

DESCRIPTION

METHOD FOR PREDICTING FRICTIONAL RESISTANCE OF ROUGH SURFACE,
AND APPARATUS FOR ESTIMATING SURFACE PERFORMANCE

TECHNICAL FIELD

5 [0001]

The present invention relates to a method which predicts
an increase in frictional resistance of a rough surface in
contact with a fluid in a simple manner, quickly and without
variations in predicted results among individuals, and to an
10 apparatus which estimates an increase in frictional resistance
using the prediction method.

BACKGROUND ART

[0002]

Increase in frictional resistance due to roughness on a
15 surface is an important issue in, for example, improving the
performance of ships or designing piping and flow channels.
The surface roughness of a rough surface is considered as the
main cause of an increase in frictional resistance. Since old
times, efforts have been made to predict an increase in
20 frictional resistance based on surface roughness. General
methods which estimate an impact of surface roughness of an
inside surface of a pipe involve Moody diagrams or Colebrook
equation as described in Non Patent Document 1. Although such
methods can be applied to fluids having various flow velocities

or various viscosities, the range of roughness wavelengths which can be studied is limited because of the fact that the surface that is evaluated is rendered rough by the attachment of sand grains and therefore the roughness wavelengths are dependent on the grain size of the sand grains. Further, these methods using Moody diagrams or Colebrook equation only take into consideration a relationship between the pipe diameter and the relative roughness height, and neglect profile parameters such as roughness wavelength. On the other hand, methods described in Patent Document 1 and Non Patent Document 2 are used to study the roughness on hulls. The methods of these documents focus on the roughness wavelength and calculate the ratio of the increase in frictional resistance based on a relationship of the square of roughness height and the roughness wavelength. However, the relationship that is studied in these documents assumes that the flow velocity is constant, with no attentions being paid to friction at various flow velocities. In practice, the resistance offered by roughness is increased to various degrees depending on the flow velocity (the speed of a ship or the flow rate in a pipe). Thus, the increase in resistance needs to be calculated at each of the expected flow velocities. However, none has disclosed a method which can predict how much a rough surface with various roughness wavelengths will increase the frictional resistance when in

contact with a fluid at various flow velocities, with high accuracy based on surface roughness.

CITATION LIST

PATENT DOCUMENT

5 [0003]

Patent Document 1: WO 2013 153877

NON PATENT DOCUMENT

[0004]

Non Patent Document 1: The Japan Society of Mechanical
10 Engineers, "Fluid Mechanics, JSME Text Series", (2007), p. 99

Non Patent Document 2: Hirohisa Mieno, Hiroshi Masuda,
"Friction Increase due to Roughness of Ship Hull Paint", Journal
of THE JAPAN INSTITUTE OF MARINE ENGINEERING, Vol. 48, No. 3
(2013), pp. 300-307

15 SUMMARY OF INVENTION

TECHNICAL PROBLEM

[0005]

The invention presents a method which predicts the
frictional resistance of a rough surface at various flow
20 velocities of a fluid, and an apparatus which estimates surface
performance using the method.

SOLUTION TO PROBLEM

[0006]

Under the circumstances discussed above, the present

inventors carried out extensive studies on methods and apparatuses which can predict an increase in frictional resistance of a rough surface having various wavelengths at various flow velocities of a fluid. As a result, the present
 5 inventors have found that the configurations described below make it possible to predict an increase in frictional resistance offered by a rough surface having various wavelengths, thus completing the present invention.

[0007]

10 [1]

A method for predicting the frictional resistance of a rough surface having a variation in roughness wavelength, and being in contact with a fluid flowing at varied velocities, the method including evaluating the total projected area A of
 15 all prominent peaks standing out above the viscous sublayer thickness per unit area (hereinafter, written as the "prominent peak projected area A"), and calculating the friction increase ratio FIR (%) using Equation (1) below or the frictional resistance increase $\Delta\tau$ using Equation (2) below.

20 [0008]

[Math. 1]

$$\text{FIR}(\%) = C \times A$$

(1)

[0009]

[Math. 2]

$$\Delta \tau = C_r \frac{1}{2} \rho A V^2 \quad (2)$$

(In Equation (1), the coefficient C is a constant
 5 dependent on the prominent peak projected area A and is
 determined by performing a frictional resistance test
 beforehand in which the frictional resistance is measured with
 respect to a plurality of rough surfaces differing in the degree
 of roughness while changing the flow velocity V and the friction
 10 increase ratios FIR (%) of the rough surfaces are calculated
 wherein the friction increase ratios FIR (%) are percentages
 of the difference $\tau_r - \tau_0$ between the frictional resistance τ_r
 of the rough surface and the frictional resistance τ_0 of a smooth
 surface, divided by τ_0 .

15 In Equation (2), the coefficient C_r is a constant
 dependent on the fluid density ρ , the prominent peak projected
 area A and the flow velocity V and is determined from the
 relation of Equation (2) using values of frictional resistance
 increase $\Delta \tau$ obtained beforehand by a frictional resistance test
 20 in which the frictional resistance is measured with respect
 to a plurality of rough surfaces differing in the degree of
 roughness while changing the flow velocity V wherein the values

of frictional resistance increase $\Delta\tau$ are the differences $\tau_r - \tau_0$ between the frictional resistance τ_r of the rough surface and the frictional resistance τ_0 of a smooth surface.)

[0010]

5 [2]

The method for predicting the frictional resistance of a rough surface described in [1], wherein the prominent peak projected area A is a value calculated using Equation (3) wherein R_{Sm} is the average length of roughness curve elements and R_c is the average height of roughness curve elements both measured as roughness wavelength λ and roughness height R , respectively, in accordance with the regulations in ISO 4287: 1997 (JIS B 0601: 2001).

[0011]

15 [Math. 3]

$$A = 0.5 \times \frac{(R_c - \delta s)^2}{R_c \times R_{Sm}} \quad (3)$$

(In Equation (3), δs is the viscous sublayer thickness measured by a frictional resistance test with respect to a smooth surface and is a value determined in such a manner that the frictional velocity u^* is calculated from the frictional resistance τ_0 of the smooth surface using Equation (5), δs is calculated from Equation (4) using the dimensionless distance

y^+ ($2 < y^+ < 8$) and the kinematic viscosity coefficient ν of a fluid of interest, and, from among the values of δs thus calculated, the value which provides a high coefficient of correlation between FIR (%) or $\Delta\tau$ and the prominent peak
 5 projected area A in Equation (1) or (2) is selected).

[0012]

[Math. 4]

$$\delta s = \frac{y^+ \times \nu}{u^*} \quad (4)$$

[0013]

10 [Math. 5]

$$u^* = \sqrt{\frac{\tau_0}{\rho}} \quad (5)$$

[0014]

[3]

The method for predicting the frictional resistance of
 15 a rough surface described in [2], wherein the prominent peak projected area A is calculated in such a manner that R_z , R_{zj} , R_q or R_a is measured as roughness height R' , the slope a of the R_c to the measured parameter is determined and the parameter is converted assuming that $R_c = a \times R'$ (R' is R_z , R_{zj} , R_q
 20 or R_a).

[0015]

[4]

The method for predicting the frictional resistance of a rough surface described in [1], wherein the prominent peak
 5 projected area A is a value calculated using Equation (6) wherein RS_m is the average length of roughness curve elements and Ra is the arithmetic average roughness both measured as roughness wavelength λ and roughness height R , respectively, in accordance with the regulations in ISO 4287: 1997 (JIS B
 10 0601: 2001).

[0016]

[Math. 6]

$$A = \frac{(Ra - \delta s)}{RS_m} \quad (6)$$

(In Equation (6), δs used to calculate the prominent peak
 15 projected area A is the viscous sublayer thickness measured by a frictional resistance test with respect to a smooth surface and is a value determined in such a manner that the frictional velocity u^* is calculated from the frictional resistance τ_0 of the smooth surface using Equation (5), δs is calculated from
 20 Equation (4) using the dimensionless distance y^+ ($2 < y^+ < 8$) and the kinematic viscosity coefficient ν of a fluid of interest, and, from among the values of δs thus calculated, the value which provides a high coefficient of correlation between FIR

(%) or $\Delta\tau$ and the prominent peak projected area A in Equation (1) or (2) is selected).

