

VENTURI COOLING SYSTEM

Technical Field

[0001] The present invention relates to combustors that may be used in combustion turbines. More specifically, the present invention relates to a system for cooling the venturi throat region between the primary and secondary combustion chambers.

Background

[0002] Gas turbines play a significant role in a number of applications, such as aircraft propulsion, marine propulsion, power generation and driving processes, such as pumps and compressors. Typically, a gas turbine includes a compressor, a combustor and a turbine. In operation, air is fed into the system where it is compressed by the compressor and mixed with fuel in the combustor. The compressed air and fuel mixture are then burned within the combustor to cause an expansion of the air flow, which is responsible for driving the turbine.

[0003] Combustion liners define the interior volume of the combustor and serve to protect the combustor casing and remaining engine parts from the extreme temperatures present within the combustor. In order to reduce NO_x emissions and extend the longevity of engine components, it is desirable to use a portion of the compressed air exiting the compressor for cooling various combustor components, including the combustion liner.

[0004] Combustion liners often include a venturi throat region that is used to influence the flame location within the combustor and to stabilize the combustion flame. In such arrangements, a lowering of the NO_x emissions is achieved by lowering peak flame temperatures through the burning of a lean, uniform mixture of fuel and air. Uniformity is typically achieved by pre-mixing fuel and air in the combustor upstream of the venturi and then firing the mixture downstream of the edge or apex of the venturi into the secondary combustion chamber. The

venturi configuration, by virtue of accelerating the flow preceding the throat portion, is intended to keep the flame from flashing back into the pre-mixing region or primary combustion chamber, upstream of the venturi throat. The flame holding region, upstream of the venturi, is also required to be cooled under certain operating conditions. Since flames can reside on each side of the venturi, both sides need to be cooled. However, if the flow of cooling air released downstream of the venturi is too large or is released too close to the venturi, it may adversely affect combustion performance.

[0005] US 5,117,636 shows a cooling passageway within the venturi that is extended downstream from the venturi apex or throat, such that the cooling air does not re-enter the flame holding zone within the secondary combustion chamber. It is intended that the functions of flame holding and venturi cooling are decoupled. US 6,430,932 shows a cooling chamber within a venturi that directs the cooling flow counter to the combustion flow and dumps cooling air upstream of the venturi.

[0006] It has been determined that the heat load on the upstream and downstream sides of the venturi are different and the dumping of cooling air near the venturi on either side has an effect on the operation of the combustor. In the prior designs, where the cooling of both sides of the venturi are linked, there has not been a balancing of the heat pickup within the venturi walls nor has there been an accommodation made for pressure drop within the cooling passageway.

Summary of the Invention

[0007] A combustor is provided for a gas turbine engine of the type having a nozzle assembly located at a first end of the combustor and a combustion chamber defined at a second end of the combustor, downstream of the nozzle. A venturi is positioned between the nozzle and the combustion chamber. The venturi defines a transition between the first end of the combustor

and the second end of the combustor. The venturi includes a passageway therein for receipt of a cooling air flow. The passageway has a first side facing the nozzle and a second side facing the combustion chamber. A compressed air source is directed into the passageway with the flow inlet being in communication with both the first and second sides of the venturi passageway. The venturi passageway directs compressed air from the source in opposite directions within the venturi. Thus, the air flow in the first side and second side of the passageway are in opposite directions.

[0008] A further aspect of the invention may include a secondary passageway that extends upstream of the second side of the venturi passageway. Further, the cooling flow inlet may be positioned closer to the nozzle or closer to the upstream side of the venturi apex.

Brief Description of the Drawings

[0009] In the drawings, there is illustrated a number of embodiments which are presently preferred.

[0010] Fig. 1 shows a simplified cross-section of a gas turbine combustion system incorporating an embodiment of the present invention.

[0011] Fig. 2 is a partial cross-sectional view of the venturi throat region of the combustion system illustrated in Fig. 1.

[0012] Fig. 3 is a partial cross-sectional view of a further embodiment of a venturi system for use in a combustion system.

