the respective pumps and one pump is running at 85 % of maximum speed for a specific period during daytime and has achieving a 10% of energy saving. The remaining pump is under standby condition.
3. The embodiment of the invention must provide significant further saving in order to justify to the user that investment in the system is worthwhile.

Action taken

1. Two variable frequency drives and one PLC is fitted on the secondary chilled water pumps. The system is coordinated with the building management system, enabling easy control of the pump's operation.
2. Two operating chilled water pumps are run at equal reduced speed between 43-50% of the maximum speed in accordance with the load,

Results

Analysis showed that a further 60% of energy can be saved in comparison with the original designed one secondary chilled pumps running at 15% of reducing speed for a specific period (4 hours daily) which can achieved 10% of energy saving only. The anticipated power consumption of the two operating secondary chilled water pumps fell by at least 70% totally. Analysis showed that the estimated energy that can be saved was 459,900 kWh per year, resulting in an annual saving of USD 59,120. In addition, an annual maintenance cost can be saved due to the lower pressure imparted to the system, reduced excessive vibration and loading on the operating pumps and the piping system.

Annual Analysis Results

System Flow Alternatives	Estimated Pumping Energy Consumed	Operating Costs, US Currency	Annual Cost Saving, US Currency
Constant Secondary	657,000 kWh	84,450	NIL
Variable Secondary	591,300 kWh	76,000	\$8,450
Running Two Pumps at Reducing Speed	197,100 kWh	25,330	\$59,120

As the analysis results show, the described embodiment can yield substantial savings when compared with a variable-secondary flow design.

Example 4

Development of an embodiment of the invention in small "single chilled water circuit".

- 5 A manufacturer with 3000 square feet floor area is equipped with two 12.5 KW chilled water pumps. One chilled water pump is used to circulate the chilled water through the single chilled water circuit of the HVAC while the other pump is in standby mode. The operating pump was allowed to run uncontrolled at its maximum flow.

Constraints

- 10 1. Chiller Manufacturer recommends a constant flow of water through the chiller evaporator (the single chilled water circuit) of the HVAC system.
2. Variable flow in the single chilled water circuit may cause instable operation or nuisance shutdown of the chiller; therefore, chiller manufacturers specify minimum limit for evaporator water flows.
- 15 3. Insufficient flow in the single chilled water circuit may lead to freeze up in the evaporator and caused severe damage to the chiller (ruptured the evaporator tube)
4. Reduced flow in chilled water circuit decreases the overall heat-transfer effectiveness of the evaporator as the convective heat transfer coefficient decreases with a reduction in flow.
- 20 5. Optional market available energy saving methods such as "chilled water reset", "colder water, series evaporator", "colder water, lower flow", or simply "variable flow" are neither technologically viable or would be economically acceptable by the HVAC owner due to the following reasons:
 - 1.High upfront investment
 - 25 2.Long payback period (3 years or above) or unsatisfactory return
 - 3.Involve complicated modification or complexity for control and operational method.
 - 4.The chiller control cannot accommodate for such arrangements.

Action taken

- 30 1. Two variable frequency drives and one PLC were fitted on the single circuit chilled water pumps.
2. Two single circuit chilled water pumps run at equal reducing speed, about 50% of the maximum speed providing the required constant flow capacity of the original system design requirement and no standby pump are provided and be necessary in this
- 35 situation as bypass and changeover capability is available in case of emergency/maintenance situations.

Results

The measures resulted in significant energy savings. The power consumption of the operating single circuit chilled water pump fell by nearly 73%. Analysis showed that the energy saved was 79,935 kWh per year, resulting in an annual saving of USD 10,270. In addition, a significant maintenance cost was saved due to the lower pressure imparted to the system, reduced excessive vibration and loading on the operating pump and the pipe line.

Example 5

An additional pump is added to the single circuit chilled water system of Example 4 so that three chilled water pumps are now used, and they are equipped with three variable frequency drives and one PLC fitted to the chilled water pumps. The system is equipped with the bypass and changeover capability, enabling safe operation of the pumping system in case of emergency/maintenance situation. The three pumps are run at substantially equal reduced speed, about 34% of the maximum speed providing the required constant flow capacity of the original systems design requirement.