[0017]

[5]

5 The method for predicting the frictional resistance of a rough surface described in [4], wherein the prominent peak projected area A is calculated in such a manner that R_z , R_{zj} , R_q or R_c is measured as roughness height R'' , the slope a of the R_a to the measured parameter is determined and the parameter
10 is converted assuming that $R_a = a \times R''$ (R'' is R_z , R_{zj} , R_q or R_c).

[0018]

[6]

A surface performance estimating apparatus for
15 estimating surface performance by the prediction method described in any of [1] to [5], the apparatus including:

a measurement section configured to measure the roughness height R and the average length R_{Sm} of roughness curve elements, and

20 a calculation unit configured to calculate the prominent peak projected area A and to calculate the friction increase ratio FIR (%) using Equation (1) or the frictional resistance increase $\Delta\tau$ using Equation (2).

[0019]

[7]

The surface performance estimating apparatus described in [6], wherein the measurement section is configured to measure the roughness height R and the average length RSm of roughness curve elements of three dimensional roughness while reading the travel distance with a rotary encoder or a linear encoder and reading the displacement with a laser displacement meter using a two-dimensional beam.

[0020]

10 [8]

The surface performance estimating apparatus described in [6] or [7], wherein the roughness height R measured is Rz, Rzjis, Rq, Rc or Ra.

ADVANTAGEOUS EFFECTS OF INVENTION

15 [0021]

The method according to the present invention can calculate an increase in frictional resistance due to roughness at various flow velocities of a fluid, directly by a very simple evaluation of roughness. The use of this method makes it possible to determine the grade of treatment or the type of treatment performed or to be performed on the surface that will be brought into contact with a fluid flowing at a flow velocity of interest.

BRIEF DESCRIPTION OF DRAWINGS

[0022]

[Fig. 1] Fig. 1 is a diagram illustrating a relationship between the friction increase ratio FIR (%) and the prominent peak projected area $A = 0.5 \times (R_c - \delta_s)^2 / (R_c \times R_{Sm})$.

5 [Fig. 2] Fig. 2 is a diagram illustrating a relationship between the friction increase ratio FIR (%) and the prominent peak projected area $A = (R_a - \delta_s) / R_{Sm}$.

[Fig. 3] Fig. 3 is a schematic view illustrating a change of δ_s and prominent peak projected area by a change in flow
10 velocity.

[Fig. 4] Fig. 4 is a schematic view illustrating how the projected area a of a peak standing out above the viscous sublayer thickness δ_s is calculated using R_c .

[Fig. 5] Fig. 5 is a set of diagrams illustrating
15 relationships of R_c and roughness parameters R_z , R_a , R_q and R_{zjis} .

[Fig. 6] Fig. 6 is a set of diagrams illustrating relationships of R_a and roughness parameters R_z , R_c , R_q and R_{zjis} .

20 [Fig. 7] Fig. 7 is a diagram illustrating a relationship between $\Delta\tau$ (N/m²) and $0.5 \times \rho \times A \times V^2$ wherein $A = 0.5 \times (R_c - \delta_s)^2 / (R_c \times R_{Sm})$.

[Fig. 8] Fig. 8 is a diagram illustrating a relationship between $\Delta\tau$ (N/m²) and $0.5 \times \rho \times A \times V^2$ wherein $A = (R_a - \delta_s) / R_{Sm}$.

[Fig. 9] Fig. 9 is a schematic view of surface roughness measurement using a two-dimensional beam.

DESCRIPTION OF EMBODIMENTS

[0023]

5 The present invention will be described in detail hereinbelow.

 An aspect of the invention resides in a method for predicting an increase in frictional resistance due to a rough surface by evaluating the total projected area A of all
10 prominent peaks standing out above the viscous sublayer thickness per unit area, and calculating the friction increase ratio FIR (%) or the frictional resistance increase $\Delta\tau$.

[0024]

Concepts of the present invention

15 Fig. 3 illustrates rough surfaces having different roughness wavelengths. As illustrated, a variation in flow velocity brings about a change in δ_s , which is the thickness of a layer free from influences of roughness, causing a change in prominent peak projected area. At a higher flow velocity,
20 the viscous sublayer becomes smaller and the prominent peak projected area A is increased; thus, the friction increase ratio FIR (%) will be higher. Provided that the flow velocity is the same, the prominent peak projected area A is larger as the roughness is larger; the prominent peak projected area A is

larger as the number of peaks is increased, i.e., as the roughness wavelength is shorter. In this case too, the friction increase ratio FIR (%) will be increased. The present inventors believed that an increase in frictional resistance due to roughness on a surface could be easily predicted if it was possible to reveal how the roughness height, the wavelength and the viscous sublayer thickness played roles in bringing about an increase in resistance, thus completing the present invention.

10 [0025]

Calculation of total prominent peak projected area A per unit area from roughness parameters

Any of R_c (the average height of roughness curve elements) and R_a (the arithmetic average roughness) is measured as the roughness height R in accordance with the regulations in JIS B 0601: 2001 (ISO 4287: 1997), and the total projected area A of all prominent peaks standing out above the viscous sublayer per unit area ((hereinafter, written as the "prominent peak projected area A ")) is calculated using Equation (1) or (2) below.

[0026]

[Math. 7]

$$A = 0.5 \times \frac{(R_c - \delta_s)^2}{R_c \times R_{Sm}} \quad (1)$$

[0027]

[Math. 8]

$$A = \frac{(R_a - \delta_s)}{R_{Sm}} \quad (2)$$

5 [0028]

R_c is the average height of roughness curve elements, and R_a is the arithmetic average roughness. These parameters are measured in accordance with the regulations in JIS B 0601: 2001 (ISO 4287: 1997). Since the average height R_c of roughness
 10 curve elements is the average of roughness heights in a profile curve, the prominent peak projected area A may be calculated by the following procedures using the roughness height R_c , R_{Sm} and the viscous sublayer height δ_s .

[0029]

15 As illustrated in Fig. 4, the prominent peak projected area a of a single peak standing out above the viscous sublayer, assuming that the peak is a circular cone, is obtained as the area of a triangle defined by the height h and the base l . Thus,
 the projected area is calculated using $0.5 \times h \times l$ as shown
 20 in Equation (3). Because l can be represented using x in Fig.

4 as $R_{Sm} - 2x$, and h can be represented as $R_c - \delta s$, Equation
 (3) can be transformed through the procedures (3) to (9) using
 R_c and R_{Sm} into the final form (9). As shown in (11), the
 5 of the prominent peak projected area A is calculated as the product
 of the prominent peak projected area a of a single peak standing
 out above the viscous sublayer, multiplied by the number N of
 peaks per unit area calculated from Equation (10).

[0030]

[Math. 9]

$$10 \quad a = 0.5 \times h \times l \quad (3)$$

[0031]

[Math. 10]

$$h = R_c - \delta s \quad (4)$$

[0032]

15 [Math. 11]

$$l = R_{Sm} - 2x \quad (5)$$

[0033]

[Math. 12]

$$\tan \theta = \frac{R_c}{\frac{1}{2} R_{Sm}} = \frac{\delta s}{x} \quad (6)$$

[0034]

[Math. 13]

$$x = \frac{\frac{1}{2} RSm \times \delta s}{Rc} \quad (7)$$

[0035]

5 [Math. 14]

$$l = RSm - \frac{RSm \times \delta s}{Rc} = \frac{RSm \times Rc - RSm \times \delta s}{Rc} = \frac{RSm \times (Rc - \delta s)}{Rc} \quad (8)$$

[0036]

[Math. 15]

$$a = 0.5 \times h \times l = 0.5 \times (Rc - \delta s) \frac{RSm (Rc - \delta s)}{Rc} = 0.5 \times \frac{RSm \times (Rc - \delta s)^2}{Rc} \quad (9)$$

10 [0037]

[Math. 16]

$$N = \frac{1}{RSm^2} (l m^2) \quad (10)$$

[0038]

[Math. 17]

$$15 \quad A = a \times N = 0.5 \times \frac{RSm \times (Rc - \delta s)^2}{Rc} \times \frac{1}{RSm^2} = 0.5 \times \frac{(Rc - \delta s)^2}{RSm} (m^2) \quad (11)$$

[0039]

Ra is the arithmetic average roughness and is a value

obtained by folding the roughness curve $f(x)$ along the mean line and dividing the area defined by the roughness curve and the mean line with the length, as shown in Equation (12). Thus, this parameter can be regarded as the projected area per unit length. Therefore, as shown in Equation (13), the subtraction of δs produces the projected area a of prominent roughness standing out above the viscous sublayer.