[0013] Fig. 4 is a partial cross-sectional view of still a further embodiment of a venturi system for use in a combustion system.

Detailed Description

[0014] In the drawings, there is shown a graphical representation of a typical combustor and a number of embodiments of the present invention. The combustor is generally designated by the numeral 10 and is contemplated to be incorporated into a gas turbine (not shown). In Fig. 1, the combustor 10 includes a combustion liner 12 that is at least partially surrounded by a heat shield or flow sleeve 14. Compressed air is provided from an upstream compressor (not shown) and fed into a surrounding channel 16 formed between the heat shield 14 and combustion liner 12.

[0015] Within the confines of the combustion liner 12 are provided a plurality of primary fuel nozzles 18 and a secondary fuel nozzle 20. A combustor flow sleeve 22 is provided surrounding the secondary fuel nozzle 20, separating the secondary nozzle 20 from the primary nozzles 18. Fuel 24 is introduced into a controller 26 and fed to the nozzles 18, 20. The area between the combustion liner 12 and combustor flow sleeve 22 defines an upstream or primary combustion chamber 28. Downstream of the secondary nozzle 20 is a venturi 30 defining an annular restriction within the area defined by the combustion liner 12. The venturi 30 includes an apex 32 that defines a venturi throat portion. The apex 32 is positioned downstream of the nozzles 18, 20. A downstream or secondary combustion chamber 34 is positioned within the combustor downstream of the venturi apex 32.

[0016] In operation, flame can reside on either side of the venturi 30, both in the upstream combustion chamber 28 and the downstream combustion chamber 34. In a primary operation, fuel 24 is provided to the primary nozzles 18, with combustion occurring in the upstream combustion chamber and directed downstream through the venturi throat 32. In a lean-lean operation, fuel 24 is provided to the primary nozzles 18 as well as to the secondary nozzle 20.

Combustion occurs in both the upstream combustion chamber 28 as well as the downstream combustion chamber 34. In addition, the combustor 10 may operate with fuel 24 directed solely to the secondary nozzle 20, creating combustion in the downstream combustion chamber 34. Additional fuel 24 may be provided to the primary nozzles 18 without flame being created in the upstream combustion chamber 28. The fuel and air pre-mix passing through the upstream chamber 28 is directed by the venturi 30 into the downstream combustion chamber 34 creating a pre-mix operation and a significant flame within the downstream chamber 34.

[0017] Since flame can reside on either side of the venturi 30, both sides of the walls of the venturi typically need to be cooled for proper operation and for longevity of the equipment. As shown in Fig. 1, cooling air in the surrounding channel 16 is directed on to the rear side of the venturi 30 and then directed along the inside walls of the venturi 30. The heat load on the upstream and downstream sides of the venturi 30 are contemplated to be different. In addition, movement of the cooling air back into the combustion chambers on either side of the venturi 30 also has an affect on operation of the combustor 10.

[0018] In Fig. 2, there is shown an enlarged schematic cross-sectional view of one side of the venturi 30 illustrated in Fig. 1. Compressed air 36 is directed into a plenum 38 on the back side of the venturi 30. In this embodiment, the inlet plenum 38 is positioned directly behind the apex 32 of the venturi 30. The cooling air directed to the venturi 30 by the inlet plenum is in fluid communication with two channels 40 and 42 that are generally parallel to a first or upstream side 41 of the venturi 30 and a second or downstream side 43 of the venturi 30. The cooling air 36 directed into the channel 40 is turned in an opposite direction from the cooling air directed into the channel 42. The air flow through the upstream channel 40 on the upstream side 41 of the venturi 30 is directed through an upstream exit 44 and into the primary combustion chamber 28.

The cooling flow passing through the downstream channel 42 and along the downstream side 43 is directed through exit 46, positioned adjacent the combustion liner 12, and into the downstream combustion chamber 34. A number of specific forms of exhaust exits for the venturi air are known in the prior art and may be incorporated into the venturi 30 as presently described.