Results

The estimated total power consumption of the three operating primary chilled water pumps fell by nearly 85%. Analysis showed that the estimate energy saving is 93,075 kWh per year, resulting in an annual saving of USD 11,960.

Annual Analysis Results

System Flow Alternatives	Pumping Energy Consumed	Operating Costs, US Currency	Annual Cost Saving US Currency
Constant Single Circuit	109,500 kWh	\$14,070	NIL
Running Two Pumps at Reducing Speed	29,565 kWh	\$3,800	\$10,270
Running Three Pumps at Reducing Speed	16,425 kWh	\$2,110	\$11,960

As the analysis results show, the described embodiments of the inventive system can yield substantial savings when compared with a constant single circuit flow design.

Example 6

A central condenser water supply plant is equipped with three 450 KW central condenser water pumps. One condenser water pump is used to circulate sea water through the region's building condenser water circuits of the HVAC Systems while the remaining two condenser pumps are in standby mode.

Constraint

1. The original system designer recommend constant flow configuration in the central condenser water supply circuit for HVAC systems' cooling in the region.
2. Variable flow in the central condenser water supply circuit may cause instable
5 operation of the chillers in the region especially the compressor's operation.
3. Insufficient flow in the central condenser water circuit may lead to unacceptable temperature rise in the region's condensers and caused damage to the respective chillers.

Action taken

- 10 Three variable frequency drives and one PLC were fitted on the central condenser water pumps. Two operating condenser water pumps run at about 50% of the maximum speed providing the required constant flow capacity of the original systems design requirement. Previously one of the pumps was allowed to run at its maximum flow while the remaining two pumps were in standby mode.

15 Results

- Estimated energy savings of 70% can be achieved. The power consumption of the two operating central condenser water pumps after incorporating the inventive features is expected to fall by 70%. Analysis showed that the estimated energy to be saved is 2,759,400 kWh per year, resulting in an annual saving of USD 354,680. In addition, an
20 annual maintenance cost to be saved due to the lower pressure imparted to the system, reduced excessive vibration and loading on the operating pumps and the piping system.

Example 7

Following the success of Example 6, a second stage improvement was implemented.

- An additional, third pump was added to the central condenser water supply circuit thus
25 totaling three condenser water pumps which run with three variable frequency drives and one PLC fitted on the condenser water pumps. The system is equipped with the bypass and changeover capability, enabling safe operation of the central condenser water supply system in case of emergency/maintenance situation.

- The three condenser pumps are run at substantially equal reduced speed, about 34% of
30 the maximum speed providing the required constant flow capacity of the original systems design requirement.

Results

- The estimated total power consumption of the three operating condenser water pumps after fitting of the embodiment of the invention fell by nearly 84%. Analysis showed
35 that the estimate energy saving is 3,311,280 kWh per year, resulting in an annual saving of USD 425,610.

Projected Annual Energy Consumption

System Flow Alternatives	Estimated Pumping Energy Consumed	Operating Costs, US Currency	Anticipated Annual Cost Saving US Currency
Constant Condenser	3,942,000 kWh	\$506,680	NIL
Running Two Pumps at Reducing Speed	1,182,600 kWh	\$152,000	\$354,680
Running Three Pumps at Reducing Speed	630,720 kWh	\$81,070	\$425,610

- 5 As the analysis results show, the inventive features can yield substantial savings when compared with a one pump constant-condenser flow design.