As shown in Equation (14), the number N of such projected areas per unit length that are found per unit area is indicated by $1/RSm$ similarly to Equation (10). Thus, the prominent peak projected area A is calculated as shown in Equation (15).

[0040]

[Math. 18]

$$Ra = \frac{1}{l} \int_0^l |f(x)| dx \quad (12)$$

15 [0041]

[Math. 19]

$$a = Ra - \delta s \quad (13)$$

[0042]

[Math. 20]

$$N = \frac{1}{RSm} (/ m^2) \quad (14)$$

20

[0043]

[Math. 21]

$$A = a \times N = \frac{(Ra - \delta s)}{RSm} (m^2) \quad (15)$$

[0044]

5 Measurement of roughness

The roughness may be measured or evaluated with a surface roughness meter such as a contact or non-contact, manual or automatic type. A stylus profilometer or a laser displacement meter is generally used and is advantageous in aspects such as simplicity. In particular, a laser displacement meter such as one using a line laser (Ultra-High Speed In-line Profilometer LJ-V7000 Series) can perform three-dimensional measurement quickly. The data obtained may be stored or may be analog/digital processed within the displacement meter.

15 [0045]

In the parameter analysis, it is desired to use the profile curve as such. When, however, the influence of waviness of 10,000 μm and higher wavelengths is not negligible, the roughness curve may be measured while using a high-pass filter having a cut-off value (wavelength) λ_c of not less than 10,000 μm in accordance with JIS B 0601: 2001 (ISO 4287: 1997).

[0046]

In the present invention, the evaluation length and the

cut-off value λ_c which are necessary to evaluate the roughness accurately are not less than 10,000 μm , and the measurement pitches are not more than 500 μm . When the measurement pitches are 500 μm , the minimum wavelength that can be measured is 2,000 μm but the measurement results of low-wavelength roughness come to have a large margin of error. In view of this, the practical measurement pitches are about 250 μm . Reducing the measurement pitches leads to an increase in measurement time and also causes the measurement results to be affected by wavelengths of low roughness height which do not play a part in increasing the frictional resistance. In view of this, the practical measurement pitches are appropriately not less than 50 μm . When roughness with short wavelength is measured with an apparatus having long measurement pitches (a low resolution), some peaks are lost and the measurement may estimate the roughness height as being lower and the wavelength as being larger than they really are. It is therefore desirable to change the measurement pitches (the resolution) in accordance with the wavelength of the roughness of interest.

[0047]

Calculation of friction increase ratio FIR (%)

When the resistance due to friction with water stream is to be measured, the frictional resistance test may be performed by any of a towing test, a circular tube test, a double cylinder

test and a cavitation tank test. In the case where a double cylinder device is used, the outer cylinder is rotated at a rotational speed of 500 rpm to 1000 rpm, and a torque which works on an inner specular cylinder is measured beforehand while increasing the rotational speed by 50 rpm. The frictional resistance τ_0 (N/m²) of the smooth surface per unit area is thus determined. Next, a cylinder with a rough surface is caused to rotate under the same conditions, and the frictional resistance τ_r (N/m²) of the rough surface is determined at the respective rotational speeds. The friction increase ratio FIR (%) is calculated using Equation (16).

[0048]

[Math. 22]

$$\text{FIR}(\%) = \frac{\tau_r - \tau_0}{\tau_0} \times 100 \quad (16)$$

15 [0049]

The thickness of the viscous sublayer δ_s at each speed is measured by the frictional resistance test with respect to the smooth surface. Specifically, the frictional velocity u^* is calculated from the frictional resistance τ_0 of the smooth surface at each speed using Equation (17), δ_s is calculated from Equation (18) using the dimensionless distance y^+ ($2 < y^+ < 8$) and the kinematic viscosity coefficient ν of the fluid

of interest, and, from among the values of δs thus calculated, the value which provides a high coefficient of correlation between FIR (%) or $\Delta\tau$ and the prominent peak projected area A in Equation (19) or (24) described later is selected.

5 [0050]

[Math. 23]

$$u^* = \sqrt{\frac{\tau_0}{\rho}} \quad (17)$$

[0051]

[Math. 24]

$$\delta s = \frac{y^+ \times \nu}{u^*} \quad (18)$$

10

Here, u^* is the frictional velocity, y^+ the dimensionless distance ($y^+ = 2$ to 8), ν the kinematic viscosity coefficient, and ρ the fluid density.

[0052]

15 Determination of coefficient C

[0053]

[Math. 25]

$$\text{FIR}(\%) = C \times A \quad (19)$$

The slope C in Equation (19) varies depending on the type

of roughness height R and the frictional resistance test method. Rough surfaces differing in the degree of roughness are prepared beforehand, and their frictional resistances are evaluated by a frictional resistance test while changing the flow velocity.

- 5 The constant C in Equation (19) is determined from the relationship between the prominent peak projected area (A) and the FIR (%) thus measured.

[0054]

When, for example, R_c and R_{Sm} are measured over an
10 evaluation length of 30 mm and the frictional resistance test is performed using a double cylinder device, the coefficient C calculated using the prominent peak projected area A is 1274. When R_a and R_{Sm} are measured over an evaluation length of 30 mm and the frictional resistance test is performed using a
15 double cylinder device, the coefficient is 1800.

[0055]

From the foregoing, the frictional performance of a rough surface can be estimated by measuring R_c or R_a and further the average length R_{Sm} of roughness curve elements with respect
20 to the rough surface, calculating the prominent peak projected area A using the viscous sublayer thickness δ_s measured on a smooth surface at the flow velocity of interest, and predicting the friction increase ratio FIR (%) from Equation (19) using a constant C previously determined.

[0056]

Calculation of roughness resistance coefficient

In general, the drag (F) of a fluid is said to be proportional to the projected area S, the fluid density ρ and the flow velocity V, and the coefficient C_D of drag acting on roughness may be calculated as shown in Equation (20). The frictional resistance increase $\Delta\tau$ due to roughness is calculated using Equation (21), and, assuming that $\Delta\tau$ is the drag F, the frictional resistance coefficient C_r of the roughness is calculated from Equation (22) using the flow velocity V, the fluid density ρ and the prominent peak projected area A. In the case of a double cylinder test, the calculation of the flow velocity is simplified as shown in Equation (23) assuming that the flow velocity is decreased by half at the center between the outer cylinder and the inner cylinder. The flow velocity, however, can be calculated by other method that is selected appropriately in accordance with the manner in which the frictional resistance test is performed.

[0057]

20 [Math. 26]

$$C_D = \frac{F}{\frac{1}{2} \rho S V^2} \quad (20)$$

[0058]

[Math. 27]

$$\Delta \tau = \tau_r - \tau_0 \quad (21)$$

[0059]

[Math. 28]

$$C_r = \frac{\Delta \tau}{\frac{1}{2} \rho A V^2} \quad (22)$$

5

[0060]

[Math. 29]

$$V = \frac{1}{2} \pi D R \frac{1}{60} (m/s)$$

D: outer cylinder diameter R(m): rotational speed (rpm) (23)

[0061]

10 Estimation of frictional resistance increase $\Delta \tau$ (N/m²) due to roughness using roughness resistance coefficient C_r

By using the roughness resistance coefficient C_r calculated beforehand, the prominent peak projected area A, the fluid density and the flow velocity V, the frictional
 15 resistance increase $\Delta \tau$ (N/m²) due to roughness can be calculated using Equation (24). The friction performance of the rough surface can be thus evaluated.