[0019] In Fig. 3, a variation of the structures of the invention is shown. The cooling air 36 is directed from the channel 16 into an inlet plenum 138 having an outlet that is positioned adjacent the downstream side 43 of the venturi 30. The inlet plenum 138 is in direct fluid communication with the downstream channel 142. Cooling air is directed in a downstream direction through the channel 142 toward the downstream exit 46. An additional flow of cooling air is directed within the downstream channel portion 142a in the opposite direction from the downstream exit 46. Cooling channel portion 142a communicates with upstream channel 140, positioned behind the upstream side 41 of the venturi 30. The cooling air within the upstream channel 140 is directed through the upstream exit 44 and into the upstream combustion chamber 28. Positioning the cooling channel on the downstream face of the venturi reduces the cooling required for the downstream portion of the venturi. Alternatively, the inlet plenum 138 may be positioned adjacent the upstream side 41 of the venturi 30.

[0020] Thus, the venturi defines a transition between the first and second ends of the combustor. The internal passageway within the venturi includes a first side and a second side on opposite sides of an apex of the venturi. The cooling inlet for the compressed air flow is in fluid communication with the first and second sides of the venturi passageway and the compressed air from the cooling inlet is directed through the venturi passageway in opposite directions within the first side and second side. The cooling inlet may communicate with the venturi passageway

at a position in line with or offset from the apex of the venturi. The cooling inlet may be formed as a radial plenum positioned behind the venturi or may take any number of other forms.

[0021] In Fig. 4, a further embodiment of a venturi cooling system is provided. The cooling air 36 is directed from the channel 16 into an inlet plenum 238. The inlet plenum 238 directs the cooling air into the channels 240, 242 on the rear side of the venturi 30. As illustrated, the inlet plenum 238 is positioned adjacent the rear of the apex 32 of the venturi 30, with the upstream channel 240 directing cooling air along the upstream side 41 and the downstream channel 242 directing cooling air along the downstream side 43. The downstream end of channel 242 is in fluid communication with a secondary passageway 248. The cooling air within the channel 242 reverses in direction as it enters the secondary passageway 248. The secondary passageway 248 extends toward the upstream side of the combustor and joins with the upstream channel 240, adjacent an exhaust channel 250. The combined flow from the upstream channel 240 and secondary passageway 248 passes through the exhaust channel 250 and is exhausted through the upstream exit 244 into the combustion chamber 28 on the upstream side of the venturi 30.

[0022] In the embodiment shown in Fig. 4 (and in the embodiments described above), it is contemplated that inlet plenum 238 may comprise something other than a continuous slot or annular channel. The inlet plenum 238 and secondary passageway 248 may be formed by a series of pipes that cross over one another. Thus, the inlet plenum 238 feeds the back side of the venturi 30 after crossing over the series of pipes that form the passageway 248. Repositioning of the inlet plenum 238 with respect to the axial position of the apex 32 of the venturi 30 may also be utilized to adjust the cooling effect on the venturi 30. In addition, as in all of the embodiments discussed herein, the relative size and formation of the channels and passageways

may be varied so as to adjust the air flow and pressure drop of the cooling air as it moves inside the venturi walls and is then directed into the combustion chambers.

[0023] A variety of modifications to the embodiments described will be apparent to those skilled in the art upon review of the present disclosure. Thus, the invention may be embodied in other specific forms without departing from the spirit or essential attributes thereof and, accordingly, reference should be made to the appended claims, rather than to the foregoing specification, as indicating the scope of the invention.