Example 8**Swimming pool water re-circulation circuit**

- 10 A residential estate's swimming pool is equipped with two 20 KW pool water recirculation pumps. One recirculation pump is used to circulate the pool water through the mechanical filtration and chemical treatment devices and then returned to the pool while the other pump is in standby mode. The operating pump was allowed to run uncontrolled at its maximum flow

15 Constraints

1. Disinfection and treatment of swimming pool water is governed by certain standard and regulations.
2. The constancy of flow rate is mandated by local standard and specifications.
3. The system designer do recommend constant flow configuration in the pool water recirculation circuit.
- 20 4. A constant quality of pool water which satisfies sanitation standards, safety and appearance, to avoid spreading bacteria or contaminant causing diseases.
5. Variable flow in the pool water recirculation circuit in accordance with the turbidity of the pool water can not satisfy the respective standard and regulations requirements fully due to the following reason;
- 25 6. Insufficient flow in the pool water recirculation circuit may lead to undesirable excess of chemicals in the pool water and cause harmful to swimmer's health.

Action taken

1. Two variable frequency drives and one PLC are fitted on the pool water recirculation pumps.
2. Two water circulating pumps are run at substantially equal reduced speed, about 50% of the maximum speed, providing the required constant flow capacity of the original system design requirement and no standby pump are provided or deemed necessary in this situation as bypass and changeover capability is available in case of emergency/maintenance situation.

Results

- 10 The measures have shown significant potential energy savings. The estimate power consumption of the operating pool water pump after incorporation of the inventive features fell by nearly 65%. Analysis showed that the energy to be saved is about 113,880 kWh per year, resulting in an annual saving of USD 14,600. In addition, a significant maintenance cost can be saved due to the lower pressure imparted to the system, reduced excessive vibration and loading on the operating pumps and the pipeline.

Projected Annual Energy Consumption

System Flow Alternatives	Pumping Energy Consumed	Operating Costs, US Currency	Anticipated Annual Cost Saving US Currency
Constant Primary	175,200 kWh	\$22,520	NIL
Running Two Pumps at Reducing Speed	61,320 kWh	\$7,880	\$14,640

20

As the analysis results indicate, the inventive system may yield substantial savings when compared with a constant-swimming pool water recirculation flow design.

- 25 It will be appreciated by persons skilled in the art that numerous variations and/or modifications may be made to the invention as shown in the specific embodiments without departing from the spirit or scope of the invention as broadly described. The present embodiments are, therefore, to be considered in all respects as illustrative and not restrictive.

CLAIMS:

1. An industrial fluid circulating system having one or more fluid circulation circuits, the system including:
 - a plurality of pump means to circulate the fluid through the or each of said
 - 5 circuits,
 - motor means driving each pump means,
 - load sensing or detecting means to sense operating loads on the system and the circuits, and
 - speed control means to vary the speed of said motor means to thereby vary the
 - 10 pumping capacity of each pump means in response to the detected load on the system.
2. An industrial fluid circulating system according to claim 1 wherein the or each said fluid circulation circuits has:
 - one or more fluid circuits; and
 - 15 said pump means comprises two or more pumps of the same capacity or similar capacity for each circulation circuit to circulate fluid through the or each of said fluid circuits, the pump operation being controlled in accordance with system load or a predetermined target or manual judgment of skilled personnel.
- 20 3. An industrial fluid circulating system according to claim 1 or claim 2 wherein said speed control means includes a plurality of variable speed drives to operate the associated pump means.
4. An industrial fluid circulating system according to any one of claims 1 to 3
- 25 wherein the system loading, or a predetermined target or manual judgment of skilled personnel, sets an operation speed of each of the plurality of pump means, each pump means running at a substantially similar speed, the speed for respective circuits being a predetermined, equally reduced speed, or an almost equal reduced speed or a similar reduced speed, simultaneously.
- 30 5. An industrial fluid circulating system according to any one of claims 1 to 4 wherein said fluid circulation circuits include a refrigeration system having:
 - one or more chilled water, condenser water, and boiler water circuits; and
 - said pump means comprises two or more pumps for each circuit to circulate
 - 35 water through each of said water circuits, the pump operation being controlled in

accordance with system load or a predetermined target or manual judgment of skilled personnel.