[0062]

[Math. 30]

$$\Delta \tau = C_r \frac{1}{2} \rho A V^2 (\text{N/m}^2) \quad (24)$$

[0063]

A surface performance estimating apparatus of the present invention uses the prediction method described hereinabove.

5 The apparatus includes a measurement section configured to measure the roughness height R and the average length RSm of roughness curve elements, and a frictional resistance calculation unit configured to calculate the prominent peak projected area A using Equations (2) and (3) and to calculate
10 the friction increase ratio FIR (%) using Equation (19) or the frictional resistance increase $\Delta \tau$ (N/m²) due to roughness using Equation (24). With this surface evaluation apparatus, the friction increase ratio of a rough surface can be predicted easily and the surface performance can be estimated.

15 [0064]

[EXAMPLES]

The present invention will be described in greater detail based on Examples hereinbelow. However, the scope of the invention is not limited to such Examples.

20 [Example 1]

Double cylinder test and prominent peak projected area $A = 0.5$
 $\times (R_c - \delta_s)^2 / (R_c \times R_{Sm})$

With a double cylinder device, the prominent peak projected area A was calculated from the average element height R_c , the average length R_{Sm} of roughness curve elements and the viscous sublayer thickness δ_s , and the relationship thereof with the friction increase ratio FIR (%) was evaluated.

[0065]

To obtain sufficient accuracy, the surface roughness of the inner cylinder on which a coating had been formed was analyzed with respect to 58 lines with use of a laser displacement meter, starting from 5 mm above the bottom of the test piece to the top of the test piece at intervals of 5 mm. The displacement data was obtained for every 250 μm over a length of 1,000 mm, namely, 4,000 points of data were obtained. The measurement pitches were thus 250 μm . The data measured with respect to one line was divided into 33 segments each corresponding to an evaluation length of 30 mm, and thereafter an approximate curve obtained by a mean square method was subtracted. A profile curve was thus obtained. Fifteen test inner cylinders A to O were provided.

20 [0066]

In actual evaluation, a roughness curve is desirably obtained while using a low-pass filter having a cut-off wavelength λ_s in order to remove the waviness influence due to long wavelengths. Here, because the cylinders used had been

processed by a high-precision technique and had no waviness influence due to long wavelengths, the profile curve was evaluated as such.

[0067]

5 From the profile curve, R_z (the highest roughness), R_{zjis} (the ten point average roughness), R_c (the average height of roughness curve elements), R_a (the arithmetic average roughness), R_q (the root mean square roughness) and R_{Sm} were calculated.

10 [0068]

Table 1 describes the results of R_z , R_{zjis} , R_c , R_a , R_q , R_{sk} , R_{ku} and R_{Sm} measured with a double cylinder device which included any of the cylinders A to O. The cylinder A alone had extremely small wavelengths and the roughness was so small for
15 250 μm pitches that the wavelengths were measured to be longer than they really were; the results thereof described in Table 1 are values obtained by remeasurement with respect to one region 30 mm $\times$ 30 mm with use of a laser displacement meter having measurement pitches of 50 μm .

20 [0069]

The double cylinder device was such that a stainless steel tank (an outer cylinder, ϕ 320 mm diameter) was filled with seawater (23°C, density $\rho = 1023.95$ (kg/m³), kinematic viscosity 9.9812×10^{-7} (m²/s)), and a vinyl chloride test

cylinder (ϕ 310 mm $\times$ H 300 mm) whose surface had been roughened by spraying of a paint thereto was placed therein. The outer cylinder was rotated at 500 rpm to 1000 rpm, and the torque which worked on the coated rough inner cylinder was measured sequentially at the respective rotational speeds. Because a variation in the thickness of the coatings formed on the coated cylinders would give rise to changes in frictional resistance τ_0 (N/m²) and viscous sublayer thickness offered by smooth surfaces, a smooth cylinder (ϕ 311 mm $\times$ H 300 mm) was provided as a cylinder having a film thickness of 500 μ m and the torque which worked thereon was measured, and another smooth cylinder (ϕ 309 mm $\times$ H 300 mm) was provided as a cylinder having a film thickness of -500 μ m and the torque which worked thereon was measured. The frictional resistance τ_0 of the smooth surface and the viscous sublayer thickness were corrected as if the thickness of the corresponding coating had been added. Table 2 describes the frictional resistance τ_0 of the smooth surface of each of the cylinders after the correction based on film thickness. Table 3 describes the frictional resistance τ_r of the rough surface of each of the cylinders.

[0070]

The friction increase ratios FIR (%) of the roughened inner cylinders at 500 to 1000 (rpm) were calculated using Equation (16), the results being described in Table 4. In this

example, the viscous sublayer thickness δ_s was calculated using Equations (17) and (18) assuming that the dimensionless distance (y^+) was 4.0, the results being described in Table 5. Using the viscous sublayer thickness δ_s in Table 5 and the average element height R_c of each cylinder described in Table 1, the prominent peak projected area A was calculated from Equation (1), the results being described in Table 6. The graph in Fig. 1 plots the prominent peak projected area A in Table 6 on the abscissa and the FIR (%) in Table 4 on the ordinate. $C = 1274$.

[0071]

An example of resistance prediction using this example will be described below.

Provided that the viscous sublayer thickness δ_s of the fluid of interest is $10 \mu\text{m}$ (dimensionless distance $y^+ = 4.0$), R_c is $40 \mu\text{m}$ and the wavelength is $3000 \mu\text{m}$, the prominent peak projected area A calculated from Equation (1) is $0.0075 \text{ (m}^2\text{)}$. Because $C = 1274$, the FIR (%) is calculated to be 9.6% from Equation (19).

[0072]

[Example 2]

Frictional resistance test Prominent peak projected area $A =$
 $(R_a - \delta_s) / R_{Sm}$

The frictional resistance measurement method and the

roughness measurement method are the same as described in [Example 1]. In this example, the viscous sublayer thickness δ_s was calculated using Equations (17) and (18) assuming that the dimensionless distance (y^+) was 2.5, the results being

5 described in Table 7. Using the viscous sublayer thickness δ_s in Table 7 and the average element height R_a of each cylinder described in Table 1, the prominent peak projected area A was calculated from Equation (2), the results being described in Table 8.

10 The graph in Fig. 2 plots the prominent peak projected area A in Table 8 on the abscissa and the FIR (%) in Table 4 on the ordinate. $C = 1800$.

An example of resistance prediction using this example will be described below.

15 [0073]

Provided that the viscous sublayer thickness δ_s of the fluid of interest is $6.5 \mu\text{m}$ (dimensionless distance $y^+ = 2.5$), R_a is $20 \mu\text{m}$ and the wavelength is $3000 \mu\text{m}$, the prominent peak projected area A calculated from Equation (2) is $0.0045 \text{ (m}^2\text{)}$.

20 Because $C = 1800$, the FIR (%) is calculated to be 8.1% from Equation (19).

[0074]

[Example 3]

Regarding the roughness parameters of the cylinders

described in Table 1, the correlations between any of R_z , R_a , R_q and R_{zjis} , and R_c are shown in Fig. 5, and the correlations between any of R_z , R_c , R_q and R_{zjis} , and R_a are shown in Fig. 6. The high correlations in all the cases show that any roughness height R may be used for the evaluation. Based on the results, when R_z , R_{zjis} or R_q is used as the roughness height R , the parameter R may be used in the calculation of the projected area with Equation (1) or Equation (2) after being converted using the slope a or a' of the parameter and R_c or R_a into:

$$R_c = a \times R \text{ or } R_a = a' \times R$$

[0075]

[Example 4]

Using the flow velocity V , the fluid density ρ and the prominent peak projected area A in Table 6 calculated in [Example 1], $0.5 \times \text{fluid density } \rho \times \text{prominent peak projected area } A \times \text{squared flow velocity } V$ was calculated, the results being described in Table 10. Here, the flow velocity V , shown in an upper row in Table 10, is a value obtained by simplified calculation with Equation (23) assuming that the flow velocity in the case of a double cylinder test would be decreased by half at the center between the outer cylinder and the inner cylinder. Fig. 7 shows a graph plotting the product of $0.5 \times \text{fluid density } \rho \times \text{prominent peak projected area } A \times \text{squared flow$

velocity V described in Table 10 on the abscissa and $\Delta\tau$ in Table 9 on the ordinate. $C_r = 0.0556$.