CLAIMS

What is claimed is:

1. A combustor for a gas turbine comprising:
 - a nozzle assembly located at a first end of the combustor;
 - a combustion chamber defined within the combustor, downstream of the nozzle;
 - a venturi positioned within the combustor and defining a transition between the first end and the combustion chamber, the venturi having a venturi wall and a radially outer wall adjacent to and running along an interior of the combustor and an internal passageway therebetween, the internal passageway having first and second passageway portions and a radial plenum extending radially inward towards the wall of the venturi and defined by sidewalls that extend in a generally radial direction, with the radial plenum being in direct fluid communication with the first and second passageway portions, wherein the first passageway portion extends along the venturi wall towards the first end of the combustor and the second passageway portion extends along the venturi wall towards the second end of the combustor, and wherein the radial plenum engages the first and second passageway portions adjacent the wall of the venturi; and
 - a compressed air source directing cooling air into the first and second passageway portions via the radial plenum, the first and second passageway portions directing the compressed air from the source in opposite directions within the venturi.
2. The combustor of claim 1 further comprising a secondary passageway communicating with a downstream end of the second side of the venturi passageway, wherein the air flow moving through the second passageway portion into the secondary passageway is reversed in direction and directed towards the first side of the of the venturi.
3. The combustor of claim 2 further comprising a cooling air exit positioned at an end of the first passageway portion directing the cooling air into the first end of the combustor.

4. The combustor of claim 3 wherein the secondary passageway and the first passageway portion are each in fluid communication with the cooling air exit.

5. The combustor of claim 4 wherein the cooling air exit directs the air flow from the secondary passageway and the first passageway portion into the first end of the combustor, upstream of the venturi.

6. The combustor of claim 1, wherein the radial plenum communicates with the first passageway portion at a position behind the upstream side of the venturi within the combustor.

7. A combustor for a gas turbine comprising: a nozzle assembly located at a first end of the combustor;

a combustion chamber defined within the combustor, downstream of the nozzle;

a venturi positioned within the combustor and defining a transition between the first end and the combustion chamber, the venturi having a venturi wall and a radially outer wall adjacent to and running along an interior of the combustor and an internal passageway therebetween, the internal passageway having a first side and a second side located on and extending along opposite sides of an apex of the venturi and a radial plenum extending radially inward towards the venturi wall and defined by sidewalls that extend on a generally radial direction, the radial plenum being in direct fluid communication with the first and second sides, wherein the radial plenum engages the first and second sides adjacent the wall of the venturi; and

a compressed air flow source in fluid communication with the first and second sides of the internal passageway, via the radial plenum, and wherein the compressed air from the compressed air flow source is directed through the venturi passageway in opposite directions within the first and second sides for cooling the venturi.

8. The combustor of claim 7, wherein the radial plenum communicates with either the first or second side of the internal passageway at a position offset from the apex of the venturi within the combustor.

9. The combustor of claim 7 further comprising a secondary passageway communicating with a downstream end of the second side of the internal passageway, wherein the air flow moving through the second side of the internal passageway into the secondary passageway is reversed in direction and directed towards the first side of the of the venturi.

10. The combustor of claim 9 further comprising a cooling air exit positioned at an end of the first side of the internal passageway for directing the cooling air into the first end of the combustor.

11. The combustor of claim 10 wherein the secondary passageway and the first side of the internal passageway are each in fluid communication with the cooling air exit.

12. The combustor of claim 11 wherein the cooling air exit directs the air flow from the secondary passageway and the first side of the internal passageway into the first end of the combustor, upstream of the venturi.

13. A venturi within a gas turbine combustor comprising: a venturi wall and a radially outer venturi wall extending along an interior surface of the combustor and defining a first and second passageway;

an upstream angled side, a downstream angled side, an apex positioned between the upstream and downstream sides and a cooling air inlet extending radially inward towards the venturi wall and defined by sidewalls that extend in a generally radial direction, are connected to the radially outer venturi wall, and located between the angled sides for supplying cooling air proximate the apex;

the first passageway directing cooling air from the cooling air inlet along inner surface of downstream angled side and then upstream thereof along a liner;

the second passageway directing cooling air from the cooling air inlet along upstream angled side; and

a cooling discharge directing cooling air from the second passageway into combustor, upstream of the apex of the venturi.

14. The venturi of claim 13 wherein the cooling air inlet comprises a central plenum extending radially around the combustor, within the venturi.

15. The venturi of claim 13 wherein the cooling discharge is positioned upstream of the upstream angled side of the venturi.