- 5 6. An industrial fluid circulating system according to claim 5 wherein cooling system loading is determined by measuring the present power consumption of operating chiller(s) compressor(s) using a power sensor.
- 10 7. An industrial fluid circulating system according to claim 5 wherein cooling system loading is determined by measuring the speed of the operating chiller(s)' compressor(s) using a tachometer.
- 15 8. An industrial fluid circulating system according to claim 5 wherein system loading is determined by measuring the discharged chilled water/boiler water temperature by a temperature sensor located in an appropriate location in the water circuit.
- 20 9. An industrial fluid circulating system according to any one of claims 5 to 8 wherein the cooling system loading or the predetermined target or manual judgment of the skilled personnel sets operation of selected ones of the plurality of condenser water pumps, chilled water pumps and boiler pumps running at their respective circuit's predetermined equal reduced speed or at almost equal reduced speed or at similar reduced speed.
- 25 10. An industrial fluid circulating system according to any one of claims 5 to 9 wherein the cooling system loading sets operation of selected ones of the plurality of condenser water pumps, chilled water pumps and boiler pumps running at their respective circuit's predetermined equal reduced speed or at almost equal reduced speed or at similar reduced speed and at a power setting that is a fixed ratio of the cooling system current power ratio or loading.
- 30 11. An industrial fluid circulating system according to any one of claims 1 to 10 wherein each pump means has a power set point which is expressed as:
$$\text{PR-SP. sub.pd} = C * \text{PR. sub.load}$$

where PR-SP.sub.pd is the power ratio (percent of maximum) set point of the
35 respective pump means being controlled,

PR.sub.load is the current loading/power ratio (percent of maximum) that is being utilized by a circuit or system, and

C is a selected constant, the formula having a low limit to prevent a power ratio set point being so low as to result in all fluid flow ceasing, and a high limit to ensure the
5 power ratio set point would never rise above undesired flow rates and pressure outputs.

12. An industrial fluid circulating system according to any one of claims 1 to 11 wherein said speed control means includes;

- a variable-speed drive circuit for powering each of the pump means;
- 10 a number of feeding pump means operatively coupled to a number of reception means with connecting line(s);
- said motor means driving at least two of said pump means;
- means for varying the speed of said motors means; and
- means for controlling the variable-speed drive circuits in response to the present
15 load on the system or act responsive to a predetermined target or manual judgment of skilled personnel so as to regulate operation of the said pump means simultaneously running at equal reduced speed or at almost equal reduced speed or at similar reduced speed.

20 13. An industrial fluid circulating system according to any one of claims 1 to 12 wherein said system includes a variable capacity, compression type, chilled fluid cooling system comprising:

- a single circuit chilled water system including:
 - at least two chilled water pumps;
 - 25 a variable-speed drive circuit for powering each of the chilled water pump;
 - a chiller with evaporator operatively coupled to a numbers of cooling loads and a suction line leading back to said chiller;
 - water passing through said chiller in a heat exchange relationship and being cooled;
 - 30 motor means for driving said chilled water pumps;
 - means for varying the speed of said motor means;
 - means for sensing the temperature of water leaving said chiller; and
 - means for controlling the variable-speed drive circuit in response to the present load on a system compressor so as to regulate operation of the said at least two chilled
35 water pumps running at equal reduced speed or at almost equal reduced speed or at similar reduced speed simultaneously in response to the load on the compressor.

14. An industrial fluid circulating system according to any one of claims 1 to 13 wherein the fluid circulation circuits are a primary-secondary heat absorbing chilled water circuit having at least two primary chilled water pumps, and at least two
5 secondary chilled water pumps, and said speed control means includes selected ones of;
a variable-speed drive circuit for powering each of the primary chilled water pump; and
a variable-speed drive circuit for powering each of the secondary chilled water pump;
10 the system further having two or more chillers with evaporators being programmed so that the two or more chillers are kept operating or at least one is kept operating and another remains shutdown and on standby in response to a predetermined loading condition operatively coupled to a number of cooling loads; and
a suction line leading back to said chillers;
15 water passing through said chillers in a heat exchange relationship and being cooled;
said motor means including motors driving selected ones of said predetermined operating primary chilled water pumps and secondary chilled water pumps;
means for varying the speed of said operating motors means;
20 means for sensing the temperature of water leaving said operating chiller(s);
means for controlling the operating variable-speed drive circuits in response to the present load on an operating compressor so as to regulate operation of selected ones of the predetermined plurality of primary chilled water pumps and plurality of secondary chilled water pumps simultaneously running at respective circuit's
25 predetermined equal reduced speed or almost equal reduced speed or similar reduced speed in response to the load on the operating compressor.