[0076]

An example of resistance prediction using this example
5 will be described below.

Provided that the fluid of interest has a density $\rho =$
1023.95 (kg/m³) and flows at a velocity of 8.4 (m/sec), the
viscous sublayer thickness δ_s is 10 μm (dimensionless distance
 $y^+ = 4.0$), R_c is 40 μm and the wavelength is 3000 μm , the
10 prominent peak projected area A calculated from Equation (1)
is 0.0075 (m²). Because $C_r = 0.0556$, the frictional resistance
increase $\Delta\tau$ is calculated to be 15 (N/m²) from Equation (24).

[0077]

[Example 5]

15 Using the frictional resistance τ_0 of the smooth surface
described in Table 2 and the frictional resistance τ_r of the
rough surface described in Table 3, the frictional resistance
increase $\Delta\tau$ (N/m²) of the rough surface compared to the smooth
surface was calculated from Equation (21), the results being
20 described in Table 9. Using the flow velocity V , the fluid
density ρ and the prominent peak projected area A in Table 8
calculated in [Example 2], $0.5 \times \text{fluid density } \rho \times \text{prominent}$
peak projected area $A \times \text{squared flow velocity } V$ was calculated,
the results being described in Table 11. Here, the flow

velocity V , shown in an upper row in Table 11, is a value obtained by simplified calculation with Equation (23) assuming that the flow velocity V in the case of a double cylinder test would be decreased by half at the center between the outer cylinder and the inner cylinder. Fig. 8 shows a graph plotting the product of $0.5 \times \text{fluid density } \rho \times \text{prominent peak projected area } A \times \text{squared flow velocity } V$ described in Table 11 on the abscissa and $\Delta\tau$ in Table 9 on the ordinate. $C_r = 0.0782$.

[0078]

10 An example of resistance prediction using this example will be described below.

 Provided that the fluid of interest has a density $\rho = 1023.95 \text{ (kg/m}^3\text{)}$ and flows at a velocity of 8.4 (m/sec) , the viscous sublayer thickness δ_s of the fluid of interest is $6.5 \mu\text{m}$ (dimensionless distance $y^+ = 2.5$), Ra is $20 \mu\text{m}$ and the wavelength is $3000 \mu\text{m}$, the total prominent peak projected area A per unit area calculated from Equation (2) is $0.0045 \text{ (m}^2\text{)}$. Because $C_r = 0.0782$, the frictional resistance increase $\Delta\tau$ is calculated to be $12.7 \text{ (N/m}^2\text{)}$ from Equation (24).

20 [0079]

[Example 6]

 Fig. 9 illustrates an example surface roughness measuring apparatus which includes LJ-V7080 manufactured by KEYENCE CORPORATION as a measurement section of a displacement meter

using a two-dimensional beam line laser, and optical scale sensor VP-90 manufactured by KEYENCE CORPORATION as a travel distance reading section. With use of a scale attached to a wheel, the optical scale sensor can record displacements at measurement pitches of about 50 μm and can output data as three-dimensional shapes. With this configuration, a three-dimensional measurement apparatus capable of quick measurement of surface roughness can be realized. According to this configuration, the projected area and number of roughness in the flowing direction can be measured at the same time, and the prominent peak projected area A can be determined very quickly. Data for resistance prediction can be thus obtained.

[0080]

[Table 1]

Table 1 Results of measurement of roughness parameters

Cylinders Nos.	Film thickness (μm)	Rz (μm)	Rzjis (μm)	Rc (μm)	Ra (μm)	Rq (μm)	Rsk	Rku	RSm (μm)
A	262	166.9	134.1	64.8	22.9	28.9	0.2	3.1	998
B	321	256.9	133.1	148.3	51.0	63.3	0.5	3.0	5604
C	242	149.9	105.7	76.3	26.7	32.8	0.1	2.7	3214
D	324	156.9	105.4	77.6	27.0	33.6	0.2	2.9	3404
E	219	140.1	97.8	68.2	24.0	29.8	0.2	2.9	3042
F	309	144.9	96.4	71.4	25.3	31.4	0.2	2.9	3449
G	257	134.6	92.9	65.4	23.2	28.8	0.2	2.9	3112
H	216	117.7	78.1	55.1	19.1	24.3	0.6	3.5	3057
I	391	104.1	57.0	55.2	19.7	24.4	0.3	2.9	5111
J	454	84.0	53.9	41.6	14.7	18.3	0.2	3.0	3777
K	311	75.2	47.1	35.0	12.4	15.9	0.6	3.7	3523
L	484	75.0	46.9	36.9	13.1	16.3	0.2	3.0	3910
M	46	62.0	40.8	28.2	10.3	13.0	0.3	3.2	3095
N	140	50.4	32.5	23.5	8.7	10.9	0.3	3.1	3385
O	283	38.7	18.6	20.9	7.8	9.6	0.1	2.8	6063

The roughness was measured at measurement pitches of 50 μm for Cylinder A and at measurement pitches of 250 μm for Cylinders B to O.

[0081]

[Table 2]

Table 2 Frictional resistance τ_s of smooth surface of cylinders (corrected based on film thickness)

(rpm)	Frictional resistance τ_0 of smooth surface (N/m ²)													
	1000	950	900	850	800	750	700	650	600	550	500			
A	150.1	137.2	124.9	113.0	101.2	90.2	80.1	70.2	60.9	52.4	44.5			
B	150.5	137.5	125.2	113.3	101.5	90.5	80.4	70.4	61.1	52.6	44.6			
C	149.9	137.0	124.7	112.9	101.1	90.2	80.0	70.1	60.9	52.4	44.4			
D	150.5	137.5	125.2	113.3	101.5	90.5	80.4	70.4	61.1	52.6	44.6			
E	149.7	136.9	124.6	112.7	101.0	90.0	79.9	70.0	60.8	52.3	44.4			
F	150.4	137.4	125.1	113.3	101.4	90.5	80.3	70.4	61.0	52.5	44.6			
G	150.0	137.1	124.8	113.0	101.2	90.2	80.1	70.2	60.9	52.4	44.5			
H	149.7	136.8	124.6	112.7	101.0	90.0	79.9	70.0	60.8	52.3	44.4			
I	151.0	138.0	125.6	113.7	101.8	90.8	80.6	70.7	61.3	52.7	44.7			
J	151.4	138.3	126.0	114.0	102.1	91.1	80.8	70.9	61.4	52.9	44.8			
K	150.4	137.5	125.2	113.3	101.4	90.5	80.3	70.4	61.1	52.5	44.6			
L	151.6	138.5	126.2	114.2	102.2	91.2	80.9	71.0	61.5	52.9	44.9			
M	148.4	135.7	123.4	111.6	100.0	89.2	79.2	69.3	60.2	51.8	44.0			
N	149.2	136.3	124.1	112.2	100.6	89.7	79.6	69.7	60.5	52.1	44.2			
O	150.2	137.3	125.0	113.1	101.3	90.3	80.2	70.3	61.0	52.5	44.5			

[0082]

[Table 3]