15. An industrial fluid circulating system according to any one of claims 1 to 14 wherein one of the fluid circulation circuits is a heat rejection condenser water circuit
30 including at least two condenser water pumps; and said speed control means includes;
a variable-speed drive circuit for powering each of the condenser water pump;
a chiller with condenser operatively coupled to a number of cooling towers
and a suction line leading back to said chiller with condenser;
water passing through said chillers with condenser in a heat exchange
35 relationship and being heated;
said motor means driving said at least two operating condenser water pumps;

means for varying the speed of said motors means;

means for sensing the temperature of water leaving said chillers; and

means for controlling the variable-speed drive circuits in response to the present load on an associated operating compressor so as to regulate operation of the at least
5 two operating condenser water pumps simultaneously running at equal reduced speed or at almost equal reduced speed or at similar reduced speed in response to the loading on the system.

16. An industrial fluid circulating system according to any one of claims 13 to 15
10 wherein the chiller includes means for regulating the flow of refrigerant gas through the compressor, and the means for determining a present load on the compressor makes that determination in response to a present state of the gas flow regulating means in the compressor.

15 17. An industrial fluid circulating system according to any one of claims 13 to 16 wherein the means for determining a present load on the compressor makes that determination in response to a level of power applied to the compressor motor or the sensed temperature of water leaving said chiller.

20 18. An industrial fluid circulating system according to any one of claims 13 to 17 wherein the means for determining a present load on the compressor and the means for controlling the variable speed drive circuit is configured to regulate the variable speed drive circuit at the predetermined percentage of full power thereby running selected
ones of the plurality of chilled water pumps, condenser water pumps and boiler pumps
25 simultaneously at respective circuits' predetermined equal reduced speed or at almost equal reduced speed or at similar reduced speed to minimize power while system loading is at maximum or below maximum loading.

19. An industrial fluid circulating system according to any one of claims 1 to 18
30 wherein said plurality of pump means comprises three or more fluid pumps for each circuit operating at equal reduced speed or at almost equal reduced speed or at similar reduced speed simultaneously.

20. An industrial fluid circulating system according to any one of claims 1 to 19
35 wherein the speed control means for varying the speed of said pump means simultaneously at equal reduced speed or at almost equal reduced speed or at

similar reduced speed acts in response to a predetermined target or the manual judgment of skilled personnel

21. A method of operating an industrial system having one or more water, or other
5 fluid circulation circuits comprising the steps of:
providing a plurality of pump means to circulate fluid through the or each of said
circuits;
operating motor means to drive each pumps means,
sensing operating loads on the system and the circuits; and
10 varying the speed of said motor means at respective circuit's predetermined equal
reduced speed or at almost equal reduced speed or at similar reduced speed to thereby
vary the pumping capacity of each pump means in response to the sensed load on the
system; or in response to a predetermined target; or in response to manual judgment of
skilled personnel.
15
22. A method according to claim 21 including the steps of circulating fluid through
the or each of said fluid circuits with two or more pumps of the same capacity or
similar capacity for each circulation circuit, and controlling the pump operation in
accordance with system load or a predetermined target or manual judgment of skilled
20 personnel.
23. A method according to claim 21 or 22 including the step of providing a plurality
of variable speed drives to operate the associated pump means.
24. A method according to any one of claims 21, 22 or 23 including the steps of
25 setting an operational speed for each of the plurality of pump means so that each pump
means runs at a substantially similar speed which is a predetermined, equally reduced
speed or an almost equal reduced speed or a similarly reduced speed, simultaneously,
for each respective circuit in response to the system loading or a predetermined target
30 or manual judgment of skilled personnel.
25. A method according to claim 21 including the steps of providing a refrigeration
system having one or more chilled water, condenser water, and boiler water circuits,
circulating water through the or each of said water circuits using said pump means
35 comprising two or more pumps for each circuit, and controlling the pump operation in

accordance with system load or a predetermined target or manual judgment of skilled personnel.

26. A method according to any one of claims 21 to 25 including the steps of
5 determining cooling system loading by measuring the present power consumption of operating chiller(s) compressor(s) using a power sensor.

27. A method according to claim 21 to 25 including the steps of determining cooling
system loading by measuring the speed of the operating chiller(s)' compressor(s) using
10 a tachometer.

28. A method according to any one of claims 21 to 25 including the steps
determining system loading by measuring the discharged chilled water/boiler water
temperature by a temperature sensor located in an appropriate location in the water
15 circuit.

29. A method according to claim 25 including the steps of setting operation of
selected ones of the plurality of condenser water pumps, chilled water pumps and boiler
pumps running at their respective circuit's predetermined equal reduced speed or at
20 almost equal reduced speed or at similar reduced speed using the cooling system
loading or the predetermined target or manual judgment of the skilled personnel.

30. A method according to claim 25 including the steps of using the cooling system
loading to set operation of selected ones of the plurality of condenser water pumps,
25 chilled water pumps and boiler pumps running at their respective circuit's
predetermined equal reduced speed or at almost equal reduced speed or at similar
reduced speed and at a power setting that is a fixed ratio of the cooling system current
power ratio or loading.

30 31. An industrial fluid circulating system having a plurality of fluid circulation
circuits, the system including:

a variable capacity, compression type, chilled fluid cooling system comprising:
a single circuit chilled water system including:

at least two chilled water pumps;
35 a chiller with evaporator operatively coupled to a numbers of cooling loads
and a suction line leading back to said chiller;

water passing through said chiller in a heat exchange relationship and being cooled;

motor means for driving said chilled water pumps;

a variable-speed drive circuit for powering each of the motor means to vary the
5 speed of the chilled water pumps;

means for sensing the temperature of water leaving said chiller; and

means for controlling the variable-speed drive circuit in response to the present
load on a system compressor so as to regulate operation of the said at least two chilled
water pumps running at equal reduced speed or at almost equal reduced speed or at
10 similar reduced speed simultaneously in response to the load on the compressor.

32. An industrial fluid circulating system according to claim 31 wherein the chiller
includes means for regulating the flow of refrigerant gas through the compressor, and
the means for determining a present load on the compressor makes that determination
15 in response to a present state of the gas flow regulating means in the compressor.

33. An industrial fluid circulating system according to claim 31 or claim 32
wherein the means for determining a present load on the compressor makes that
determination in response to a level of power applied to the compressor motor or the
20 sensed temperature of water leaving said chiller.

34. An industrial fluid circulating system according to any one of claims 31 to 33
wherein the means for determining a present load on the compressor and the means for
controlling the variable speed drive circuit is configured to regulate the variable speed
25 drive circuit at the predetermined percentage of full power thereby running the selected
ones of the plurality of chilled water pumps, condenser water pumps and boiler pumps
at respective circuits' predetermined equal reduced speed or at almost equal reduced
speed or at similar reduced speed to minimize power while system loading is at
maximum or below maximum loading.

30

35. An industrial fluid circulation system according to any one of claims 1 to 20
wherein the fluid is a liquid.

36. A method according to any one of claims 21 to 30 wherein the fluid is a liquid.
35

37. An industrial system substantially as hereinbefore described with reference to the accompanying drawings.

38. A method of operating an industrial system having one or more water, or other
5 fluid circulation circuits having the combination of steps substantially as hereinbefore described with reference to the accompanying drawings and/or the Examples.