Table 3 Frictional resistance t_r of rough surface of cylinders

(rpm)	Frictional resistance t_r of rough surface (N/m ²)													
	1000	950	900	850	800	750	700	650	600	550	500			
A	191.0	173.9	156.7	141.3	126.3	111.8	98.5	86.1	74.0	63.2	53.1			
B	172.0	157.0	143.1	128.7	115.4	102.9	91.0	79.5	69.0	58.9	49.9			
C	165.6	151.3	137.5	123.9	111.1	98.4	87.3	76.2	66.0	56.4	47.7			
D	165.9	151.8	137.9	124.3	111.3	99.3	87.5	76.3	66.1	56.3	47.7			
E	167.6	152.8	138.9	124.7	111.9	99.2	87.6	76.5	66.2	56.7	47.7			
F	168.1	153.0	138.6	125.0	111.8	99.3	87.6	76.6	66.3	56.7	47.9			
G	170.3	155.0	140.9	127.0	113.5	100.7	88.9	77.9	67.3	57.5	48.7			
H	166.0	150.9	137.0	123.9	110.3	97.8	86.8	75.7	65.2	56.0	47.3			
I	157.1	143.4	130.5	117.5	105.6	94.3	83.1	72.8	63.2	53.9	45.6			
J	158.0	144.2	131.2	118.3	106.1	94.7	83.8	73.4	63.8	54.9	46.4			
K	158.2	144.6	130.9	118.1	105.8	94.6	83.2	72.6	63.2	54.0	45.8			
L	158.2	144.5	131.2	118.7	106.4	95.1	83.9	73.6	63.7	54.7	46.4			
M	153.3	139.9	127.3	115.0	102.7	91.8	81.0	70.8	61.4	52.7	44.5			
N	152.5	139.3	126.8	114.2	102.4	91.0	80.7	70.8	61.2	52.3	44.5			
O	152.9	139.7	127.2	115.0	103.6	92.1	81.1	71.6	61.8	53.2	45.0			

[0083]

[Table 4]

Table 4 Friction increase ratio FIR (%) of coated cylinders

(rpm)	1000	950	900	850	800	750	700	650	600	550	500
A	27.3	26.8	25.5	25.1	24.8	23.9	22.9	22.6	21.4	20.6	19.4
B	14.3	14.2	14.3	13.6	13.8	13.6	13.3	12.8	12.9	12.1	11.8
C	10.5	10.4	10.2	9.8	9.9	9.1	9.1	8.7	8.4	7.7	7.2
D	10.2	10.3	10.1	9.6	9.6	9.7	8.8	8.4	8.3	7.1	6.9
E	11.9	11.6	11.5	10.7	10.8	10.2	9.5	9.2	8.9	8.4	7.4
F	11.8	11.3	10.8	10.3	10.2	9.8	9.1	8.8	8.5	7.9	7.4
G	13.5	13.0	12.9	12.4	12.2	11.6	11.0	11.0	10.4	9.8	9.5
H	10.9	10.2	10.0	9.9	9.3	8.6	8.6	8.1	7.3	7.1	6.6
I	4.0	3.9	3.9	3.3	3.7	3.9	3.1	3.0	3.2	2.2	1.9
J	4.4	4.2	4.2	3.7	3.9	4.0	3.6	3.6	3.9	3.8	3.5
K	5.2	5.2	4.6	4.3	4.3	4.5	3.6	3.2	3.6	2.9	2.6
L	4.3	4.3	4.0	3.9	4.0	4.3	3.6	3.7	3.7	3.3	3.4
M	3.3	3.1	3.1	3.0	2.7	2.9	2.3	2.1	2.0	1.8	1.3
N	2.2	2.2	2.2	1.7	1.8	1.4	1.3	1.5	1.0	0.3	0.8
O	1.8	1.7	1.8	1.7	2.3	1.9	1.1	1.9	1.3	1.5	1.1

[0084]

[Table 5]

Table 5 Viscous sublayer thickness δ_s ($y^* = 4$) of cylinders

(rpm)	1000	950	900	850	800	750	700	650	600	550	500
A	10.43	10.91	11.43	12.02	12.70	13.45	14.27	15.24	16.37	17.64	19.15
B	10.41	10.89	11.42	12.00	12.68	13.43	14.25	15.22	16.34	17.62	19.12
C	10.43	10.91	11.44	12.02	12.71	13.45	14.28	15.25	16.37	17.65	19.16
D	10.41	10.89	11.42	12.00	12.68	13.43	14.25	15.22	16.34	17.62	19.12
E	10.44	10.92	11.45	12.03	12.71	13.46	14.29	15.26	16.38	17.66	19.17
F	10.42	10.90	11.42	12.00	12.68	13.43	14.25	15.23	16.35	17.62	19.13
G	10.43	10.91	11.43	12.02	12.70	13.45	14.27	15.25	16.37	17.65	19.15
H	10.44	10.92	11.45	12.03	12.71	13.46	14.29	15.26	16.39	17.67	19.17
I	10.40	10.88	11.40	11.98	12.66	13.40	14.23	15.19	16.32	17.59	19.09
J	10.38	10.86	11.38	11.96	12.64	13.38	14.21	15.17	16.30	17.56	19.07
K	10.42	10.90	11.42	12.00	12.68	13.43	14.25	15.22	16.35	17.62	19.13
L	10.37	10.85	11.37	11.95	12.63	13.38	14.20	15.16	16.29	17.55	19.06
M	10.49	10.97	11.50	12.09	12.77	13.53	14.36	15.34	16.46	17.75	19.27
N	10.46	10.94	11.47	12.06	12.74	13.49	14.32	15.30	16.42	17.70	19.21
O	10.42	10.90	11.43	12.01	12.69	13.44	14.26	15.24	16.36	17.63	19.14

[0085]

[Table 6]

Table 6 Prominent peak projected area $A = 0.5 \times (Rc - \delta s)^2 / (Rc \times RSm)$ of cylinders

(rpm)	1000	950	900	850	800	750	700	650	600	550	500
A	0.0229	0.0225	0.0220	0.0215	0.0210	0.0204	0.0197	0.0190	0.0181	0.0172	0.0161
B	0.0114	0.0114	0.0113	0.0112	0.0111	0.0109	0.0108	0.0107	0.0105	0.0103	0.0100
C	0.0088	0.0087	0.0086	0.0084	0.0082	0.0081	0.0078	0.0076	0.0073	0.0070	0.0067
D	0.0085	0.0084	0.0083	0.0081	0.0080	0.0078	0.0076	0.0074	0.0071	0.0068	0.0065
E	0.0080	0.0079	0.0078	0.0076	0.0074	0.0072	0.0070	0.0068	0.0065	0.0062	0.0058
F	0.0076	0.0074	0.0073	0.0072	0.0070	0.0068	0.0066	0.0064	0.0062	0.0059	0.0055
G	0.0074	0.0073	0.0072	0.0070	0.0068	0.0066	0.0064	0.0062	0.0059	0.0056	0.0053
H	0.0059	0.0058	0.0057	0.0055	0.0053	0.0051	0.0049	0.0047	0.0044	0.0042	0.0038
I	0.0036	0.0035	0.0034	0.0033	0.0032	0.0031	0.0030	0.0028	0.0027	0.0025	0.0023
J	0.0031	0.0030	0.0029	0.0028	0.0027	0.0025	0.0024	0.0022	0.0020	0.0018	0.0016
K	0.0025	0.0024	0.0023	0.0021	0.0020	0.0019	0.0017	0.0016	0.0014	0.0012	0.0010
L	0.0024	0.0024	0.0023	0.0022	0.0020	0.0019	0.0018	0.0016	0.0015	0.0013	0.0011
M	0.0018	0.0017	0.0016	0.0015	0.0014	0.0012	0.0011	0.0009	0.0008	0.0006	0.0005
N	0.0011	0.0010	0.0009	0.0008	0.0007	0.0006	0.0005	0.0004	0.0003	0.0002	0.0001
O	0.0004	0.0004	0.0004	0.0003	0.0003	0.0002	0.0002	0.0001	0.0001	0.0000	0.0000

[0086]

[Table 7]

Table 7 Viscous sublayer thickness δ_s of cylinders (Example 2)

	(rpm)	δ_s ($y^+ = 2.5$) (μm)										
		1000	950	900	850	800	750	700	650	600	550	500
A	6.52	6.82	6.82	7.15	7.51	7.94	8.40	8.92	9.53	10.23	11.03	11.97
B	6.51	6.81	6.81	7.13	7.50	7.93	8.39	8.91	9.51	10.21	11.01	11.95
C	6.52	6.82	6.82	7.15	7.51	7.94	8.41	8.92	9.53	10.23	11.03	11.97
D	6.51	6.81	6.81	7.13	7.50	7.92	8.39	8.91	9.51	10.21	11.01	11.95
E	6.52	6.82	6.82	7.15	7.52	7.95	8.41	8.93	9.54	10.24	11.04	11.98
F	6.51	6.81	6.81	7.14	7.50	7.93	8.39	8.91	9.52	10.22	11.01	11.95
G	6.52	6.82	6.82	7.15	7.51	7.94	8.41	8.92	9.53	10.23	11.03	11.97
H	6.53	6.83	6.83	7.15	7.52	7.95	8.41	8.93	9.54	10.24	11.04	11.98
I	6.50	6.80	6.80	7.12	7.49	7.91	8.38	8.89	9.50	10.20	10.99	11.93
J	6.49	6.79	6.79	7.11	7.48	7.90	8.37	8.88	9.48	10.19	10.98	11.92
K	6.51	6.81	6.81	7.14	7.50	7.93	8.39	8.91	9.52	10.22	11.01	11.95
L	6.48	6.78	6.78	7.11	7.47	7.90	8.36	8.87	9.48	10.18	10.97	11.91
M	6.55	6.85	6.85	7.19	7.56	7.98	8.45	8.97	9.59	10.29	11.09	12.04
N	6.54	6.84	6.84	7.17	7.54	7.96	8.43	8.95	9.56	10.26	11.06	12.01
O	6.51	6.81	6.81	7.14	7.51	7.93	8.40	8.91	9.52	10.22	11.02	11.96

[0087]

[Table 8]

Table 8 Prominent peak projected area $A = (Ra - \delta s)/RSm$ of cylinders (Example 2)

(rpm)	1000	950	900	850	800	750	700	650	600	550	500
A	0.0164	0.0161	0.0158	0.0154	0.0150	0.0145	0.0140	0.0134	0.0127	0.0119	0.0110
B	0.0079	0.0079	0.0078	0.0078	0.0077	0.0076	0.0075	0.0074	0.0073	0.0071	0.0070
C	0.0063	0.0062	0.0061	0.0060	0.0058	0.0057	0.0055	0.0053	0.0051	0.0049	0.0046
D	0.0060	0.0059	0.0058	0.0057	0.0056	0.0055	0.0053	0.0051	0.0049	0.0047	0.0044
E	0.0057	0.0056	0.0055	0.0054	0.0053	0.0051	0.0050	0.0048	0.0045	0.0043	0.0040
F	0.0054	0.0054	0.0053	0.0052	0.0050	0.0049	0.0048	0.0046	0.0044	0.0041	0.0039
G	0.0054	0.0053	0.0052	0.0050	0.0049	0.0048	0.0046	0.0044	0.0042	0.0039	0.0036
H	0.0041	0.0040	0.0039	0.0038	0.0036	0.0035	0.0033	0.0031	0.0029	0.0026	0.0023
I	0.0026	0.0025	0.0025	0.0024	0.0023	0.0022	0.0021	0.0020	0.0019	0.0017	0.0015
J	0.0022	0.0021	0.0020	0.0019	0.0018	0.0017	0.0015	0.0014	0.0012	0.0010	0.0007
K	0.0017	0.0016	0.0015	0.0014	0.0013	0.0011	0.0010	0.0008	0.0006	0.0004	0.0001
L	0.0017	0.0016	0.0015	0.0014	0.0013	0.0012	0.0011	0.0009	0.0007	0.0005	0.0003
M	0.0012	0.0011	0.0010	0.0009	0.0007	0.0006	0.0004	0.0002	0.0000	-0.0003	-0.0006
N	0.0006	0.0006	0.0005	0.0003	0.0002	0.0001	-0.0001	-0.0003	-0.0005	-0.0007	-0.0010
O	0.0002	0.0002	0.0001	0.0000	0.0000	-0.0001	-0.0002	-0.0003	-0.0004	-0.0005	-0.0007

[0088]

[Table 9]

Table 9 Frictional resistance increase Δt (N/m²)

(rpm)	Frictional resistance increase Δt (N/m ²)														
	1000	950	900	850	800	750	700	650	600	550	500				
A	41.0	36.7	31.8	28.3	25.1	21.6	18.4	15.9	13.1	10.8	8.6				
B	21.5	19.5	17.9	15.4	14.0	12.3	10.7	9.0	7.9	6.3	5.3				
C	15.7	14.3	12.8	11.1	10.0	8.2	7.3	6.1	5.1	4.0	3.2				
D	15.4	14.2	12.6	10.9	9.8	8.7	7.1	5.9	5.1	3.8	3.1				
E	17.8	15.9	14.3	12.0	10.9	9.2	7.6	6.4	5.4	4.4	3.3				
F	17.7	15.6	13.5	11.7	10.3	8.8	7.3	6.2	5.2	4.2	3.3				
G	20.3	17.9	16.1	14.0	12.4	10.5	8.8	7.7	6.4	5.1	4.2				
H	16.3	14.0	12.4	11.2	9.4	7.8	6.9	5.7	4.4	3.7	2.9				
I	6.1	5.4	4.9	3.8	3.8	3.5	2.5	2.1	2.0	1.2	0.9				
J	6.6	5.9	5.2	4.3	4.0	3.6	2.9	2.5	2.4	2.0	1.6				
K	7.8	7.1	5.8	4.9	4.4	4.1	2.9	2.3	2.2	1.5	1.2				
L	6.6	6.0	5.0	4.5	4.1	3.9	3.0	2.7	2.3	1.8	1.5				
M	4.8	4.2	3.9	3.4	2.7	2.6	1.8	1.4	1.2	0.9	0.5				
N	3.3	2.9	2.8	2.0	1.9	1.3	1.1	1.0	0.6	0.2	0.3				
O	2.7	2.4	2.2	1.9	2.3	1.7	0.9	1.3	0.8	0.8	0.5				

[0089]

[Table 10]

Table 10 Calculation of $0.5 \times \text{fluid density} \times \text{prominent peak projected area} \times \text{squared flow velocity}$ V (Example 4)

{rpm) (m/sec)	0.5 × ρ × A × V ² A = 0.5 × (Rc - δs) ² / (Rc × RSm)											
	1000	950	900	850	800	750	700	650	600	550	500	
	8.4	8.0	7.5	7.1	6.7	6.3	5.9	5.4	5.0	4.6	4.2	
A	821.6	728.5	641.2	559.4	482.8	412.3	347.7	288.4	234.7	187.0	144.8	
B	411.1	368.4	328.1	290.2	254.5	221.2	190.4	161.8	135.5	111.7	90.2	
C	317.8	282.7	249.7	218.7	189.6	162.8	138.1	115.4	94.7	76.2	59.8	
D	307.0	273.1	241.3	211.5	183.5	157.6	133.8	111.8	91.9	74.0	58.1	
E	288.9	256.4	225.9	197.4	170.6	146.0	123.3	102.5	83.7	66.9	52.0	
F	271.3	241.0	212.6	186.0	161.0	137.9	116.7	97.3	79.6	63.8	49.8	
G	266.7	236.6	208.2	181.7	156.9	134.0	113.1	93.8	76.4	60.9	47.2	
H	212.7	187.9	164.6	142.9	122.6	104.0	87.1	71.5	57.6	45.2	34.4	
I	127.8	112.9	99.0	86.0	73.8	62.6	52.4	43.1	34.7	27.2	20.8	
J	111.4	97.5	84.6	72.6	61.4	51.2	42.0	33.7	26.4	20.0	14.5	
K	88.1	76.4	65.6	55.7	46.4	38.1	30.7	24.1	18.3	13.3	9.2	
L	87.6	76.2	65.7	56.0	46.9	38.8	31.4	24.9	19.0	14.1	9.9	
M	64.6	55.2	46.5	38.6	31.4	24.9	19.3	14.4	10.2	6.8	4.1	
N	38.4	32.2	26.5	21.4	16.7	12.7	9.3	6.4	4.1	2.3	1.0	
O	15.6	12.8	10.3	8.1	6.1	4.4	3.1	1.9	1.1	0.5	0.1	

[0090]

[Table 11]

Table 11 Calculation of $0.5 \times \text{fluid density} \times \text{prominent peak projected area} \times \text{squared flow velocity}$ V (Example 5)

	$0.5 \times \rho \times A \times V^2$ $A = (Ra - \delta s)/RSm$														
	(rpm)	1000	950	900	850	800	750	700	650	600	550	500			
(m/sec)		8.4	8.0	7.5	7.1	6.7	6.3	5.9	5.4	5.0	4.6	4.2			
A		590.1	522.8	459.6	400.5	344.9	293.7	246.8	203.5	164.3	129.4	98.4			
B		285.3	255.8	227.8	201.5	176.8	153.7	132.3	112.4	94.2	77.6	62.6			
C		225.6	200.6	177.0	155.0	134.2	115.0	97.4	81.1	66.3	53.0	41.2			
D		216.3	192.4	169.9	148.7	128.9	110.5	93.6	78.0	63.8	51.1	39.7			
E		206.4	183.1	161.2	140.6	121.4	103.6	87.2	72.2	58.5	46.3	35.5			
F		195.8	173.8	153.3	134.0	115.8	99.1	83.7	69.5	56.6	45.0	34.8			
G		192.6	170.7	150.1	130.9	112.8	96.1	80.8	66.7	53.9	42.5	32.4			
H		147.8	130.2	113.7	98.3	83.9	70.6	58.6	47.5	37.5	28.7	20.9			
I		92.8	81.9	71.6	62.0	53.0	44.8	37.2	30.3	24.0	18.5	13.6			
J		78.1	67.9	58.5	49.7	41.4	33.9	27.1	21.0	15.5	10.7	6.6			
K		60.1	51.5	43.5	36.1	29.2	23.0	17.5	12.4	8.0	4.3	1.1			
L		60.8	52.4	44.6	37.4	30.6	24.5	19.0	14.1	9.7	5.9	2.7			
M		43.5	36.1	29.3	23.0	17.2	12.1	7.5	3.5	0.0	-2.8	-5.1			
N		23.0	17.8	13.2	8.9	5.0	1.6	-1.3	-3.9	-6.0	-7.6	-8.8			
O		7.6	5.3	3.2	1.3	-0.5	-2.0	-3.2	-4.3	-5.2	-5.8	-6.2			

CLAIMS

[Claim 1]

A method for predicting the frictional resistance of a rough surface having a variation in roughness wavelength , and
5 being in contact with a fluid flowing at varied velocities, the method comprising evaluating the total projected area A of all prominent peaks standing out above the viscous sublayer thickness per unit area (hereinafter, written as the "prominent peak projected area A"), and calculating the friction increase
10 ratio FIR (%) using Equation (1) below or the frictional resistance increase $\Delta\tau$ using Equation (2) below:

[Math. 1]

$$\text{FIR}(\%) = C \times A \quad (1)$$

[Math. 2]

$$\Delta \tau = C_r \frac{1}{2} \rho A V^2 \quad (2)$$

15

(in Equation (1), the coefficient C is a constant dependent on the prominent peak projected area A and is determined by performing a frictional resistance test beforehand in which the frictional resistance is measured with
20 respect to a plurality of rough surfaces differing in the degree

of roughness while changing the flow velocity V and the friction increase ratios FIR (%) of the rough surfaces are calculated wherein the friction increase ratios FIR (%) are percentages of the difference $\tau_r - \tau_0$ between the frictional resistance τ_r of the rough surface and the frictional resistance τ_0 of a smooth surface, divided by τ_0 , and

in Equation (2), the coefficient C_r is a constant dependent on the fluid density ρ , the prominent peak projected area A and the flow velocity V and is determined from the relation of Equation (2) using values of frictional resistance increase $\Delta\tau$ obtained beforehand by a frictional resistance test in which the frictional resistance is measured with respect to a plurality of rough surfaces differing in the degree of roughness while changing the flow velocity V wherein the values of frictional resistance increase $\Delta\tau$ are the differences $\tau_r - \tau_0$ between the frictional resistance τ_r of the rough surface and the frictional resistance τ_0 of a smooth surface).

[Claim 2]

The method for predicting the frictional resistance of a rough surface according to Claim 1, wherein the prominent peak projected area A is a value calculated using Equation (3) wherein R_{Sm} is the average length of roughness curve elements and R_c is the average height of roughness curve elements both measured as roughness wavelength λ and roughness height R ,

respectively, in accordance with the regulations in ISO 4287:
1997 (JIS B 0601: 2001),

[Math. 3]

$$A = 0.5 \times \frac{(Rc - \delta s)^2}{Rc \times RSm} \quad (3)$$

5 (in Equation (3), δs is the viscous sublayer thickness
measured by a frictional resistance test with respect to a
smooth surface and is a value determined in such a manner that
the frictional velocity u^* is calculated from the frictional
resistance τ_0 of the smooth surface using Equation (5), δs is
10 calculated from Equation (4) using the dimensionless distance
 y^+ ($2 < y^+ < 8$) and the kinematic viscosity coefficient ν of
a fluid of interest, and, from among the values of δs thus
calculated, the value which provides a high coefficient of
correlation between FIR (%) or $\Delta\tau$ and the prominent peak
15 projected area A in Equation (1) or (2) is selected)

[Math. 4]

$$\delta s = \frac{y^+ \times \nu}{u^*} \quad (4)$$

[Math. 5]

$$u^* = \sqrt{\frac{\tau_0}{\rho}} \quad (5)$$

[Claim 3]

The method for predicting the frictional resistance of a rough surface according to Claim 2, wherein the prominent peak projected area A is calculated in such a manner that Rz, Rzjis, Rq or Ra is measured as roughness height R', the slope a of the Rc to the measured parameter is determined and the parameter is converted assuming that $R_c = a \times R'$ (R' is Rz, Rzjis, Rq or Ra).

[Claim 4]

The method for predicting the frictional resistance of a rough surface according to Claim 1, wherein the prominent peak projected area A is a value calculated using Equation (6) wherein RSm is the average length of roughness curve elements and Ra is the arithmetic average roughness both measured as roughness wavelength λ and roughness height R, respectively, in accordance with the regulations in ISO 4287: 1997 (JIS B 0601: 2001)

[Math. 6]

$$A = \frac{(Ra - \delta s)}{RSm} \quad (6)$$

(in Equation (6), δs used to calculate the prominent peak projected area A is the viscous sublayer thickness measured by a frictional resistance test with respect to a smooth surface and is a value determined in such a manner that the frictional

velocity u^* is calculated from the frictional resistance τ_0 of the smooth surface using Equation (5), δ_s is calculated from Equation (4) using the dimensionless distance y^+ ($2 < y^+ < 8$) and the kinematic viscosity coefficient ν of a fluid of interest, and, from among the values of δ_s thus calculated, the value which provides a high coefficient of correlation between FIR (%) and the prominent peak projected area A in Equation (1) is selected).

[Claim 5]

10 The method for predicting the frictional resistance of a rough surface according to Claim 4, wherein the prominent peak projected area A is calculated in such a manner that R_z , R_{zj} , R_q or R_c is measured as roughness height R'' , the slope a of the R_a to the measured parameter is determined and the
15 parameter is converted assuming that $R_a = a \times R''$ (R'' is R_z , R_{zj} , R_q or R_c).

[Claim 6]

A surface performance estimating apparatus for estimating surface performance by the prediction method described in any
20 of Claims 1 to 5, the apparatus comprising:

a measurement section configured to measure the roughness height R and the average length R_{Sm} of roughness curve elements, and

a calculation unit configured to calculate the prominent

peak projected area A and to calculate the friction increase ratio FIR (%) using Equation (1) or the frictional resistance increase $\Delta\tau$ using Equation (2).

[Claim 7]

5 The surface performance estimating apparatus according to Claim 6, wherein the measurement section is configured to measure the roughness height R and the average length R_{Sm} of roughness curve elements of three dimensional roughness while reading the travel distance with a rotary encoder or a linear
10 encoder and reading the displacement with a laser displacement meter using a two-dimensional beam.

[Claim 8]

 The surface performance estimating apparatus according to Claim 6 or 7, wherein the roughness height R measured is
15 R_z , R_{zjis} , R_q , R_c or R_a .